

CONFERENCE ORGANIZERS



43rd AIVC

11th TightVent & 9th venticool Conference

Ventilation, IEQ and health
in sustainable buildings

October
4-5
2023

Copenhagen
Denmark

PRESENTATIONS



www.aivc2023conference.org

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Congress Venue

Aalborg University A. C. Meyers Vænge 15, 2450 København -Denmark

Registration Desk Hours

Registration Desk will be open during the following dates and times:

Tuesday 3 October, 2023 / 19:00 – 20:00

Wednesday 4 October, 2023 / 08:00 – 19:00

Thursday 5 October, 2023 / 08:30 – 18:00

Poster display information

- Posters should be set up on Wednesday 4 October, 2023 from 09:00
 - Dismantling of posters should be finished by **Thursday 5 October 2023 at 16:30**
- Professional Congress Organizer and Organizers have no liability for posters left behind

Poster dimensions

A0) size, 120CM Height x 80CM Width

Poster presentation session

Wednesday October 4, 2023 at 19.00 – 21.00

Authors are kindly requested to be in front of their poster to be able to reply to any questions

Long & Short Oral Presentation information

- Long Oral Presentations (indicated within the programme) are expected to last 12 minutes; another 3 minutes are foreseen for questions and answers (15 minutes in total)
- Short Oral Presentations (indicated within the programme) are expected to last 3 minutes; another 2 minutes are foreseen for questions and answers (5 minutes in total)

Social Events

Welcome Reception

Tuesday 3 October, 2023

19:00 – 20:00

Aalborg University,
Kantine Area

Poster Presentations & Student Competition – Industry stands – Cocktail Reception

Wednesday 4 October, 2023

18:30 – 20:00

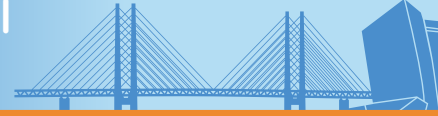
Aalborg University,
Poster Area

Gala Dinner

Thursday 5 October, 2023

20:00 – 22:30

Kosmopol ApS
(Fiolstræde 44 DK-1171
København)

Tuesday October 3th, 2023

19:00-20:00 Registration & Welcome reception



ROOM D (1.042)

Wednesday October 4th, 2023

08:00-09:00 Registration

09:00-10:30 Opening - Plenary session

Chairs: Alireza Afshari, Arnold Janssens**Welcome on behalf of AIVC, venticool, TightVent**
Arnold Janssens, INIVE/AIVC/Ghent University**Welcome on behalf of Aalborg University Copenhagen**
Alireza Afshari, Aalborg University Copenhagen, Denmark**Tomorrow's Ventilation Solutions for Future Hospital Demands**
Trond Thorgeir Harsem, Nordconsult, Norway**Users and practices in heating and ventilating homes -
why do they behave different than we think?**
Kirsten Gram-Hanssen, Aalborg University, Denmark**What we know about smart ventilation**
Gaëlle Guyot, Cerema, France**Dallying with DALYs: Why acceptable IAQ should consider harm**
Benjamin Jones, University of Nottingham, United Kingdom

10:30-11:00

Coffee Break

Wednesday October 4th, 2023

**ROOM A (1.008)**

11:00-12:30

**Session 1A - Topical Session
(Airborne cross infection and engineering solutions)****Chairs:** Peter V. Nielsen, Chen Zhang

The COVID-19 pandemic raises the public's attention on cross infections in the indoor environment. WHO has identified airborne transmission as a principal route for SARS-CoV-2, especially in crowded and poorly ventilated indoor environments. In fact, airborne transmission is one of the primary routes for many infectious diseases, such as anthrax, chickenpox, and influenza. The airborne cross-infection risk between people is influenced by many parameters, such as distance, relative position, respiratory activities, room ventilation, face mask, or other protection equipment. Effective control of these influencing factors can be important to mitigate airborne transmission risk between people. Many prevention measures were recommended by authorities during COVID-19, but their protective effects regarding airborne transmission are still under discussion.

The main objective of this workshop is to discuss the mechanism of airborne transmission and the engineering control solutions. Through the discussion, the workshop will find out some effective and efficient control measures to reduce airborne cross-infection risk. Discussion topics include:

1. What are the parameters influencing the cross-infection risk?
2. What are the challenges to reduce infection risk indoors?
3. How to design indoor airflow distribution to reduce the indoor exposure?
4. Are there any other engineering control solutions and what are their efficiency?
5. Where these engineering control solutions may be applied

Opening

Alireza Afshari, Aalborg University, Denmark

Human exposure against airborne pathogens in an office environment

Risto Kosonen, Aalto University, Finland

Discussion on minimum ventilation rates for infection control

Yuguo Li, University of Hong Kong, Hong Kong

Mitigation of airborne transmission of respiratory viruses by ventilation – past, present and future

Arsen Krikor Melikov, Technical University of Denmark, Denmark

Point source ventilation effectiveness in infection risk-based post-COVID ventilation design

Jarek Kurnitski, Tallinn University of Technology, Estonia

Airborne transmission of disease in stratified and non-stratified flow

Peter V. Nielsen, Aalborg University, Denmark

Discussion with the audience



Wednesday October 4th, 2023



ROOM B (1.001)

11:00-12:30 **Session 1B - Long Oral Presentation Session**
(Building airtightness testing & prediction)

Chairs: Laure Mouradian, Iain Walker

Acoustic method for measurement of airtightness - field testing on three different existing office buildings in Germany

Bjorn Schiricke, DLR (German Aerospace Center), Germany

Pulse tests in highly airtight Passivhaus standard buildings

Xiaofeng Zheng, University of Nottingham, United Kingdom

Correlation analysis between ACH50 and Air permeability considering the floor area of a residential buildings

Suji Choi, Inha University, Republic of Korea

Airtightness predictive model from measured data of residential buildings in Spain

Irene Poza Casado, University of Valladolid, Spain

Bridging The Mechanical / Enclosure Gap

David de Sola, 3iVE LLC, United States of America



ROOM C (2.1.042)

11:00-12:30 **Session 1C - Topical Session**
(Summer comfort and energy efficiency in hot periods:
interest of mixed mode cooling and need of occupant feedback)

Chairs: Maxime Boulinguez, Gwénaëlle Haese, Arnaud Jay

This session explores combined passive, soft and active cooling solutions, some occupant feedback and performance indicators. The presentation will be based on case studies in different climate conditions. The first part of the session will focus on windows and ceiling fan occupant behaviour model coupling methodology with Building Energy Models. This presentation will rely on a tropical case study. Then, an innovative approach to better understand hot discomfort will be highlighted based on the measurement of global human responses. This presentation will lie on an application to end users of mixed-mode cooled buildings under tropical climate conditions. Thirdly, a Windows coach for office workers will be introduced. The coach's objective is to advise occupants on the proper action to take on their windows (open or close) to optimise their thermal comfort, IAQ and energy efficiency. Feedback on two summers' experimental campaigns for occupant thermal comfort in a naturally ventilated building in a continental climate has been used to design the coach. This feedback will be presented. Finally, these works led to a new research project, CoolDown, funded by the French National Research Agency (ANR). It aims to develop new tools and methodologies to target energy and comfort performance in mixed-mode cooled buildings from early design to on-site performance. COOL-DOWN methodology will be introduced in the last presentation and will serve as a starting point for the discussion of this session.

Wednesday October 4th, 2023

11:00-12:30 Windows and ceiling fan occupant behaviour model coupling methodology with building energy models, a tropical case study

Maxime Boulinguez, *Université La Réunion - PIMENT, France*

An innovative approach to better understand hot discomfort, based on the measurement of global human responses, including physiological and sensory indicators - application to end users of mixed mode cooled buildings under tropical climate conditions

Gwénaëlle Haese, *CSTB, France*

An IAQ and thermal comfort coach prototype to improve comfort and energy consumption thanks to adequate management of natural ventilation: development and first feedback results

Arnaud Jay, *CEA Liten - INES, France*

Towards an alternative cooling: Optimisation of the successive use of the cooling systems from passive to active - Development of design and control strategies of the hybrid cooling

Arnaud Jay, *CEA Liten - INES, France*

12:30-13:30

Lunch Break



ROOM C (2.1.042)

**13:30-14:45 Session 2A - Topical Session
(Energy Performance of Gas Phase Air Cleaning)**

Chairs: Bjarne W. Olesen, Sasan Sadrizadeh

The session will focus on the energy performance of gas phase air cleaning. Standalone air cleaners may improve air quality by delivering a certain Clean Air Delivery Rate (CADR). For the same level of air quality, the ventilation rate can be reduced by a similar amount. However, standalone air cleaners are also using energy. Air cleaners built into the ventilation system may increase pressure drop and using some power, which both increase the energy use. A couple of studies based on models and dynamic building simulations on energy use for heating, cooling, and ventilation have been used to study the overall energy implications of using gas phase air cleaners. The results will be presented and discussed in this session.

13:30-14:45 Introduction to IEA EBC Annex 78
Bjarne W. Olesen, *Technical University of Denmark, Denmark*

Air cleaner as an alternative to increased ventilation rates in buildings: a simulation study for an office

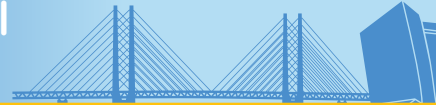
Alireza Afshari, *Aalborg University Copenhagen, Denmark*

Exploring the Energy-Saving Benefits of Gas-Phase Air Cleaning in Nordic Buildings

Sasan Sadrizadeh, *KTH Royal Institute of Technology, Sweden*

Gas phase air cleaning effects on ventilation energy use and indicators for energy performance

Dragos-Ioan Bogatu, *Technical University of Denmark, Denmark*

**ROOM B (1.001)**

13:30-14:45

**Session 2B - Topical Session
(Revision of ISO 9972: Improvements in the reliability
of airtightness measurements)****Chairs:** Valérie Leprince, Gary Nelson

ISO 9972 is an international standard describing the measurement procedure and calculation methods for determining the air permeability of buildings using the fan pressurisation method. Given the impact of airtightness on building energy use, more and more tests are performed, a lot of them required by regulations. Indeed, environmental conditions during the test, and more specifically wind and temperature differences, may cause significant errors and thus increase the result uncertainty. However, when a target value must be reached in a mandatory context, knowing the test uncertainty is crucial. Recent works have underlined the need to improve the reliability of the method to measure a building's air leakage rate as described in ISO 9972.

To address these challenges, Cerema has launched a project to review ISO 9972 and has set up a working group of international experts in the field of building airtightness testing to identify the relevant issues with the current standard and propose improvements. This project aims to lay a foundation for a thorough review and revision of the current ISO 9972 standard and this session will present the first results towards this revision.

13:30-14:45

Introduction to the project of ISO 9972 revision

Valérie Leprince, Cerema, France

**On the integration of envelope pressure inhomogeneity and autocorrelation
in fan pressurization uncertainty analysis**

Martin Prignon, Buildwise, Belgium

**Statistical analysis of the correlations between buildings
air permeability indicators**

Bassam Moujalled, Cerema, France

Proposal for new implementations in ISO 9972

Benedikt Kölsch, Cerema, France

Discussion with the audience

Wednesday October 4th, 2023



ROOM C (2.1.042)

13:30-14:45

Session 2C - Long & Short Oral Presentation Session
(Climate change & Resilient cooling)

Chairs: Yun Gyu Lee, Pilar Linares

Which design parameters impact the resilience to overheating in a typical apartment building? (Long Oral Presentation)

Abantika Sengupta, *KU Leuven, Belgium*

Renewable ventilative cooling? Insights from an Irish perspective (Long Oral Presentation)

Adam O' Donovan, *Munster Technological University, Ireland*

Urban context and climate change impact on the thermal performance and ventilation of residential buildings: a case-study in Athens (Long Oral Presentation)

Maria Kolokotroni, *Brunel University London, United Kingdom*

Thermography-based assessment of mean radiant temperature and occupancy in healthcare facilities (Long Oral Presentation)

Paul Seiwert, *RWTH Aachen University, Germany*

Analyzing natural ventilation and cooling potential in a communal space building in Belgium under future climate conditions (Short Oral Presentation)

Shiva Khosravi, *Archipelago Leuven, Belgium*

A study of indoor environment and window use in French dwellings monitored during a summer with heatwaves (Short Oral Presentation)

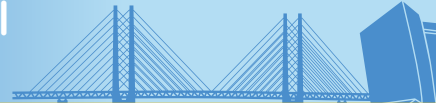
Mathilde Hostein, *Cerema/ENTPE, France*

Importance of thermal stack effect in ventilative cooling concepts for residential buildings (Short Oral Presentation)

Diederik Verscheure, *Vero Duco NV, Belgium*

14:45-15:00

Room Change

**ROOM A (1.008)**

15:00-16:30

Session 3A - Topical Session**(Real performance of (smart) residential ventilation - performance assurance, fault detection, continuous commissioning)****Chairs:** Gaëlle Guyot, Jakub Kolarik

This topical session is organized by the fourth subtask of the IEA EBC Annex 86 Energy Efficient IAQ Management in residential buildings entitled "Ensuring performance of smart ventilation". One of our focus points is the quality assurance of implemented residential ventilation systems & strategies, whether they can be called "smart" or not.

In this topical session, we want to present and discuss results and experiences from different stakeholders representing both industry and academia, regarding real performance of residential ventilation. Furthermore, we want to discuss how do their data stand against existing quality management approaches and inspection protocols for residential ventilation. Can we identify the crucial issues specific to "smart systems"? We aim to discuss examples of approaches to ensure reliable operation beyond the commissioning phase.

15:00-16:30

Welcome and introduction

Gaëlle Guyot, Cerema, France

Performance 2 project - Winter IAQ campaigns in 13 dwellings equipped with Humidity-based DCV systems: analyses of the ventilation performance after 15 years of use

Adeline Melois, Cerema, France & Juan Rios, Aereco, France

Checking and assuring real IAQ and energy performances through demand control and cloud connectivity

Ivan Pollet, Renson, Belgium

Data driven models for fault detection - Combining thermal and indoor air quality grey box models

Gabriel Rojas, University of Innsbruck, Austria

Evaluation of supply temperature set-points and airflow imbalance using smart ventilation data

Kevin Smith, Technical University of Denmark, Denmark

Technologies in balanced ventilation systems to maintain optimal performance in energy and comfort

Bart Cremers, Zehnder Group, The Netherlands

Discussion

Wednesday October 4th, 2023

**ROOM B (1.001)****15:00-16:30** **Session 3B - Topical Session**
(Building and ductwork airtightness regulations in various countries)**Chairs:** Irene Poza Casado, Nolwenn Hurel

In 2008 a series of Ventilation Information Papers (VIP)s (from VIP 17 to VIP 27) were published by the AIVC, detailing the "Trends in the building ventilation market and drivers for changes" for 10 countries. Regulations have however evolved a lot in most countries since then. A new series of VIPs is being developed to get an update on the current regulations in European countries regarding building and ductwork airtightness. They include for both, when relevant, information on: national requirements and drivers: airtightness indicator, requirements in the regulation, energy programs, airtightness justifications, sanctions, etc.; if it is included in the energy calculations and how; the airtightness test protocol: qualification for the testers, guidelines, requirements on measuring devices; tests performed: tested buildings/ductworks, database, evolution with time; guidelines to build airtight buildings/ductworks.

Eight VIPs have been already published in this new series, and most of them have already been presented at the last AIVC Conference in Rotterdam (for Belgium, Czech Republic, Estonia, France and Greece). Contributions from other countries are in preparation, and a total of about 15 publications is expected to give an overview of the building and ductwork airtightness trends in various countries. Some of them are presented in this Topical Session.

15:00-16:30 **Introduction: Presentation of the series of AIVC VIPs on building and ductwork airtightness regulations**
Nolwenn Hurel, PLEIAQ, France**Building and ductwork airtightness in Norway:
national trends and requirements**

Tormod Aurlien, NMBU, Norway

**Building and ductwork airtightness in The Netherlands:
national trends and requirements**

Niek-Jan Bink, ACIN Instrumenten, The Netherlands

**Building and ductwork airtightness in Spain:
national trends and requirements**

Irene Poza Casado, University of Valladolid, Spain

**Building and ductwork airtightness in Latvia:
national trends and requirements**

Andrejs Nitijevskis, IRBEST Ltd, Latvia

Air tightness and its impact on energy consumption in multi-family residential buildings in Montenegro (Short Oral Presentation)

Esad Tombarevic, University of Montenegro, Montenegro

**ROOM C (2.1.042)**

15:00-16:30

**Session 3C - Topical Session
(Resilient Cooling of Buildings meets Resilient Cooling in Cities)****Chairs:** Hilde Breesch, Patryck Czarnecki

In this interactive session the challenges of resilient cooling of buildings and their relation to their urban surroundings will be addressed. After a short presentation of main outcomes of EBC Annex 80 Resilient Cooling of Buildings, the audience will work in small groups on a prepared set of building cases and challenges. With the Technology Profiles from Annex 80 and the Resilient Cooling Guidelines at hand different solutions shall be discussed and most suitable ones identified. The discussion of these different approaches in the plenum shall foster peer learning and create a better understanding of the nexus of resilient cooling of buildings and resilient cooling in cities.

15:00-16:30

Resilient Cooling Technology Profiles from the EBC Annex 80
Peter Holzer, Institute of Building Research & Innovation, Austria**Resilient Cooling Guidelines from the EBC Annex 80**
Vincenzo Corrado, Politecnico di Torino, Italy**Work in small Design Groups****Open discussion and collective conclusion in plenum**

16:30-17:00

Coffee Break

Wednesday October 4th, 2023



ROOM A (1.008)

17:00–18:15

Session 4A - Short Oral Presentation Session (Indoor Air Quality & Health)

Chairs: Max Sherman, Pawel Wargocki

Health risks of residential indoor and outdoor exposure to fine particle-bound phthalates

Jiayao Chen, *University College Dublin, Ireland*

HEPA filters to improve vehicle cabin air quality – advantages and limitations

Dixin Wei, *Volvo Cars, Sweden*

Experimental study of an innovative wet scrubber concept in regards to particle filtration and pressure loss

Nhat Nguyen, *RWTH Aachen University, Germany*

An evaluation of CO2 emission rates by Chilean school children

Nicolas Carrasco, *Pontificia Universidad Catolica de Chile, Chile*

The Effects of Bedroom Mechanical Ventilation on Health and Sleep Quality

Jeongwon Kim, *Dankook University, Republic of Korea*

Analysis of PM2.5 indoor-outdoor ratio in lobby floor according to configurations of entrance

Soyi Park, *Inha University, Republic of Korea*

Proposal of an effort-benefit diagram to compare unit and room air-change rates applied to a literature review

Sven Auerswald, *Fraunhofer ISE, Germany*

Experimental Investigation of Indoor Air Quality in an Open Office Environment

Mustafa Zeki Yilmazoglu, *Untes Heating Air Conditioning Corp., Turkey*

Hygienic Air Handling Unit Certification Program: the new necessity for a guaranteed indoor air quality

Ali Nour Eddine, *Eurovent Certita Certification, France*

Car traffic or emissions from heating sources: What is responsible for IAQ?

Katarzyna Ratajczak, *Poznan University of Technology, Poland*

Monitoring VOCs' concentrations in a circular biobased residential building using low-cost sensors (Student Competition)

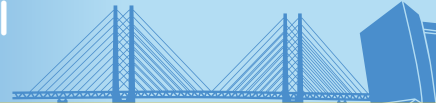
Yannick Thienpont, *KU Leuven, Belgium*

Smart & Predictive Air Quality Solution

Paul Brasser, *Prometech, The Netherlands*

Energy Implications of Increased Ventilation in Commercial Buildings to Mitigate Airborne Pathogen Transmission

David Artigas & Sean M. O'Brien, *Simpson Gumpertz & Heger Inc., United States of America*



Wednesday October 4th, 2023



ROOM B (1.001)

17:00-18:15 Session 4B - Short Oral Presentation Session
(Ventilation strategies & thermal comfort)

Chairs: Jaap Hogeling, Maria Kolokotroni

Reflections on alternative modelling approaches regarding occupants' window operation behaviour

Christiane Berger, *Aalborg University, Denmark*

Development of air supplied ceiling radiant air conditioning system using the Coanda effect (Student Competition)

Satoshi Noguchi, *the University of Kitakyusyu, Japan*

Wind Tunnel Experiment of Wind-Induced Single-sided Ventilation under Generic Sheltered Urban Area

Zitao Jiang, *Osaka University, Japan*

A study on desiccant system regenerated by waste heat from home-use solid oxide fuel cell cogeneration system

Keita Mizuno, *Misawa Homes Institute of Research & Development Co., Ltd., Japan*

Method for Evaluating an Air-Conditioning System with Natural Ventilation by Coupled Analysis of a Building Energy Simulation Tool and Computational Fluid Dynamics (Student Competition)

Ryuichi Yasunaga, *the University of Kitakyushu, Japan*

Performance comparison of different ventilation strategies in elderly care homes in Belgium

Hilde Breesch, *KU Leuven, Belgium*

Sea Water Air Conditioning (SWAC): A Resilient and Sustainable Cooling Solution for hot and humid climates - Energy Performance and Numerical Modeling

Kanhan Sanjiv, *University of French Polynesia, French Polynesia*

The Effects of Lowering Temperature Setpoints on Perceived Thermal Comfort -An experimental study in office buildings (Student Competition)

Beatriz Coutinho, *University of Coimbra - ADAI, Portugal*

Long-term energy performance of dew-point indirect evaporative cooler under the climate change world scenario (Student Competition)

Maria Jesus Romero-Lara, *University of Cordoba, Spain*

On the assessment of the pressure coefficient on the mixed ventilation modeling

Marcos Batistella Lopes, *Associacao Paranaense de Cultura - APC - PUCPR, Brazil*

Construction of operational control rules for an earth-to-air heat exchanger through transfer reinforcement learning (Student Competition)

Yuki Adachi, *the University of Kitakyusyu, Japan*

Ventilation and Thermal Performance Examination of Slot Line Diffuser for Perimeter Usage by CFD Simulation

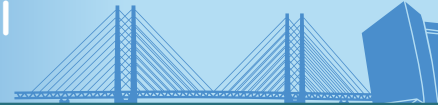
Shaoyu Sheng, *Osaka University, Japan*

Wednesday October 4th, 2023



ROOM A (1.008)

18:15-19:00	Industry Presentations Acin Instrumenten, BCCA, Blowerdoor, DooApp, Lindab, Mez-Technik, Renson, Retrotec, Soudal, Velux, WindowMaster
19:00-21:00	Poster presentations & Student Competition – Industry stands – Cocktail reception with snacks

**ROOM A (1.008)**

09:00-10:30

**Session 5A - Topical Session
(The Role of Building Ventilation on Building Decarbonization)****Chairs:** Núria Casquero-Modrego, Iain Walker

Decarbonization of buildings leads to a significant reduction of CO2 emissions, which is essential in order to meet climate goals. However, this is a complex undertaking, especially when decarbonization is an emergent topic for the construction sector and households. Building ventilation is connected with many factors related to building decarbonization. Considering building ventilation requirements is essential when improving the energy efficiency of buildings (for example when improving the airtightness of existing buildings). Appliance electrification aspects of building decarbonization may also provide opportunities for better IAQ for the occupants and consequently decreasing the possibility of health issues. Furthermore, designing and proposing better building ventilation strategies can help us to meet climate goals. This session aims to bring people with expertise in building ventilation and IAQ, who are working on assessing building ventilation and IAQ strategies for building decarbonization. The goal is to generate a dialogue that addresses the relationship between building ventilation and climate change and promote awareness of the need to integrate ventilation and building decarbonization research.

