

## RESULTS FROM FIELD-TESTING OF AN PRESENCE CONTROLLED VENTILATION SYSTEM IN AN OCCUPIED OFFICE BUILDING

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### Abstract

The use of IR detectors to steer the ventilation is in principle an attractive approach for optimising the ventilation according to the occupants needs. In order to evaluate the performances under real conditions, one of the BBRI office buildings in Limelette (some 31 offices with in total 51 persons and a variable occupation load) was equipped with a mechanical supply ventilation system in which each terminal is controlled by an IR detector. During a two week period, the performances of the ventilation system were measured in detail (total air flow rate, functioning of each individual terminal, pressure control function, energy consumption, acoustical measurements,...).

The paper presents first the concept of the ventilation system and some of the measured performances at component level. The overall performance of the system is then discussed as well as the impact of the building and ductwork airtightness. Finally conclusions and recommendations for further improvements are given.

### 1. Introduction

Ventilation is a major energy consumer in office buildings and it is therefore interesting to control it so as to find a compromise between indoor air quality and energy consumption.

Although research in the recent years has showed that occupants are not the only source of pollution in office building (but also building materials, furniture, etc.), it is clear that, up to the present, it is the only source upon which a realistic ventilation control can be based.

The use of infrared detectors to steer the ventilation is a possible approach for optimising the ventilation as a function of the occupancy in the ventilated spaces. Other methods like people counting or carbon dioxide measurements can be appropriate as well.

In order to evaluate its performances under real conditions, one of the BBRI office buildings was equipped with a mechanical supply system in which each terminal is controlled by an infrared sensor.

After a description of the investigated system, this paper presents the results of the performed monitoring: evaluations at component level (pressure control, ductwork,...) are first presented, secondly the overall system performances. Then comes a chapter that shows how the ductwork airtightness and the building airtightness impact on the performances of the system. Finally conclusions and recommendations are given.

## 2. System description

### 2.1 General

The studied IR controlled ventilation system is installed in BBRI office building B which is situated in Limelette, 20 km South of Brussels, Belgium. It is a 2-storey office building with a volume of 3000 m<sup>3</sup>, counting 31 rooms with 1 to 4 working places each, or 51 working places in total.

The mechanical air supply installation is made of two separated systems, one for each storey. They consist of a constant speed fan, a filter providing the pressure control function, a pre-heating system, ducts and presence controlled ventilation terminals.

The system is designed to provide 25 m<sup>3</sup>/h of outside air per working place in the rooms that are effectively occupied. When all the rooms are occupied, this provides a total air flow rate of  $51 \times 25 = 1275$  m<sup>3</sup>/h for the whole building. Extraction is taking place in the toilets via the corridors.

The ventilation system is operating from 6:00 AM to 8:00 PM.

### 2.2 Supply terminals

The ventilation terminal is controlled by a motion sensor based upon infrared detection. If presence of occupants is detected in a room, the ventilation is activated thanks to an electro/pneumatic device. When the occupants leave the room, the ventilation remains activated during 15 minutes before switching off.

The terminal can be manually set in four different opening positions corresponding with four different air flow rates: 25, 50, 75 and 100 m<sup>3</sup>/h. This setting is made once for all in function of the number of working places in the ventilated space, one working place corresponding with a rate of 25 m<sup>3</sup>/h. The pressure in the ductwork should stay between 70 Pa and 130 Pa to ensure a proper operation of the terminals.

