

ANALYSIS OF DRAFT QUANTITY THROUGH VENTILATED AIR SPACE

Akihito OZAKI*, Takashi SUGAI*, Toshiyuki WATANABE**,
Kyoji KOJINA* and Kohei ONO*

*Department of Architecture, Faculty of Engineering, Fukuoka University
8-19-1 Nanakuma, Jyonan-ku, Fukuoka 814-80, Japan
Tel:+81-92-871-6631, Fax:+81-92-865-3109

**Department of Architecture, Faculty of Engineering, Kyushu University
6-10-1 Hakozaki, Higashi-ku, Fukuoka 812, Japan
Tel:+81-92-642-3339, Fax:+81-92-642-3341

ABSTRACT A draft quantity through a ventilated air space is analyzed by numerical calculation and model experiment, and a correlation between the draft quantity and buoyancy, following a temperature difference between the ventilated air space and circumference, is discussed. At first, a dimensionless theoretical analysis method for the draft quantity is shown, and a relation between a dimensionless draft quantity and dimensionless height $\ell/(bG_r)$ of ventilated air space is defined. Next, the relation between both values is experimentally verified. As the results, it is made clear that the draft quantity shows a proportional relationship with the temperature difference in case a thickness of the ventilated air space is the same, and the dimensionless draft quantity can be expressed as a function of the dimensionless number $(b/\ell)G_r$ (modified Rayleigh number).

1. INTRODUCTION

In an exterior wall of recent residences, a ventilated air space is generally constructed for the purpose of air ventilation. However a draft quantity of the ventilated air space is hardly researched. The present report therefore aims at quantitatively explaining a correlation between the draft quantity and buoyancy, following a temperature difference between the ventilated air space and circumference, and at theoretically obtaining the draft quantity in response to any different condition in temperature difference, thickness and height of ventilated air space and opening area of air vent.

2. NUMERICAL CALCULATION METHOD OF DRAFT QUANTITY THROUGH VENTILATED AIR SPACE

A draft quantity through a ventilated air space, which is constructed between a heating and adiabatic surface as in Fig.1, can be estimated by equations (1)~(3) of continuity, energy and motion.

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0 \quad (1)$$

$$\rho c_p u \frac{\partial T}{\partial x} + \rho c_p v \frac{\partial T}{\partial y} = \lambda \frac{\partial^2 T}{\partial y^2} \quad (2)$$

$$u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} = v \frac{\partial^2 u}{\partial y^2} - \frac{1}{\rho} \frac{\partial p_d}{\partial x} + \beta g(T - T_o) \quad (3)$$

$$\left(\because p_d = p - p_o, \beta = \frac{\rho_o - \rho}{\rho(T - T_o)} \right)$$

Boundary conditions are expressed by equations (4)~(8).

$$x = 0, 0 < y < b : u = \bar{u}, v = 0, T = T_o \quad (4)$$

$$y = 0, x \geq 0 : u = 0, v = 0, T = T_w \quad (5)$$

$$y = b, x \geq 0 : u = 0, v = 0, \partial T / \partial y = 0 \quad (6)$$

$$x = 0 : p_d = 0 \quad (7)$$

$$x = l : p_d = 0 \quad (8)$$

The equations as described above can be rearranged by introducing dimensionless numbers defined by following equations (9)~(16)

$$U = \frac{bu}{vG_r}, V = \frac{bv}{v} \quad (9), (10)$$

$$X = \frac{x}{bG_r}, Y = \frac{y}{b} \quad (11), (12)$$

$$P = \frac{p_d b^2}{\rho v^2 G_r^2}, \Theta = \frac{T - T_o}{T_w - T_o} \quad (13), (14)$$

$$G_r = \frac{g\beta(T_w - T_o)b^3}{v^2}, P_r = \frac{\mu c_p}{\lambda} \quad (15), (16)$$

Continuity, energy and motion equations are rewritten as follows:

$$\frac{\partial U}{\partial X} + \frac{\partial V}{\partial Y} = 0 \quad (17)$$

$$U \frac{\partial \Theta}{\partial X} + V \frac{\partial \Theta}{\partial Y} = \frac{1}{P_r} \frac{\partial^2 \Theta}{\partial Y^2} \quad (18)$$

$$U \frac{\partial U}{\partial X} + V \frac{\partial U}{\partial Y} = \frac{\partial^2 U}{\partial Y^2} - \frac{\partial P}{\partial X} + \Theta \quad (19)$$