09:00-10:30

Quantifying the Potential Health Impacts of Unvented Combustion in Homes - A Meta-Analysis

Nuria Casquero-Modrego, LBNL, United States of America

**How to create a performance-based regulation on ventilation
- the French Experience**

Valérie Leprince, Cerema, France

Comparative Analysis Between Indoor Temperatures of Dwellings at Urban Scale During a Typical and Extreme Summers in a Temperate Climate

Ainhoa Arriazu-Ramos, University of Navarra, Spain

Decarbonization and IAQ in Spain: a roadmap

Marta Sorribes Gil, IETCC-CSIC, Spain

Discussion Time: The Role of Building Ventilation on Building Decarbonization

Thursday October 5th, 2023



ROOM B (1.001)

09:00-10:30

**Session 5B - Long Oral/Topical Presentation Session
(Indoor Air Quality & ventilation)**

Chairs: Simon Jones, James McGrath

**Ventilation behaviour of occupants driven by outdoor temperature:
12 case studies**

Sonia Garcia-Ortega, IETCC-CSIC, Spain

**Indoor air quality in Austrian classrooms – Assessing different ventilation
strategies with a citizen science approach**

Simon Beck, University of Innsbruck, Austria

**Measurement of ventilation effectiveness and indoor air quality
in toilets at mass gathering events**

Filipa Adzic, University College London, United Kingdom

**Impact of the building airtightness and natural driving forces
on the operation of an exhaust ventilation system in social housing in Chile**

Gilles Flamant, Pontificia Universidad Catolica de Chile, Chile

**Metal Oxide Semiconductor sensors (MOS) for measuring Volatile Organic
Compounds (VOC) - performance evaluation in residential settings**

Jakub Kolarik, Technical University of Denmark, Denmark

**Towards performance-based approaches for smart residential ventilation:
a robust methodology for ranking the systems and decision-making**

Baptiste Poirier, Cerema, France

**ROOM C (2.1.042)**

09:00-10:30

Session 5C - Topical Session

(Importance of good resilient building design and standards to ensure good ventilative cooling performance to reduce overheating and environmental impact)

Chairs: Christoffer Plesner, Jannick K. Roth

The purpose of this session is to discuss and showcase how ventilative cooling can be part of the following three key elements, in which the building sector are facing:

Resiliency: Robustness and resilience are key indicators when designing future buildings in terms of ventilative cooling.

Indoor climate: The focus on the indoor climate including limiting overheating is a main point due to rapid changes in the outdoor environment, fx. climate change.

Environmental impact: Sustainability will be, and is already, a key parameter when assessing technologies in the built environment.

All three above mentioned key elements are to some extent bound to standards and legislation. Hence, standards and legislation are essential to push new requirements, while setting the bar for future building design.

09:00-10:30

Introduction

Christoffer Plesner, VELUX A/S, Denmark & **Jannick Roth**, WindowMaster International A/S, Denmark

Update on Resilient cooling and indicators from the IEA EBC Annex 80

Peter Holzer, Institute of Building Research & Innovation, Austria

Resilient Ventilative cooling in Design Practice: Where next?

Paul O'Sullivan, Munster University, Ireland

Life cycle assessment: A design element for ventilation system selection

Jannick Roth, WindowMaster International A/S, Denmark

Lessons Learned from Irish Schools: Early-stage Insights on Overheating

Paul O'Sullivan, Munster University, Ireland

Resilient cooling in office buildings: case study in Belgium

Hilde Breesch, KU Leuven, Belgium

Design procedures for ventilative cooling integrated in new standards

Christoffer Plesner, VELUX A/S, Denmark & **Jannick Roth**, WindowMaster International A/S, Denmark

Questions and open Discussion

10:30-11:00

Coffee Break

Thursday October 5th, 2023



ROOM A (1.008)

11:00-12:30 **Session 6A - Topical Session**
(The role of carbon dioxide and particulate matter for assessing ventilation and respiratory disease transmission in buildings)

Chairs: Justin Berquist, Svein Ruud

The objective of this topical session is to discuss some of the potential applications and limitations of CO₂ and PM concentration measurements for assessing ventilation and filtration performance, IAQ, and respiratory disease transmission in buildings. Researchers with a breadth of expertise and publications in this area will present their research. The session programme outlined below includes the individuals selected to present, the expected presentation titles, and the corresponding presentation objective.

11:00-12:30 **Sensitivity Analysis of CO₂ Concentrations as Ventilation Metrics**
Oluwatobi Oke, National Institute of Standards and Technology, (NIST),
United States of America

Evaluation of Uncertainties of Using CO₂ for Studying Ventilation Performance and Indoor Airborne Contaminant Transmissions
Liangzhu (Leon) Wang, Concordia University, Canada

Effects of ventilation on airborne transmission: particle measurements and performance evaluation
Huijuan Chen, Research Institute of Sweden (RISE), Sweden

Impact and benefits of the air cleaning measures implemented in two schools
Liang (Grace) Zhou, National Research Council Canada (NRC), Canada

**ROOM B (1.001)**

11:00-12:30

**Session 6B - Long Oral Presentation Session
(Indoor Environmental Quality)****Chairs:** Alireza Afshari, Sonia Garcia**Critical reflections on indoor-environmental quality constructs**Ardeshir Mahdavi, *TU Graz, Austria***Ventilation and sleep quality**Pawel Wargocki, *Technical University of Denmark, Denmark***Applicability and sensitivity of the TAIL rating scheme using data
from the French national school survey**Pawel Wargocki, *Technical University of Denmark, Denmark***An investigation of MVHR system performance based on health and comfort
criteria in bedrooms of low-carbon social housing in South-Wales, UK**Faisal Farooq, *Cardiff University, United Kingdom***Impact of optimized residential ventilation with energy recovery
on health and well-being**Martin Kremer, *RWTH Aachen University, Germany***A detailed investigation of the impact of an innovative dynamic façade system
on indoor environmental quality in offices**Magdalena Hajdukiewicz, *Eindhoven University of Technology (TU/e),
The Netherlands*

Thursday October 5th, 2023



ROOM C (2.1.042)

11:00-12:30 **Session 6C - Long Oral Presentation Session
(Ventilative cooling & Natural Ventilation)**

Chairs: Dong Hwa Kang, Jensen Zhang

A methodology for evaluating the ventilative cooling potential in early-stage building design

Valentina Radice Fossati, *Eurac Research, Italy*

Ventilation reliability: A pilot study on window opening behaviour in a primary school (Student Competition)

Lara Tookey, *Massey University, New Zealand*

A survey of building design practitioner perceptions of ventilative cooling in their building design processes

Maha Sohail, *Munster Technological University, Ireland*

Can naturally ventilated office buildings cope with dusty outdoor air?

Evangelos Belias, *EPFL, Switzerland*

Distribution of Particulate Matter Concentration and Temperature Stratification Examined by Zonal Model and Experimental Measurements in Room with A Novel Portable Displacement Ventilation Cooling Unit

Toshio Yamanaka, *Osaka University, Japan*

Thermal comfort and risk of draught with natural ventilation - assessment methods, experiences and solutions

Jannick Roth, *Windowmaster International A/S, Denmark*

12:30-13:30

Lunch Break



ROOM A (1.008)

13:30-14:30 **Session 7A - Long Oral Presentation Session
(Air Cleaning)**

Chairs: Wouter Borsboom, Arnold Janssens

Evaluation of sensor-based air cleaners to remove PM2.5 and TVOC from indoors with pollutant sources of smoking and burning candles (Student Competition)

Li Rong, *Aarhus University, Denmark*

Developing methodology for testing of gas-phase air cleaners based on perceived air quality

Pawel Wargocki, *Technical University of Denmark, Denmark*

Evaluating the impact of air cleaning on bioaerosols and other IAQ indicators in Belgian daycare facilities

Sarah Paralovo, *VITO, Belgium*

Removal of Odorants in Nursing Homes Using Air Cleaners

Stig Koust, *Danish Technological Institute, Denmark*



Thursday October 5th, 2023

**ROOM B (1.001)**

13:30-14:30 **Session 7B - Long Oral Presentation Session
(Ventilation & infection risk)**

Chairs: Gaëlle Guyot, Jelle Laverge

**What can CO2 measurements tell us about ventilation
and infection risk in classrooms?**

Carolanne Vouriot, *University of Cambridge, United Kingdom*

**Indoor air modelling and infection risk assessment
in a naturally ventilated patient room**

Natalia Lastovets, *Tampere University, Finland*

**Performance of Local Ventilation System Combined with Underfloor
Air Distribution as Preventative Measures for Infectious Diseases
in Consulting Room (Student Competition)**

Jun Yoshihara, *Osaka University, Japan*

**The numerical investigation of human micro-climate
with different human simulators**

Haruna Yamasawa, *Osaka University, Japan*

**ROOM C (2.1.042)**

13:30-14:30 **Session 7C - Topical Session
(Personalized Environmental Control Systems (PECS)
operation and evaluation)**

Chairs: Dragos-loan Bogatu, Bjarne W. Olesen

The session will introduce IEA-EBC Annex 87 and discuss aspects related to PECS operation and evaluation. Current indoor environmental quality (IEQ) and energy performance evaluation methods alongside used key performance indicators (KPIs) will be described. Examples of both traditional and advanced control strategies will be presented. A secondary objective of the session is to gather input from the conference audience on the aforementioned topics.

13:30-14:30 **Introduction to IEA EBC Annex 87**
Bjarne W. Olesen, *Technical University of Denmark, Denmark*

**Indoor environmental quality (IEQ) and energy performance
evaluation of PECS**

Douaa Al-Assaad, *KU Leuven, Belgium*

Physiological sensing for thermal comfort assessment

Dragos-loan Bogatu, *Technical University of Denmark, Denmark*

14:30-14:45 **Room Change**

Thursday October 5th, 2023

**ROOM A (1.008)****14:45-16:15** **Session 8A - Topical Session
(Post Pandemic Pontifications)****Chairs:** Benjamin Jones, Max Sherman

The COVID-19 pandemic highlighted the importance of indoor air quality and appropriate ventilation in buildings to prevent airborne disease transmission. As we move into the endemic phase, it remains crucial that we continue to prioritize effective ventilation and air cleaning to keep occupants as healthy as possible. This topical session will cover the latest research on airborne transmission mechanisms, including implications for ventilation system design, sizing, and operation. Presenters will share case studies of transmission mitigation and lessons learned from the pandemic response. There will be an emphasis on strategies for improving air quality in public buildings within the constraints of operating budgets. Attendees will gain practical knowledge to assess systems to make impactful upgrades, and communicate the ongoing importance of ventilation and air quality to stakeholders in a post-pandemic context. The goal is to ensure the health, safety, productivity, and wellbeing of building occupants now and in the future.

14:45-16:15 **ASHRAE 241-2023 Control of Infectious Aerosols (Long Oral Presentation)**Max Sherman, *University of Nottingham, United Kingdom***Can the Wells-Riley model universally assess airborne pathogen infection risk?**
(Long Oral Presentation)Benjamin Jones, *University of Nottingham, United Kingdom***Flow dynamic of human cough and measuring techniques: A review**
(Long Oral Presentation)Chen Zhang, *Aalborg University, Denmark***Evaluating the impact of air cleaning and ventilation of airborne pathogens and human bio-effluents at two primary schools in Belgium**
(Long Oral Presentation)Klaas De Jonge, *Ghent University, Belgium***Review of international standards describing air cleaner test methods**
(Long Oral Presentation)Hannelore Scheipers, *Ghent University, Belgium***Rethinking different ventilation strategies in a post-pandemic era:
a CFD assessment (Short Oral Presentation)**Alicia Murga Aquino, *Kobe University, Japan***How the COVID Pandemic and the Energy Crisis Have Influenced Indoor
Environmental Conditions in non-residential Buildings (Short Oral Presentation)**Aurora Monge, *University of Navarra, Spain*



Thursday October 5th, 2023

**ROOM B (1.001)**

14:45-16:15 **Session 8B - Long Oral Presentation Session**
(Impact factors on IAQ)

Chairs: Marie Coggins, Andy Persily

The impact of increased occupancy on particulate matter concentrations in mechanically-ventilated residential buildings in a subtropical climate
German Hernandez Herrera, *Universidad Politécnica de Madrid (UPM), Spain*

On-Site Capture Efficiency of Kitchen Range Hood Based on Particle Diameters and Exhaust Flow Rates
Shinhye Lee, *Seoul National University, Republic of Korea*

An investigation of cooking-related pollutants in the residential sector
Daniela Mortari, *University Savoie Mont Blanc, France*

Fine dust measurement in ducts of balanced ventilation systems
Bart Cremers, *Zehnder Group Zwolle, The Netherlands*

The Impact of Deep Energy Renovations on Indoor Air Quality and Ventilation in Irish Dwellings
Hala Hassan, *University of Galway, Ireland*

Financial impact of leaky ductwork in buildings – a calculation tool to raise awareness
Nolwenn Hurel, *PLEIAQ, France*

16:15-16:45

Coffee Break

**ROOM D (1.042)**

16:45-18:15 **Closing session**

Chairs: Arnold Janssens, Alireza Afshari

Decoding 30 Years of Insights: Conclusions from ISIAQ's Landmark Webinar Series on Indoor Air Quality and Climate
Ying Xu, *Tsinghua University, China*

Summing up of the "Smart ventilation, IAQ & Health" track
Benjamin Jones, *University of Nottingham, United Kingdom*

Summing up of the "Building and ductwork airtightness" track
Valerie Leprince, *Cerema, France*

Summing up of the "Ventilative and resilient cooling" track
Hilde Breesch, *KU Leuven, Belgium*

Best paper/poster award & Student Competition awards

Announcement of 2024 conference

Closing

20:00

Gala Dinner (ticket required)

43rd AIVC 11th TightVent & 9th venticool Conference



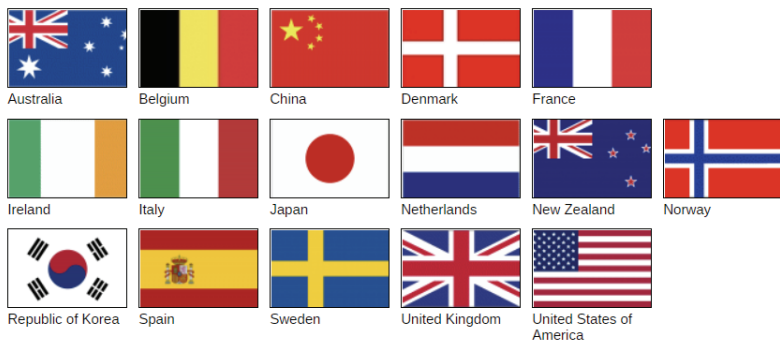
Welcome on behalf of AIVC, TightVent and venticool

Arnold Janssens
Peter Wouters
Operating Agents AIVC - INIVE



1

Welcome on behalf of AIVC (1979 - ...)

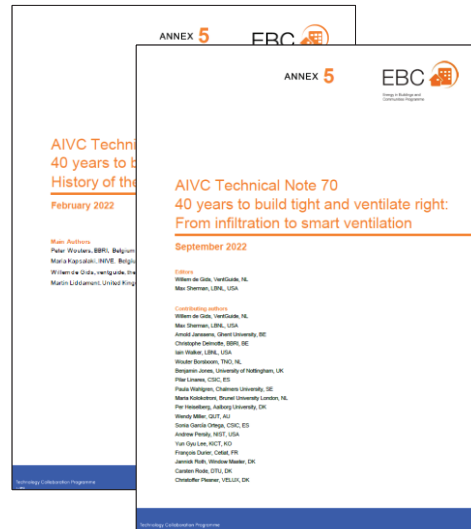


2


AIVC and Denmark




- 1 of 8 founding countries of AIVC
- (Almost) uninterrupted membership since creation of AIVC
- AIVC-conferences in Denmark:
 - 1993, Copenhagen, 'Energy Impact of Ventilation and Air Infiltration'
 - 2012, Copenhagen, 'Optimising Ventilative Cooling and Airtightness for [Nearly] Zero-Energy Buildings, IAQ and Comfort'
 - 2023, Copenhagen, 'Ventilation, IEQ and health in sustainable buildings'



3



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Events

The AIVC holds a conference each year in September/October, a workshop in March/April and several webinars, covering a wide range of topics in the field of infiltration and ventilation in buildings. The conferences and workshops take place in one of the AIVC participating countries. Since 1980, the AIVC annual conferences have been an international meeting point for presenting and discussing major developments and results regarding infiltration and ventilation in buildings.


Click on the links below to know more.



Conferences



Workshops



Webinars

- **Annual conferences** in collaboration with TightVent and venticool platforms
- **Annual workshops** in collaboration with local hosts on themes of local interest
- **Webinars** presenting results of projects or publications.

4

43rd AIVC 11th TightVent & 9th Venticool Conference



Welcome on behalf of TightVent (2011 - ...) and Venticool (2013-...)



5

More focusing on knowledge generation aspects



More focusing on awareness raising and market implementation



...Building and ductwork airtightness...
...Ventilative cooling...



6

FAQs

Select a Category

▼ nothing selected

Search

[What is building airtightness?](#)

[What are the methods to detect leaks of the building envelope?](#)

[What are existing methods to estimate building airtightness? Are there other test methods apart from fan pressurization?](#)

[What is ductwork airtightness?](#)

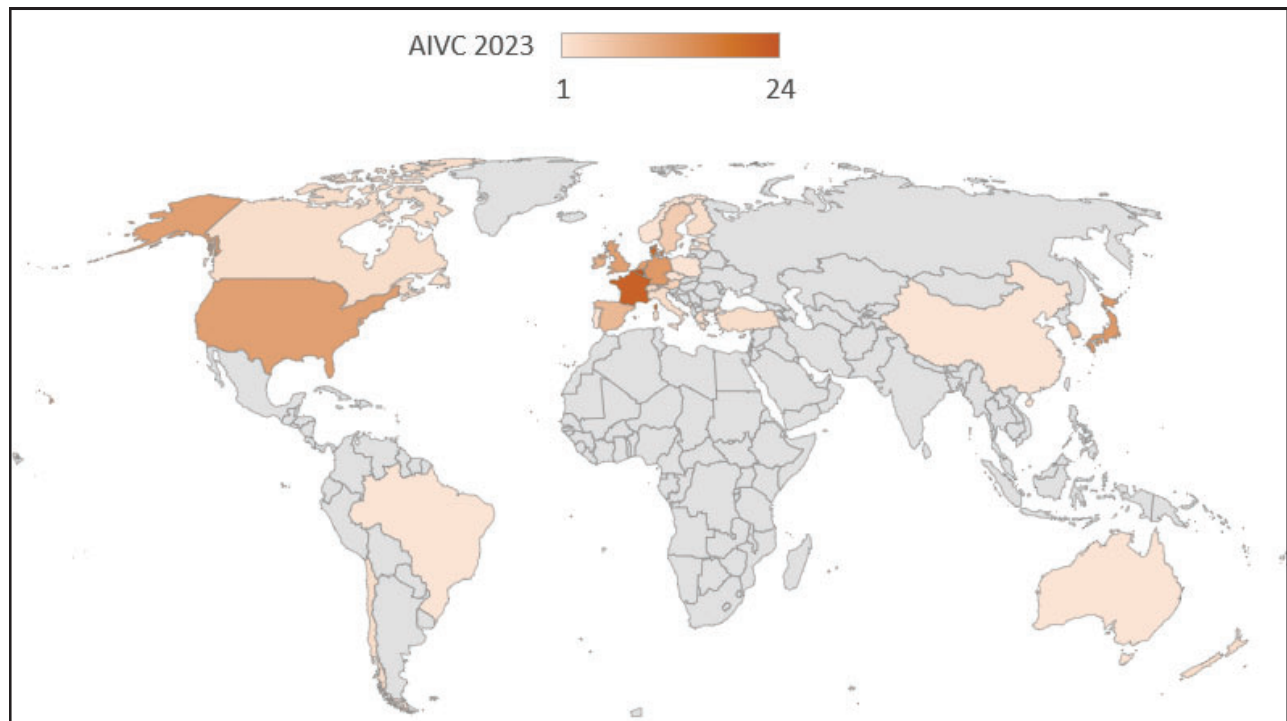
[What is infiltration/exfiltration?](#)

[How is the building airtightness quantified?](#)

[How is the ductwork airtightness quantified?](#)

[What are the most common air leakage/infiltration paths?](#)

7



8

Programme October 4

	Room A	Room B	Room C
09:00-10:30	Opening - Plenary session		
10:30-11:00	Coffee break		
11:00-12:30	Topical Session (Airborne cross infection and engineering solutions)	Long Oral Presentation Session (Building airtightness testing & prediction)	Topical Session (Summer comfort and energy efficiency in hot periods: interest of mixed mode cooling and need of occupant feedback)
12:30-13:30	Lunch break		
13:30-14:45	Topical Session (Energy Performance of Gas Phase Air Cleaning)	Topical Session (Revision of ISO 9972: Improvements in the reliability of airtightness measurements)	Long & Short Oral Presentation Session (Climate change & Resilient cooling)
14:45-15:00	Room change		
15:00-16:30	Topical Session (Real performance of (smart) residential ventilation - performance assurance, fault detection, continuous commissioning)	Topical Session (Building and ductwork airtightness regulations in various countries)	Topical Session (Resilient Cooling of Buildings meets Resilient Cooling in Cities)
16:30-17:00	Coffee break		
17:00-18:15	Short Oral Presentation Session (Indoor Air quality & Health)	Short Oral Presentation Session (Ventilation strategies & thermal comfort)	
18:15-19:00	Industry presentations (Acin Instrumenten, BCCA, Blowerdoor, DooApp, Lindab, Mez-Technik, Renson, Retrotec, Soudal, Velux, WindowMaster)		
19:00-21:00	Poster presentations & Student Competition - Industry stands - Cocktail reception with snacks		

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Programme October 5

	Room A	Room B	Room C
09:00-10:30	Topical Session (The Role of Building Ventilation on Building Decarbonization)	Long Oral Presentation Session (Indoor Air Quality & ventilation)	Topical Session (Importance of good resilient building design and standards to ensure good ventilative cooling performance to reduce overheating and environmental impact)
10:30-11:00	Coffee break		
11:00-12:30	Topical Session (The role of carbon dioxide and particulate matter for assessing ventilation and respiratory disease transmission in buildings)	Long Oral Presentation Session (Indoor Environmental Quality)	Long Oral Presentation Session (Ventilative cooling & Natural Ventilation)
12:30-13:30	Lunch break		
13:30-14:30	Long Oral Presentation Session (Air Cleaning)	Long Oral Presentation Session (Ventilation & infection risk)	Topical Session (Personalized Environmental Control Systems (PECS) operation and evaluation)
14:30-14:45	Room change		
14:45-16:15	Topical Session (Post Pandemic Pontifications)	Long Oral Presentation Session (Impact factors on IAQ)	
16:15-16:45	Coffee break		
16:45-18:15	Closing Session		
20:00	Gala Dinner**		

10

Awards in closing session

- Best paper award
- Best poster award
- Student's competition award
 - Paper and poster based upon a bachelor or master's thesis
 - Presentations indicated in programme



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Thank you!