### 2.3 Autocontrolling filter

#### 2.3.1 Filtering and pressure control functions

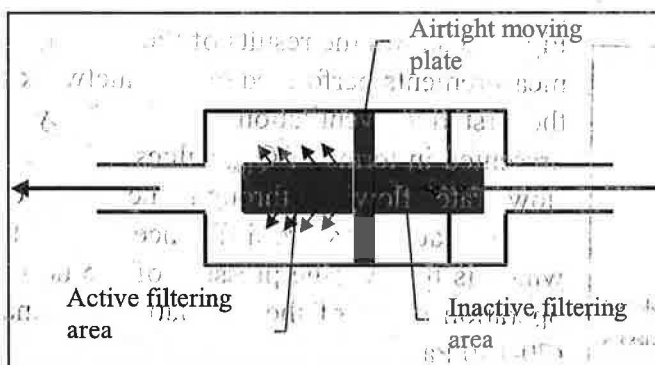


Figure 1 - Pressure control function of the filter

Besides the usual filtering function, this autocontrolling filter is designed to maintain the pressure in the ductwork between certain limits. This control function is of course essential for a ventilation system allowing the total air flow rate to vary with the demand.

The pressure control function is obtained by moving an airtight plate which modifies the active filtering area through which the air is passing (see figure) hence the pressure drop through the filter. The pressure in the ductwork is measured and the position of the moving plate is adjusted in order to keep this pressure between the chosen limits.

Furthermore, this equipment is designed to change automatically the filter when the accumulated particulate matters provoke too high pressure losses in the filter to ensure a normal operation of the ventilation system.

### 3. Measurement set-up

Various parameters were continuously recorded during the monitoring of the whole system operation: the total supply air flow rates using tracer gas technique, the outside temperature, the temperatures after the pre-heating units, the pressure at the end of the ductwork and just after the fan, the state (open or closed) of each OPTO ventilation terminal of the ground floor system and the position of the airtight moving plate in the autocontrolling filters.

Figure 2 shows the ventilation system with its different components and the positions of the different measurement points.

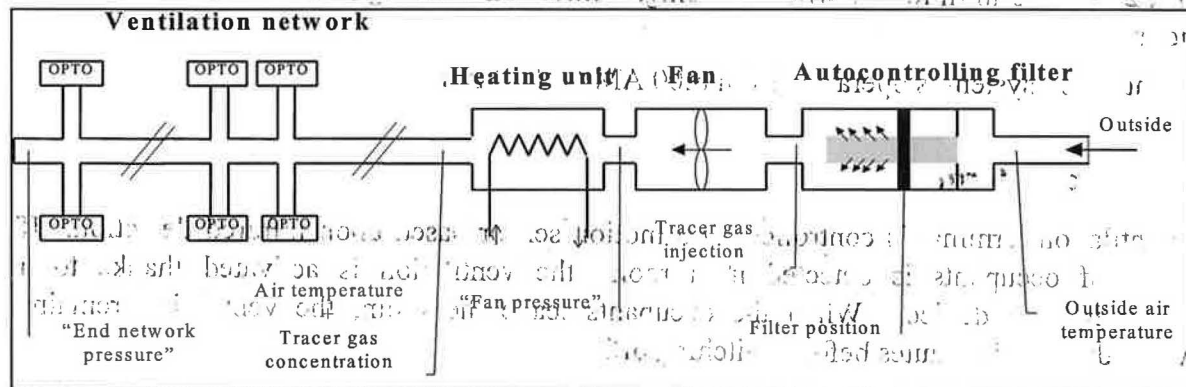


Figure 2 - Monitoring of the whole system operation - Measurement points

Spot measurements of the overall building airtightness, the ductwork airtightness, the acoustical level, the air flow characteristics of terminals and air flow rates at ventilation terminals were also performed.

