

THERMAL PERFORMANCE EVALUATION OF PASSIVE AND HYBRID SYSTEMS USING A LOW COST METHODOLOGY

S. ZABOT

Regione Lombardia, Servizio Energia  
Via F. Filzi 22, 20100 Milano - Italy

A. V. BHARGAVA (\*), S. FUMAGALLI and L. SOHA  
ENEA-FARE Engineering Division  
21020 Ispra (Varese) - Italy

S. MAHAJAN (\*\*) and F. CONTI  
Commission of European Communities  
JRC Ispra, 21020 Ispra (VA) - Italy

- (\*) - On study leave from Ramjas College, Delhi University, India. Under ENEA and ICTP Research and Training Program in Italian Laboratories.  
(\*\*) - On Sabbatical leave from California State University, Sacramento, California, USA.

### 1. INTRODUCTION

In 1980 the Region of Lombardy, School Building Service, started a pilot program for energy conservation and solar energy applications in educational buildings. During the period 1981-1984 preliminary energy audits were carried out, funds were allocated and several energy saving measures were implemented.

In order to assess the cost-effectiveness as well as technical performances it was decided that field monitoring should be performed on these buildings.

As a first step a suitable methodology was developed. The main aim of this methodology is to answer the question:

how well do the retrofit measure perform ?

No attempt is made to understand the details of the performance mechanism. The cost and ease of implementation of the methodology were important considerations for the final choice.

In the fall 1984 the methodology was set-up to monitor two school buildings. The Montorfano School and the Lanzada School were instrumented as case studies in the late fall of 1984 and they are presented here.

### 2. SITE AND BUILDINGS DESCRIPTION

The Montorfano Elementary School (Fig. 1A) is located about 5 Km. south east of the city of Como. The microclimate of this site feels the effect of the region around the lake of Como with moderately cold winters and warm summer.

The school building is made with concrete frames. Two class room wings are built on a gently sloping hill side and are partly below ground level on the corridor side (Fig. 1B). The section connecting the two wings is two storey. The ground floor has a custodian's apartment and school offices. The first floor contains the gym.

- The major retrofit measures were:
- increased envelope insulation,
  - double glazing of the classroom windows,
  - installation of a ventilation system with a solar air heater
  - improvement of regulation system

The Lanzada Elementary School (Fig. 2A) is located 20 Km. north of the city of Sondrio in Regione Lombardia. It is at the end of Valmalenco (a side valley of Valtellina), below the Bernina Range - in the Alps.

The microclimate is typical of the high alpine belt: snowy winter and spring with sunny days and cool nights. Summers are moderately warm.

The school is a masonry building made with massive stones, internally covered with bricks (average thickness 70 cm.). The school has a compact shape with all the classrooms facing south. It is a three storey building with a small gym at the lowest level (Fig. 2B).

- The major retrofit measures were:
- Increased envelope insulation on North side.
  - Double glazing of all windows on the North side.
  - Insulation of the loft.
  - Installation of solar air heaters on South facade connected with a ventilation system.

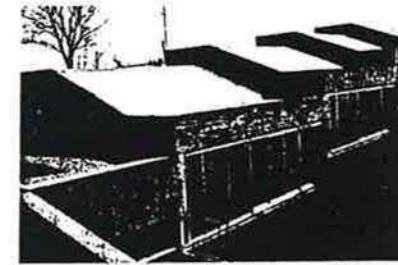


Figure 1 B

SECTION

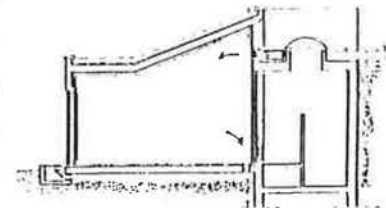


Figure 2 A

LANZADA SCHOOL  
FACADE AFTER RETROFIT

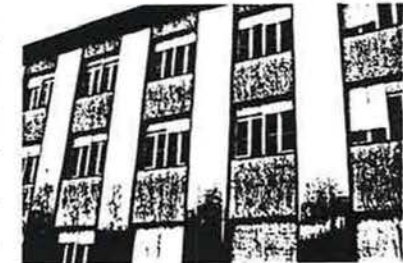
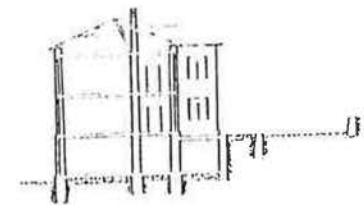


Figure 2 B

SECTION



### 3. METHODOLOGY AND DATA COLLECTION

The chosen methodology is aimed to determine the passive gains and to evaluate the thermal performance of various building subsystems (conventional boiler, solar air heater, etc). The conduction heat loss coefficient (UA) and the infiltration rate were obtained respectively by means of co-heating and tracer gas measurement techniques. Both of these are standard techniques and are briefly described later in this section.

All energy sources, such as the heat supplied from auxiliary heater and solar air heaters and internal gains were monitored. Using these monitored data, energy balance for the whole building is performed. The following equation is used to determine passive solar gains:

$$Q_{\text{passive}} = Q_{\text{loss}} - Q_{\text{auxiliary}} - Q_{\text{internal}} - Q_{\text{collector}} \quad (1)$$

$Q_{\text{loss}}$  is made up of three parts:

$$Q_{\text{loss}} = Q_{\text{conduction}} + Q_{\text{infiltration}} + Q_{\text{ventilation}} \quad (2)$$

where  $Q$ 's are the various net heat losses or gains through the building envelope. The measured indoor-outdoor temperature difference, total heat loss coefficient (UA) and infiltration rate are used to calculate  $Q_{\text{loss}}$  assuming steady state conditions.

