FINAL REPORT

## ASHRAE RESEARCH PROJECT 438-RP

# EVALUATION OF THE TECHNIQUES <br> FOR THE MEASUREMENT OF <br> AIR LEAKAGE OF BUILDING COMPONENTS 

Submitted by:
Dr. Donald G. Colliver, P.E.
Dr. William E. Murphy, P.E. Wei Sun

Agricultural Engineering Department University of Kentucky
Lexington, KY 40546-0276
December 30, 1992

## TABLE OF CONTENTS

Introduction ..... 1-1
Research Objectives ..... 2-1
Background ..... 3-1
Definition of Terms ..... 3-1
Methods of Measuring Air Tightness/Leakage ..... 3-2
Methods of Measuring Air Infiltration ..... 3-5
Methods of Reporting Air Leakage/Infiltration ..... 3-6
Methods of Determining Building Component Air Leakage ..... 3-11
Methods ..... 4-1
Literature Investigation ..... 4-1
Development of Database ..... 4-1
Transformation to Common Base Pressure ..... 4-7
Theoretical/Experimential Investigation ..... 4-10
Theoretical Analysis - Single Openings ..... 4-10
Theoretical Analysis - Multiple Openings ..... 4-16
Experimential Procedure ..... 4-20
Experimential Design ..... 4-27
Experimential Data Analysis ..... 4-31
Results/Discussion/Conclusions ..... 5-1
Literature Database ..... 5-1
Determination of Independent $\mathrm{C}_{1}, \mathrm{C}_{2}$ and $\mathrm{C}_{3}$ Parameters ..... 5-5
Fit of New Equation ..... 5-10
ELA and $\mathrm{C}_{\mathrm{d}}$ Analysis ..... 5-33
Summary ..... 6-1
References ..... 7-1BibliographyReference List by AIVC Number Code8-1
Reference List by Author ..... 8-10
Appendices
A - Listing of Literature Leakage Values (Raw Data)
B - Listing of Literature Leakage Values (Transformed Data)
C - Notes to Tables
D - Conversion Table
E - Derivation of Dimensionless Crack flow Equation
F - Definition and Combination of Crack/Position Coding
G - Data Tables for Individual Openings
H - Data Tables for Openings in Parallel
I - Data Tables for Openings in Series
J - Data Tables for Component Leakage Tests

## INTRODUCTION

Knowledge of the amount of air leaking into a building through the various building components is important for a wide variety of reasons. Initially the interest in these values was so that estimates could be made on the amount of energy to be added or removed to heat or cool air that was infiltrating into the structure. Selection of new and replacement building materials was done partially on the amount of energy costs that would be saved by the selection of that component.

More recently however as structures are being built to tighter standards in order to conserve energy, there has been an increasing interest in determining the air flow though building components in order to predict the amount of outside air entering the structure through the building envelope. There is a need to be able to estimate the air leakage in a building envelope in order to estimate the forced ventilation which might be required in the structure. These values are also used by designers as they locate the leakage for ventilation purposes. Rather than sealing the structure airtight and going back and installing openings at the desired locations, if we were able to accurately predict the leakage it would be possible to design the envelope with sufficient (but not excessive) leakage.

As more locations have energy and indoor air quality codes and standards being applied to buildings in their jurisdiction, it will be even more critical to have good estimation methods to predict the amount of leakage in a structure. This will be important not only for the code enforcement official who might be applying a performance based code but also for designers and builders attempting to meet performance requirements. It is even more important for those developing prescriptive codes and standards since they must know a-priori that their specifications will meet or exceed the desired leakage recommendations.

The establishment of the appropriate air leakage value to use for the various components is not an easy task. There has been much discussion about testing techniques, values to be used and the accuracy of the data. The problem is confounded even further when it is considered that many of the air leakage components or sites are manufactured on the construction site and not on an assembly line where quality control can be maintained.

Therefore the goal of this research was to evaluate the existing data of component leakage and determine an appropriate technique to enable the estimation of potential rates of air leakage through various building components.

## RESEARCH OBJECTIVES

The objectives of this research were:

1. Compile and catalog from the recent available literature, the available data on leakage areas of building components commonly used in North America in residential construction,
2. Assess the adequacy of the component ELA concept and develop and test alternates which would be based on the fundamental principles governing flow through openings,
3. Evaluate the different methods of reporting the air leakage, and
4. Recommend a system of reporting air leakage for components.

## CHAPTER 3 - BACKGROUND

The concept of a term which can be used to describe the flow of air into a structure is an commendable one. A method is needed to estimate the leakage of a structure before it is built. The development of a term to accurately predict the leakage of the structure over varying weather conditions would also be helpful so that evaluations such as the blower door test would not have to be run on the structure.

### 3.1 Definition of Airleakage, Airtightness, Air Infiltration

Several key terms which oftentimes are used interchangeably in the building industry need to be discussed and differentiated between in order to understand the research being reported.

Airleakage refers to the movement of air across or through the building envelope due to some differential pressure. In this report the term airleakage will be referring to the case in which testing is being done where the differential pressure is artificially imposed at levels above those found due to weather conditions and typical operation. When testing for airleakage, this pressure is artificially imposed by some device such as a fan and the measured flow required to maintain that pressure is considered to be the airleakage.

Airtightness refers to the ability of the envelope of the structure to resist the flow of air through it. The more airtight a structure, the higher the pressure must be to maintain flow of air through it. Airtightness can be thought of as the resistance of an enclosure to allow air to cross its boundaries. It is the inverse of airleakage. Typical units of airtightness are volumetric flow rate per unit of surface at some stated constant differential pressure.

Air infiltration refers to the naturally occurring flow of air across the building envelope due to differential pressures naturally occurring during the operation of the building such as weather effects (stack and wind), occupant effects (opening and closing doors) and equipment effects (vented combustion equipment, vents, etc). Air infiltration varies depending upon the response of the building to these effects. It is not constant over time and therefore the value must be time-averaged or the rate stated for some given condition such as the differential pressure. Typically air infiltration will be quoted in units of volume per unit time.

In order to completely evaluate the leakage performance of a building or its components it is necessary to eliminate the other variables which influence
infiltration and evaluate the air leakage characteristics of the building envelope and its components only.

### 3.2 Methods of Measuring Air Tightness/Leakage

The measurement of either the building leakage or air tightness is done to describe the building envelope without weather, equipment or operator influence. Since one is basically the inverse of the other, only one of the parameters needs to be measured. Two major approaches have been used to determine the airtightness; DC pressurization and AC pressurization. DC pressurization is the predominate technique used.

### 3.2.1 DC Pressurization

DC pressurization has been used for many years, studied considerably and there are several commercially available units available (ASTM, 1987; CGSB, 1986; Gadsby and Harrje, 1985; and Murphy et al 1991). Commonly called the "biower door" or "fan pressurization deviee (FPD)", it serves as the basis for several national standards and is used by both researchers and field personnel to identify the airflow-pressurization characteristics of buildings and/or locating sources of air flowing through the building skin. The majority of the air leakage data reported in the literature were obtained with this technique. It can be assumed that the data from the literature reported in this work was obtained using this technique unless stated otherwise.

The technique involves placing a powerful variable speed fan in an opening in the building envelope (usually by replacing a door or window) through which air is blown into (pressurization) or out of (depressurization) the building. A uniform, artificial, static pressure is imposed across the entire building envelope and the amount of air being moved by the fan to create this pressure differential is determined. A relationship between the imposed pressure difference and flow rate through the fan may then be determined. The amount of air leakage or building tightness is determined from this relationship.

The air flow rate through the fan is usually determined from: a) measurements of the pressure drop across a known flow restriction, or b) the fan rotational speed and calibration curve. The differential pressure across the building shell is determined from internal and external static pressure taps and a differential pressure transducer.

There are several standards which use the FPD (Canadian Standard CAN/CGSB 149.10 M86, ASTM E 779-87, ASTM E 783-84 and ISO DP 9972). All these standards use the DC pressurization technique, however they differ in salient points such as: pressurization/depressurization, pressure tap location, differential pressure range, limiting weather conditions (wind and temperature differences), expression of results and stated accuracy (Charlesworth, 1988). A summary of comparison between the ASTM 779 and CAN/CGSB 149.10 is

Table 3-1 A Comparison Between Air Tightness Measurement Standards (From Charlesworth, 1988)

| Standard | Pressure Tap Location | Differential Pressure Range | Number of Readings | Preparation of Openings | Results | Equation for <br> Linear Least <br> Squares <br> Regression | Limiting Conditions |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| CAN/CGSB 149.10M86 | Minimum of four taps - located around building connected to an averaging container to dampen flucutations | $15-50 \mathrm{~Pa}-$ depressurization | Every 5 Pa from high to low | Detailed instructions as to position of dampers and sealing of vents | Equivalent Leakage Area at $10 \mathrm{~Pa}, \mathrm{Cd}$ $=0.611$ | Log <br> transformation of flow weighted by flow squared | Wind speed $<5.6 \mathrm{~m} / \mathrm{s}$ |
| ASTM E 779-87 | One tap - location not specified | $12.5-75 \mathrm{~Pa}$ pressurization or depressurization | Every 12.5 Pa | Dampers closed - other openings as normal operation | Effective <br> Leakage area ot $4 \mathrm{~Pa}, \mathrm{Cd}=$ 1.0 | Log transformation no weighting for equal spacing | Wind speed $<2 \mathrm{~m} / \mathrm{s}$ <br> Temperature $5-35^{\circ} \mathrm{C}$ |

given in Table 3.1. There have been no direct comparisons between the various standards so there are no recommendations as to validity of one over the other or standard calibrations for converting the data between them. The 1989
ASHRAE Handbook - Fundamentals gives equations to make conversions between leakage and flows given different discharge coefficients and reference pressures.

It has been assumed that the air leakage of the structure when it is under pressurization (air being blown in) is different than when it is being depressurized (air being blown out of the structure) because many times the two curves do not look the same. It has also been assumed that this nonreversibility is due to some building element (such as the vapor retarder) acting as a flap valve, the asymmetric geometry of some of the cracks or the presence of wind and stack pressures during the measurement. In a study of pressurization/ depressurization measurements on 196 houses, Sherman et al. 1986, found that there were no significant differences (i.e. the differences are within the measurement errors) in either the flow exponent or leakage area or a systematic ditterence between pressurization and depressurization but that significant uncertainty is associated with an individual measurements.

In addition it has been found (Murphy et al. 1991) that there can be substantial differences between different FPDs on the same house with the same operators and data analysis technique. It was found that ordinary use of FPDs by typical operators to determine envelope airtightness levels in existing houses may do little better than $\pm \mathbf{2 5 \%}$ accuracy.

Therefore there are substantial difficulties in attempting to make subtle comparisons between data in the literature which has been collected from different sources using different equipment.

### 3.2.2 AC Pressurization

The AC pressurization technique is another technique which has been used to examine building air leakage (Modera and Sherman, 1985). While the quantity of data collected with this technique is very small, the technique used is substantially different and merits mentioning. It was developed for determining air tightness directly at the small pressure differences typically found in natural infiltration conditions without introducing large flows through the building envelope or introducing atypical pressure differences (and thus atypical flow regimes [turbulent rather than laminar] through the openings).

AC pressurization creates a periodic pressure difference across the building envelope that can be distinguished from the naturally occurring pressure fluctuations. This pressure change with its amplitude and phase cause a corresponding volume change. The flow through the envelope can then be determined from the continuity equation for a compressible medium, provided accurate estimates are provided of the building's capacity, internal pressure and
its derivative.

### 3.3 Methods of Measuring Air Infiltration

There are several methods which have been used to determine the air infiltration characteristics of a structure. All the most common ones use some form of gas tracing. Gas tracing is the addition of a detectable gas into an airstream or air volume for the purpose of tracking the mass movement of air; or, more typically, the determination of the amount of exchange of air contaminated with the tracer with non-contaminated air. The tagging of air with tracers is usually done by inserting the tracer gas or volatile liquid into the air and then quantitatively detecting or tracing its presence over time.

There are three major techniques of determining the air infiltration rates using gas tracers: a) slug injection/exponential decay (concentration decay), b) variable injection to achieve constant concentration and c) constant emission/ injection. The general governing mass balance equation is the same for all three techniques.

The concentration decay is the most straightforward technique of the three. In this method the tracer is released into the space, time is allowed for it to thoroughly mix with all the air volume and then periodically the decay of the concentration is measured (Hunt, 1980). The decay is due to the dilution of the tagged air with incoming fresh air not containing the tracer. The faster the decay, the higher the input rate of fresh air. This technique is commonly used when large numbers of samples or structures need to be tested with minimal equipment setup or on-site maintenance. Grab bag sampling is often used.

The second major technique varies the source generation or injection to achieve constant concentration. A direct-feedback, automated control system is required for this technique. The infiltration rate becomes directly proportional to the tracer gas generation/release rate. This technique can detect short term variations in infiltration rates and can do multizone measurements. It can only be used when the injection is controlled automatically and typically is used only on structures requiring elaborate testing. Examples of this type technique are Princeton's CCTG and Lawrence Berkeley Laboratory's MTMS systems.

In the constant emission technique, the tracer is released at a constant known rate and the concentration is monitored over time. After steady state is reached, the average concentration may be used to determine the average air exchange over the time period the sampler was exposed. Thus this technique corrects for variations of air exchange over time due to fluctuations in weather conditions, operator effects or equipment effects. Multiple tracers which do not interfere with each other may be used for multizone applications. The Brookhaven PFT method is an example of this technique.

The result of all these techniques is an estimation of the amount of
outside air that has infiltrated into the building envelope through openings due to the driving pressure forces which existed during the sampling period.

Thus if we were able to determine a) the driving forces and b) the response of the openings in the building envelope to these driving forces we would be able to estimate the air infiltration over time.

### 3.4 Methods of Reporting Air Leakage/Infiltration

Several different empirical methods of reporting air leakage characteristics have been used previously. This also makes it difficult to make comparisons between different values in the literature. In most fan pressurization measurements; the flow is recorded as a function of the imposed differential pressure for several ( 5 to 10 ) different pressures between the range of 10 to 75 Pa . The way the data are reduced after these five to ten data points are found is where there are significant differences, so it is important to identify them and how they are used.

Reporting in situations where there were several replications was usually done by giving the average value and some measure of dispersion. This is usually the maximum and minimum values or the standard deviation value. It was noticed that in some cases when both the max/min values and the standard deviation values were given that the average minus the standard deviation was less than the reported minimum value. It is assumed that this is due to a log normal distribution of the readings rather than the Gaussian distribution commonly associated with the standard deviation term.

### 3.4.1 Flow Coefficient and Flow Exponent (C and n, dimensionless)

Empirically it has been found that the pressure vs flow data follow a power law relationship. Gabrielesson et al. (1968) proposed the expression:

$$
Q=C A(\Delta P)^{n}
$$

where:
$\mathbf{Q}=$ volumetric flow rate, $\mathrm{m}^{3} / \mathrm{h}$
$C=$ crack flow coefficient, $\mathrm{m}^{3} / \mathrm{h}(\mathrm{Pa})^{\text {n }}$
$A=$ crack section flow area, $\mathrm{m}^{3}$
$\Delta \mathbf{P}=$ pressure drop across the opening, Pa
$\mathrm{n}=$ flow exponent, dimensionless
Shaw (1974) presented another equation on the basis of mass flow rate:

$$
F=K(\Delta P)^{n}
$$

where:

$$
\begin{aligned}
& \mathrm{F}=\text { mass flow rate, } \mathrm{kg} / \mathrm{h} \\
& \mathrm{~K}=\text { constant }, \mathrm{kg} /\left(\mathrm{hr} \bullet \mathrm{~Pa}^{n}\right)
\end{aligned}
$$

Warren (1978) considered that the flow length might make a difference and introduced length into the formula:

$$
\Delta P=[Q / K L]^{1 / n}
$$

where:
$K=$ constant, $\mathrm{m}^{2}(\mathrm{~Pa})^{\mathrm{n}} / \mathrm{h}$
$L=$ crack length, $m$.
The most common form of equation to describe air leakage characteristics is the "power law" equation (Irving 1979, Sherman 1980, ASHRAE, 1989):

$$
\mathrm{Q}=\mathrm{C}(\Delta \mathrm{P})^{\mathrm{n}}
$$

where:
$\mathrm{Q}=$ Air flow, $\mathrm{m}^{3} / \mathrm{s}$
$\Delta P=$ Pressure differential, Pa
$C=$ Flow coefficient, ( $\mathrm{m}^{3} / \mathrm{s}$ at 1 Pa )
$\mathrm{n}=$ Flow exponent, dimensionless
The most common way of determining coefficients is to do a log transformation on the data:

$$
\ln Q=\ln C+n * \ln (\Delta P) 3.4 .5
$$

and then do a least squares regression on the linear transformed data to determine the slope ( $n$ ) and the intercept ( $C$ ) of the line.

There are no fundamental fluid flow principles for this relationship. We expect that the exponent should lie between 0.5 (approximating orifice flow) and 1.0 (approximating fully developed laminar flow). There is no physical interpretation beyond this explanation. It is commonly assumed that the variations between these two values account for the physical changes which occur between fully developed turbulent flow and fully developed laminar flow. This includes such things as the development of laminar flow and its effect by the entrance/exit losses, the developing length and other minor loss parameters such as bends, area changes, etc.

Etheridge (1977) and Chastain et al. (1987) derived a semi-empirical dimensionless flow equation based on the distribution of total pressure drop in the loss of fully-developed flow in constant area opening and the loss of developing section, inlet-outlet friction, area change and bend effects:

$$
\frac{\Delta \mathrm{P}}{\frac{1}{2} \rho \overline{\mathrm{~V}}^{2}}=\frac{\mathrm{B}}{\operatorname{Re}} \frac{\mathrm{Z}}{\mathrm{D}_{\mathrm{h}}}+\mathrm{K}
$$

The derivation of this dimensionless crack flow equation and further discussion of this analysis is done in Section 4.2.1.

### 3.4.2 Air Flow Rate at $50 \mathrm{~Pa}\left(\mathrm{O}_{50}, \mathrm{~m}^{3}\right)$

Several countries have adopted an air flow rate with 50 Pa imposed differential pressure as a standard when classifying buildings in terms of airtightness (Charlesworth, 1988). This single point reference number can be easily obtained. It is obtained by simply pressurizing or depressurizing the structure to 50 Pa and determining the fan flow rate required to achieve this pressurization (depressurization) or by substitution into Eqn 3.4.1.

### 3.4.3 Air Change Rate at $50 \mathrm{~Pa}\left(\mathbf{N}_{50}, \mathrm{ACH}\right.$ or $\left.\mathbf{h}^{-1}\right)$

This term is also commonly used as a single point reference. It is found from the $\mathrm{Q}_{50}$ value described above divided by the building volume, $\mathrm{V}\left(\mathrm{m}^{3}\right)$.

$$
N_{50}=\quad Q_{50} / V
$$

(ACH or $\mathrm{h}^{-1}$ )
The major difficulty in determining this number is estimating the applicable building volume. Questions arise about inclusion of closet volume, interior walls, cabinets, etc.

### 3.4.4 Effective Leakage Area (ELA $\mathbf{4}_{4}$ or EfLA, $\mathrm{cm}^{2}$ or $\mathrm{m}^{2}$ )

Another popular measure of leakage introduced by Sherman and Grimsrud (1980) is the building Effective Leakage Area. They identified that the behavior of the actual leakage curve closely resembles that expected for turbulent flow and could be modeled by the classical flow equation for a sharpedge orifice if the discharge coefficient is defined to be unity. (See Appendix E for a theoretical derivation from first principles.) Thus they assumed that the flow was proportional to the square-root of the applied pressure:

$$
Q_{\text {ref }}=A C_{d}\left(2 \Delta P_{\text {ref }} / \rho\right)^{0.5}
$$

where:
$\mathrm{Q}_{\text {rof }}=$ flow at the reference pressure, $\left(\mathrm{m}^{3} / \mathrm{s}\right)$
$A=$ effective leakage area ( $\mathrm{m}^{2}$ )
$C_{d}=$ Discharge coefficient (1.0)
$\rho=$ density of the air, $\left(1.22 \mathrm{~kg} / \mathrm{m}^{3}\right)$

$$
\Delta P_{\text {rof }}=\text { the applied reference pressure, (4 Pa) }
$$

Thus a term which lumps the area and the orifice discharge coefficient together representing the effective area of an orifice (with $C_{d}=1.0$ ) to produce the same amount of flow at a reference can be described as:

$$
\begin{align*}
& E L A_{4} \triangleq A C_{d}=A \\
& E L A_{4}=10,000 Q_{\text {rof }}\left(\rho / 2 \Delta P_{\text {ref }}\right)^{0.5}
\end{align*}
$$

where:
$E L A_{4}=$ effective leakage area, $\left(\mathrm{cm}^{2}\right)$.
The ELA does not bear any simple relationship to physical opening areas in the building but instead represents the summation of the overall effect of all the openings. One thing to note is that it is the opening area of an effective orifice with a discharge coefficient of one. It should be stressed that the effective leakage area is not the actual leakage area and it should not be confused.

The ELA depends upon which pressure is used to calculate its value. An applied reference pressure of 4 Pa is used to calculate the ELA . This value is $^{\text {. }}$ commonly used in the USA.

### 3.4.5 Equivalent Leakage Area (ELA $\mathbf{1 0}_{0}$ or EqLA, $\mathrm{cm}^{2}$ or $\mathbf{m}^{2}$ )

Another commonly used term (and often confused with the Effective Leakage Area term) is the Equivalent Leakage Area. This also comes from the theoretically derived orifice equation. The derivation of the effective leakage area follows the derivation of the orifice equation with the exception of the assumption of a orifice discharge coefficient of 1.0 is relaxed and replaced with a value of 0.611 (a value found to be representative of the types of openings being described):

$$
\begin{align*}
& E L A_{10} \triangleq A C_{d}=A \bullet 0.611 \\
& E L A_{10}=10,000\left(Q_{\text {rof }} / C_{d}\right)\left(\rho / 2^{*} \Delta P_{\text {rof }}\right)^{0.5}
\end{align*}
$$

where:
$E L A_{10}=$ equivalent leakage area, $\left(\mathrm{cm}^{2}\right)$
$\mathrm{O}_{\text {rof }}=$ flowrate at the reference pressure difference, ( $\mathrm{m}^{3} / \mathrm{s}$ )
$C_{d}=$ discharge coefficient, 0.611 (dimensionless)
$\Delta P_{\text {ref }}=$ reference pressure difference, (10 Pa).
The equivalent leakage area is used by several countries in their standards (eg. Canada and the Netherlands). The potential exists for errors to occur by interchanging the ELA $A_{4}$ and ELA ${ }_{10}$ values. In this report the $\mathrm{C}_{\mathrm{d}}$ value
will be listed in the data tables to avoid confusion. It will be assumed that reported ELA values from Canada and the Netherlands will be equivalent leakage areas unless otherwise noted.

### 3.4.6 Specific Leakage Area (SLA ${ }_{4}$ or SLA $_{10}, \mathrm{~cm}^{2} / \mathrm{m}^{2}$ )

The effective or equivalent leakage area is representative of the total leakage of all the envelope of the building. It is possible that two buildings could have the same leakage yet differ in envelope area. Thus the development of a "normalizing" term which would take into account the size of the building. The specific leakage area is either the ELA ${ }_{4}$ or the ELA ${ }_{10}$ divided by the floor area of the building:

$$
\mathrm{SLA}_{4 \text { or } 10}=\mathrm{ELA}_{4 \text { or } 10} / A_{f}
$$

where:
$A_{f}=$ floor area, $\left(\mathrm{m}^{2}\right)$

The specific leakage area enables the comparison of leakage between buildings. The floor area is chosen as the normalizing term because it is easily obtainable since it is the most distinguishable number to be recalled by the resident. It should be noted that the value used for floor area can be a source of error. Care must be used in determining if this represents net area (outside dimensions less any area for exterior and interior walls, closets, etc.) or it represents the gross floor area (the outside dimensions).

### 3.4.7 Normalized Leakage Area ( $\mathrm{NLA}_{4}$ or $\mathrm{NLA}_{10}, \mathrm{~cm}^{2} / \mathrm{m}^{2}$ )

A building's leakage areas are in the building envelope which consists of more than just the floor area. It is possible to have structures to have the same floor area yet differ widely in exposed surface areas. The normalized leakage area term was developed to take into consideration the area of the building envelope which may be exposed to the pressure differentials which drive airflow through the skin of the building. This is all the exposed surfaces above the grade line and includes the walls, ceilings, and floor above grade (but not the floor on a slab in direct contact with the soil). The CGSB standard uses all the envelope area except the basement floor to normalize. Thus the normalized leakage area was defined as:

$$
\mathrm{NLA}_{4 \text { or } 10}=\mathrm{ELA}_{4 \text { or } 10} / \mathrm{A}_{0}
$$

where:

$$
A_{0}=\text { exposed envelope surface area, }\left(m^{2}\right) .
$$

It is assumed that the normalized leakage area is considered to be the most comprehensive, and best representative number of the leakage area for
comparison purposes on surfaces.

### 3.4.8 Leakage per Unit Length ( $\mathrm{Q} / \mathrm{L}, \mathrm{I} / \mathrm{s}-\mathrm{m}$ )

The leakage flow rate per unit length of crack has commonly been given when well defined, easily measured openings are present (eg. window sash or door seal length). It is known in these circumstances that the amount of flow is proportional to the length of the crack - not to the surface area. Thus the leakage is expressed as the flow rate per unit length of opening:

$$
\mathrm{Q}_{\mathrm{ref}}=\mathrm{k}_{\mathrm{ref}}
$$

where:
$\mathrm{O}_{\text {rof }}=$ the flow rate per unit length at the reference pressure, ( $1 / \mathrm{s}-\mathrm{m}$ )
ref $=$ reference pressure for the flow determination, $(\mathrm{Pa})$.
$k_{\text {rof }}=$ constant at the reference pressure

### 3.4.9 Percentage of Total Leakage (\%)

There are several cases in the literature when the authors reported only the flow through a particular opening relative to the total flow for the entire structure (ie. \% of flow). There was not sufficient information to determine the flow rate through the particular opening. This happened most often when the primary interest was in locating and sealing the leaks in the building and not on quantifying the volumetric flow rate.

### 3.5 Methods of Determining Building Component Air Leakage

The leakage characteristics previously discussed were primarily developed to report the air leakage of entire buildings as determined by pressurizing and/or depressurizing the entire building and analyzing the resulting flow-differential pressure data. The ASTM Standard E779-87 and Canadian CGSB Standard 149.10-M86 standards are commonly applied to whole house testing in North America. The purpose of these two standards is to establish a uniform technique to determine the leakage rates through a building envelope under controlled fan pressurization or depressurization. The leakage characteristics of individual building components can also be determined from on-site measurements (ASTM E1186-87 and others). These techniques will be discussed individually. Charlesworth (1988) should be consulted for more specific in-depth details.

The desire is to determine the leakage characteristics of the building component insitu in order to be able to accurately predict the leakage of individual parts of the building so tradeoffs might be made.

### 3.5.1 Sealing the Component with a Chamber

This is the simplest form of direct measurement of the component leakage. It consists of putting a chamber over the interior face of the building element; supplying air (pressurization) or pulling air (depressurization) out of the chamber at a rate required to maintain a pressure difference; and then measuring the flow rate required to maintain this pressure. The analysis and data reporting may be done using any of the methods in Section 3.4.

This method can be made more accurate by balancing the pressure in the room containing the chamber with the pressure in the chamber. This is used to assure that the leakage flow being measured is that flowing through the component in question rather than around or through the chamber.

Another technique used to compensate for the chamber flow resistance is the compensating flow rate meter (Phaff, 1987). This measuring device has a resistance compensating device (an integral fan) to make up for any resistance that the chamber placed over the component might add to the flow path. When this device is correctly adjusted the device does not influence the airflow and the rate through the component sealed by the chamber may be directly obtained.

### 3.5.2 Balanced Fan Pressurization

This technique is used primarily in situations where the component can not be isolated or sealed and it is known that typically the component does not have a pressure differential across it (Shaw, 1980). A prime example is the party wall between two townhouses. The party wall can not be sealed to prevent the air leaking through it when one of the townhouses is pressurized. Thus an erroneously high reading of leakage would be obtained because during normal operations there would not be a significant differential pressure across the party wall. However the pressurization device creates a uniform static pressure in the structure which is "seen" by all surfaces (including the party wall). The balanced fan pressurization technique provides for a compensating pressurization device to be located on the opposite side of the building component not to be included in the test (eg. party wall). The same pressure would be applied to both sides of the component so that there would not be a driving force creating air flow through the component. Thus there would not be additional air flow required of the testing fan (and the component flow measuring device) to blow through the party wall.

### 3.5.3 Selective Progressive Sealing

Selective progressive sealing is an indirect determination of the air leakage through a building component which has been sealed with an impermeable cover. This technique assumes that all the air flowing through a component can be stopped by sealing and the resulting reduction or subtraction
in total air flow in the building can be attributed to stopping the flow penetrating the component which was sealed. The subtraction of the two tests then quantifies the air leakage through the component sealed.

This technique has been commonly used to identify and quantify large leakage sites. In most situations this has involved retrofit applications to quantify the effects of various retrofit options. However a thorough analysis of the errors involved has not been completed. A potential problem has been noted in that usually the accuracy errors involved in readings of the total airflow might make the errors in the differential readings quite large compared to the actual values of the readings. Another item often noted is that there appears to be some hysterisis in the sealing order. The reduction in building leakage does not match the increase in building leakage if the components are unsealed in a different order. This might indicate that the sealing is not independent of what other components are sealed. This could possibly indicate that there is some communication of air between the components being sealed.

### 3.5.4 Controlled Laboratory Conditions

Building component measurements can also be made under the controlled conditions of a laboratory. A number of standards exist which specify how these measurements are to be made. Usually the test specimens are placed in a test chamber where the airflow and pressures can be carefully monitored. The airflow to and through the specimen and the pressure differentials across the specimen can be accurately monitored and controlled without the influences of wind, stack or occupant induced pressures, drastic humidity changes and equipment calibration errors due to transportation. In addition, replications can be made under similar conditions to get a better understanding of the systematic errors and biases.

It has been noted that laboratory based measurements have produced significantly different results from site measurements of similar components (Charlesworth 1988, Weidt et al. 1979). There may be several factors which contribute to these differences such as installation differences, weathering, workmanship, etc.. However it should be noted that the instruments used in the laboratory typically have much higher accuracies with lower error bands, greater access to calibration standards and there are no effects of climate which may indicate that the field values have more instrument and systematic errors than anticipated.

## CHAPTER 4 - METHODS

In order to accomplish the objectives previously stated, the problem was broken down into two major parts: the literature investigation and analysis, and the theoretical and experimental development of a more well defined term(s) to accurately describe the air leakage characteristics of the building component openings.

### 4.1 LITERATURE INVESTIGATION

### 4.1.1 Development of Database

A through examination of library records and discussions with internationally recognized air infiltration researchers revealed that the most comprehensive source of air infiltration related literature in the world was held by the Air Infiltration and Ventilation Centre (AIVC) located in Coventry, Great Britain. This centre is Annex V of, and is supported by, the International Energy Agency, Energy Conservation in Buildings and Community Systems Programme. Its purpose is to "provide technical support to those engaged in the study and prediction of air ieakage and the consequential losses of energy in buildings. The aim is to promote the understanding of the complex air infiltration processes and to advance the effective application of energy saving measures in both the design of new buildings and the improvement of existing building stock." (General cover statement on their documents.)

The AIVC's library has extensive documentation from the IEA participating countries on many items relating to air infiltration. An electronic database (AIRBASE) covering their extensive library holdings has been developed (Limb, 1989). Each source previously identified was found to also be in this database. A copy of this database was obtained and installed on a PC in our department.

A general search of this database was undertaken using an extensive keyword search including broad topic, narrow topic and related topic terms contained in the AIRBASE thesaurus. This identified approximately 3500 references. Each abstract was then read and evaluated for its potential use as a source of data for this study. Those articles which were not originally in English or did not have an English translation were not investigated further and were dropped from the list. The source list was narrowed to approximately 425 references which included mainly journal articles, books, and technical research reports.

Each of these references was then subjectively rated into five groups on its potential source of useful data. The groupings were based on indications in the abstracts that the paper included information on:

1. Component leakage data
2. Whole buildings leakage or pressurization data
3. Air infiltration data
4. IAQ, moisture or heat transfer data

## 5. Non-North American or pre-1970 data

Attempts were then made to obtain copies of the papers or in the case of the research reports at least the sections which might have available useful data. Approximately 98\% of the articles in the first two groupings were obtained. Each paper obtained was then scanned to see if it contained potential data or additional reference sources. Additional references located in this manner were then included in the active search list and processed like the others.

With the concurrence of the Project Monitoring SubCommittee, it was decided that the scope of the data should be limited to data obtained after 1970, structures in North America, and not include data that was questionable or obtained from public press type articles. These restrictions significantly reduced the amount of information that was available. There were several articles published in the 1920's and 1930's which presented air leakage test values of building components. However the construction appeared to be significantly different from what is common today. The restriction of limiting the data to construction of North America was a more limiting constraint however. There has been much more research in Europe than in North America on measuring the flow of air through specific building components. It is believed however that this restriction is justified due to the differences in construction techniques and materials.

The remaining approximately 175 papers were then read, analyzed and data extracted. Data was pulled from the papers as they were read and put into a database format.

A list of the references included in the database are included in the Bibliography. Since AIRBASE was the primary source of information, the AIRBASE source identification number was also used in this work as the reference number. Part I of the Bibliography is given in numerical order based on the reference code. Part II is given in the traditional reference notation, alphabetized by author.

A major task of the literature search was sorting and grouping the leakage values obtained. The data collected was initially categorized by the source of leakage (ie. component type) based upon the grouping of leakage areas found in Table 3, Chapter 23 of the 1989 ASHRAE Handbook - Fundamentals.

As the database grew the classification evolved to that presented in Table 4-1.

Table 4-1. Components Used to Classify Leakage

| CG | Ceiling - General |
| :--- | :--- |
| CG | Ceiling - Drop |
| CH | Chimney |
| CP | Ceiling Penetrations - Whole House Fans |
| CP | Ceiling Penetrations - Recessed Lights |
| CP | Ceiling Penetrations - Ceiling/Flue Vent |
| CP | Ceiling Penetrations - Surface Mounted Lights |
| CS | Crawl Space |
| CS | Crawl Space - 8x16" Vents |
| DAC | Doors - Attic/Crawl Space |
| DAFD | Doors - Attic Fold Down |
| DAG | Doors - Attic from Garage |
| DD | Doors - Double |
| DE | Doors - Elevator (passenger) |
| DFRAME | Door Frame - Generai |
| DFRAME | Door Frame - Masonry |
| DFRAME | Door Frame - Wood |
| DFRAME | Door Frame - Trim |
| DFRAME | Door Frame - Jamb |
| DFRAME | Door Frame - Threshold |
| DG | Doors - General |
| DIP | Doors - Interior Pocket |
| DIS | Doors - Interior Stairs |
| DMS | Door Mail Slot |
| DSP | Doors - Sliding Exterior Glass Patio |
| DSTW | Doors - Storm (difference with/without) |
| DS | Doors - Single |
| DV | Doors - Vestibule |
| EOS | Electrical Outlets/Switches |
| F | Furnace - Sealed or no combustion |
| F | Furnace - Retention Head or Stack Damper |
| F | Furnace - Retention Head and Stack Damper |
| FLCS | Floors over Crawl Spaces |
| FWDOC | Fireplace W Damper Open/Closed |
| FWG | Fireplace with Glass Doors |
| FWIDOC | Fireplace with Insert \& Damper Open/Closed |
| GWH | Gas Water Heater |
| J | Joints (general) |
| JCW | Joints - Ceiling-Wall |
| JSP | Joints - Sole Plate |
| JTP | Joints - Top Plate |
| PPWP | Piping/Plumbing Wiring Penetrations |
| V | Vents |
|  |  |

Table 4-1. Components Used to Classify Leakage (Continued)

| VBWDO | Vents - Bathroom With Damper Closed/Open |
| :--- | :--- |
| VDWOD | Vents - Dryer With (O)ut Damper |
| VKWDO | Vents - Kitchen With Damper Closed/Open |
| VKWDO | Vents - Kitchen With Tight Gasket |
| WAEX | Wall Exterior |
|  | Cast-in-place Concrete |
|  | Clay Brick Cavity Wall - Finished |
|  | Continuous Air Infiltration Barrier |
|  | LW Concrete Block - unfinished/finished |
|  | HW Concrete Block - unfinished |
|  | Precast Concrete Panel |
|  | Rigid Sheathing |
| WIA | Window - Awning |
| WICA | Window - Casement |
| WIDH | Windows - Double Hung |
|  | with/without storm |
| WIDS | Windows - Double Horizontal Sliders |
| WIFM | Windows - Framing Masonry |
| WIFW | Windows - Framing Wood |
| WIJ | Windows - Jalousie |
| WIL | Windows - Lumped |
| WISHS | Windows - Single Horizontal Slider |
| WISH | Windows - Single Hung |
| WISILL | Windows - Sill |
| WIST | Windows - Storm |

After going through several papers it was realized that some structure to the information obtained would have to be developed. Data was being reported using several different methods of leakage indication (see Section 3.4). The key methods of reporting air leakage/infiltration were investigated to attempt to structure the information. The parameters to be obtained from the papers were those most commonly given.

It was decided to record the information in Table 4-2 about each test reported (one test per line) if the information was given in the paper. Very few cases provided sufficient information to complete all the fields of a single line. Often there would be multiple lines of data from a single reference due to the reporting of the details of individual tests. For example, if a paper contained the average ELA ${ }_{4}$ for 6 windows, the average and range (if given) would be reported as a single entry. If however the values for each of the six windows were given, there would be six lines of information. The exception to this general operational rule was when whole house information was being reported. Although initially detailed information was recorded ün tile inuliviutuà hưuses, this väas uisconitinued in favor of retaining the grouped or averaged data due to the magnitude of the records involved and the inability to obtain sufficient house descriptive information to derive the leakage of individual components.

The data were entered into a Quattro Pro 3.0 spreadsheet running on a 386 DOS machine. Putting the data into a spreadsheet allowed the data to be sorted by columns which was used extensively for error checking, converting to metric units, assuring all data entries for the same reference used a similar reference pressure and discharge coefficient, sorting components, etc.

A copy of the data obtained from the references is contained in Appendix A. The data in fields 7-11 and 13-17 have been converted to metric units using the conversion factors contained in Appendix D.

A metric conversion of the constants $C$ and $n$ were done in order for the result of the equation to be in metric. The following equations were used to make the transformation:

$$
\begin{align*}
& C_{s-1}=C_{t-p} *(1.572)(1 / 248.66)^{n t p} \\
& n_{s-1}=n_{t-p}
\end{align*}
$$

In several cases pressurization-flow data were presented in graphical or tabular form. If an equation was not given for this data, the data were fit to the power law equation (3.4.4) using a least squares regression on the linearly transformed data. Data points given in graphical format only (ie. graphs only - no numerical data) were digitized using an enlarged photocopy of the graph and a digitizing pad. When observed data points were not indicated on the plot, five equally spaced points along the line were digitized.

## Table 4-2. Database Format for Information Recorded From Literature

## Field Item

AIRBASE reference number - This is the number assigned to the reference by the AIRBASE literature data base. The complete reference citation is included in the Bibliography
2 Class or category of leakage - An identification of the component or source of the leakage. See Table 4-1 for a listing of the leakage categories. For those cases where it was not possible to get a clear indication of the type of component, were lumped together under "general" by component.
3 Identifier tag or ID number of test - If the reference indicated an identifier for the particular test or site it was included in this field.
4 Number of cases or replications
5 Technique the source used to obtain the data point (See Appendix $C$ for explanation of the code number)
6 Code to identify units and explain how C and n values were obtained (See Appendix C for explanation of the code number)
7 Reported C value - The constant for the power equation
8 Reported $n$ value - The exponent for the power equation
9 Flow value - The flow through the opening at the reference pressure specified. This value included in this column is the value reported in the reference. For situations with multiple samples, this is the average value reported.
10 Minimum flow value - If a range for multiple samples was given the smaller number was assumed to be the minimum flow value.
11 Lower limit sample standard deviation flow - The average value minus the standard deviation when the range of values was given by the standard deviation.
12 Upper limit sample standard deviation flow - The average value plus the standard when the range of values was given by the standard deviation.
13 Maximum flow value - The maximum flow value with multiple replications.
14 Units for flow - Units reported in the reference (conversions to other flow units ie. I-P to S-I are given in Appendix D)
15 Leakage area term - The leakage area reported in the reference, expressed in units as given in column 18. No attempt has been made here to change the discharge coefficient or reference pressure
16 Minimum area value - If a range of values was given the smaller number was assumed to be the minimum area.
17 Area lower limit sample standard deviation - The average value minus the standard deviation when the spread of values was described by the standard deviation
18 Area upper limit sample standard deviation - The average value plus the standard deviation when the spread of values was described by the standard deviation.
19 Maximum area value - The maximum area value reported with multiple replications.
20 Units for area - Units reported in the reference (conversions to other area terms are given in Appendix D)
21 Discharge coefficient - Value reported or assumed. This was not easily determined. A value of 1 was assumed when 4 Pa was the reference pressure.
22 Reference pressure for reported values ( Pa ) - Reference pressure reported for the flow or area terms.
23 \% of total building leakage - Often reported in whole house or selective sealing testing methods.
Note \# - Notes to aid in further describing the data. A key to the numbers is in Appendix C. Other key or descriptive information - Other brief descriptive information indicated in the reference.

Initially all data was entered into the database. When one paper reported data which was obtained from another source, the data was entered with the original reference in the source field with a notation in the "other" field that this data was reported by the second source. These data were deleted from the data base when it was verified that the original data from the original source was included in the database. There were many cases which were found in which a data value in one reference had propagated to several references. Thus, several times initially it was incorrectly assumed that there were considerable data on a component when in reality it was several duplications of previously reported data.

### 4.1.2 Transformation to Common Base Pressure and Discharge Coefficient

As previously discussed, the data found in the literature was reported different ways, obtained at several different pressures and used different discharge coefficients. In general however the data could be broken into three main categories:
a. An equation was given for the data (or curves were presented),
b. The fiow was given at a particuiar pressure difiference, andior
c. The leakage area was reported for a given reference pressure and discharge coefficient.

In order to make comparisons between the sources it was necessary to transform the results to a common reference. It was decided to transform all the data to an ELA using 4 Pa as the base and a discharge coefficient of 1.0 since this was the most common format.

When an equation was given for the data, the flow at 4 Pa was calculated. The effective leakage area was then calculated (using a discharge coefficient of 1.0 and a reference pressure of 4 Pa ) from (eqn 23.28, HOF):

$$
\mathrm{L}=\left(\mathrm{C}_{\text {ref }} / \mathrm{C}_{\mathrm{d}}\right)\left[\rho /\left(2 \Delta \mathrm{P}_{\text {rof }}\right)\right]^{0.5}
$$

where:
$L=$ Effective Leakage Area at reference pressure
$\mathrm{Q}_{\text {rof }}=$ Flow at reference pressure
$C_{d}=$ Discharge coefficient
$\rho=$ Air density (assumed standard value of $1.2 \mathrm{~kg} / \mathrm{m}^{3}$ )
$\Delta P=$ Reference pressure
When the source had reported the data in terms of the flow at a given reference pressure and discharge coefficient, the leakage area at the reference pressure was calculated from Eqn 4.1.3. The effective leakage area at 4 Pa was then calculated from (eqn 23.29 HOF ):

$$
L_{r, 2}=L_{r, 1}\left(C_{d, 1} / C_{d, 2}\right)\left[\Delta P_{r, 2} / \Delta P_{r, 1}\right]^{n-0.5}
$$

where:

$$
\begin{aligned}
& L_{r, 1}=\text { area at } P_{1} \\
& L_{r, 2}=\text { area at } P_{2} \\
& C_{d, 1}=\text { discharge coefficient at } 1 \\
& C_{d, 2}=\text { discharge coefficient at } 2(=1.0) \\
& \Delta P_{r, 1}=\text { reference pressure used by literature source } \\
& \Delta P_{r, 2}=\text { reference pressure used in calculation (4.0 Pa) } \\
& n=\text { flow exponent }
\end{aligned}
$$

A value of 0.65 was assumed for the flow exponent if an equation for the data was not given.

When the source reported the data in terms of leakage area, the effective leakage area at the reference pressure of 4 Pa with a discharge coefficient of 1.0 was calculated from Eqn 4.1.4.

The effective leakage area was calculated for each of the values reported for the minimum, average and maximum flows and areas. A single effective leakage area was calculated when the data were reported in the power equation form.

The effective leakage areas thus calculated are found in columns 13-19 in the data contained in Appendix B.

The units for the ela are $\mathrm{cm}^{2}$ per whatever unit was used by the source. For example, if the source gave the flow in $1 / \mathrm{sm}^{2}$ of component, the ela would be $\mathrm{cm}^{2} / \mathrm{m}^{2}$ of component area. In some instances for the same component there were units of $\mathrm{cm}^{2}$ per: entire house, unit (eg. door), $\mathrm{m}^{2}$ of floor area, and/or per linear unit ( m ) of crack or sash.