## 4. Evaluation at component level

### 4.1 Ductwork

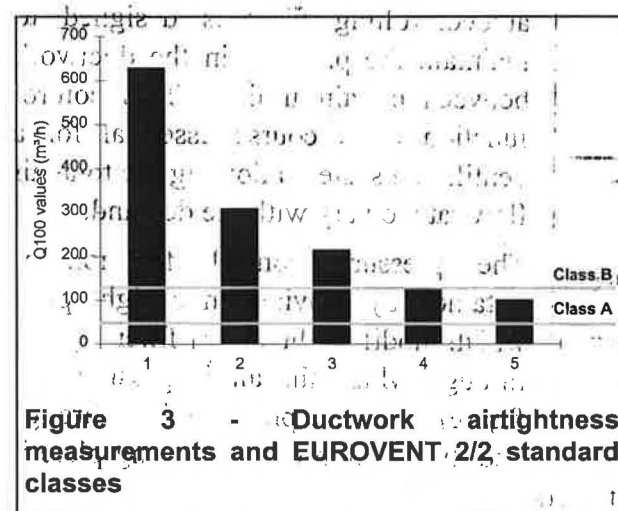


Figure 3 - Ductwork airtightness measurements and EUROVENT 2/2 standard classes

Figure 3 shows the results of the airtightness measurements performed on the ductwork of the first floor ventilation system. They are presented in terms of  $Q_{100}$  values, i.e. the air flow rate flowing through the ductwork leakages at a pressure difference of 100 Pa, which is the average pressure of the normal operation range of the ventilation terminals (70-130 Pa).

The EUROVENT 2/2 standard, "Air leakage rate in sheet metal air distribution systems", proposed two classes for ductwork airtightness. The  $Q_{100}$  values according to

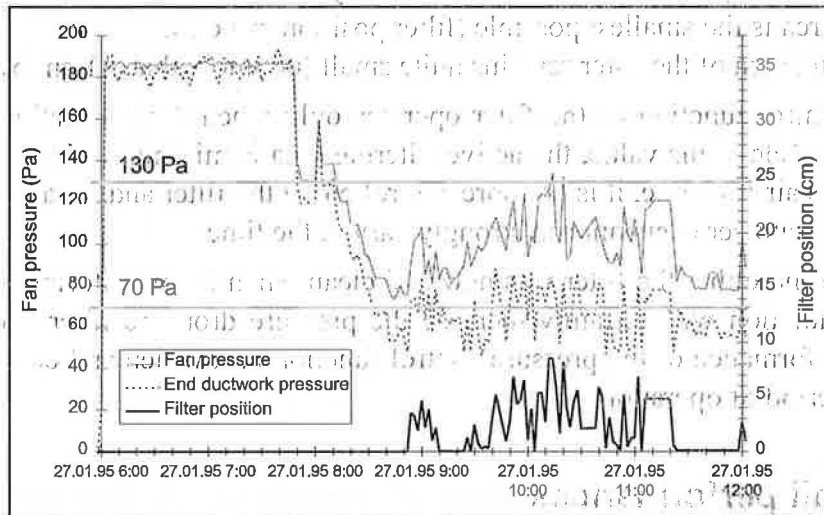
this standard are also shown.

As it can be seen, the initial  $Q_{100}$  value was about  $630 \text{ m}^3/\text{h}$  which is almost equal to the nominal ventilation air flow rate ( $625 \text{ m}^3/\text{h}$ ). Under these conditions, the system could not operate normally. Several improvements were necessary to reach an acceptable value. The last test (n°5) gave a value slightly better than the class A of the EUROVENT 2/2 standard.

## 4.2 Autocontrolling filter

The pressure control function should allow in theory to maintain the pressure in the ductwork in the operating range of the ventilation terminal (70-130 Pa)

The following figure gives, for the ground floor, the measured pressures just after the fan and at the end of the ductwork as well as the filter position during a typical day. The maximum filtering area corresponds with the top of the scale (38 cm) and the minimum filter area with the bottom of the scale (0 cm).