Boundary conditions are represented as follows:

$$X = 0, 0 < Y < 1 : U = Q, V = 0, \Theta = 0 \quad (20)$$

$$Y = 0, X \geq 0 : U = 0, V = 0, \Theta = 1 \quad (21)$$

$$Y = 1, X \geq 0 : U = 0, V = 0, \partial \Theta / \partial Y = 0 \quad (22)$$

$$X = 0 : P = 0 \quad (23)$$

$$X = L : P = 0 \quad (24)$$

Then Q (dimensionless draft quantity) and L (dimensionless height of ventilated air space) are defined on referring to the dimensionless numbers.

$$Q = \frac{b\bar{u}}{vG_r} = \int_0^1 U dY \quad (25)$$

$$L = \frac{l}{bG_r} \quad (26)$$

In case difference coord from equation (27) by u every height level of the

$$Q = \frac{\Delta Y}{3} \left(4 \sum_{j=1}^{(K-1)/2} U_{m+1,2} \right)$$

If the values of Q and P_r, U, V, Θ and P can be so basis of the boundary con Q, which is obtained from found out (the value of P obtained.

3. EXPERIMENTAL MI THROUGH VENTILA

Fig.3 shows an experimen can be arbitrarily change plywood and an transpa aluminum plate, of which material put on outside c system. We measured dra space with smoke wire m acrylic board is a window the acrylic board. We gen electric current through a acrylic board and to which loci of smoke through the emitting strobe light into the velocity of flow from the d time (We read the distance In the upper part of the v approaches that of heating between both surfaces as in assumption that the measur The draft quantities were thickness (10, 15, 20, 25, 30 (5, 10, 15K) between the framework of air vent is atta

4. ANALYSIS OF MEASU

Photo.1 and Fig.6 show exa that the thickness of the ve between the ventilated air s strobe light is 0.5sec, and th the ventilated air space. The Fig.7 and 8 (results witho correlation between the dra

- (3)
- (4)
- (5)
- (6)
- (7)
- (8)
- (9),(10)
- (11),(12)
- (13),(14)
- (15),(16)
- (17)
- (18)
- (19)
- (20)
- (21)
- (22)
- (23)
- (24)
- (25)
- (26)

In case difference coordinates are decided as in Fig.2, the value of Q is obtained from equation (27) by use of Simpson's low, because the draft quantity is equal in every height level of the ventilated air space.

$$Q = \frac{\Delta Y}{3} \left(4 \sum_{j=1}^{(K-1)/2} U_{m+1,2j} + 2 \sum_{j=1}^{(K-3)/2} U_{m+1,2j+1} \right) = \text{Constant draft quantity} \quad (27)$$

If the values of Q and P, are given as the initial calculation conditions, the values of U, V, Θ and P can be solved from the equations (17)~(19) in every height level on basis of the boundary conditions by convergence calculation concerning the value of Q, which is obtained from the right side term of equation (27). If the value of L is found out (the value of P becomes 0 in this height), the corresponding value of Q is obtained.

3. EXPERIMENTAL MEASUREMENT METHOD OF DRAFT QUANTITY THROUGH VENTILATED AIR SPACE

Fig.3 shows an experimental model of a ventilated air space, thickness of its space can be arbitrarily changed. One side of ventilated air space is constructed by plywood and an transparent acrylic board, another side is constructed by an aluminum plate, of which temperature can be controlled freely by a flat calorific material put on outside of this plate. Fig.4 shows an experimental measurement system. We measured draft quantities (velocity of flow) through the ventilated air space with smoke wire method, as the visible method of a moving smoke. The acrylic board is a window to observe a locus of smoke. A camera is set up in front of the acrylic board. We generated smoke in the ventilated air space by passage of an electric current through a nichrome wire, which was stretched on under level of the acrylic board and to which paraffin oil was applied. Then we took photographs of the loci of smoke through the acrylic board, in a state of opening a stop, by periodically emitting strobe light into the ventilated air space from the side, and we obtained the velocity of flow from the distance of photographed loci in the regular intervals of a time (We read the distance of the loci at the middle part of the ventilated air space). In the upper part of the ventilated air space (a temperature of adiabatic surface approaches that of heating surface), velocity distribution of flow describe a parabola between both surfaces as in Fig.5. Therefore we calculated the draft quantities on the assumption that the measured velocity of flow is the maximum value of the parabola. The draft quantities were measured under different experimental conditions in thickness (10, 15, 20, 25, 30mm) of the ventilated air space, temperature differences (5, 10, 15K) between the ventilated air space and circumference, and a joint framework of air vent is attached or not at lower part of the ventilated air space.