- Organizing committee
- Scientific committee
- Session chairs
- Sponsors and exhibitors

- You: the authors and presenters!




12

Tomorrow's Ventilation Solutions for Future Hospital Demands

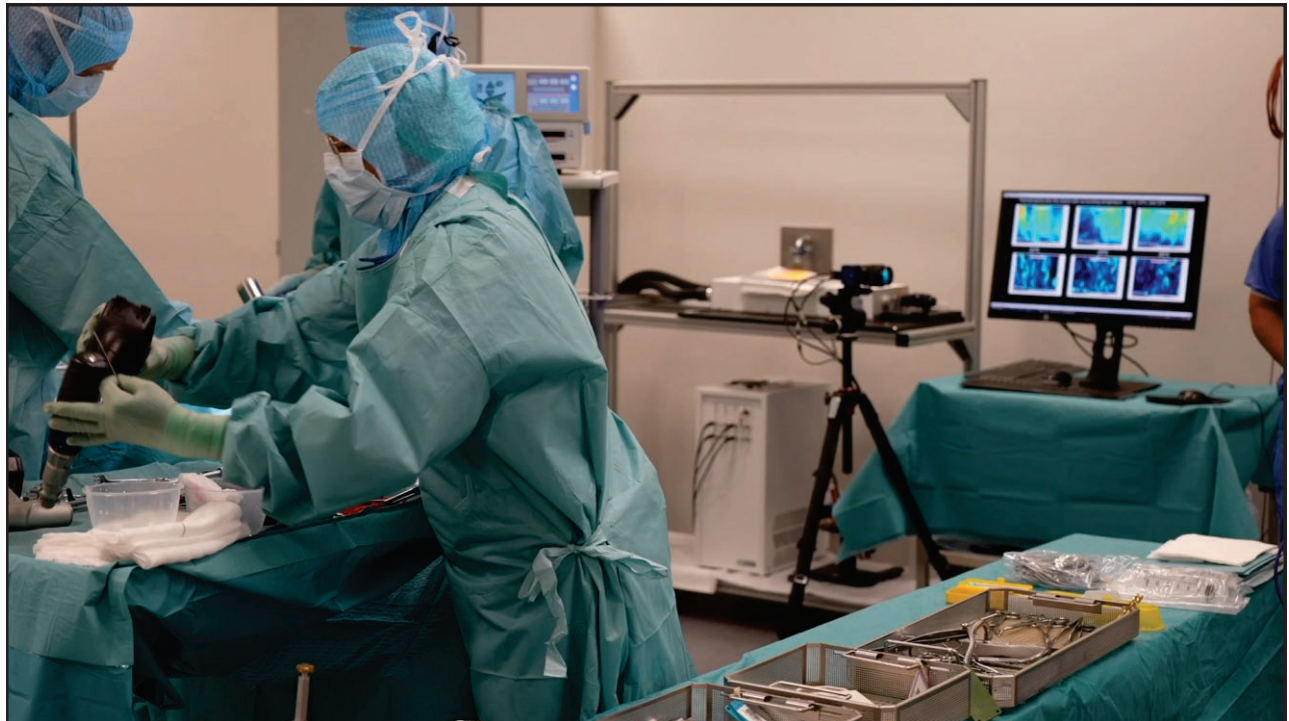
Reduction of Postoperative Surgical
Infection through Fusion of CFD and XR

Presenter / Project manager:
Trond Thorgeir Harsem
Trond.thorgeir.harsem@norconsult.com

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1




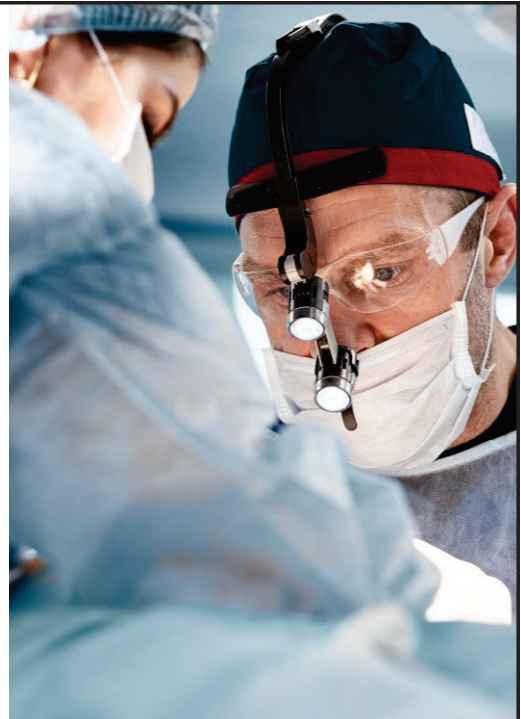
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Tomorrow's Ventilation Solutions for Future Hospital Demands

Reduction of Postoperative Surgical Infection through Fusion of CFD and XR

Presenter / Project manager:
Trond Thorgeir Harsem
Trond.thorgeir.harsem@norconsult.com

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3

Reduction of Postoperative Surgical Infection through Fusion of CFD and XR

Project partners:

NTNU - Norwegian University of Science and Technology

KTH - Royal Institute of Technology, Stockholm

CTH - Chalmers University of Technology

DTU - Technical University of Denmark

AAU - Aalborg University

Norconsult Digital, Sandvika, Norway

St. Olavs University hospital, Trondheim, Norway

The Project is financed by the Research Council of Norway


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4

Motivation

- ▶ Postoperative Surgical site Infections (POSI) are a major problem in Norway and the rest of the world
- ▶ The human body emits 10,000 particles per minute, 10% of which contain bacteria
- ▶ World Health Organization (WHO): 50% of Bacteria that contribute to POSI will be antibiotic resistant in the near future
- ▶ In Europe 800 000 is affected every year for POSI (Health First Europe Launched in 2020)
- ▶ Approximately 1 500 patients die every year due to postoperative infections in Sweden (Socialstyrelsen i Sverige)
- ▶ Compared to 324 death in car accident in 2018 in Sweden (Parastoo Sadeghian – PhD Defence 2022)

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5

Research Project Goal

Increase the understanding of the airborne particle movement by developing an Extended Reality solution to reduce Postoperative Surgical site Infections (POSI).

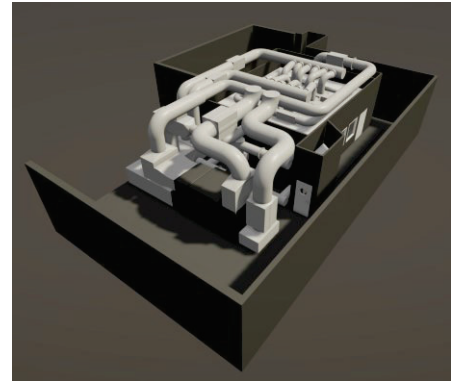



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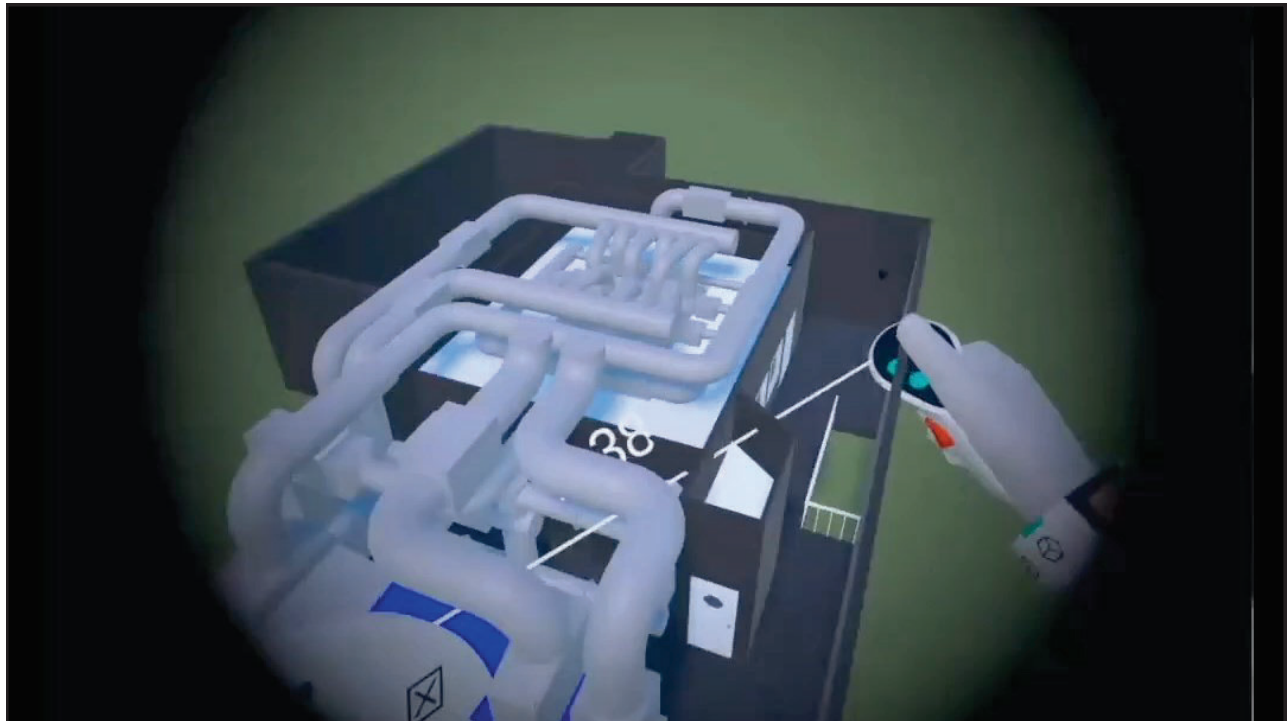
Methodology

- ▶ Build up an Operating Room Laboratory with different ventilation solution
- ▶ Computational Fluid Dynamics simulations will be validated against laboratory measurements
- ▶ Result visualization using custom developed Extended Reality Technology
- ▶ Visualization of sensor data and movement



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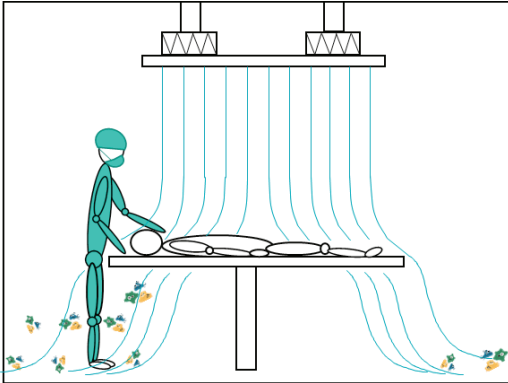
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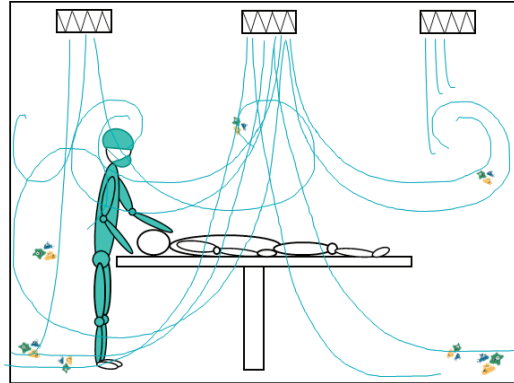
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Normal Different Ventilation Solution in Operating Room

Laminar air flow (LAF)



Mixing ventilation (MV)



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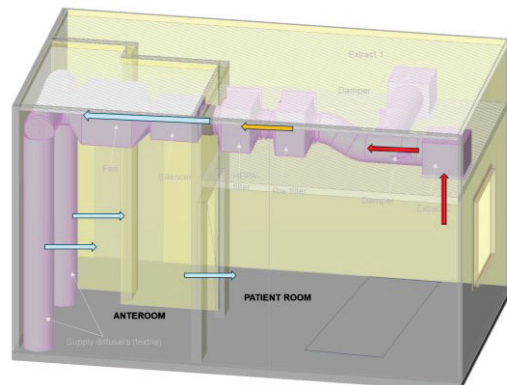
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How to Make a Ultra Clean Operating Room

Where do the bacteria come from?

- ▶ Patient
- ▶ Health care workers
- ▶ Instruments
- ▶ Particles from the outside of the operating room

- ▶ To eliminate one of this factors we include results from an earlier research project – 2017-2019
 - ▶ “Upgraded patient room”
 - ▶ Results:
 - ▶ 99% reduced particle from patient room out to Corridor



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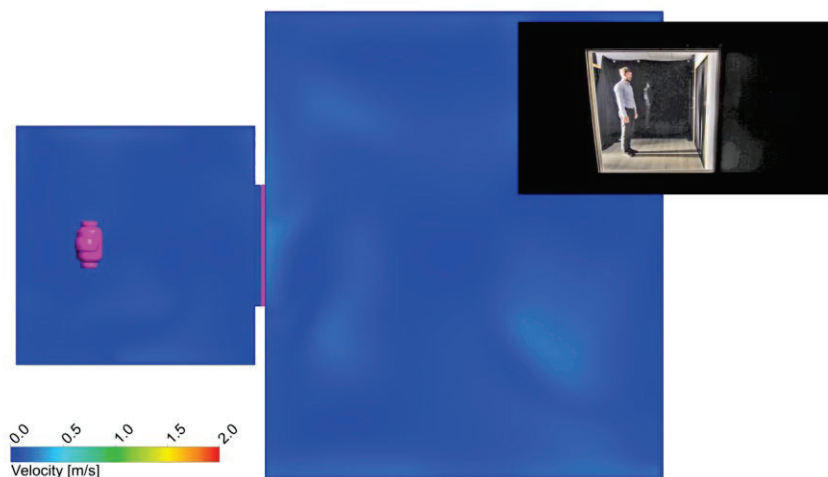
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What The World Health Organization (WHO) says about Pandemic Influenza:

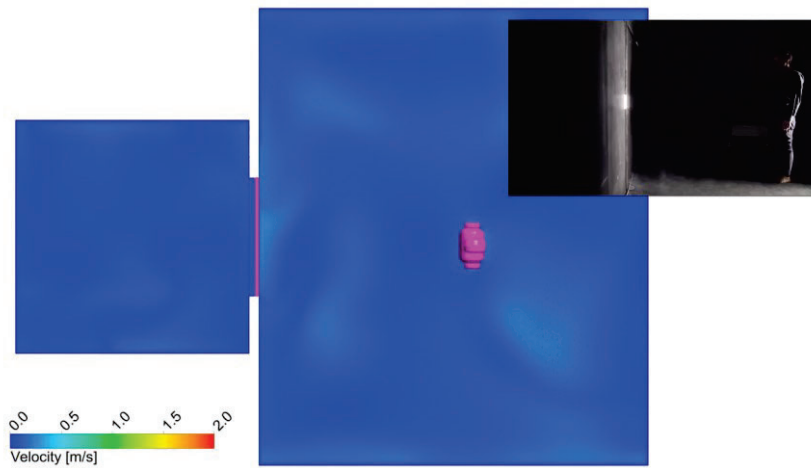
- We do not know what it will look like
- We do not know where it will come from
- We do not know who or how hard it will hit
- But it will come
- It is a question of “when” - not about “if”
- Sooner or later, the world will again be affected by a pandemic influenza



CFD and smoke visualization – entering patient room



CFD and smoke visualisation – exit patient room



13

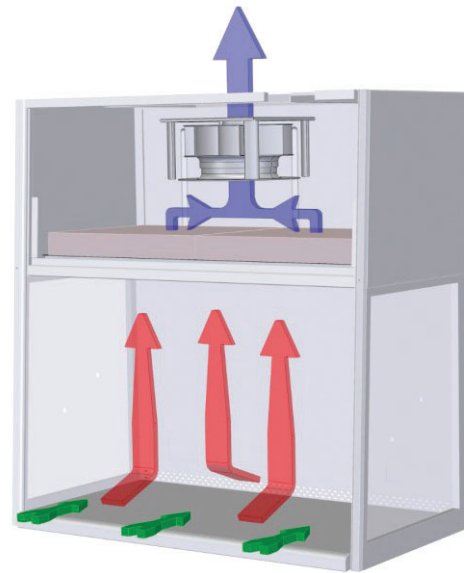
Summary of the results from the baseline cases CFD simulations and laboratory experiments

Description	Baseline cases			
	Door only (1)	Exit (2)	Entry (3. fast)	Entry (3.slow)
CFD model	755 litres	780 litres	1100 litres	680 litres
Laboratory experiments	765 litres	730 litres	800 litres	

14

Inspiration: Fume Hoods

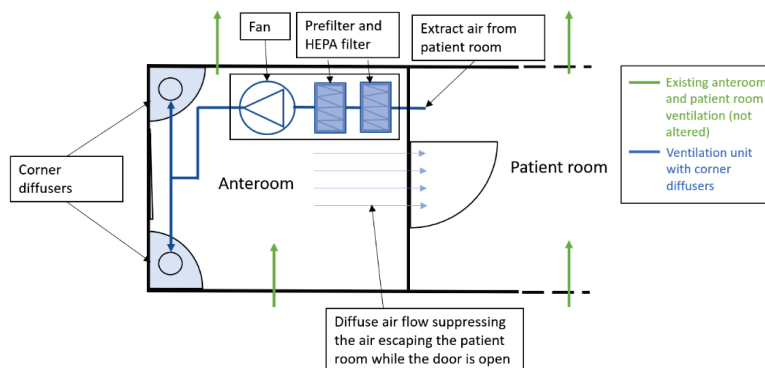
- ▶ Ventilation device that limits the exposure to hazardous or toxic fumes that are exhausted by material inside the device
- ▶ 0,4-0,6 m/s over the opening area



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15

Installing a ventilation unit into the anteroom

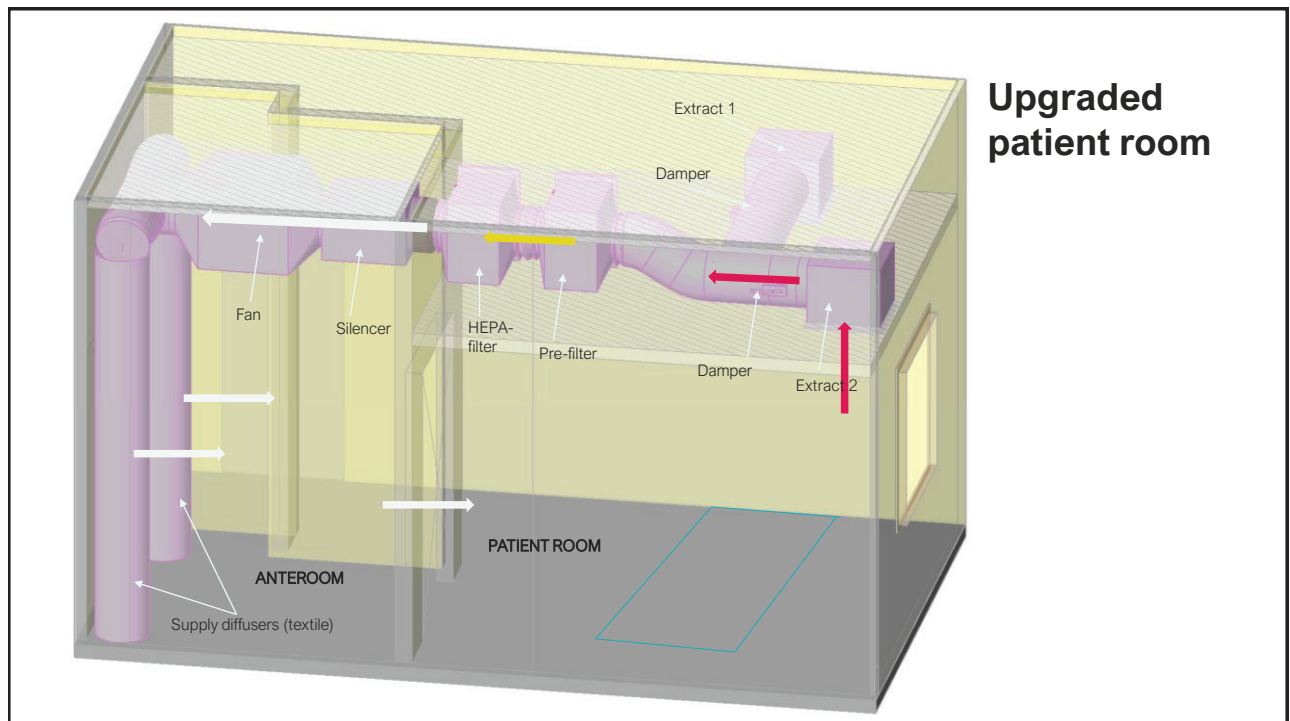


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Ventilation solution through a hinged door



17



18

Smoke visualization without recirculated air



19

**Patient room WITH
air Barrier**



20

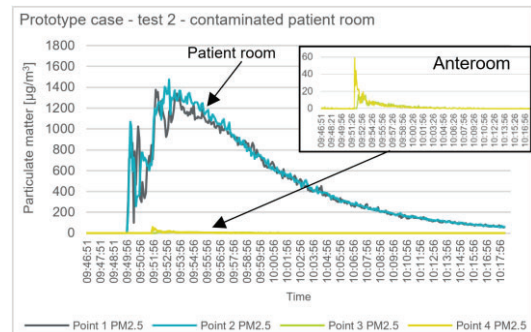
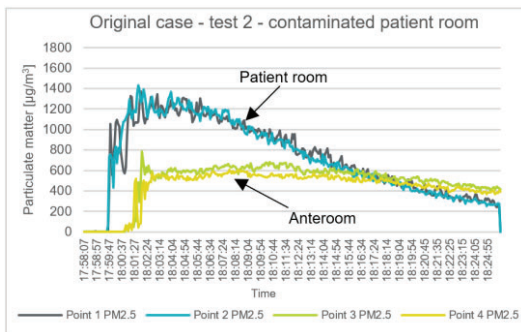
Smoke visualization with recirculated air



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Results

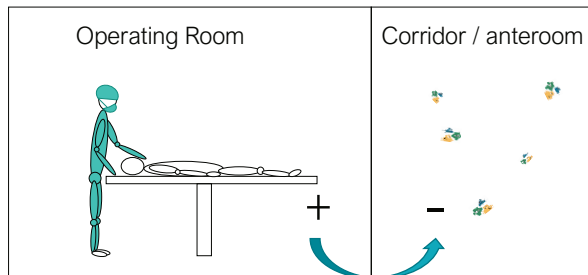
- ▶ Approximately 99% reduction of particle transfer to the anteroom with the presented ventilation solution compared to the base case



22

Pressure differential Operating Room – Corridor /Anteroom

- ▶ The Operating Room is kept at positive pressure related to their adjacent spaces to ensure that no contaminated air can enter the room by ensuring the correct air flow direction
- ▶ When opening the door between to room => **No pressure difference**



23

Reasons for Traffic Flow in an Operating Room

Typical reasons for traffic flow in an Operating Room

- 👤 Expert consultation
- 👤 Need for more instruments
- 👤 Lunch or coffee breaks
- 👤 Team member leaving or entering after incision or before closure
- 👤 Student visits

The number of door openings can rise up to *37 openings per hour* ⁷

24

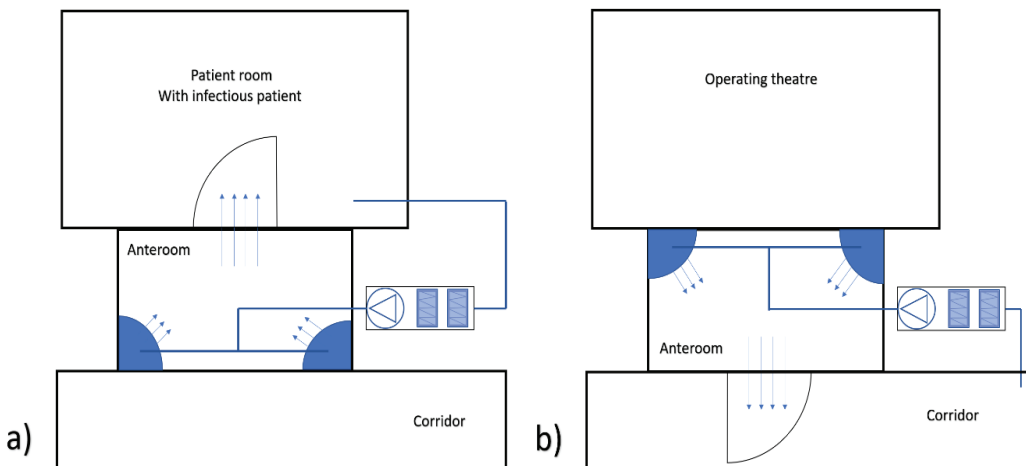
Research Question

- ▶ Is it possible to *reduce the particle migration to the Operating Room* caused by door opening and passage?