Weidt et al (1979) indicated that varying the expression of air leakage rate between crack length, sash area and free ventilating area dramatically shifts the relative performance of the tested window operation type. Data concerning the area of the component were included in the "other" field when it was available, however only limited attempts (when sufficient information was given) were made to transform the data from one set of ela units to another (eg. $\mathrm{cm}^{2} / \mathrm{m}^{2}$ to $\mathrm{cm}^{2} / \mathrm{lmc}$ ). When this was done it was indicated in the "other" field.

The selection of the ela to represent a component was made by selecting the minimum and maximum elas and then attempting to subjectively determine a weighted average for the overall average ela. The weighing was based upon: the number of samples, the source of data, the age of data, and the grouping of independent data.

There was a large number of references which reported whole house or ductwork leakage values without supplying sufficient information about the structures to separate the component values from the data. It was determined that cataloging of this data was not going to yield usable information for the project and was beyond the scope of this project. In addition it is known that there are currently several significant
projects underway to obtain the leakage of ductwork. Since these data were not available, they could not be included in this report. Therefore the whole house and ductwork data were not included in the database.

### 4.2 THEORETICAL/EXPERIMENTAL INVESTIGATION

### 4.2.1.1 Theoretical Analysis - Single Openings

As discussed in a previous section, there are three major ways to deal with the $\mathrm{Q}-\Delta \mathrm{p}$ relation. These are:

1) Power equation,
2) Orifice equation, and
3) Dimensionless crack flow equation.

It is convenient to use the power equation for any shaped crack when the dimensions of the cracks are not known. However C and n are the products of regression only and they have no corresponding physical meaning since the equation is not theoretically derived. A further disadvantage is that the equation lacks generality because it is not dimensional homogenous. Hence its application is mainly because it is easy to use and it statistically fits data well. There is no theoretical basis involved

The orifice equation is theoretically derived from the Bernoulli equation. The constants $C_{d}, A_{0}$ and $\rho$ have clear physical meanings. However the relation that $Q$ is proportional to the square root of $\Delta P$ is restrictive because it neglects minor losses.

Using the dimensionless crack flow equation is an improvement; however, there are still restrictions in its application: a) the cross-section area of crack needs to be known to calculate the average velocity, $\overline{\mathrm{V}}$, thus it is difficult to calculate for cracks with irregular or unknown shapes, and b) there is not an easily solved relationship which can be derived from a pressurization test.

In general, it can be seen that using the dimensionless crack flow equation is a better approach. However the restrictions need to be loosened before it can be used. If the dimensionless crack flow equation can be arranged so as to have Q as a function of $\Delta P$, then the equation developed will have the benefits of the power equation and the orifice equation. Besides, if an approximate cross-sectional area of a crack can be determined automatically and statistically with original data sets rather than using an assumed dimension as an input, this will be a great improvement.

The dimensionless crack equation has been derived from first principles for idealized openings in Appendix $E$ in the form of:

$$
\frac{\Delta P}{\frac{1}{2} \rho \bar{V}^{2}}-\frac{B}{R e} \frac{Z}{D_{h}}+K
$$

The volumetric flow rate $Q=\bar{V} A$, and $D_{h}$ is the hydraulic diameter of cracks. After obtaining a new equation, it will be expanded to irregular cracks in a later section.

Substituting for velocity and the Reynolds Number into equation [4.1],

$$
\frac{\Delta P}{\frac{1}{2} \rho\left(\frac{Q}{A}\right)^{2}}-\frac{B}{\left(\frac{Q D_{h}}{A v}\right)} \frac{Z}{D_{h}}+K
$$

and simplifying, yields:

$$
\frac{2 \Delta P A^{2}}{\rho Q^{2}}-\frac{B Z_{v A}}{Q D_{h}^{2}}+K
$$

Multiplying $\mathrm{Q}^{2} / \mathrm{K}$ to both sides of the above equation and rearranging in the form of a quadratic equation, it can be solved (only the positive root is meaningful) as:

$$
\begin{gather*}
Q^{2}+\frac{B Z v A}{K D_{h}^{2}} Q-\frac{2 \Delta P A^{2}}{K \rho}=0 \\
Q=A\left[\sqrt{\left(\frac{B Z v}{2 K D_{h}^{2}}\right)^{2}+\frac{2 \Delta P}{K \rho}}-\frac{B Z v}{2 K D_{h}^{2}}\right]
\end{gather*}
$$

This $\mathrm{Q}-\Delta \mathrm{P}$ expression is derived from the dimensionless crack flow equation and is still based on the $\mathrm{Q}-\Delta \mathrm{P}$ data obtained from blower door tests. Each parameter or constant has a clear meaning. Now the question is can the equation be expanded to also include openings with irregular shapes and sizes?

Equation 4.2 may be rewritten in simplified form:

$$
Q-C_{1}\left[\left(C_{2}^{2}+C_{3} \Delta P\right)^{0.5}-C_{2}\right]
$$

where:

| $\mathrm{C}_{1}=\mathrm{A}$ | $\mathrm{m}^{2}$ |
| :--- | :--- |
| $\mathrm{C}_{2}=\mathrm{BZv/2KD}_{\mathrm{h}}$ | $\mathrm{m} / \mathrm{s}$ |
| $\mathrm{C}_{3}=2 / \mathrm{K} \rho$ | $\mathrm{m}^{3} / \mathrm{kg}$ |

For well-defined openings, it is not difficult to use the above equation because each geometric term has a clear meaning. For irregular cracks, where the sectional area, $A$, is variable, we can still use equation (4.3) to get an area value, but it will be the equivalent sectional area. $D_{h}$ may be approximately defined as:

$$
D_{h}-\sqrt{\frac{4 A}{\pi}}
$$

which is derived from $A=\pi D_{h}{ }^{2} / 4$. Based on this definition, $D_{h}$ will be the equivalent diameter of the crack.

Thus the three geometric parameters $\mathrm{C}_{1}, \mathrm{C}_{2}$, and $\mathrm{C}_{3}$ may be determined from dimensional measurements for well defined openings. (It can be shown that Eqn 4.3 reduces to the orifice equation for a flow length equal to zero.) A problem however occurs with openings where $B, Z, D_{h}$, and $K$ are not well defined and there is not a single solution to the set of equations.

Nonlinear regression techniques were used to determine values for the constants which minimized the error between the prediction equation and the data obtained from the fan pressurization tests.

It is known that the coefficients have some physical limits, so bounds were placed on the range the coefficients could assume so that physically infeasible solutions would not be provided.

Let's cencider ceofficiont $\mathrm{C}_{3}$. The density of air, $\hat{\rho}$, has a limit ûin the variatiun of its value. If the test is conducted at sea-level atmosphere pressure ( $101,325 \mathrm{~Pa}$ ), and temperature of air is in range of $-30^{\circ} \mathrm{C}$ to $+40^{\circ} \mathrm{C}$, the variation of air density is about 1.453 to $1.128 \mathrm{~kg} / \mathrm{m}^{3}$. The minor loss coefficient, K , also has a limited variation (1.2-2.3) as shown in previous literature (Etheridge 1977 and Chastain et al.1987). Therefore it is not difficult for us to estimate the $\mathrm{C}_{3}$ range, and provide bounds on $\mathrm{C}_{3}$ in the regression routine.

It is also clear that $C_{1}>0$ and $C_{2}>0$. Therefore, let $C_{3}$ be a bounded-coefficient and $C_{1}, C_{2}$ be semi-free coefficients to be determined in regression.

There are several nonlinear statistical packages available. The routine selected for this work was the SAS procedure NLIN (SAS, 1985). This procedure produces least squares estimates of the parameters of a nonlinear model. The form of the equation, initial estimates of parameter starting values, and derivatives of the model with respect to the parameters are required inputs. It evaluates the residual sum of squares at each combination of initial parameter values over the range provided to determine the best set of values to start the iterative algorithm. The Marquardt method of iteration was used. This method regresses the residuals onto the partial derivative of the model with respect to the parameters until the iterations converge.

The three coefficients $\mathrm{C}_{1}, \mathrm{C}_{2}$ and $\mathrm{C}_{3}$ obtained from the regression analysls can then be used to estimate the parameters of the openings. If the air density value in the test was known, a value of the minor loss, $K$, can be calculated from $\mathrm{C}_{3}$ directly. (It should be noted that this minor loss value is just an average value.) Then by substituting $C_{1}, C_{2}$ and $C_{3}$ values, we can obtain the $B \cdot Z /\left(\operatorname{Re} \cdot D_{h}\right)$ vs. $\Delta P /\left(1 / 2 \rho \bar{V}^{2}\right)$ relationship which is the dimensionless crack equation representation of the $Q-\Delta P$ data set:

$$
\begin{aligned}
& \frac{B Z}{R e D_{h}}-\frac{B Z}{\left(\frac{\bar{V} D_{h}}{v}\right) D_{h}}-\frac{\frac{B Z v}{D_{h}^{2}}}{\frac{Q}{A}}-\frac{A \frac{B Z v}{2 K D_{h}^{2}}}{\frac{2}{k \rho}} \frac{4}{\rho Q}-\frac{C_{1} C_{2}}{C_{3}} \frac{4}{\rho Q} \\
& \frac{\Delta P}{\frac{1}{2} \rho \bar{V}^{2}}-\frac{2 \Delta P}{\rho\left(\frac{Q}{A}\right)^{2}}-A^{2} \frac{2 \Delta P}{\rho Q^{2}}-C_{1}^{2} \frac{2 \Delta P}{\rho Q^{2}}
\end{aligned}
$$

This means each $\Delta \mathrm{P} /\left(1 / 2 \rho \overline{\mathrm{~V}}^{2}\right)$ value and the corresponding $\mathrm{B} \cdot \mathrm{Z} /\left(\operatorname{Re} \cdot D_{h}\right)$ value can be obtained directly from the $\mathrm{Q}-\Delta \mathrm{P}$ data set and the nonlinear regression products $C_{1}, C_{2}$ and $C_{3}$. In this technique there is no need to make further assumptions or provide the dimensions of the crack to get the $B \cdot Z /\left(\operatorname{Re} \cdot D_{h}\right)$ and $\Delta P /\left(1 / 2 \rho \bar{V}^{2}\right)$ values. This is a significant improvement over the previous methods of evaluating the physical based models (Etheridge 1977, Chastain et.al 1987).

From equation (4.1), it can be shown that the minor loss is the difference between the total pressure and the major loss (all three terms have dimensionless units):

$$
K-\frac{\Delta P}{\frac{1}{2} \rho \bar{V}^{2}}-\frac{B Z}{\operatorname{Re} D_{h}}
$$

It can be seen that as $\Delta P$ changes, the $K$ value is not a constant after the above subtraction for each data point. We know that for a certain crack; $D_{h}, Z$ and $B$ are constants, leaving only the variables, $\Delta P, Q$ or $R e$ to account for the change in $K$. Actually $R e$ is a function of $Q$ only for a certain crack, while $\Delta P$ is independent of $Q$. Hence one $\Delta \mathrm{P}$ is exactly corresponding to one Q or Re; they are not independent of each other and have some functional relationship. For most air leakage studies, $\Delta P$ is used as the independent variable, hence we define:

$$
\begin{equation*}
K(\Delta P)=K(Q)-K(R e)-\frac{\Delta P}{\frac{1}{2} \rho \bar{V}^{2}}-\frac{B Z}{R e D_{h}} \tag{4.5}
\end{equation*}
$$

The original minor loss coefficient, $K$, was previously taken as a constant. It is actually the average residual between the total and major loss:

$$
\begin{equation*}
\bar{K}-\frac{\sum_{i-1}^{n} K\left(\Delta P_{j}\right)}{n} \tag{4.6}
\end{equation*}
$$

Where n is the number of data points.
A number of different functional forms may be regressed to get the $K(\Delta P)$ expression. If the scatter plot of $K(\Delta P)-\Delta P$ appears linear, we may use a simple linear approximation:

$$
\begin{equation*}
K(\Delta P)-a(\Delta P-b)+\bar{K} \tag{4.7a}
\end{equation*}
$$

Or if the scatter points fits a curve, a quadratic approximation may be applied.

$$
\begin{equation*}
K(\Delta P)-a(\Delta P-b)^{m}+\bar{K} \tag{4.7b}
\end{equation*}
$$

Theoretically describing the functional form of the minor loss is beyond the scope of this project. From the $K(\Delta P)-\Delta P$ relation, in practice, we can regress and predict to get a $K(\Delta P)$ to substitute into equation (4.1) and (4.3), where constants a and b , or $\mathrm{a}, \mathrm{b}$ and m can be determined consequently. For a simple calculation, the linear approximation may be suggested as a better choice:

$$
\begin{equation*}
Q=A\left[\sqrt{\left(\frac{B Z v}{2 K(\Delta P) D_{h}^{2}}\right)^{2}+\frac{2 \Delta P}{K(\Delta P) \rho}}-\frac{B Z v}{2 K(\Delta P) D_{h}^{2}}\right] \tag{4.8}
\end{equation*}
$$

where $K(\Delta P)$ is defined by [4.7a].
The orifice equation previously discussed is a special case of this equation. For the dimensionless crack flow equation, if $Z / D_{h}{ }^{2}$ is close to zero (flow length approximating zero), then $\mathrm{C}_{2}=\mathrm{BZv} /\left(2 \mathrm{~K}(\Delta \mathrm{P}) \mathrm{D}_{\mathrm{h}}{ }^{2}\right)$ will approach zero also. Equation [4.8] reduces to:

$$
Q=A \sqrt{\frac{2 \Delta P}{K(\Delta P) \rho}}-\frac{A}{\sqrt{K(\Delta P)}} \sqrt{\frac{2 \Delta P}{\rho}}
$$

Comparison with the orifice equation in which flow length $Z=0$, and $E Q L A=A$ yields:

$$
Q-E Q L A C_{d} \sqrt{\frac{2 \Delta P}{\rho}}-A C_{d} \sqrt{\frac{2 \Delta P}{\rho}}
$$

Hence another discharge coefficient expression for the orifice is:

$$
\begin{equation*}
C_{d}-\frac{1}{\sqrt{K(\Delta P)}} \tag{4.9}
\end{equation*}
$$

This indicates that the orifice equation is just a special case for the derived dimensionless crack flow equation. On the other hand, we find that the curve performance of the new model is very close to that of the power equation for different kind of cracks. That is, it is a theoretical derivation of the orifice equation and yields statistical results as good as the power equation. Therefore the new equation has the benefits of the power and the orifice equations plus sufficient parameters to make judgement on how the air is flowing in complex flow paths.

It can also be shown that the ELA and $C_{d}$ are dependent on $\Delta P$ in the general case. By definition, ELA is:

$$
E L A=\frac{Q}{\sqrt{\frac{2 \Delta P}{\rho}}}=C_{d} \text { EQLA }
$$

substituting for Q of equation [4.8] yields:

$$
\begin{equation*}
E L A=A\left[\sqrt{\left(\frac{C_{2}}{\sqrt{\frac{2 \Delta P}{\rho}}}\right)^{2}+\frac{1}{K(\Delta P)}}-\frac{C_{2}}{\sqrt{\frac{2 \Delta P}{\rho}}}\right] \tag{4.10}
\end{equation*}
$$

Thus the importance of treating K as a function rather than a constant can be shown by looking at the change in the discharge coefficient:

$$
\begin{equation*}
C_{d}-\sqrt{\left(\frac{C_{2}}{\sqrt{\frac{2 \Delta P}{\rho}}}\right)^{2}+\frac{1}{K(\Delta P)}}-\frac{C_{2}}{\sqrt{\frac{2 \Delta P}{\rho}}} \tag{4.11}
\end{equation*}
$$

### 4.2.1.2 Theoretical Analysis - Multiple Openings Connected in Series or Parallel

Crack flow resistance was defined previously as the inverse of the flow coefficient of the empirically regressed power equation (Cale and Zawacki 1980, Bassett 1986). That is,

$$
Q-C(\Delta P)^{n}-\left(\frac{1}{R}\right)(\Delta P)^{n}
$$

where $R=1 / C$ and is called the "resistance to crack flow" with units of $\mathrm{Pa}^{\mathrm{n}} \cdot \mathrm{S} / \mathrm{m}^{3}$.
There is limited literature which gives the definition of resistance of crack flow, and there is no theoretical derivation. The basic idea for the concept should be:

1) crack flow resistance is the ratio of the driving force and the transfer rate.
2) it is necessary to satisfy the "parallel and series theorem", i.e, for parallel path flow, $R_{\text {total }}=1 / \Sigma\left(1 / R_{i}\right)$; for series path flow, $R_{\text {total }}=\Sigma R_{i}$.

Figures 4-1 and 4-2 illustrate the cracks in parallel and series connections with their resistance relationships.


Figure 4-1. Crack parallel connection


Figure 4-2. Crack series connection
An analogy can be made between the concepts of resistance in electrical circuits and heat conduction and crack flow resistance.

In electrical circuits:

$$
I-\frac{V}{R}
$$

In heat conduction:

$$
q-\lambda \Delta T-\frac{\Delta T}{R}
$$

where I and q are the transfer rates, called current intensity and heat flux respectively, V and $\Delta \mathrm{T}$ are the driving forces, called voltage and temperature difference respectively.

The relationship between the driving force and transfer rate depends on the characteristic of the resistance. If the electrical resistance $R$ and heat resistance $R$ are constant, the I-V and $\mathrm{q}-\Delta \mathrm{T}$ relationships should be linear, otherwise, they will be nonlinear for nonconstant $R$.

In the problem of flow through cracks, $\Delta P$ or some term involving $\Delta P$, is the driving force and Q is the transfer rate. Before trying to make judgement on the validation of the previous concept of the crack flow resistance, we look at an example to explore some of the problems involved in the previous definition.

In Figure 4-3 two power equations are presented which were fit to fan pressurization test data on two different cracks.

Crack 1: $\mathrm{Q}=\mathrm{C} \cdot(\Delta \mathrm{P})^{0.5}$
Crack 2: $\mathrm{Q}=\mathrm{C} \cdot(\Delta \mathrm{P})^{1.0}$


Figure 4-3. Two power equations with $n=1$ and $n=0.5$
On the basis of the previous definition, if the values of flow coefficients are identical then the tlow resistances tor the two cracks will be the same. That is,

$$
R_{1}-\frac{1}{C}-R_{2}
$$

However, when these two curves are plotted, it can be seen that:
a) When $\Delta \mathrm{P}<1 \mathrm{~Pa}$, crack 1 allows more flow than crack 2 for the same pressure difference, thus, $R_{1}<R_{2}$ and
b) When $\Delta P>1 \mathrm{~Pa}, \mathrm{R}_{1}>\mathrm{R}_{2}$ for the same reason.

Unfortunately there is only one point, $\Delta P=1 \mathrm{~Pa}$ which satisfies $R_{1}=R_{2}$.
This example illustrates the fallacy the original definition of crack flow resistance being the inverse of the regression coefficient $C$.

Another proposed definition is based on the effective leakage area (ELA) which characterizes the air leakage. The resistance of crack flow with units of $\mathrm{m}^{-2}$ can be defined as:

$$
R-\frac{1}{E L A}
$$

Based on the ELA definition formula:

$$
Q-\frac{1}{R} \sqrt{\frac{2 \Delta P}{\rho}}
$$

where $\sqrt{\frac{2 \Delta P}{\rho}}$ is considered as a driving force.

These two definitions create different formulas for the resistance of flow through openings which are in parallel and series connections. Table 4-3 presents the difference between these resistance definitions. Examples to test if there is any improvement in prediction due to the use of the new definition will be given in the Results Section.

Table 4-3. Crack flow resistance definition

|  | Previous definition: $R=1 / C$ | New definition: $R=1 / E L A$ |
| :---: | :---: | :---: |
| Parallel path | $\begin{aligned} & \frac{1}{R_{\text {total }}}-\sum\left(\frac{1}{R_{1}}\right) \\ - & C_{\text {totalal }}-\sum C_{1} \end{aligned}$ | $\begin{array}{r} \frac{1}{R_{\text {total }}}-\sum\left(\frac{1}{R_{i}}\right) \\ E L A_{\text {total }}-\sum E L A_{1} \end{array}$ |
| Series path | $\begin{gathered} R_{\text {total }}-\sum R_{1} \\ \frac{1}{C_{\text {total }}}-\sum\left(\frac{1}{C_{1}}\right) \end{gathered}$ | $\begin{gathered} R_{\text {total }}-\sum R_{1} \\ \frac{1}{E L A_{\text {total }}}-\sum\left(\frac{1}{E L A_{1}}\right) \end{gathered}$ |

### 4.2.2 Experimental Procedure

A series of experiments were run in order to verify and validate the theoretical developments of the previous section and to gather additional information about the parameters for actual building components. A series of pressure-flow data were taken in the laboratory over a wide pressure range for:
a) a number of geometrically well defined, straight opening specimens tested individually,
b) pairs of well defined openings placed in parallel such that flow would go through them independently,
c) pairs of well defined openings placed in series so that the air would have to travel through both of them, and
d) a number of building components mounted in $2.44 \times 2.44 \mathrm{~m}\left(8^{\prime} \times 8^{\prime}\right)$ wood frame wall sections.

The basic idea for this experiment was to use well-defined openings (openings with straight wells and a flow path with known dimensions) with knowin $C_{1}$ values to find the other two opening flow characteristics $\mathrm{C}_{2}$ and $\mathrm{C}_{3}$, and the n values from the power equation, to establish a numerical transform formula. Secondly, based on the established formula (Equation [4.3]), the constant $C_{2}$ was determined and the other two constants obtained for several building components. The quality of the new fitted model for analyzing building components was then statistically tested.

In addition to model calibration and validation there were other reasons to test the equation. There was a desire to check some of the coefficients to determine:
a) if there was a difference between the goodness-of-fit of this equation compared to the power or orifice equations over the range of the data taken, and
b) if any difference could be shown between the equation types between the flow and predictions at $5 \mathrm{~Pa}\left(\mathrm{O}_{4}\right.$ was not chosen because data were not taken at that pressure difference).

### 4.2.2.1 Test Apparatus

A system to produce and sense a differential pressure across the specimen was constructed in the Energy and Environment laboratory in the Agricultural Engineering Department. This system consisted of: a variable speed fan to produce the airflow and differential pressures; an airflow monitoring chamber to measure airflow; an air distribution and straightening plenum; a specimen and building component holder; temperature, and barometric and differential pressure transducers and the necessary electronics for data acquisition. A schematic of the test apparatus is given in Figure 4.2.1.


Figure 4.2.1 A schematic of the air leakage measurement system for testing the well defined cracks and building components.

## Fan

A variable speed, six radial blade fan with a 26.8 cm (10 9/16") wheel was used to generate the pressure and airflow. The fan shaft was attached to a $1.5 \mathrm{~kW}(2$ hp ) variable speed DC motor via belt with $2: 1$ sheave ratio. The rotational speed of the fan was sensed with a permanently installed tachometer detecting light bouncing off a reflective tape strip attached to the fan shaft. The output of the tachometer was taken to a 4 digit LED display. Correct operation of the tachometer was checked at the beginning of the study. The DC motor speed was controlled with a variable 0-10V input which was adjusted with a 10 turn 5 K ohm potentiometer. The speed controller specifications indicated a time constant of 6 seconds and a very stable long term speed control (within $+/-1 \%$ ). Observation of the digital fan speed indicator showed the fan speed to be very stable. The fan was rated at $140 \mathrm{l} / \mathrm{s}(300 \mathrm{cfm})$ output at $17.8 \mathrm{~cm}\left(7^{\prime \prime}\right)$ static pressure and 3500 rpm .

## Airflow Measuring Station

The airflow was measured using a multiple nozzle outlet chamber built to the specifications of ANSI/ASHRAE Standard 51-1985 (ANSI/AMCA Standard 210-85). (See Figure 12 in Standard 51. ) The chamber was 122ㄴ122×305 cm (18×18×120"), made with 2.54 cm square steel tubing frame and covered with 18 gage steel sheets. The sheet metal was attached to the outside of the tubing frame with metal bonding, double-faced adhesive tape and blind pop rivets. All joints were sealed inside and out with a high grade silicone caulk. Access was provided to the inside of the chamber with a $45 \times 45 \mathrm{~cm}$ metal plate door on either side of the nozzle plane. A seal was produced at the edges of the doors where they overlapped the frame. Each door was held tightly closed with eight fasteners. The settling means was provided at the locations specified in the standard with one layer each of $40 \%$ and $60 \%$ open, 24 gage metal sheets attached to cross bracing on the interior of the chamber. Velocity readings were taken during system testing on a 10 cm grid across the face of the settling mesh and indicated a uniform flow. Ten aluminum spun nozzles without throat taps ( $L=0.6 D$ ), $(D=12.7,17.5,25.4,40.6,50.8,63.5,76.2,101.6,127$, and 1.52 .4 mm ) were installed on the nozzle plane inside the chamber. The nozzles were located relative to each other on the plane so any combination of nozzles could be operated simultaneously. Static pressure taps constructed as specified in the standard ( 0.16 cm diameter) were placed in the chamber as specified.

Differential pressure across the nozzles was measured with a 25 cm WG f.s. variable capacitance diaphragm transducer (accuracy $=< \pm 1.0 \%$ fs, repeatability $=$ $<0.3 \% \mathrm{fs}$ ). All output voltages were obtained from $41 / 2$ digit voltmeters with an RMS averaging function. Averaging time windows were approximately 30 seconds. Static pressure upstream of the nozzles was measured similarly with one side of the differential transducer being open to the room atmosphere. Barometric pressure was measured in the room with a mercury barometer with 0.1 mm resolution. Dry bulb room temperature was determined with a mercury thermometer and relative humidity was read from a recording hydrothermograph ( $\pm 5 \% \mathrm{rh}$ ). Incline manometers (resolution $0.05{ }^{\prime \prime} \mathrm{wg}$ ) were piped in parallel to the electronic transducers to enable a quick periodic check on the electronic devices.

During data analysis an error was found which indicated that there was a significant systematic error in the measuring chamber. After considerable investigation with smoke pencils, it was found that there were some leaks across the nozzle plane through the structural tubing due to leakage through a weld. Substantial effort was spent calibrating this additional leakage across the nozzle plane. (See Section 4.2.2.3.1 Overall System and Chamber Background Leaks Correction.) These calibration data were obtained before the leakage was stopped thus calibration could be done on the original data. This systematic error will be referred to in the Results Section as the system and chamber background leakage.

Equations specified in the ASHRAE Standard 51-1985 were used to calculate the airflow at standard temperature and pressure.

## Air Plenum

The output from the airflow monitoring chamber was ducted to a wooden plenum chamber which expanded from the $15 \mathrm{~cm}\left(6^{\prime \prime}\right)$ outlet to the $2.44 \times 2.44 \mathrm{~m}$ testing face. This expansion chamber was made of marine grade plywood, had two coats of varnish applied to the inside and outside surfaces, and was caulked extensively. The unit was checked for leaks with a smoke stick when the unit was pressurized to four times the maximum operating pressure.

Two planes of 6.3 mm pegboard were installed between the inlet and exit to create backpressure and assure uniform air distribution at the testing face of the unit. Uniform air flow across the face of the pegboard was checked by taking air velocity readings with a hot wire anemometer on a 15 cm grid at the face of the pegboard plane closest to the test face.

Static pressure taps for the high pressure side ("interior") of the differential pressure to be applied across the openings to be tested were mounted on the straight section sides of the plenum approximately 60 cm from the face of the test section.

The direction of airflow was always to the exterior, simulating building pressurization. Depressurization tests were not run.

## Downstream Air Wind Shield

A shield was built to be placed downstream of the test specimen so the "outside" surface of the test specimens and the "exterior" static pressure taps would be shielded from any air currents produced by the diffusers of the building HVAC system. The shield had $1.22 \times 2.44 \mathrm{~m}$ top, bottom and two sides perpendicular to the specimen. After the specimen holder was put in place the shield would be rolled to meet with the specimen holder. Static pressure taps were mounted on the inside of each of the four sides of the shield for the low side of the differential pressure transducer across the specimen.

## Specimen Holder

A wooden frame was built to mount the well defined cracks (Figs 4.2.2a-b). This frame was designed to be able to hold the openings individually, two in parallel or


Figure 4.2.2a Specimen Crack Holder Frame


Figure 4.2.2b Cross section of two cracks in series mounted on the outer and inner layers of the frame
two in series. The effective open face on the specimen holder was $91 \times 122 \mathrm{~cm}$. The unit was sandwiched between the plenum and the air shield and pipe clamps were used to hold the three pieces together. It was recognized that these joints had the potential for creating a systematic error. Therefore extraordinary effort was expended putting in seals between the plenum and the specimen holder and/or wall sections to prevent air from leaking out around the joint. Pipe clamps were used extensively to apply pressure on the seals to prevent them from leaking. Attempts were made to tighten the clamps to the same pressure each time and the joints were checked to locate and fix any obvious leaks.

## Well Defined Openings

Several well defined straight-through openings had been previously used in a study of determining the discharge coefficients for laminar flow in rectangular openings (Chastain and Colliver, 1987). These openings (See Figure 4.2.3) were thin rectangular openings with straight walls and square edged openings which were manufactured to very tight tolerances. They were made from 6 mm clear acrylic plastic sheet. Six of these openings which varied in open cross sectional area by a factor of approximately 16, varied in flow length by a factor of $31 / 2$ and opening height were selected. There were two geometries which had two crack specimens each which were used as replicates. The specifications for the openings are given in Table 4.4.

## Building Component Wall Sections

A number of building components and wall penetrations were tested in $2.44 \times 2.44 \mathrm{~m}$ wall sections (Table 4.5 ). A separate $2 \times 4$ wood frame wall was constructed for each type component tested (with exception of things which could be changed without disturbing the wall such as outlet or switch gaskets). Typical single bottom plate, double top plate, $400 \mathrm{~mm}\left(16{ }^{\prime \prime}\right)$ O.C. $50 \times 100 \mathrm{~mm}$ (2X4") SYP (southern yellow pine) stud walls were constructed by a summer student worker with supervision given by a carpenter. The side representing the interior wall surface was covered with 13 mm gypsum board drywall which was tapped and mudded. The exterior side was covered with 13 mm foil backed polyisosynurate insulating board fastened with 32 mm drywall nails approximately every 150 mm on the edges and 200mm on the interior. No vapor barrier was installed. Several components such as premium awning, premium double hung, economy double hung, premium casement, economy casement, copper pipes and electric outlets and switches were installed in wall sections made like the base wall. Six outlets/switches were installed on one wall with each plastic switch box in an individual wall cavity between the studs. Wire penetrations went through the top plate. (Airflow through penetrations in the top and bottom plates was not blocked by the experimental apparatus and was exposed to the same pressures as the "exterior" side of the wall.) There were no wire holes between the vertical studs. $1 / 2^{\prime \prime}$ copper water pipe was used in the 1 " holes in the bottom plate between each stud space to represent water pipe penetrations. A list of the components tested is given in Table 4.5. Several different types of windows were tested. For each test case two windows from the same manufacturer of the same type, style and size were installed in a wall section constructed like Case A. The values presented are for the entire test section unless otherwise indicated.


Figure 4.2.3 A typical rectangular crack (opening)

| Crack | Section <br> Area | d | Z | W |
| :---: | :---: | :---: | :---: | :---: |
| A | 400 | 0.8 | 25.4 | 500.1 |
| B1, B2 | 850 | 1.7 | 50.8 | 500.1 |
| E | 3145 | 6.3 | 88.9 | 499.3 |
| F1, F2 | 6431 | 12.9 | 50.8 | 498.5 |
| (Unit) | $\left(\mathrm{mm}^{2}\right)$ | $(\mathrm{mm})$ | $(\mathrm{mm})$ | $(\mathrm{mm})$ |

Table 4.4 Crack Geometry Specifications

## Table 4.5 List of the Building Components Tested

| CO-1 | Exterior Frame: Gypsum Board (Base Case A - no penetrations) |
| :---: | :---: |
| CO-2 | Exterior Frame: Insulating Board (Base Case B - no penetrations) |
| CO-3 | Wall Penetration: 6 Outlets with No Wire Holes in Studs, or Top and Bottom Plates |
| CO-4 | Wall Penetration: 6 Outlets with Gaskets and No Wire Holes |
| CO-5 | Wall Penetration: 6 Outlets, Wire Holes in Top Plate |
| CO-6 | Wall Penetration: 6 Outlets, Top Wire Holes Sealed |
| CO-7 | Wall Penetration: 6 Outlets with Gaskets, Top Plate Wire Holes Not Sealed |
| CO-8 | Wall Penetration: 6 Outlets with Gaskets, Top Plate Wire Holes Sealed |
| CO-9 | Wall Penetration: 6 Copper Water Lines through Bottom Plate |
| CO-10 | Wall Penetration: 6 Switches, Wire Holes in Top Plate |
| CO-11 | Wall Penetration: 6 Switches, Top Plate Wire Holes Sealed |
| CO-12 | Wall Penetration: 6 Switches with Gaskets, Top Plate Wire Holes Sealed |
| CO-13 | Wall Penetration: 6 Switches with Gaskets, Top Plate Wire Holes Not Sealed |
| CO-14 | 2 Premium Awning windows installed in Base Case Wall |
| CO-15 | 2 Premium Double Hung windows installed in Base Case Wall |
| CO-16 | 2 Economy Double Hung windows installed in Base Case Wall |
| CO-17 | 2 Premium Casement windows installed in Base Case Wall |
| CO-18 | 2 Economy Casement windows installed in Base Case Wall |

Specimen Differential Pressure Measurement
The differential pressure across a test specimen was measured using the same equipment as the differential pressure across the nozzles with the exception that the full scale range of the transducer was 65 Pa for low range and 625 Pa for high pressure measurements.

### 4.2.2.2 Experimental Design

To verify and validate the proposed equation, several applications of the same experiment were run. This involved obtaining data for flow versus differential pressure
across an opening(s) for several points between 5 and 125 Pa. A description of the experiment and the application of this experimental procedure to the test cases will be presented.

## Experimental Procedure

Each experiment involved placing an opening(s) in the specimen holder (for the case of the well defined cracks) or a wall section (in the case of building components) in the test apparatus and tightening the sandwich of plenum/specimen holder/air shield with pipe clamps. The seals around the joints were checked for leakage.

The fan would then be adjusted to produce at least 125 Pa across the specimen. Nozzles in the airflow chamber would then be opened or closed (and speed readjusted) to obtain the smallest nozzle which would allow the fan capacity to create sufficient airflow to provide the necessary 125 Pa . This procedure was done in an attempt to provide maximum resolution of airflow measurement.

Data would then be taken on the room air conditions (barometric pressure, dry bult temperature, relative humidity) and transducers (S/N's and offeet voltage with no pressure differential) being used. The wet bulb temperature used in the calculation of air density was obtained from a psychrometric program used in teaching the undergraduate environmental design classes.

The fan speed would then be adjusted to create a differential pressure of 5 Pa across the specimen. After approximately 30 seconds, if the pressure had stabilized, a pressure reading across the nozzle would be initiated (ie. the RMS averaging function on the voltmeter turned on). The voltmeter would integrate the voltage signal coming in and output a time running average of the signal. The differential pressure across the nozzle would thus be averaged for approximately 30 seconds before the reading was recorded. (This averaging was necessary because there was a small, low frequency [approximately 0.5-1.0 hz as determined from a digital spectrum analyzer] signal that was superimposed on the transducer signal. It was concluded that this signal was coming from changes in the internal pressures of the airflow chamber due to the diaphragm action of the chamber walls moving very slightly. This conclusion was reached by noting the frequency of the movement of the walls.) The actual pressure readings across the test specimen and the static pressure on the upstream side of the nozzles would be taken during the time the nozzle pressure reading was being averaged. (The specimen pressures were stable and did not require any time averaging.) This process was repeated for each pressure/flow data point taken.

The actual pressure readings ( $\pm 0.25 \mathrm{~Pa}$ ) across the test specimen and nozzle were thus taken for pressures from 5 to 125 Pa in nominal increments of 5 Pa .

The temperature, relative humidity and barometric pressure readings taken at the beginning of the test were retaken at the conclusion of each test. The starting and ending conditions were then averaged (if the temperature had changed) to estimate the air state points.

No test was accepted unless the entire pressure range could be completed at one time.


#### Abstract

A series of tests was initially run to determine if there was any hysteresis in the system. No significant differences were found in the readings if the order of pressures was increasing from 5 to 125 or if the data were collected with the setpoint pressures decreasing from 125 to 5 . Therefore the tests only used the pressure increasing from 5 to 125 with increments of 5 Pa . Only the data between 5 and 75 Pa were used in the data analysis for this project since this is the range commonly used with blower door testing. A comparison of analysis techniques using the range of data collected as one of the parameters to be investigated is being planned for future work. The wide data range was collected for use in that project.


## Experiments Performed

Four groups of tests were run using this method. They will be referred to as: individual openings, parallel, series and component openings. An identification of the cracks used for the various experiments and the experiment ID codes are presented in Appendix F.

Individual Openings
The purpose of this test was to obtain pressure-flow data on the well defined, simple geometry, straight-through cracks previously described (Table 4.1) when they were mounted in the specimen holder. The openings and flow covered a range of openings sizes and flow ranging from laminar to turbulent flow. There were two geometries which had two "identical" cracks each for replication purposes. Thus there were four different geometries tested.

An additional feature was added by testing the effects of mounting location on each of the four geometries. They were first tested mounted on the inner layer of the specimen holder (with only the mounting plate on the outer layer) and then mounted on the outer layer (with only the mounting plate on the inner layer). When there were no cracks installed on the outer layer there were two large openings ( $\approx 100 \times 520 \mathrm{~mm}$ ) for placement of the cracks in the mounting plate; thus it was possible for this plate to have some effect on the flow. The same is true for when the cracks were mounted on the outer plate with no cracks mounted on the inner plate.

Each test had three replications. The order of testing for the individual, parallel and series openings was determined from a random number table. If two tests of the same opening were in order, the crack was taken out and remounted.

## Parallel Openings

The purpose of this test was to obtain the pressure-flow data on the well defined cracks when the air would be going through them in parallel. This would be a test to see how the flow coefficients for individual cracks would combine when there were multiple openings such as the case when there are many different components acting independently to the total static pressure which is applied to all surfaces in whole house testing.

The combination of cracks chosen for testing is presented in Appendix F. These combinations included pairs of "identical" cracks to see if the predicted area coefficient, $\mathrm{C}_{1}$, doubled; and all possible combinations of the remaining geometries. Three replications were run on each test.

## Series Openings

The purpose of this test was to obtain the pressure-flow data on the welldefined cracks when the air would be going through them in series. This would be a check to see how the coefficients would combine when the air would be flowing through multiple restrictions. An example would be the case when the air goes into the electrical outlet, through the box, up the cavity space and then out the hole in the top plate cut for the electrical wire.

The combinations used in this series of tests was to place the largest opening at the inner mounting position and then place all the other geometries individually on the outer plate. This would simulate air flowing through a large hole first and then smaller holes. Tests were also made on the reverse placement (large on outside and smal!er on inside! Throe replications were made of each test.

## Building Components

The purpose of this test was to determine the flow coefficients for some typical building components ranging in quality from construction/economy grade units used in lower cost construction to premium grade, high quality units. In addition to the previous differences determined for the other groups of tests, there were additional differences to be investigated:
a) Could any physical difference be shown by the coefficients (eg. Is the flow through one large leak or many small ones?), and
b) Was there a general trend in the coefficients among the components?

A list of the building components tested was presented in Table 4.5. Tests were conducted on blank frame walls, a series of combinations of electrical outlets/switches and openings for the wire, and various types and quality of commercially available residential windows.

The building component tests were not completely randomized. All tests on a wall section that did not require sealing were run before caulking was applied to the openings. Sealing was done with high quality silicone caulk.

### 4.2.2.3 Experimental Data Analysis

### 4.2.2.3.1 Overall system and chamber background leaks correction

The measurement system was carefully sealed to prevent air leakage however calibration tests indicated that some air leakage existed in the system. The "overall system background leak" refers to the air leaks from the wooden collection chamber as well as from the frame holding the well-defined cracks. This term is only associated with testing the well-defined cracks.

The term "chamber background leak" refers to the leakage from the wooden collection chamber only, without including the leakage from the frame which holds the cracks. The chamber background leak is only for component testing.

Two leakage correction equations were used to deduct the air leakage from the data sets of flow measurements for well-defined cracks and building components individually. This correction procedure was included in the pressurization data of Appendices G-J. See the "Flow Rate" columns in the tables in Appendices G-J. The data in the replication column are the uncorrected values. The "System Leak" or the "Chamber Leak" refers to the "overall system background leak" or the "chamber background leak" respectively. The "Corrected Mean" values were obtained by subtracting the leak terms from the average values of the uncorrected three replications. Thus the "Corrected Mean" values are the actual flows entering the specimen being tested.

### 4.2.2.3.2 Nozzle chamber corrections

There were two problems in the air flow measuring system. The first was the Reynolds number test range for flow through the nozzles. Since the system was based on ANSI/ASHRAE Standard 51-1985, the Re through the nozzle should be greater than 12,000 for the minimum flow as a criteria in this Standard. (The formula used to calculate the discharge coefficient, Cd, was for Re greater than 12,000.) However due to low flow rates at the lower pressures, some of the measurements always occurred below the minimum Re specified even for the smallest nozzle (1/2"). To find a calibration method, the Cd curve produced by the equation given in the Standard with $\beta=0$ ( $\beta$ is the ratio of the nozzle exit diameter to the approach duct diameter) at lower Reynolds numbers was compared with another Cd curve with $\beta=0.2$ (ASME 1959). This was applicable for a Reynolds number range of 2500 to 12,000. It was found that these two curves are very close in the Re range of 2500 to 12,000 as shown in Figure 4.2.6. Therefore the error introduced by extending the previous curve to the lower Reynolds number range is not very significant (about $2 \%$ difference at most) compared to the second problem.

## Cd Curve for Wide Re Range



Figure 4.2.4. Comparison of Nozzle Discharge Coefficient Variation over Wide Re Range

Note: Extrapolating the Cd curve (for chamber $\beta=0$ ) for $\operatorname{Re}>12000$ range into lower Re range ( $\mathrm{Re}<12000$ ) would not introduce significant error since it is very close to another Cd curve with $\beta=0.2$ which is applicable over the lower Re range.

The second problem involved leaks in and through the metal nozzle chamber. A group of rotameters (Dwyer RMC-102\&103, +-2\% accuracy) was used to check the flow rate of the nozzle chamber outlet. There were significant differences between the nozzle readings and the rotameter readings especially for the smaller nozzles. The problem was later found to be caused by two leaks in the nozzle chamber. One was an "external leak", denoted $\mathrm{Q}_{\text {el }}$, which occurred at the corners of the chamber where an airtight seal had not been produced. The second one called the "internal leak", denoted $\mathrm{O}_{\mathrm{i}}$, was caused by leakage through the foam plugs which were used to seal the nozzles not being used and by leakage through a defect in a weld in the metal chamber. Thus the data previously obtained could not be used until the error from the flow through these leaks could be determined and taken into account.

It was assumed that these two leaks could be modeled by the power law, which led us to use pressurization tests on the two openings individually. Figure 4.2.5 contains data used in the calibration of these two leaks. The external leak, $\mathrm{Q}_{\text {el }}$, is a function of the pressure difference, $\mathrm{P}_{\text {el }}$, between the chamber on the downstream side of the nozzle and the room pressure. The expression to represent this leakage is:

$$
Q_{e l}=0.6314 * P_{\text {el }}^{0.6445}
$$

with $r^{2}=0.9593$ and C.V. $=6.2796 \%$ ( $Q$ unit is $c f m, P$ unit is in.wg). The internal leak $Q_{i i}$, is a function of pressure drop, $P_{n}$, across the testing nozzle plane. This leakage was represented by the relationship:

$$
\mathrm{Q}_{\mathrm{il}}=15.782 * \mathrm{P}_{\mathrm{n}}^{0.8734}
$$

with $\mathrm{r}^{2}=0.9935$ and C.V. $=3.789 \%$.
The relationship between the uncorrected nozzle flow, $Q_{n}$, rotameter flow, $Q_{r}$, (which is the actual flow amount entering the air plenum) and the "external" and "internal" leaks $\mathrm{C}_{\mathrm{i}}$ and $\mathrm{C}_{\text {el }}$ is:

$$
Q_{r}-\left.Q_{n}\right|_{\Delta P_{\text {mexzs }}}+\left.Q_{i l}\right|_{\Delta P_{\text {moztio }}}-\left.Q_{\theta 1}\right|_{\Delta P_{a l}}
$$

These two leaks should be individually included in data corrections according to the referred pressure drop. All the pressurization flow data in Appendices G-J have been corrected for these two leaks.