For short time intervals (less than a few days) this assumption is not valid, since the time constants of the buildings are of the order of few days. However, for periods greater than a week the thermal mass effects are small, and fully negligible for seasonal energy balance.

The heat delivered to the building is composed by: (i) heat given off by the hot water circulating through the convectors, (ii) the heat supplied by the forced hot air, which is preheated by the solar air heater and further heated by passing over a hot water radiator (only for the Montorfano system) and (iii) the passive gain through glazings and opaque surfaces.

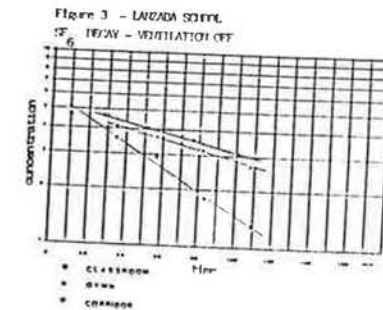
The measured values of inlet-outlet temperature differences and the flow rates of hot water and forced air at various points along the flow paths, are used to determine the heat delivered by the hot water convectors and the solar air collectors. This information, when used in equation (1), yields both opaque as well as transparent passive contributions.

The data collection was limited to a level adequate for carrying out the performance evaluation using the methodology outlined above. This includes the measurements of temperatures, insolation, auxiliary heating, hot water flow rates, mixing fan on/off time, humidity, wind speed etc. All probes were calibrated in the laboratory before field installation.

The Solartron Schlumberger ORION is used for data logging. It allows some conversion on probes, counters unit and measures of events. The data were recorded on a cassette tape and later transferred on to the VAX 11/750 system via Modems and Telephone Line.

#### Tracer gas measurements.

The tracer gas measurements were done at different places in each building: class rooms, corridors and gymnasium. The measurements were made under natural as well as forced ventilation. The gas (SF<sub>6</sub>) was injected and allowed to mix for approximately half an hour. Four to five samples were taken from each space at successive intervals of 15 minutes. The concentration of SF<sub>6</sub> was then measured using gas chromatography. From the SF<sub>6</sub> concentration decay rates, the air exchange rates were obtained. Figure 3 shows gas decay at Lanzada school in three different building zones.



#### Coheating measurements.

The main purpose behind coheating technique was to check the calculated heat loss coefficients with field measured values. Due to several technical problems, the standard coheating method using electrical heating could not be carried out for the whole buildings. It was therefore decided to perform tests on single rooms and to use the results for evaluating the conduction loss coefficient for other part of the buildings. The following procedure was used: electrical resistance heaters replaced normal auxiliary system in the zone under test. Corridors and adjacent rooms were kept at constant temperature by using the regular heating systems. From the information on temperatures, electrical auxiliary heating supplied and the SF<sub>6</sub> decay rate during the test, the heat loss coefficient could be calculated. Since the buildings have large thermal mass, special care was taken to ensure that the indoor temperatures were constant. Furthermore the experiments were conducted during a night proceeded by many overcast days. It was found that this brings to values 30-40% lower than the values calculated according to Italian Standard UNI-CTI procedures. (see Table I)

TABLE I - UA Values (W/C)

	(UNI-CTI)		Measured After
	Before	After	
Montorfano Classroom	313	187	130
Montorfano Whole Bldg.	6054	5044	3430
Lanzada Whole Bldg.	2311	1744	1145

TABLE II - Infiltration Rates (ACH)

	Occupied Fan ON	Unoccupied Fan OFF
	MONTORFANO	
Classroom	2.78	0.50
Corridor	2.28	0.50
Gymn	1.96	0.50
LANZADA		
Fan OFF		
Classroom	0.60	0.41
Corridor	2.00	1.86

#### 4. DATA ANALYSIS

The main objective of the data analysis was to determine the total heat losses, auxiliary energy used, internal and solar gains. The solar energy contributions consist of two parts: the energy provided by the solar air collectors (active part) and the passive solar gains.

Total Heat Losses. The heat losses through the building, due to conduction, infiltration and ventilation, were calculated assuming steady state conditions. As mentioned earlier, this assumption is valid for periods longer than the time constant of the building. The conduction heat losses were calculated using the value of conduction loss coefficient (UA) as determined above and the measured indoor-outdoor temperature difference (DT).

$$Q_{\text{conduction}} = \sum_j (UA)_j \cdot DT_j \cdot Dt \quad (3)$$

The infiltration heat losses were calculated using the infiltration rate determined from tracer gas measurements and indoor-outdoor temperature difference.

$$Q_{\text{infiltration}} = \sum_j (V_j \cdot R_j \cdot r_a \cdot C_a \cdot DT_j \cdot Dt) \quad (4)$$

The Ventilation Losses were calculated using the ventilation rate and indoor-outdoor temperature difference. An equation very similar to Equation (4) is used to calculate ventilation rate for the fan-on periods.