4. ANALYSIS OF MEASURED AND CALCULATED DRAFT QUANTITY

Photo.1 and Fig.6 show examples of the loci of smoke and its tracing, for the case that the thickness of the ventilated air space is 20mm, the temperature difference between the ventilated air space and circumference is 15K, the interval of emitting strobe light is 0.5sec, and there is not the joint framework of air vent at lower part of the ventilated air space. The locus of smoke moves at regular distance. Fig.7 and 8 (results without or with the joint framework of air vent) show a correlation between the draft quantities and the temperature differences. The draft

introducing dimensionless

follows:

less height of ventilated air
pers.

quantities indicate a proportional relationship with the temperature differences, irrespectively without or with the joint framework, in case the thickness of the ventilated air space is the same. Fig.9 and 10 show the correlations between the draft quantities or the dimensionless draft quantities Q and dimensionless number $(b/\ell)G_r P_r$ (modified Rayleigh number: P_r times the reciprocal of L). The solid lines in the figures represent the calculated values, provided that modified Rayleigh number is revised by using a correction value A'/A in the case with the joint framework. The values of Q and $(b/\ell)G_r P_r$ show one-to-one correspondence. The calculated values agree well with the measured values, whether there is a joint framework of air vent or not, namely the draft quantities can be expressed as the function of modified Rayleigh number.

5. CONCLUSION

We showed a dimensionless theoretical analysis method, on the basis of equations of continuity, energy and motion, to obtain a draft quantity through a ventilated air space, which is constructed between a heating and adiabatic surface, and defined a relation between a dimensionless draft quantity and dimensionless height $\ell/(bG_r)$ of ventilated air space. Furthermore, a correlation between the draft quantity and buoyancy, following a temperature difference between the ventilated air space and circumference, is explained by a visible experiment (smoke wire method).

The summary of the results is: The draft quantity indicates a proportional relationship with the temperature difference in case a thickness of the ventilated air space is the same. The dimensionless draft quantity and dimensionless number $(b/\ell)G_r P_r$ (modified Rayleigh number) show one-to-one correspondence, and the values of calculated draft quantities agree well with that of measured values. In case a joint framework of air vent is attached at lower part of the ventilated air space, the calculated values agree with the measured one by making revision for the modified Rayleigh number by a correction value A'/A . The draft quantity can be expressed as the function of modified Rayleigh number.

NOTATION

A : sectional area of ventilated air space, A' : opening area of air vent, b : thickness of ventilated air space [m], c_p : specific heat of air [J/(kg·K)], G_r : Grashof number [-], g : gravitational acceleration [m/s²], K : horizontal grid number [-], L : dimensionless height [-], ℓ : height of ventilated air space [m], P_r : dimensionless differential pressure between ventilated air space and circumference [-], P_r : Prandtl number [-], p_s : differential pressure between ventilated air space and circumference [Pa], Q : dimensionless draft quantity [-], q : draft quantity [m³/s], T : absolute temperature [K], U : dimensionless velocity of vertical direction [-], u : velocity of vertical direction [m/s], V : dimensionless velocity of horizontal direction [-], v : velocity of horizontal direction [m/s], X : dimensionless coordinates of vertical direction [-], x : coordinates of vertical direction [m], Y : dimensionless coordinates of horizontal direction [-], y : coordinates of horizontal direction [m], β : coefficient of volume expansion [1/K], ΔX : grid interval of vertical direction [-], ΔY : grid interval of horizontal direction [-], Θ : dimensionless air temperature [-], λ : thermal conductivity [W/(m·K)], μ : viscosity [Pa·s], ν : kinematic viscosity [m²/s], ρ : specific weight of air [kg/m³]