Figure 4.2.5 External and internal leak calibrations of the nozzle flow measuring chamber

### 4.2.2.3.3 Statistical Techniques

The data collected were fit to the theoretical nonlinear model:

$$
\begin{equation*}
Q-C_{1} \cdot\left[\sqrt{C_{2}^{2}+C_{3} \cdot \Delta P}-C_{2}\right] \tag{4.3}
\end{equation*}
$$

where:

$$
\begin{array}{ll}
\mathrm{C}_{1}=\mathrm{A} & \mathrm{~m}^{2} \\
\mathrm{C}_{2}=\mathrm{BZv/2KD}_{\mathrm{h}}{ }^{2} & \mathrm{~m} / \mathrm{s} \\
\mathrm{C}_{3}=2 / K \rho & \mathrm{~m}^{3} / \mathrm{kg}
\end{array}
$$

Each coefficient has a clear physical meaning in the model. The $\mathrm{C}_{1}$ is the equivalent sectional area, $C_{2}$ is a constant involving geometry and minor loss $K$, and $C_{3}$ is proportional to the inverse of $K$, which may be thought as a friction indicator.

The flow, Q , and the pressure drop, $\Delta \mathrm{P}$, were used as the dependent and independent variables respectively to statistically determine the three coefficients. It is obvious that these three coefficients (parameters) have a nonlinear relationship.

The statistical technique used to solve the nonlinear regression was the SAS NLIN method (SAS 1985, section of NLIN) with the Marquardt option. This option was chosen because it is one which appears to work well in many circumstances and was a practical choice (Draper and Smith 1966, pp. 263-273; SAS 1985, section of NLIN). In this nonlinear regression package, the grid ranges for the relevant parameters need to be provided. The grid for $\mathrm{C}_{2}$ was set from 0 to 10 by a step of 0.2 , and $\mathrm{C}_{3}$ from 0.6 to 1.7 by a step of 0.05 . Different ranges and steps were tested. The results produced after these changes insignificantly different. This indicated that once the dependency problem was solved, the results were unique. Discussion about the dependency problem is contained in the Results section. A similar technique was used to determine the constants $\mathrm{C}_{1}$ and $\mathrm{C}_{3}$ for the building component testing.

## CHAPTER 5 - RESULTS / DISCUSSION / CONCLUSIONS

### 5.1 LITERATURE DATABASE

The data from the literature were grouped by components and the effective leakage area for each citation was calculated as described in Section 4.1 at 4 Pa using a discharge coefficient of 1.0. The ela data and supporting material for each citation are contained in Appendix B. (A general shorthand notation throughout the project was used to identify the components and if there was weatherstripping applied. A "W" as the last letter in the component label indicates the presence of weatherstripping and "NW" indicates no weatherstripping.)

The minimum, maximum and best estimate ela values were determined for each component. A summary of the data is contained in Table 5-1. There was considerable variation in the units which were used to provide the basis for the flow or areas reported (eg. $\mathrm{cm}^{2}$ per: house, each unit, meter of sash, meter of crack, $\mathrm{m}^{2}$ of component area). Conversions between these units were not attempted unless there was suificieni infümatioun given in the ieference to make a conversion. In genera! the units used in the summary were those used by the majority of the data sources.

The best estimate selected from the data was a weighted estimate based upon the number of samples, the age of the data, the "quality" of data (a "best estimate" versus a measured value), and if independent sources predicted similar values. There was somewhat of a problem in determining the best estimate for some of the components since there was considerable overlap of the sources with no independent replication. For several of the cases the "best estimate" was taken as that assumed by the original source.

The best estimate for cases in which there were values given before and after sealing (eg. chimney) was assumed to be an average of multiple replications of the differences and/or direct measurements.


#### Abstract

Also contained in Table 5-1 are the values in the 1989 ASHRAE Handbook Fundamentals Chapter 23 Table 3 Effective Leakage Area of Building Components also calculated at a reference pressure of 4 Pa . The new table greatly expands the table in the HOF. There is considerable similarity between the best estimate values selected from the two sources. The selection of the best estimate values to report were made without observation of the values contained in the ASHRAE Handbook Fundamentals and thus were independent of those numbers. The similarity of the two sets of data is indicative of the use of the same data and in many cases the data were identical. It should be noted that although many entries in the table are unchanged, they have been rigorously reviewed and checked for quality and consistency.


One of the most significant differences is in the data for windows. This is due to the units used as a basis for the ela. A major project to investigate the air leakage of installed windows was done by Weidt et al (1979). The variation of the
performance of a number of windows was demonstrated based upon the air leakage. Large shifts in relative performance of different types of windows was identified based on which expression of leakage was used. The air leakage rates were calculated using three different ways: per linear foot of crack, per square foot of window sash area, and per square foot of ventilating area. Since standards and specifications are based on a per linear measure of crack calculation this unit was the basis used for this project. Exceptions to this are the cases of the awning windows and window framing in which there was insufficient information to determine the leakage based upon a length of crack.

It was initially anticipated that there would be some differences that could be identified between the different methods of reporting air leakage, the reference pressures and discharge coefficients selected for the testing and the different testing methods. No significant differences could be detected due to the scatter and insufficient number of data. While this scatter may be due to the anticipated differences in testing and reporting, the results of the round robin testing of the fan pressurization devices suggests that the errors in the devices introduce similar or greater errors.

It should be noted that although the number of different building components listed in the table is greater than previously identified, there are still gaps in the data for some groupings of components which need estimates (or better estimates) of leakage values. There are currently several significant projects underway to obtain the leakage of ductwork. Since these data were not available, they were not included in this report. Other components which need data (or better additional data) include: building construction joints (joints of dissimilar materials like masonry and wood or insulating board and wood; sole plate/baseboard; band joists; building corner joints; butt joints of sheathing, etc), window and door framing in masonry and wood wall construction, and the combined effects of air infiltration barriers and vapor retarders.


TABLE 5-1 Summary of Effective Leakage Areas from Literature ( cm 2 at $4 \mathrm{~Pa}, \mathrm{Cd}=1$ )

|  | Units | RP-438 |  |  | Chapter 23 - Table 31989 HOF |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | $\begin{aligned} & \hline \hline \text { Best } \\ & \text { Estimate } \end{aligned}$ | Minimum | aximum | Best Estimate | Minimum | Maximúm | Units if Different |
| Clay Brick cavity wall - finished | cm2/m2 | 0.68 | 0.05 | 2.3 |  |  |  |  |
| Precast Concrete Panel | cm2/m2 | 1.2 | 0.28 | 1.65 |  |  |  |  |
| WIANW Window: Awning NotWS | cm2/m2 | 1.6 | 0.8 | 2.4 | 1.6 | 0.8 | 2.4 |  |
| WIAW Window: Awning w weatherstripping | cm2/m2 | 0.8 | 0.4 | 1.2 | 0.8 | 0.4 | 1.2 |  |
| WICW Windows: Casement w weatherstripping | cm2/mc | 0.24 | 0.1 | 3 | 0.8 | 0.4 | 1.2 | cm2/m2 |
| WICW Windows: Casement w/o ws | cm2/mc | 0.28 |  |  | 1.6 | 0.8 | 2.4 | $\mathrm{cm} 2 / \mathrm{m} 2$ |
| WIDHSW Windows: Double Horiz Slider w/o ws | cm2Amc | 1.1 | 0.019 | 3.4 | 5.2 | 2.8 | 7.6 | $\mathrm{cm} / \mathrm{m} 2$ |
| WIDHSW Windows: Dbl Hor Sldr - wood w w/s | cm2/mc | 0.55 | 0.15 | 1.72 | 2.6 | 3.8 | 1.4 | cm2/m2 |
| WIDHSW Windows: Dbl Hor Sldr - al w w/s | cm2/mc | 0.72 | 0.58 | 0.8 |  |  |  |  |
| WIDHW Windows: Double Hung w/o ws | cm2/mc | 2.5 | 0.86 | 6.1 | 6 | 3.2 | 8.8 | cm2/m2 |
| WIDHW Windows: Double Hung w ws | cm2/mc | 0.65 | 0.2 | 1.9 | 3 | 1.6 | 4.4 | cm2/m2 |
| WIDHW Wdows: Dbl Hung w/o ws, w storm | cm2/me | 0.97 | 0.48 | 1.7 |  |  |  |  |
| WIDHW Wdows: DJ Hung w ws, w storm | cm2/mc | 0.79 | 0.44 | 1 |  |  |  |  |
| WIDHW Wdows: Dbl Hung w ws, w pressurized trackst | cm2/mc | 0.48 | 0.39 | 0.56 |  |  |  |  |
| WIFM Windows: Framing - Masonary - uncaulked | cm2/m2 | 6.5 | 5.7 | 10.3 | 6.5 | 5.7 | 10.3 |  |
| WIFM Windows: Framing - Masonary - caulked | cm2/m2 | 1.3 | 1.1 | 2.1 | 1.3 | 1.1 | 2.1 |  |
| WIFW Windows: Framing - Wood - uncaulked | cm2/m2 | 1.7 | 1.5 | 2.7 | 1.7 | 1.5 | 2.7 |  |
| WIFW Windows: Framing-Wood - caulked | cm2/m2 | 0.3 | 0.3 | 0.5 | 0.3 | 0.3 | 0.5 |  |
| WIJ Windows: Jalousie | cm2/louvre | 3.38 |  |  |  |  |  |  |
| WIL Windows: Lumped | cm2/ms | 0.471 | 0.009 | 2.06 |  |  |  |  |
| WISHSW Windows: Single Horizontal Slider | cm2/ms | 0.67 | 0.2 | 2.06 | 1.8 | 0.9 | 2.7 | cm2/m2 |
| WISHSW Windows: Single Horizontal Slider w/o ws |  |  |  |  | 3.6 | 1.8 | 5.4 | cm2/m2 |
| WISHSW Windows: Single Hor Slider - aluminum | cm2/ms | 0.8 | 0.27 | 2.06 |  |  |  |  |
| WISHSW Windows: Single Hor Sldr - wood | cm2/ms | 0.44 | 0.27 | 0.99 |  |  |  |  |
| WISHSW Windows: Single Hor Sidr - wood clad | cm2/ms | 0.64 | 0.54 | 0.81 |  |  |  |  |
| WISHW Windows: Single Hung - WS | cm2/ms | 0.87 | 0.62 | 1.24 | 2.2 | 1.8 | 2.9 | cm2/m2 |
| WISHW Windows: Single Hung - non ws |  |  |  |  | 4.4 | 3.6 | 5.8 | cm2/m2 |
| WISILL Windows: Sill | cm2/mc | 0.21 | 0.139 | 0.212 |  |  |  |  |
| WIST Windows: Storm Inside - heat shrink | cm2/ms | 0.018 | 0.009 | 0.018 |  |  |  |  |
| WIST Windows: Storm Inside - rigid w magnetic seals | cm2/ms | 0.12 | 0.018 | 0.24 |  |  |  |  |
| WIST Windows: Storm Inside - flex sheets w mech seals | $\mathrm{cm} 2 / \mathrm{lms}$ | 0.154 | 0.018 | 0.833 |  |  |  |  |
| WIST Windows: Storm Inside - rigid w mechanical seals | cm2/ms | 0.4 | 0.045 | 0.833 |  |  |  |  |
| WISTM Windows - Storm Outside (Storm only) |  |  |  |  |  |  |  |  |
| WISTM Windows - Storm Outside - pressurized track | cm2/mc | 0.528 |  |  |  |  |  |  |
| WISTM Windows - Storm Outside 2 track | cm2/mc | 1.23 |  |  |  |  |  |  |
| WISTM Windows - Storm Outside - 3 track | cm2/mc | 2.46 |  |  |  |  |  |  |

NOTE: Units are cm2 per
$\mathrm{m} 2=$ square meters of surface area
Imc = lineal meter of crack
Ims = lineal meter of sash
m = lineal meter

### 5.2 DETERMINATION OF INDEPENDENT $\mathrm{C}_{1}, \mathrm{C}_{2}$ AND $\mathrm{C}_{3}$ PARAMETERS

Initially the statistical technique attempted was to make a direct nonlinear regression based on equation [4.3]. The three-parameter nonlinear regression results showed that the estimated constants were not stable as different grids (and therefore search starting points) were selected. In addition, the standard error for one of the parameters always went to zero. This numerical instability indicated that the direct three-parameter regression was over parameterized. When the model was mathematically analyzed further it was recognized that the constants are not independent of each other and actually only two independent parameters existed as in the following form:

$$
Q-\sqrt{\left(C_{1} C_{2}\right)^{2}+C_{1}^{2} C_{3} \Delta P}-C_{1} C_{2}
$$

Thus there was a problem in attempting to determine the values for $\mathrm{C}_{1}, \mathrm{C}_{2}$ and $\mathrm{C}_{3}$ when there were only two independent parameters.

For well-defined cracks the section area of the crack is known. If the area is treated as an input, the model in the form [4.3] will automatically be reduced to a two-parameter non-linear regression model. When this was done for the well-defined cracks the resulting values of $\mathrm{C}_{2}$ and $\mathrm{C}_{3}$ were very stable. This is a significant improvement over a previous approach which needed the assumption of $K=1.5$ (Chastain et al. 1987).

The original desire was to apply the model to openings in real building components with unknown crack geometries. None of the three parameters would be known in this case. Therefore a relationship was needed to find one of the three coefficients either theoretically or numerically. It was found that the theoretical relationship(s) derived from equation [4.3] are dependent on the equation and therefore the dependency problem could not be solved.

Thus an attempt was made to determine a numerical relationship between the parameters.

The power equation is presently the most common expression used to represent the $\mathrm{Q}-\Delta \mathrm{P}$ curve. It is also independent of the KY model since it is not theoretically based. Therefore it is a good reference to numerically relate the constants in the new model. The relationship of $\mathrm{C}_{2}$ in the KY model and the n in the power model were investigated.

## For the KY model case:

The model is defined by Eqn 4.3

$$
Q-C_{1}\left[\left(C_{2}^{2}+C_{3} \Delta P\right)^{0.5}-C_{2}\right]
$$

where:

$$
\begin{array}{ll}
\mathrm{C}_{1}=\mathrm{A} & \mathrm{~m}^{2} \\
\mathrm{C}_{2}=\mathrm{BZv/2KD}_{\mathrm{h}} & \mathrm{~m} / \mathrm{s} \\
\mathrm{C}_{3}=2 / \mathrm{K} \rho & \mathrm{~m}^{3} / \mathrm{kg}
\end{array}
$$

When $\mathrm{C}_{2}=0$ (i.e. flow length, $\mathrm{Z}=0$ ), the equation reduces to:

$$
Q-C_{1} \sqrt{C_{3} \Delta P}
$$

and therefore $Q \propto \sqrt{ } \Delta P$.
The first derivative, $d Q / d \Delta P$ for the model is given by:

$$
\frac{d Q}{d \Delta P}-\frac{C_{1} C_{3}}{2 \sqrt{C_{2}^{2}+C_{3} \Delta P}}
$$

Figure 5.1 illustrates a typical plot of $d Q / d \Delta P$ vs. $\Delta P$. When $C_{2}$ is increasing, the derivative term, $d Q / d \Delta P$, is approaching a constant. When $C_{2}$ increases to a large number, say $C_{2}>10 \mathrm{~m} / \mathrm{s}$ in this plot, it can be assumed that the derivative $d Q / d \Delta P$ will not significantly change with $\Delta P$. This means that the slope of $Q$ vs. $\Delta P$ is approaching to a constant, that is,

When $C_{2}^{2}>C_{3} \Delta P, \quad \frac{d Q}{d \Delta P} \sim$ constant $\quad \therefore \quad Q \propto \Delta P$

For the power model case:

$$
Q-C \Delta P^{n}
$$

When $\mathrm{n}-0.5$,
$Q-C \sqrt{\Delta P}$
$\mathbf{Q} \propto \sqrt{\Delta \mathbf{P}}$

When $n-1$,
$Q-C \Delta P$,
$Q \propto \Delta P$

Comparing these two model cases suggests that the function of n in the power model plays a similar role as $\mathrm{C}_{2}$ in the new model. This observation implies that there may be a numerical relationship between the $C_{2}$ and $n$.

Using data from well-defined cracks enabled the establishment of this
calibration relationship since the sectional area of crack, $\mathrm{C}_{1}$, was known and stable $\mathrm{C}_{2}$ and $C_{3}$ values could be obtained by the two-parameter regression. Meanwhile the power model was also applied to fit the $\mathrm{Q}-\Delta \mathrm{P}$ data from the same well-defined cracks to produce the corresponding $C$ and $n$. Figure 5.2 is the plot of $C_{2}$ vs. $n$ for 19 welldefined cracks. This demonstrates a strong linear correlation between $\mathrm{C}_{2}$ and n represented by:

$$
C_{2}-11.85(n-0.5)
$$

## $\mathrm{dQ} / \mathrm{dP}$ vs. P when C 2 increases



Figure $5.1 \mathrm{dQ} / \mathrm{d} \Delta \mathrm{P}$ vs. $\Delta \mathrm{P}$ relationship in the $K Y$ model

## C2 vs. n plot for the same cracks



Figure $5.2 \quad C_{2}$ in the $K Y$ model vs. $n$ in the power model

It is recognized that an assumption is needed in order to apply this numerical relationship to real building components. It is assumed that there is no significant difference in the $\mathrm{Q}-\Delta \mathrm{P}$ relationship between the well-defined cracks and building components. This assumption had been recognized and used in previous research (Hopkins and Hansford 1974, Etheridge 1977 and Chastain et al.1987).

It is now easy to apply the model to evaluate the leakage performance of real building components. In summary, the following are the procedures for well-defined cracks and building components respectively:

## For well-defined cracks:

Input the known crack sectional area $C_{1}$ into the model equation [4.3] to obtain the other two constants $C_{2}$ and $C_{3}$ by a two-parameter nonlinear regression.

## For building components:

1) Obtain the regression exponent $n$ based on the power model.
2) Calculate the $\mathrm{C}_{2}$ from the numerical relation of equation 5.3.
3) Substitute the $\mathrm{C}_{2}$ value into the new model [4.3] to obtain the other two constants $C_{1}$ and $C_{3}$ by a two-parameter nonlinear regression.

To determine the two estimated parameters, the least squares concept is still used. The error sum of squares, $\Phi\left(\mathrm{C}_{2}, \mathrm{C}_{3}\right)$ or $\Phi\left(\mathrm{C}_{1}, \mathrm{C}_{3}\right)$ for the nonlinear model and the given data is:

$$
\begin{aligned}
& \phi\left(C_{2}, C_{3}\right)=\sum_{K-1}^{m}\left[Q_{K}-C_{1}\left(\sqrt{C_{2}^{2}+C_{3} \Delta P_{K}}-C_{2}\right)\right]^{2} \\
& \phi\left(C_{1}, C_{3}\right)=\sum_{K-1}^{m}\left[Q_{K}-C_{1}\left(\sqrt{C_{2}^{2}+C_{3} \Delta P_{K}}-C_{2}\right)\right]^{2}
\end{aligned}
$$

Where $\left(Q_{k}, \Delta P_{k}\right)$ is a group of $m$ observations and $1 \leq K \leq m$. The parameter with " $\cap$ " means that the parameter is known.

For example, take the equation of $\Phi\left(\mathrm{C}_{2}, \mathrm{C}_{3}\right)$ to obtain the least squares estimates of $C_{2}$ and $C_{3}$. We need to differentiate this equation with respect to $C_{2}$ and $\mathrm{C}_{3}$ respectively. This procedure provides a system of equations with two independent equations and two unknown parameters to be estimated. The equation for $\Phi\left(\mathrm{C}_{1}, \mathrm{C}_{3}\right)$ can be handled in a similar way. Finding the estimates by solving the system of equations is very complicated and iterative methods must be employed. There are several methods available for obtaining the parameters by routine computer
calculations. The Marquardt's method is one which appears to work well in many circumstances thus it is a practical choice (Draper and Smith 1966, pp. 263-273). The detailed information about the detail of solving the system of equations involving the Marquardt iterative methods can be found in Draper and Smith (1966). Commercial computer software is also available to solve nonlinear regressions. The SAS NLIN programs are very powerful in handling nonlinear regression (SAS 1985). This program was chosen with the Marquardt's method option to estimate the two parameters in this study.

All iterative procedures require initial values of the parameters to be selected. Applying different grids in these cases should not significantly impact the results. It is also clear from their physical meanings that $C_{1}>0, C_{2}>0$ and $C_{3}>0$. Hence these constants are set to be semi-free-positive coefficients to be determined by the regression.

From the regression of a specified $\mathrm{Q}-\Delta \mathrm{P}$ data set for building components, the three coefficients $\mathrm{C}_{1}, \mathrm{C}_{2}$ and $\mathrm{C}_{3}$ are then obtained. If the air density value in the test is known, the $K$ value can be calculated from $C_{3}$ directly. The $B \cdot Z /\left(R e \cdot D_{h}\right)$ vs. $\Delta \mathrm{P} /\left(1 / 2 \rho \overline{\mathrm{~V}}^{2}\right)$ relationship can also be determined by using the original $\mathrm{Q}-\Delta \mathrm{P}$ data set and the corresponding regression results of $C_{1}, C_{2}$ and $C_{3}$, i.e.

$$
\begin{aligned}
& \frac{B Z}{R e D_{h}}-\frac{B Z}{\left(\frac{\bar{V} D_{h}}{v}\right) D_{h}}-\frac{\frac{B Z v}{D_{h}^{2}}}{\frac{Q}{A}}-\frac{A \frac{B Z v}{2 K D_{h}^{2}}}{\frac{2}{k \rho}} \frac{4}{\rho Q}-\frac{C_{1} C_{2}}{C_{3}} \frac{4}{\rho Q} \\
& \frac{\Delta P}{\frac{1}{2} \rho \bar{V}^{2}}-\frac{2 \Delta P}{\rho\left(\frac{Q}{A}\right)^{2}}-A^{2} \frac{2 \Delta P}{\rho Q^{2}}-C_{1}^{2} \frac{2 \Delta P}{\rho Q^{2}}
\end{aligned}
$$

This means each $\Delta P /\left(1 / 2 \rho \bar{V}^{2}\right)$ value and the corresponding $B \cdot Z /\left(\operatorname{Re} \cdot D_{h}\right)$ value can be obtained directly from the $Q-\Delta P$ data set using the calculation and regression products $\mathrm{C}_{1}, \mathrm{C}_{2}$ and $\mathrm{C}_{3}$. In this technique, no assumption has to be made about the dimensions of the crack in order to get $B \cdot Z /\left(\operatorname{Re} \cdot D_{h}\right)$ and $\Delta P /\left(1 / 2 \rho \bar{V}^{2}\right)$ values. This is a significant improvement over the previous methods of Etheridge (1977) and Chastain et.al (1987).

### 5.3 FIT OF NEW EQUATION

### 5.3.1 Measure of Goodness-of-Fit

When a new model is proposed, it is essential to quantify how good it fits measured data. The power model is based on a linear regression of a log-transform of the $\mathrm{Q}-\Delta \mathrm{P}$ data. The KY model is derived from the nonlinear $\mathrm{Q}-\Delta \mathrm{P}$ relationship. Therefore a common measure should be chosen to judge their statistical performance.

Often the success of the analytical exercise depends on the proper choice of the quantitative criteria used to determine the quality of the fitted model. The coefficient of determination, $\mathrm{R}^{2}$, is a commonly used measure; however it is often used improperly and is frequently misunderstood as a measure of the fit of the regression line (Raymond 1990). By definition,

$$
R^{2}=\frac{S S_{\text {Reg }}}{S S_{\text {Total }}}-\frac{\sum_{i-1}^{n}\left(\hat{y}_{1}-\bar{y}\right)^{2}}{\sum_{i-1}^{n}\left(y_{1}-\bar{y}\right)^{2}}-1-\frac{S S_{\text {Res }}}{S S_{\text {Total }}}
$$

where:

| $\mathrm{SS}_{\text {Rea }}$ | is the regression sum of square, |
| :--- | :--- |
| $\mathrm{SS}_{\text {Res }}$ | is the residual sum of square, |
| $\mathrm{SS}_{\text {Total }}$ | is the total sum of square, |
| $\mathrm{y}_{\mathrm{i}}$ | is the predicted y for the corresponding independent variable $x_{i}$. |

This coefficient of determination represents the proportion of variation in the response data that is explained by the model. Clearly $0 \leq R^{2} \leq 1$. Raymond (1990) illustrated that $\mathbf{R}^{2}$ can appear to be artificially high either because the slope of the regression is large or because the spread of the regressor data $x_{1}, x_{2}, \ldots x_{n}$ is great.

The coefficient of variation, CV, is a less common criterion but is a reasonable one for representing quality of fit and measuring spread of noise around the regression line. The CV is defined as:

where n is the number of observations.

The CV is interpreted as the residual estimate of error standard deviation, measured as a percent of the average response values. The CV was chosen to be the criterion used for the comparison of the models.

### 5.3.2 Overall Fit

A summary of the results of the unweighted power model and the KY model applied to the data obtained from the well-defined cracks and building components are presented in Table 5.3.1 and Table 5.3.2 respectively. The arithmetic average CV value is $1.86 \%$ for the power model and $0.92 \%$ for the KY model for the same welldefined cracks. The CV value was lower for the new model in 22 of the 26 cases tested. For the building components tests, the CV is $2.63 \%$ based on the power model and $1.93 \%$ on the KY model. The CV was lower for the new model in 10 of the 18 cases tested.

These results indicate that the KY model statistically fits as well as the power model. However the efficacy of the new equation is measured in the ability to determine additional information about the openings. The model can predict the actual equivalent sectional area of any type of crack or openings in building components and other parameters such as the minor loss coefficient. This is considered as another factor that measures and describes the crack leakage performance. In other words, the model should provide a reasonable prediction of the sectional area chracteristics of the opening. The summary table of well-defined cracks also indicates that the average relative error between the predicted section areas and the section areas is $5.98 \%$.

It should be noted that the five $K$ values predicted which are out of the range previously suggested for well defined openings (1.2-2.3) are for component wall sections. This indicates that as expected the minor losses for complex openings is larger than well defined openings. Four of the five cases are associated with cases which substantially underestimate the flow at 5 Pa . This indicates that further investigation is needed on the relationship between $\mathrm{C}_{2}$ and n . The equation developed relating these two was taken from data obtained from the well defined openings.

Comparisons were also made between the observed air flow at 5 Pa and the flow predicted for that pressure difference by both models. The predicted air flow using the new model was closer in 6 of the 26 well-defined crack cases and 3 of the 18 building component cases. It should be noted here that the measurement and computation of the observed air flow at 5 Pa has the following problems: 1) It has a higher inaccuracy of flow measurement than the higher pressure difference readings. 2) For most crack flow measurements, the 5 Pa pressure drop corresponds to the lowest Re (which is often lower than 12,000), therefore the discharge coefficient, $C_{d}$, which had the largest error had to be chosen to compute the flow. 3) Its reading may also be easily influenced by the surroundings (especially in field measurements). Because of these reasons, a wide pressure range from 5 Pa to 75 Pa is used to include more high accuracy data in order to obtain a general regression equation to extrapolate to the lower pressure data. Therefore the errors between the observed and the predicted data at 5 Pa are sometimes higher in the new model than that in the

Table 5.3.1-Summary of Well Defined Cracks


Table 5.3.2 - Summary of Building Components

| Component | Power Model |  |  |  |  |  |  | New Model |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | C <br> $\left(m^{+}+3 t^{\circ} P_{x}+1\right)$ | n | C.V. (\%) | $\begin{gathered} \text { ELA } \\ (\mathrm{at} 4 \mathrm{~Pa}) \\ \left(\mathrm{m}^{\wedge} 2\right) \end{gathered}$ | $\begin{gathered} \text { Qpredi. } \\ \text { (at5 Pa) } \\ \left(m^{\wedge} / / s e c .\right) \end{gathered}$ | $\begin{aligned} & \text { Qmeasured } \\ & \text { (at } 5 \mathrm{~Pa}) \\ & \left(\mathrm{m}^{\wedge} / 3 / \mathrm{sec} .\right) \end{aligned}$ | Error of Q <br> (at 5 Pa ) <br> (\%) | C1 $\left(m^{\wedge} 2\right)$ | $\begin{aligned} & \mathrm{C} 2 \\ & (\mathrm{n} / \mathrm{s}) \end{aligned}$ | $\begin{gathered} C 3 \\ \left(\mathrm{~m}^{\wedge} 3 \mathrm{~kg}\right) \end{gathered}$ | c.V. (\%) | ELA <br> (at 4 Pa ) <br> ( $\mathrm{m}^{\wedge} 2$ ) | $\begin{gathered} \text { Q predi. } \\ \text { (at } 5 \mathrm{~Pa} \text { ) } \\ \left(\mathrm{m}^{\wedge} 3 / \mathrm{sec} .\right) \end{gathered}$ | Error of Q <br> (at 5 Pa ) <br> (\%) | A predi. $\left(\mathrm{cm}^{\wedge} 2\right)$ | K |
| CO-1 | 1.01E-05 | 1.0145 | 10.521 | 1.60E-05 | 5.17E-05 | 0.00004 | 29.23 | 1.260E-04 | 6.04 | 1.35 | 2.909 | 2.11E-05 | 6.74E-05 | 68.55 | 1.26 | 1.23 |
| CO-2 | 8.19E-04 | 0.5554 | 3.469 | 6.86E-04 | $2.00 \mathrm{E}-03$ | 0.00189 | 5.93 | 3.940E-04 | 0.71 | 7.11 | 2448 | 7.13E-04 | 2.09E-03 | 10.37 | 3.94 | 0.23 |
| CO3 | 3.12E-04 | 0.6720 | 0.935 | 3.07E-04 | 9.20E-04 | 0.00090 | 2.24 | 7.800E-04 | 2.01 | 1.07 | 1.159 | 2.64E-04 | 8.22E-04 | -8.62 | 7.80 | 1.56 |
| CO-4 | 274E-04 | 0.6879 | 2.237 | 2.68E-04 | $8.03 \mathrm{E}-04$ | 0.00078 | 2.92 | 5.810E-04 | 8.01 | 135 | 1.803 | 239E-04 | 7.41E-04 | -5.04 | 5.81 | 1.23 |
| ca-5 | 3.49E-04 | 0.6459 | 0.678 | 3.31E-04 | 9.87E-04 | 0.00098 | 0.71 | 7.730E-04 | 1.78 | - 1.05 | 1.770 | 280E-04 | 8.67E-04 | -11.54 | $\because 73$ | 1.58 |
| CO6 | 2.12E-04 | 0.7167 | 0.534 | 2.22E-04 | 6.72E-04 | 0.00068 | 1.80 | 7.590E-04 | : 1.61 | 0.91 | 1.557 | 1.83E-04 | $5.77 \mathrm{E}-04$ | -12.51 | 7.59 | 1.83 |
| C0.7 | 229E-04 | 0.7081 | 0.777 | 236E-04 | 7.13E-04 | 0.00070 | 1.92 | 7210E-04 | \$249 | 4.01 | 1.450 | 1.98E-04 | $6.23 \mathrm{E}-04$ | -11.00 | 7.21 | 1.55 |
| CO-8 | 1.70E-04 | 0.7047 | 1.287 | 1.75E-04 | 528E-04 | 0.00050 | 5.69 | 5.000E-04 | :37 | 1.09 | 0.928 | $1.53 \mathrm{E}-04$ | $4.78 \mathrm{E}-04$ | -4.33 | 5.00 | 1.53 |
| co-9 | 1.67E-05 | 0.8518 | 5.119 | 211E-05 | 6.58E-05 | 0.00005 | 31.58 | 1.080E-04 | 4.15 | 1.10 | 2771 | 209E-05 | 6.66E-05 | 33.23 | 1.08 | 1.51 |
| ca-10 | 222E-04 | 0.7393 | 1.649 | $2.40 \mathrm{E}-04$ | 730E-04 | 0.00068 | 730 | 8.040E-04 | 2.84 | 1.08 | 1.315 | 208E-04 | 6,56E-04 | 3.53 | 8.04 | 1.57 |
| C0.11 | $1.48 \mathrm{E}-04$ | 0.7318 | 1.811 | 1.58E-04 | 4.81E-04 | 0.00045 | 6.80 | 5.130E-04 | 273 | 1.08 | 1.337 | 137E-04 | $4.31 \mathrm{E}-04$ | -4.11 | 5.13 | 1.57 |
| CO-12 | 933E-05 | 0.6758 | 1.004 | 9:23E-05 | 2.77E-04 | 0.00028 | -1.12 | 2.620E-04 | 2.13 | 0.94 | 1.795 | 7.62E-05 | $238 \mathrm{E}-04$ | -14.92 | 2.62 | 1.77 |
| CO-13 | 6.62E-05 | 0.8962 | 5.858 | 8.89E-05 | 2.80E-04 | 0.00022 | 27.31 | 5.150E-04 | 4.74 | 1.17 | 1.466 | 939E-05 | 2.99E-04 | 36.11 | 5.15 | 1.42 |
| CO.14 | $3.81 \mathrm{E}-04$ | 0.7687 | 1.741 | 4.29E-04 | 1.31E-03 | 0.00131 | 0.22 | 2.567E-03 | 3.20 | 0.57 | 2.685 | 3,37E-04 | $1.07 \mathrm{E}-03$ | -18.09 | 25.67 | 2.92 |
| CO-15 | $6.00 \mathrm{E}-04$ | 0.7577 | 1.807 | 6.65E-04 | 2.03E-03 | 0.00211 | -3.73 | 3.982E-03 | 3.08 | 0.54 | 2.581 | 5.13E-04 | $1.64 \mathrm{E}-03$ | -22.45 | 39.82 | 3.08 |
| CO. 16 | 5.05E-03 | 0.6222 | 0.821 | 4.64E-03 | 1.37E-02 | $0.0133<$ | 3.05 | 8.463E-03 | 1.42 | 132 | 0.725 | 4.20E-03 | $1.28 \mathrm{E}-02$ | -3.86 | 84.63 | 1.26 |
| CO-17 | 7.7TE-04 | 0.7721 | 2345 | 8.78E-04 | 2.69E-03 | 0.00299 | -9.96 | 6.701E-03 | 3.20 | 0.43 | 2.648 | 6.71E-04 | 2.14E-03 | -28.30 | 67.01 | 3.87 |
| CO-18 | 1.10E-03 | 0.7745 | 4.713 | $125 \mathrm{E}-03$ | $3.83 \mathrm{E}-03$ | 0.00416 | . 8.03 | 1.416E-02 | 3.20 | 0.28 | 4.033 | 8.70E-04 | $2.79 \mathrm{E}-03$ | -32.92 | 141.60 | 6.40 |
| (Average) |  |  | 2.628 |  |  |  |  |  |  |  | 1.965 |  |  |  |  | 2.01 |

power model.
The plots in Figures 5-3-3 through 5-3-11 illustrate the curve performances of the power equation and the new dimensional flow equation. The pressurization data sets of the 18 building components are used as a sample. The conclusion gained from these plots is that the KY model curve fits the data as well and is very close to the power equation model.

The following symbols are used in Figure 5-3-3 to Figure 5-3-11:
**** 1: Average value of the three observed data points
------- 2: Power model predicted curve

- 3: New model predicted curve

Comparison of new model to power model: CO-3
 $\operatorname{crapp} 2+$ Pow row


Compariscin of new model to power model: CO-4
$P=$ Prestre differenoe $\mathrm{Pa} Q=$ Flow rate $\mathrm{m} 3 / \mathrm{s}$ a
Group 2-Power mpo forthal datas
Graw
 0


Figure 5-3-4. Model comparison on component 3 and 4

Comparison of new model to power model: CO-1




Comparison of new model to power model: CO-2 $\mathrm{P}=$ Prosare diferenoe Pa $\mathrm{Q}=$ Flow rate $\mathrm{m} 3 / \mathrm{s}$



Figure 5-3-3. Model comparison on component 1 and 2

Comparison of new model to power model: CO-5 $\mathrm{P}=$ Fressure diference $\mathrm{Pa} \quad \mathrm{Q}$ = Fiow rate $\mathrm{m} 3 / \mathrm{s}$ Grap 2-P Group 1-Otichal datay


Comparison of new model to power model: CO-6 $P=$ Prossurs diference $\mathrm{Pa} Q=$ Fiow rate $\mathrm{m} 3 / \mathrm{s}$ Group 2-Pow crap 1-arichel dates



Figure 5-3-5. Model comparison on component 5 and 6



Comparison of new model to power model: co-8 $P=$ Preesure determen Pa a, Frow rate m3/s Croup 2Power mpder $0=0.000770$ n= 0.7047



Figure 5-3-6. Model comparison on component 7 and 8

Comparison of new model to power model: CO-9 $\mathrm{P}_{\mathrm{n}}$ Prosere dilemence $\mathrm{Pa} Q=\mathrm{Flow}$ rato $\mathrm{m3} / \mathrm{s}$



Comparlison of new model to power modet CO-10 $P=$ Presare difteracos $P a \quad Q=$ Fiow rate m3/s



Figure 5-3-7. Model comparison on component 9 and 10


Figure 5-3-8. Model comparison on component 11 and 12

Comparison of new model to power model: CO-13 $P=$ Presure diference $\begin{gathered}\text { Pa } \\ \text { Group forlal dates }\end{gathered}$



Comparlsin of new model to power model: CO-18 $\mathrm{P}=\mathrm{Provare}$ cirlemoe Pa O . Frow rate $\mathrm{m} 3 / \mathrm{s}$



Figure 5-3-9. Model comparison on component 13 and 14

Comparison of new model to power model: CO-15 $P=$ Pressurs difference $\mathrm{Pa} \quad \mathrm{Q}=$ Flow rate $\mathrm{m} 3 / \mathrm{s}$



Comparison of new model to power model: CO-18 $\mathrm{P}=$ Preesure differumoe $\mathrm{Pa} \quad \mathrm{Q}=$. Flow rate $\mathrm{m} 3 / \mathrm{s}$



Figure 5-3-10. Model comparison on component 15 and 16

## Comparison of new model to power model: CO-17 




Comparison of new model to power model: CO-18 $P=$ Presurs differsone $P a$ an $Q_{a}$ Flow rate m3/a



Figure 5-3-11. Model comparison on component 17 and 18

### 5.3.3 Parallel and series analysis based on well-defined cracks

A comparison was made between two definitions given in Table 4-3 for the resistance to airflow: a) the inverse of the ELA (based on $\mathrm{Pa}=4$ ) and b) the inverse of the power regression equation coefficient $C$. The data obtained from the well-defined cracks were used to test the new and previous definitions to see which one worked better. The values of the ELA and C are from Table 5-3-1. (It should be recognized that the coefficients of leakage curves with different $n$ values have to be added to apply the parallel and series analogy. This incompatibility further reflects the problem of using nonhomogeneous equations however it is necessary in order to make these comparisons.)

In Table 5.3.3 the previous definition of crack resistance is used to compare the parallel flow theory based on the power equation model coefficient C. In Table 5.3.4 and Table 5.3.5 the new definition of resistance to airflow being the inverse of the ELA is used for checking parallel flow based on the ELA values calculated from the power and KY models respectively. The comparison between combining the two C coefficients versus the value of C obtained from the combined flow indicates a $12.5 \%$ overall average difference between them. This value is reduced to $8.7 \%$ and $8 \%$ when the ELA values from the power model or KY model respectively are used. The ELA and $C$ values used for each crack in these equations are those obtained from tests where each crack was run separately. Tables 5.3.6, 5.3.7 and 5.3.8 are similar to the previous three tables while being applied to the series flow application. The average differences are 13.8, 12.2 and $10.7 \%$ for the series application. From these six tables it is obvious that defining the inverse of the ELA as a first approximation of the crack resistance is more effective and yields a satisfactory prediction of parallel and series flow through cracks. Secondly, these examples show that the ELA value from the KY model is slightly more accurate than that from power model.

This theory may also be used to analyze the leakage performance of various building components which may have parallel or series connections as well as whole building structures.

Table 5.3.3 Using $R=1 / C$ as definition of crack resistance to check the parallel flow theory based on the power model coefficient C (C units: $\left.\mathrm{m}^{3} / \mathrm{h}(\mathrm{Pa})^{n}\right)$

| Cracks connected in <br> parallel | $C \Leftrightarrow C 1+C 2 \quad\left(\times 10^{-4}\right)$ | Relative <br> error (\%) |
| :--- | :--- | :--- |
| A@B1 | $3.21 \Leftrightarrow 0.57+3.66=4.23$ | 31.8 |
| A@E | $29 \Leftrightarrow 0.57+26.4=26.97$ | -7.0 |
| A@F1 | $67.9 \Leftrightarrow 0.57+64.1=64.67$ | -4.8 |
| B1@B2 | $7.77 \Leftrightarrow 3.66+3.06=6.72$ | -13.5 |
| B1@E | $26.4 \Leftrightarrow 3.66+26.4=30.06$ | 13.9 |
| B1@F1 | $73.3 \Leftrightarrow 3.66+64.1=67.76$ | -7.6 |
| F1@F2 | $139.2 \Leftrightarrow 64.1+62.4=126.5$ | -9.1 |
| Absolute Average |  | 12.5 |

Note: $\mathrm{C}=$ Coefficient from the power equation using the total flow through both cracks mounted in parallel.
C1 = Coefficient from power equation for crack 1 determined individually C2 $=$ Coefficient from power equation for crack 2 determined individually

Relative Er. $-\frac{\mathrm{C}-(\mathrm{C} 1+\mathrm{C} 2)}{\mathrm{C}} * 100$

Table 5.3.4 Using $R=1 / E L A$ as definition of crack resistance to check the parallel flow based on ELA values at 4 Pa from the power model (ELA units: $\mathrm{m}^{2}$ )

| Cracks connected <br> in parallel | ELA $\Leftrightarrow$ ELA $_{1}+$ ELA $_{2} \quad\left(\times 10^{-4}\right)$ | Relative <br> error $(\%)$ |
| :--- | :--- | :--- |
| A@B1 | $3.57 \Leftrightarrow 0.72+3.62=4.34$ | 21.6 |
| A@E $\cdot$ | $23.1 \Leftrightarrow 0.72+21.4=22.12$ | -4.2 |
| A@F1 | $53.7 \Leftrightarrow 0.72+51.2=51.92$ | -3.3 |
| B1@B2 | $7.54 \Leftrightarrow 3.62+3.09=6.71$ | -11 |
| B1@E | $23.1 \Leftrightarrow 3.62+21.4=25.02$ | 8.3 |
| B1@F1 | $58.1 \Leftrightarrow 3.62+51.2=54.82$ | -5.6 |
| F1@F2 | $109 \Leftrightarrow 51.2+50.4=101.6$ | -6.8 |
| Absolute Average |  | 8.7 |

Note: ELA $A_{1}$ is quoted from Table 5-3-1 for the power model with flow through crack 1.
$E L A_{2}$ is quoted from Table 5-3-1 for power model with flow through crack 2.
ELA was determined for flow through both the two cracks mounted in parallel.