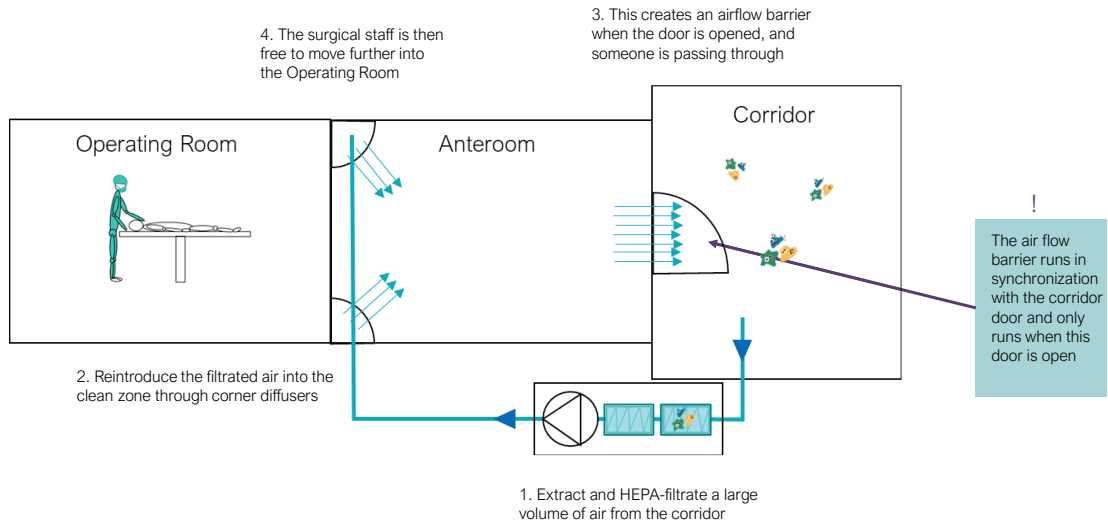
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Ida: Installation of recirculation unit in the ante room into the operation room



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Solution Transferred from Earlier Research Project



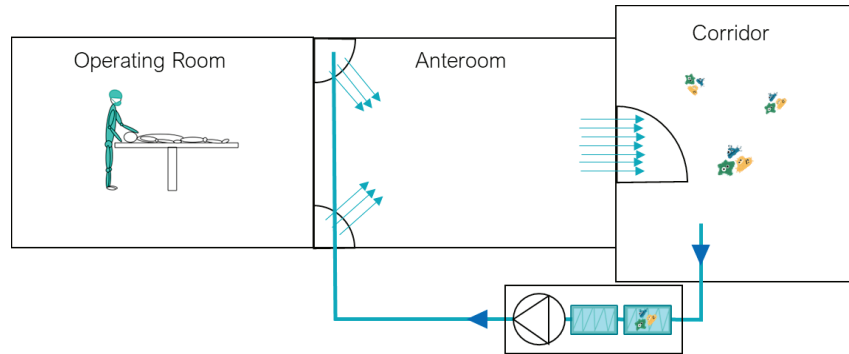
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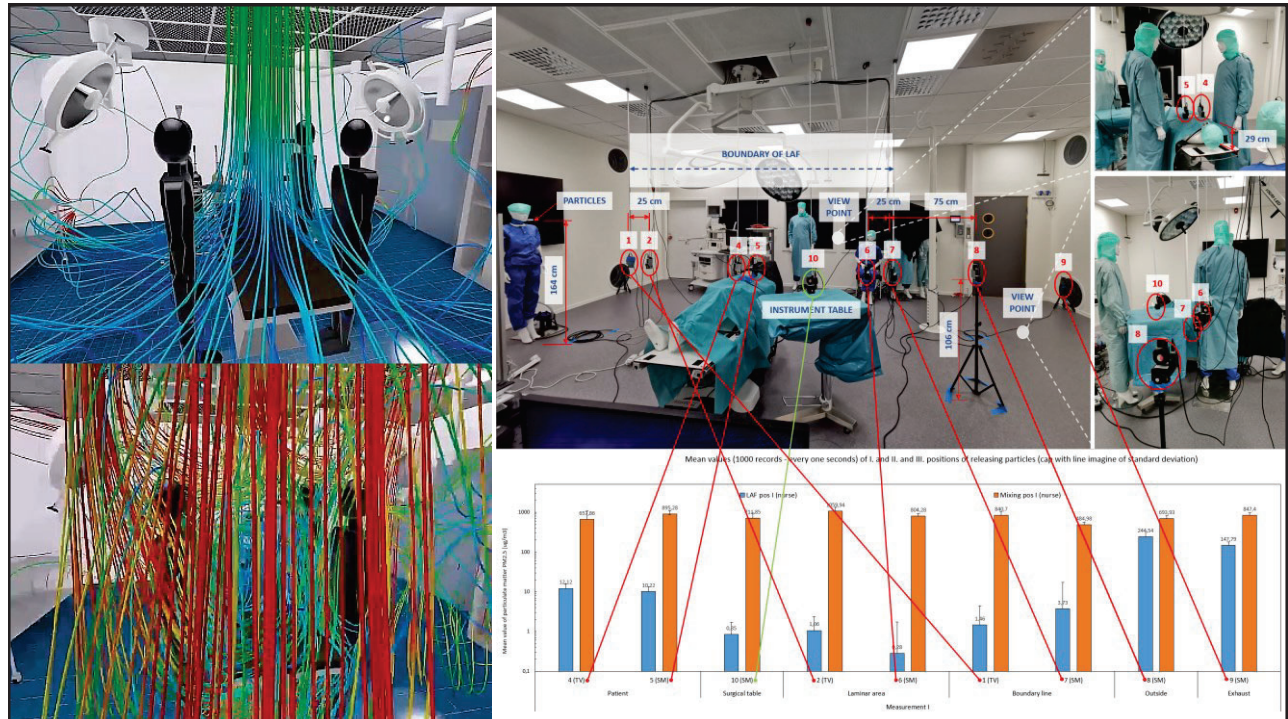
28

Where do the bacteria come from? One of four sources are eliminated!

- ▶ Patient
- ▶ Health care workers
- ▶ Instruments
- ▶ **Outside of the operating room**



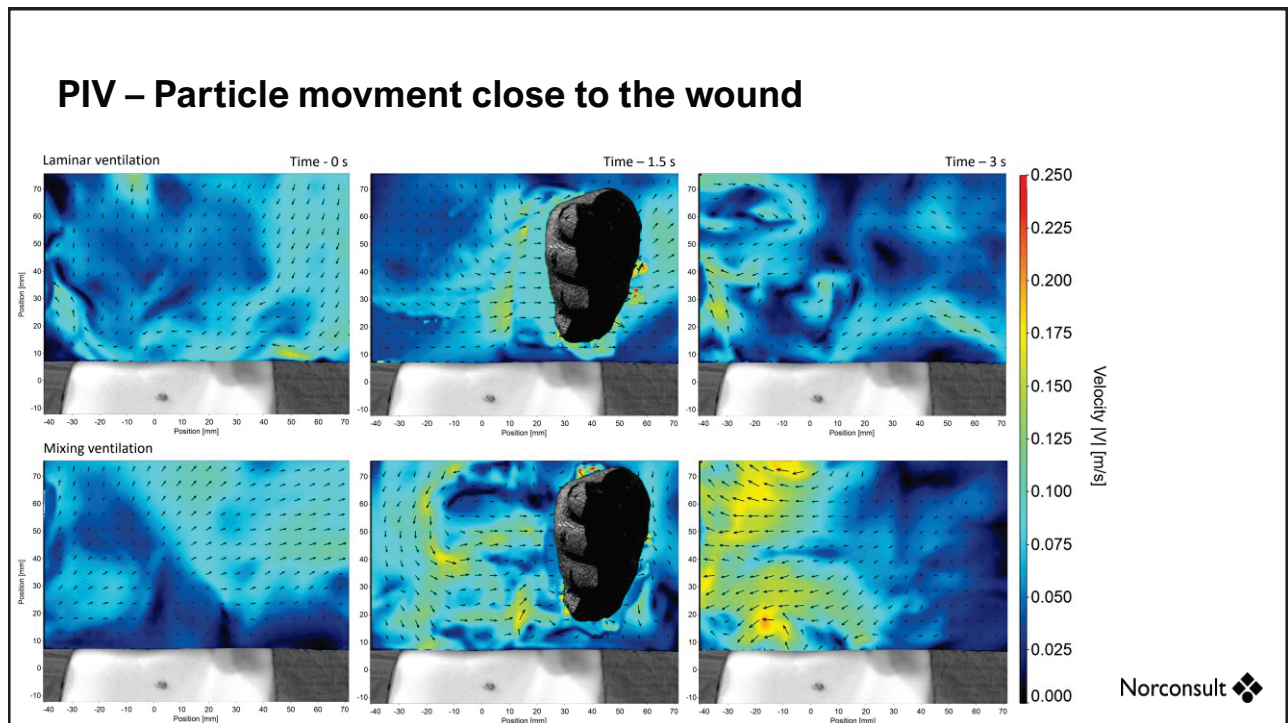
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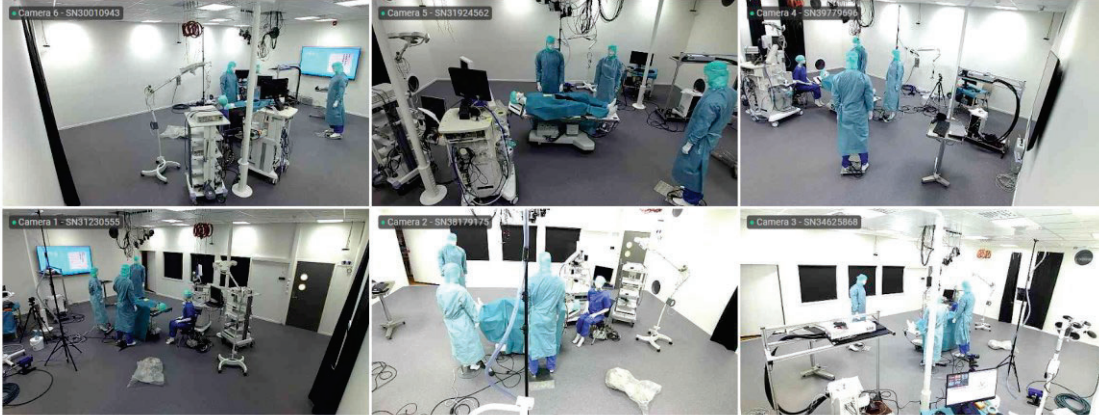
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Motion Capture 6 Stereo CAM

SKS Video Wall

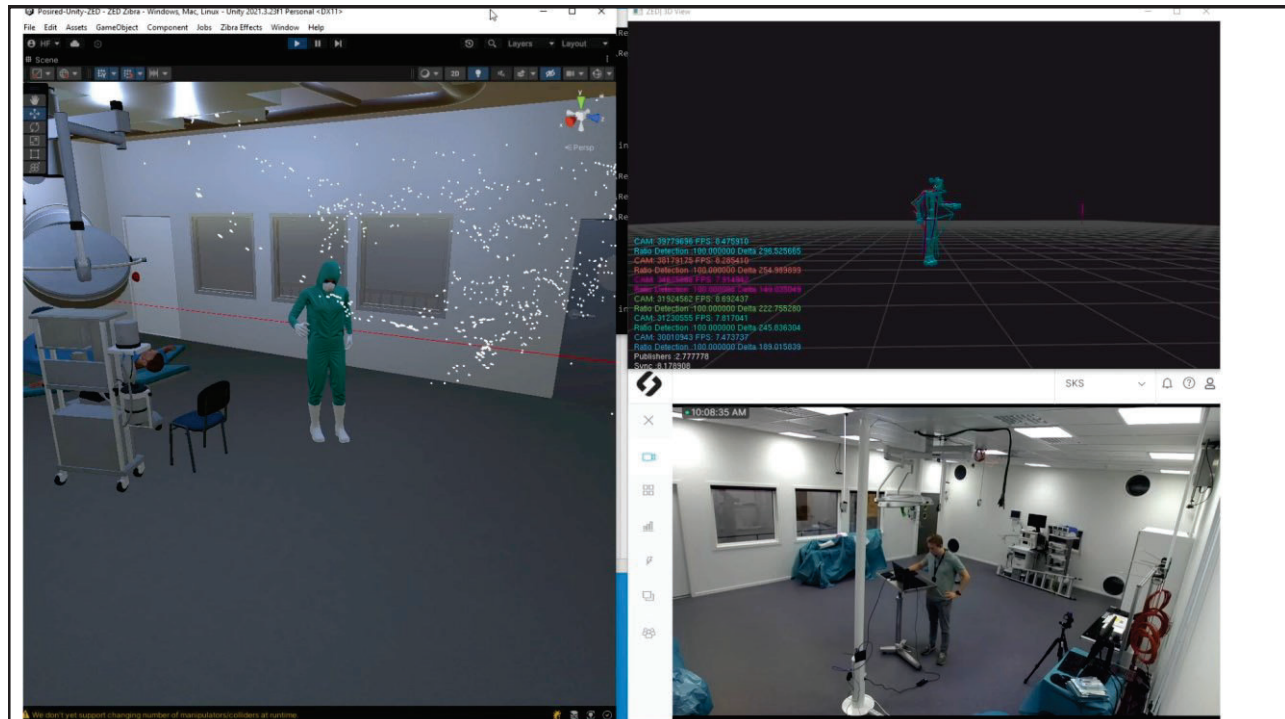
Full Screen

All devices

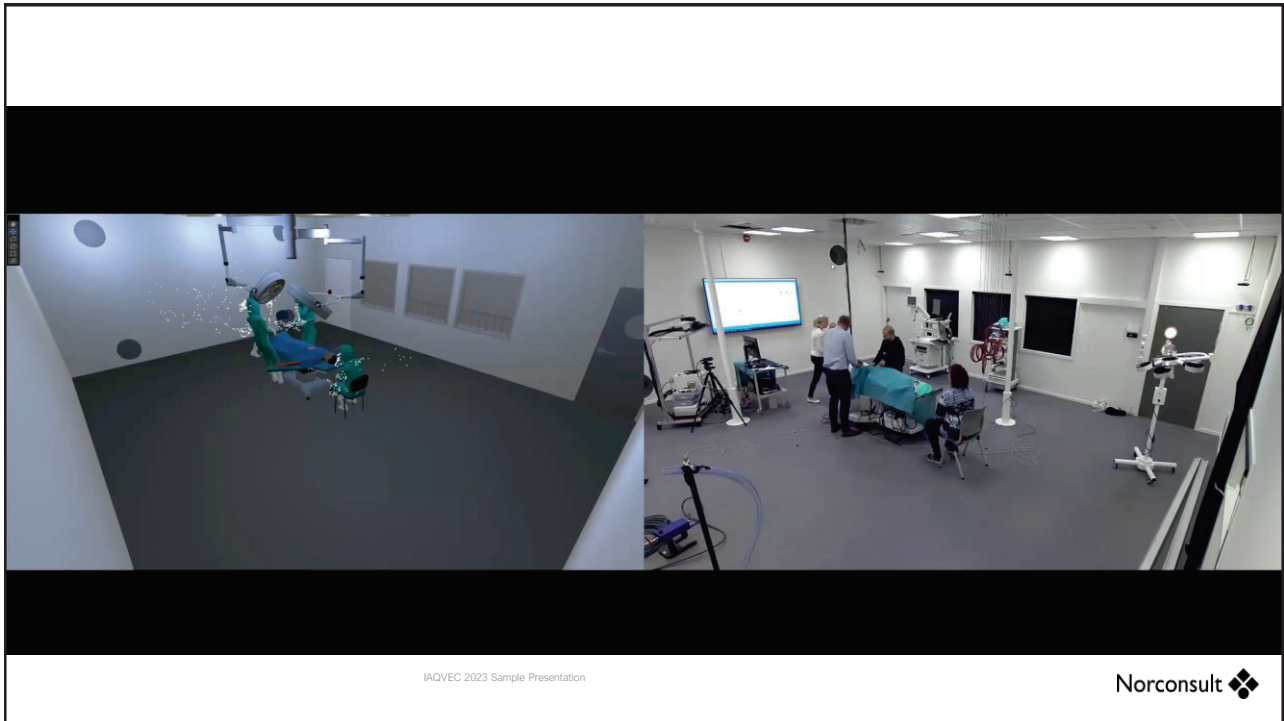


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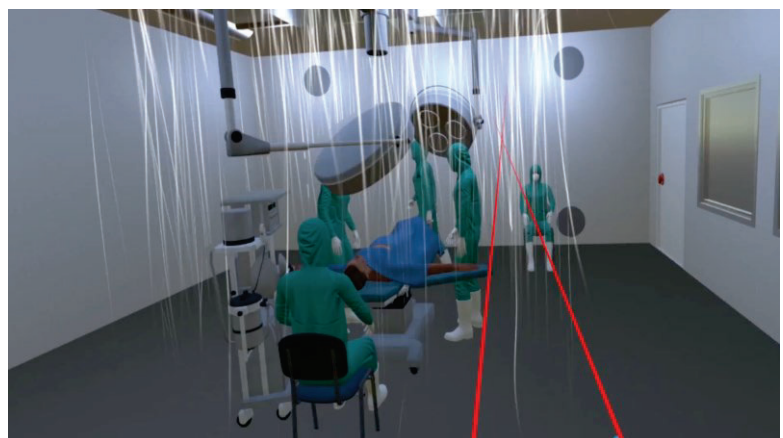
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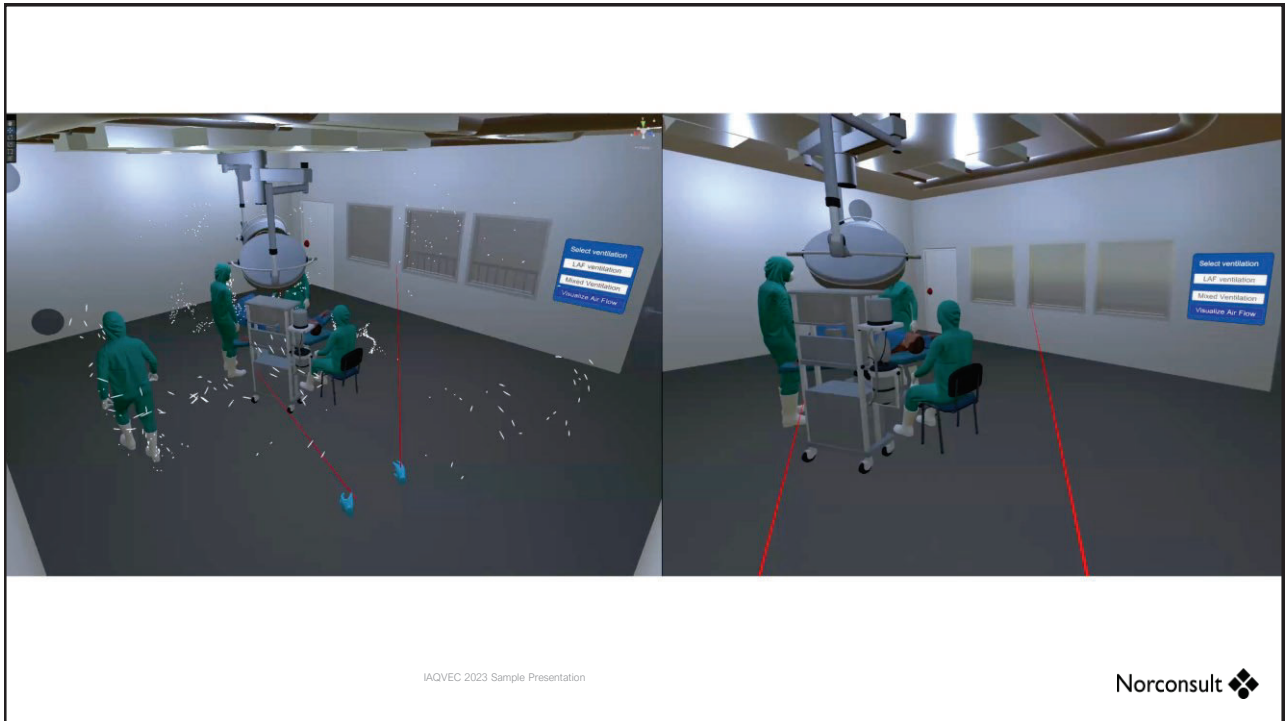
XR Solution