Auxiliary energy is provided by hot water radiators. The energy supplied by the hot water was calculated from the information on inlet-outlet water temperature differences and water flow rates, and the on-time of the circulating pump.

$$Q_{\text{auxiliary}} = F \cdot r_w \cdot C_w \cdot DT \cdot Dt \quad (5)$$

were:  $V_j$  = Volume of the  $j$ th space  
 $R_j$  = Air Infiltration Rate for the  $j$ th space  
 $DT_j$  = Temperature difference between the  $j$ th space and outdoor  
 $r_a/r_w$  = Density of air/water  
 $C_a/C_w$  = Specific Heat Capacity of air/water  
 $Dt$  = Durations  
 $F$  = Water Flow Rate

Internal Gains: the information on the number of occupants, time of occupancy and the lighting level was used to evaluate internal gains.  
 Boiler Efficiency: the fuel flow rate and the burner on-time gives the total oil burned and finally the oil burned along with the auxiliary useful energy delivered, gives the heating system efficiency.  
 Solar contributions: the measured values of the air flow rate through the collector, the inlet-outlet air temperature difference and the time for which the fan is on, were used to calculate solar active contributions supplied by the solar air collectors. An equation similar to Eq. (5) is used for this calculation. Finally the passive solar contributions are calculated subtractively using Equation (1). However any errors made in the calculation of  $Q_{\text{loss}}$  and  $Q_{\text{aux}}$  are lumped into the passive contributions.

#### 5. RESULTS AND DISCUSSION

##### Montorfano Elementary School

The heat balance for the building was performed in three stages. In the first stage only the classroom heat balances were analyzed (see Table III A). The class rooms have large glazings and hence collect a considerable amount of solar energy. The value of 28% (20% passive; 8% active) for the 10 day period looks good.

The corridors which have strong thermal coupling with rooms receive very little direct solar energy. As expected the solar contributions drop to 15% (11% passive, 4% active) for 10 day period (see Table III B).

For the whole building the solar contribution is 18% (15% passive and 3% active). The slight increase in solar gains over that for the rooms and corridor case is due to the fact that the main building has considerable south facing glazing (see Table III B).

The heating system efficiency varied between 50.1% to 58.7%. The boiler was sized for the heating requirements before the retrofit measures. Since the retrofit measures reduced the heat load considerably, the boiler operates more intermittently, thus lowering the efficiency.

The highest hourly efficiency for the solar air collector system is 50.7% and the average for the 10 days period presented here is 34.6%. These values are rather low and the reason is due to the fact that the glazing of some collectors were found defective. For clear sky days, the solar air heater system is nearly 40% efficient with instantaneous value as high as 65.7%.

TABLE III A  
CLASSROOM - MARCH '85

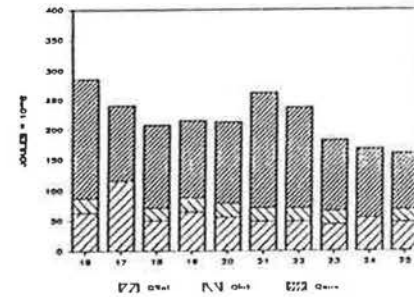


TABLE III B  
CLASSROOMS & CORRIDORS

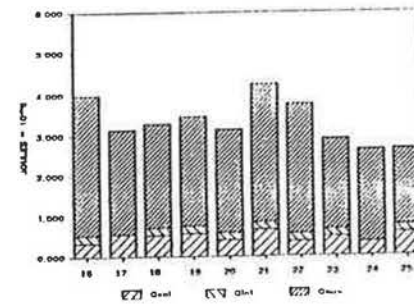
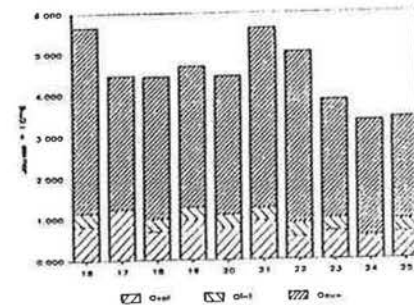


TABLE III C  
WHOLE BUILDING



To understand the performance of the solar collector, the heat balance for one day is presented in Table IV.

The solar contributions to the heating energy requirements are quite reasonable although, for building licensing constraints, only 42.5 m<sup>2</sup> of solar air collector were installed instead of the 130 m<sup>2</sup> originally planned.

#### Lanzada Elementary School

Two 12 day periods were analyzed. The two periods are representative of typical spring and autumn climate.

During the twelve day period in autumn (November) six days were sunny and six days were overcast. In March, eight days were sunny and four days were overcast. The average ambient temperature is nearly the same during these periods.

The bar chart in Table V shows the hourly auxiliary delivered to the building and the solar energy incident on collector plane for a period of three sunny days.

The heat balance calculations were performed for the whole building. The results for 12 days period in March and November 1985 are presented (Table VI and VII).

The daily energy balance has little meaning, because of different shifts in the various contributions due to the transient effects through the thick walls, but the results averaged over twelve day periods are nearly free of thermal mass effects.