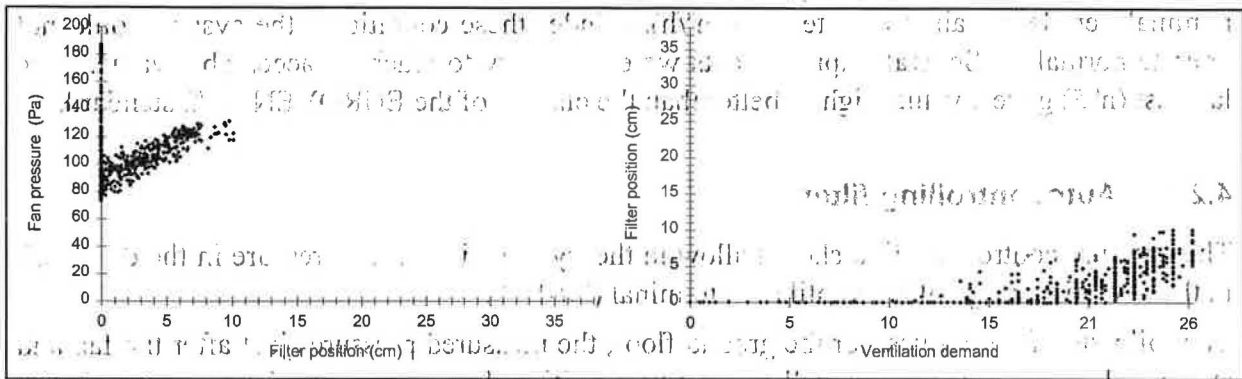


**Figure 4 - Filter position, fan pressure and end ductwork pressure (ground floor)**

The following remarks can be made:

- In the morning, when the ventilation demand is zero or very low, the filtering area is minimal which should provoke a sufficient pressure drop to decrease the air flow rate through the fan so as to obtain a ductwork pressure between 70 and 130 Pa. However, the measured ductwork pressures are near 180 Pa.
- Since the pressure at all ventilation terminals is somewhere between the two measured pressures, it is likely that most of them function within the operating range (70-130 Pa). However, the pressure at the end of the ductwork falls under 70 Pa several times. On this subject, it must be stressed that too low pressures prevent the terminal to close completely and some air can flow into unoccupied rooms.
- During high demand periods the pressure drop in the ductwork can reach values up to 70 Pa. In this event, it is normal that the end ductwork pressure falls under the working range since the control is based on the pressure measured at the beginning of the ductwork. The problem could be solved by using duct of larger diameter so as to reduce the pressure losses or, more simply but less energy efficiently, by putting the control pressure measurement point at a more central place in the ductwork.

The following figures show the fan pressure as a function of the filter position and the filter position as a function of the ventilation demand.



**Figure 5. Operation of the pressure control function of the filter**

As it can be seen, the pressure after the fan can vary from 180 Pa to about 75 Pa when the active filtering area is the smallest possible (filter position = 0 cm). One can also observe that the active filtering area of the filter remains quite small (not more than 11 cm over 38 cm).

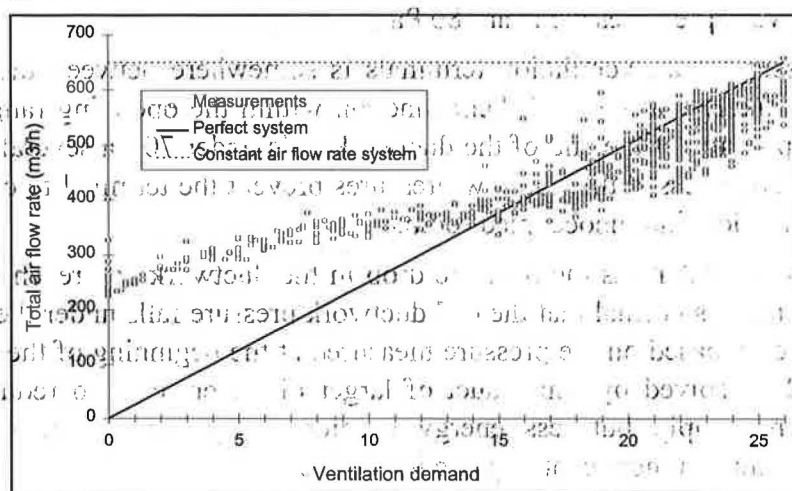
The pressure control function of the filter operates only when the ventilation demand goes over 10 people. Below this value, the active filtering area is minimal and the pressure varies in function of the air flow rate, it is no more controlled by the filter and, as a consequence, the supplied air flow rate per occupant can strongly vary in the time.