- ▶ Clean Air is an essential factor
- ▶ Understanding how operating personnel's movement and ventilation solution affect particle dispersion
- ▶ Visible virtual air and particle flow

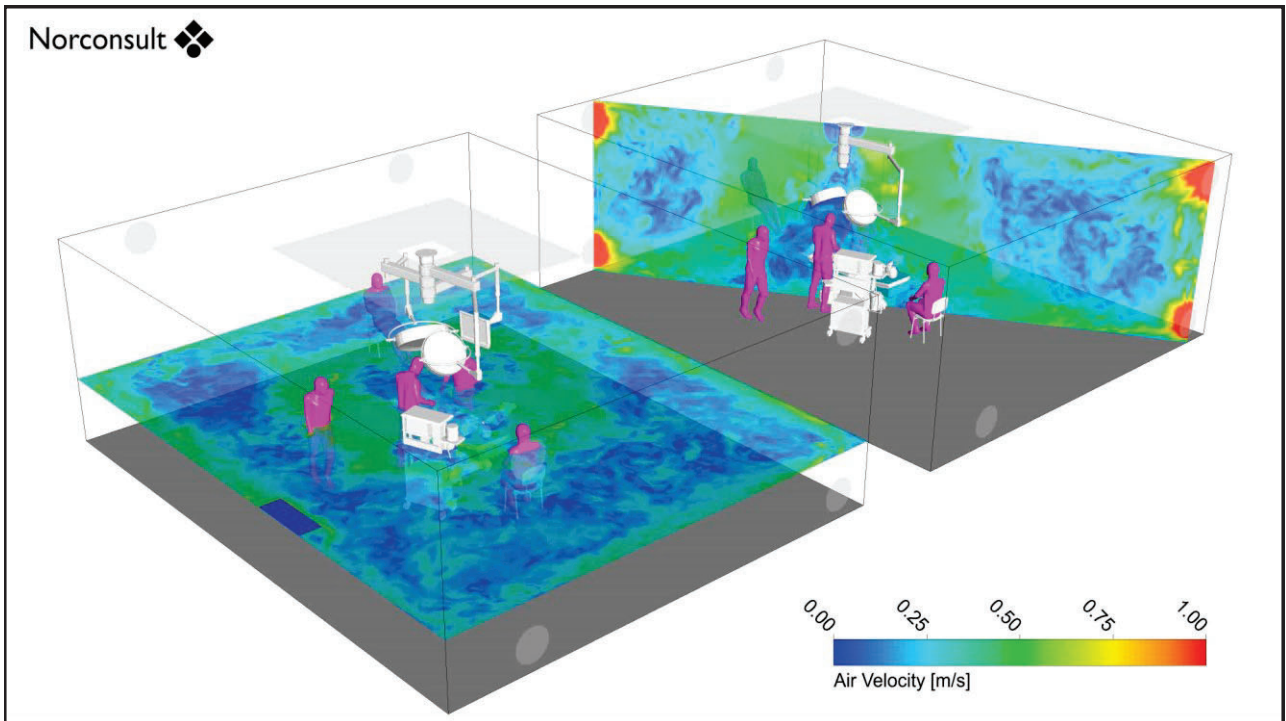


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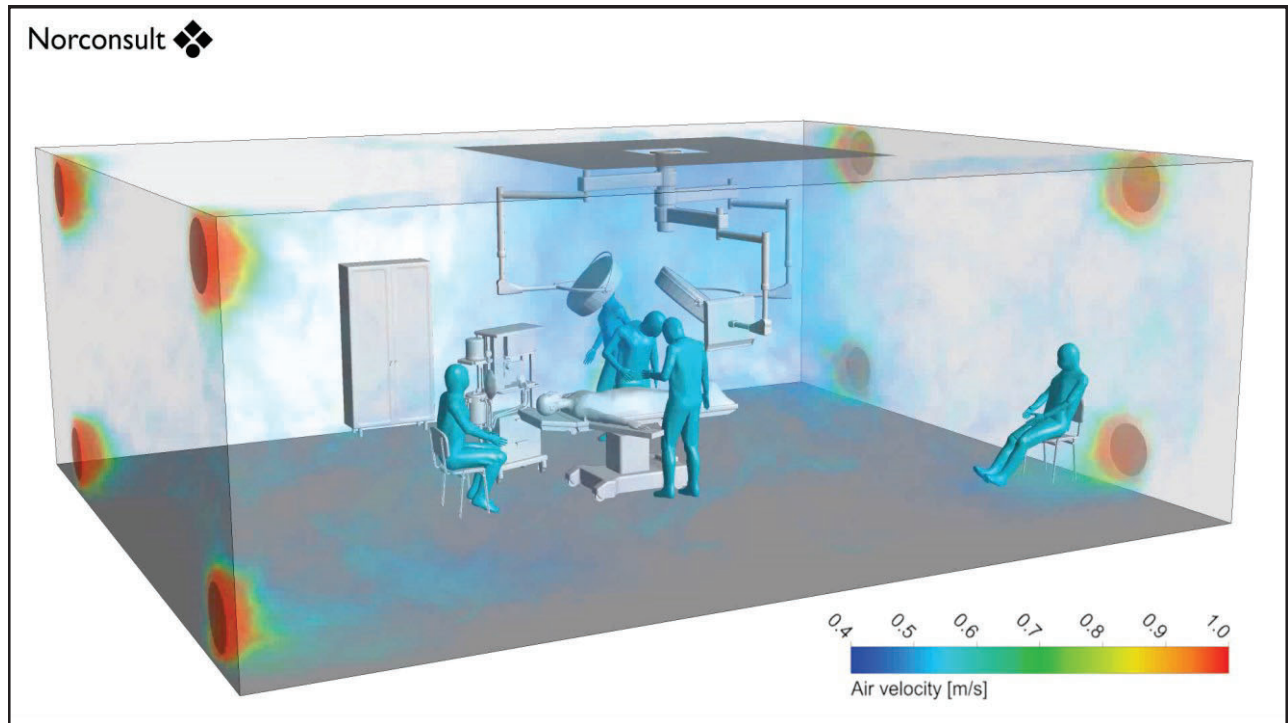
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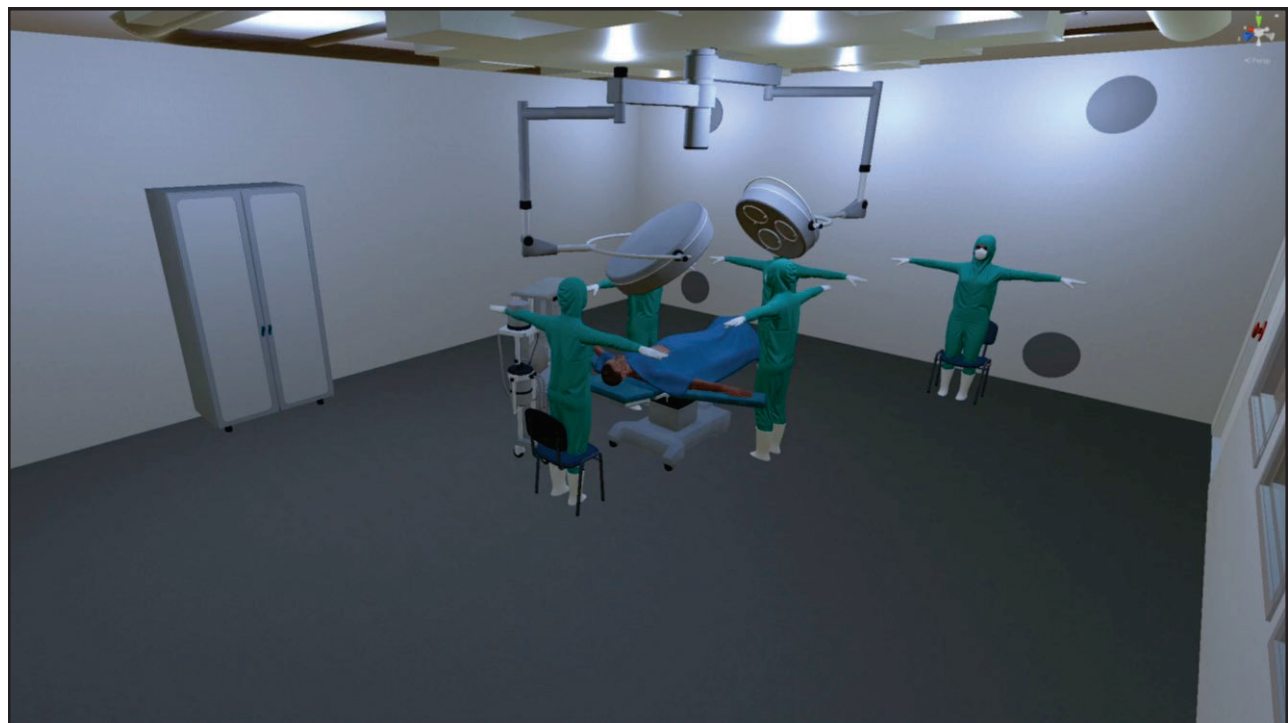
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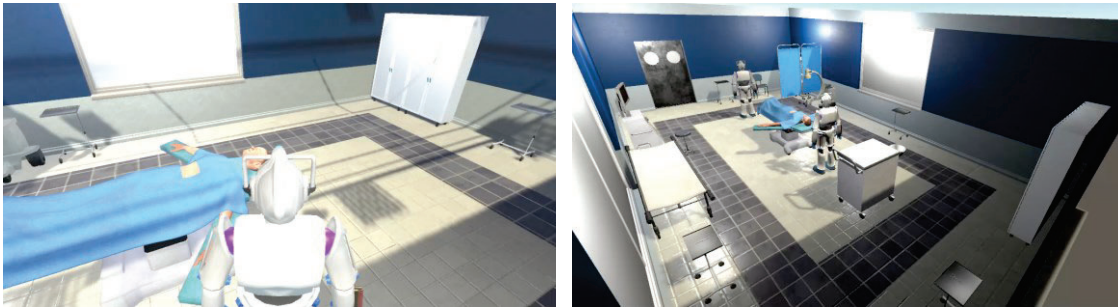
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Machine Learning

- ▶ Machine learning to make agents perform simple tasks



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Results and Conclusions

- ▶ Training of medical personnel with use of XR
- ▶ Increase the knowledge of medical persons and designers
- ▶ This knowledge will reduce the amount of Post Operative Surgical Infections
- ▶ Further research will contribute to increasing understanding of which factors affect particle concentration (read bacteria) in the patient's wound area, ie:
 - ▶ Pre- and post-surgery
 - ▶ Ventilation solutions
 - ▶ Temperature conditions
 - ▶ Behavior and habit of operating personnel



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**Thank you for
your attention**

Trond Thorgeir Harsem^{1,2}
trond.thorgeir.harsem@norconsult.com

¹ Norconsult, Sandvika, Norway
² KTH - Royal Institute of Technology, Stockholm

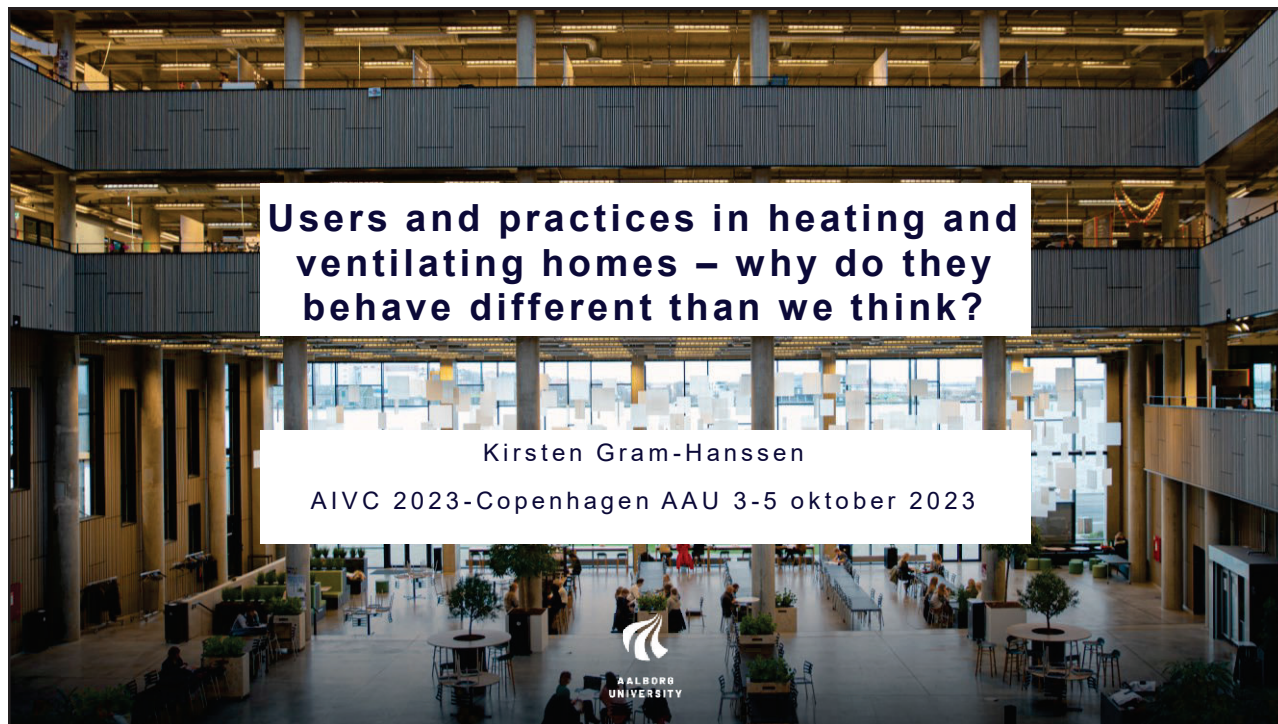


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Every day we improve everyday life



Users and practices in heating and ventilating homes – why do they behave different than we think?


Kirsten Gram-Hanssen
AIVC 2023-Copenhagen AAU 3-5 oktober 2023



1

What I will talk about....

- ➊ Healthy indoor air quality *and* reduced energy consumption are compulsory
- ➋ For both goals the interplay between users, buildings and technologies is the key
- ➌ Users are different – and different from what designers think – and we need to understand this



2

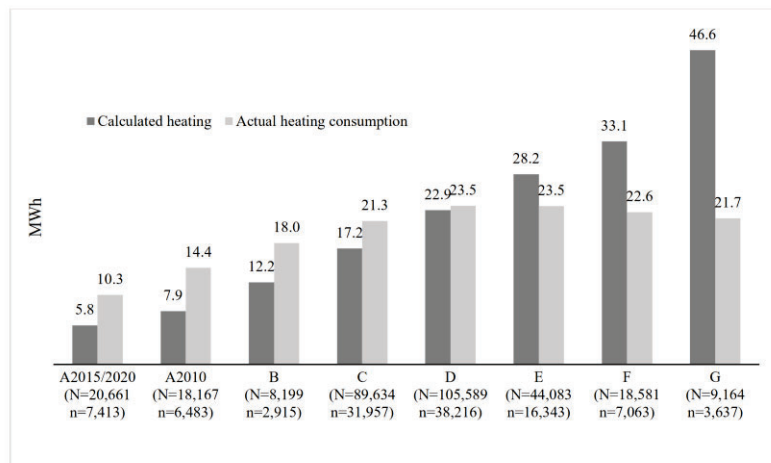
When designing technology or modeling buildings we anticipate behavior

3

But often user behavior is different from anticipations....

4

- Comparing calculated and actual heating in homes show systematic huge difference
- G-labeled homes use much less than expected - and A-labeled much more
- The building efficiency impact behavior
- This has policy relevance for reducing energy consumption



(Hansen & Gram-Hanssen, 2023).

Figure 1. Comparison of means of calculated heating demand and actual heating consumption in houses in MWh for 2019 to 2021 across labels. N=314,078 observations. n=114,022 households.

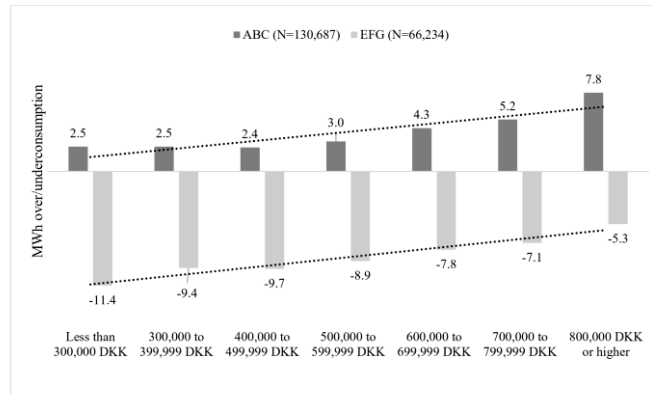
4

Do income explain all of this discrepancy?

5

- There are over-consumption in all ABC-labeled houses
- There are under-consumption in all EFG-labeled houses
- Households' economy influence how much over- and under-consumption.
- But economy do not explain all: the efficiency of the house impact behavior

Figure 4. Comparison of over- and underconsumption in MWh from lower to higher income households.



(Hansen & Gram-Hanssen, 2023).

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5

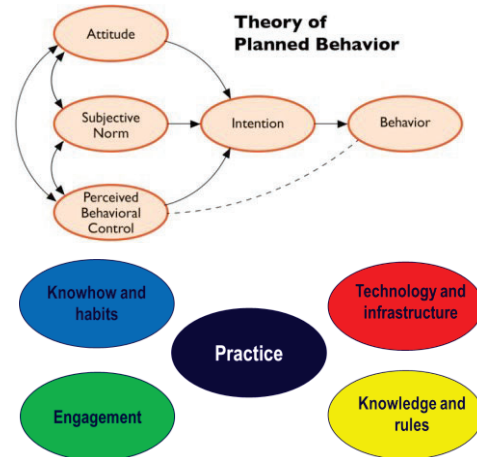
Users are important – how to conceptualise them?

6

Practice theoretical approaches *versus* Behavioural

7

- Routinised *versus* intentional
- Focus on the practices we all carry *versus* on the individual person
- Technology and infrastructure included *versus* not included as forming practices
- Understand how something become normal and change over time



(Gram-Hanssen, 2014)



7

Practices are collective – but with socio-economic variations

8

We all do the same - though with variations related to:

- Our material possessions and housing type (depend on economy)
- Routines and cultures from upbringing we (unconsciously) bring with us
- Knowledge and skills often related to specific materiality
- Much of this relates to gender, education and age



(Gram-Hanssen, 2021)



8

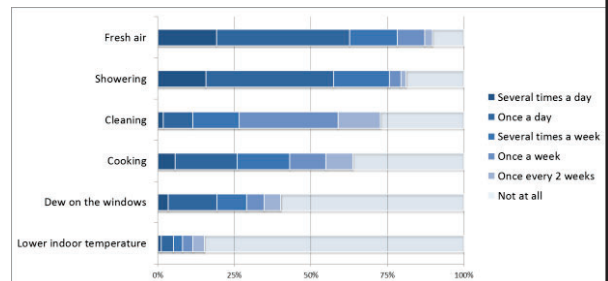
How often have you aired and why?as part of routines and daily practices

9

Little socio-demographic variation:

- Age, income and education show no significant correlations
- Families with children air more often
- Households with immigrant or descendant air more often
- The age of the house (and thus air-tightness) *do not* influence airing practices

	Which of the following statements fits your way of airing best?
I am airing after a routine	38% (464)
I do not have a fixed routine, but airing out when I think there is need for it	55% (671)
3. Don't know or missing	7%



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(Hansen, et al 2018)

9

Changing routines of airing and heating by means of payment

10

- An experiments where residents pay for poor indoor climate rather than for heating
- Cloud based solution with indoor climate measurements of CO₂, temperature, and humidity
- To prevent people from having an unhealthy indoor air quality, e.g. turning down too much the heat and airing too little to save money



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(Gram-Hansen, et al 2023)

10

Indoor climate measurements: Feedback and payments to influence users

11

- High CO₂, temperature, and humidity mean more payment; but also low temperature and low humidity (to prevent overventilation/waste of heat) means more payment

- Mixed experiences:
 - › Better communication is needed about the system
 - › Measurements can be good for the dialog between resident and building managers
 - › Some understood the system some did not
 - › Some got angry with the ideas

Table 1. Examples of specifications of limits for indoor climate measures used for calculating heating cost distribution.

	Good – basic payment	Less good – additional payment	Poor – higher additional payment
Temperature, °C	18-21	16-18 and 21-23	<16 and >23
Humidity, %RH	30-50	20-30 and 50-60	<20 and >60
CO ₂ concentration, ppm	<800	800-1000	>1000



(Gram-Hanssen, et al 2023)



11

Are smart technology a solution to “user-problems” ?

12

Discrepancies between professional's view and real-life users

13

Professionals tend to think that:



Users tend to:

"The less occupant interfere with regulating the better. We should automate and control it for them"

"The more automation the better comfort and convenience for the user"

"Ordinary people are not interested and do not understand HVAC technologies"

Experience systems they don't understand, and which works contra intuitive

Experience systems failure and problems of interoperability

Find workarounds if the automation do not work as they prefer or they do not understand it



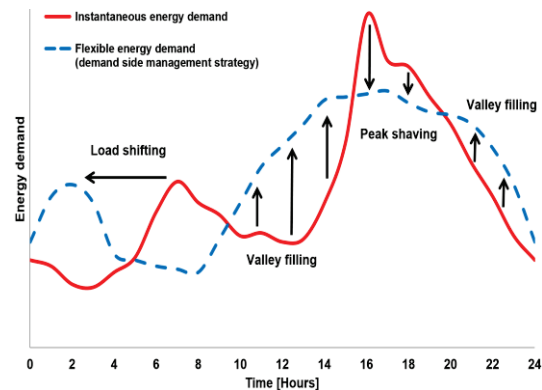
(Andersen et al, 2022; Aagaard, 2021)

13

It can get even more complicated in the future energy system...