Table 5.3.5 Using $R=1 / E L A$ as definition of crack resistance to check the parallel theory based on the KY model ELA values at 4 Pa . (ELA units: $\mathrm{m}^{2}$ )

| Cracks <br> connected in <br> parallel | $E L A \Leftrightarrow \mathrm{ELA}_{1}+\mathrm{ELA}_{2} \quad\left(\times 10^{-4}\right)$ | Relative <br> error (\%) |
| :--- | :--- | :--- |
| A@B1 | $3.09 \Leftrightarrow 0.67+3.12=3.79$ | 22.7 |
| A@E | $22.5 \Leftrightarrow 0.67+21.3=21.97$ | -2.4 |
| A@F1 | $53.7 \Leftrightarrow 0.67+51.2=51.87$ | -3.4 |
| B1@B2 | $6.13 \Leftrightarrow 3.12+2.66=5.78$ | -5.7 |
| B1@E | $22.4 \Leftrightarrow 3.12+21.3=24.42$ | 9.0 |
| B1@F1 | $57.8 \Leftrightarrow 3.12+51.2=54.32$ | -6.0 |
| F1@F2 | $109 \Leftrightarrow 51.0+50.5=101.5$ | -6.9 |
| Absolute Average |  | 8.0 |

Table 5.3.6 Using $\mathrm{R}=1 / \mathrm{C}$ as definition of crack resistance to check the series theory based on power model coefficient C. (C units: $\left.\mathrm{m}^{3} / \mathrm{h}(\mathrm{Pa})^{\mathrm{n}}\right)$

| Cracks connected <br> in series | $\frac{1}{C_{\text {Total }}}-\Sigma\left(\frac{1}{C_{1}}\right)\left(\times 10^{4}\right)$ | Re. <br> eror <br> $(\%)$ |
| :--- | :---: | :--- |
| A~B1 | $\frac{1}{0.312}-3.21-\left(\frac{1}{0.33}+\frac{1}{3.66}\right)-3.3$ | 2.8 |
| B1~A | $\frac{1}{0.358}-2.79-\left(\frac{1}{3.18}+\frac{1}{0.57}\right)-2.07$ | -25.8 |
| B1~E | $\frac{1}{3.40}-0.29-\left(\frac{1}{3.18}+\frac{1}{26.4}\right)-0.352$ | 21.4 |
| E~B1 | $\frac{1}{3.08}-0.32-\left(\frac{1}{24.4}+\frac{1}{3.66}\right)-0.314$ | -1.9 |
| E~F1 | $\frac{1}{21.5}-0.0465-\left(\frac{1}{24.4}+\frac{1}{64.1}\right)-0.0566$ | 21.7 |
| F1~B1 | $\frac{1}{3.17}-0.315-\left(\frac{1}{67.1}+\frac{1}{3.66}\right)-0.288$ | -8.5 |
| Absolute Average | $\frac{1}{21.8}-0.0459-\left(\frac{1}{67.1}+\frac{1}{26.4}\right)-0.0528$ | 15.0 |

Table 5.3.7 Using $R=1 / E L A$ as definition of crack resistance to check the series theory based on ELA from power model at 4 Pa . (ELA units: $\mathrm{m}^{2}$ )

| Cracks connected in series | $\frac{1}{E L A_{\text {Total }}}-\Sigma\left(\frac{1}{\text { ELA }_{1}}\right) \quad\left(\times 10^{4}\right)$ | Re. error (\%) |
| :---: | :---: | :---: |
| A ~ B1 | $\frac{1}{0.441}-2.27-\left(\frac{1}{0.474}+\frac{1}{3.62}\right)-2.39$ | 5.3 |
| B1 ~ A | $\frac{1}{0.519}-1.93-\left(\frac{1}{3.19}+\frac{1}{0.717}\right)-1.71$ | -11.4 |
| B1~E | $\frac{1}{3.3}-0.303-\left(\frac{1}{3.19}+\frac{1}{21.4}\right)-0.36$ | 18.8 |
| E~B1 | $\frac{1}{3.11}-0.322-\left(\frac{1}{20.1}+\frac{1}{3.62}\right)-0.326$ | 1.2 |
| E~F1 | $\frac{1}{17.9}-0.0559-\left(\frac{1}{20.1}+\frac{1}{51.2}\right)-0.0693$ | 24.0 |
| F1~B1 | $\frac{1}{3.22}-0.311-\left(\frac{1}{51.2}+\frac{1}{3.62}\right)-0.296$ | -4.8 |
| F1~E | $\frac{1}{18.1}-0.0552-\left(\frac{1}{51.2}+\frac{1}{21.4}\right)-0.0663$ | 20.1 |
| Absolute Average |  | 12.23 |

Table 5.3.8 Using $R=1 / E L A$ as definition of crack resistance to check the series theory based on ELA from KY model at 4 Pa .
(ELA units: $\mathrm{m}^{2}$ )

| Cracks connected in series | $\frac{1}{E L A_{\text {Total }}}-\Sigma\left(\frac{1}{E L A}\right) \quad\left(\times 10^{4}\right)$ | Re. error (\%) |
| :---: | :---: | :---: |
| A ~ B1 | $\frac{1}{0.472}-2.12-\left(\frac{1}{0.522}+\frac{1}{3.12}\right)-2.24$ | 5.7 |
| B1 ~ A | $\frac{1}{0.582}-1.72-\left(\frac{1}{2.84}+\frac{1}{0.667}\right)-1.85$ | 7.6 |
| B1 ~ E | $\frac{1}{2.91}-0.34-\left(\frac{1}{2.84}+\frac{1}{21.3}\right)-0.40$ | 17.6 |
| E~B1 | $\frac{1}{2.68}-0.37-\left(\frac{1}{20.1}+\frac{1}{3.12}\right)-0.37$ | 0 |
| E~F1 | $\frac{1}{17.9}-0.056-\left(\frac{1}{20.1}+\frac{1}{51.2}\right)-0.069$ | 23.2 |
| F1~B1 | $\frac{1}{2.82}-0.35-\left(\frac{1}{53}+\frac{1}{3.12}\right)-0.34$ | -2.9 |
| F1~E | $\frac{1}{18.0}-0.056-\left(\frac{1}{53}+\frac{1}{21.3}\right)-0.066$ | 17.9 |
| Absolute Average |  | 10.7 |

### 5.3.4 Building component analysis

Various openings of building components may work as cracks which are connected in parallel and/or series. The $Q-\Delta P$ relationships and ELAs have been measured and calculated for 18 building components. Among the 18 components there are two useful groups which have identifiable series/parallel flow paths. These are the test wall sections which contain electrical switches, duplex outlets and/or wire holes through the top plate with or without sealing of the holes and with or without switch/outlet gaskets. These examples will be used to analyze the flow path details, where the ELA values are produced from the new model.

The ELA values determined for combinations of these test cases are presented in Table 5.3.9. Note that the ELA values are for the total of the six boxes (switches or outlets) in a single test wall.

The following symbols are used to represent the combinations:
S --- Electrical switch, coverplate and box in the wall, no wiring hole penetrations through the studs or the top plate
O --- Electrical duplex outlet, cover plate and box in the wall, no wiring hole penetrations through the studs or the top plate
G --- Gasket (foam) on the switch or outlet
T/O -- Wire hole opening (1") in the top plate of the wall section with $12 / 3$ romex wire run through each hole
Seal - Top plate wire hole sealed with caulking

Table 5.3.9 Crack leakage resistance of the switch group

| Component | Connection | ELA (cm 2$)$ | $R(=1 /$ ELA $)$ |
| :--- | :--- | :--- | :--- |
| CO-10 | S + T/O | 2.08 | 0.48 |
| CO-11 | S + T/O + Seal | 1.37 | 0.73 |
| CO-12 | S + T/O + G + Seal | 0.762 | 1.31 |
| CO-13 | S+T/O + G | 0.939 | 1.06 |

If it is considered that applying a gasket or sealing the wire hole works as connecting a resistance in series (Figure 5-3-12), a value of $R=0.58\left(\mathrm{~cm}^{-2}\right)$ is obtained for applying the gaskets between CO-10 and CO-13 or between CO11 and $C O-12$. A value of $R=0.25\left(\mathrm{~cm}^{-2}\right)$ is found for sealing the top hole between $\mathrm{CO}-10$ and $\mathrm{CO}-11$ or $\mathrm{CO}-12$ and $\mathrm{CO}-13$. These values demonstrate the application of the series resistance theory.


Figure 5-3-12. Switch group resistance analysis
Table 5.3.10 contains the results of applying gaskets or sealing the top holes for the walls containing outlets. If we assume these work as connecting a resistance in series, we obtain $R=0.04\left(\mathrm{~cm}^{-2}\right), 0.15\left(\mathrm{~cm}^{-2}\right)$ and $0.10\left(\mathrm{~cm}^{-2}\right)$ between $\mathrm{CO}-3$ and CO-4, between $\mathrm{CO}-5$ and $\mathrm{CO}-7$ and between $\mathrm{CO}-6$ and CO 8 respectively. The average total resistance value for the six gaskets is 0.10 $\left(\mathrm{cm}^{-2}\right)$. (Note that there is not a dramatic difference in the order in which it is applied. This implies that there are significant other leaks.) The resistance values for the seal equal to $0.19\left(\mathrm{~cm}^{-2}\right)$ and $0.14\left(\mathrm{~cm}^{-2}\right)$ between CO-5 and CO6 and CO-7 and CO-8 respectively. Therefore the average total resistance value for sealing the six top holes is $0.17\left(\mathrm{~cm}^{-2}\right)$.

Table 5.3.10 Crack leakage resistance of the outlet group

| Component | Connection | ELA $\left(\mathrm{cm}^{\wedge} 2\right)$ | $R(=1 /$ ELA $)$ |
| :--- | :--- | :--- | :--- |
| CO-3 | 0 | 2.64 | 0.38 |
| CO-4 | O+G | 2.39 | 0.42 |
| CO-5 | O+T/O | 2.80 | 0.36 |
| CO-6 | O+T/O + Seal | 1.83 | 0.55 |
| CO-7 | O+T/O + G | 1.98 | 0.51 |
| CO-8 | O+T/O + G + Seal | 1.53 | 0.65 |

There is a problem in attempting to determine the resistance of the T/O (Top plate wire hole) by subtracting the resistance of $\mathrm{CO}-3$ from the resistance of CO-5. If we assume that O (outlet on the wall) and T/O (top hole) are
connected in series, the resistance of CO-5 should be higher than the resistance of CO-3, while the result shows the resistance of CO-5 is lower than that of $\mathrm{CO}-3$. This indicates that the function of having a wire hole in the top plate with an outlet may work in neither simple parallel nor simple series, it work as a combination of parallel and series.

Overall the definition presented of crack resistance and parallel/series flow theory describes the component leakage well.


Figure 5-3-13. Outlet group resistance analysis

### 5.4 ELA and $C_{d}$ Curves Analysis

In previous air leakage studies, researchers set ELA and $C_{d}$ as constants evaluated at $4,10,50$ or 75 Pa . The results from the FPD tests use these standard values for simplicity since the flow is not simple orifice flow. Actually they are functions of the pressure difference, $\Delta P$. In this analysis, the ELA and $\mathrm{C}_{\mathrm{d}}$ are calculated based on their original definitions. The ELA curves in Figures $5-4-1$ and 5-4-2 are based on the derived equation 4-10 for the 18 building components. They illustrate what has been previously known. That is that ELA has a significant variation as $\Delta \mathrm{P}$ changes, which means that the choice of reference pressure drops ( 4 Pa or 10 Pa ) affects the ELA results. The $\mathrm{C}_{\mathrm{d}}$ curves in Figure 5-4-3 are based on equation 4-11 with five different $C_{2}$ values, which illustrate the significant errors by setting $C_{d}=1.0$ (ASTM 1984, 1987) or $\mathrm{C}_{\mathrm{d}}=0.611$ (CGSB 1986) for the calculation of any crack leakage. Figure 5-4-4 presents the $\mathrm{C}_{\mathrm{d}}$ charts for the KY model at 4 Pa and 10 Pa . These three dimensional contours illustrate how $\mathrm{C}_{\mathrm{d}}$ values vary instead of previously assumed constant settings of $C_{d}=0.611$ or $C_{d}=1.0$. This chart is created for calculation purposes especially for the situation of computing the EQLA values. In this chart of typical values (input $K=1.5$ as a known value), we find that the $C_{d}$ values change dramatically for low pressure difference rather than being a constant. Hence the $\mathrm{C}_{\mathrm{d}}$ value at low pressures is more sensitive.

## ELA Curves for Components




Figure 5-4-1. ELA curves for building components 1 to 12
difference between the minimum and maximum values recorded for several components was much larger. This it to be expected since there were more sources of data with a greater variability between the different sources. It was often found that the variability between the "same" component between two sources using the same reporting format was greater than the variation between component types from the same source.

It should be noted that although the number of different building components listed in the table is greater than previously identified, there are still gaps in the data for some groupings of components which need estimates (or better estimates) of leakage values. There are currently several significant projects underway to obtain the leakage of ductwork. Since these data were not available, they were not included in this report. Other components which need data (or better additional data) include: building construction joints (joints of dissimilar materials like masonry and wood or insulating board and wood; sole plate/baseboard; band joists; building corner joints; butt joints of sheathing, etc), window and door framing in masonry and wood wall construction, and the combined effects of air infiltration barriers and vapor retarders.

## Evaluate and Give Alternatives to the ELA Concept:

A comparison of the predicted effective leakage areas using the commonly used reference pressures and discharge coefficients was done for the building components tested in the laboratory. As is commonly known, the ELA varies with the reference pressures selected. These differences were significant (30\%) in the ELA predicted using 4 Pa and 10 Pa as the reference pressures for the components tested. Curves were also produced to illustrate the significant variations introduced by selecting different discharge coefficients. In addition to these variations, the known correlation which exists between n and C propagate variations which are not readily apparent. These type of variations cause confusion when comparisons are attempted between reported values when different authors use different reference values and discharge coefficients, especially when these values and the C and n are not stated.

A theoretically-based air leakage model to define the flow rate versus the pressure differential across building components was derived from the dimensionless crack equation. It was validated and compared to commonly accepted ELA calculation techniques using a number of well-defined cracks which were experimentally tested. It was found that the three-parameter model developed describes the flow versus pressure relationship accurately however it is not as easy to use. It requires a nonlinear regression solution technique and has a numerical restriction from the power model. A benefit of the model developed is that the coefficients obtained represent physical parameters describing the characteristics of the opening tested.

Experiments were also run to test the model developed for situations where the air may be flowing in a series path through several openings or in parallel across different openings. The new model was able to fit the flow versus pressure data as well as the commonly used techniques. It was also demonstrated that the physical parameters obtained independently with the new model on each opening could be combined to predict the total resistance when the openings were combined in series.

Evaluate Different Methods and Recommend a Method of Reporting Leakage:
There were no direct comparisons found in the literature between the various DC pressurization standards so there are no recommendations as to the validity of one over the other or standard calibrations for converting the data between them.

Several different methods of reporting leakage were observed. It was observed that the leakage data obtained in the available literature were given in three main ways: constants for the fit of the data to the power equation, the flow at a given differential pressure across the component, or the equivalent/effective leakage area. These methods were evaluated in order to make the transformations to compare components tested by different sources. It was not possible to get ELA values from some of the techniques such as those which gave leakage as a percentage of the total leakage of the structure. The most common technique of giving a $C$ and $n$ value for the component was adequate provided the value used for the discharge coefficient and the unit of area or length was known. There was some confusion in the literature between effective and equivalent leakage areas. It is critical that leakage reports give sufficient information to avoid this confusion.

It was recognized that there are advantages to using the power and/or orifice equations such as ease of use and generally having a good fit. Disadvantages to these techniques were also identified (e.g. dimensionally nonhomogeneous, constants have no physical basis, not theoretically based and incompatibility for use in series/parallel flow analysis).

It was found that there were several key parameters to successfully reporting sufficient information about leakage in order to make comparisons. In the ideal case the following need to be included: the actual flow/pressure difference values for the test points; the area over which the pressure is being maintained and flow is being measured (or the length of the crack which is exposed to the pressure difference); and other conditions which influence the flow such as type of weatherstripping, etc. If the actual data points cannot be reported, the coefficients ( $C_{1}, C_{2}, C_{3}$ or $C \& n$ ) and the range of pressures used for testing should be reported in addition to the others indicated.

## REFERENCES

ASHRAE. 1989. ASHRAE Handbook - Fundamentals. American Society of Heating, Refrigerating, and Air Conditioning Engineers. Atlanta, GA.

ASTM. 1987. ASTM Standard E779-87, Standard test method for determining air leakage rate by fan pressurization. American Society for Testing Materials. Phila., PA.

ASTM. 1984. ASTM Standard E783-84, Standard method for field measurement of air leakage through installed exterior windows and doors. American Society for Testing Materials. Phila., PA

Bassett, M.R. 1986. Building site measurements for predicting air infiltration rates. In Measured Air Leakage of Buildings. ASTM STP 904. American Society for Testing and Materials. pp365-383. Phila., PA.

CGSB. 1986. Standard CAN/CGSB-149.10-M86, Determination of the airtightness of building envelopes by the fan depressurization method. Canadian General Standards Board. Ottawa, Canada.

Charlesworth, P.S. 1 Y४8. Aır exchange rate and arrtightness measurement techniques - Ản appiications guide. Air Infiltration and Ventilation Centre, Coventry, Great Britain.

Chastain, J.P., D.G. Colliver and P.W. Winner. 1987. Computation of discharge coefficeints for laminar flow in rectangular and circular openings. ASHRAE Transactions 93(2):22259-2281.

Cole, J.T. and T.S. Zawacki. 1980. Application of a generalized model of air infiltration to existing homes. ASHRAE Transactions 86(1):765-777.

Draper, N.W. and H. Smith. 1966. Applied regression analysis. John Wiley \& Sons, Inc. New York
Etheridge, D.W. 1977. Crack flow equations as scale effect. Building and Environment 12:282-289.
Gabrielsson, J. and P. Porra. 1968. Calculation of infiltration and transmission heat loss in residential buildings by digital computers. J. Inst. Heat Vent. Eng. 35:357-368.

Gadsby, K.J., and D.T. Harrje. 1985. Fan pressurization of buildings: Standards, calibration and field experience. $\Lambda$ SHRAE Transactions 91 (2B):95-101.

Hopkins, L.P. and B. Hansford. 1974. Air flow through cracks. Building Services Engineer. Vol 42:123132.

Hunt, C.M. 1980. Air infiltration: A review of some existing measurement techniques and data. Building Air Change Rate and Infiltration Measurements. ASTM STP-719. American Society for Testing Materials. pp 3-23.

Limb, M.J. 1989. AIRGUIDE - A Guide to the AIVC's AIRBASE. Air Infiltration and Ventilation Centre. Coventry, Great Britian

Modera, M.P. and M.H. Sherman. 1985. AC pressurization - A technique for measuring leakage area in residential buildings. ASHRAE Transactions 91 (2B):120-132.

Murphy, W.E., D.G. Colliver and L.R. Piercy. 1991. Repeatability and reproducibility of fan pressurization devices in measuring building air leakage. ASHRAE Transactions 97(2):885-895.

Phaff, H. 1987. Flowrate measurements with a pressure compensating device. 8th AIVC Conference. Ventilation Technology Research and Application, Supplement to Porceedings, pp 167-170.

Raymond, H.M. 1990. Classical and modern regression with applications. 2nd. Ed. PWS-Kent Publishing Company. Boston, MA.

SAS. 1985. SAS User's Guide - Statistics. 5th Ed. SAS Institute Inc., Cary, NC.
Shaw, C.Y., D.M. Sander and G.T. Tamura. 1974. A Fortran V program to simulate stair shaft pressurization systems in multi-story buildings. DBR Computer Program No. 38. National Research Council of Canada.

Sherman, M.H. 1980. Air infiltration in buildings. Ph.D. Thesis. University of California, Berkeley, CA.
Sherman, M.H. and D.T. Grimsrud. 1980. Infiltration-pressurization correlation: simplified physical modeling. ASHRAE Transactions 86(1):778-807.

Sun, W. 1992. Modeling air leakage characteristics of residential building components. Unpublished M.S. Thesis, University of Kentucky. Lexington, KY.

Weidt, J.L. 1979. Field air leakage of newly installed residential windows. Proceedings of the ASHRAE/DOE-ORNL Conference. Thermal Performance of the Exterior Envelopes of Buildings. Orlando, FL.

## BIBLIOGRAPHY Part 1 - BY AIRBASE NUMBER

\#NO40 Tamura, G. 1975. Measurement of air leakage characteristics of house enclosures. ASHRAE Transactions 81(1):202-208, 1 fig, 5 tabs. \#AIC 1048.
\#NO41 Hunt C.M., Burch D. 1975. Air infiltration measurements in a four-bedroom townhouse using sulphur hexafluoride as a tracer gas. ASHRAE Transactions 81(1):186-201, 5 figs, 4 tabs, 18 refs. \#AIC 229.
\#NO42 Stricker S. 1975. Measurement of air tightness of houses. ASHRAE Transactions 81(1):148-167, 9 figs. 1 tab. 3 refs. \#AIC 1093.
\#NO44 Shaw C.Y., Sander D.M., Tamura G.T. 1973. Air leakage measurements of the exterior walls of tall buildings. ASHRAE Transactions 79(2):40-48, 10 figs. 6 refs. D.B.R.research paper no. 601. \#AIC 34.
\#NO70 Grimsrud, D.T., Sherman M.H., Diamond R.C., and Sonderegger R.C. 1979. Air leakage, surface pressures and infiltration rates in houses. 2nd International C.I.B. Symposium on Energy Conservation in the Built Environment, Copenhagen, May 28 - June 1st 1979. Preprints - session 2, 111-120, 5 figs, 2 tabs, 4 refs. \#AIC 23.
\#NO86 Funkhouser, P.E. 1979. Air infiltration effects on the thermal transmittance of concrete building systems. ASHRAE Transactions 85(1):918-925, 4 figs. \#AIC 16.
\#NO89 Blomsterberg A.K., Harrje D.T. 1979. Approaches to evaluation of air infiltration energy losses in buildings. ASHRAE Transactions 85(1):797-815, 10 figs, 2 tabs, 20 refs. \#AIC 13, also "Evaluating air infiltration energy losses" ASHRAE Journal 21 (5):25-32.
\#NO90 Goldschmidt, V.W., Wilhelm, D.R. 1979. Summer infiltration rates in mobile homes. ASHRAE Transactions 85(1):840-850, 15 figs, 1 tab, 11 refs. \#AIC 15.
\#NO91 Tamura, G.T. 1979. The calculation of house infiltration rates. ASHRAE Transactions 85(1):58-71, 7 figs, 5 tabs, 9 refs. \#AIC 3.
\#NO92 Caffey, G.E. 1979. Residential air infiltration. ASHRAE Transactions 85(1):41-57, 12 figs, 5 tabs, 1 ref. \#AIC 2.
\#NO101 Bursey T., Green G.H. 1970. Combined thermal and air leakage performance of double windows. ASHRAE Transactions 76(2):215-226, 13 figs, 6 refs. \#AIC 1222.
\#NO113 Grubbs W.J. 1967. Leaky prime windows. ASHRAE Journal 9(1):109-112, 7 figs, 6 tabs, 2 refs. \#AIC 1225.
\#NO119 Lowinski, J.F. 1979. Thermal performance of wood windows and doors. ASHRAE Transactions 85(1):548-566, 10 tabs, \#AIC 1289.
\#NO142 Thorogood R.P. 1979. Resistance to air flow through external walls. Building Research Establishment Information Paper. 14/79 \#AIC 76.
\#NO159 Tamura, G.T. 1974. Predicting air leakage for building design. 6th C.I.B. Congress on the Impact of Research on the Built Environment Budapest 3-10 October 1974 preprints vol 1/1 p368-374, D.B.R. technical paper no. 437 \#AIC 41.
\#NO173 Schutrum L.F., Ozisik N., Baker J.T., and Humphreys C.M. 1961. Air infiltration through revolving doors. ASHRAE Journal 3(11):43-50, \#AIC 1445.
\#NO176 Tamura G.T. and Shaw C.Y. 1976. Studies on exterior wall air tightness and air infiltration of tall buildings. ASHRAE Transactions $82(1): 122-134$, N.R.C.C. Building Research paper no.706, \#AIC 33.
\#NO177 Sasaki J.R. 1973. Air leakage testing. Spec. Ass. 15(5):15-18, N.R.R.C. Division of Building Research technical paper no. 407, \#AIC 36.
\#NO208 Keast D.N., Pei, H-S. 1979. The use of sound to locate infiltration openings in buildings. Proceedings ASHRAE/DOE Conference "Thermal performance of the exterior envelopes of buildings" Florida, December 3-5, 1979, pp85-93, \#AIC 71.
\#NO210 Sasaki J.R., Wilson A.G. 1965. Air leakage values for residential windows. ASHRAE Transactions $71(2): 81-88,7$ figs, 17 refs, National Research Council of Canada, Division of Building Research paper no. 329, \#AIC 37.
\#NO221 Collins J.O. 1979. Air infiltration measurement and reduction techniques on electrically heated homes. Proceedings ASHRAE/DOE Conference on "Thermal Performance of the Exterior Envelopes of Buildings "Florida, 3-5 December 1979, 4 tabs 5 refs, \#AIC 74.
\#NO251 Potter I.N. 1979. Effect of fluctuating wind pressures on natural ventilation. ASHRAE Transactions 85(2):445-457, 8 figs, 3 refs, \#AIC 9.
\#NO260 Grimsrud D.T., Sherman M.H., and Diamond R.C. 1979. Infiltration - pressurization correlations: detailed measurements on a California house. ASHRAE Transactions 85(1):851-865, 7 figs, 1 tab, 5 refs, \#AIC 10.
\#NO286 Houghten F.C. and Schrader C.C. 1924. Air leakage through the openings in buildings. ASHVE Transactions 30:105-120, 11 figs, 2 tabs, \#AIC 1240.
\#NO287 Schrader C.C. 1924. Air leakage around window openings. ASHVE Transacitons 30:313-322, \#AIC 1481.
\#NO288 Stewart M.B. Jacob T.R. and Winston J.G. 1979. Analysis of infiltration by tracer gas technique, pressurization tests and infrared scans. Proceedings ASHRAE/DOE Conference "Thermal performance of the exterior envelopes of buildings" Florida December 3-5th 1979, 10 figs, 3 tabs, 3 refs. \#AIC 72.
\#NO299 Tamura G.T. and Shaw C.Y. 1976. Air leakage data for the design of elevator and stair shaft pressurization system. ASHRAE Transactions 82(2):179-190, 8 figs, 3 tabs, 8 refs. \#AIC 35.
\#NO311 Shaw C.Y. 1980. Methods for conducting small-scale pressurization tests, and air leakage data of multi-storey apartment buildings. ASHRAE Transactions 86(1):241-250, 11 figs, 1 tab; 4 refs. \#AIC 103.
\#NO314 Shah, M.M. 1980. Estimated rate of pressurization and depressurization of buildings. ASHRAE Transactions 86(1):251-257, 2 refs. \#AIC 102.
\#NO320 Harrje, D.T., Blomsterberg, A., and Persily, A. 1979. Reduction of air infiltration due to window and door retrofits in an older home. Princeton University, Center for Energy and Environmental Studies, report PU/CEES 85,25 pg 10 figs 2 tabs. 10 refs, \#AIC 64.
\#NO328 Kronvall, J.Air. 1978. Leakage of buildings-a literature list. Lund Institute of Technology, Divisision of Building Technology, report 77, \#AIC 54.
\#NO339 Tietsma, G.J., Peavy, B.A. 1978. The thermal performance of a two-bedroom mobile home. National Bureau of Standards Building Science Series, 102:55 pg 156 figs 2 refs. \#AIC
1287.
\#NO344 Sasaki, J.R. and Wilson, A.G. 1962. Window air leakage. National Research Council Canada, Division of Building Research, Building Digest no 25, \#AIC 1671.
\#NO398 Grot, R.A. and Clark, R.E. 1979. Air leakage characteristics of low-income housing and the effectiveness of weatherization techniques for reducing air infiltration. Proceedings ASHRAE/DOE Conference "Thermal performance of the exterior envelopes of buildings" Kissimmee, Florida 3-5 December 1979, 8 tabs 10 figs 5 refs, \#AIC 157.
\#NO457 Blomsterberg, A., Sherman, M.H. and Grimsrud, D.T. 1979. A model correlating air tightness and air infiltration in houses. Proceedings ASHRAE/DOE Conference "Thermal performance of the exterior envelopes of buildings" DEC. 3-5 1979 Florida, \#AIC 342.
\#NO458 Weidt, J.L., Weidt, J., and Selkowitz, S. 1979. Field air infiltration performance of new residential windows. Proceedings. ASHRAE/DOE Conference "Thermal performance of the exterior envelopes of buildings" 3-5. Dec. 1979 Florida, Lawrence Berkeley Laboratory report LBL 9937, \#AIC 387.
\#NO459 Sherman, M.H., Grimsrud, D.T., and Sonderegger, R.C. 1979. Low pressure leakage function of a building. Proceedings ASHRAE/DOE Conference "Thermal performance of the exterior envelopes of buildings" Dec. 3-5 1979 Florida, 6 figs, 5 refs, \#AIC 20.
\#NO461 Tsongas, G.A., Odell, F.G., and Thompson, J.C. 1979. A field study of moisture damage in walls insulated without a vapour barrier. Proceedings ASHRAE/DOE Conference "Thermal performance of the exterior envelopes of buildings" 3-5 December 1979 Florida, \#AIC 1372.
\#NO466 Burch, D.M., Luna, D.E. 1980. A mathematical model for predicting attic ventilation rates required for preventing condensation on roof sheathing. ASHRAE Transactions 86(1):201220, 10 figs, 3 tabs, 14 refs, \#AIC 175.
\#NO478 Sherman, M.H., Grimsrud, D.T. 1980. Infiltration-pressurization correlation: simplified physical modeling. ASHRAE Transactions 82(2):778-807, LBL 10163, 4 figs, 16 refs, \#AIC 192.
\#NO509 Hunt, C.M., Porterfield, J.M., and Ondris, P. 1978. Air leakage measurements in three apartment houses in the Chicago area. National Bureau of Standards Interagency Report NBSIR 78-1475 24p, 12 figs, 9 refs, \#AIC 205.
\#NO526 Hunt, C.M., Treado, S.J., and Peavy, B.A. 1976. Air leakage measurements in a mobile home. National Bureau of Standards Interagency Report, NBSIR 76-1063, 23 pg, 9 figs, 5 tabs, 4 refs, \#AIC 210.
\#NO569 Stricker, S. 1974. Measurement of air leakage of houses. Ontario Hydro Research quarterly, 26(4):11-18, 7 figs, 2 refs, \#AIC 220.
\#NO597 Sasaki, J.R. 1968. Air leakage characteristics of some brick and concrete block walls. National Research Council of Canada, Division of Building Research, Technical note no. 525, 5p, 2 figs, 1 tab, \#AIC 273.
\#NO618 Beach, R.K. 1979. Relative tightness of new housing in the Ottawa area. Division of Building Research, National Research Council of Canada. Building research note no. 149 7p, 6 figs, 3 tabs, \#AIC 272.
\#NO646 Treado, S.J., Burch, D.M., Hunt, C.M. 1979. An investigation of air infiltration characteristics and mechanisms for a townhouse. National Bureau of Standards Technical Note, 992, 31 pg, 7 figs, 8 refs, \#AIC 311.
\#NO706 Shaw, C.Y. and Tamura, G.T. 1980. Mark XI energy research project, Air tightness and air infiltration measurements. Division of Building Research, National Research Council of Canada. Building Research Note no. 162, Ottawa 7pg, 14 figs, 1 tab, 12 refs, \#AIC 609.
\#NO708 Scanada Consultants Ltd. 1979. Effect of high levels of insulation on the heating fuel consumption of Canadian houses. Report for the HUDAC Technical Research Committee, Canada, Project T80-78-30, 17p, 8 tabs, 4 figs, \#AIC 348.
\#NO712 Hunt, C.M. 1980. Air infiltration: A review of some existing measurement techniques and data. In "Building Air Change Rate and Infiltration Measurements" Proceedings, ASTM Conference Gaithersburg 13 March 1978 C.M. Hunt, J.C. King, H.R. Trechsel eds. pp 3-23, 5 figs, 51 refs, AIC.
\#NO788 Grimsrud, D.T., Sonderegger, R.C., and Sherman M.H. Infiltration measurements in audit and retrofit programs. Lawrence Berkeley Laboratory report. \#AIC 388.
\#NO891 Dumont R.S., Orr H.W., and Figley D.A. 1981. Air tightness measurements of detached houses in the Saskatoon area. Building Research Note no. 178 Division of Building Research NRCC 18p, 2 figs, 7 tabs, 6 refs.
\#NO993 Retrospectors, Inc. 1981. Air tightness testing and sealing of homes in Ottawa, Ontario. Ontario Ministry of Municipal Affairs and Housing, Report no. 11501 13pp, 2 figs, 1 tab.
\#NO1004 Scheunemann, E.C. 1982. Mark XI energy research project. Summary of results 1978-1981. Building Research Practice Note no. 27 National Research Council of Canada 18pp, 3 figs, 10 tabs, 13 refs.
\#NO1008 Shaw, C.Y., and Brown, W.C. 1982. Effect of a gas furnace chimney on the air leakage characteristic of a two-storey detached house. Preprint 3rd AIC Conference "Energy efficient domestic ventilation systems" London 20-23 September, 1982 7pp, 10 figs.
\#NO1056 Shaw, C.Y. 1981. A correlation between air infiltration and airtightness for houses in a developed residential area. ASHRAE Transactions 87(2):333-341, 9 figs, 6 refs.
\#NO1057 Hollowell, C.D., Young, R.A., Berk, J.V., and Brown, S.R. 1982. Energy conserving retrofits and indoor air quality in residential housing. ASHRAE Transactions 88(1):875-893, 3 figs, 7 tabs, 9 refs.
\#NO1065 Krinkel, D.L., Dickeroff, D.J., Casey, J. and Grimsrud, D.T. 1980. Pressurization test results: Bonneville Power Administration Energy Conservation Study. LBL Report no. 10996, 13pp, 4 figs, 1 tab, 2 refs.
\#NO1070 Offermann, F.J., Gurman, J.R., and Hollowell, C.P. 1981. Midway house-tightening project: a study of indoor air quality. LBL Report 12777 27pp, 4 tabs, 2 figs, 15 refs.
\#NO1076 Blomsterberg, A.K., Modera, M.P., Grimsrud, D.T. 1981. The Mobile Infiltration Test Unit - Its design and capabilities:Preliminary experimental results. LBL Report no. 12259 20pp, 5 figs, 2 tabs, 5 refs.
\#NO1155 Dickinson, J.B., D.T. Grimsrud, D.L. Krinkel, R.D. Lipschutz. 1982. Results of the Bonneville Power Adminstration Weatherization and Tightening Projects at the Midway Substation residential community. February. LBL-12742.
\#NO1 156 Lipschutz, R.D., Dickinson, J.B., and Diamond, R.C. 1982. Infiltration and leakage measurements in new houses incorporating energy efficient features. 1982 Summer Study in Energy Efficient Buildings Santa Cruz CA 22-28 August 1982, LBL Report no. 14733.
\#NO1157 Dickerhoff, D.J., Grimsrud, D.T., and Lipschutz, R.D. 1982. Component leakage testing in residential buildings. 1982 Summer Study in Energy Efficient Buildings, Santa Cruz CA August 22-28 1982, LBL Report no. 14735.
\#NO1158 Dickinson, J.B., Lipschutz, R.D., O’Regan, B., and Wagner, B.S. 1982. Results of recent weatherization retrofit projects. 1982 Summer Study in Energy Efficient Buildings, Santa Cruz CA August 22-28 1982, LBL Report no. 14734, 19pp, 5 tabs, 19 refs.
\#NO1170 Grimsrud, D.T., Sonderegger, R.C., and Sherman, M.H. 1981. Infiltration measurements in audit and retrofit programs. Energy Audit Workshop 13-15 April 1981 SCBR Document D21, pp 115-139, 5 figs, 4 tabs, 14 refs.
\#NO1175 Giesbrecht, P. 1982. An abstract on airtightness in houses. Ener-Corp Management Report. 25pp, 22 figs, 11 refs, \#AIC 731,
\#NO1184 Persily, A.K., and Linteris, G.T. 1983. A comparison of measured and predicted infiltration rates. ASHRAE Transactions $89(2 \mathrm{~B}): 183-200,17$ refs, 6 figs, 6 tabs.
\#NO1207 Persily, A.K. and Grot, R.A. 1983. Air infiltration and building tightness measurements in passive solar residences. Reprint Solar Engineering ( ):116-121, 2 tabs, 5 figs, 15 refs.
\#NO1227 O'Riordan, M.C., James, A.C., Rae, S. and Wrixon, A.D. 1983. Human exposure to radon decay products inside dwellings in the United Kingdom. National Radiological Protection Board R152 41pp, 8 tabs, 110 refs.
\#NO1234 Crall, C.P. Development of the air infiltration model for the energy performance design system. ASHRAE Transactions 89(2B):201-210, 2 figs, 10 refs.
\#NO1259 Bassett, M. 1983. Air infiltration in New Zealand houses. 4th AIC Conference "Air infiltration reduction in existing buildings" Switzerland, 26-28 September 1983 p.14.1-14.18, 6 figs, 1 tab, 17 refs.
\#NO1261 Reinhold, C., and Sonderegger, R. 1983. Component leakage areas in residential buildings. 4th AIVC Conference "Air infiltration reduction in existing buildings" Switzerland, 26-28 September 1983 p.16.1-16.3, 13 tabs, 5 figs, 37 refs.
\#NO1277 Klems, J.H. 1983. Methods of estimating air infiltration through windows. Energy and Buildings 5(4):243-252, 4 tabs, 21 refs.
\#NO1 284 Dumont, R.S., Orr, H.W., and Lux, M.E. 1982. Low energy prairie housing - a survey of some essential features. DBR Building Practice Note No. 38 pp 10, 4 tabs, 10 refs.
\#NO1336 Brandle, K. and Boehm, R.F. 1982. Airflow windows - performance and applications. Proc. ASHRAE/DOE Conference "Thermal performance of the exterior envelope of the building II" Dec 6-9 1982 USA p.361-379, 12 figs, 5 tabs.
\#NO1338 Persily, A. 1982. Repeatability and accuracy of pressurization testing. Proc. ASHRAE/DOE Conference "Thermal performance of the exterior envelope of the building II" Dec 6-9 1982 USA p.380-390, 7 figs, 1 tab, 19 refs.
\#NO1340 Grimsrud, D.T. Sherman, M.H., and Sonderegger, R.C. 1982. Calculating infiltration implications for a construction quality standard. Proc. ASHRAE/DOE Conference "Thermal performance of the exterior envelope of the building II" Dec 6-9 1982 USA p.422-450, 9 tabs, 5 figs, 30 refs.
\#NO1354 Nagda, N.L., Harrje, D.T., Koontz, M.D. and Purcell, G.G. 1984. A detailed investigation of the air infiltration characteristics of two houses. ASTM Symposium on measured air leakage performance of buildings Philadelphia USA April 2-3 1984. STP -904, 4 refs.
\#NO1356 Gammage, R.B., Hawthorne, A.R. and White, D.A. 1984. Parameters affecting air infiltration and air tightness in 31 east Tennessee homes. ASTM Symposium on measured air leakage performance of buildings, Philadelphia USA April 2-3 198413 pp., 2 tabs.
\#NO1357 Bassett, M.R. 1984. Building site measurements for predicting air infiltration rates. ASTM Symposium on measured air leakage performance of buildings Philadelphia USA April 2-3 1984 STP-904, 7 figs, 6 tabs, 12 refs.
\#NO1358 Kim, A., and Shaw, C.Y. 1984. Seasonal variation in airtightness of two detached houses. ASTM Symposium on measured air leakage performance of buildings, Philadelphia USA April 2-3 1984. STP 904, 9 figs, 3 tabs.
\#NO1382 Cunningham M.J. 1984. Further analytical studies of building cavity moisture concentrations. Building and Environment 19(1):21-28, 6 figs, 1 tab, 15 refs.
\#NO1431 Sulatisky, M. 1989. Air tightness tests on 200 new houses across Canada. Summary of results. Canada Buildings Energy Technology Transfer Program publication 84.01, 51pp, 6 figs, 34 tabs, 6 refs.
\#NO1484 USA Department of Energy. 1983. Energy efficient windows. A key to energy performance. USA Dept. of Energy, 8pp, 4 figs, 18 refs.
\#NO1514 Harrje, D.T., and Born, G.J. 1982. Cataloging air leakage components in houses. Center for Energy and Environmental Studies, Princeton University, 22pp, 21 figs.
\#NO1515 Jacobson, D., Dutt G S., and Socolow R H. 1984. Pressurization testing, infiltration reduction and energy savings. PU/CEES Report No. 173, Center for Energy and Environmental Studies, Princeton University, 37pp, 1 fig, 11 tabs, 14 refs. Presented at the ASTM Symposium on the Measured Air Leakage Performance of Buildings, Philadelphia, 1984.
\#NO1594 Carisson, A., Kronvall, J. 1984. Constancy of air tightness in buildings. 5th AIC
Conference 'The implementation and effectiveness of air infiltration standards in buildings' Reno, Nevada, 1-4 October 1984, pp 16.1-16.13, 6 figs, 1 tab, 2 refs.
\#NO1647 Quackenboss, J.J. et al. 1984. Residential indoor air quality, structural leakage and occupant activities for 50 Wisconsin homes.Indoor Air. Vol 5. Buildings, Ventilation and Thermal Climate. Edited by B. Berglund, T. Lindvall, J. Sundell. Swedish Council for Building Research, pp. 411-420, 2 tabs, 23 refs.
\#NO1649 Gammage, R B. et al. 1984. Parameters affecting air leakage in East Tennessee homes. Indoor Air. Vol 5. Buildings, Ventilation and Thermal Climate. Edited by B Berglund, T Lindvall, J Sundell. Swedish Council for Building Research, pp. 429-434, 2 tabs, 6 refs.
\#NO1670 Poreh, M, and Hassid, S. 1982. Simulation of buoyancy and wind induced ventilation. Wind Tunnel Modeling for Civil Engineering Applications. Proceedings of the International Workshop on Wind Tunnel Modeling Criteria and Techniques in Civil Engineering Applications, Gaithersburg, USA, April 14-16, 1982. Edited by T A Reinhold. Cambridge,UK:CUP. pp. 558-566. 3 figs, 6 refs.
\#NO1721 lobst, J. et al. 1984. Interior storm windows. Rodale Product Testing Report No 117-T. Emmaus, Pennsylvania, USA. Rodale Press. 41 p, 3 figs, 8 tabs, 5 refs.
\#NO1755 Klote, J.H. 1985. Smoke control in VA hospitals. ASHRAE Journal, 27(4):42-45, 1 fig,

2 tabs, 8 refs.
\#NO1853 Persily, A.K. and Grot, R.A. 1985. Accuracy in pressurization data analysis. ASHRAE Transactions $91(2 B): 105-119,5$ tabs, 18 refs.
\#NO 1856 Herrlin, M.K. 1985. MOVECOMP: a static-multicell-airflow-model. ASHRAE Transactions $91(2 \mathrm{~B}): 1989-1996,2$ figs, 7 refs.
\#NO1859 Silberstein, S. and Grot, R.A. 1985. Air exchange rate measurements of the National Archives Building. ASHRAE Transactions 91(2A):503-510, 3 figs, 1 tab, 6 refs.
\#NO1869 Yuill, G. 1985. Determination of the effective leakage areas of houses by multilinear regression analysis of the energy consumption data. ASHRAE Transactions 91 (2B):133-143, 2 figs, 4 tabs, 2 refs.
\#NO1870 Lipschutz, R.D., Girman, J.R., Dickinson, J.B., Allen, J.R., and Traynor, G.W. 1981. Infiltration and indoor air quality and indoor air quality in energy efficient houses in Eugene, Oregon. Berkeley, California, USA. Lawrence Berkeley Laboratory, LBL-12924 UC-95d. 49p., 5 figs, 9 tabs, 27 refs.
\#NO1872 Modera, M.P., and Sherman, M.H. 1985. AC pressurization: a technique for measuring leakage area in residential buildings. ASHRAE Transactions $91(2 \mathrm{~B}): 120-132,4$ figs, 2 tabs, 20 refs.
\#NO1873 Gadsby, K.J., and Harrje, D.T. 1985. Fan pressurization of buildings: standards, calibration, and field experience. ASHRAE Transactions 91 (2B):95-104, 6 figs, 1 tab, 24 refs.
\#NO1874 Trechsel, H.R., Achenbach, P.R., and Ebbets, J.R. 1985. Effect of an exterior air-infiltration barrier on moisture condensation and accumulation within insulated frame wall cavities. ASHRAE Transactions 91 (2A):545-559, 6 figs, 5 tabs, 7 refs.
\#NO1999 Weidt, J.L., Weidt, J., and Selkowitz, S. 1979. Field air leakage of newly installed residential windows. Presented at the ASHRAE/DOE Conference, Kissimmee, Florida, December $3-5,1979.17 \mathrm{p} .4$ figs, 7 tabs, 12 refs.
\#NO2062 Wilson, A.G. and Sasaki, J.R. 1972. Evaluation of Window Performance. Division of Building Research, NRCC, Ottawa, Canada. Nat. Bur. Stand. Special Publication 361, pp. 385-394 Ottawa, Canada.
\#NO2074 Kehrli, D.W. 1985. Window air leakage performance as a function of differential temperatures and accelerated environmental aging. Rochester, New York, USA:Schlegel Corporation. 28p. 13 figs, 1 tab, 10 refs.
\#NO2251 Lagus, P.L., and King, J.C. 1984. Air leakage and fan pressurization measurements in selected naval housing. In: Measured air leakage in buildings. A symposium on performance of building constructions, Philadelphia, 2-3 April 1984. ASTM Special Technical Publication 904. Edited by H R Trechsel and P L Lagus. ASTM. p5-16. 4 figs, 6 tabs, 9 refs.
\#NO2252 Persily, A.K. 1986. Measurements of air infiltration and airtightness in passive solar homes. Measured air leakage of buildings. A symposium on performance of building constructions, Philadelphia, 2-3 April 1984. ASTM Special Technical Publication 904. Edited by H R Trechsel and P L Lagus. ASTM. p46-60. 4 figs, 3 tabs, 20 refs.
\#NO2253 Goldschmidt, V.W. 1986. Average infiltration rates in residences: comparison of electric and combustion heating systems. In: Measured air leakage of buildings. A symposium on performance of building constructions, Philadelphia 2-3 April 1984. ASTM Special Technical

Publication 904. Edited by H R Trechsel and P L Lagus. ASTM 1986. p70-98. Tabs, 68 refs.
\#NO2257 Persily, A.K. and Grot, R.A. 1986. Pressurization testing of federal buildings. Measured air leakage of buildings. In: A symposium on performance of building constructions, Philadelphia 2-3 April 1984. ASTM Special Technical Publication 904. Edited by H R Trechsel and P L Lagus. ASTM. p184-200. 3 figs, 5 tabs, 21 refs.
\#NO2258 Verschoor, J.D., and Collins, J. O. 1984. Demonstration of air leakage reduction program in Navy family housing. In: Measured air leakage of buildings. A symposium on performance of building constructions, Philadelphia 2-3 April 1984. ASTM Special Technical Publication 904. Edited by H R Trechsel and P L Lagus. ASTM. p294-303. 5 figs, 1 tab.
\#NO2259 Weimar, R.D., and Luebs, D.F. 1986. Field performance of an air infiltration barrier. In: Measured air leakage of buildings. A symposium on performance of building constructions, Philadelphia 2-3 April 1984. ASTM Special Technical Publication 904. Edited by H R Trechsel and P L Lagus. ASTM. p304-311. 2 figs, 4 tabs.
\#NO2260 Giesbrecht, P., and Proskiw, G. 1986. An evaluation of the effectiveness of air leakage sealing. In: Measured air leakage of buildings. A symposium on performance of building constructions, Philadelphia 2-3 April 1984. ASTM Special Technical Publication 904. Edited by H R Trechsel and P L Lagus. ASTM. p312-322. 1 fig, 3 tabs, 11 refs.
\#NO2378 Tanaka, H., and Lee, Y. 1986. Scale model verification of pressure differentials and infiltration induced across the walls of a high-rise building. J Wind Engng Ind Aerodynam, 125 (1): $1-14,5$ figs, 6 refs.
\#NO2592 Dumont, R.S. 1987. Improved methods for air sealing residences. Proceedings, Fifth Annual Conference, Energy Efficient Buildings Association, Minneapolis, Minnesota, USA, April, 10p, 15 figs, 10 refs.
\#NO2702 Lecompte, J.G.N. 1987. Airtightness of masonry walls. 8th AIVC Conference,
'Ventilation Technology - Research and Application', 21-24 September 1987, Proceedings, Ueberlingen, West Germany, p21.1-21.10, 4 figs, 8 refs.
\#NO2703 Dudek, S.J.M., and Valentine, G. 1987. Condensation damage to timber frame housing. 8th AIVC Conference, 'Ventilation Technology - Research and Application', 21-24 September 1987, Proceedings, Ueberlingen, West Germany, p22.1-22.7, 2 figs, 7 refs.
\#NO2739 Brunsell, J.T. 1987. The effect of vapour barrier thickness on air tightness. 8th AIVC Conference, 'Ventilation Technology - Research and Application', Ueberlingen, Federal Republic of Germany, 21-24 September, Supplement to Proceedings, p63-74, 9 figs, 3 refs.
\#NO2808 Matthews, T.G., Thompson, C.V., and Monar, K.P. 1987. Impact of HVAC operation and leakage on ventilation and intercompartment transport: studies in a research house and 39 Tennessee Valley homes. Indoor Air'87, Proceedings of the 4th International Conference on Indoor Air Quality and Climate, Berlin (West), 17-21 August 1987, Vol 3, Institute for Water, Soil and Air Hygiene, p209-213, 2 figs, 2 refs.
\#NO2857 Nisson J.D.N. 1982. Testing for airtightness. Energy Design Update, September-October, p8-14, 1 fig.
\#NO2861 Anon. 1983. Specifications for airtightness. Energy Design Update, 2(3):7-9, November, 1 fig.
\#NO2866 Anon. 1985. Testing for airtightness - a guide to the blower door depressurization test. Energy Design Update, 4(4):8-23, 11 figs.
\#NO2869 Anon. 1987. Taping for tightness - some astonishing results. Energy Design Update, 6(4):3-9, 1 fig, 4 tabs.
\#NO2881 Love, J.A. 1987. Airtightness testing methods for row housing. ASHRAE Transactions 93(1):1359-1370, 5 figs, 8 tabs, 6 refs.
\#NO3004 Platts, R.E. 1988. Wet walls: apparent incidence of excessive condensation in house envelope construction in Canada. In: Symposium on air infiltration, ventilation and moisture transfer, Fort Worth, Texas, USA, Building Envelope Coordinating Council, p82-90, 4 figs, 5 refs.
\#NO3009 Houston, A.J. 1988. Improved performance standards for polyethylene sheet vapour barriers. in: Symposium on air infiltration, ventilation and moisture transfer, Fort Worth, Texas, USA, Building Envelope Coordinating Council, p141-150. 6 figs, 6 refs.
\#NO3017 Harris, J. 1988. Comparison of measured air leakage rates and indoor air pollutant concentrations with design standards for energy efficient residential buildings. In: Symposium on air infiltration, ventilation and moisture transfer, Fort Worth, Texas, USA, Building Envelope Coordinating Council, p229-242, 13 figs, 9 tabs, 8 refs.
\#NO3019 Swinton, M.C., Moffatt, S., and White, J.H. 1988. Residential combustion venting failures - A systems approach. In: Symposium on air infiltration, ventilation and moisture transfer, Fort Worth. Texas, IJSA. Building Thermal Envelope Coordinating Council, pp 33-48, 10 figs, 1 tab.
\#NO3020 Kehrli, D. 1988. Fenestration air tightness limitations : serviceability/durability. in: Symposium on air infiltration, ventilation and moisture transfer, Fort Worth, Texas, USA, Building Envelope Coordinating Council, p275-280, 3 refs.
\#NO3021 Lischkoff, J., Quirouette, R., and Stritesky, V. 1988. Design, construction and performance evaluation of air barrier systems. In: Symposium on air infiltration, ventilation and moisture transfer, Fort Worth, Texas, USA, Building Envelope Coordinating Council, p281-285.
\#NO3032 Scheuneman, E., and Wilson, A.G. 1988. The impact of commercial air sealing of houses on air tightness and fuel consumption. In: Symposium on air infiltration, ventilation and moisture transfer, Fort Worth, Texas, USA, Building Envelope Coordinating Council, p411-419, 18 tabs, 12 refs.
\#NO3034 Tuluca, A., Keyes, P.A. 1988. Leakage areas for opaque wood frame walls - a preliminary study. In: Symposium on air infiltration, ventilation and moisture transfer, Fort Worth, Texas, USA, Building Envelope Coordinating Council, p433-442, 7 tabs, 4 refs.
\#NO3035 Nantka, M.B. 1988. A study of air infiltration and natural ventilation in dwelling houses. In: Symposium on air infiltration, ventilation and moisture transfer, Fort Worth, Texas, USA, Building Envelope Coordinating Council, p443-455, 7 figs, 10 refs.
\#NO3273 Riley, M. 1988. Comparison of airtightness retesting results. Canada, Ottawa, Energy Mines and Resources, Revised November, 15pp, 5 figs, 5 tabs.
\#NO3332 Dickson, D.J. 1988. Infiltration rates in a low energy house with a fireplace. Building Research and Practice, 4:237-245, 13 figs, 1 ref.
\#NO3365 Gettings, M.B. 1989. Blower door directed infiltration reduction procedure description and field test. ASHRAE Transactions 95(1):58-63, 2 tabs.
\#NO3398 Peterson, R.A., and Hendricks, L.T. 1988. Ceiling airtightness and the role of air barriers and vapour retarders. USA, University of Minnesota, Cold Climate Housing Information Center, 9pp, 10 figs, 8 refs.