REFERENCES

- 1) Osamu MIYATAKE and Tetsu FUJII : "A Numerical Analysis of Natural Convective Heat Transfer between Two Parallel Vertical Plates", The Reports of Research Institute of Industrial Science, Kyushu University No.55, pp.9-25, 1972
- 2) Akihito OZAKI, Takashi SUGAI, Toshiyuki WATANABE et al.: "Analysis of Heat and Moisture Transfer through Ventiladed Air Space", Research Report of 35th session, Kyushu Branch, Architectural Institute of Japan, pp.105-108, 1996

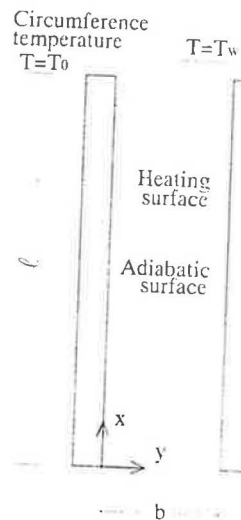


Fig.1 Analytical model

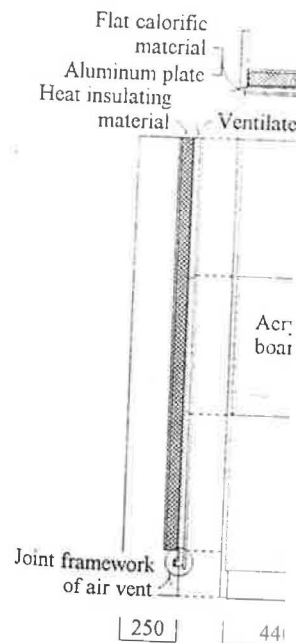


Fig.3 Experimental model air space

temperature differences, case the thickness of the relations between the draft and dimensionless number (reciprocal of L). The solid lines ed that modified Rayleigh in the case with the joint o-one correspondence. The s, whether there is a joint es can be expressed as the

on the basis of equations of ty through a ventilated air atic surface, and defined a dimensionless height $\ell/(bG_r)$ of en the draft quantity and he ventilated air space and e wire method).

indicates a proportional ckness of the ventilated air and dimensionless number e correspondence, and the of measured values. In case he ventilated air space, the g revision for the modified uantity can be expressed as

at, b : thickness of ventilated air $[-]$, g : gravitational acceleration $[-]$, h : height of ventilated air space and circumference $[-]$, P_r : space and circumference $[\text{Pa}]$, Q : temperature $[\text{K}]$, U : dimensionless $[\text{m/s}]$, V : dimensionless $[\text{m/s}]$, X : dimensionless $[\text{m}]$, Y : dimensionless $[\text{m}]$, β : coefficient of ΔY : grid interval of horizontal $[\text{W}/(\text{m}\cdot\text{K})]$, μ : air $[\text{kg}/\text{m}^3]$