14

- The future energy system will rely on renewable energy – flexible demand and load shifting will be increasingly relevant
- Buildings can serve as short time flexibility generator – including preheating and short time turning down heating
- Will involve even more complexity to HVAC management and the interaction between technologies, buildings and users.

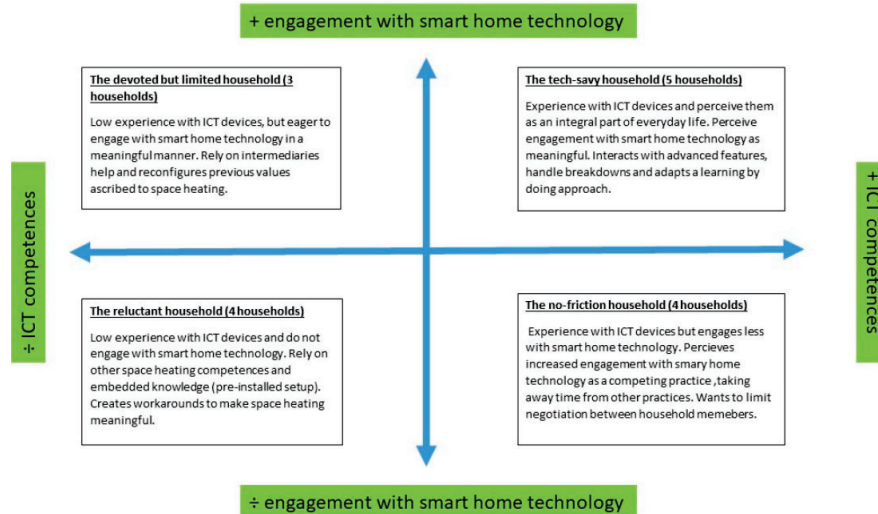


(Larsen & Gram-Hanssen, 2020)

14

Different types of users of smart control

15



(Larsen & Gram-Hanssen, 2020)

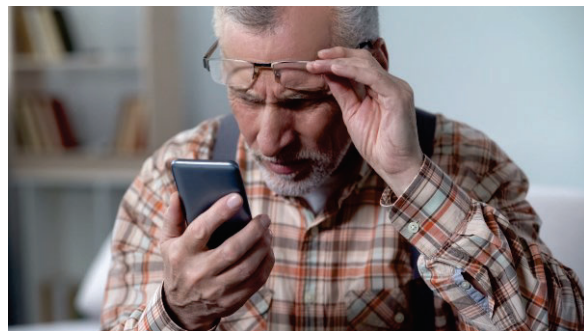
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15

Socio-economics of smart tech competences and interest

16

- The older generation will (long time ahead) not be as good as the young ones to understand and use smart tech
- The most tech interested are male. Not that females don't have competences - or that all male are tech nerds
- If AI should be part of the solutions, we need systems that work for *all* users



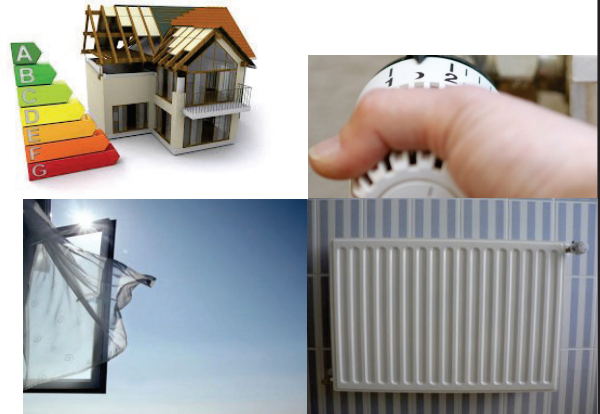
(Strengers et al, 2022)

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16

What I tried to say

- ❶ Users are important – we will not achieve healthy indoor air quality or reduced energy consumption without involving the users
- ❷ Technology and behavior cannot be separated
- ❸ User are different when it comes to competences and engagements
- ❹ Real knowledge, not assumptions, of users should be included already in the design of new technologies and building system



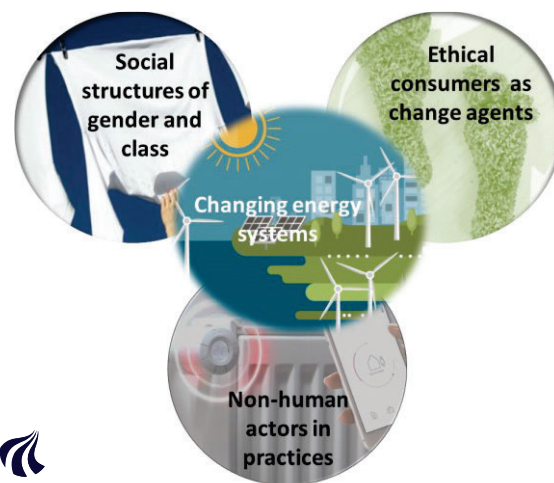
17

The project behind

18

ERC advanced grant eCAPE:
New Energy Consumer roles and smart technologies – Actors, Practices and Equality. Grant agreement number 786643

www.ecape.aau.dk



18

References

19

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- Strengers, Y., Gram-Hanssen, K., Dahlgren, K., & Aagaard, L. K. (2022). Energy, emerging technologies and gender in homes. *Buildings and Cities*, 3(1), 842–853.



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> WHAT WE KNOW ABOUT SMART VENTILATION

Dr Gaëlle Guyot, France

AIVC Conference 2023, 4-5th October, Copenhagen, Keynote

1

AGENDA

- Context
 - About the definition
- What we know
 - About the history of the terminology
 - About the strategies
- What we need
 - About the performances

2

AGENDA

- Context
- What we know
- What we need

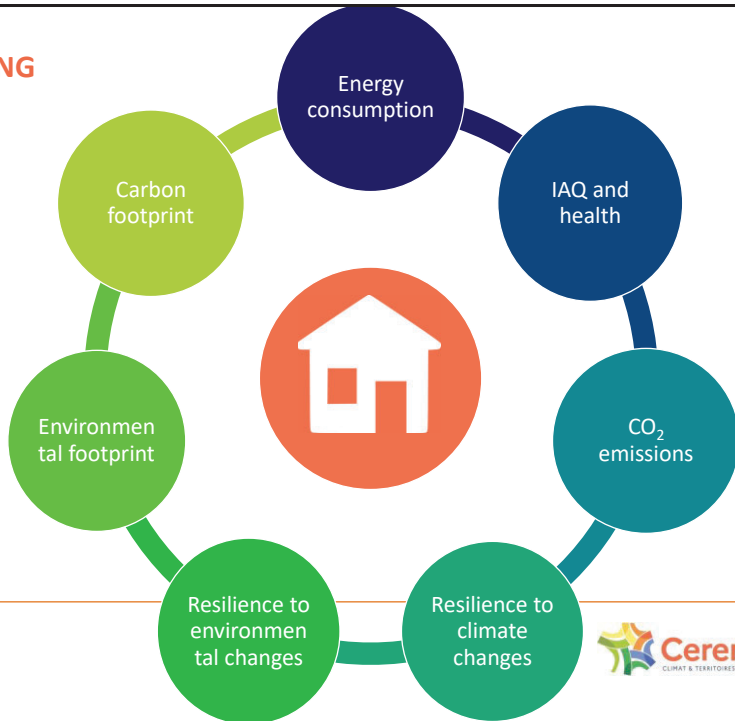
From researchers
From industrials
From policy makers

3

> **CONTEXT**

4

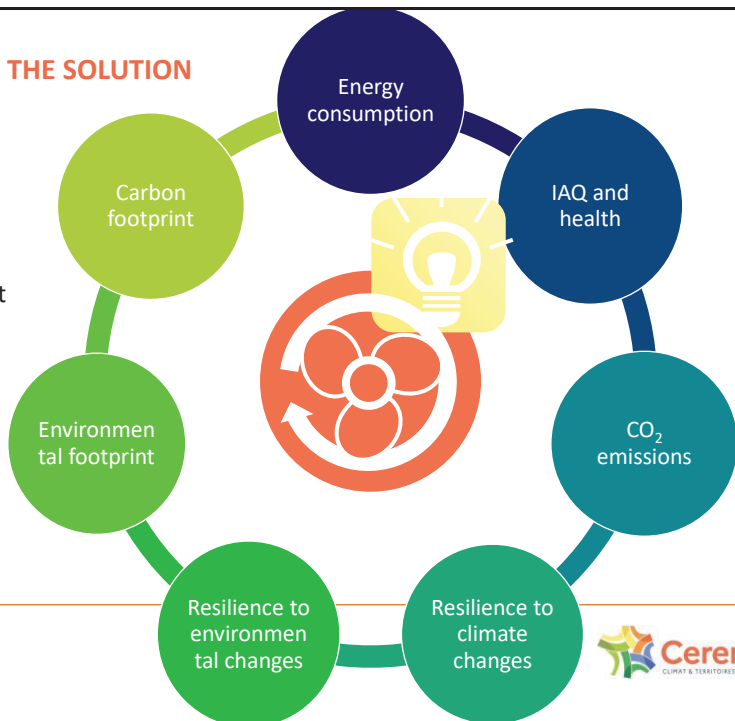
THE BUILDINGS' SECTOR IS FACING MULTIPLE CHALLENGES



5

SMART VENTILATION IS PART OF THE SOLUTION

Ventilation in buildings is at the heart of all these challenges



6

IEA-EBC ANNEX 86

The IEA-EBC Annex 86 : Energy Efficient Indoor Air Quality Management in Residential Buildings
ST4-smart ventilation : pushing an international effort about the promotion of smart ventilation strategies

IEA EBC HOME LINKS SEARCH SITE MAP EBC-LOGIN

EBC
Energy in Buildings and
Communities Programme

HOME ABOUT SUBTASKS PUBLICATIONS PARTICIPANTS NEWS MEETINGS MEMBER AREA

IEA EBC - Annex 86 - Energy Efficient Indoor Air Quality Management in Residential Buildings

From the overview of the state of the art, it is clear that the issues raised in the previous section can't be solved directly from existing knowledge. Partial answers to each of these issues are available, but a new annex is needed to address the gaps and integrate the available solutions in a coherent and operable rating method.

International collaboration is a prerequisite for this effort since market access for innovative IAQ management.

ANNEX INFO & CONTACT
Status: Ongoing (2020 - 2025)
OPERATING AGENT
Dr Jelle Laverge
Assistant Professor
Ghent University

7

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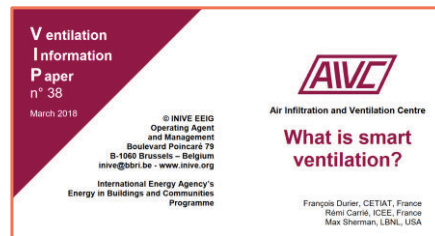
Cerema
CLIMAT & TERRITOIRES DE DEMAIN

> WHAT WE KNOW ?

8

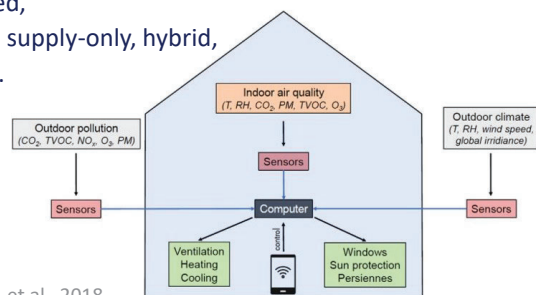
DEFINITION OF SMART VENTILATION

“Smart ventilation is a process to continually adjust the ventilation system in time, and optionally by location, to provide the desired IAQ benefits while minimizing energy consumption, utility bills and other non-IAQ costs (such as thermal discomfort or noise). (...)”



DEFINITION OF SMART VENTILATION

- Demand-controlled ventilation (DCV) = a specific subset of smart ventilation
- A wide range of systems currently available in the literature and on the market depending on
 - the type of sensing parameters : **indoor**, humidity, occupancy, VOC, **outdoor** climate, pollution, temperature, etc.,
 - the type of sensing combinations,
 - The type of air terminal devices (ATD),
 - the type of installations: centralized/decentralized,
 - the types of techniques: balanced, exhaust-only, supply-only, hybrid,
 - and the types of controls and control algorithms.



Schieweck, et al. 2018.

Possible manipulated variables (indoors and outdoors) for the control of ventilation, indoor climate and indoor pollutants

HISTORY

- “smart ventilation” first used fairly recently by LBNL researchers (Walker et al., 2014; Less and Walker, 2016; Lubliner et al., 2016)
- These terms have been increasingly used since
- DCV emerged after the oil crisis of the early 1980 (Anon, 1983; Barthez and Soupault, 1984; Nicolas, 1985).
- More recently, a favorable context – pushed by the 2003 EPBD in Europe
 - with more than 20-30 DCV systems approved and available in countries such as Belgium, France and the Netherlands (Guyot et al., 2018b).

11

PERFORMANCES OF RESIDENTIAL SMART VENTILATION – FIRST STATE OF THE ART IN 2018

- With various smart ventilation systems based on CO₂-, humidity-, combined CO₂- and TVOC-, occupancy-, outdoor temperature-controlled ventilation, **energy savings up to 60% could be obtained without compromising, and sometimes improving, IAQ**
 - 38 studies from 1979 to 2016

⇒ a clear potential for improve IAQ and save energy

12

PERFORMANCES OF RESIDENTIAL SMART VENTILATION – STATE OF THE ART IN 2023

- On-going
- In the Framework of the IEA-EBC Annex 86
- And of a new AIVC Technical note



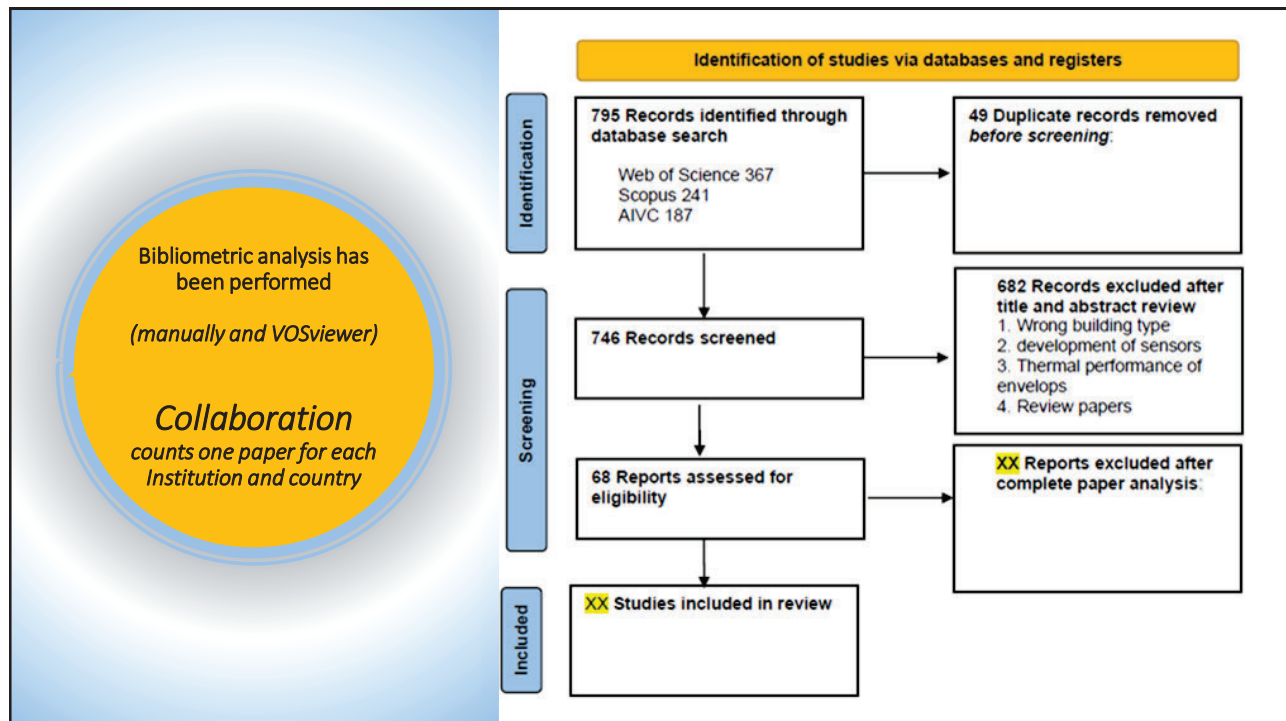
Dr Daniela MORTARI, PUCPR, Cerema, Univ. Savoie Mont Blanc, Brasil-France

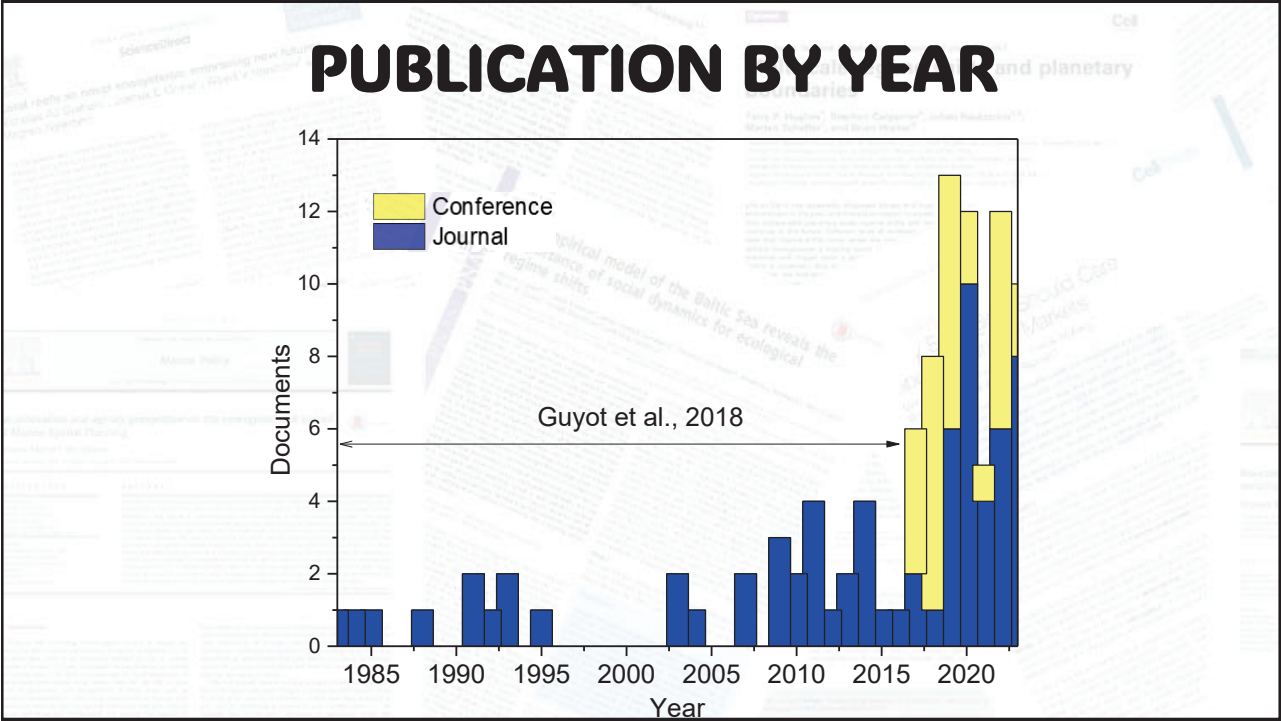


Dr Yu WANG, BRANZ, New Zealand



Keynote – smart ventilation, AIVC conference 2023





15

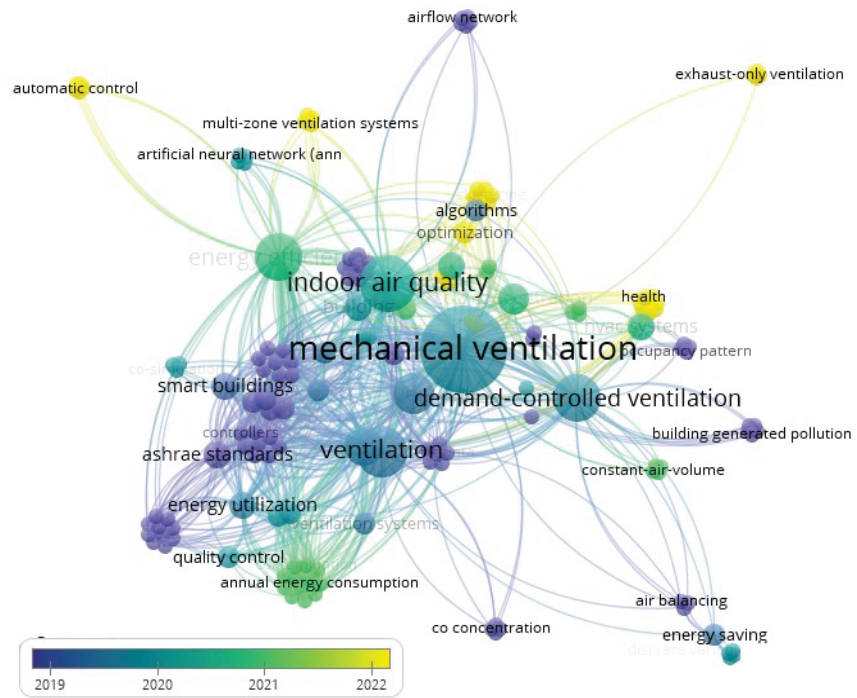
RANK OF INSTITUTIONS ACCORDING TO THE NUMBER OF ARTICLES

RANK	INSTITUTION	DOCUMENTS	PROPORTION
1	Ghent University	15	22.1%
2	Lawrence Berkeley National Laboratory	8	11.7%
3	Renson Ventilation, Belgium	6	8.8%
4	Technical University of Denmark	6	8.8%
5	CEREMA, Lyon, France	4	5.8%
6	The Ohio State University	4	5.8%
7	Belgian Building Research Institute (BBRI)	3	4.4%
8	Université Savoie Mont Blanc	3	4.4%
9	Nanyang Technological University	3	4.4%
10	Fraunhofer Institute for Solar Energy systems (ISE)	2	2.9%
11	AERECO SA France	2	2.9%
12	CETHIL-EDF Joint Laboratory, BHEE, France	2	2.9%
13	City University of Hong Kong	2	2.9%
14	Dankook University	2	2.9%
15	Flemish Institute for Technical Research, Belgium	2	2.9%
16	National Institute of Standards and Technology, USA	2	2.9%
17	Norwegian University of Science and Technology	2	2.9%
18	Seoul National University	2	2.9%
19	Southeast University, China	2	2.9%
20	Université Grenoble Alpes	2	2.9%
21	Université de Lyon	2	2.9%
22	Vrije Universiteit Brussel	2	2.9%

16

VOSViewer Results

Keywords
occurrence by year



17

> **WHAT WE NEED ?**

18

AGENDA

- Context
- What we know
- What we need

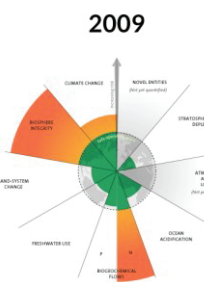
From researchers
From industrials
From policy makers



19

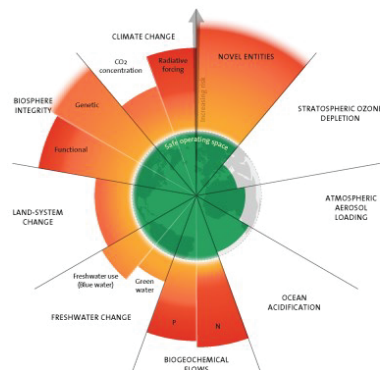
1- SOBER SYSTEMS ADAPTED TO THE CHALLENGES OF OUR DECADE

André Souch | #232860294

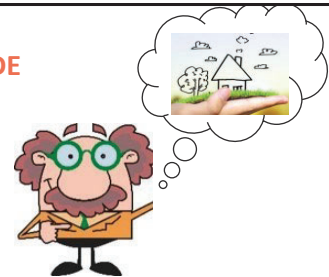


3 boundaries crossed

2023



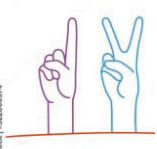
6 boundaries crossed



- At this time on the history, we need ventilation systems :
 - Sober (energy and resources),
 - Reliable,
 - With long-term performances **over the ventilation system's lifetime – 20 years** (Durmisevic, 2006; Feist et al., 2020),
- Many of them already exist !