It is important to note that the filter was new and clean when the monitoring was performed. The dust accumulation will certainly increase the pressure drop the filter can provoke and therefore the performance of the pressure control function at low demand could be improved after a certain period of operation.

## 5. Overall performances

The overall performance analysis is based on the ground floor measurements (26 places).

The next figure shows the air flow rate effectively supplied to the ground floor in function of the ventilation demand as well as the air flow rate that would be supplied by a perfect system (25 m<sup>3</sup>/h per person). The air flow rate supplied by a constant air flow rate system installed in the same building, that is 26 x 25 m<sup>3</sup>/h, is also shown.



**Figure 6 -Air flow rate supplied in function of the ventilation demand - Measurements, perfect controlled system and constant air flow strategy**

One can observe that the total air flow rate is quite higher than expected for the low ventilation demand (< 15 people). This is on the one hand due to the high ductwork pressures recorded for the low demands and, on the other hand, to the ductwork leakages. The situation could progressively improve as the filter is getting dirtier.

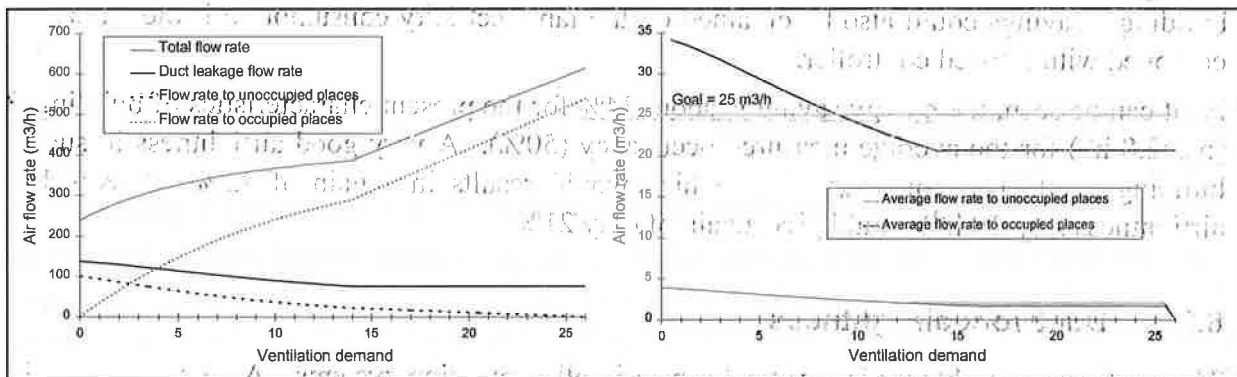
It must be stressed that the total air flow rate shown on the previous figure represents the amount of air supplied by the fan and not the amount of air supplied to the occupied rooms. Indeed, a part of the air is supplied through the ductwork leakages to the corridor and another part is supplied through the closed ventilation terminals to the unoccupied rooms.

The previous figure gives a correct image of the system performance from the energy point of view but we need to go further to evaluate the system from an IAQ point of view.

### 5.1 Split-up of the total air flow rate

The total air flow rate supplied by the ventilation system (ground floor) is made of three components: the air flow rate supplied to occupied rooms through open terminals; the air flow rate supplied to unoccupied rooms through closed terminals; the air flow rate supplied to the corridor or the basement through ductwork leakages.

On the basis of the air flow characteristics of the ventilation terminals and the ductwork leakages, it has been possible to model the behaviour of the ventilation system and to split the total air flow rate into its different components. This is shown on the next figure as well as the average air flow rate per person in occupied and unoccupied rooms.