s of Natural Convective Heat Research Institute of Industrial

t al.: "Analysis of Heat and port of 35th session, Kyushu

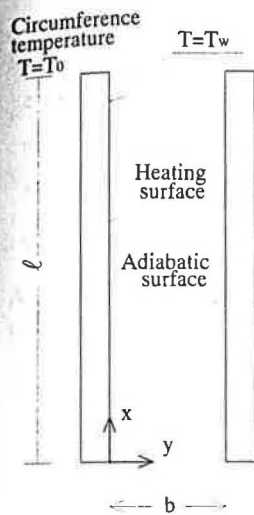


Fig.1 Analytical model

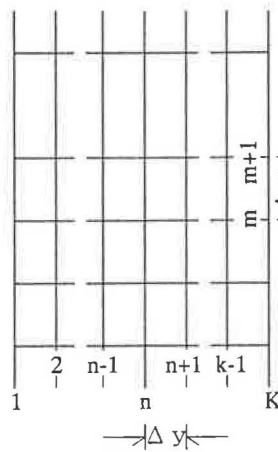


Fig.2 Difference coordinates

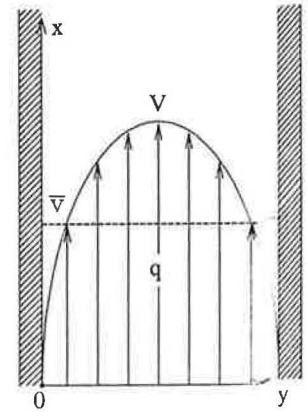


Fig.5 Velocity distribution of flow

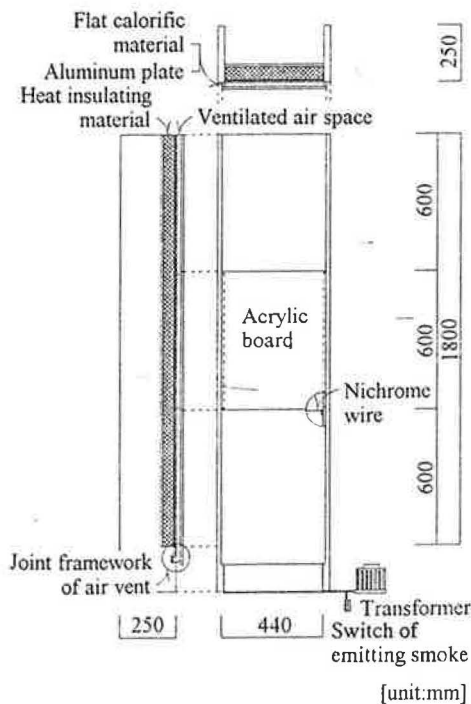


Fig.3 Experimental model of ventilated air space

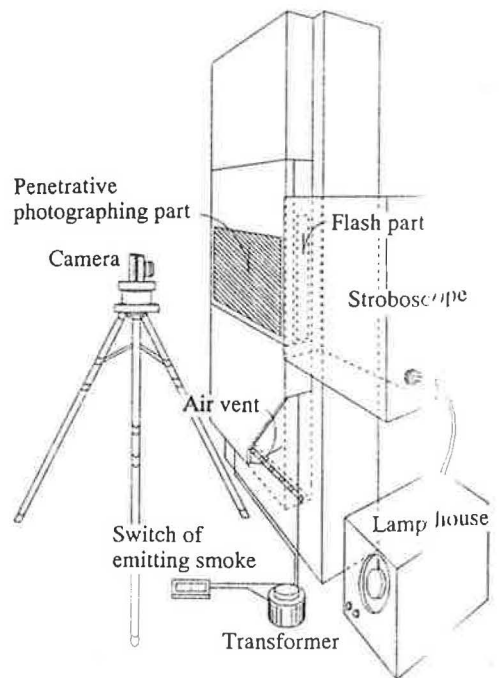


Fig.4 Experimental measurement system

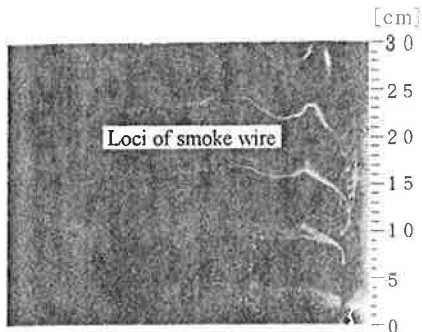


Photo.1 Photograph of smoke wire

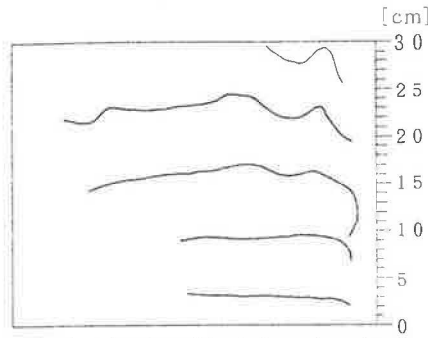


Fig.6 Tracing of loci of smoke wire

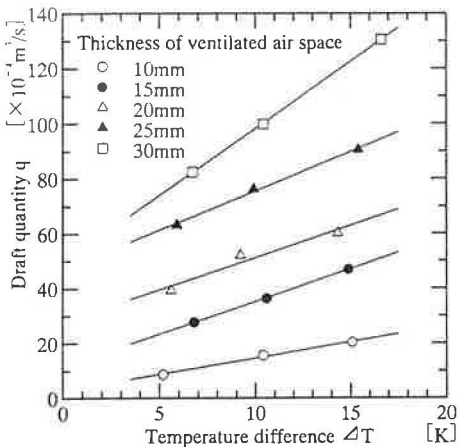


Fig.7 Correlation between draft quantity and temperature difference without a joint framework of air vent (sectional area of ventilated air space: $1 \sim 3 \times 10^2 \text{ cm}^2/\text{m}$)