20

Adeline Sirek | 0322060578



2-ROBUST AND IMPROVED PERFORMANCE-BASED ASSESSMENT SCHEMES

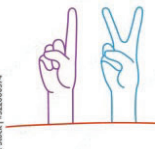


- Remains a barrier in many countries !
- Final report of the IEA EBC Annex 9 (1982-1986) described the 2 approaches for ventilation standards:
 - The prescriptive approach, in which an outdoor air flow rate is stated
 - The air quality approach, in which a limiting maximum pollutant concentration is defined and the building designer or user, is required to supply sufficient air to ensure that this is not exceeded = Performance-based approach but not only IAQ



Spekkink, D. 2005. Key note presentation on PeBBu, CIB Conference, Helsinki, 2005

Adeline Sirek | 0322060578



2-ROBUST AND IMPROVED PERFORMANCE-BASED ASSESSMENT SCHEMES



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"Turn left at the next traffic lights, then take the fourth street to the right, go right ahead at the first roundabout, turn to the right at the second roundabout and keep the left lane, then turn"

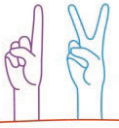


"To the airport!"



Spekkink, D. 2005. Key note presentation on PeBBu, CIB Conference, Helsinki, 2005

Spekkink, D. 2005. Key note presentation on PeBBu, CIB Conference, Helsinki, 2005



2-ROBUST AND IMPROVED PERFORMANCE-BASED ASSESSMENT SCHEMES

- To date, however, most national regulations and standards => the prescriptive approach
- In the field of smart ventilation
 - SV must undergo procedures/calculations in order to demonstrate their performances
 - before being authorised or used in buildings,

Ventilation Information Paper
n° 39
March 2019

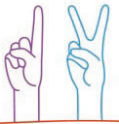
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International Energy Agency's
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Programme

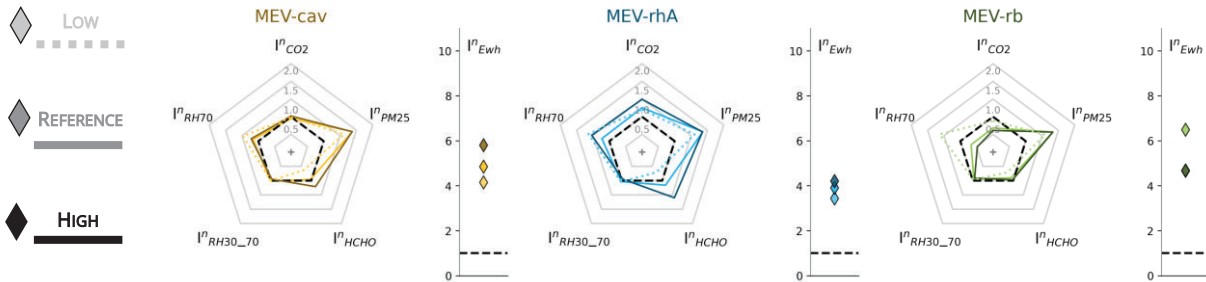
AIVC
Air Infiltration and Ventilation Centre

A review of performance-based approaches to residential smart ventilation

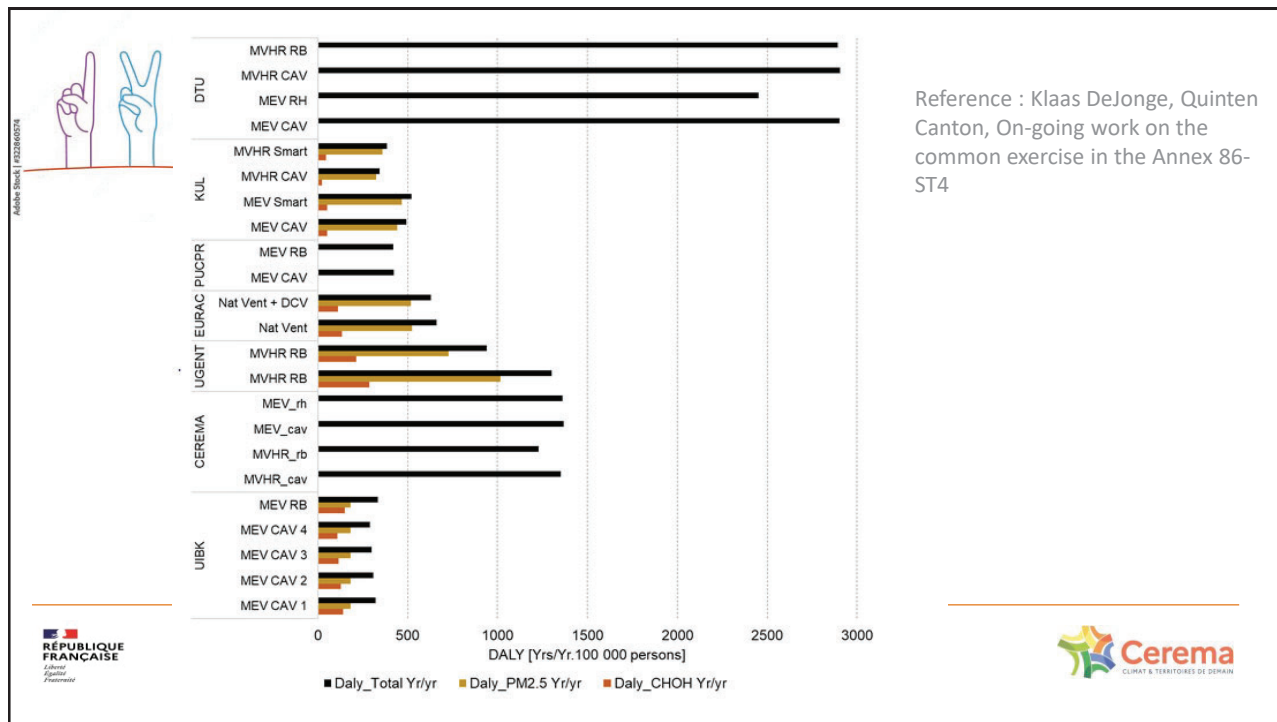
Gaëlle Guyot, Cerema, University SMB, France
Iain Walker and Max Sherman, LBNL, USA
Pilar Linares and Sonia Garcia Ortega, CSIC, Spain
Samuel Caillou, BBRI, Belgium



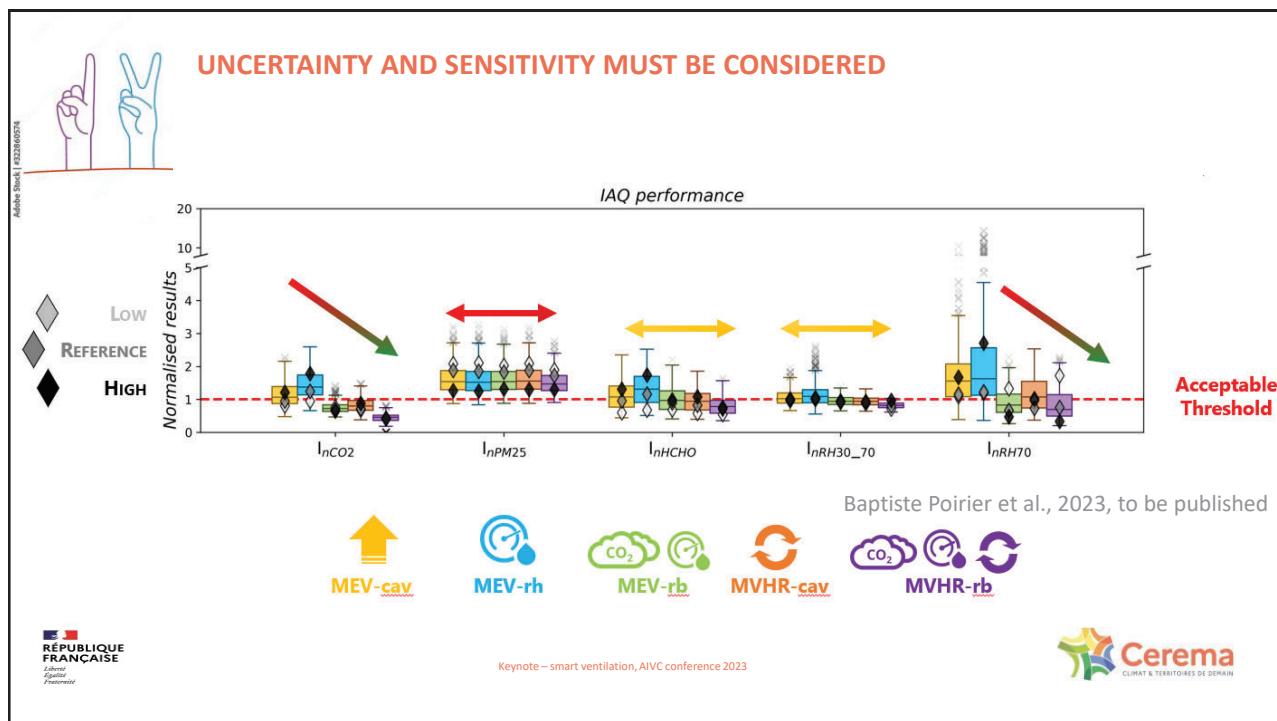
EXAMPLE OF ENERGY AND IAQ PERFORMANCE RESULTS



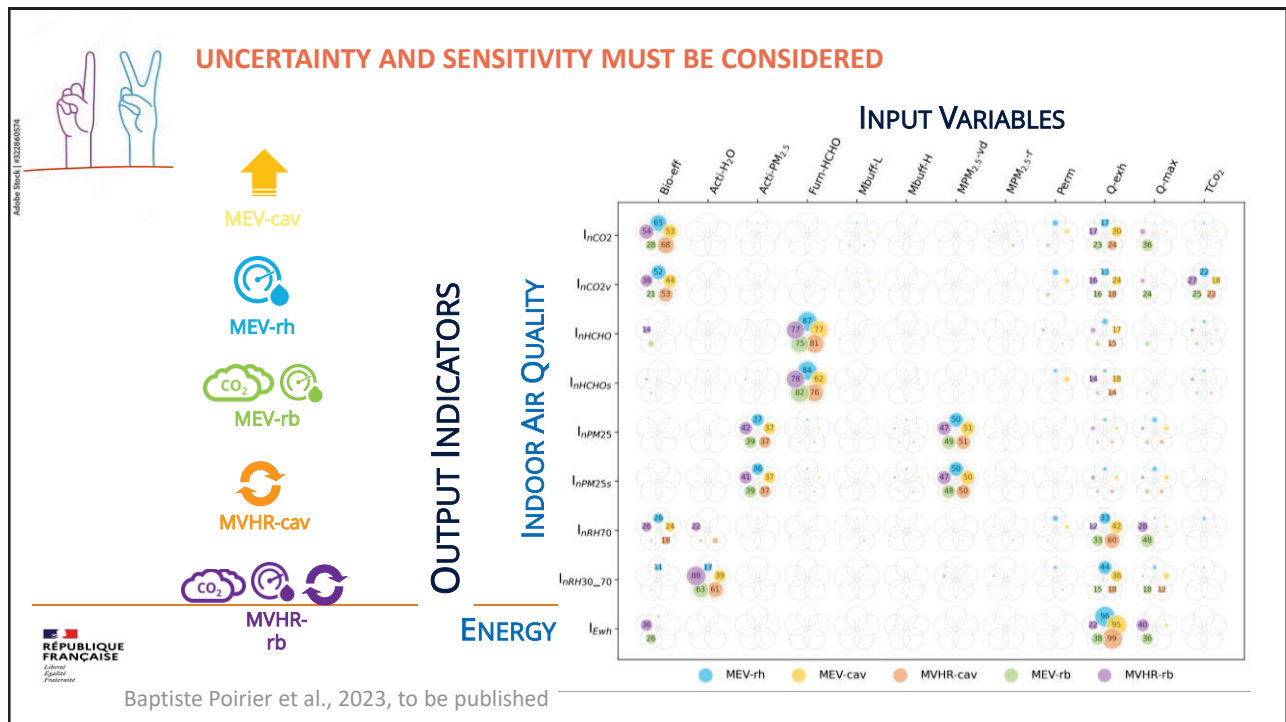
Reference : Baptiste Poirier, et al., 2023, to be published



25



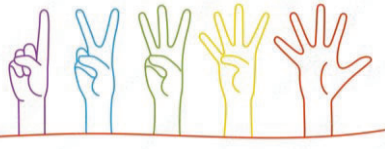
26



27

SMART VENTILATION NEEDS SPECIAL ATTENTION


- Ventilation in general suffers from a lack of quality (design, installation, maintenance)
- With smart ventilation, we generally allow lower airflows at some times when needs are low (no occupancy, low emissions, etc...)
 - ⇒ We have to secure even more the performances **over the ventilation lifetime – 20 years**
 - ⇒ 3 - Requirements for sensors and actuators accuracy / longevity
 - ⇒ 4 - Secure long-term performances
 - ⇒ 5 - Adapted inspection protocols



Alphée Smeck | #322660274

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Keynote – smart ventilation, AIVC conference 2023



Cerema
CLIMAT & TERRITOIRES DE DEMAIN

28



3- REQUIREMENTS ON SENSORS OR ACTUATORS



The only example : Belgium

MONITEUR BELGE — 13.11.2015 — BELGISCH STAATSBAD

68615

Art. 2. Le présent arrêté entre en vigueur le 1^{er} décembre 2015.
Bruxelles, le 29 octobre 2015.

La Ministre de l'Enseignement de Promotion Sociale, de la Jeunesse, des Droits des femmes et de l'Egalité des chances,
Mme I. SIMONIS

VERTALING

MINISTERIE VAN DE FRANSE GEMEENSCHAP

[C - 2015/29552]

29 OKTOBER 2015. — Ministerieel besluit tot goedkeuring van het referatodossier van de onderwijseenheid « Burgerzin » (code 050202U11D1), gerangschikt op het niveau van het lager secundair overgangsonderwijs voor sociale promotie



3.2.4 Incertitude des capteurs de détection

Les capteurs utilisés pour la détection des besoins, comme spécifié dans la suite du texte, doivent avoir une incertitude maximale sur la valeur du paramètre mesuré comme suit :

- Pour les capteurs de concentration en CO_2 : ± 40 ppm $+ 5$ % de la valeur, entre 300 et 1 200 ppm (exemple pour une exigence de 950 ppm, l'intervalle de tolérance est compris entre 862 ppm et 1 038 ppm);
- Pour les capteurs d'humidité relative (RH) : ± 5 points de pourcentage d'humidité relative, entre 10 % et 90 % (exemple pour une exigence de 35 % d'humidité relative, l'intervalle de tolérance est compris entre 30 % et 40 % d'humidité relative).

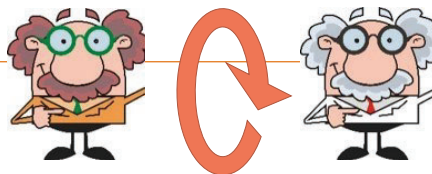
29



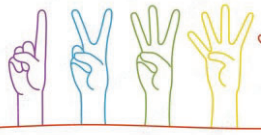
4- MORE STUDIES ABOUT LONG-TERM DURABILITY OF PERFORMANCES

- Ventilation lifetime – 20 years !

1. the intrinsic technological durability of all the components, sensors, controllers of these intelligent systems during their aging and their life in the building;
2. the robustness in use, i.e., the robustness of all these components to the use made of them by the occupants: maintenance (or absence), cleaning (or absence), unforeseen use (obstruction, clogging, ...) linked to different reasons specific to the occupants, to their understanding of the issues, of the systems, and to the way the systems take into account their needs and expectations.

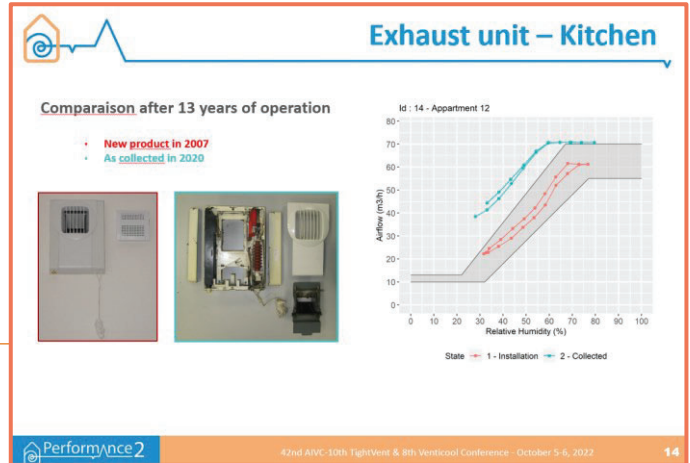
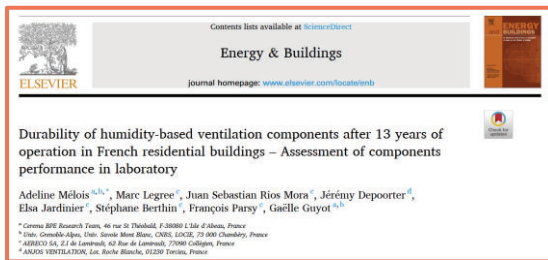


30



4- MORE STUDIES ABOUT LONG-TERM DURABILITY OF PERFORMANCES

- Very few studies found in the literature
 - 3 in 2023
 - Zero in the Annex 86 group (so far)
 - « Long-term »: sometimes only one year ...



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5- ADAPTED INSPECTION PROTOCOLS REGULARLY REQUIRED

- Emerging inspection protocols and quality management schemes should include specificities due to smart ventilation
- Including a frequency of inspections adapted to the long-term performances offered by the systems and/or the sensors
 - Not only at building's commissioning, **Ventilation lifetime – 20 years !**
- Smart ventilation can offer online and continuing commissioning = a huge opportunity !