**Figure 7 - Split-up of the total air flow rate supplied to the ground floor and average air flow rate per person.**

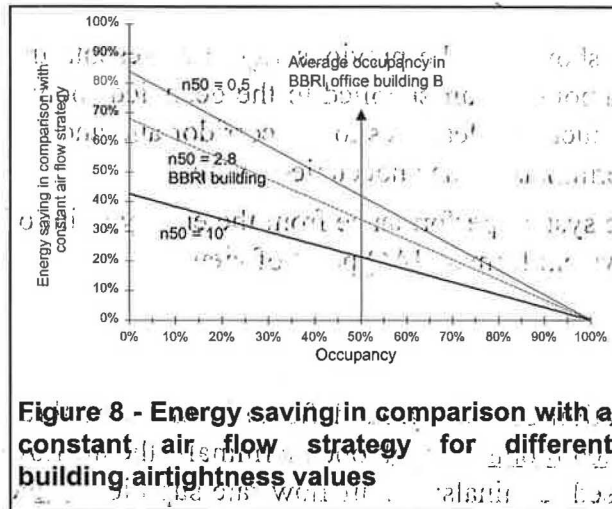
As one can see, the air flow rate through the ductwork leakages varies from 100 m<sup>3</sup>/h for the high demand up to 130 m<sup>3</sup>/h for the low demand. Moreover, when the ventilation demand is very low, the air flow rate to the unoccupied rooms is about 100<sup>3</sup>/h.

The average air flow rate supplied to the people (in occupied rooms) can vary from 34 m<sup>3</sup>/h to about 20 m<sup>3</sup>/h. These variations are due, on the one hand, to the high pressures in the ductwork for the low demand and, on the other hand, to the high ductwork pressure losses for the high demand combined with the position of the control pressure measurement point.

## 6. Impact of ductwork and building airtightness

### 6.1. Building Airtightness

It is clear that the building airtightness plays a very important role in the performance of ventilation systems. Indeed, the air infiltration results in an additional air flow rate which is totally uncontrolled and thus increase the energy consumption. In case of controlled ventilation systems, the natural infiltration has a direct impact on the energy savings that can be obtained as illustrated hereafter.



**Figure 8 - Energy saving in comparison with a constant air flow strategy for different building airtightness values**

through the envelope.

Figure 8 shows the energy savings that can be obtained thanks to an IR controlled ventilation system in comparison with a constant air flow strategy and for 3 degrees of airtightness. It was assumed that the pressure control was perfect (100 Pa). In this case study, the energy savings are related to the heating energy consumption only since no cooling system equips the building. Savings could also be obtained on the fan electricity consumption if the latter was equipped with a speed controller.

As it can be seen, the energy saving is about 34% for the present characteristics of building B ( $n_{50} = 2.8 \text{ h}^{-1}$ ) for the average measured occupancy (50%). A very good airtightness in such a building would be about  $n_{50} = 0.5 \text{ h}^{-1}$  which would result in a gain of 42%. A very bad airtightness ( $n_{50} = 10 \text{ h}^{-1}$ ) would give a gain of only 21%.

### 6.2 Ductwork airtightness

The ductwork airtightness is a critical aspect in all ventilation systems. A comparison of its impact on the performances of a constant air flow system and of a IR controlled system is made to illustrate that matter.

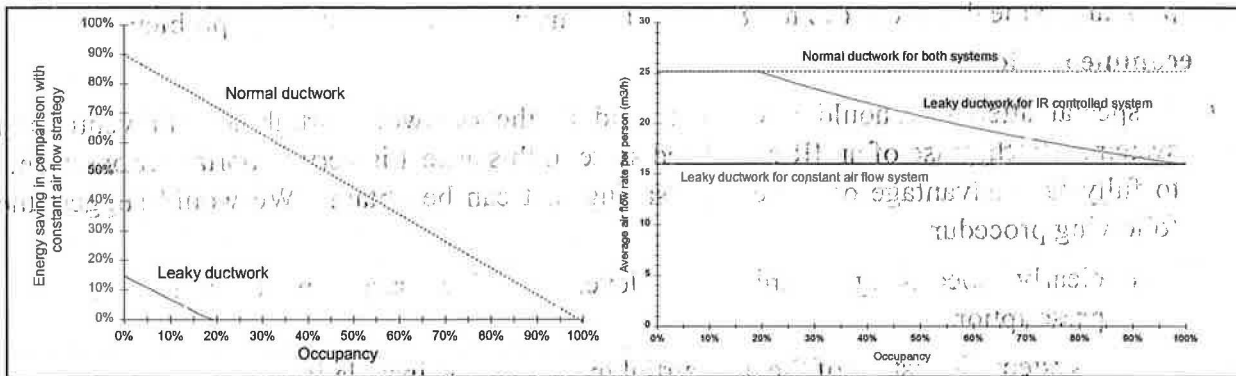
It will be assumed that the installed fan has been chosen to provide a maximal air flow rate which takes into account normal ductwork leakages (class A in EUROVENT 2/2 standard) and 25 m<sup>3</sup>/h for each working place. The fan is strictly limited to that maximal value.