## BIBLIOGRAPHY Part II - BY AUTHOR

Anon. 1983. Specifications for airtightness. Energy Design Update, 2(3):7-9.
Anon. 1985. Testing for airtightness - a guide to the blower door depressurization test. Energy Design Update, 4(4):8-23.

Anon. 1987. Taping for tightness - some astonishing results. Energy Design Update, Vol.6, No.4:39.

Bassett, M. 1983. Air infiltration in New Zealand houses. 4th AIC Conference "Air infiltration reduction in existing buildings" ${ }^{\text {S }}$ witzerland, 26-28 September 18:1-14.

Bassett, M.R. 1984. Building site measurements for predicting air infiltration rates.
ASTM Symposium on measured air leakage performance of buildings Philadelphia, USA. April 2-3 1984. STP-904.

Beach, R.K. 1979. Relative tightness of new housing in the Ottawa area. Division of Building Research, National Research Council of Canada. Building research note no. 149 pg. 17.

Blomsterberg A.K., and Harrje D.T. 1979. Approaches to evaluation of air infiltration energy losses in buildings. ASHRAE Transactions 85(1):797-815, also "Evaluating air infiltration energy losses" ASHRAE Journal. 21 (5):25-32.

Blomsterberg, A., Sherman, M.H. and Grimsrud, D.T. 1979. A model correlating air tightness and air infiltration in houses. Proceedings ASHRAE/DOE Conference "Thermal performance of the exterior envelopes of buildings" DEC. 3-5 1979 Florida.

Blomsterberg, A.K., Modera, M.P., and Grimsrud, D.T. 1981. The Mobile Infiltration Test Unit - Its design and capabilities:Preliminary experimental results. LBL Report no. 1225920 pp.

Brandle, K. and Boehm, R.F. 1982. Airflow windows - performance and applications. Proc. ASHRAE/DOE Conference "Thermal performance of the exterior envelope of the building II" USA pg. 361-379.

Brunsell, J.T. 1987. The effect of vapour barrier thickness on air tightness. 8th AIVC Conference, 'Ventilation Technology - Research and Application', Ueberlingen, Federal Republic of Germany, 21-24 September, Supplement to Proceedings, pg. 63-74.

Burch, D.M., Luna, D.E. 1980. A mathematical model for predicting attic ventilation rates required for preventing condensation on roof sheathing. ASHRAE Transactions 86(1):201-220.

Bursey T., Green G.H. 1970. Combined thermal and air leakage performance of double windows. ASHRAE Transactions 76(2):215-226.

Caffey, G.E. 1979. Residential air infiltration. ASHRAE Transactions 85(1):41-57.
Carisson, A., Kronvall, J. 1984. Constancy of air tightness in buildings. 5th AIC Conference 'The implementation and effectiveness of air infiltration standards in buildings' Reno, Nevada, 1-4, pg. 16.1-16.13.

Collins, J.O. 1979. Air infiltration measurement and reduction techniques on electrically heated homes. Proceedings ASHRAE/DOE Conference on "Thermal Performance of the Exterior Envelopes of Buildings "Florida, 3-5 December 1979 28p.

Crall, C.P. 1989. Development of the air infiltration model for the energy performance design system. ASHRAE Transactions $89(2 B): 201-210,2$ figs, 10 refs.

Cunningham, M.J. 1984. Further analytical studies of building cavity moisture concentrations. Building and Environment 19(1):21-28.

Dickerhoff, D.J., Grimsrud, D.T., and Lipschutz, R.D. 1982. Component leakage testing in residential buildings. 1982 Summer Study in Energy Efficient Buildings, Santa Cruz CA August 22-28 1982, LBL Report no. 14735.

Dickinson, J.B., D.T. Grimsrud, D.L., Krinkel, R.D. Lipschutz. 1982. Results of the Bonneville Power Administration Weatherization and Tightening Projects at the Midway Substation residential community. February. LBL Report No. 1272.

Dickinson, J.B., Lipschutz, R.D., O'Regan, B., and Wagner, B.S. 1982. Results of recent weatherization retrofit projects. 1982 Summer Study in Energy Efficient Buildings, Santa Cruz CA August 22-28 1982, LBL Report no. 14734.

Dickson, D.J. 1988. Infiltration rates in a low energy house with a fireplace. Building Research and Practice 4:237-245.

Dudek, S.J.M., and Valentine, G. 1987. Condensation damage to timber frame housing. 8th AIVC Conference, 'Ventilation Technology - Research and Application', 21-24 September, Proceedings, Ueberlingen, West Germany, pg. 22.1-22.7.

Dumont, R.S. 1987. Improved methods for air sealing residences. Proceedings, Fifth Annual Conference, Energy Efficient Buildings Association, Minneapolis, Minnesota, USA, April, 10p.

Dumont, R.S., Orr, H.W., and Lux, M.E. 1982. Low energy prairie housing - a survey of some essential features. DBR Building Practice Note no. 38:pp. 10, 4 tabs, 10 refs.

Figley, D.A., Dumont, R.S. and Orr H.W. 1981. Air tightness measurements of detached houses in the Saskatoon area. Building Research Note no. 178 Division of Building Research NRCC.

Funkhouser, P.E. 1979. Air infiltration effects on the thermal transmittance of concrete building systems. ASHRAE Transactions 85(1):918-925.

Gadsby, K.J., and Harrje, D.T. 1985. Fan pressurization of buildings: standards, calibration, and field experience. ASHRAE Transactions 91 (2B):95-104.

Gammage, R B. 1984. Parameters affecting air leakage in East Tennessee homes. Indoor Air. Vol 5. Buildings, Ventilation and Thermal Climate. Edited by B Berglund, T Lindvall, J Sundell. Swedish Council for Building Research, pp. 429-434.

Gammage, R.B., Hawthorne, A.R. and White, D.A. 1984. Parameters affecting air infiltration and air tightness in 31 east Tennessee homes. ASTM Symposium on measured air leakage performance of buildings Philadelphia USA April 2-3 1984. STP 904, 13 pp., 2 tabs.

Gettings, M.B. 1989. Blower door directed infiltration reduction procedure description and field test. ASHRAE Transactions, 95(1):58-63.

Giesbrecht, P. 1982. An abstract on airtightness in houses. Ener-Corp Management Report. 25pp.
Giesbrecht, P., and Proskiw, G. 1986. An evaluation of the effectiveness of air leakage sealing. In: Measured air leakage of buildings. A symposium on performance of building constructions, Philadelphia 2-3 April 1984. ASTM Special Technical Publication 904. Edited by H R Trechsel and P

Goldschmidt, V.W., and Wilhelm, D.R. 1979. Summer infiltration rates in mobile homes. ASHRAE Transactions 85(1):840-850.

Goldschmidt, V.W. 1986. Average infiltration rates in residences: comparison of electric and combustion heating systems. In: Measured air leakage of buildings. A symposium on performance of building constructions, Philadelphia 2-3 April 1984. ASTM Special Technical Publication 904. Edited by H R Trechsel and P L Lagus. ASTM 1986. pg. 70-98.

Grimsrud, D.T., Sonderegger, R.C., and Sherman M.H. 1981. Infiltration measurements in audit and retrofit programs. Lawrence Berkeley Laboratory report.

Grimsrud, D.T., Sherman, M.H., and Diamond, R.C. et.al. 1979. Infiltration - pressurization correlations: detailed measurements on a California house. ASHRAE Transactions 85(1):851-865.

Grimsrud, D.T. Sherman, M.H., and Sonderegger, R.C. 1982. Calculating infiltration - implications for a construction quality standard. Proc. ASHRAE/DOE Conference "Thermal performance of the exterior envelope of the building $I^{\prime \prime}$ Dec 6-9 1982 USA pg. 422-450.

Grimsrud, D.T., Sherman M.H., Diamond R.C., and Sonderegger R.C. 1979. Air leakage, surface pressures and infiltration rates in houses. 2nd International C.I.B. Symposium on Energy Conservation in the Built Environment, Copenhagen, May 28 - June 1st 1979. Preprints - session 2, pg. 111-120.

Grimsrud, D.T., Sonderegger, R.C., and Sherman, M.H. 1981. Infiltration measurements in audit and retrofit programs. Energy Audit Workshop 13-15 April 1981 SCBR Document D21, pg. 115-139.

Grot, R.A. and Clark, R.E. 1979. Air leakage characteristics of low-income housing and the effectiveness of weatherization techniques for reducing air infiltration. Proceedings ASHRAE/DOE Conference "Thermal performance of the exterior envelopes of buildings" Kissimmee, Florida 3-5 December 1979.

Grubbs W.J. 1967. Leaky prime windows. ASHRAE Journal 9(1):109-112.
Harris, J. 1988. Comparison of measured air leakage rates and indoor air pollutant concentrations with design standards for energy efficient residential buildings. In: Symposium on air infiltration, ventilation and moisture transfer, Fort Worth, Texas, USA, Building Envelope Coordinating Council, pg. 229-242.

Harrje, D.T., Blomsterberg, A., and Persily, A. 1979. Reduction of air infiltration due to window and door retrofits in an older home. Princeton University, Center for Energy and Environmental Studies, report PU/CEES 85, 25 pg.

Harrje, D.T., and Born, G.J. 1982. Cataloging air leakage components in houses.
Center for Energy and Environmental Studies, Princeton University, 22pp.
Herrlin, M.K. 1985. MOVECOMP: a static-multicell-airflow-model. ASHRAE Transactions 91 (2B):1989-1996.

Hollowell, C.D., Young, R.A., Berk, J.V., and Brown S.R. 1982. Energy conserving retrofits and indoor air quality in residential housing. ASHRAE Transactions 88(1):875-893.

Houghten, F.C. and Schrader, C.C. 1924. Air leakage through the openings in buildings. ASHVE Transactions 30:105-120.

Houston, A.J. 1988. Improved performance standards for polyethylene sheet vapour barriers. In: Symposium on air infiltration, ventilation and moisture transfer, Fort Worth, Texas, USA, Building Envelope Coordinating Council, pg. 141-150.

Hunt, C.M. 1980. Air infiltration: A review of some existing measurement techniques and data. In "Building Air Change Rate and Infiltration Measurements" Proceedings, ASTM Conference Gaithersburg 13 March 1978 C.M. Hunt, J.C. King, H.R. Trechsel eds. pp 3-23.

Hunt, C.M., Porterfield, J.M., and Ondris, P. 1978. Air leakage measurements in three apartment houses in the Chicago area. National Bureau of Standards Interagency Report NBSIR 78-1475 24p.

Hunt, C.M., Treado, S.J., and Peavy, B.A. 1976. Air leakage measurements in a mobile home. National Bureau of Standards Interagency report, NBSIR 76-1063, 23 pg.

Hunt C.M., and Burch D. 1975. Air infiltration measurements in a four-bedroom townhouse using sulphur hexafluoride as a tracer gas. ASHRAE Transactions, 81(1):186-201.
lobst, J. et al. 1984. Interior storm windows. Rodale Product Testing Report no. No 117-T. Emmaus, Pennsylvania, USA. Rodale Press.

Jacobson, D., Dutt G S., and Socolow R H. 1984. Pressurization testing, infiltration reduction and energv savinas. PU/CEES Report No. 173, Center for Energy and Environmental Studies, Princeton University, Presented at the ASTM Symposium on the Measured Air Leakage Performance ot Buildings, Philadelphia, 1984.

Keast D.N., and Pei, H-S. 1979. The use of sound to locate infiltration openings in buildings. Proceedings ASHRAE/DOE Conference "Thermal performance of the exterior envelopes of buildings" Florida, December 3-5, 1979 pg. 85-93.

Kehrli, D.W. 1985. Window air leakage performance as a function of differential temperatures and accelerated environmental aging. Rochester, New York, USA:Schlegel Corporation. 28p.

Kehrli, D. 1988. Fenestration air tightness limitations : serviceability/durability. In: Symposium on air infiltration, ventilation and moisture transfer, Fort Worth, Texas, USA, Building Envelope Coordinating Council, pg. 275-280.

Kim, A., and Shaw, C.Y. 1984. Seasonal variation in airtightness of two detached houses. ASTM Symposium on measured air leakage performance of buildings, Philadelphia, USA, April 2-3 1984. STP-904.

Klems, J.H. 1983. Methods of estimating air infiltration through windows. Energy and Buildings 5(4):243-252.

Klote, J.H. 1985. Smoke control in VA hospitals. ASHRAE Journal, 27(4):42-45.
Krinkel, D.L., Dickeroff, D.J., Casey, J. and Grimsrud, D.T. 1980. Pressurization test results:
Bonneville Power Administration Energy Conservation Study. LBL Report no. 10996, 13pp.
Kronvall, J.Air. 1978. Leakage of buildings-a literature list. Lund Institute of Technology, Divisision of Building Technology, report 77.

Lagus, P.L., and King, J.C. 1984. Air leakage and fan pressurization measurements in selected naval housing. In: Measured air leakage in buildings. A symposium on performance of building constructions, Philadelphia, 2-3 April. ASTM Special Technical Publication 904. Edited by H R Trechsel and P L Lagus. ASTM, pg. 5-16, 4 figs, 6 tabs, 9 refs.

Lecompte, J.G.N. 1987. Airtightness of masonry walls. 8th AIVC Conference, 'Ventilation Technology - Research and Application, 21-24 September, Proceedings, Ueberlingen, West Germany, pg. 21.1-21.10.

Lipschutz, R.D., Dickinson, J.B., and Diamond, R.C. 1982. Infiltration and leakage measurements in new houses incorporating energy efficient features. 1982 Summer Study in Energy Efficient Buildings Santa Cruz CA, 22-28 August, LBL Report no. 14733.

Lipschutz, R.D., Girman, J.R., Dickinson, J.B., Allen, J.R., and Traynor, G.W. 1981. Infiltration and indoor air quality in energy efficient houses in Eugene, Oregon. Berkeley, California, USA. Lawrence Berkeley Laboratory, LBL-12924 UC-95d. 49p.

Lischkoff, J., Quirouette, R., and Stritesky, V. 1988. Design, construction and performance evaluation of air barrier systems. In: Symposium on air infiltration, ventilation and moisture transfer, Fort Worth, Texas, USA, Building Envelope Coordinating Council, pg. 281-285.

Love, J.A. 1987. Airtightness testing methods for row housing. ASHRAE Transactions 93(1):1359-1370.

Lowinski, J.F. 1979. Thermal performance of wood windows and doors. ASHRAE Transactions 85(1):548-566.

Matthews, T.G., Thompson, C.V., and Monar, K.P. 1987. Impact of HVAC operation and leakage on ventilation and intercompartment transport: studies in a research house and 39 Tennessee Valley homes. Indoor Air'87, Proceedings of the 4th International Conference on Indoor Air Quality and Climate, Berlin (West), 17-21 August 1987, Vol 3, Institute for Water, Soil and Air Hygiene, pg. 209-213.

Modera, M.P., and Sherman, M. H. 1985. AC pressurization: a technique for measuring leakage area in residential buildings. ASHRAE Transactions 91 (2B):120-132.

Nagda, N.L., Harrje, D.T., Koontz, M.D. and Purcell, G.G. 1984. A detailed investigation of the air infiltration characteristics of two houses. ASTM Symposium on measured air leakage performance of buildings Philadelphia USA April 2-3 1984. STP-904.

Nantka, M.B. 1988. A study of air infiltration and natural ventilation in dwelling houses. In: Symposium on air infiltration, ventilation and moisture transfer, Fort Worth, Texas, USA, Building Envelope Coordinating Council, pg. 443-455.

Nisson J.D.N. 1982. Testing for airtightness. Energy Design Update, September-October, pg. 8-14.
Offermann, F.J., Gurman, J.R., and Hollowell, C.P. 1981. Midway house-tightening project: a study of indoor air quality. LBL Report no. 12777 27pp.

O'Riordan, M.C., James, A.C., Rae S. and Wrixon, A.D. 1983. Human exposure to radon decay products inside dwellings in the United Kingdom. National Radiological Protection Board R152 41pp.

Persily, A. 1982. Repeatability and accuracy of pressurization testing. Proc. ASHRAE/DOE Conference "Thermal performance of the exterior envelope of the building II" Dec 6-9 USA p.380-390.

Persily, A.K. 1986. Measurements of air infiltration and airtightness in passive solar homes. In: Measured air leakage of buildings. A symposium on performance of building constructions, Philadelphia, 2-3 April. ASTM Special Technical Publication 904. Edited by H R Trechsel and P L Lagus. ASTM. p46-60.

Persily, A.K. and Grot, R.A. 1983. Air infiltration and building tightness measurements in passive solar residences. Reprint Solar Engineering ( ):116-121.

Persily, A.K. and Grot, R.A. 1985. Accuracy in pressurization data analysis. ASHRAE Transactions $91(2): 105-119$.

Persily, A.K. and Grot, R.A. 1986. Pressurization testing of federal buildings. In: Measured air leakage of buildings. A symposium on performance of building constructions, Philadelphia 2-3 April 1984. ASTM Special Technical Publication 904. Edited by H R Trechsel and P L Lagus. ASTM. pg. 184-200.

Persily, A.K., and Linteris, G.T. 1983. A comparison of measured and predicted infiltration rates. ASHRAE Transactions 89(2B):183-200.

Peterson, R.A., and Hendricks, L.T. 1988. Ceiling airtightness and the role of air barriers and vapour retarders. USA, University of Minnesota, Cold Climate Housing Information Center, 9pp. Platts, R.E. 1988. Wet walls: apparent incidence of excessive condensation in house envelope construction in Canada. In: Symposium on air infiltration, ventilation and moisture transfer, Fort Worth, Texas, USA, Building Envelope Coordinating Council, pg. 82-90.

Poreh, M, and Hassid, S. 1982. Simulation of buoyancy and wind induced ventilation. Wind Tunnel Modeling for Civil Engineering Applications. Proceedings of the International Workshop on Wind Tunnel Modeling Criteria and Techniques in Civil Engineering Applications, Gaithersburg, USA, April 14-16. Edited by T A Reinhold. Cambridge, UK:CUP. pp. 558-566.

Potter, I.N. 1979. Effect of fluctuating wind pressures on natural ventilation. ASHRAE Transactions 85(2):445-457.

Quackenboss, J.J. 1984. Residential indoor air quality, structural leakage and occupant activities for 50 Wisconsin homes. Indoor Air. Vol 5. Buildings, Ventilation and Thermal Climate. Edited by B. Berglund, T. Lindvall, J. Sundell. Swedish Council for Building Research, pp 411-420.

Reinhold, C., and Sonderegger, R. 1983. Component leakage areas in residential buildings. 4th AIVC Conference "Air infiltration reduction in existing buildings" Switzerland, 26-28 September, pg. 16.1-16.3.

Retrospectors, Inc. 1981. Air tightness testing and sealing of homes in Ottawa, Ontario. Ontario Ministry of Municipal Affairs and Housing, Report no. 11501 13pp.

Riley, M. 1988. Comparison of airtightness retesting results. Canada, Ottawa, Energy Mines and Resources, Revised November, 15pp.

Sasaki, J.R. 1968. Air leakage characteristics of some brick and concrete block walls. National Research Council of Canada, Division of Building Research, Technical note no. 525:5, 2 figs, 1 tab.

Sasaki, J.R. and Wilson, A.G. 1962. Window air leakage. National Research Council Canada, Division of Building Research, Building Digest no. 25.

Sasaki, J.R., and Wilson A.G. 1965. Air leakage values for residential windows.
ASHRAE Transactions 71 (2):81-88, National Research Council of Canada, Division of Building Research paper no. 329.

Sasaki, J.R. 1973. Air leakage testing. Spec. Ass. 15(5):15-18, N.R.R.C. Division of Building Research technical paper no. 407.

Scanada Consultants Ltd. 1979. Effect of high levels of insulation on the heating fuel consumption of Canadian houses. Report for the HUDAC Technical Research Committee, Canada, Project T80-78-30.

Scheuneman, E.C. 1982. Mark XI energy research project. Summary of results 1978-1981. Building Research Practice Note no. 27 National Research Council of Canada, 18pp.

Scheuneman, E.C., and Wilson, A.G. 1988. The impact of commercial air sealing of houses on air tightness and fuel consumption. In: Symposium on air infiltration, ventilation and moisture transfer, Fort Worth, Texas, USA, Building Envelope Coordinating Council, pg. 411-419.

Schutrum L.F., Ozisik N., Baker J.T., and Humphreys C.M. 1961. Air infiltration through revolving doors. ASHRAE Journal 3(11):43-50.

Shah, M.M. 1980. Estimated rate of pressurization and depressurization of buildings. ASHRAE Transactions 86(1):251-257.

Shaw, C.Y. 1980. Methods for conducting small-scale pressurization tests, and air leakage data of multi-storey apartment buildings. ASHRAE Transactions 86(1):241-250.

Shaw, C.Y. 1981. A correlation between air infiltration and airtightness for houses in a developed residential area. ASHRAE Transactions 87(2):333-341.

Shaw, C.Y., and Brown, W.C. 1982. Effect of a gas furnace chimney on the air leakage characteristic of a two-storey detached house. Preprint 3rd AIC Conference "Energy efficient domestic ventilation systems" London 20-23 September, 1982 7pp.

Shaw C.Y., Sander D.M., and Tamura G.T. 1973. Air leakage measurements of the exterior walls of tall buildings. ASHRAE Transactions 79(2):40-48. Research paper no. 601.

Shaw, C.Y. and Tamura, G.T. 1980. Mark XI energy research project, Air tightness and air infiltration measurements. Division of Building Research, National Research Council of Canada. Building Research Note no. 162, Ottawa, 7 pg.

Sherman, M.H., Grimsrud, D.T. 1980. Infiltration-pressurization correlation: simplified physical modeling. ASHRAE Transactions 82(2):778-807, LBL no. 10163.

Sherman, M.H., Grimsrud, D.T., and Sonderegger, R.C. 1979. Low pressure leakage function of a building. Proceedings ASHRAE/DOE Conference "Thermal performance of the exterior envelopes of buildings" Dec. 3-5 1979 Florida.

Silberstein, S. and Grot, R.A. 1985. Air exchange rate measurements of the National Archives Building. ASHRAE Transactions $91(2 A): 503-510$.

Stewart, M.B., Jacob, T.R., and Winston, J.G. 1979. Analysis of infiltration by tracer gas technique, pressurization tests and infrared scans. Proceedings ASHRAE/DOE Conference "Thermal performance of the exterior envelopes of buildings" Florida December 3-5th.

Stricker S. 1975. Measurement of air tightness of houses. ASHRAE Transactions, 81 (1):148-167.
Stricker, S. 1974. Measurement of air leakage of houses. Ontario Hydro Research quarterly, 26(4):11-18.

Sulatisky, M. 1989. Air tightness tests on 200 new houses across Canada. Summary of results. Canada Buildings Energy Technology Transfer Program publication 84.01.

Swinton, M.C., Moffatt, S., and White, J.H. 1988. Residential combustion venting failures - a systems approach. In: Symposium on air infiltration, ventilation and moisture transfer, Fort Worth, Texas, USA, Building Thermal Envelope Coordinating Council, pg. 33-48.

Tamura, G.T. 1979. The calculation of house infiltration rates. ASHRAE Transactions 85(1):58-71.
Tamura, G. 1975. Measurement of air leakage characteristics of house enclosures. ASHRAE Schrader, C.C. 1924. Air leakage around window openings. ASHVE Transactions 30:313-322.

Tamura, G.T. 1974. Predicting air leakage for building design. 6th C.I.B. Congress on the Impact of Research on the Built Environment Budapest 3-10 October 1974 preprints vol 1/1 pg. 368-374, D.B.R. technical paper no. 437.

Tamura, G.T. and Shaw, C.Y. 1976. Air leakage data for the design of elevator and stair shaft pressurization system. ASHRAE Transactions 82(2):179-190.

Tamura, G.T. and Shaw C.Y. 1976. Studies on exterior wall air tightness and air infiltration of tall buildings. ASHRAE Transactions 82(1):122-134, N.R.C.C. Building Research paper no. 706. Tanaka, H., and Lee, Y. 1986. Scale model verification of pressure differentials and infiltration induced across the walls of a high-rise building. J Wind Engng Ind Aerodynam, 25(1):1-14.

Thorogood R.P. 1979. Resistance to air flow through external walls. Building Research Establishment Information Paper.

Tietsma, G.J., Peavy, B.A. 1978. The thermal performance of a two-bedroom mobile home. National Bureau of Standards Building Science Series, 102. 55 pg. 1.

Treado, S.J., Burch, D.M., Hunt, C.M. 1979. An investigation of air infiltration characteristics and mechanisms for a townhouse. National Bureau of Standards Technical Note, 992, 31 pg.

Trechsel, H.R., Achenbach, P.R., and Ebbets, J.R. 1985. Effect of an exterior air-infiltration barrier on moisture condensation and accumulation within insulated frame wall cavities. ASHRAE Transactions 91 (2A):545-559.

Tsongas, G.A., Odell, F.G., and Thompson, J.C. 1979. A field study of moisture damage in walls insulated without a vapour barrier. Proceedings ASHRAE/DOE Conference "Thermal performance of the exterior envelopes of buildings" 3-5 December 1979 Florida.

Tuluca, A., Keyes, P.A. 1988. Leakage areas for opaque wood frame walls - a preliminary study. In: Symposium on air infiltration, ventilation and moisture transfer, Fort Worth, Texas, USA, Building Envelope Coordinating Council, pg. 433-442.

USA Department of Energy. 1983. Energy efficient windows. A key to energy performance. USA Dept. of Energy.

Verschoor, J.D., and Collins, J. O. 1984. Demonstration of air leakage reduction program in Navy family housing. In: Measured air leakage of buildings. A symposium on performance of building constructions, Philadelphia 2-3 April 1984. ASTM Special Technical Publication 904. Edited by H R Trechsel and P L Lagus. ASTM. p294-303.

Weidt, J.L., Weidt, J., and Selkowitz, S. 1979. Field air infiltration performance of new residential windows. Proceedings. ASHRAE/DOE Conference "Thermal performance of the exterior envelopes of buildings" 3-5. Dec. 1979 Florida, Lawrence Berkeley Laboratory report LBL 9937.

Weimar, R.D., and Luebs, D.F. 1986. Field performance of an air infiltration barrier. In: Measured air leakage of buildings. A symposium on performance of building constructions, Philadelphia 2-3 April 1984. ASTM Special Technical Publication 904. Edited by H R Trechsel and P L Lagus. ASTM. pp.304-311.

Wilson, A.G. and Sasaki, J.R. 1972. Evaluation of Window Performance. Division of Building Research, NRCC, Ottawa, Canada. Nat. Bur. Stand. Special Publication 361, pp.385-394 Ottawa, Canada.

Yuill, G. 1985. Determination of the effective leakage areas of houses by multilinear regression analysis of the energy consumption data. ASHRAE Transactions 91 (2B):133-143.

## APPENDIX A

Listing of Literature Leakage Values




A-5




|  | er | crass |  | case | $\left\lvert\, \begin{array}{l\|} \hline \mathrm{D} \\ \mathrm{~A} \end{array}\right.$ |  | $\begin{gathered} C \\ \hline \text { value } \\ \text { Mixed } \end{gathered}$ | $\begin{gathered} n \\ \text { value } \end{gathered}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 354 | 1261 | WIDSNW |  |  |  |  |  |  |
| 355 | 1514 | WDSW | W-14-18 |  | 3 |  |  |  |
| 358 | 1514 | WIDSW | W-3 |  | 3 |  |  |  |
| 357 | 1281 | WDSW |  |  |  |  |  |  |
| 358 | 458 | WDSW |  | 27 | $\theta$ |  |  |  |
| 359 | 458 | WDSW |  | 33 | 9 |  |  |  |
| 360 | 458 | WDSW |  | 6 | 9 |  |  |  |
| 361 | 942 | WF |  |  |  |  |  |  |
| 362 | 1514 | WFM | W-5 |  | 3 |  |  |  |
| 363 | 1514 | WFM | W-8 |  | 3 |  |  |  |
| 364 | 1261 | WFM |  |  |  |  |  |  |
| 365 | 1261 | WFM |  |  |  |  |  |  |
| 368 | 311 | WIFM | cq2 | 1 | 4 | 8 | 0.02 | 0.68 |
| 367 | 1514 | WFW | W-5 |  | 3 |  |  |  |
| 368 | 1514 | WIFW | W-8 |  | 3 |  |  |  |
| 368 | 1261 | WIFW |  |  |  |  |  |  |
| 370 | 1281 | WIFW |  |  |  |  |  |  |
| 371 | 208 | WFW | lab 1 |  |  |  |  |  |
| 372 | 339 | WFW | 1 |  |  |  |  |  |
| 373 | 1261 | WL |  | 19 |  |  |  |  |
| 374 | 1281 | WL |  | 11 |  |  |  |  |
| 375 | 82 | WL |  | 50 | 2 |  |  |  |
| 376 | 92 | WL. | H1 | 2 | 2 |  |  |  |
| 377 | 92 | WL |  |  |  |  |  |  |
| 378 | 2257 | WL |  |  |  |  |  |  |
| 379 | 1721 | WLL |  |  | 1 |  |  |  |
| 380 | 176 | WLL |  |  |  |  |  |  |
| 381 | 1721 | WLL |  |  | 1 |  |  |  |
| 382 | 176 | WL |  |  | 5 |  |  |  |
| 383 | 178 | WLL |  |  | 5 |  |  |  |
| 384 | 458 | WL |  | 192 | 1 |  |  |  |
| 385 | 1357 | WO | 1 |  | 3 |  |  |  |
| 386 | 1277 | wo |  | 1 | , |  |  |  |
| 387 | 127 | WQ |  | 1 | 3a |  |  |  |
| 388 | 1277 | WGa |  | 1 | 3a |  |  |  |
| 389 | 708 | WQ | H4 | 1 | 4 | 6 | 0.13 | 0.33 |
| 390 | 311 | WSHS | Bldg A | 1 | 4 | 6 | 0.1 | 0.74 |
| 391 | 311 | WSHS | Bldg A | 1 | 4 | 6 | 0.15 | 0.68 |
| 392 | 311 | WSHS | Bldg C | 1 | 4 | e | 0.02 | 0.72 |
| 393 | 311 | WSHS | Bldg A | 1 | 4 | - | 0.17 | 0.69 |
| 394 | 311 | WSHS | Bldg V | 1 | 4 | 6 | 0.07 | 0.67 |
| 395 | 311 | WSHS | Bldg V | 1 | 4 | - | 0.1 | 0.68 |
| 396 | 311 | WSHS | Bldg C | 1 | 4 | 6 | 0.03 | 0.83 |
| 397. | 311 | WSHS | Bldg C | 1 | 4 | 6 | 0.04 | 0.77 |
| 398 | 311 | WISHS | Bldg C | 1 | 4 | 8 | 0.03 | 0.72 |
| 399 | 311 | WSHS | Bldg C | 1 | 4 | 8 | 0.01 | 1.01 |
| 400 | 311 | WSHS | Bldg A | 1 | 4 | 8 | 0.09 | 0.72 |
| 401 | 311 | WSHS | Bldg ${ }^{\text {T }}$ | 1 | 4 | 7 | 0.06 | 0.67 |
| 402 | 311 | WSHS | Bldg V | 1 | 4 | 8 | 0.05 | 0.75 |
| 403 | 311 | WSHS | Bldg T | 1 | 4 | 7 | 0.03 | 0.64 |
| 404 | 311 | WSHS | Bldg C | 1 | 4 | 6 | 0.03 | 0.71 |
| 405 | 458 | WSHS |  | 8 | 9 |  |  |  |
| 406 | 458 | WSHS |  | 3 | 9 |  |  |  |
| 407 | 458 | WSHS |  | 31 | $\theta$ |  |  |  |
| 408 | 458 | WSHS |  | 22 | 9 |  |  |  |
| 409 | 1514 | WISHSN | W-16 |  | 3 |  |  |  |
| 410 | 1514 | WSHSN | W-4 |  | 3 |  |  |  |
| 411 | 1261 | WSHSNW |  |  |  |  |  |  |
| 412 | 1514 | WISHSW | W-16-18 |  | 3 |  |  |  |



## APPENDIX B

## Listing of Literature Leakage Values

With Calculated ELA using 4 Pa and $\mathrm{C}_{\mathrm{d}}=1$









# APPENDIX C Notes to Literature Leakage Tables 

Legend for Data Testing Coding - Column 6

(Reported by the source)

1. Single reading at reference pressure-75 Pa
2. Single reading at reference pressure - 62 Pa
3. Single reading at reference pressure-50 Pa

3a. $\quad$ Single reading at reference pressure - 26.8 $\mathrm{Pa}(15 \mathrm{mph})$
3b. Single reading at some other reference pressure
4. Multiple readings over a pressure range with eqn for readings
5. Multiple readings over a pressure range - calculated at $P_{\text {ref }}$
6. E779-81
7. CGSB CAN2-149.10
8. ASTM C-236
9. ASTM C-283
A. Multiple readings over a pressure range - data for $26.8 \mathrm{~Pa}(15 \mathrm{mph})$

## Legend for C and n Coding - Column 7

1. Equation originally given (SI)
2. Equation given I-P not converted yet
3. Equation given I-P still in I-P
4. Equation comes from subtraction of two equations given
5. Data given - Equation regressed by RP 438
6. Data points shown on graph - digitized and regressed by RP 438
7. Graph given - points selected, digitized and regressed by RP 438
8. Equation converted to SI

## Legend for Note \#'s - Column 24

1. Data point pulled from graph
2. Data pulled from graph and regress equation
3. Data values given, equation regressed by RP438
4. Data from regression equation given
5. Data from difference between open and closed
6. Lightweight concrete block wall with polystyrene pellets in the cores $12 \times 8 \times 16$ ( 2 core) 2 cores latex on the exterior
7. Fixed glazing area $24 \%$ to $38 \%$ of wall area
8. Lightweight concrete block (expanded clay aggregate) 2 core ( $8 \times 8 \times 16$ )
9. Lightweight concrete block (expanded slag aggregate) 3 core ( $8 \times 8 \times 16$ )
10. Concrete block (sand and gravel aggregrate) 3 core ( $8 \times 8 \times 16$ )
11. Two rows brick ( $23 / 8 \times 33 / 4 \times 8$ ) with 2 " cavity
12. Single row SCR brick ( $21 / 6 \times 51 / 2 \times 111 / 2$ ) with $3 / 8$ vented furring strips, sheathing paper, furring strips with fiberglass insulation, vapor barrier,
tempered wallboard
13. Same as 107 except $3 / 8^{\prime \prime}$ air space not vented.

109a. Single stud wall conventional vapor barrier location - whole house values
109b. Single stud wall sandwiched vapor barrier location - whole house values
110. Double stud wall conventional vapor barrier locaton - whole house values
111. Double stud wall sandwiched vapor barrier location - whole house values
112. 50 Pa with $C_{d}=0.611$
113. 50 Pa with orifice equation $\mathrm{C}_{\mathrm{d}}=0.60$
201. $\quad Q=C A(\Delta P)^{n}$ where $A=$ area of building envelope, $228 \mathrm{~m}^{2}$
301. Only if in unconditioned space
302. Only if in conditioned space
303. Unheated flue with 0.15 m diameter, 0.075 m restriction orfice and rain cap
304. Area, vol, C, n, SLA, $\mathrm{ACH}_{50}, \mathrm{ACH}$ given for each house (SLA based on envelope area)
305. Fan on - flow is corrected for standard restrictions like $1 / 4$ " screening, louvers, elbows, straight duct and grease filter
306. Leakage areas for opague walls: data values, $C, n, r^{2}$ also given in paper, no significant differences between sidings
307. Gives house specifics (including ELA, SLA and average ACH for each) for 312 houses which have been found in literature - big whole house leakage data set
308. Mobile home in lab-storm windows inside louvered jalousie type windows. Window area of $6 \mathrm{~m}^{2}$
309. "Very leaky" - loosely fitting window, much worst than average ( $1.3 \mathrm{cfm} / \mathrm{ft}$ @ 25 Pa )
310. "Leaky" - Average fit, unweatherstripped or weatherstripped with loose fit (0.45 cfm @ 25 Pa )
311. "Tight" ( $0.5 \mathrm{cfm} / \mathrm{ft}$ @ 75 Pa )
312. $A=Q / 2400(\Delta P)^{0.5}$
313. Negative values were given in some cases becasue paint seal was broken
314. Exclusive of windows and doors, but including leakage between wall and door and window frames [ area term given by $A=Q /\left(2400\left(\Delta P 0.3^{\prime \prime}\right)^{0.5}\right)$ ]
315. Approximately $17 \mathrm{ft} \mathrm{crack} /$ window
316. Exterior wall leakage rates on tall buildings
317. Subtraction of registers sealed measurement value from registers unsealed value
318. Units $=\mathrm{m}^{3} / \mathrm{s}-1000 \mathrm{~m}^{2}$

Note: The numbers are not sequential due to not using and/or combining or deleting some of the raw data.