The building collects a considerable amount of solar energy: the south facade has 84 m<sup>2</sup> of direct gain aperture (windows) and 130 m<sup>2</sup> of collector area out

TABLE IV  
MONTORFANO MARCH 20, 1985  
SOLAR COLLECTOR HEAT BALANCE

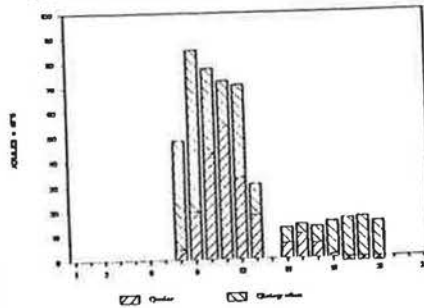


TABLE V  
LANZADA 14-15-15- NOVEMBER '85

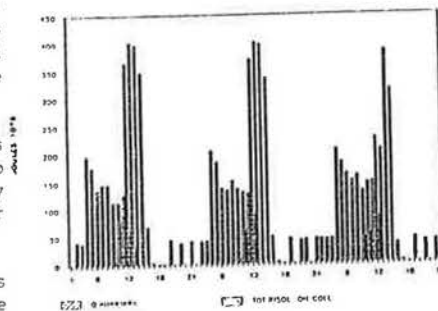
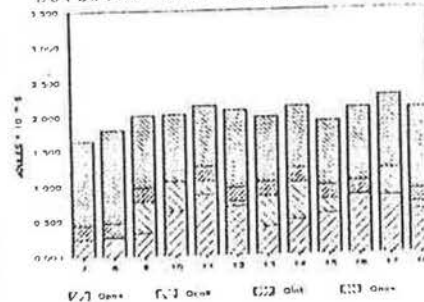


TABLE VI  
LANZADA MARCH '85  
BUILDING HEAT BALANCE



of a total of 380 m<sup>2</sup> of the south facade area.

The collector efficiency in the two periods doesn't exceed 30% and 20% respectively. This low efficiency was mainly due to incorrect positioning of the fan in the air circulation loop (at the inlet to the collector instead of the outlet) which provoked excessive air leakages in the region of the collectors. The measured air flowrate is 1200 m<sup>3</sup>/h leaving the collectors versus 3200 m<sup>3</sup>/h of air forced into the collectors and pushed through.

The total solar contribution, with respect to the total load, is 30% in November and 39% in March; respectively 25% passive, 5% active and 27% passive, 12% active. The results are good and the collector contribution will increase if the existent ventilation unit is moved to the attic as suggested.

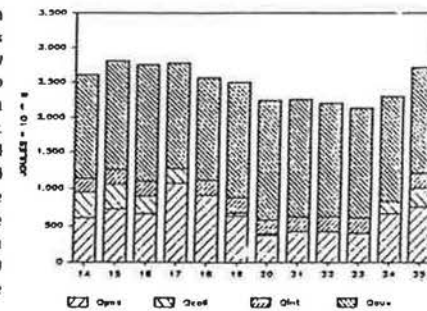
#### 6. CONCLUSIONS

The results show how trivial errors in installing solar components can lead to lower energy performance and considerable waste of energy. To date, many solar systems have been installed in Italy. However, due to lack of detailed knowledge of solar systems and to underestimation of increased complexity of hybrid systems, the effect of these systems often does not correspond to design expectations. These constructive deficiencies are very dangerous, because, by giving a negative demonstration, they reduce the potential uptake of correctly installed systems. The defects noted show an impelling need for the training of qualified personnel in design, installation and management in hybrid solar systems.

#### REFERENCES:

- (1) Palmiter L, Hamilton, Holtz M. "Low Cost Performance Evaluation of Passive Solar Heating and Cooling". SERI, report RR-63-223. October 1979.
- (2) Mahajan S, Shea M, Newcomb C and Mort D. "Performance of Passive Solar and Energy Concerning Houses in California". SERI report STR-254-2017. November 1983.
- (3) Shea M, Mort D, Mahajan S and Newcomb C. "Documentation of Class B Data Analysis". SERI report STR-254-2055. September 1983.
- (4) Zabot S, Mahajan S, Bhargava A K, Soma L, Fumagalli S and Conti F. "A Low Cost Methodology for Thermal Performance Monitoring of Public Schools in the Regione Lombardia". EUR 10905 Ispra - Nov. 86