In this case, if the airtightness of the ductwork is worse than expected, it would result in a too low operating pressure hence air flow rate when the demand is high.

This case is illustrated on the next figures. The first one represents the energy saving that can be achieved in comparison with a constant air flow strategy for two degrees of airtightness.

The ductwork airtightness of the "leaky ductwork" is taken equal to the first measured value in the BBRI building B, before any improvement and the ductwork airtightness of the "normal

ductwork” is taken equal to the last measured value in the BBRI building B, after all the performed improvements.



**Figure 9 - Energy saving obtained by IR control in comparison with a constant air flow strategy for two values of ductwork airtightness - Average air flow rate per person**

As it can be seen, the energy savings that can be made thanks to an IR controlled system are limited when the ductwork airtightness is very bad. In case of leaky ductwork and when the occupancy is more than 20%, the energy consumption becomes the same as for a constant air flow system.

From the IAQ point of view, it can be observed that the average air flow rate per person is always higher with the IR controlled system. Indeed, using IR control the required working pressure in the ductwork can be obtained for the low demand. With IR control the average air flow rate delivered to each occupied office reduces if the total occupancy increases.

## 7. Conclusions

The following conclusions can be drawn from the measurements and observations made.

### Regarding the performances of the ventilation terminals:

- The IR detection function of the ventilation terminal seems to work in a satisfactory way.
- Observations showed that under normal conditions, the air velocities in occupied spaces are low enough to avoid draught complaints. Furthermore, the noise levels are acceptable.

### Regarding the overall system performances

- The measurements have highlighted the tremendous importance of the duct airtightness. In the investigated building and system, the original airtightness was so poor that the IR control had little impact on the total air flow rate and as a consequence on the energy bill.
- A lot of efforts were needed to improve the ducts airtightness so as to come near an acceptable value. It is a very labour intensive activity hence a high cost activity and, therefore, it is clear that the only realistic solution is to achieve a better airtightness from the beginning and clear performance requirements in the technical prescription. It seems sufficient to require ductwork leakage levels as given in the EUROVENT standard.
- The pressure variation in the ductwork was considerable (from 60 to 180 Pa). The high pressures are due to a too small pressure drop in the autocontrolling filter which could improve as the filter gets older. The low pressures are due to high pressure losses in the ductwork for high demand and a bad position of the control pressure measurement point. This could be improved without changing the principle of the pressure control system.



- During periods of non-occupation of the building or periods of very low ventilation demand, the noise coming from the ventilation system is not negligible because of the high pressure in the ductwork (up to 180 Pa). It seems quite easy to solve this problem.

### Recommendations

- A special attention should always be paid to the ductwork airtightness in ventilation systems. In the case of an IR controlled system, this aspect is very important if one wants to fully take advantage of the energy saving that can be obtained. We would suggest the following procedure:
  1. clearly specifying a minimum level for duct airtightness in the technical prescriptions.
  2. a systematic testing of the duct airtightness after the installation.
  3. perform improvements if needed.
- In the investigated system the position of the pressure control measurement point was not appropriate. This should always be carefully checked.
- The duct pressure losses for high ventilation demands are quite high (up to 70 Pa). The ductwork of a controlled ventilation system should be designed so as to obtain reasonable losses for the whole range of air flow rates.

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