## APPENDIX D <br> Conversion Factors

Physical Quantity

Length in Area

Volume

Mass
Density
Flow
Flow
Velocity

Pressure

Spec Leakage area

To convert From To
0.0254

| m | 0.3048 |
| :--- | :--- |
| $\mathrm{~m}^{2}$ | 0.09294 |
| $\mathrm{~cm}^{2}$ | 6.452 |
| $\mathrm{~m}^{3}$ | 0.02832 |
| l | 28.32 |
| kg | 0.4536 |
| $\mathrm{~kg} / \mathrm{m}^{3}$ | 16.02 |
| $\mathrm{~m}^{3} / \mathrm{s}$ | $4.719 * 10^{-4}$ |
| $\mathrm{l} / \mathrm{s}$ | 0.4719 |
| $\mathrm{~m} / \mathrm{s}$ | 0.00508 |
| $\mathrm{~m} / \mathrm{s}$ | 0.44704 |
| $\mathrm{~km} / \mathrm{h}$ | 1.609 |
| Pa | 248.66 |
| Pa | 3386.4 |


| $\mathrm{in}^{2} / \mathrm{ft}^{2}$ | $\mathrm{~cm}^{2} / \mathrm{m}^{2}$ | 69.44 |
| :--- | :--- | :--- |
| $\mathrm{~cm}^{2} / \mathrm{ft}^{2}$ | $\mathrm{~cm}^{2} / \mathrm{m}^{2}$ | 0.0929 |
| $\mathrm{~cm}^{2} / \mathrm{ft}$ | $\mathrm{cm}^{2} / \mathrm{m}$ | 0.3048 |
| $\mathrm{~cm}^{2} / 100 \mathrm{ft}$ | $\mathrm{cm}^{2} / \mathrm{m}$ | 30.48 |


| $\mathrm{cm}^{2} / \mathrm{m}^{2}$ | $\mathrm{in}^{2} / \mathrm{ft}^{2}$ |
| :--- | :--- |
| $\mathrm{~cm}^{2} / \mathrm{m}^{2}$ | $\mathrm{~cm}^{2} / \mathrm{ft}^{2}$ |
| $\mathrm{~cm}^{2} / \mathrm{m}$ | $\mathrm{cm}^{2} / \mathrm{ft}$ |
| $\mathrm{cm}^{2} / \mathrm{m}$ | $\mathrm{cm}^{2} / 100 \mathrm{ft}$ |
| $\mathrm{cm}^{2} / \mathrm{m}$ | in 2 $/ \mathrm{ft}$ |

0.0144
10.76
3.281
0.0328
0.5086
3.80
0.636
6949.
11.81
0.1969
196.8

Other Conversions:

```
\(1 \mathrm{~N}=1 \mathrm{~kg}-\mathrm{m} / \mathrm{s}^{2}\)
Pressure (in wg) \(=0.000482 \mathrm{~V}^{2} \quad(\mathrm{mph})\)
Pressure \((\mathrm{Pa})=0.1200575 \mathrm{~V}^{2}(\mathrm{mph})\)
Pressure \((\mathrm{Pa})=0.601 \mathrm{~V}^{2}(\mathrm{~m} / \mathrm{s})\)
Pressure \((\mathrm{Pa})=0.601[(1 / 3.6) \mathrm{V}]^{2} \quad(\mathrm{~km} / \mathrm{h})\)
\(15 \mathrm{mph}=27 \mathrm{~Pa}, 25 \mathrm{mph}=75 \mathrm{~Pa}\)
Standard Air Density \(=0.075 \mathrm{lb} / \mathrm{ft}^{3}=1.20 \mathrm{~kg} / \mathrm{m}^{3}\)
```


## APPENDIX E

Derivation of Dimensionless Crack Flow Equation
From Sun, 1992

## THEORY AND DEVELOPMENTS

### 3.1 Theory Preparation

As mentioned in previous chapters, three basic alternative equations (orifice equation, power equation and dimensionless flow equation) are used in the airtightness literature. It is important to demonstrate their developments and indicate some of the assumptions involved. From these examples we might get some ideas for improvements which will enable us to make equation(s) which more closely describe the natural performance of crack flow.

The fundamental theorem to describe the airtightness problem is the Bernoulli equation. It is widely used in hydrodynamics, especially in one dimensional steady flow problems. The basic point of the Bernoulli equation is the energy balance in which the pressure changes and velocity along a streamline are related. For crack flow, the flow velocity and flow rate have a simple relation. Hence with the help of the Bernoulli equation, we can obtain the pressure and flow relationship which governs airtightness and crack flow problems.

### 3.1.1 Introduction of the derivation of Bernoulli equation based on energy conservation law

The first law of thermodynamics applies to a thermodynamic, volume system which is originally at rest and after some event, is finally at rest again. Under these conditions, it is stated that the "change in internal energy, due to the event, is equal to the sum of the total work done on the system during the course of the event and any heat which was added " (Currie, 1974). Consider a control volume as shown below:


Figure 3-1. Differential controll volume of a flow pipe
$\mathrm{d} \tau$-----differential control volume
dA-----differential control surface
$\overrightarrow{\mathrm{n}}$----unit normal to the surface of the body
$\overrightarrow{\mathrm{v}}_{1}, \overrightarrow{\mathrm{v}}_{2}$--flow velocity vectors
$\mathrm{A}_{1}, \mathrm{~A}_{2}$--cross sectional areas
Control volume---a finite length mini stream tube which is a region whose sidewalls are made up of streamlines.

The energy of the fluid consists of two parts (on the basis of per unit mass) : 1) internal energy, $e$, and 2) the kinetic energy, $(\vec{v} \cdot \vec{v}) / 2$. Hence the total energy contained in the control volume will be $\int_{\tau} \rho(e+((\vec{v} \cdot \vec{v}) / 2) d \tau$.

Two types of external forces which may act on the fluid mass are the body force and the surface stress. Body force per unit mass is denoted by the vector, $\overrightarrow{\mathrm{f}}$. Then the total work done due to this body force will be $\int_{\tau} \overrightarrow{\mathrm{V}} \cdot \rho \stackrel{\rightharpoonup}{\mathrm{f}} \mathrm{d} \tau$. The magnitude of the surface stress per unit area is represented by the vector, $\stackrel{\rightharpoonup}{P}_{n}$. Then the total work done is $\int_{A} \vec{v} \cdot \vec{P}_{n} d A$. Finally, if the vector $\vec{q}$ denotes the conductive heat flux leaving the control volume, then the quantity of heat leaving the fluid mass per unit time per unit surface area will be $\vec{q} \cdot \stackrel{\rightharpoonup}{n}$, and the net amount of heat leaving the fluid per unit time will be $\int_{A} \vec{q} \cdot \vec{n} d A$.

The law of energy conservation requires that the rate of change of total energy is equal to the rate at which work is being done plus the rate at which heat is being added, that is:

$$
\begin{equation*}
\frac{D}{D t} \int_{\tau} \rho\left(e+\frac{1}{2} \vec{v} \cdot \vec{v}\right) d \tau=\oint_{A} \vec{v} \cdot \vec{P}_{r} d A+\int_{\tau} \vec{v} \cdot \rho \vec{f} d \tau-\oint_{A} \vec{q} \cdot \vec{n} d A \tag{3.1}
\end{equation*}
$$

where Lagrangian derivatives D/Dt is employed to a specific mass of fluid which is arbitrarily chosen. For the control volume system, we may convert to Eulerian derivatives by use of Reynolds' Transport Theorem, suppose any physical parameter

$$
\alpha=\alpha(\mathrm{x}, \mathrm{y}, \mathrm{z}, \mathrm{t})
$$

$$
\frac{D}{D t} \int_{\tau} \alpha d \tau=\int_{\tau}\left[\frac{\partial \alpha}{\partial t}+\nabla \cdot(\alpha \vec{v})\right] d \tau
$$

Then the left side of equation [3.1] is,
$\frac{D}{D t} \int_{\tau} \rho\left(e+\frac{1}{2} \vec{v} \cdot \vec{v}\right) d \tau$
$=\int_{\tau} \frac{\partial}{\partial t}\left(\rho e+\frac{1}{2} \rho \vec{v} \cdot \vec{v}\right) d \tau+\int_{\tau} \nabla \cdot\left[\rho \vec{v}\left(e+\frac{1}{2} \vec{v} \cdot \vec{v}\right)\right] d \tau$
$=\int_{\tau} \frac{\partial}{\partial t}\left(\rho e+\frac{1}{2} \rho \vec{v} \cdot \vec{v}\right) d \tau+\oint_{A} \vec{n} \cdot \rho \vec{v}\left(e+\frac{1}{2} \vec{v} \cdot \vec{v}\right) d A$
( using Gauss theorem in the second term )

If we assume,

1) steady flow, $\partial / \partial t=0$, that is, $\partial / \partial t(\rho e+1 / 2 \rho \vec{v} \cdot \vec{v})=0$
2) no heat transfer involved, $|\stackrel{\rightharpoonup}{q}|=0$
3) body force is conservative, such as gravity, then $\overrightarrow{\mathbf{f}}$ may be written as the gradient of some scaler function $U$, that is, $\overrightarrow{\mathrm{f}}=-\nabla \mathrm{U}$
4) inviscid fluid without shear stress to resist deformation, therefore the normal stress is the only stress exerted on the surface, hence $\overrightarrow{\mathrm{P}}_{\mathrm{n}}=-\overrightarrow{\mathrm{n}} \cdot \mathrm{P}$, which means the surface stress is the pressure in outward normal direction. P is the static pressure.
is:

$$
\oint_{A} \overrightarrow{\mathrm{v}} \cdot \vec{P}_{\mathrm{P}} \mathrm{dA}=\oint_{A} \overrightarrow{\mathrm{v}} \cdot(-\overrightarrow{\mathrm{n}} \mathrm{P}) \mathrm{dA}
$$

$$
=-\oint_{\mathrm{A}} \overrightarrow{\mathrm{v}} \cdot \overrightarrow{\mathrm{n}} \mathrm{PdA}
$$

the second term of the right side is (assuming $\rho$ constant):

$$
\int_{\tau} \overrightarrow{\mathrm{v}} \cdot(\rho \overrightarrow{\mathrm{f}}) \mathrm{d} \tau=\int_{\tau} \rho \overrightarrow{\mathrm{v}} \cdot \overrightarrow{\mathrm{f}} \mathrm{~d} \tau
$$

$=\int_{\tau} \rho \overrightarrow{\mathrm{v}} \cdot(-\nabla \mathrm{U}) \mathrm{d} \tau=\int_{\tau}-\rho(\overrightarrow{\mathrm{v}} \cdot \nabla \mathrm{U}) \mathrm{d} \tau$

$$
\left.=\int_{\tau}-\rho[\nabla \cdot(\overrightarrow{\mathrm{V}} \mathrm{U})-U \nabla \cdot \overrightarrow{\mathrm{~V}})\right] \mathrm{d} \tau
$$

$$
=-\int_{\tau} \rho \nabla \cdot(\vec{v} U) d \tau+\int_{\tau} \rho U \nabla \cdot \vec{v} d \tau
$$

$$
\left.=-\int_{\tau} \rho \nabla \cdot(\overrightarrow{\mathrm{v}} \mathrm{U}) \mathrm{d} \tau \quad \text { (continuity: } \nabla \cdot \overrightarrow{\mathrm{v}}=0\right)
$$

$$
=-\oint_{A} \rho U(\vec{v} \cdot \vec{n}) d A \quad \text { (Gauss theorm) }
$$

and the third term,

$$
-\oint_{A} \vec{q} \cdot \vec{n} d A=0
$$

In all, equation [3.1] based on above assumptions reduces to be:

$$
\oint_{A} \vec{n} \cdot \rho \vec{v}\left(e+\frac{1}{2} \stackrel{\rightharpoonup}{v} \cdot \vec{v}\right) d A=-\oint_{A} \vec{v} \cdot \vec{n} P d A-\oint_{A} \rho U(\vec{v} \cdot \vec{n}) d A
$$

that is:

$$
\begin{equation*}
\oint_{A} \rho \vec{v} \cdot \vec{n}\left[e+\frac{1}{2} \vec{v} \cdot \vec{v}+\frac{P}{\rho}+U\right] d A=0 \tag{3.2}
\end{equation*}
$$

Equation [3.2] may be thought of as another energy conservation equation based on the assumptions of: steady flow, no heat transfer, invisicid fluid and body force is conservative, where,
e-------internal energy per unit mass, $\stackrel{\rightharpoonup}{\mathrm{V}} \cdot \overrightarrow{\mathrm{v}} / 2---\mathrm{kinetic}$ energy per unit mass, $P / \rho----$ pressure potential energy per unit mass, and U-------body force potential energy related with gravity, hence $U=g Z$ which may be called elevation energy per unit mass.

The meaning of equation [3.2] is that the algebraical sum of total energy which is flowing in or out the control surface A is zero during unit time. That is to say, accompanied with fluid
flowing in or out of any differential area $d A$, the net gain of energy flow in all the control surface $A$ is zero during unit time. Where $e+1 / 2 \vec{v} \cdot \vec{v}+P / \rho+g Z$ is considered as total energy per unit mass.

For the following special case, if the control volume can be thought of as a very small stream tube where there is no flow cross the tube surface, except for the section areas 1-1 and 2-2 in Figure 3-1, and also any parameters on these section areas are treated as being uniform, we can obtain from equation [3.2]:

$$
\left(\int_{\mathrm{A} 1}+\int_{\mathrm{A} 2}\right)\left[\rho \overrightarrow{\mathrm{v}} \cdot \overrightarrow{\mathrm{n}}\left(\mathrm{e}+\frac{1}{2} \overrightarrow{\mathrm{v}} \cdot \overrightarrow{\mathrm{v}}+\frac{\mathrm{P}}{\rho}+g z\right) \mathrm{dA}\right]=0
$$

That is:

$$
\rho \vec{v}_{i} \vec{n}_{1}\left(e_{1}+\frac{1}{2} \vec{v}_{i} \vec{v}_{1}+\frac{P_{1}}{\rho}+g Z_{1}\right) A_{1}+\rho \vec{v}_{2} \vec{n}_{2}\left(e_{2}+\frac{1}{2} \vec{v}_{\dot{2}} \vec{v}_{2}+\frac{P_{2}}{\rho}+g Z_{2}\right) A_{2}=0
$$

In $A_{1}$ section area, $\vec{v}_{1} \cdot \vec{n}_{1}=-V_{1} \quad\left(\vec{v}_{1}, \vec{n}_{1}\right.$ different directions) and in $A_{2}$ section area, $\vec{v}_{2} \cdot \vec{n}_{2}=V_{2} \quad\left(\vec{v}_{2}, \vec{n}_{2}\right.$ same direction ). Where $V_{1}=\left|\vec{v}_{1}\right|, V_{2}=\left|\vec{v}_{2}\right|$. That is, $V_{1}$ and $V_{2}$ are the magnitude of the vector $\overrightarrow{\mathrm{v}}_{1}$ and $\overrightarrow{\mathrm{v}}_{2}$ respectively. We know from the continuity condition $A_{1} \cdot V_{1}=A_{2} \cdot V_{2}$, therefore, we get the equation commonly called the Bernoulli equation:

$$
e_{1}+\frac{1}{2} v_{1}^{2}+\frac{P_{1}}{\rho}+g Z_{1}=e_{2}+\frac{1}{2} v_{2}^{2}+\frac{P_{2}}{\rho}+g z_{2}
$$

This equation can be satisfied at two distinct sectional
areas along the stream tube and of course be satisfied at two distinct points along stream line.

We can further assume: internal energy keeps constant because of no heat transfer and friction involved, and there is no (or negligible) elevation change, i.e, $e_{1}=e_{1}$, and $Z_{1}=Z_{2}$.

Hence we obtain the following result which is important to introduce the orifice flow equation and the dimensional crack flow equation:

$$
\begin{equation*}
\frac{1}{2} v_{1}^{2}+\frac{P_{1}}{\rho}=\frac{1}{2} v_{2}^{2}+\frac{P_{2}}{\rho} \tag{3.3}
\end{equation*}
$$

In summary, to obtain the above equation [3.3], several important assumptions had to be applied to the basic energy conservation equation [3.1]:
(1) steady flow, $\partial / \partial t=0$
(2) incompressible flow, $\rho=$ constant inside the stream tube
(3) frictionless flow, $e=$ constant
(4) no elevation change, $Z=$ constant
(5) no heat transfer involved, $|\stackrel{\rightharpoonup}{q}|=0$
(6) inviscid fluid, $\overrightarrow{\mathrm{P}}_{\mathrm{n}}=-\overrightarrow{\mathrm{P}} \cdot \mathrm{n}$, sometimes however, it is used to approximate viscid fluid flow, and
(7) uniform parameters at any two distinct sectional areas along small stream tube.

### 3.1.2. Application of Bernoulli equation to orifice flow



Figure 3-2. Orifice flow

As shown in the figure, the mainstream flow continues to accelerate from the orifice throat to form a vena contracta and then decelerates again to fill the duct. We set section (1) at a uniform flow inlet part. In the vena contracta section, the flow area is minimum, streamlines are essentially straight, and the pressure is uniform across the channel section. Hence section (2) is set at the vena contracta. Then the Bernoulli equation is applied in form of equation (3),

$$
\frac{\mathrm{v}_{1}^{2}}{2}+\frac{\mathrm{P}_{1}}{\rho}=\frac{\mathrm{v}_{2}^{2}}{2}+\frac{\mathrm{P}_{2}}{\rho}
$$

Using the continuity condition $\mathrm{V}_{1} \cdot \mathrm{~A}_{1}=\mathrm{V}_{2} \cdot \mathrm{~A}_{2}$,

$$
\begin{gathered}
V_{2}=\sqrt{\frac{2\left(P_{1}-P_{2}\right)}{\rho\left[1-\left(\frac{A_{2}}{A_{1}}\right)^{2}\right]}} \\
Q=A_{2} V_{2}=A_{2} \sqrt{\frac{2\left(P_{1}-P_{2}\right)}{\rho\left[1-\left(\frac{A_{2}}{A_{1}}\right)^{2}\right]}}
\end{gathered}
$$

There are the following four points which should be mentioned:

The section (2) area $A_{2}$ at the vena contracta is hard to measure. The location of the vena contracta section and the section area $A_{2}$ vary with flow conditions which might change with different test requirements. In practice, we may use orifice area $A_{0}$ instead of $A_{2}$.

Secondly, it is difficult to place pressure tap(s) exactly at the location of the vena contracta to measure $P_{2}$. The location of the pressure taps influences the empirically determined discharge coefficient $C_{d}$. For practical orifice measurement, location of the pressure taps consistent with $C_{d}$ may be selected (Doebelin 1966, pp.466, Fox and McDonald 1978, pp.453).

Thirdly, we should note that the Bernoulli equation is
derived from inviscid flow case. In order to expand it to viscid fluid, the internal friction which might cause actual velocity less than the ideal velocity should be included.

Furthermore, we have to use some coefficient (s) to correct for the differences caused by the above factors. Therefore contributed by these three factors, the discharge coefficient, $\mathrm{C}_{\mathrm{d}}$, may be introduced. The following equation may also be satisfied by $C_{d}$ adjustment, hence

$$
\begin{equation*}
Q=C_{d} A_{0} \sqrt{\frac{2\left(P_{1}-P_{2}\right)}{\rho\left[1-\left(\frac{A_{0}}{A_{1}}\right)^{2}\right]}} \tag{3.4a}
\end{equation*}
$$

If and only it $A_{0} \ll A_{1}$, i.e, orifice area is much smaller than the duct section area, we get a simpler and common expression:

$$
\begin{equation*}
Q=C_{d} A_{0} \sqrt{\frac{2 \Delta P}{\rho}} \tag{3.4b}
\end{equation*}
$$

Equation [3.4b] is often called the orifice equation. It is obvious that equation [3.4b] is an approximation from equation [3.4a] by neglecting $A_{0} / A_{1}$. Hence the $C_{d}$ value in equation [3.4b] is also affected by the value of $A_{0} / A_{1}$ which is neglected.

In conclusion, on the basis of the discussion above, the discharge coefficient $C_{d}$ based on equation [3.4b] is influenced by:
a) $A_{2} / A_{0}$. In which $A_{2}$ varies with flow condition, therefore it is hard to measure
b) location of pressure taps
c) viscid friction
d) $A_{0} / A_{1}$, denoted by $\beta$, which can be determined by the structure itself and does not vary with flow condition.

We may write $C_{d}$ as the following function:
$C_{d}=f\left(A_{2} / A_{0}\right.$, tap location, viscid friction, $\beta$ )

If and only if the tap locations have been determined, it can be written as:

$$
C_{d}=f\left(A_{2} / A_{0}, \text { viscid friction, } \beta\right),
$$

It is known that the Reynold number, $R e$, can be written as: $\operatorname{Re}=\left(\overline{\mathrm{V}} \cdot D_{h}\right) / v=\left(Q \cdot D_{h}\right) /(A \cdot v)$. It involves the flow condition, the viscid friction and the geometry terms. As noted in point a) above, $A_{2} / A_{0}$ varies with flow condition. Hence a more general expression for $C_{d}$ is, $C_{d}=f(R e, \beta)$, if and only if the location of pressure taps has been determined (Fox and McDonald 1978, Doebelin 1966).
$D_{h}$ is the hydraulic diameter and $\nu$ is viscosity.

### 3.1.3 Application of orifice equation to crack flow

The equation [3-4b], called the orifice equation, is widely applied to crack flow studies and is considered as a fundamental equation, although the power equation is more commonly used (ASHRAE Handbook Fundamentals 1989, pp.23.6 \& 23.11). But it is a way for us to treat the crack flow problem more analytically than empirically.

Crack flow is much more complicated than orifice flow either in geometry or in flow model. There are several things which cause the difference. One is that we assume the flow length in orifice flow is zero while in real crack flow it typically is not. Secondly in orifice flow, we neglect the viscid friction and assume $\mathrm{P} / \rho+1 / 2 \mathrm{~V} \cdot \mathrm{~V}$ is constant along the stream tube, and the friction effect is simply contributed to the $C_{d}$ term; while in crack flow, the $P / \rho+3 / 2 \mathrm{~V} \cdot \mathrm{~V}$ values across the crack might be significantly different and there may be better ways to describe them. Thirdly, the relation that $Q$ is proportional to $\sqrt{\overline{\Delta P}}$ may be suitable for orifice flow only, but it is really a restriction for crack flow because it just covers a square root relationship of $\mathrm{Q}-\Delta \mathrm{P}$, it is lack of generality of application.

### 3.1.4 Application of Bernoulli equation to crack flow

For orifice flow we neglect the effect of friction in the derivation and treat the flow length as zero. But for crack flow we are concerned with the geometry of the crack channel which causes an amount of pressure loss due to friction. It is commonly called "Head loss".


Figure 3-3. Crack flow

To simplify the analysis, the total head loss denoted by $h_{t}$ may be divided into major loss, denoted by $h_{m}$, and minor loss, denoted by $h_{n}$. See Figure 3-4 (Fox and McDonald 1978, pp.369), note that the entrance length for the developing section is estimated by the formula of, $L \approx 0.06 \mathrm{Re} \cdot \mathrm{D}_{\mathrm{h}}$,



Figure 3-4. Variation of static pressure in a pipe inlet section (From Fox and McDonald 1973)

The major loss $h_{m}$ is due to friction in fully-developed laminar flow in constant area portions of the system. The minor loss $h_{n}$ is due to the difference between fully-developed laminar
and developing laminar flow, bends, entrance and exit losses, and any other nonconstant area frictional effects. Refer to Figure $3-3$, we have:

$$
\begin{equation*}
\left(\frac{v_{1}^{2}}{2}+\frac{P_{1}}{\rho}\right)-\left(\frac{v_{2}^{2}}{2}+\frac{P_{2}}{\rho}\right)=h_{T}=h_{m}+h_{n} \tag{3.5}
\end{equation*}
$$

It is obvious that if we assume no flow length (which means no developing and developed regimes, and no pressure drops on the length), no inlet-outlet and no bends, the total head loss $h_{t}$ will be zero. Then equation [3.5] reduces to the Bernoulli equation.

Let us first find the major loss expression due to friction in a constant sectional area crack with fully-developed laminar flow, $A_{1}=A_{2}$, hence $V_{1}=V_{2}$ from continuity, also with $h_{n}=0$. Therefore, $\left(P_{1}-P_{2}\right) / \rho=h_{\mathrm{t}}=h_{\mathrm{m}}$ for fully-developed flow in a constant area crack.

Based on this condition, we can easily obtain the $h_{m}$ expression for fully-developed laminar flow for regular geometry channels such as infinite parallel plate and circular opening flows with the derivation from basic fluid mechanics theory. The following is the derivation for the fully-developed laminar flow for infinite parallel plate with a limited width as shown in Figure 3-5 as the first example, this result can also be applied to the rectangular channel with small aspect ratios:


Figure 3-5. Laminar flow between infinite parallel plates

The Navier-Stokes equation for laminar viscid flow is (Currie 1974, pp.220-225):

$$
\frac{\partial \stackrel{\rightharpoonup}{v}}{\partial t}+(\stackrel{\rightharpoonup}{\mathrm{V}} \cdot \nabla) \stackrel{\rightharpoonup}{\mathrm{v}}=-\frac{1}{\rho} \nabla \mathrm{p}+v \nabla^{2} \overrightarrow{\mathrm{v}}+\stackrel{\rightharpoonup}{\mathrm{f}}
$$

where
$\overrightarrow{\mathrm{v}}$---velocity vector
p---pressure
$\overrightarrow{\mathrm{f}}$---body force ( $=-\overrightarrow{\mathrm{g}}$, gravity vector)
t---time
p--air density
$\mu--$ dynamic viscosity.

Using the following assumptions:

1) steady flow $\partial / \partial t=0$
2) one dimension flow $\vec{V}=u \vec{k}$,ie, $u=u(y), v=0, w=0$
3) since no flow in $x, y$ directions, $P=P(z)$
4) body force (gravity) is negligible , $|\overrightarrow{\mathrm{f}}|=0$
5) boundary condition, $u(0)=0, u(D)=0$
6) pressure is linear distribution in flow direction, $d P / d z=-\Delta P / Z$

Hence

$$
\begin{aligned}
& (\vec{v} \cdot \nabla) \vec{v}=\left(u \vec{k} \cdot \frac{\partial}{\partial z} \vec{k}\right) u \vec{k}=0 \quad(u=u(y)) \\
& \nabla P=\frac{\partial P}{\partial x} \vec{i}+\frac{\partial p}{\partial y} \vec{j}+\frac{\partial p}{\partial z} \vec{k}=\frac{d P}{d z} \vec{k} \quad(P=P(z)) \\
& \nabla^{2} \vec{v}=\frac{\partial^{2} u}{\partial x^{2}}+\frac{\partial^{2} u}{\partial y^{2}}+\frac{\partial^{2} u}{\partial z^{2}} \quad(\vec{v}=u \vec{k}) \\
& =\frac{d^{2} u}{d y^{2}} \quad(u=u(y)) \\
& \frac{\partial \vec{v}}{\partial t}=0 \quad\left(\text { Steady flow: } \frac{\partial}{\partial t}=0\right)
\end{aligned}
$$

Therefore the Navier-Stokes equation reduces to be:

$$
0=\frac{1}{\rho}\left(-\frac{d P}{d z}\right)+\mu\left(\frac{d^{2} u}{d y^{2}}\right)
$$

By integrating the velocity variable $u$ with respect to $y$ twice, then a fully-developed laminar velocity profile is given:

$$
u(y)=\frac{1}{\mu} \frac{d p}{d z}\left(\frac{y^{2}}{2}+A y+B\right)
$$

Applying the boundary conditions to determine integration constants A and B

$$
\begin{cases}u(0)=0, & B=0 \\ u(y)=0, & A=-\left(\frac{D}{2 \mu}\right) \frac{d P}{d z}\end{cases}
$$

Then,

$$
u(y)=\frac{1}{2 \mu} \frac{d p}{d z} y(y-D)
$$

Assuming that the width of the infinite parallel is $w$, then the volumetric flow rate Q for the rectangular channel with a small ratio is:

$$
Q=\int_{0}^{w} \int_{0}^{d} u(y) d y d x=-\frac{b^{3} w}{12 \mu} \frac{d P}{d z}=\frac{D^{3} w}{12 \mu} \frac{\Delta P}{Z}
$$

Average velocity $\overline{\mathrm{V}}$ :

$$
\overline{\mathrm{V}}=\frac{\mathrm{Q}}{\mathrm{~A}}=\frac{\mathrm{Q}}{\mathrm{wD}}=\frac{\mathrm{D}^{2}}{12 \mu} \frac{\Delta \mathrm{P}}{\mathrm{Z}}
$$

$$
\Delta P=\frac{12 \mu Z \bar{V}}{D^{2}}
$$

Applying hydraulic diameter $\mathrm{D}_{\mathrm{h}}=2 \mathrm{D}$ for parallel plates:

$$
\Delta P=\frac{48 \mu Z \bar{V}}{D_{h}^{2}}
$$

Using major loss expression $h_{m}=\Delta \mathrm{P} / \rho$

$$
\mathrm{h}_{\mathrm{m}}=\frac{48 \mathrm{v} \overline{\mathrm{~V}}}{D_{\mathrm{h}}^{2}}=\frac{96 \mathrm{Z}}{\left(\frac{\overline{\mathrm{v}} D_{\mathrm{h}}}{v}\right) D_{\mathrm{h}}} \frac{\overline{\mathrm{v}}^{2}}{2}=\frac{96 \mathrm{Z}}{\operatorname{Re} D_{\mathrm{h}}} \frac{\overline{\mathrm{v}}^{2}}{2}
$$

Using a similar method for fully-developed laminar flow in a circular pipe, we obtain:

$$
\mathrm{h}_{\mathrm{m}}=\frac{64 \mathrm{Z}}{\operatorname{Re} \mathrm{D}_{\mathrm{h}}} \frac{\overline{\mathrm{v}}^{2}}{2}
$$

where $D_{h}=D$ for the circular case.

In general, we find the difference existing in the major loss expression for rectangular or circular openings is the coefficient only. We may define $B$ as friction coefficient and $f$ as friction factor for the fully-developed laminar flow of any type of crack as:

$$
h_{m}=\frac{B}{\operatorname{Re}} \frac{Z}{D_{h}} \frac{\bar{v}^{2}}{2}=f \frac{Z}{D_{h}} \frac{\bar{v}^{2}}{2}
$$

This is the form for the major loss for fully-developed laminar flow; where $f=B / R e, B=64$ for circular pipe, $B=96$ for rectangular channel and friction factor $f$ is linear with the $\operatorname{Re}$ number.

We note that sometimes when the pressure drop across an orifice, pipe or even building components is higher, the Re number may be in the turbulent regime in which case the friction factor, $f$, will not be linearly related to the Re number. There are several empirical correlations for different cases (Fox and McDonald 1978, pp.467):

For turbulent flow in smooth pipes ( $\mathrm{Re} \leq 100,000$ ), the Blasius correlation is:

$$
f=\frac{0.316}{R e^{0.25}}
$$

For turbulent flow in the fully-rough flow regime, the Von Karman correlation is (e/D is the relative roughness, see details in Fox and McDonald 1978 pp. 467):

$$
f=\frac{1}{4\left[0.57-\log _{10}\left(\frac{e}{D}\right)\right]^{2}}
$$

Actually for most medium-roughness pipes, the correlation curves of the friction factor may be found numerically (see Figure 8.12 in Fox and McDonald 1978).

In summary, the general form for the major loss is:

$$
h_{m}=f \frac{Z}{D_{h}} \frac{\bar{V}^{2}}{2}
$$

Where for fully-developed laminar cases: $f=64 / \operatorname{Re}$ and $f=96 / \operatorname{Re}$
for circular and rectangular pipes respectively. For flow in the turbulent regime, $f$ depends on the relative roughness (using Blasius or Von Karman correlations or others) and the $\operatorname{Re}$ number.

We then consider minor loss which is due to inlet-outlet loss, hydrodynamic development loss, section area change, the roughness of internal surface and bends. Because it has such a large variation and is affected by so many factors, the minor loss, in practice, is impossible to be theoretically estimated. However, we might express minor head loss as:

$$
h_{n}=K \frac{\bar{v}^{2}}{2}
$$

$K$ is called the minor loss coefficient, which must be determined experimentally for each case or each similar group. Some suggested $K$ values can be found in the literature for pipe entrances, exits, enlargements and contractions, gradual contraction and bends (Fox and McDonald 1978, pp.368-374).

In conclusion

$$
\frac{\Delta P}{\rho}=h_{T}=h_{m}+h_{n}=\frac{B}{R e} \frac{Z}{D_{h}} \frac{\bar{V}^{2}}{2}+K \frac{\bar{V}^{2}}{2}
$$

An alternative form with dimensionless pressure drop expression of the above equation can be written as:

$$
\begin{equation*}
\frac{\Delta \mathrm{P}}{\frac{1}{2} \rho \overline{\mathrm{~V}}^{2}}=\frac{\mathrm{B}}{\operatorname{Re}} \frac{\mathrm{Z}}{\mathrm{D}_{\mathrm{h}}}+\mathrm{K} \tag{3.6}
\end{equation*}
$$

where

| $\Delta \mathrm{P}$ | the dimensionless pressure drop |
| :---: | :---: |
| $\frac{1}{1 / 2 p \bar{V}^{2}}$ $B Z$ | the dimensionless major loss coefficient due to |
| $\mathrm{ReD}_{\mathrm{h}}$ | frictional effects in fully-developed flow in constant area opening. |
| K | the dimensionless minor loss coefficient due |
|  | to the difference between fully-developed and developing flow, inlet and outlet losses, area changes and bends. |

The equation [3.6] is called the dimensionless crack flow equation. The derivation of the equation in this way may be more stringent than other approaches used, such as the power model or the orifice model. A number of researchers have applied this dimensionless crack flow equation (Hopkins and Hansford 1974, Etheridge 1977, Kronvall 1980 and Chastain et al.1987).

## APPENDIX F

Definition and Combination of Crack/Position Coding

## Definition and Combination of Crack/Position Coding

1) Individual Crack

| Crack | A | B1 | B2 | E | F1 | F2 |
| :--- | :--- | :--- | :--- | :--- | :--- | :--- |
| Name (Outer Layer) | A-1 | B1-1 | B2-1 | E-1 | F1-1 | F2-1 |
| Name (Inner Layer) | A-2 | B1-2 | B2-2 | E-2 | F1-2 | F2-2 |

Note: $\quad$ Refer to Figure 4.2.2 for placement location and Table 4.1 for dimensions of the individual cracks.
2) Pāälié Cūmectioũin

| Parallel <br> Type | A and B1 | A and E | A and F1 | B1 and B2 | B1 and F1 | B1 and E | F1 and F2 |
| :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- |
| Name | A@B1 | A@E | A@F1 | B1@B2 | B1@F1 | B1@E | F1@F2 |

## 3) Series Connection

| Inner | A | B1 | B1 | E | E | F1 | F1 |
| :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- |
| Outer | B1 | A | E1 | B1 | F1 | B1 | E |
| Name | A $\sim$ B1 | B1 $\sim$ A | B1 $\sim$ E1 | E $\sim$ B1 | E~F1 | F1 $\sim$ B1 | F1 $\sim$ E |

## APPENDIX G

## Data Tables for Individual Openings



Crack Leakage Calculation By Individual Crack: A-2



Crack Leakage Calculation By Individual Crack: B1-2

| Pressume Dillerence Across Crack |  | Flow Rate |  |  |  |  |  | Power Model Preoicteda <br> ( $\mathrm{m}^{2} 3 / \mathrm{sec}$ ) |  |  | Now Model <br> Predicted a <br> ( $m^{\wedge} 3 / s e c$ ) |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| $(\mathrm{I} \sim \mathrm{Wg})$ | (Pa) | $\begin{aligned} & \text { Rep. } 1 \\ & \text { (cims) } \end{aligned}$ | Repa <br> (chri) | $\begin{array}{r} \text { Repa3 } \\ \text { (ckmet } \end{array}$ | Systom Leak <br> (din) | Correcte <br> (clant) | odMean ( $\mathrm{m}^{2} 3 /$ sec. $)$ |  |  |  |  |  |  |
| 0.02 | 5 | 233 | 233 | 246 | 0.40 | 1.98 | 0.00038 | 0.00096 |  |  | 000088 |  |  |
| 0.04 0.06 | 10 15 | 3.86 5.13 | 3.18 5.18 | 3.97 5.23 | 0.56 0.68 | 3.11 4.50 | $\begin{aligned} & 0.00147 \\ & 0.00212 \end{aligned}$ | 0.00154 0.00203 |  |  | $000207$ |  |  |
| 0.08 | 20 | 6.14 | 6.19 | 6.19 | 0.78 | 539 | 0.00254 | $0.00248$ |  |  | 0.00254 |  |  |
| 0.10 | 25 | 7.04 | 7.13 | 7.15 | 0.87 | 6.24 | 0.00294 | 0.00889 |  |  | 0.00296 |  |  |
| 0.12 | 30 | 7.99 | 803 | 8.08 | 0.95 | 7.08 | 0.00334 | $\begin{aligned} & 0.00327 \\ & 0.00363 \end{aligned}$ |  |  | 0.00370 |  |  |
| 0.14 | 35 | 8.83 | 883 | 892 | 1.03 | 7.83 | 0.00370 |  |  |  |  |  |  |
| 0.16 | 40 | 9.56 | 9.67 | 9.62 | 1.10 | 852 | 0.00402 | $\begin{aligned} & 0.00363 \\ & 0.00398 \end{aligned}$ |  |  | $0.00404$ |  |  |
| 0.18 | 45 | 10.28 | 1040 | 10.40 | 1.16 | 920 | 0.00434 | $0.00432$ |  |  | $0.00436$ |  |  |
| 0.20 | 50 | 10.86 | 11.10 | 11.10 | 122 | 9.79 | 0.00462 | 0.00464 |  |  | 0.00466 |  |  |
| 0.22 | 55 | 11.57 | 11.65 | 11.65 | 128 | 1034 | 0.00488 | 0.00495 |  |  | 0.00494 |  |  |
| 0.24 | $\boldsymbol{\infty}$ | 12.35 | 1243 | 1247 | 1.34 | 11.08 | 0.00523 | 0.00526 |  |  | 0.00522 |  |  |
| 0.26 | 65 | 12.97 | 12.86 | 1297 | 1.39 | 11.54 | 0.00545 | 0.00555 |  |  | 0.00549 |  |  |
| 0.28 | 70 | - 13.54 | 1380 | 13.73 | 144 | 1224 | 0.00578 | $\begin{aligned} & 0.00584 \\ & 0.00613 \end{aligned}$ |  |  | $\begin{aligned} & 0.00574 \\ & 0.00599 \end{aligned}$ |  |  |
| 030 | 75 | 14.19 | 14.22 | 14.26 | 149 | 1273 | 0.00601 |  |  |  |  |  |  |
| 0.32 | 80 | 14.88 | 14.75 | 14.96 | 1.54 | 1332 | 0.00629 |  |  |  |  |  |  |
| 0.34 | 85 | 15.44 | 15.41 | 15.52 | 1.59 | 13.87 | 0.00655 |  |  |  |  |  |  |
| 0.36 | 90 | 15.91. | 15.97 | 15.93 | 1.63 | 14.30 | 0.00675 | C= 3.18E-04 |  |  | Cf= 8.50E-04 |  |  |
| 0.38 | 95 | 16.54 | 16.58 | 16.58 | 1.68 | 14.89 | 0.00703 |  |  |  |  |  |  |
| 0.40 | 100 | 17.05 | 17.08 | 17.12 | 1.72 | 15.36 | 0.00725 | $n=0.6852$ |  |  | C2. | 1.96 |  |
| 0.42 | 105 | 17.63 | 17.72 | 17.70 | 1.76 | 15.92 | 0.00751 |  |  |  |  |  |  |
| 0.44 | 110 | 18.05 | 18.12 | 18.09 | 180 | 16.29 | 0.00769 |  |  |  | C3- 1.03 |  |  |
| 0.46 | 115 | 18.55 | 18.75 | 18.63 | 184 | 16.80 | 0.00793 |  |  |  |  |  |  |  |
| 0.48 0.5 | 120 | 19.18 19.54 | 19.22 | 19.25 19.68 | 1.88 | $17.34$ | 0.00818 0.00036 | C.V. $=$ | 1.9047 | (\%) | C.V. $=$ | 1.0089 | (\%) |
| 0.5 | 125 | 19.54 | 19.68 | 19.68 | 1.92 | 17.71 | 0.0006 |  |  |  |  |  |  |

Crack Leakage Calculation By Individual Crack: B2-1

| Pressure Ditierence Across Crack |  | Flow Rate |  |  |  |  |  | Power Model Predicted $Q$ <br> ( $\left.\mathrm{m}^{n} 3 / \mathrm{sec}\right)$ |  |  | Now Model <br> Predicled Q <br> ( $\mathrm{m}^{\wedge} 3 / \mathrm{sec}$ ) |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | Repoi (clmi) | Rop. 2 <br> (cirm | $\begin{gathered} \hline \text { Repas } \\ \text { (cimet } \end{gathered}$ | Syctom Leak (dme) | $\begin{aligned} & \text { Corroctio } \\ & \text { (dind) } \end{aligned}$ | odmean <br> (in'3/sec) |  |  |  |  |  |  |
| 0.02 | 5 | 242 | 236 | 216 | 0.40 | 1.92 | 0.00090 |  | 0.00093 |  |  | 0.00083 |  |
| 0.04 | 10 | 380 | 3.80 | 3.63 | 0.56 | 3.19 | 000150 |  | 0.00150 |  |  | 0.00146 |  |
| 0.08 | 15 | 4.99 | 4.99 | 4.74 | 0.68 | 4.23 | 0.00200 |  | 0.00198 |  |  | 0.00199 |  |
| 0.08 | 20 | 6.13 | 699 | 5.73 | 0.78 | 5.17 | 0.00344 |  | 000242 |  |  | 0.00245 |  |
| 0.10 | 25 | 7.05 | 7.05 | 680 | 0.87 | 6.10 | 000288 |  | 0.00282 |  |  | 0.00287 |  |
| 0.12 | 30 | 789 | 789 | 7.58 | 0.95 | 6.83 | 0.00322 |  | 0.00320 |  |  | 0.00325 |  |
| 0.14 | 35 | 8.79 | 8.62 | 8.43 | 1.03 | 7.59 | 0.00358 |  | 000355 |  |  | 0.00361 |  |
| 0.16 | 40 | 2.89 | 846 | 206 | 1.10 | 827 | 0.00390 |  | 0.00390 |  |  | 000394 |  |
| 0.18 | 45 | 10.28 | 1020 | 0.87 | 1.16 | 895 | 0.00423 |  | 000423 |  |  | 0.00426 |  |
| 0.20 | 50 | 11.05 | 11.01 | 10.56 | 122 | 2.65 | 0.00455 |  | 000454 |  |  | 000456 |  |
| 022 | 55 | 11.81 | 11.68 | 1126 | 128 | 10.23 | 0.00483 |  | $000485$ |  |  | $0.00484$ |  |
| 024 | 60 | 12.35 | 1230 | 1201 | 1.34 | 1088 | 0.00514 |  | 000515 |  |  | $000512$ |  |
| 0.26 | 65 | 13.11 | 1299 | 1248 | 1.39 | 11.47 | 0.00541 |  | 000545 |  |  | 0.00538 |  |
| 028 | 70 | 13.71 | 1365 | 13.06 | 1.44 | 12.03 | 0.00568 |  | 0.00573 |  |  | 0.00564 |  |
| 0.30 | 75 | 14.29 | 14.21 | 13.64 | 149 | 12.55 | 0.00592 |  | 0.00601 |  |  | 0.00589 |  |
| unic | 0 | 1454 | 44, | 4.96 | 1.94 | 1318 | 0100618 |  |  |  |  |  |  |
| 0.34 | 85 | 15.51 | 15.47 | 14.88 | 1.59 | 13.70 | 0.00647 |  |  |  |  |  |  |
| 0.36 | 90 | 16.06 | 16.19 | 15.50 | 1.63 | 14.28 | 0.00674 | C= | 306E-04 |  | C1- | 8.50E-04 |  |
| 038 | 95 | 16.68 | 16.60 | 15.95 | 1.68 | 14.73 | 0.00695 |  |  |  |  |  |  |
| 0.40 | 100 | 17.15 | 17.17 | 16.52 | 1.72 | 1523 | 0.00719 |  |  |  | C2. | 217 |  |
| 0.42 | 105 | 17.79 | 17.68 | 17.07 | 1.76 | 15.75 | 0.00743 | $\mathrm{n}=$ | 0.6897 |  |  |  |  |
| 0.44 | 110 | 18.35 | 18.32 | 17.64 | 1.80 | 1630 | 0.00769 |  |  |  | $\mathbf{C m}$ | 1.04 |  |
| 0.46 | 115 | 18.81 | 1891 | 18.15 | 1.84 |  | 0.00792 |  |  |  |  |  |  |
| 0.48 | 120 | 19.30 | 19.41 | 18.59 | 1.88 | 1722 | 0.00813 | C.V. $=$ | 1.0808 | (\%) | C.V $=$ | 0.9215 | (\%) |
| 0.5 | 125 | 20.03 | 19.90 | 19.33 | 1.92 | 17.84 | 0.00842 |  |  |  |  |  |  |