TABLE VII - LANZADA - NOV. '85  
BUILDING HEAT BALANCE



ERDBEDECKTES BUEROHAUS MIT HYBRIDEM SONNENENERGIESYSTEM,  
KONVEKTIVER CHAMAELEON-FASSADE UND GERÖLLSPEICHER

KURT HAAS

Haas + Partner AG, Ingenieure + Planer  
CH-8645 Jona

1. Konzept

Der längliche Bürotrakt von 10/20 m nimmt die ursprüngliche Hügelkuppe links ab und ersetzt diese in künstlicher, erdbedeckter Bauweise. Das ganze Gelände steht dem Wohnhaus als Grünfläche wieder zur Verfügung. Die konsequente Ausbildung einer Temperaturschichtung von Süden nach Norden, in Form von Pufferzonen, reduziert die Wärmeverluste. Das Fensterband ist mit 3.20 m im Süden weit geöffnet. Dies leuchtet am 22. Dez., mittags, den Arbeitsraum in seiner ganzen Tiefe aus und macht so aus der "erdbedeckten Höhle" einen lichtdurchfluteten Arbeitsraum. Mit dem Luft- und Fensterkollektor wird bei Sonneneinstrahlung warme Luft erzeugt, die durch einen Ventilator in den Geröllspeicher geblasen wird. Dieser erwärmt sich und gibt die Wärme passiv, durch eine Zwischenisolation verzögert, an den Raum ab. Die Arbeitsräume sind von diesem geschlossenen Wärmeluftsystem unberührt und werden individuell durch Fensterflügel belüftet. Freie Luftzirkulation zwischen Büro und Reduit sowie Oberflächenmaterialien mit hoher Wärmeleitfähigkeit führen die eingestrahlte Wärme in die Gebäudemasse ab und sorgen so für eine hohe thermische Trägheit und somit für ein ausgeglichenes Raumklima. Zur Erweiterung des Strahlungsangebotes, ist vor der Fassade ein weisses Band aus Steinplatten als Reflektor angelegt.

Bild 1. Gesamtansicht von Süden; links erdbedeckter Bürobau, rechts Wohnhaus



Bild 2. Dachaufsicht erdbedeckt, bepflanzt. Die Spielwiese ist vollständig begrünt und mit einer Randbepflanzung versehen



gebotes, ist vor der Fassade ein weisses Band aus Steinplatten als Reflektor angelegt.

Das Fensterband ist mit 3.20 m im Süden weit geöffnet. Dies leuchtet am 22. Dez., mittags, den Arbeitsraum in seiner ganzen Tiefe aus und macht so aus der "erdbedeckten Höhle" einen lichtdurchfluteten Arbeitsraum. Mit dem Luft- und Fensterkollektor wird bei Sonneneinstrahlung warme Luft erzeugt, die durch einen Ventilator in den Geröllspeicher geblasen wird. Dieser erwärmt sich und gibt die Wärme passiv, durch eine Zwischenisolation verzögert, an den Raum ab. Die Arbeitsräume sind von diesem geschlossenen Wärmeluftsystem unberührt und werden individuell durch Fensterflügel belüftet. Freie Luftzirkulation zwischen Büro und Reduit sowie Oberflächenmaterialien mit hoher Wärmeleitfähigkeit führen die eingestrahlte Wärme in die Gebäudemasse ab und sorgen so für eine hohe thermische Trägheit und somit für ein ausgeglichenes Raumklima. Zur Erweiterung des Strahlungsangebotes, ist vor der Fassade ein weisses Band aus Steinplatten als Reflektor angelegt.

2. Gebäudehülle, Erdbedeckung

Der Gebäudehülle kommt bei jedem energiebewusst konzipierten Gebäude grosse Bedeutung zu. Dies gilt natürlich umso mehr bei erdbedeckter Bauweise. Der Dachaufbau ist wie folgt ausgeführt:

18 - 24 cm	Beton
	Dampfsperre
2 x 60 mm	PU
1.2 mm	Kunststoff-Folie
60 mm	Polystyrol-Sickerplatten
200 g/m <sup>2</sup>	Filtermatte
30 - 50 cm	Humus

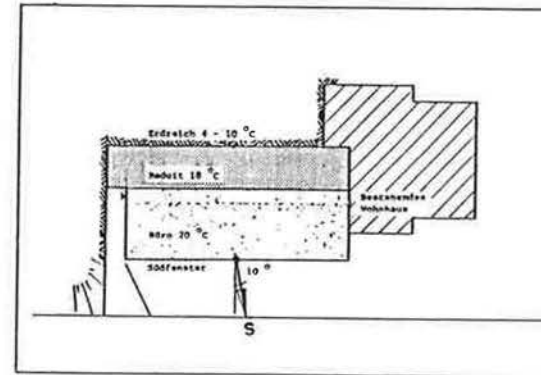


Bild 3. Pufferzonen. Die Temperatur nimmt von Süden nach Norden ab

Zudem wird durch die Erdbedeckung ein erheblicher, oekologischer Beitrag gegen die Verbauung unserer Landschaft geleistet.

Der Wand- und Bodenaufbau ist analog. Die Isolation ist im Bereich Boden/Wand in einer Stärke von 200 mm durchgezogen. Nur statisch notwendige Punktaufleger durchdringen diese.

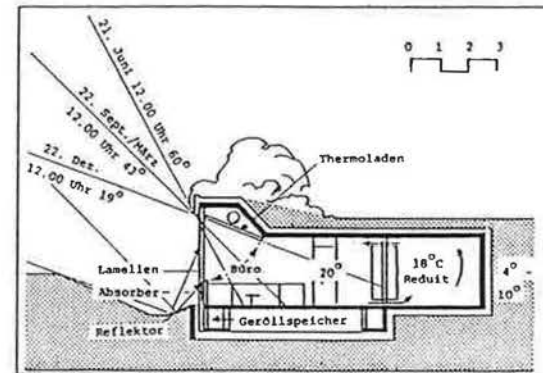
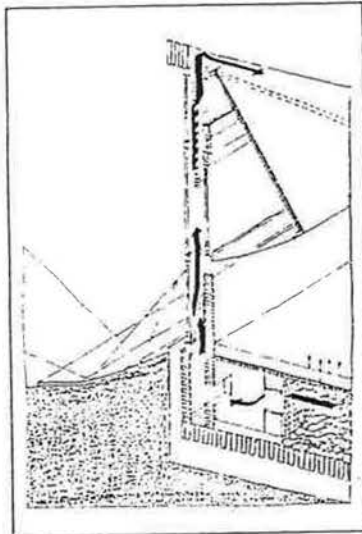


Bild 4. Schnitt durch das Gebäude

3. Kollektorfassade, Geröllspeicher



Die Kollektorfassade besteht vertikal aus drei verglasten Teilen: der Brüstung, dem Sichtteil und dem Oblicht. Im Querschnitt von aussen nach innen folgen: Stahlfenster, thermisch getrennt, Isolierglas 2-fach; Luftzwischenraum 15 - 20 cm; Holzfenster, Isolierglas 2-fach, resp. Brüstung als Sandwich aus Eternit, weiss, und 100 mm PU.