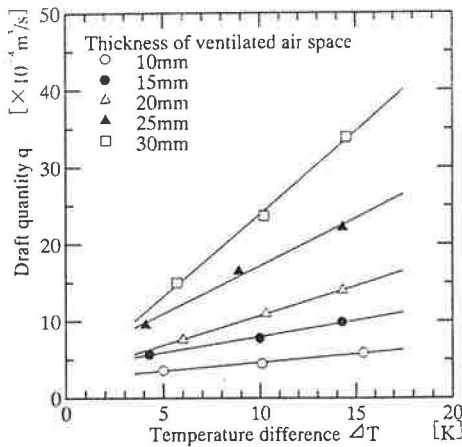


Fig.8 Correlation between draft quantity and temperature difference with a joint framework of air vent (opening area of air vent: $7.31 \text{ cm}^2/\text{m}$)

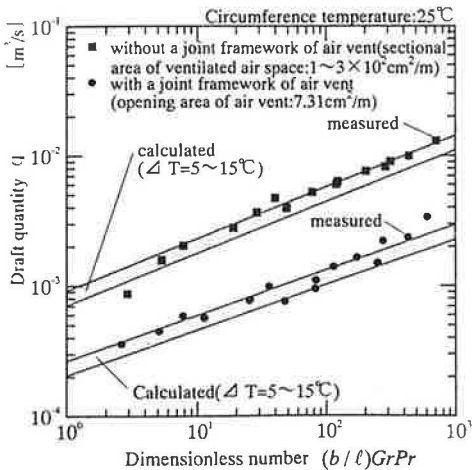


Fig.9 Correlation between draft quantity and dimensionless number

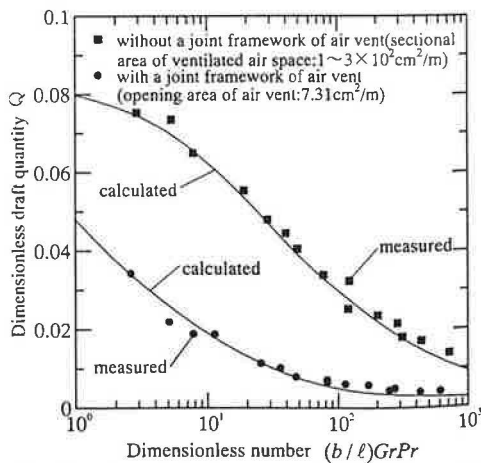


Fig.10 Correlation between dimensionless draft quantity and dimensionless number

STUDY ON S

Takashi IN
Dept. of Architecture, S
Takaharu
Nikken Sekke

ABSTRACT A solar shading performance was verified (with roll blinds in the cavity between the double skins of the atrium under natural ventilation) in the presence of solar radiation, air temperature and humidity. The simulation was carried out in the atrium under a natural thermal environment was carried out. The results also suggested that in autumn, natural ventilation was more effective than in summer. The model of the double skins was also developed and adopted. The model which describes the natural ventilation was also proved the possibility of utilizing the simulation model developed and adopted.

1. INTRODUCTION

Because an atrium has a large volume, under various conditions, the concern is that the atrium is heated by solar radiation and an increase in temperature. An example where a double skin is used in the atrium in order to control the temperature in the atrium.

2. SUMMARY OF ATRIUM

The atrium was located in the building envelope except the side that faces the double skin (outside: heat reflective glass, inside: a roll blind was installed). The roll blind provided around the double skin was used for the ventilation, forced ventilation in the upper part of the double skin (Fig. 2) (measured value), or natural ventilation in the lower part (Fig. 3) may be selected.