Crack Leakage Calculation By Individual Crack : B2-2

| Pressure Difference Across Crack |  | Flow Rate |  |  |  |  |  | Power Model Predicted 0 <br> ( $\mathrm{m}^{\prime} 3 / \mathrm{sec}$ ) |  |  | Now Model Predicted Q <br> ( $\mathrm{m}^{+} 3 / \mathrm{sec}$.) |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| ( $\mathrm{ln} . \mathrm{Wg}$ ) |  | Rop. <br> (cime) | Rep 2 <br> (ctri) | $\begin{aligned} & \text { Rop.3 } \\ & \text { (ctrm) } \end{aligned}$ | Systern Leak (dimp) | CorroctrodMean (chrif) ( $\mathrm{m}^{2} 3 / \mathrm{sec}$ ) |  |  |  |  |  |  |  |
| 0.02 | 5 | 216 | 216 | 2.16 | 0.40 | 1.76 | 0.00083 |  | 0.00087 |  |  | 0.00081 |  |
| 0.04 0.06 | 10 15 | 3.63 4.69 | 3.49 4.69 | 3.43 4.74 | 0.56 0.68 | 296 4.03 | $\begin{aligned} & 0.00140 \\ & 0.00190 \end{aligned}$ |  | 0.00141 0.00188 |  |  | 0.00141 0.00192 |  |
| 0.08 | 20 | 5.77 | 6.77 | 5.78 | 0.78 | 4.99 | 0.00236 |  | 0.00230 |  |  | 0.00236 |  |
| 0.10 | 25 | 6.71 | 6.75 | 6.75 | 0.87 | 5.87 | 0.00277 |  | 0.00269 |  |  | 0.00276 |  |
| 0.12 0.14 | 30 35 | 7.59 8.43 | 7.51 8.38 | 747 8.34 | 0.95 1.03 | 6.57 7.36 | 0.00310 0.00347 0.00379 |  | 0.00305 0.00340 |  |  | 0.00313 0.00347 |  |
| 0.16 | 40 | 9.10 | 2.15 | 9.15 | 1.10 | 204 | 0.00379 |  | 0.00374 |  |  | 0.00379 |  |
| 0.18 | 45 | 9.78 | 9.87 | 9.81 | 1.16 | 866 | 0.00409 |  | 0.00406 |  |  | 0.00409 |  |
| 0.20 | 50 | 10.46 | 10.62 | 10.63 | 122 | 935 | 0.00441 |  | 000437 |  |  | 0.00438 |  |
| 0.22 | 55 | 11.04 | 11.23 | 11.24 | 128 | 9.89 | 0.00467 |  | 0.00467 |  |  | 0.00466 |  |
| 024 | 60 | 11.35 | 11.75 | 11.83 | 1.34 | 1030 | 0.00486 |  | 0.00497 |  |  | 0.00492 |  |
| 0.26 | 65 | 1215 | 1237 | 1241 | 1.39 | 10.92 | 0.00515 |  | 0.00526 |  |  | 0.00517 |  |
| 0.28 | 70 | 1292 | 12.88 | 1291 | 144 | 11.47 | 0.00541 |  | 0.00554 |  |  | 0.00542 |  |
| 0.30 | 75 | 13.49 | 13.53 | 13.49 | 149 | 1201 | 0.00567 |  | 0.00581 |  |  | 0.00565 |  |
| 0.32 | 80 | 14.06 | 14.13 | 14.13 | 1.54 | 12.56 | 0.00593 |  |  |  |  |  |  |
| 0.34 | 85 | 14.66 | 14.73 | 14.73 | 1.59 | 13.12 | 0.00619 |  |  |  |  |  |  |
| 0.36 | 90 | 15.15 | 15.33 | 15.33 | 1.63 | 13.64 | 0.00644 | C- | 280E-04 |  | C1 $=$ | 8.50E-04 |  |
| 038 | 95 | 15.73 | 15.70 | 15.70 | 1.68 | 14.03 | 0.00662 |  |  |  |  |  |  |
| 0.40 | 100 | 16.25 | 16.36 | 16.38 | 1.72 | 14.61 | 0.0069 |  |  |  | $\cdots=$ | 203 |  |
| 0.42 | 105 | 16.75 | 16.69 | 16.72 | 1.76 | 14.96 | 0.00706 | $\mathrm{n}=$ | 0.7025 |  |  |  |  |
| 0.44 | 110 | 17.30 | 17.23 | 1730 | 180 | 15.47 | 0.00730 |  |  |  | C3F | 0.95 |  |
| 0.46 | 115 | 17.77 | 17.74 | 17.80 | 184 | 15.93 | 0.00752 |  |  |  |  |  |  |
| 0.48 | 120 | 18.30 | 18.29 | 18.33 | 1.88 | 16.43 | 0.00775 | C.Vm | 2226 | (\%) | C.V. $=$ | 0.6390 | (\%) |
| 0.5 | 125 | 18.72 | 18.69 | 18.75 | 1.92 | 16.80 | 0.00793 |  |  |  |  |  |  |



Crack Leakage Calculation By Individual Crack: E-2



Crack Leakage Calculation By Individual Crack: F1-2


| Pressure Difference Across Crack |  | Flow Rate |  |  |  |  |  | Power Model Prodicted 0 <br> ( m r $3 /$ /sec. ) |  | Now Model Predicted a <br> ( $\mathrm{m}^{2} 3 / \mathrm{sec}$.) |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| ( $n . W \mathrm{Wg}$ ) |  | $\begin{array}{r} \text { Rop. } 1 \\ (\text { dfmal } \end{array}$ | $\begin{aligned} & \text { Reop2 } \\ & \text { (dimt) } \end{aligned}$ | $\begin{aligned} & \text { Reop } 3 \\ & \text { (dimet) } \end{aligned}$ | $\begin{aligned} & \text { Systom } \\ & \text { Lakk } \\ & \text { (dims) } \end{aligned}$ | $\begin{aligned} & \text { Correcte } \\ & \text { (citres } \end{aligned}$ | $\begin{aligned} & \text { odmazan } \\ & (n+3 / s e c) \end{aligned}$ |  |  |  |  |  |
| 0.02 | 5 | 30.09 | 3009 | 30.73 | 0.40 | 2950 | 0.01411 | 0.01464 |  |  | 0.01472 |  |
| 0.04 | 10 | 45.59 | 45.58 | 45.58 | 0.56 | 45.03 | 002125 | 0.02113 |  |  | 0.02137 |  |
| 0.06 | 15 | 57.75 | 57.75 | 56.63 | 0.68 | 56.70 | 0.08676 | 0.02619 |  |  | 0.02647 |  |
| 0.08 | 20 | 66.71 | 68.71 | 66.09 | 0.78 | 6592 | 0.03111 | 0.03050 |  |  | 0.03077 |  |
| 0.10 | 25 | 74.43 | 74.43 | 74.12 | 0.87 | 7346 | 0.03467 | 0.03433 |  |  | 0.03456 |  |
| 0.12 | 30 | 81.79 | 81.79 | 8122 | 0.95 | 80.65 | 0.03906 | 0.03781 |  |  | 0.03798 |  |
| 0.14 | 35 | 80.37 | 8837 | 89.34 | 103 | 87.33 | 0.0412 | 0.04103 |  |  | 0.04115 |  |
| 0.16 | 40 | 94.55 | 94.55 | 94.29 | 1.10 | 9337 | 0.04406 | 0.04403 |  |  | 004408 |  |
| 0.18 | 45 | 100.64 | 100.64 | 10038 | 1.16 | 99.39 | 0.04691 | 0.04687 |  |  | 004684 |  |
| 0.20 | 50 | 10599 | 10599 | 108.18 | 122 | 104.83 | 0.04947 | 0.04956 |  |  | 0.04945 |  |
| 0.22 | 55 | 111.13 | 111.13 | 110.89 | 1.28 | 109.76 | 0.05180 | 0.05213 |  |  | 0.05193 |  |
| 0.24 | 60 | 116.48 | 116/8 | 11624 | 134 | 11506 | 005430 | 0.05458 |  |  | 0.05430 |  |
| 0.36 | 65 | 120.65 | 12065 | 121.01 | 139 | 11938 | 0.05634 | 0.05695 |  |  | 005657 |  |
| 028 | 70 | 125497 | 12547 | 125.44 | 144 | 124.02 | 0.05853 | 0.05923 |  |  | 005876 |  |
| 0.30 | 75 | 130.32 | 13032 | 130.11 | 149 | 128.76 | 0.00077 | 0.06143 |  |  | 0.06087 |  |
| 0.32 | 60 | 134.50 | 134.50 | 134.10 | 1.54 | 13282 | 0.06209 |  |  |  |  |  |
| 0.34 | 85 | 13768 | 13768 | 138.18 | 1.59 | 13626 | 0.06431 |  |  |  |  |  |
| 036 | 90 | 14253 | 14253 | 142.50 | 1.63 | 14089 | 006649 | C= 0.00624 |  | C1- | 6.431E-03 |  |
| 0.38 | 95 | 146.25 | 14625 | 146.54 | 1.68 | 144.67 | 0.06828 |  |  |  |  |  |
| 040 | 100 | 15022 | 15022 | 15001 | 1.72 | 148.43 | 0.07005 |  |  | C2- | 022 |  |
| 042 | 105 | 154.42 | 154A2 | 15390 | 1.76 | 15248 | 0.07196 | $n=0.5297$ |  |  |  |  |
| 0.44 | 110 | 157.90 | 157.90 | 157.55 | 1.80 | 155.98 | 007362 |  |  | C. | 125 |  |
| 046 | 115 | -161.03 | 161.63 | 160.98 | 1.84 | 159.57 | 0.07531 |  |  |  |  |  |
| 0.48 | 120 | 164.84 | 164.84 | 164.65 | 188 | 16290 | 007688 | C.V. 1.0847 | (x) | C.V. | 0.5709 | (\%) |
| 0.5 | 125 | 168.15 | 168.15 | 167.96 | 1.92 | 166.16 | 0.07842 |  |  |  |  |  |

Crack Leakage Calculation By Individual Crack: F2-2

| Pressure Ditlerence Across Crack |  | Flow |  |  | Rate |  |  | Power Model Predicted Q <br> ( $\mathrm{m}^{2} 3 / \mathrm{sec}$ ) |  |  | Now Model Predicted Q <br> ( $\mathrm{m}^{2} 3$ /sec) |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| ( n Wg) |  | Rop. 1 (cfris) | $\begin{aligned} & \text { Rept2 } \\ & \text { (cint) } \end{aligned}$ | $\begin{array}{r} \text { Rep.3 } \\ \text { (ctrre) } \end{array}$ |  | Corracte <br> (cirr) | odMean <br> ( $\mathrm{m}^{2} 3 / \mathrm{sec}$ ) |  |  |  |  |  |  |
| 0.02 | 5 | 35.40 | 33.65 | 34.77 | 0.40 | 34.21 | 0.01615 |  | 0.01606 |  |  | 0.01562 |  |
| 0.04 | 10 | 47.62 | 4886 | 47.11 | 0.56 | 4731 | 0.02233 |  | 0.00250 |  |  | 0.02209 |  |
| 0.06 | 15 | 58.98 | 5927 | 58.91 | 0.68 | 58.38 | 0.02755 |  | 0.02741 |  |  | 0.02706 |  |
| 0.08 | 20 | 67.74 | 67.66 | 67.66 | 0.78 | 66.91 | 0.03158 |  | 0.03152 |  |  | 0.03124 |  |
| 0.10 | 25 | 74.44 | 75.23 | 75.52 | 0.87 | 74.19 | 0.03502 |  | 0.03514 |  |  | 0.03493 |  |
| 0.12 | 30 | 8282 | 8247 | 81.65 | 0.95 | 81.36 | 0.03840 |  | 0.03840 |  |  | 0.03926 |  |
| 0.14 | 35 | 88.78 | 88.33 | 88.15 | 1.03 | 87.58 | 0.04134 |  | 0.04139 |  |  | 0.04133 |  |
| 0.16 | 40 | 94.17 | 94.79 | 94.55 | 1.10 | 9341 | 0.04408 |  | $0.04417$ |  |  | $0.04418$ |  |
| 0.18 | 45 | 100.01 | 100.58 | 100.12 | 1.16 | 99.08 | 0.04676 |  | $0.04677$ |  |  | $0.04696$ |  |
| 0.20 | 50 | 105.13 | 106.11 | 10523 | 122 | 10427 | 0.04921 |  | 0.04923 |  |  | 0.04940 |  |
| 0.22 | 55 | 110.05 | 111.19 | 110.14 | 1.28 | 109.18 | 0.05153 |  | 0.05157 |  |  | 0.05181 |  |
| 0.24 | 60 | 115.41 | 11589 | 115.48 | 134 | 114.25 | 0.05392 |  | 0.05380 |  |  | 0.05411 |  |
| 0.26 | 65 | 119.58 | 121.20 | 119.64 | 1.39 | 118.74 | 0.05604 |  | 0.05593 |  |  | 0.05632 |  |
| 0.28 | 70 | 124.20 | 124.83 | 124.25 | 1.44 | 12298 | 0.05804 |  | 0.05799 |  |  | 0.05845 |  |
| 030 | 75 | 128.50 | 128.92 | 129.27 | 149 | 127.41 | 0.06013 |  | 0.05997 |  |  | 0.06050 |  |
| 0.32 | 80 | 13287 | 133.62 | -13272 | 1.54 | 131.53 | 0.06208 |  |  |  |  |  |  |
| 0.34 | 85 | 136.77 | 137.67 | 137.14 | 1.59 | 135.61 | 0.06400 |  |  |  |  |  |  |
| 0.36 | 90 | 140.75 | 141.45 | 141.11 | 1.63 | 13947 | 0.06582 | C = | 0.00734 |  | $\mathrm{Cl}=$ | 6.431E-03 |  |
| 0.38 | 95 | 144.47 | 145.48 | 144.98 | 1.68 | 143.30 | 0.06763 |  |  |  |  |  |  |
| 0.40 | 100 | 147.11 | 149.43 | 148.77. | 1.72 | 146.72 | 0.06924 |  |  |  | $\mathrm{C}=$ | . 0.00 |  |
| 0.42 | 105 | 15201 | 15297 | 15232 | 1.76 | 150.67 | 0.07111 | $\mathrm{n}=$ | 0.4865 |  |  |  |  |
| 0.44 | 110 | 155.20 | 156.45 | 155.81 | 1.80 | 154.02 | 0.07269 |  |  |  | C3- | 1.18 |  |
| 0.46 | 115 | 158.95 | 159.86 | 159.54 | 1.84 | 157.61 | 0.07438 |  |  |  |  |  |  |
| 0.48 | 120 | 16217 | 16336 | 162.90 | $1.88$ | $160.93$ | $0.07595$ | C.V. $=$ | 0.2495 | (\%) | C.V. $=$ | 0.7401 | (\%) |
| 0.5 | 125 | 166.09 | 168.66 | 166.22 | $1.92$ | $164,40$ | $0.07759$ |  |  |  |  |  |  |

## APPENDIX H

Data Tables for Openings in Parallel

Parallel Crack Leakage Calculation: (A@B1)


Parallel Crack Leakage Calculation: (A@E)

| Pressure Diference Across Crack |  | Flow Rato |  |  |  |  |  | Power Model Predicted O$\left(n^{\prime} 3 / \mathrm{sec}\right)$ |  |  | Now Model Predicted Q <br> ( $\mathrm{m}^{1} 3 / \mathrm{sec}$ ) |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| ( $\mathrm{n} . \mathrm{Wg}$ ) | (Pa) | Rep. 1 (chrit | Rep2 <br> (cliti) | Rep3 <br> (clart) | Systom Leak (dinc) | Corncte | dmean $\left(\begin{array}{rr} \\ 3 / s e c\end{array}\right)$ |  |  |  |  |  |  |
| 0.02 | 5 | 1448 | 14.98 | 15.24 | 0.40 | 14.50 | 0000884 | 000669 |  |  | 000656 |  |  |
| 0.04 | 10 | 20.19 | 2060 | 20.80 | 0.56 | 19.97 | 0.00943 | 0.009600.01185 |  |  | 0.00956 |  |  |
| 0.06 | 15 | 25.16 | 25.68 | 25.97 | 0.68 | 24.93 | 0.01176 |  |  | 001185 | 001186 |  |  |
| 0.08 | 20 | 29.48 | 29.94 | 30.10 | 0.78 | 29.06 | 0.01372 | 0.01376 0.01545 |  |  |  |  |  |
| 0.10 | 25 | 33.24 | 33.52 | 3380 | 0.87 | 3265 | 0.01541 |  |  |  | 0.01552 |  |  |
| 0.12 | 30 | 36.58 | 3685 | 36.72 | 0.95 | 35.77 | 0.01688 | 0.01698 |  |  | 001707 |  |  |
| 0.14 | 35 | 40.21 | 40.70 | 40.82 | 1.03 | 39.55 | 0.01866 | 0.01840 |  |  | 0.01849 |  |  |
| 0.16 | 40 | 4268 | 43.16 | 4328 | 1.10 | 41.94 | 0.01980 | 0.01972 |  |  | 0.01982 |  |  |
| 0.18 | 45 | 4528 | 45.73 | 45.85 | 1.16 | 44.46 | 0.02098 | 0.08097 |  |  | 0.02106 |  |  |
| 020 | 50 | 48.01 | 48.44 | 48.55 | 122 | 47.11 | 000223 | 002215 |  |  | 0.0224 |  |  |
| 0.22 | 65 | 60.62 | 51.04 | 51.14 | 128 | 49.65 | 0.02343 | 0.02327 |  |  | . 0.02336 |  |  |
| 024 | 60 | 5275 | 63.16 | 53.16 | 1.34 | 51.68 | 0.02439 | 0.08435 0.02538 |  |  | 0.02443 |  |  |
| 026 | 65 | 54.91 | 55.20 | 55.30 | 1.39 | 53.75 | 0.02536 |  |  |  | 0.02546 |  |  |
| 0.28 | 70 | 5721 | 57.50 | 57.60 | 1.44 | 66.00 | 0.02643 | 0.02638 |  |  | 0.02645 |  |  |
| 0.30 | 75 | 5935 | 68.73 | 59.82 | 1.49 | 58.14 | 0.02744 | 0.02734 |  |  | 002740 |  |  |
| 0.32 | 80 | 61.46 | 61.82 | 61.91 | 1.54 | 60.19 | 0.02841 |  |  |  |  |  |  |
| 0.34 | 85 | 63.42 | 63.78 | 6887 | 1.59 | 6210 | 0.02331 |  |  |  |  |  |  |
| 0.36 | 90 | 65.33 | 65.69 | 65.69 | 1.63 | 63.94 | 0.03017 | C- 00029 |  |  | Cl $=3.545 E-03$ |  |  |
| 0.38 | 95 | 67.31 | 67.66 | 67.74 | 1.68 | 6589 | 0.03110 |  |  |  |  |  |  |
| 0.40 | 100 | 68.99 | 69.42 | 6951 | - 1.72 | 67.59 | 0.03190 |  |  |  | $\mathbf{C 2}$ | 021 |  |
| 0.42 | 105 | 71.06 | 71.56 | 71.73 | 1.76 | 69.69 | 0.03289 | $\mathrm{n}=$ | 0.5197 |  |  |  |  |
| 0.44 | 110 | 72.69 | 73.18 | 7334 | 1.80 | 71.27 | 0.03363 |  |  |  | C3F $\quad 0.84$ |  |  |
| 0.46 | 115 | 74.68 | 75.17 | 75.33 | 184 | 7322 | 0.03456 |  |  |  |  |  |  |  |
| 0.48 | 120 | 76.42 | 76.74 | 76.90 | 188 | 74.80 | 0.03530 | C.V. | 0.6479 | (\%) | C.V $=$ | 0.6876 | (\%) |
| 0.5 | 125 | 77.67 | 78.06 | 78.22 | 1.92 | 76.07 | 0.03590 |  |  |  |  |  |  |

Parallel Crack Leakage Calculation: (A@F1)


Parallel Crack Leakage Calculation: (B1@B2)



Parallel Crack Leakage Calculation: ( $\mathrm{B} 1 @ \mathrm{~F} 1$ )


Paralle! Crack Leakage Calculation: (F1@F2)


## APPENDIX I

Data Tables for Openings in Series


Series Crack Leakage Calculation: (Bi~A)

| Pressure Difference Across Crack |  | Flow Rade |  |  |  |  |  | Power Model Predicted 0 <br> ( $\left.\mathrm{m}^{2} 3 / \mathrm{sec}\right)$ |  |  | Now Model Predicted Q <br> ( $\mathrm{m}^{2} 3 /$ sec) |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| ( $\mathrm{ln} . \mathrm{Wg}$ ) | (Pa) | $\begin{aligned} & \text { Fiep. } 1 \\ & \text { (dim) } \end{aligned}$ | Rep. 2 <br> (cimb) | $\begin{gathered} \text { Repa } \\ \text { (ctrin) } \end{gathered}$ | $\left.\begin{gathered} \text { Systom } \\ \text { Leak } \\ \text { (cfme } \end{gathered} \right\rvert\,$ | $\begin{aligned} & \text { Correct } \\ & \text { foind } \end{aligned}$ | codmean <br> ( $\quad \mathrm{m}^{2} 3 / \mathrm{sec}$ ) |  |  |  |  |  |  |
| 0.02 | 5 | 0.70 | 0.62 | 0.72 | 0.40 | 0.28 | 0.00013 |  | 0.00017 |  |  | 0.00019 |  |
| 0.04 0.06 | 10 15 | 1.30 1.85 | 1.30 | 1.22 | 0.56 0.68 | 0.71 1.12 | 0.00034 0.00053 |  | $\begin{aligned} & 0.00032 \\ & 0.00047 \end{aligned}$ |  |  | $\begin{aligned} & 000036 \\ & 000052 \end{aligned}$ |  |
| 0.08 | 20 | 221 | 227 | 2.15 | 0.78 | 1.43 | 0.00067 |  | 0.00062 |  |  | 0.00068 |  |
| 0.10 | 25 | 263 | 253 | 257 | 087 | 1.74 | 0.00082 |  | 0.00076 |  |  | 0.00082 |  |
| 0.12 | 30 | 307 | 301 | 3.02 | 0.95 | 208 | 000098 |  | 000091 |  |  | 0.00096 |  |
| 0.14 | 35 | 333 | 338 | 333 | 1.03 | 232 | 0.00109 |  | 0.00105 |  |  | 0.00109 |  |
| 0.16 0.18 | 40 | 3.70 4.04 | 3.70 399 | 3.70 4.04 | 1.10 1.16 | 260 286 | 0.00123 0.00135 |  | 0.00120 0.00134 |  |  | 0.00122 |  |
| 0.18 | 45 | 4.04 | 3.99 | 4.04 | 1.16 | 286 | 0.00135 |  | 000134 |  |  | 0.00135 |  |
| 0.20 | 50 | 4.38 | 4.43 | 4.34 | 122 | 3.16 | 0.00149 |  | 0.00148 |  |  | 0.00147 |  |
| 022 | 55 | 4.62 | 4.57 | 4.62 | 128 | 332 | 0.00157 |  | 0.00162 |  |  | 0.00158 |  |
| 0.24 | 60 | 5.01 | 4.94 | 4.96 | 1.34 | 363 | 000171 |  | 0.00176 |  |  | 0.00169 |  |
| 0.26 | 65 | 5.08 | 523 | 5.13 | 1.39 | 3.75 | 0.00177 |  | 0.00190 |  |  | 0.00180 |  |
| 028 | 70 | 5.49 | 5.44 | 5.49 | 144 | 4.03 | 0.00190 |  | 0.00204 |  |  | 0.00191 |  |
| 030 | 75 | 5.80 | 5.70 | 5.80 | 149 | 427 | 0.00202 |  | 0.00217 |  |  | 0.00201 |  |
| 0.32 | 80 | 5.96 | 6.07 | 5.96 | 1.54 | 4.45 | 0.00210 |  |  |  |  |  |  |
| 0.34 | 85 | 6.27 | 627 | 6.27 | 1.59 | 4.09 | 0.00221 |  |  |  |  |  |  |
| 0.36 | 90 | 6.58 | 6.49 | 6.58 | 1.63 | 4.92 | 0.00232 |  | 3.58E-05 |  | C1 - | 4.262E-04 |  |
| 0.38 | 95 | 6.78 | 6.71 | 6.79 | 1.68 | 5.08 | 0.00240 |  |  |  |  |  |  |
| 0.40 | 100 | 7.00 | 6.91 | 6.99 | 1.72 | 524 | 0.00247 |  |  |  | $C 2=$ | 533 |  |
| 0.42 | 105 | 7.20 | 7.11 | 7.20 | 1.76 | 5.41 | 0.00255 | $\mathrm{n}=$ | 0.9511 |  |  |  |  |
| 0.44 | 110 | 7.53 | 7.44 | 7.49 | 1.80 | 5.68 | 0.00268 |  |  |  | $\mathbf{C 3}=$ | 0.97 |  |
| 0.46 | 115 | 7.59 | 7.69 | 7.60 | 1.84 | 5.79 | 0.00273 |  |  |  |  |  |  |
| 0.48 | 120 | 7.71 | 7.76 | 7.72 | 1.88 | 6.85 | 0.00276 | C.V.e | 6.7966 | (\%) | C.V. | 1.8753 | (\%) |
| 0.5 | 125 | 7.96 | 8.01 | 7.93 | 1.92 | 6.05 | 0.00285 |  |  |  |  |  |  |


| Pressure Diflerence Across Crack |  | Flow |  |  | Pate |  |  | Power Model Predicted 0 <br> ( $\mathrm{m}^{2} 3 /$ sec. $)$ |  |  | Now Model <br> Preoficted 0 <br> ( $\mathrm{m}^{+} 3 / \mathrm{sec}$ ) |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| ( n Wg) | ( P ) | Rapl <br> (din) | $\begin{aligned} & \text { Fop. } 2 \\ & \text { (cind) } \end{aligned}$ | $\begin{aligned} & \text { Repos } \\ & \text { (ctri) } \end{aligned}$ | Systern Leak (dari) | $\begin{aligned} & \text { Correcte } \\ & \text { (cdrn) } \end{aligned}$ | dodMean (m³tsec.) |  |  |  |  |  |  |
| 0.02 | 5 | 240 | 243 | 243 | 0.40 | 202 | 0.00095 |  | 0.00099 |  |  | 0.00090 |  |
| 0.04 | 10 | 3.91 | 3.91 | 3.91 | 0.56 | 3.36 | 0.00158 |  | 0.00156 |  |  | 0.00154 |  |
| 0.06 | 15 | 4.93 | 6.18 | 5.08 | 0.68 | 4.39 | 0.00207 |  | 0.00205 |  |  | 0.00206 |  |
| 0.08 | 20 | 6.05 | 6.05 | 6.00 | 0.78 | 5.25 | 0.00248 |  | 0.00248 |  |  | 0.00251 |  |
| 0.10 | 25 | 701 | 7.01 | 7.11 | 0.87 | 6.17 | 0.00291 |  | 0.00287 |  |  | 0.00291 |  |
| 0.12 | 30 | 7.95 | 7.77 | 786 | 095 | 691 | 0.00326 |  | 0.00324 |  |  | 0.00388 |  |
| 0.14 | 35 | 809 | 8.63 | 8.63 | 1.03 | 7.03 | 0.00360 |  | 0.00359 |  |  | 000362 |  |
| 0.16 | 40 | 839 | 231 | 831 | 1.10 | 824 | 000399 |  | 000392 |  |  | 0.00394 |  |
| 0.18 | 45 | 10.15 | 10.01 | 10.05 | 1.16 | 8.91 | 0.00420 |  | 000424 |  |  | 0.00424 |  |
| 0.20 | 60 | 10.83 | 10.71 | 10.83 | 122 | 9.57 | 000452 |  | 000455 |  |  | 000453 |  |
| 0.22 | 65 | 11.51 | 11.47 | 11.47 | 128 | 1020 | 0.00481 |  | 000484 |  |  | 0.00480 |  |
| 024 | 60 | 1209 | 1202 | 1205 | 134 | 10.72 | 0.00506 |  | 0.00513 |  |  | 000506 |  |
| 026 | 65 | 1276 | 12.75 | 1275 | 139 | 11.36 | 000536 |  | 0.00541 |  |  | 000532 |  |
| 028 | 70 | 13.07 | 1334 | 13.18 | 144 | 11.75 | 000555 |  | 0.00568 |  |  | 0.00556 |  |
| 0.30 | 75 | 13.90 | 13.79 | 1383 | 149 | 1235 | 0.00583 |  | 0.00595 |  |  | 0.00579 |  |
| 0.32 | 80 | 14.46 | 14.35 | 14.42 | 1.54 | 1287 | 0.00607 |  |  |  |  |  |  |
| 034 | 65 | 15.03 | 14.99 | 14.99 | 1.59 | $13.42$ | 0.00633 |  |  |  |  |  |  |
| 036 | 90 | 1534 | 15.54 | 15.51 | 1.63 | $1383$ | 0.00653 |  | 3.40E-04 |  | C1 = | 7260E-04 |  |
| 0.38 | 95 | 16.06 | 16.5 | 16.12 | 1.68 | 14.42 | 0.00680 |  |  |  |  |  |  |
| 0.40 | 100 | 16.64 | 16.61 | 16.68 | 1.72 | 1492 | 0.00704 |  |  |  | C2\% | 1.9 |  |
| 0.42 | 105 | 16.87 | 17.08 | 16.90 | 1.76 | 15.19 | 000717 |  | 0.6628 |  |  |  |  |
| 0,44 | 110 | 17.45 | 17.58 | 17.37 | 1.80 | 15.67 | 0.00739 |  |  |  | C. | 125 |  |
| 0.46 | 115. | 18.05 | 18.12 | 17.98 | 1.84 | 1621 | 0.00765 |  |  |  | C.V |  |  |
| 0.48 | 120 | 18.45 | 18.61 | 18.51 | 1.88 | 16.64 | 000786 | C.V | 1.6334 | (\%) | C.V- | 0.9164 | (x) |
| 05 | 125 | 1901 | 19.01 | 18.94 | 1.92 | 17.07 | 000806 |  |  |  | , |  |  |

Series Crack Leakage Calculation: (E~B1)


| Pressure Diference Across Crack |  | Flow Rate |  |  |  |  |  | Power Model Predkted 0 <br> ( $\mathrm{m}^{\wedge} 3 / \mathrm{sec}$.) |  | Now Model <br> Predicled O <br> ( $\mathrm{m}^{\wedge} 3 / \mathrm{sec}$ ) |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | Rep. 1 <br> (cim) | Rep. 2 <br> (dm) | $\begin{aligned} & \text { Rep. } 3 \\ & \text { (cfrot) } \end{aligned}$ | $\left\lvert\, \begin{aligned} & \left.\begin{array}{l} \text { yssom } \\ \text { Leak } \\ \text { (dmy } \end{array} \right\rvert\, \end{aligned}\right.$ |  | dMean <br> ( $\mathrm{m}^{2} 3 /$ /sec.) |  |  |  |  |  |
| 0.02 | 5 | 11.29 | 10.97 | 10.63 | 0.40 | 10.57 | 0.00499 | 0.00524 |  |  | 0.00527 |  |
| 0.04 | 10 | 17.16 | 17.40 | 17.16 | 0.56 | 16.68 | 0.00787 | 0.00769 |  |  | 0.00784 |  |
| 0.06 | 15 | 21.16 | 21.56 | 21.36 | 0.68 | 2068 | 000976 | 0.00962 |  |  | 000981 |  |
| 0.08 | 20 | 25.17 | 25.35 | 25.18 | 0.78 | 24.45 | 0.01154 | 0.01128 |  |  | 0.01148 |  |
| 0.10 | 25 | 2842 | 28.75 | 28.59 | 0.87 | 27.72 | 0.01308 | 0.01277 |  |  | 0.01295 |  |
| 0.12 | 30 | 31.55 | 31.44 | 31.27 | 0.95 | 30.47 | 0.01438 | 0.01412 |  |  | 0.01428 |  |
| 0.14 | 35 | 34.21 | 33.96 | 33.81 | 1.03 | 32.97 | 0.01556 | 0.01538 |  |  | 0.01550 |  |
| 0.16 | 40 | 3634 | 3662 | 36.49 | 1.10 | 35.39 | 001670 | 0.01656 |  |  | 001664 |  |
| 0.18 | 45 | 3899 | 38.77 | 38.65 | 1.16 | 37.64 | 0.01776 | 0.01767 |  |  | 001771 |  |
| 0.20 | 50 | 4094 | 40.96 | 41.08 | 1.22 | 39.77 | 001877 | 001873 |  |  | 0.01873 |  |
| 0.22 | 55 | 42.83 | 4307 | 42.95 | 128 | 41.70 | 001968 | 0.01875 |  |  | 0.01969 |  |
| 0.24 | © | 45.09 | 45.11 | 44.99 | 1.34 | 43.72 | 0.02064 | 0.02072 |  |  | 0.02061 |  |
| 0.26 | 65 | 46.40 | 46.97 | 46.86 | 1.39 | 4535 | 0.02140 | 0.02166 |  |  | 002149 |  |
| 0.28 | 70 | 48.55 | 48.57 | 48.46 | 1.44 | 47.09 | 0.0232 | 0.02257 |  |  | 0.02234 |  |
| 0.301 | 751 | sa.13 | 5uso | 50 ${ }^{\text {cis }}$ | isis | - | смma: | anoves |  |  | 0.02316 |  |
| 032 | 80 | 51.96 | 51.89 | 51.68 | 1.54 | 60.30 | 0.02374 |  |  |  |  |  |
| 034 | 85 | 53.66. | 5359 | 53.59 | 1.59 | 5202 | 002455 |  |  |  |  |  |
| 0.36 | 90 | 54.94 | 55.25 | 55.35 | 1.63 | 53.55 | 0.02527 | C. 0.00215 |  | C1- | 1.808E-03 |  |
| 0.38 | 95 | 56.67 | 5688 | 56.59 | 1.68 | 55.03 | 0.02597 |  |  |  |  |  |
| 040 | 100 | 58.27 | 58.11 | 57.91 | 1.72 | 5638 | 008661 |  |  | C2\% | 0.59 |  |
| 0.42 | 105 | 59.75 | 59.78 | 59.59 | 1.76 | 57.94 | 0.02735 | $n=0.5534$ |  |  |  |  |
| 0.44 | 110 | 61.11 | 60.31 | 60.30 | 1.80 | 58.78 | 0.02774 |  |  | C3- | 239 |  |
| 0.46 | 115 | 6256 | 6266 | 62.66 | 1.84 | 60.79 | 0.02869 |  |  |  |  |  |
| 0.48 | 120 | 63.88 | 64.18 | 64.09 | 188 | 62.17 | 0.02934 | C.V $=1.5720$ | (\%) | C.V. $=$ | 0.7352 | (\%) |
| 0.5 | 125 | 65.45 | 6539 | 6521 | 1.92 | 63.43 | 0.02994 |  |  |  |  |  |

Series Crack Leakage Calculation: (F1~B1)


| Pressure DMference Across Crack |  | Flow Rate |  |  |  |  |  | Fower Model Predicted Q <br> ( $\mathrm{m}^{\mathrm{r}} 3 / \mathrm{sec}$ ) |  |  | New Model <br> Predicted 0 <br> ( $\mathrm{m}^{\wedge} 3 /$ sec) $)$ |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | $\begin{aligned} & \text { Rep.1 } \\ & \text { (dimi) } \end{aligned}$ | Rep2 <br> (ctril) | Repa <br> (ctmi) | Systom Leak (cimp) | $\begin{aligned} & \text { Cortecte } \\ & \text { (ctros) } \end{aligned}$ | dMean <br> ( $\left.\mathrm{m}^{2} 3 / \mathrm{sec}\right)$ |  |  |  |  |  |  |
| 0.02 | 5 | 10.96 | 1130 | 10.85 | 040 | 10.64 | 0.00502 |  | 0.00529 |  |  | 0.00530 |  |
| 0.04 | 10 | 17.15 | 1740 | 1764 | 0.56 | 16.84 | 000795 |  | 0.00774 |  |  | 000786 | + |
| 0.06 | 15 | 21.74 | 21.76 | 21.78 | 0.68 | 21.08 | 0.00995 |  | 0.00968 |  |  | 0.00984 |  |
| 0.08 | 20 | 2534 | 2535 | 2520 | 0.78 | 24.52 | 0.01157 |  | 001134 |  |  | 001151 |  |
| 0.10 | 25 | 28.42 | 28.74 | 2861 | 0.87 | 27.72 | 001308 |  | 001292 |  |  | 001298 |  |
| 0.12 | 30 | 51.13 | 31.44 | \$1.31 | 0.95 | 3034 | 0.01432 |  | 0.01417 |  |  | 001431 |  |
| 0.14 | 35 | 34.07 | 3423 | 33.98 | 1.03 | 33.07 | 0.01561 |  | 0.01543 |  |  | 001553 |  |
| 0.16 | 40 | 3647 | 36.49 | 3638 | 1.10 | 3535 | 0.01668 |  | 001660 |  |  | 0.01667 |  |
| 0.18 | 45 | 38.76 | 38.90 | 3880 | 1.16 | 37.65 | 0.0177 |  | 001772 |  |  | 001774 |  |
| 020 | 60 | 40.94 | 41.08 | 40.88 | 1.22 | 39.74 | 001876 |  | 0.01877 |  |  | 001875 |  |
| 0.22 | 55 | 4293 | 43.19 | 43.11 | 128 | 41.80 | 0.01973 |  | 001979 |  |  | 0.01972 |  |
| 024 | 60 | 44.97 | 45.44 | 45.37 | 134 | 4392 | 0.02073 |  | 0.02076 |  |  | 0.02064 |  |
| 0.26 | 65 | 48.95 | 46.58 | 46.90 | 1.39 | 45.55 | 0.02150 |  | 0.02169 |  |  | 0.02152 |  |
| 028 | 70 | 4887 | 48.79 | 48.62 | 1.44 | 47.32 | 0.02233 |  | 0.02859 |  |  | 0.02237 |  |
| 0.30 | 75 | 50.43 | 60.56 | 50.40 | 149 | 48.97 | 0.02311 |  | 0.02347 |  |  | 0.02319 |  |
| 0.32 | 80 | 62.23 | 5239 | 52.13 | 1.54 | 60.72 | 0.02394 |  |  |  |  |  |  |
| 034 | 85 | 5396 | 54.09 | 53.94 | 1.59 | 5241 | 0.02474 |  |  |  |  |  |  |
| 036 | 90 | 55.52 | 6565 | 55.50 | 1.63 | 5392 | 002545 | C= | 000218 |  | C1- | .750E03 |  |
| 038 | 95 | 57.33 | 67.07 | 57.03 | 1.68 | 55.47 | 0.02618 |  |  |  |  |  |  |
| 0.40 | 100 | 58.73 | 58.76 | 58.63 | 1.72 | 5698 | 0.02690 |  |  |  | C2- | 0.59 |  |
| 0.42 | 105 | 6020 | 60.42 | 6029 | 1.76 | 5854 | 0.02763 | $\boldsymbol{n}=$ | 0.5504 |  |  |  |  |
| 0.44 | 110 | 61.57 | 61.88 | 61.66 | 1.80 | 69.90 | 0.08827 |  |  |  | C\% | 255 |  |
| 046 | 115 | 6309 | 63.48 | 63.45 | 184 | 61.50 | 002902 |  |  |  |  |  |  |
| 048 | 120 | 64.67 | 64.79 | 64.58 | 188 | 62.80 | 0.02964 | CVe | 1.3651 | (\%) | C.V. | 0.6415 | (3) |
| 0.5 | 125 | 6589 | 6592 | 65.72 | 1.92 | 6352 | 0.05017 |  |  |  |  |  |  |

Component Leakage Test: Exterior Frame / Gypsum Board: CO-1

| Pressure Diflerence Across Crack |  | Flow Rate |  |  |  |  |  | Power Model Predicted $Q$ <br> ( $\mathrm{m}^{2} 3 /$ sec ) |  | Now Model Predicted a <br> ( $\left(\mathrm{m}^{2} 3\right.$ /sec) |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | Rep. 1 <br> (cmid) | Rep2 <br> (dme) | Rep 3 <br> (cfing) | Chariber Leak (cinct) | Corrected Mean (clm) ( $\mathrm{m}^{1} 3 / \mathrm{sec}$ ) |  |  |  |  |
| ( $\mathrm{n} . \mathrm{Wg}$ ) | (Pa) |  |  |  |  |  |  |  |  |
| 0.02 | 5 | 0.37 | 0.37 | 0.35 | 0.29 | 0.08 | 0.00004 |  | 0.00005 |  | 0.00007 |
| 0.04 | 10 | 0.70 | 0.70 | 0.57 | 0.41 | 0.24 | 0.00011 | 0.00010 |  | 0.00013 |
| 0.06 | 15 | 0.94 | 0.94 | 0.84 | 0.51 | 0.39 | 0.00019 | 0.00016 |  | 0.00019 |
| 0.08 | 20 | 1.17 | 1.17 | 1.09 | 0.60 | 0.55 | 0.00026 | 0.00021 |  | 0.00024 |
| 0.10 | 25 | 1.31 | 1.39 | 1.15 | 0.67 | 0.62 | 0.00029 | 0.00026 |  | 0.00029 |
| 0.12 | 30 | 1.52 | 1.51 | 1.37 | 0.74 | 0.73 | 0.00034 | 0.00032 |  | 0.00034 |
| 0.14 | 35 | 1.64 | 1.71 | 149 | 0.80 | 0.81 | 0.00038 | 0.00037 |  | 0.00039 |
| 0.16 | 40 | 1.83 | 1.83 | 1.75 | 0.86 | 0.95 | 0.00045 | 0.00043 |  | 0.00044 |
| 0.18 | 45 | 1.95 | 201 | 1.88 | 0.91 | 1.03 | 0.00049 | 0.00048 |  | 0.00048 |
| 0.20 | 50 | 213 | 2.13 | 1.99 | 0.97 | 1.12 | 0.00053 | 0.00053 |  | 0.00052 |
| 0.22 | 65 | 223 | 2.24 | 216 | 1.01 | 120 | 0.00056 | 0.00059 |  | 0.00056 |
| 0.24 | 60 | 240 | 240 | 227 | 1.06 | 1.30 | 0.00061 | 0.00084 * |  | 0.00060 |
| 026 | 65 | 251 | 2.51 | 238 | 1.11 | 1.36 | 0.00064 | 0.00070 |  | 0.00064 |
| 0.28 | 70 | 262 | 262 | 249 | 1.15 | 143 | 0.00067 | 0.00075 |  | 0.00068 |
| 0.30 | 75 | 273 | 2.79 | 259 | 1.19 | 1.51 | 0.00071 | 0.00081 |  | 0.00072 |
| 0.32 | 80 | 283 | 2.89 | 275 | 1.24 | 1.59 | 0.00075 |  |  |  |
| 0.34 | 85 | 292 | 2.98 | 286 | 1.28 | 1.64 | 0.00078 |  |  |  |
| 0.36 | 90 | 303 | 3.08 | 290 | 1.31 | 1.69 | 0.00080 | C- 1.01E-05 |  | $C 1=1260 E-04$ |
| 0.38 | 95 | 3.13 | 3.18 | 301 | 120 | 1.15 | บuTios |  |  |  |
| 0.40 | 100 | 3.23 | 3.28 | 3.16 | 1.39 | 1.83 | 0.00087 |  |  | C2\% 6.04 |
| 0.42 | 105 | 338 | 3.38 | 325 | 1.43 | 1.91 | 0.00090 | $n=1.0145$ |  |  |
| 0.44 | 110 | 3.47 | 3.52 | 3.40 | 146 | 200 | 0.00095 |  |  | $\mathrm{CK}=\quad 1.35$ |
| 0.46 | 115 | 3.56 | 3.62 | 350 | 1.49 | 207 | 0.00088 |  |  |  |
| 0.48 | 120 | 3.66 | 3.71 | 3.60 | 1.53 | 2.13 | 0.00100 | C.V- 10.5207 | (\%) | C.V $=29093$ |
| 0.5 | 125 | 3.81 | 3.86 | 369 | 1.56 | 223 | 0.00105 |  |  |  |