Die Fassade kann - einem Chamäleon gleich - der thermischen Situation entsprechend, farblich verändert werden. In der Speicherperiode (21. Aug. bis 13. Apr.) werden im Brüstungsbereich schwarze Absorber aus Steckmetall eingesetzt; im Oblicht- und Sichtbereich stehen die Lamellenstoren - mit der braunen Seite nach aussen - unter 45°. Die Fassade ist dunkel, auf Absorption eingestellt. Im Sommer werden die Absorber aus dem Brüstungsbereich entfernt, der weisse Eternit wird sichtbar; im Oblicht- und Sichtbereich stehen die Lamellenstoren mit der weissen Rückseite zur Sonne. Die Fassade

Bild 5. Perspektive Kollektorfassade/Geröllspeicher



Bild 6. Die Chamäleonfassade dunkel, auf Absorption eingestellt

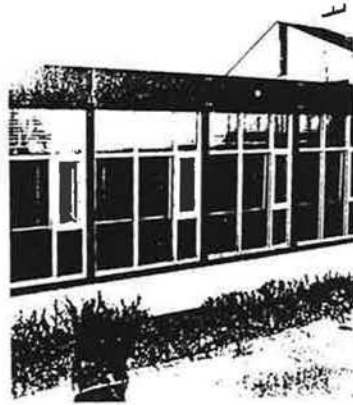


Bild 7. Die Chamäleonfassade weiss, auf Reflexion eingestellt

ist weiss, auf Reflexion eingestellt. Grosse Thermo-Klappläden, die am Tag einen Teil der Decke bilden, werden im Winter in der Nacht und übers Wochenende von Hand heruntergeklappt und reduzieren so den Bruttowärmebedarf um 28%. Ein zweistufiger Ventilator wälzt die Luft temperaturgesteuert um. Thermo-siphonzirkulation in der Nacht kann nicht ausgeschlossen werden.

Bild 8. Thermoläden beim Herunterklappen

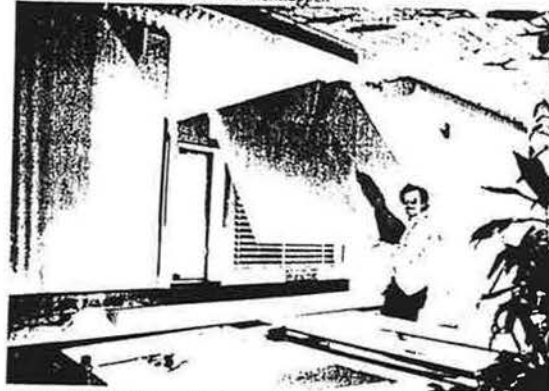
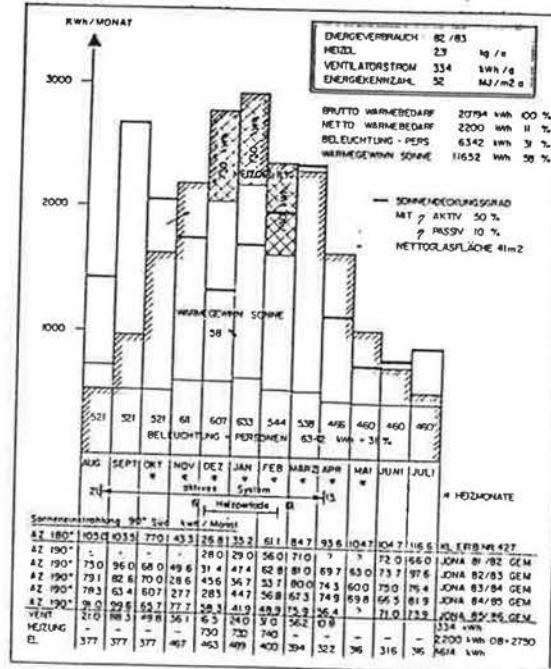


Bild 9. Energiebilanz 1982/83, Sonneneinstrahlungswerte AZ 190° vertikal, Jona gemessen



4. Energiebilanz 82/83, Energiekennzahl

Durch die konsequente Isolation beträgt der Bruttoenergiebedarf noch 94.8 kW/m<sup>2</sup>a. Dieser wird zu 31 % durch Beleuchtung und Personen (Strom gemessen) und zu 11 % als Nettowärmebedarf durch die Ölheizung des Wohnhauses gedeckt (gemessen). Die Wärmeverteilung erfolgt durch Niedertemperaturheizkörper mit Thermostatventilen. Die Heizsaison dauert vom 6. Dez. bis 19. Febr. Es wird nur an Wochentagen - vor und während der Arbeitszeit - geheizt. Der Restwärmebedarf (ca. 58 %) wird durch die Sonne gedeckt. Ein Solarimeter misst die momentane, tägliche und monatliche Einstrahlung auf die vertikale Fassadenfläche. Der Stromverbrauch für den Ventilator beträgt 1-1.5 % der Bruttoeinstrahlung.

Die Heizenergiekennzahl beträgt 52 MJ/m<sup>2</sup>.a und liegt, verglichen mit konventionellen Bauten, bei ca. 10 %.

## 5. Behaglichkeit, Gebädeträgheit

In Diagrammen wurden die gemessenen Temperaturwerte als Wochenminima und -maxima aufgetragen. Die Raumtemperatur (Bild 11) erreicht in den Sonnenspitzenmonaten das Maximum am Nachmittag bei  $+29^{\circ}\text{C}$ . Durch Nachlüftung sind dann die Temperaturen am Morgen wieder auf  $24-25^{\circ}\text{C}$ . Die grössten Schwankungen pro Woche sind in den Monaten Dezember und Januar zu verzeichnen, da bei diesem kleinen Sonnenangebot - auch bei einem strahlenden Tag - die Storen kaum gezogen werden, sodass volle passive Sonnenenergienutzung vorliegt. In der Periode vom 16. Dezember bis 5. Januar wurde das Gebäude während 18 Tagen ohne Fremdenergie sich selbst überlassen. Die Raumtemperatur betrug am 5. Januar noch  $+17^{\circ}\text{C}$ .

Die Speichertemperatur (Bild 12) hat Ende August ca. Raumtemperatur angenommen und steigt dann bei Speicherbeginn, durch das grosse Sonnenangebot im September, um ca.  $+3^{\circ}\text{C}/\text{Woche}$ . Aus den Kurven ist zu entnehmen, dass

nicht ein Wochen-/Monats- oder Saisonspeicher definiert werden kann. Grob gesagt, lässt sich die Septemberwärme in den November "retten". Dies sind immerhin 2 Monate.

Die Bodentemperatur (Bild 10) weist wesentlich kleinere Schwankungen auf als die Speicher- oder Raumtemperatur. Die Klinkerplatten wirken sehr temperaturausgleichend.

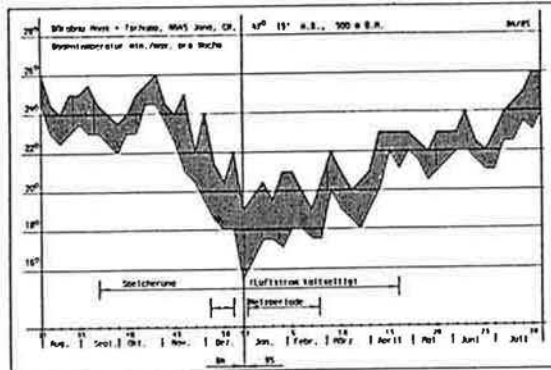


Bild 10. Bodentemperatur, min./max. pro Woche

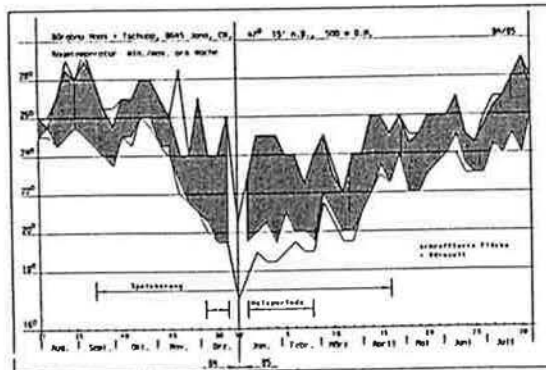


Bild 11. Raumtemperaturen min./max. pro Woche (schrattierte Fläche = Bürozeit)