Component Leakage Test: Exterior Frame / Insulating Board: CO-2

| Pressure Difierence Across Crack |  | Fow Rate |  |  |  |  |  | Power Model Predicted Q ( $\mathrm{m}^{2} 3 / \mathrm{sec}$.) |  | New Model <br> Predicted Q <br> ( $\mathrm{m}^{\wedge} 3 /$ sec) |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | Rep. 1 | Rep2 | Repa | Charber Leak | Correcied Mean |  |  |  |  |
| ( n W9) | (Pa) | (cim) | (cfrel) | (chind | ( $\mathrm{dm} \mathrm{m}^{\text {d }}$ | (ctm) | ( $\mathrm{m}^{\wedge} 3 / \mathrm{sec}$ ) |  |  |  |
| 0.02 | 5 | 3.72 | 4.42 | 4.73 | 0.29 | 4.00 | 0.00189 | 0.00200 |  | 0.00209 |
| 0.04 | 10 | 5.95 | 6.97 | 6.88 | 0.41 | 6.19 | 0.00292 | 0.00294 |  | 0.00305 |
| 0.06 | 15 | 745 | 8.91 | 8.78 | 0.51 | 7.86 | 0.00371 | 0.00369 |  | 0.00380 |
| 0.08 | 20 | 8.97 | 10.80 | 10.47 | 0.60 | 9.48 | 0.00447 | 0.00432 |  | 0.00443 |
| 0.10 | 25 | 10.12 | 1219 | 1206 | 0.67 | 10.79 | 0.00509 | 0.00489 |  | 0.00498 |
| 0.12 | 30 | 11.23 | 1334 | 13.42 | 0.74 | 11.93 | 0.00563 | 0.00542 |  | 0.00548 |
| 0.14 | 35 | 1251 | 14.49 | 14.09 | 0.80 | 12.90 | 0.00609 | 0.00590 |  | 0.00594 |
| 0.16 | 40 | 13.41 | 15.45 | 15.20 | 0.86 | 13.83 | 0.00653 | 0.00635 |  | 0.00637 |
| 0.18 | 45 | 14.39 | 16.27 | 13.46 | 0.91 | 14.63 | 0.00090 | 0.00678 |  | 0.00677 |
| 0.20 | 50 | 15.26 | 17.06 | 16.67 | 0.97 | 15.37 | 0.00725 | 0.00719 |  | 0.00715 |
| 0.22 | 55 | 16.16 | 17.79 | 1731 | 1.01 | 16.07 | 0.00759 | 0.00758 |  | 0.00752 |
| 0.24 | 60 | 16.71 | 1847 | 17.88 | 1.06 | 16.62 | 0.00785 | 0.00796 |  | 0.00786 |
| 0.26 | 6 | 17.45 | 18.99 | 18.44 | 1.11 | 17.19 | 0.00811 | 0.00832 |  | 0.00819 |
| 0.28 | 70 | 17.96 | 19.58 | 18.76 | 1.15 | 1761 | 0.00831 | 0.00067 |  | 0.00851 |
| 0.30 | 75 | 1861 | 20.13 | 19.37 | 1.19 | 18.17 | 0.00658 | 0.00901 |  | 0.00882 |
| 032 | 80 | 1921 | 20.55 | 1981 | 124 | 18.62 | 0.00879 |  |  |  |
| 0.34 | 85 | 19.87 | 20.85 | 20.34 | 1.28 | 19.08 | 0.00900 |  |  |  |
| 0.36 | 90 | 20.26 | 21.25 | 20.78 | 1.31 | 19.45 | 0.00918 | $C=8.19 E-04$ | - | C1 = 3.940E-04 |
| 0.38 | 95 | 20.54 | 21.53 | 21.06 | 1.35 | 19.69 | 0.00929 |  |  |  |
| 0.40 | 100 | 21.02 | 21.81 | 21.65 | 1.39 | 20.10 | 0.00949 |  |  | $\mathrm{C}=0.71$ |
| 0.42 | 105 | 21.25 | 2228 | 2215 | 1.43 | 20.47 | 0.00966 | $n=0.5554$ |  |  |
| 0.44 | 110 | 21.39 | 2276 | 22.51 | 146 | 20.78 | 0.00981 |  |  | C3- 7.11 |
| 0.46 | 115 | 21.61 | 2297 | 23.01 | 1.49 | 21.03 | 0.00993 |  |  |  |
| 0.48 | 120 | 21.93 | 2363 | 22.46 | 1.53 | 21.48 | 0.01014 | C.V.- 3.4685 | (\%) | C.V. $=2.4459$ |
| 0.5 | 125 | 2234 | 2387 | 23.84 | 1.56 | 21.79 | 0.01028 |  |  |  |

## APPENDIX J

Data Tables for Component Leakage Tests

Component Leakage Test: Wall Penetrations / Outlets: CO-3


Component Leakage Test: Wall Penetrations / Oullets wilh Gaskets: CO-4



Component Leakage Test: Wall Penetrations/ Outlets, Top Wire Holes Sealed: CO-6


| Pressure Diference Across Crack |  | Flow Rate |  |  |  |  |  | Power Model Prodicted Q$\left(m^{\wedge} 3 / s e c .\right)$ |  |  | Now Model Predicted O <br> ( $\mathrm{m}^{\wedge} 3 / \mathrm{sec}$ ) |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | Rep. 1 <br> (dm) | Rep. 2 <br> (clme) | Rep. 3 <br> (chmit) | Chamber Laak (cims) | Corrected Mean (dmi) $\left(m^{\wedge} 3 / \mathrm{sec}\right.$ ) |  |  |  |  |  |
| $(\mathrm{n} 2 \mathrm{Wg})$ | (Pa) |  |  |  |  |  |  |  |  |
| 0.02 | 5 | 1.81 | 1.66 | 1.81 | 0.29 | 147 | 0.00070 |  |  |  | 0.00071 |  | 0.00062 |
| 0.04 | 10 | 2.87 | 287 | 300 | 0.41 | 2.50 | 0.00118 |  | 0.00116 |  | 0.00112 |
| 0.06 | 15 | 3.72 | 3.77 | 3.91 | 0.51 | 329 | 0.00155 |  | 0.00155 |  | 0.00154 |
| 0.08 | 20 | 4.61 | 4.56 | 4.87 | 0.00 | 4.08 | 0.00193 |  | 0.00190 |  | 0.00191 |
| 0.10 | 25 | 5.28 | 6.33 | 5.56 | 0.67 | 4.72 | 0.0023 |  | 0.00232 |  | 000225 |
| 0.12 | 30 | 5.97 | 6.09 | 6.19 | 0.74 | 5.35 | 0.00252 |  | 0.00253 |  | 0.00256 |
| 0.14 | 35 | 6.62 | 683 | 6.92 | 0.80 | 5.99 | 0.00283 |  | 000282 |  | 0.00285 |
| 0.16 | 40 | 725 | 7.48 | 7.65 | 086 | 6.60 | 0.00312 |  | 000310 |  | 0.00313 |
| 0.18 | 45 | 781 | 7.97 | 831 | 0.91 | 7.12 | 0.00336 |  | 0.00337 |  | 000339 |
| 0.20 | 50 | 8.42 | 8.40 | 800 | 0.97 | 761 | 0.00359 |  | 0.00363 |  | 0.00363 |
| 022 | 65 | 8.94 | 231 | 9.33 | 1.1 | 8.18 | 000396 |  | 0.00388 |  | 000387 |
| 0.24 | $\infty$ | 9.49 | 2.78 | 9.93 | 1.06 | 8.67 | 000409 |  | 0.00412 |  | 0.00410 |
| 0.26 | 65 | 1008 | 1031 | 10.63 | 1.11 | 2.23 | 000436 |  | 000436 |  | 0.00432 |
| 0.28 | 70 | 10.50 | 1087 | 11.03 | 1.15 | 9.65 | 0.00455 |  | 0.00460 |  | 0.00453 |
| บั̇ | \% | 1:\% | 4.485 | 41.90 | 1.10 | 10.20 | 0.00481 |  | 0.00483 |  | 0.00473 |
| 0.32 | 80 | 11.47 | 11.85 | 1203 | 124 | 10.55 | 0.00498 |  |  |  |  |
| 034 | 85 | 11.99 | 1258 | 1270 | 128 | 11.15 | 0.00526 |  |  |  |  |
| 0.36 | 90 | 1239 | 13.06 | 1335 | 1.31 | 11.02 | 0.00548 | - $\mathbf{C =}$ | ERR |  | $\mathrm{Cl}=721 \mathrm{E}-04$ |
| 0.38 | 95 | 12.97 | 13.60 | 13.67 | 1.35 | 1206 | 0.00569 |  |  |  |  |
| 0.40 | 100 | 13.40 | 14.05 | 14.24 | 1.39 | 1251 | 0.00590 |  |  |  | C2= 249 |
| 0.42 | 105 | 13.75 | 14.51 | 14.66 | 1.43 | 1288 | 0.00608 | nm | 0.7061 |  |  |
| 044 | 110 | 14.14 | 1500 | 14.93 | 146 | 1323 | 0.00624 |  |  |  | C3\% 1.01 |
| 0.46 | 115 | 14.81 | 15.56 | 15.74 | 149 | 13.87 | 0.00655 |  |  |  |  |
| 0.48 0.5 | 120 | 15.19 15.64 | 15.79 1627 | 16.10 | 1.53 1.56 | $14.17$ | $0.00609$ | C.V $=$ | 0.7768 | (\%) | C.V.- 1.4510 |
| 0.5 | 125 | 15.64 | 1627 | 16.41 | 1.56 | 14.55 | $0.00696$ |  |  |  |  |

Component Leakage Test: Wall Penetrations / Outlets with Gaskets, Top Wire Holes Sealed: CO-8

| Pressure Difierence Across Crack |  | Flow Rate |  |  |  |  | Power Model Predicted 0 |  | Now Model Precicted $O$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | Hep. 1 | Repa 2 | Rep3 | Chambor Leak | Corrected Mean |  |  |  |
| $(\mathrm{n} W \mathrm{Wg})$ | (Pa) | (dme) | (chas) | (crmer | (ctrrs) | (cmil) (m'3/sec) | ( $\mathrm{m}^{\wedge} 3 / \mathrm{sec}$.) |  | ( $\mathrm{m}^{\wedge} 3 / \mathrm{sec}$ ) |
|  |  |  |  |  |  |  |  |  |  |
| 0.02 | 5 | 1.35 | 1.31 | 1.39 | 029 | 1.00000000 | 0.00053 |  | 0.00048 |
| 0.04 | 10 | 2.33 | 223 | 236 | 0.41 | 1.890 .00089 | 0.00066 |  | 0.00085 |
| 0.06 | 15 | 2.93 | 2.93 | 3.05 | 0.51 | 246000116 | 0.00115 |  | 0.00116 |
| 0.08 | 20 | 3.64 | 3.58 | 369 | 0.60 | 3.040 .00143 | 0.00140 |  | 0.00143 |
| 0.10 | 25 | 4.19 | 4.14 | 4.25 | 0.67 | 3.520 .00166 | 0.00164 |  | 0.00168 |
| 0.12 | 30 | 4.73 | 4.68 | 4.88 | 0.74 | 4.020 .00190 | 0.00187 |  | 0.00191 |
| 0.14 | 35 | 5.24 | 5.19 | 5.39 | 0.80 | 4.470 .00211 | 0.00208 |  | 0.00212 |
| 0.16 | 40 | 5.79 | 5.65 | 5.79 | 0.86 | 4.880 .00231 | 0.00229. |  | 0.00232 |
| 0.18 | 45 | 6.18 | 6.14 | 623 | 0.81 | 527000049 | 0.00249 |  | 0.00251 |
| 0.20 | 50 | 6.57 | 6.62 | 6.71 | 0.97 | 5.87000067 | 0.00268 |  | 0.00269 |
| 0.22 | 55 | 7.04 | 7.00 | 7.18 | 1.01 | 6.060 .00296 | 0.00236 |  | 0.00286 |
| 024 | 60 | $7 A 1$ | 737 | 7.03 | 1.06 | 6.410 .00303 | 0.00304 |  | 0.00303 |
| 0.26 | 65 | 7.83 | 7.80 | 8.00 | 1.11 | 6.770 .00319 | 0.00322 |  | 0.00319 |
| 0.28 | 70 | 8.25 | 8.07 | 2.45 | 1.15 | 7.100 .00335 | 0.00339 |  | 0.00334 |
| 0.30 | 76 | 8.60 | 8.56 | 8.76 | 1.19 | 7450.00351 | 0.00356 |  | 0.00349 |
| 0.32 | 80 | 8.92 | 8.87 | 8.15 | 1.24 | 7.750 .00366 |  |  |  |
| 0.34 | 85 | 9.31 | 2.15 | 9.54 | 128 | 8.060 .00390 |  |  |  |
| 0.36 | 90 | 9.74 | 9.58 | 9.89 | 1.31 | 8.420 .00397 | $C=1.70 E-04$ |  | C1 = 5.00E-04 |
| 0.38 | 95 | 10.00 | 10.01 | 10.20 | 1.35 | 8.720 .00411 |  |  |  |
| 0.40 | 100 | 10.35 | 1027 | 10.66 | 1.39 | 9.040 .00427 |  |  | $\mathrm{C} 2=\quad 237$ |
| $0.42^{+}$ | 105 | 10.77 | 10.65 | 10.91 | 1.43 | 9.350 .00441 | $n=0.7047$ |  |  |
| 0.44 | 110 | 11.14 | 10.98 | 11.29 | 146 | 9.68 0.00457 |  |  | C3- $\quad 1.09$ |
| 0.46 | 115 | 1141 | 11.17 | 11.63 | 1.49 | 9.910 .00468 |  |  |  |
| 0.48 | 120 | 11.84 | 11.70 | 11.96 | 1.53 | 10.300 .00486 | C.V. $=1.2873$ | (\%) | C.V.- 0.9263 |
| 0.5 | 125 | 1213 | 12.01 | 12.35 | 1.56 | $10.60 \quad 0.00500$ |  |  |  |

Component Leakage Test: Wall Penetrations / Copper Water Line: CO-9

| Pressure Diference Across Crack |  | Flow Prato |  |  |  |  |  | Power Model Predicled O |  |  | New Model |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | Rep. 1 | Rep. 2 | Repa 3 | Chamber Leak | Corrected Mean |  |  |  |  |  |  |
| (n.Wg) | (Pa) | (dmer) | (chral | (chm) | (chmi) | (dm) | ( $\mathrm{m} / 3 / \mathrm{sec}$ ) | ( $\left.\mathrm{m}^{\wedge} 3 / \mathrm{sec}.\right)$ |  |  | ( $\mathrm{m}^{*} 3$ /sec) |  |  |
| 0.02 | 5 | 040 | 0.40 | 0.40 | 029 | 0.12 | 0.00005 |  | 0.00007 |  |  |  | 0.00007 |  |
| 0.04 | 10 | 0.72 | 0.72 | 0.62 | 0.41 | 0.27 | 0.00013 |  | 0.00012 |  |  | 0.00013 |  |
| 0.06 | 15 | 0.89 | 0.89 | 0.89 | 0.51 | 0.38 | 0.00018 |  | 0.00017 |  |  | 0.00018 |  |
| 0.08 | 20 | 1.13 | 1.05 | 1.05 | 0.60 | 0.48 | 0.00023 |  | 0.00021 |  |  | 000023 |  |
| 0.10 | 25 | 1.27 | 1.27 | 1.27 | 0.67 | 0.60 | 0.00028 |  | 0.00026 |  |  | 0.00027 |  |
| 0.12 | 30 | 148 | 141 | 141 | 0.74 | 0.70 | 0.00033 |  | 0.00030 |  |  | 0.00032 |  |
| 0.14 | 35 | 1.61 | 1.54 | 1.54 | 0.80 | 0.76 | 0.00036 |  | 0.00035 |  |  | 0.00036 |  |
| 0.16 | 40 | 1.74 | 1.67 | 1.00 | 006 | 0.81 | 0.00038 |  | 000039 |  |  | 0.00040 |  |
| 0.18 | 45 | 1.86 | 1.79 | 1.72 | 0.91 | 0.88 | 0.00041 |  | 0.00043 |  |  | 000043 |  |
| 0.20 | 50 | 1.98 | 1.98 | 1.91 | 0.97 | 0.99 | 0.00047 |  | 0.00047 |  |  | 000047 |  |
| 022 | 55 | 214 | 209 | 1.98 | 1.01 | 1.05 | 0.00050 |  | 0.00051 |  |  | 0.00050 |  |
| 0.24 | $\infty$ | 226 | 221 | 208 | 1.06 | 1.12 | 0.00053 |  | 0.00055 |  |  | 0.00054 |  |
| 0.26 | 65 | 237 | 237 | 219 | 1.11 | 1.20 | 0.00057 |  | 0.00058 |  |  | 0.00057 |  |
| 0.28 | 70 | 248 | 242 | 229 | 1.15 | 1.24 | 0.00059 |  | 0.00062 |  |  | 0.00060 |  |
| 0.30 | 75 | 265 | 252 | 246 | 1.19 | 135 | 0.00064 |  | 0.00066 |  |  | 000063 |  |
| 0.32 | 80 | 269 | 263 | 257 | 1.24 | 140 | 0.00056 |  |  |  |  |  |  |
| 034 | 85 | 285 | 274 | 268 | 1.28 | 148 | 0.00070 |  |  |  |  |  |  |
| 036 | 90 | 296 | 284 | 273 | 1.31 | 1.53 | 0.00072 | Cm | 1.67E-05 |  | Ci* | 1.08E-04 |  |
| 038 | 95 | 3.06 | 2.95 | 2.83 | 1.35 | 1.59 | 0.00075 |  |  |  |  |  |  |
| 040 | 100 | 3.16 | 305 | 293 | 1.39 | 1.66 | 0.00078 |  |  |  | $\mathbf{C 2}$ | 4.15 |  |
| 0.42 | 105 | 320 | 320 | 3.04 | 1.43 | 1.72 | 0.00081 | $\mathrm{n}=$ | 0.8518 |  |  |  |  |
| 0.44 | 110 | 335 | 325 | 3.08 | 1.46 | 1.76 | 0.00083 |  |  |  | $\boldsymbol{\sim}$ | 1.10 |  |
| 0.46 | 115 | 3.45 | 335 | 3.18 | 1.49 | 1.83 | 0.00006 |  |  |  |  |  |  |
| 0.48 | 120 | 3.49 | 3.43 | 328 | 1.53 | 1.87 | 0.00088 | C.V. | 5.1187 | (\%) | C.V. | 27706 | (*) |
| 0.5 | 125 | 3.68 | 3.53 | 338 | 1.56 | 1.94 | 0.00091 |  |  |  |  |  |  |

Component Leakage Test: Wall Penetrations / Switches, Top Wire Holes: CO-10

| Pressure Dtierence Across Crack |  | Flow Rato |  |  |  |  |  | Power Model Predicled O ( $\mathrm{m}^{2} 3 / \mathrm{sec}$ ) |  | Now Model Predicted 0 <br> ( $\mathrm{m}^{2} 3$ /sec.) |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | Pep. 1 | Rep 2 | Rep. 3 | Chamber Leak (ctrid) | Corrected Mean <br> (dmi) ( $\mathrm{m}^{2} 3 /$ sec. $)$ |  |  |  |  |  |  |
| ( $\mathrm{n} . \mathrm{Wg}$ ) | (Pa) | ( d | (cfme) | (ctris) |  |  |  |  |  |  |  |
| 0.02 | 5 | 1.67 | 1.74 | 1.74 | 0.29 | 143 | 0.00068 |  | 0.00073 |  |  | 0.00066 |  |
| 0.04 | 10 | 3.08 | 320 | 3.08 | 0.41 | 271 | 0.00128 | 0.00122 |  |  | 0.00119 |  |
| 0.06 | 15 | 4.08 | 4.19 | 4.03 | 0.51 | 358 | 0.00169 | 0.00164 |  |  | 0.00165 |  |
| 0.08 | 20 | 5.10 | 6.10 | 4.95 | 0.60 | 4.46 | 0.00210 | 0.00203 |  |  | 0.00207 |  |
| 0.10 | 25 | 5.83 | 5.88 | 5.75 | 0.67 | 5.15 | 0.00243 | 0.00240 |  |  | 000244 |  |
| 0.12 | 30 | 6.71 | 609 | 6.36 | 0.74 | 585 | 0.00276 | 0.00274 |  |  | 0.00279 |  |
| 0.14 | 35 | 748 | 7.57 | 7.06 | 0.80 | 657 | 0.00310 | 0.00308 |  |  | 0.00312 |  |
| 0.16 | 40 | 8.10 | 8.19 | 7.86 | 0.86 | 7.19 | 0.00340 | 0.00339 |  |  | 0.00343 |  |
| 0.18 | 45 | 8.83 | 887 | 8.66 | 0.91 | 788 | 0.00372 | 0.00370 |  |  | 0.00372 |  |
| 0.20 | 50 | 9.50 | 2.50 | 9.14 | 0.97 | 8.42 | 0.00397 | 0.00400 |  |  | 0.00400 |  |
| 0.22 | 65 | 10.17 | 10.20 | 2.78 | 1.01 | 8.03 | 0.00426 | 0.00430 |  |  | 000427 |  |
| 0.24 | 60 | 10.74 | 1086 | 1039 | 1.06 | 9.60 | 0.00453 | 0.00458 |  |  | 000452 |  |
| 0.26 | 65 | 11.41 | 11.37 | 11.04 | 1.11 | 10.17 | 0.00480 | 0.00486 |  |  | 0.00477 |  |
| 0.28 | 70 | 1201 | 1205 | 11.44 | 1.15 | 10.68 | 0.00504 | 0.00513 |  |  | 0.00501 |  |
| 0.30 | 75 | 1267 | 1266 | 12.12 | 1.19 | 11.29 | 0.00533 | 0.00540 |  |  | 0.00524 |  |
| 0.32 | 80 | 1321 | 13.13 | 1258 | 124 | 11.74 | 0.00554 |  |  |  |  |  |
| 0.34 | 85 | 13.71 | 13.71 | 13.23 | 1.28 | 1228 | 0.00579 |  |  |  |  |  |
| 0.36 | 90 | 14.29 | 14.35 | 13.70 | 1.31 | 1280 | 0.00604 | $C=222 E-04$ |  | C1 = | 8.04E-04 |  |
| 0.38 | 95 | 14.92 | 14.81 | 14.20 | 1.35 | 1329 | 0.00627 |  |  |  |  |  |
| 0.40 | 100 | 15.54 | 15:47 | 14.83 | 1.39 | 13.89 | 0.00656 |  |  | C2- | 2.84 |  |
| 0.42 | 105 | 15.95 | 16.12 | 15.35 | 1.43 | 14.38 | 0.00679 | $n=0.7393$ |  |  |  |  |
| 0.44 | 110 | 16.66 | 16.56 | 15.91 | 1.46 | 14.91 | 0.00704 |  |  | C\% | 1.06 |  |
| 0.46 | 115 | 17.10 | 17.00 | 16.41 | 1.49 | 15.34 | 0.00724 |  |  |  |  |  |
| 0.48 | 120 | 17.71 | 1765 | 17.02 | 1.53 | 15.93 | 0.00752 | C.V. $=1.6489$ | (\%) | C.V $=$ | 1.3151 | (\%) |
| 0.5 | 125 | 18.25 | 18.25 | 17.73 | 1.56 | 16.51 | 0.00779 |  |  |  |  |  |

Component Leakage Test: Wall Penetrations / Switches, Top Wire Holes Sealed: CO-11


Component Leakage Test: Wall Penetrations / Switches with Gaskets, Top Wire Holes Sealed: CO-12


Component Leakage Test: Wall Penetrations / Switches with Gaskets, Top Wire Holes: co-13

| Pressure Diference Across Crack |  | Flow Rato |  |  |  |  |  | Power Model Predicteda <br> (m³/sec) |  |  | Now Model Predicted Q <br> ( $\mathrm{m}^{\wedge} 3 / \mathrm{sec}$.) |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| ( $\mathrm{n} . \mathrm{Wg}$ ) | (Pa) | Reg. 1 (dme) | Rop 2 <br> (chme | Rep3 <br> (ctrm | Chanter Leak (dim) | Corrocted Mean (dind) ( $\mathrm{m}^{2} 3 / \mathrm{sec}$ ) |  |  |  |  |  |  |  |
| 0.02 | 5 | 0.69 | 0.79 | 0.79 | 0.29 | 0.47 | 0.00023 |  | 0.00028 |  |  | 0.00030 |  |
| 0.04 | 10 | 1.72 | 1.50 | 1.5 | 0.11 | 121 | 0.00057 |  | 0.00052 |  |  | 000057 |  |
| 0.06 | 15 | 235 | 229 | 235 | 0.51 | 1.82 | 0.00086 |  | 0.00075 |  |  | 0.00082 |  |
| 0.08 | 20 | 287 | 209 | 287 | 0.50 | 221 | 0.00104 |  | 0.00097 |  |  | 0.00105 |  |
| 0.10 | 25 | 329 | 320 | 352 | 0.67 | 267 | 0.00126 |  | 0.00118 |  |  | 0.00126 |  |
| 0.12 | 30 | 378 | 3.72 | 403 | 0.74 | 3.10 | 000146 |  | 0.00140 |  |  | 000147 |  |
| 0.14 | 35 | 4.33 | 4.10 | 4.57 | 0.80 | 3.53 | 0.00167 |  | 0.00160 |  |  | 0.00168 |  |
| 0.16 | 40 | 4.65 | 4.55 | 4.99 | 0.86 | 387 | 0.00183 |  | 000181 |  |  | 0.00185 |  |
| 0.18 | 45 | 5.17 | 4.93 | 6.55 | 0.91 | 4.30 | 0.00203 |  | 000201 |  |  | 0.00802 |  |
| 020 | 50 | 5.53 | 534 | 589 | 0.97 | 4.62 | 0.00218 |  | 000221 |  |  | 0.00219 |  |
| 022 | 55 | 5.93 | 5.74 | 634 | 1.01 | 4.99 | 0.00235 |  | 0.00240 |  |  | 0.00236 |  |
| 0.24 | ${ }^{\circ}$ | 628 | 6.19 | 681 | 1.06 | 536 | 0.00253 |  | 0.00280 |  |  | 000052 |  |
| 026 | 65 | 6.63 | 6.44 | 7.15 | 1.11 | 5.63 | 0.00866 |  | 0.00279 |  |  | 000267 |  |
| 028 | 70 | 6.92 | 6.82 | 7.56 | 1.15 | 5.95 | 0.00831 |  | 0.00298 |  |  | 0.00232 |  |
| 0.30 | 75 | 725 | 721 | 7.93 | 1.19 | 627 | 0.00206 |  | 0.00317 |  |  | 0.00897 |  |
| 0.32 | 80 | 7.71 | 7.54 | 838 | 1.24 | 6.64 | 0.00313 |  |  |  |  |  |  |
| 034 | 85 | 7.96 | 7.90 | 8.85 | 128 | 6.96 | 000329 |  |  |  |  |  |  |
| 0.36 | 90 | 8.32 | 824 | 2.17 | 1.31 | 7.26 | 0.00343 | C- | ERR |  | C1 | S150E-04 |  |
| 0.38 | 95 | 8.55 | 8.51 | 948 | 1.35 | 7.49 | 0.00354 |  |  |  |  |  |  |
| 0.40 | 100 | 8.95 | 882 | 9.90 | 1.39 | 784 | 0.00370 |  |  |  | C2. | 4.74 |  |
| 0.42 | 105 | 926 | 214 | 10.29 | 1.43 | 8.13 | 0.00384 | $\mathrm{n}=$ | 08962 |  |  |  |  |
| 0.44 | 110 | 9.52 | 9.40 | 10.58 | 146 | 8.38 | 0,00395 |  |  |  | C3- | 1.17 |  |
| 0.46 | 115 | 9.75 | 2.75 | 11.00 | 1.49 | 8.67 | 0.00409 |  |  |  |  |  |  |
| 0.48 | 120 | 10.14 | 10.13 | 11.33 | 1.53 | 9.01 | 0.00425 | C.V. | 58579 | (x) | C. $\mathrm{V}_{\text {m }}$ | 14660 | ( $x_{1}$ |
| 0.5 | 125 | 1036 | 1035 | 11.56 | 1.56 | 220 | 0.00434 |  |  |  |  |  |  |

Component Leakage Test: Premium Awning on the Wall: CO-14

| Pressure Dherence Across Crack |  | Flow Prato |  |  |  |  |  | Power Model Predicted Q <br> ( $\mathrm{m}^{2} 3 / \mathrm{sec}$ ) |  | New Model Predicted O <br> ( $\mathrm{m}^{\wedge} 3 / \mathrm{sec}$ ) |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | Rep. 1 | Rep2 | Rep 3 | Chambor Leak (chni) | Corrected Mean$\text { (dme ( } \left.m^{\wedge} 3 \text { sec }\right)$ |  |  |  |  |
| (h.Wg) | (Pa) | (dim) | (chrn) | (ctin) |  |  |  |  |  |
| 0.02 | 5 | 3.41 | 268 | 3.12 | 0.29 | 2.78 | 0.00131 |  | 0.00131 |  | 000107 |
| 0.04 | 10 | 5.41 | 4.89 | 5.25 | 0.41 | 4.77 | 0.00225 | 0.00224 |  | 0.00803 |
| 0.06 | 15 | 730 | 686 | 7.05 | 0.51 | 6.56 | 0.00310 | 0.00305 |  | 0.00291 |
| 0.08 | 20 | 9.05 | 8.37 | 8.59 | 0.60 | 8.07 | 0.00381 | 0.00381 |  | 0.00373 |
| 0.10 | 25 | 10.58 | 998 | 10.00 | 0.67 | 252 | 0.00449 | 0.00452 |  | 0.00449 |
| 0.12 | 30 | 12.14 | 11.25 | 11.64 | - 0.74 | 10.94 | 0.00516 | 0.00520 |  | 0.00521 |
| 0.14 | 35 | 13.56 | 1279 | 13.14 | 0.80 | 1236 | 0.00583 | 0.00586 |  | 0.00589 |
| 0.16 | 40 | 15.10 | 14.01 | 14.20 | 0.86 | 13.58 | 0.00641 | 0.00649 |  | - 0.00654 |
| 0.18 | 45 | 1624 | 15.43 | 15.72 | 0.91 | 14.88 | 0.00702 | 0.00711 |  | 0.00716 |
| 0.20 | 50 | 17.61 | 16.74 | 17.07 | 0.97 | 16.17 | 0.00763 | 0.00771 |  | 0.00776 |
| 0.22 | 55 | 18.98 | 18.07 | 18.10 | 1.01 | 1737 | 0.00820 | 0.00829 |  | 0.00834 |
| 0.24 | 60 | 20.40 | 19.45 | 19.81 | 1.06 | 18.82 | 0.00888 | 0.00887 |  | 0.00890 |
| 026 | 65 | 21.60 | 20.75 | 21.05 | 1.11 | 20.08 | 0.00945 | 0.00943 |  | 0.00944 |
| 028 | 70 | 25.01 | 21.91 | 22.16 | 1.15 | 21.87 | 0.01032 | 0.00998 |  | 0.00996 |
| 0.30 | 75 | 24.41 | 2323 | 23.29 | 1.19 | 2245 | 0.01060 | 0.01053 |  | 0.01047 |
| 0.32 | 80 | 25.74 | 24.31 | 24.55 | 1.24 | 23.63 | 0.01115 |  |  |  |
| 0.34 | 85 | 27.16 | 25.55 | 26.09 | 1.28 | 24.99 | 0.01179 |  |  |  |
| 0.36 | 90 | 28.29 | 26.94 | 27.19 | 1.31 | 26.16 | 0.01235 | C= 3.81E-04 |  | $C 1=2.567 \mathrm{E}-03$ |
| 0.38 | 95 | 29.56 | 28.22 | 28.34 | 1.35 | 27.36 | 0.01291 |  |  | - |
| 0.40 | 100 | 30.91 | 29.33 | 29.68 | 1.39 | 28.58 | 0.01349 |  |  | C2\% 32 |
| 0.42 | 105 | 3223 | 30.56 | 30.92 | 1.43 | 29.81 | 0.01407 | $n=0.7687$ |  |  |
| 0.44 | 110 | 33.41 | 32.00 | 32.17 | 1.46 | 31.06 | 0.01466 |  |  | © - 0.57 |
| 0.46 | 115 | 34.71 | 33.02 | 33.53 | 149 | 3226 | 0.01522 |  |  |  |
| 0.48 | 120 | 36.08 | 34.30 | 34.71 | 1.53 | 33.50 | 0.01581 | C.V $=1.7409$ | (\%) | C.V. $=26847$ |
| 0.5 | 125 | 37.02 | 35.57 | 35.78 | 1.56 | 34.56 | 0.01631 |  |  |  |

Component Leakage Test: Premlum Double Hung on the Wall: CO-15

| Pressure Diference Across Crack <br> (h.Wg) (Pa) |  | Fiow Reto |  |  |  |  |  | Power Model Predicted Q <br> ( $\mathrm{m}^{2} 3 / \mathrm{sec}$ ) |  | $\cdot$ | New Model Predicied 0 <br> ( $\left.\mathrm{m}^{2} 3 / \mathrm{sec}.\right)$ |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | Rep. 1 <br> (dm) | Rep2 <br> (clris) | Rep3 <br> (clme) | Chamber Leak (cime) | Corrected Mean (drif) ( $\mathrm{m}^{1} 3 /$ sec $)$ |  |  |  |  |  |  |
| 0.02 | 5 | 4.71 | 4.87 | 4.67 | 0.29 | 4.46 | 0.00211 |  | 000203 |  |  | 0.00164 |
| 0.04 | 10 | 7.72 | 747 | 7.60 | 0.41 | 7.18 | 0.00339 |  | 0.00343 |  |  | 0.00310 |
| 0.06 | 15 | 10.17 | 1006 | 10.19 | 0.51 | 2.63 | 0.00454 |  | 0.00467 |  |  | 0.00443 |
| 0.08 | 20 | 14.77 | 1243 | 1263 | 0.60 | 1258 | 0.00598 |  | 0.00581 |  |  | 0.00567 |
| 0.10 | 25 | 15.34 | 14.73 | 14.61 | 0.67 | 14.22 | 0.00671 |  | 000688 |  |  | 0.00683 |
| 0.12 | 30 | 17.61 | 17.08 | 1685 | 0.74 | 16.44 | 0.00776 |  | 000790 |  |  | 0.00792 |
| 0.14 | 35 | 19.87 | 1225 | 18.59 | 080 | 18.44 | 0.00870 |  | 000887 |  |  | 0.00895 |
| 0.16 | 40 | 21.99 | 21.58 | 21.06 | 086 | 2069 | 0.00976 |  | 0.00982 |  |  | 0.00994 |
| 0.18 | 45 | 24.12 | 2351 | 2289 | 0.91 | 22.50 | 0.01066 |  | 0.01073 |  |  | 001088 |
| 0.20 | 50 | 26.15 | 25.53 | 24.72 | 0.97 | 24.50 | 0.01156 |  | 0.01163 |  |  | 0.01179 |
| 0.22 | 55 | 28.24 | 27.64 | 2669 | 1.01 | 29.51 | 0.01251 |  | 0.01250 |  |  | 0.01266 |
| 0.24 | 60 | 30.45 | 29.51 | 28.57 | 1.06 | 2845 | 0.01343 |  | 0.01335 |  |  | 0.01351 |
| 0.26 | 65 | 3237 | 31.65 | 30.45 | 1.11 | 3038 | 0.01434 |  | 0.01418 |  |  | 0.01432 |
| 020 | $\cdots$ | 34.46 | 3268 | 3038 | 1.15 | 32.35 | 0.01527 |  | 0.01500 |  |  | 0.01512 |
| 030 | 75 | 36,44 | 3586 | 34.16 | 1.19 | 34.301 | 0.01619 |  | uvisal |  |  | 2atcon |
| 0.32 | 80 | 38.33 | 37.64 | 36.13 | 1.24 | 36.13 | 0.01705 |  |  |  |  |  |
| 0.34 | 85 | 40.52 | 39.68 | 38.13 | 1.28 | 38.20 | 0.01803 |  |  |  |  |  |
| 036 | 90 | 4258 | 41.56 | 39.69 | 1.31 | 39.96 | 0.01886 | C= | ERA |  | C1- | 988E-03 |
| 0.38 | 95 | 44.48 | 43.59 | 41.57 | 1.35 | 41.86 | 0.01976 |  |  |  |  |  |
| 0.40 | 100 | 46.41 | 45.57 | 43.36 | 1.39 | 43.72 | 0.02064 |  |  |  | $\mathbf{C 2}$ | 308 |
| 0.42 | 105 | 48.55 | 47.75 | 4520 | 1.43 | 45.74 | 0.02159 | nm | 0.7577 |  |  |  |
| 0.44 | 110 | 50.87 | 49.81 | 47.05 | 1.46 | 47.78 | 0.02255 |  |  |  | $\boldsymbol{\sim}$ | 0.54 |
| 0.46 | 115 | 5289 | 51.36 | 48.92 | 1.49 | 49.56 | 0.02339 |  |  |  |  |  |
| 0.48 | 120 | 54.88 | 53.87 | 50.91 | 1.53 | 51.08 | 0.02440 | C.V $=$ | 1.8069 | (\%) | C.V. ${ }_{\text {m }}$ | 25906 |
| 0.5 | 125 | 56.77 | 55.75 | 53.37 | 1.56 | 53.74 | 0.02536 |  |  |  |  |  |

Component Leakage Test: Economy Double Hung on the Wall: CO-16

| Pressure DHerence Across Crack |  | Flow Raso |  |  |  |  |  | Power Model Predicted Q |  |  | New Model |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | Rep. 1 | Rep2 | Rep3 | Chambor Leak | Corrected Mean |  |  |  |  |  |  |  |
| ( $\mathrm{l} . \mathrm{W}$ g) | (Pa) | (ctris) | (chrif) | (camb) | (clmi) | (cim) | ( 1 (re3/sec.) | ( $\left.\mathrm{m}^{2} 3 / \mathrm{sec}\right)$ |  |  | ( $\left.\mathrm{m}^{2} 3 / \mathrm{sec}\right)$ |  |  |
| 0.02 | 5 | 2860 | 27.20 | 2987 | 029 | 28.27 | 0.01334 |  | 0.01375 |  |  | 0.01282 |  |
| 0.04 | 10 | 45.56 | 45.49 | 46.35 | 0.41 | 45.38 | 0.02142 |  | 002116 |  |  | 0.02100 |  |
| 0.06 | 15 | 61/42 | 67.00 | 58.60. | 0.51 | 58.79 | 0.02775 |  | 0.02723 |  |  | 0.02751 |  |
| 0.08 | 20 | 71.92 | 6947 | 68.86 | 0.60 | 69.49 | 0.03279 |  | 0.03257 |  |  | 0.03310 |  |
| 0.10 | 25 | 83.22 | 80.00 | 79.46 | 0.67 | 80.23 | 0.03786 |  | 0.03742 |  |  | 0.03806 |  |
| 0.12 | 30 | 8236 | 88.93 | 88.91 | 0.74 | 89.33 | 0.04216 |  | 004191 |  |  | 0.04258 |  |
| 0.14 | 35 | 101.17 | 98.43 | 97.53 | 0.80 | 98.24 | 0.04637 |  | 0.04613 |  |  | 0.04675 |  |
| 0.16 | 40 | 109.33 | 107.15 | 105.93 | 0.86 | 106.81 | 0.05031 |  | 0.05013 |  |  | 0.05064 |  |
| 0.18 | 45 | 118.09 | 114.92 | 113.76 | 0.91 | 114.68 | 0.05412 |  | 0.05394 |  |  | 0.05431 |  |
| 0.20 | 50 | 125.26 | 121.88 | 121.49 | 0.97 | 121.91 | 0.05754 |  | 0.05760 |  |  | 0.05778 |  |
| 0.22 | 55 | 133.09 | 129.17 | 128.13 | 1.01 | 129.11 | 0.06094 |  | 0.06111 |  |  | 0.06109 |  |
| 0.24 | 60 | 140.19 | 136.77 | 135.13 | 1.06 | 136.30 | 0.06433 |  | 0.06451 |  |  | 0.06425 |  |
| 0.26 | 65 | 14639 | 14185 | 141.80 | 1.11 | 14224 | 0.06713 |  | 0.06781 |  |  | 0.06729 |  |
| 0.28 | 70 | 153.85 | 149.46 | 148.23 | 1.15 | 149.36 | 0.07049 |  | 0.07101 |  |  | 0.07022 |  |
| 0.30 | 75 | 160.13 | 155.61 | 155.58 | 1.19 | 155.91 | 0.07356 |  | 0.07412 |  |  | 0.07304 |  |
| 0.32 | 80 | 166.49 | 16209 | 160.94 | 1.24 | 161.94 | 0.07643 |  |  |  |  |  |  |
| 0.34 | 85 | 17289 | 167.82 | 166.43 | 1.28 | 167.77 | 0.07918 |  |  |  |  |  |  |
| 0.36 | 90 | 178.85 | 173.13 | 17283 | 1.31 | 173.62 | 0.08194 | C = | 5.05E-03 |  | C1 $=$ | 463E-03 |  |
| 0.38 | 95 | 184.14 | 179.30 | 178.00 | 1.35 | 179.13 | 0.08454 |  |  |  |  |  |  |
| 0.40 | 100 | 190.27 | 185.06 | 183.53 | 1.39 | 184.90 | 0.08726 |  |  |  | C2- | 1.42 |  |
| 0.42 | 105 | 196.45 | 190.41 | 188.68 | 1.43 | 190.42 | 0.08587 | $\mathrm{n}=$ | 0.0222 |  |  |  |  |
| 0.44 | 110 | 201.80 | 196.12 | 194.66 | 1.46 | 196.07 | 0.09253 |  |  |  | C3- | 1.32 |  |
| 0.46 | 115 | 206.58 | 20146 | 199.56 | 1.49 | 201.03 | 009488 |  |  |  |  |  |  |
| 0.48 | 120 | 212.56 | 207.13 | 205.26 | 1.53 | 206.79 | 0.09759 | C.V.- | 0.8210 | (\%) | C.V. $=$ | 0.7252 | (\%) |
| 0.5 | 125 | 217.88 | 213.07 | 210.16 | 1.56 | 21217 | 0.10013 |  |  |  |  |  |  |

## Component Leakage Test: Premium Casement on the Wall: CO-17



Component Leakage Test: Economy Casement on the Wall: CO-18

| Pressure DHerence Across Crack |  | Flow Plete |  |  |  |  |  | Power Model Predicted $a$ <br> ( $\mathrm{m}^{\wedge} 3 / \mathrm{sec}$ ) |  | New Model <br> Predicted Q <br> ( $\mathrm{m}^{2} 3 / \mathrm{sec}$.) |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | Rep. 1 | Rep 2 | Rep 3 | Chamber Leak (din) | Corrected Mean <br> (dimi) $\left(\mathrm{m}^{2} 3 / \mathrm{sec}\right.$ ) |  |  |  |  |  |  |
| ( $\mathrm{n} . \mathrm{Wg}$ ) | (Pa) | (chm) | (ctar) | (cmin) |  |  |  |  |  |  |  |
| 0.02 | 5 | 9.15 | 9.09 | 9.03 | 0.29 | 8.81 | 0.00416 |  | 0.00383 |  |  | 0.00279 |  |
| 0.04 | 10 | 14.28 | 14.53 | 14.45 | 0.41 | 14.01 | 0.00661 | 0.00654 |  |  | 0.00543 |  |
| 0.06 | 15 | 18.62 | 18.97 | 19.36 | - 0.51 | 18.47 | 0.00872 | 0.00896 |  |  | 0.00793 |  |
| 0.08 | 20 | 22.93 | 23.35 | 23.42 | 0.60 | 2264 | 0.01068 | 0.01120 |  |  | 0.01033 |  |
| 0.10 | 25 | 27.21 | 27.59 | 27.54 | 0.67 | 26.77 | 0.01264 | 0.01331 |  |  | 0.01262 |  |
| 0.12 | 30 | 31.33 | 31.44 | 31.77 | 0.74 | 30.77 | 0.01452 | 0.01533 |  |  | 0.01483 |  |
| 0.14 | 35 | 35.49 | 35.73 | 36.07 | 0.80 | 34.97 | 0.01650 | 0.01727 |  |  | 0.01696 |  |
| 0.16 | 40 | 39.51 | 40.04 | 40.41 | 0.86 | 39.13 | 0.01847 | 0.01915 |  |  | 0.01901 |  |
| 0.18 | 45 | 44.03 | 44.48 | 44.96 | 0.91 | 43.58 | 0.02057 | 0.02098 |  |  | 0.02101 |  |
| 0.20 | 50 | 48.33 | 49.03 | 49.42 | 0.97 | 47.96 | 0.02264 | 0.02276 |  |  | 0.02295 |  |
| 0.22 | 55 | 52.93 | 53.12 | 53.46 | 1.01 | 5215 | 0.02461 | 0.02451 |  |  | 0.02483 |  |
| 0.24 | 60 | 57.50 | 57.50 | 57.87 | 1.06 | 56.56 | 0.02670 | 0.08622 |  |  | 0.02666 |  |
| - 0226 | 65 | 61.75 | 6227 | 6239 | 1.11 | 61.03 | 0.02880 | 0.02789 |  |  | 0.02845 |  |
| 0.28 | 70 | 66.24 | 67.61 | 66.77 | 1.15 | 65.72 | 0.03102 | 0.02954 |  |  | 0.03020 |  |
| 0.30 | 75 | 71.16 | 71.65 | 71.61 | 1.19 | 70.28 | 0.03317 | 0.03116 |  |  | 0.03190 |  |
| 0.32 | 80 | 75.84 | 76.74 | 76.22 | 1.24 | 75.03 | 0.03541 |  |  |  |  |  |
| 0.34 | 85 | 80.44 | 81.35 | 81.05 | 1.28 | 79.67 | 0.03760 |  |  |  |  |  |
| 0.36 | 90 | 84.23 | 6638 | 85.92 | 1.31 | 84.19 | 0.03974 | C. $1.10 \mathrm{E}-03$ | - | $C 1$ - 14157E-02 |  |  |
| 0.38 | 96 | 87.24 | 90.76 | 90.45 | 1.35 | 88.13 | 0.04159 |  |  |  |  |  |
| 0.40 | 100 | 90.67 | 95.81 | 95.42 | 1.39 | 9258 | 0.04369 |  |  | C2- | 3.2 |  |
| 0.42 | 105 | 93.94 | 99.83 | 99.45 | 1.43 | 96.31 | 0.04546 | $n=0.7745$ |  |  |  |  |
| 144 | 110 | 97.16 | 103.37 | 103.51 | 1.46 | 99.89 | 0.04714 |  |  | $\boldsymbol{\sim}=$ | 0.26 |  |
| 46 | 115 | 101.41 | 106.40 | 106.63 | 1.49 | 103.32 | 0.04876 |  |  |  |  |  |
| 8 | 120 | 10533 | 109.80 | 109.69 | 1.53 | $106.76$ | $0.05039$ | C.V $=4.7133$ | (\%) | C.V.- | 4.0332 | (\%) |
| ; | 125 | 111.74 | 113.70 | 113.71 | 1.56 | 11149 | 0.05262 |  |  |  |  |  |