Bild 12. Speichertemperatur min./max. pro Woche

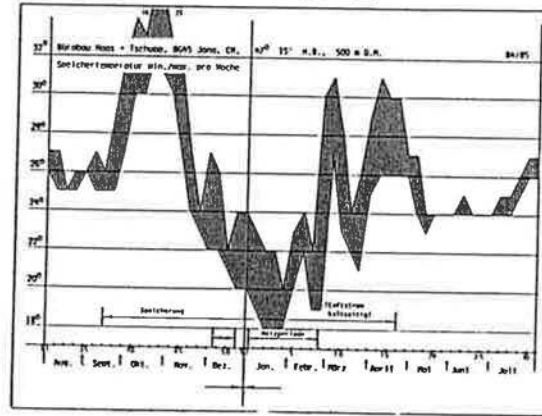


Bild 13. Aussentemperatur min./max. pro Woche

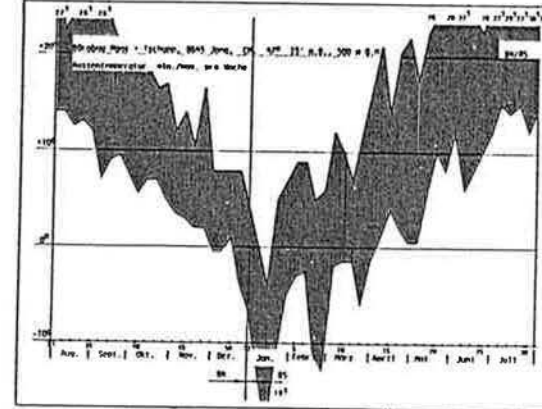


Tabelle 1. Klima und Gebäudedaten

Klima:	
geogr. Breite	$47^{\circ} 15'$
Höhe über Meer	500 m
Fassadenorientierung	AZ $190^{\circ}$
Heizgradtage (ZHSMA)	3718 HGT (Jahr)
Gebäudeabmessungen:	
Energiebezugsfläche	$213 \text{ m}^2$
$\text{m}^3$ SIA mit Speicher	$1180 \text{ m}^3$
$\text{m}^3$ SIA ohne Speicher	$1075 \text{ m}^3$
Kollektorflächen netto	$41 \text{ m}^2$
Geröllspeicher	$60 \text{ m}^3$
Thermische Gebäudemasse	$C = 80 \text{ kWh/K}$

## 6. Kosten, Wirtschaftlichkeit

Vergleicht man die Jahresgesamtkosten (Kapitalkosten + Heizkosten) pro m<sup>3</sup> für den Bürobau H + P mit einem konventionellen Bau, setzt den konventionellen Bau 10 % billiger, die Energiekosten aber 10 x höher ein, so ergibt sich (Zins 6 %, 50 J. a = 6.34 %, Heizöl Fr. 80.--/100 kg, Geschosshöhe 3.0 m):

Bürobau	H + P	konventionell
m <sup>3</sup> -Preis	Fr. 347.--	Fr. 312.--
EH MJ/m <sup>2</sup> a	52	520
Kap. + Amort./m <sup>3</sup> .a	22.43	19.78
Energiekosten/m <sup>3</sup> .a	0.33	3.25
Gesamtkosten/m <sup>3</sup> .a	Fr. 22.76	Fr. 23.03
	98.8 %	100 %

Das Gebäude ist also auch gegenüber einem günstigeren Bürobau mit einem relativ kleinen Energieverbrauch wirtschaftlich. Rechnet man jedoch das gleiche Gebäude mit und ohne aktive Sonnenenergieanlage, so lassen sich die Fr. 86'000.-- Mehrkosten kaum amortisieren.

Tabelle 2  
K-Werte, Brutto-Wärmebedarf, Energiekennzahl

Decke	12 cm PU,	0.210 W/m <sup>2</sup> K
Wand	12 cm PU,	0.230 W/m <sup>2</sup> K
Boden	20 cm PU,	0.170 W/m <sup>2</sup> K
Fenster:	Tag	1.600 W/m <sup>2</sup> K
	Nacht	0.350 W/m <sup>2</sup> K
Tag	Betrieb	QH = 6400 W (n = 0.30)
Nacht		QH = 4600 W (n = 0.15)
Energiekennzahl:		
EH	Öl (η = 80%)	= 46 MJ/m <sup>2</sup> a
EH	Ventilator	= 6 MJ/m <sup>2</sup> a
EH	= Heizenergiekennzahl	= 52 MJ/m <sup>2</sup> a
EEL	= Energiekennzahl Strom	= 70 MJ/m <sup>2</sup> a
E	= Total	= 122 MJ/m <sup>2</sup> a

## 7. Probleme

Die Isolation über dem Geröllkoffer ist zu dünn, es liesse sich länger und höher speichern (ev. ohne Fremdenergie). Der sommerliche Wärmeschutz kann noch verbessert werden. Im Winter setzt sich Kondenswasser im Luftkollektor an (ästhetisch), das aber problemlos abgeleitet werden kann. Das Betreiben der Anlage ist technisch einfach (nur ein Ventilator), bedingt aber gute Wetterbeobachtung und ein Gefühl für die einzelnen Einflussgrößen. Durch Fehleinschätzungen des Wetters, können die Solltemperaturen in den Randzeiten unterschritten werden.

Tabelle 3. Daten Luftkollektor/Geröllspeicher  
(Vergleiche mit weiteren Luftkollektoren siehe [4], S. 117)

Ventilatorstufen	ST 1	ST 2
Ventilatorleistung	265 W	630 W
Ventilatorleistung/m <sup>2</sup> Koll.	6.5 W/m <sup>2</sup>	15.4 W/m <sup>2</sup>
Luftgeschwindigkeit		
Kanal	1.53 m/s	4.56 m/s
Kollektor	0.07 m/s	0.20 m/s
Max. Stundenleistung	590 m <sup>3</sup> /h	1760 m <sup>3</sup> /h
Max. Stundenleistung/m <sup>2</sup> Koll.	14.4 m <sup>3</sup> /h	43 m <sup>3</sup> /h
Steuerung:		
Temperaturdifferenz Lamelle/Speicher	10 °C	-
Kollektortemperatur	-	40 °C
Max. Tagesleistung	14 670 m <sup>3</sup> /d	
Kollektorfläche netto	41 m <sup>2</sup>	
Geröllspeicher	60 m <sup>3</sup> , 30 kWh/K	
Geröllspeicher/m <sup>2</sup> Koll	0.73 kWh/K.m <sup>2</sup>	
Druckdifferenz total	1.46 m <sup>3</sup> /m <sup>2</sup>	
	11 mm H <sub>2</sub> O	