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WHAT WE NEED ? TO RESUME

1. Sober systems adapted to the challenges of our decade
2. Robust and improved performance-based assessment schemes E+IAQ
3. Requirements on sensors or actuators
4. More studies about long-term durability of performances
5. Adapted inspection protocols regularly required

INDUSTRIALS / RESEARCHERS = PULL

POLICY MAKERS / RESEARCHERS = PUSH

INDUSTRIALS / POLICY MAKERS / RESEARCHERS = PULL & PUSH

INDUSTRIALS / RESEARCHERS = PULL

POLICY MAKERS / RESEARCHERS = PUSH



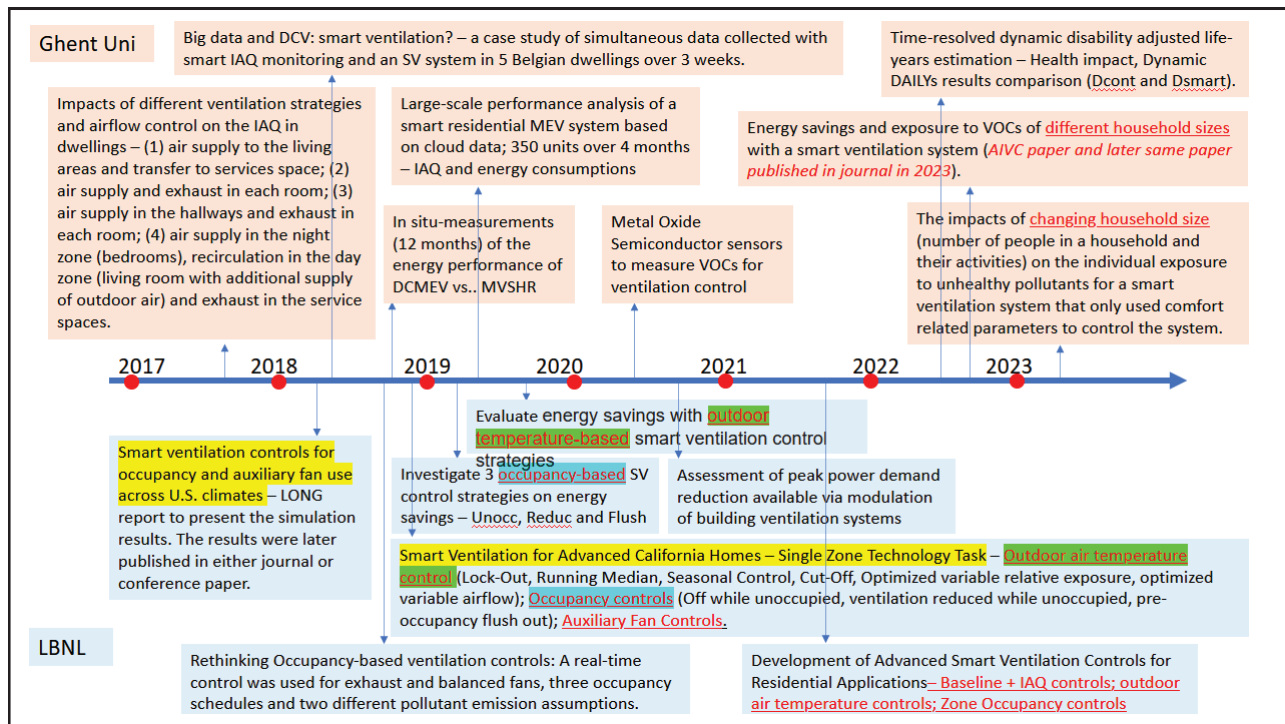
Thank you for your attention

Special thanks to

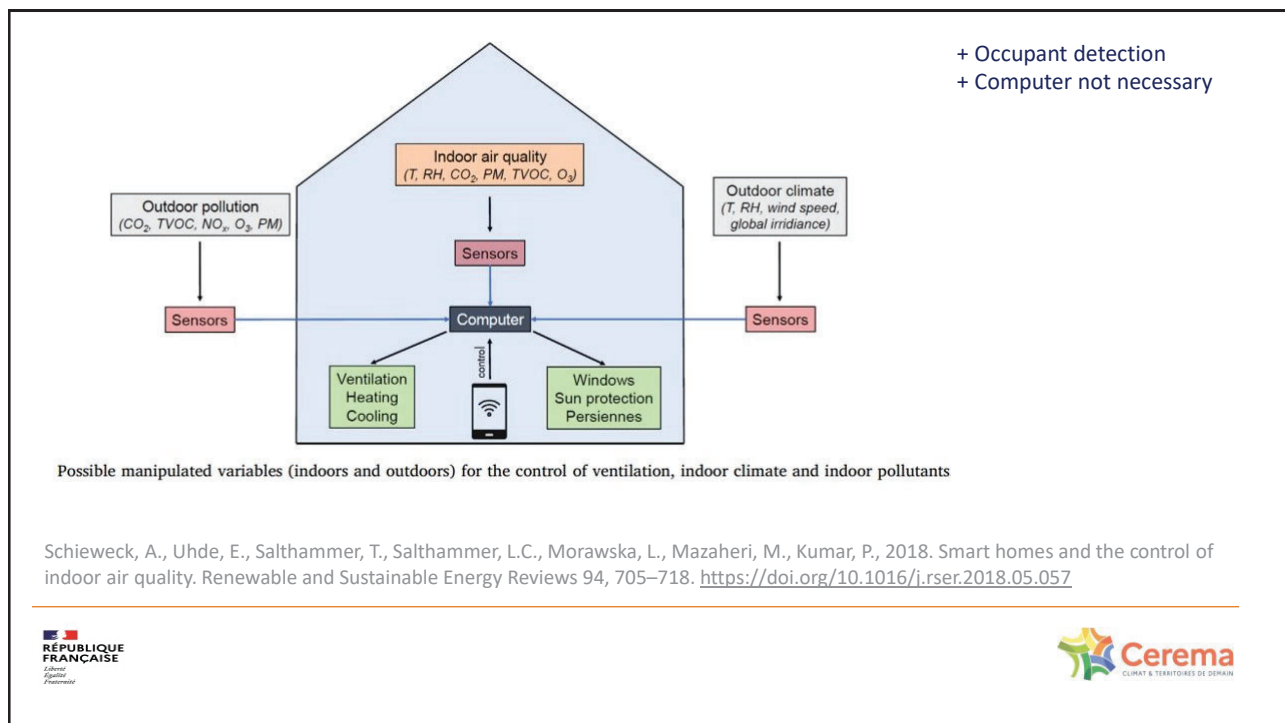
Dr Daniela Mortani, PUCPR, Cerema

Dr Yu Wang, BRANZ

Colleagues of IEA-EBC Annex 86 & my dear co-leader Dr Jakub Kolarik



35



36

IMPACT OF THE SELECTED IAQ PERFORMANCE INDICATOR

- “Zonal controls that saved energy by reducing outside airflow achieved typical reductions in ventilation-related energy of 10% to 30%, compared to the 7% savings from the unzoned control. However, this was at the expense of increased personal concentrations for some contaminants in most cases. In addition, care is required in the design and evaluation of zonal controls, because control strategies may reduce exposure to some contaminants, while increasing exposure to others”

Walker, I., Less, B., Lorenzetti, D., Sohn, M.D., 2021. Development of Advanced Smart Ventilation Controls for Residential Applications. *Energies* 14, 5257. <https://doi.org/10.3390/en14175257>

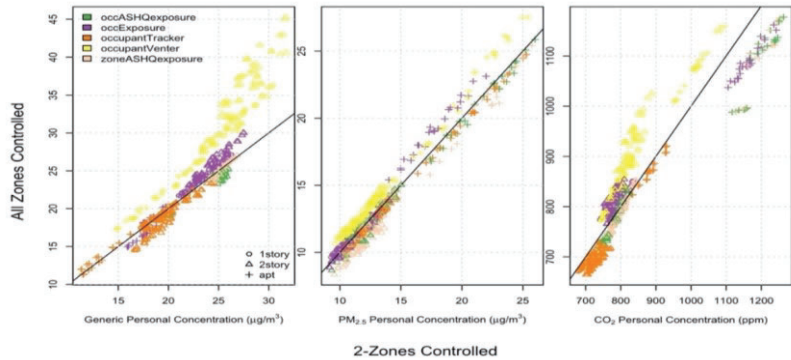


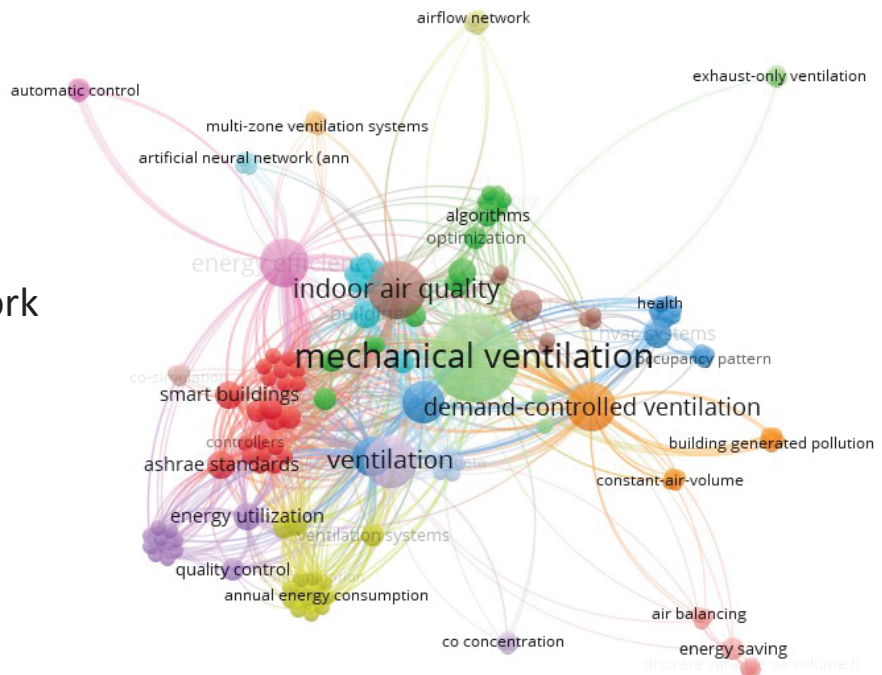
Figure 5. Comparison of annual median personal contaminant exposure by for systems controlling all zones or only two zones for different control strategies (colours) and dwelling types (symbols).

37

VOSViewer results

Keywords

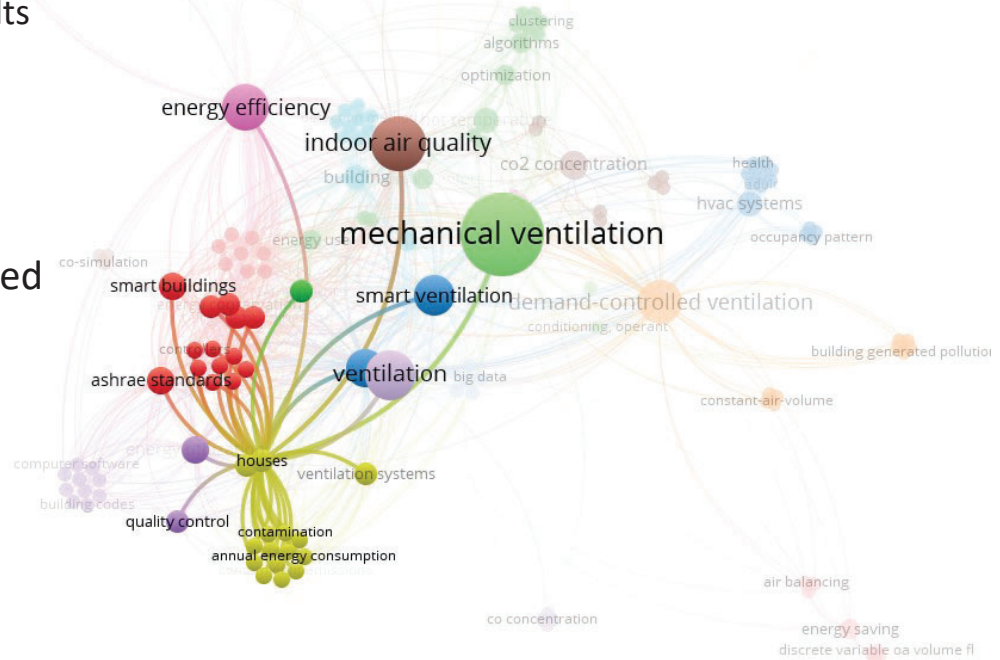
co-occurrence network



38

VOSViewer Results

Keywords incidence related to “houses”



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« HOUSES ARE DUMB WITHOUT SMART VENTILATION »

Isn't it dumb to provide a constant ventilation airflow rate ?

- Everywhere throughout a building,
- whatever the boundary conditions: climate, outdoor pollution, seasons, ...
- whatever the needs of the occupants;
- whatever the risks of damage to the buildings.

=> totally unsuited to our challenging changing world and our need to adapt



ERNEST ORLANDO LAWRENCE
BERKELEY NATIONAL LABORATORY

Houses are Dumb without Smart Ventilation

Iain Walker, Max Sherman and Brennan Less

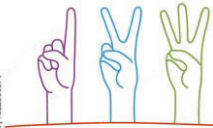
Environmental Energy Technologies Division
Lawrence Berkeley National Laboratory Berkeley,
CA 94720

May 2014

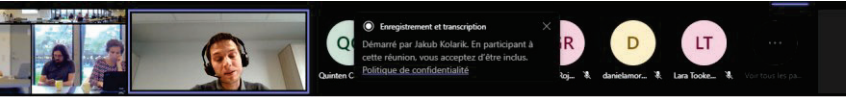
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41



3- REQUIREMENTS ON SENSORS OR ACTUATORS



Where do the tolerance values come from ?



- **Final choice of the values**

Sensitivity of calculated IAQ to sensor accuracy (f_{reduc} study)


Analysis of the accuracy existing sensors on the market

Final choice of tolerances, insuring:

- That the IAQ is not too deteriorated compared to "perfect sensors"
- That the criteria are not too restrictive compared to actual existing sensors (exclude bad products, but margin large enough to have concurrency on the market, and not only high-end products)

Source : Sebastien Pecceu, Buildwise,
Annex 86 meeting, 2 October, Copenhagen



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DALYing with DALYs

Why *acceptable* IAQ should consider harm

Dr Benjamin Jones

Associate Professor
University of Nottingham
benjamin.jones@nottingham.ac.uk



1



indoor air quality

2

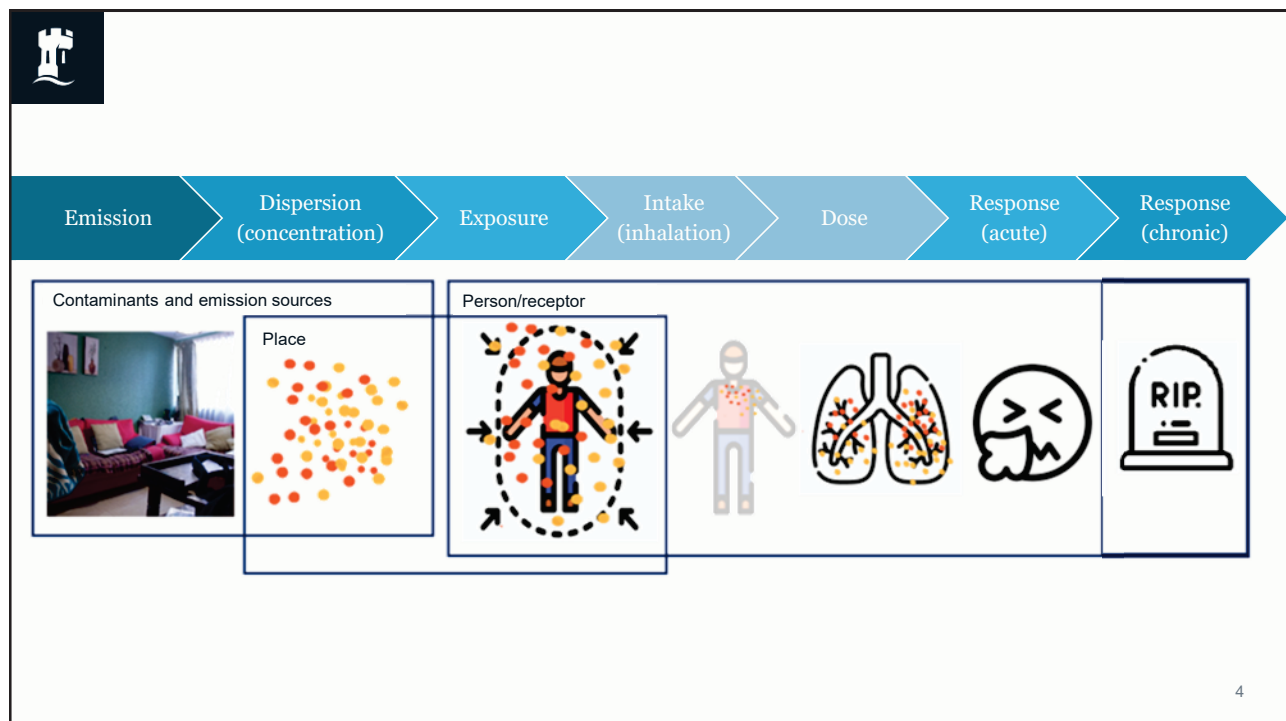
2



Acceptable indoor air quality: *air in which there are no known contaminants at harmful concentrations, as determined by cognizant authorities, and with which a substantial majority (80% or more) of the people exposed do not express dissatisfaction.*

3

3

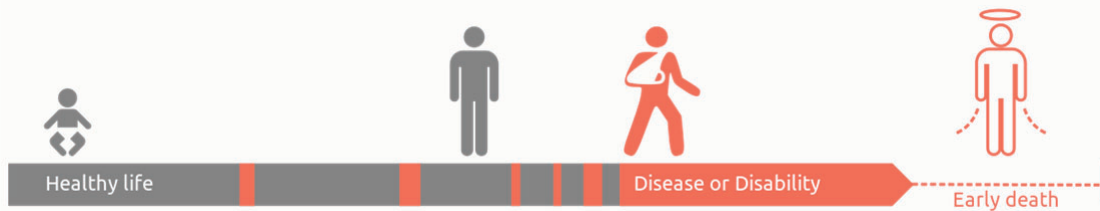


4

4



Chronic harm



Metrics Overview: AIVC VIP#36

Ventilation Information Paper n° 36
September 2017

© INIVE EERG
Operating Agent and Management
Bolevard Poelaert 79
B-1000 Brussels - Belgium
inive@bvt.be - www.inive.org

International Energy Agency
Energy Conservation in Buildings
and Community Systems Programme

Abstract

In a recent review of 31 green building certification schemes used around the world, IAQ was found to contribute to only 7.5% of the final score on average¹. As policy makers strive to reduce the energy demands of buildings by sealing or reducing outdoor air ventilation rates, an unintended consequence could be the reduction in the quality of indoor air with corresponding negative health effects at a population scale. This article summarizes the discussion of an Air Infiltration and Ventilation Centre workshop on IAQ metrics held in March 2017². It first identifies the types of contaminants found in many buildings today, the mechanisms of exposure to them, and methods of mitigating their effects. It then explores metrics that could be used to quantify the quality of indoor air.

¹W. W. Ramalho, O. Manfrot, C. Dubois on quality requirements in green building certification. Building and Environment, 2015, 92: 16-9.
²AIVC: "Ventilation for the sake of indoor air quality control in buildings? Do we need performance-based approaches?" AIVC Workshop held in Brussels, Belgium, 14th-15th March, 2017.



Metrics of Health Risks from Indoor Air

Dr Benjamin Jones
University of Nottingham, UK

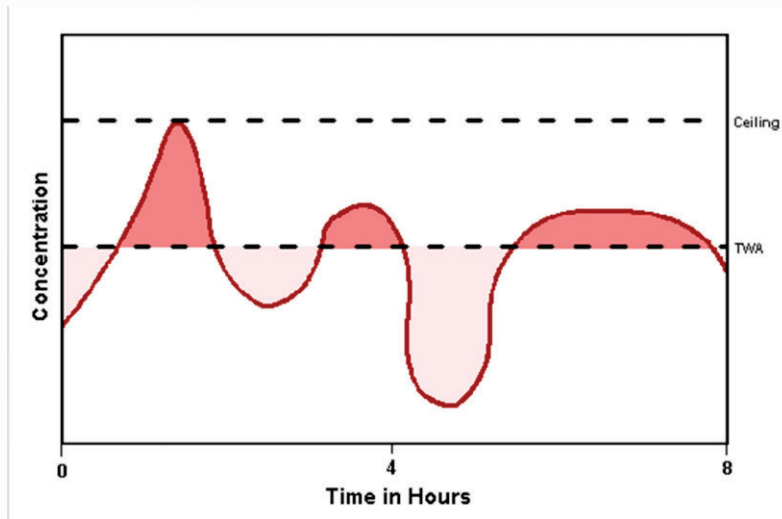
1 Problems

Building materials and systems, and the activities carried out in them, can be a source of contaminants that are harmful to human health. For example, there is evidence that some of the materials used to construct and furnish buildings emit harmful gases and harbor biological organisms. Unvented combustion processes for space and food heating emit gaseous and particulate contaminants and can be a source of moisture that is a primary driver of biological growth. Human activities, such as cooking and vacuum cleaning, also emit particulates. Cleaning and disinfecting products emit gaseous contaminants and particulates, and smoking emits over 7000 different compounds of which many are harmful³. Poor indoor and transport biological contamination, and can themselves be allergens. People and pets also emit gaseous by-products that are digestible to mold, and harbor pathogens that produce disease. These examples show the many potential harmful and contaminant sources in buildings, for which there are multiple exposure pathways, and not all of them are airborne.

³ CDC. New Tobacco Smoke Causes Disease: The Biology and Behavioral Basis for Smoking-Attributable Disease. Centers for Disease Control, Atlanta, Georgia, U.S.A., U.S. Public Health Service, 2010.



Current acceptability

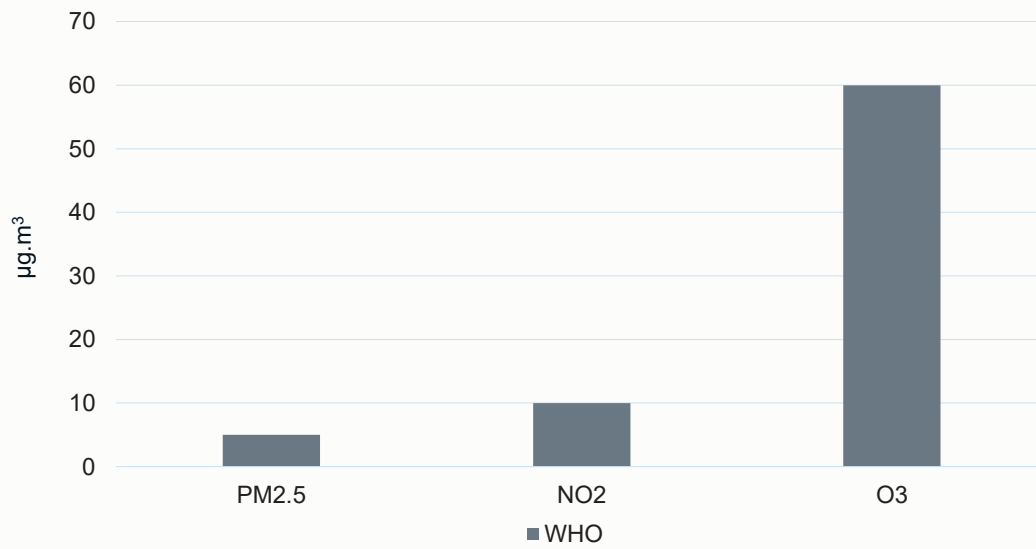


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WHO threshold values

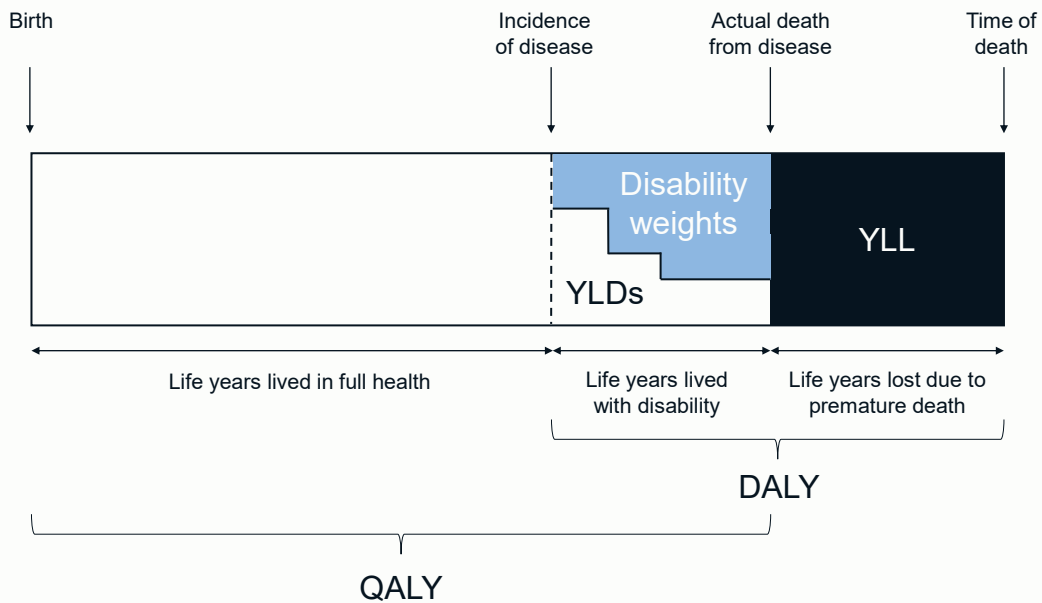


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8



Health Adjusted Life Years



9

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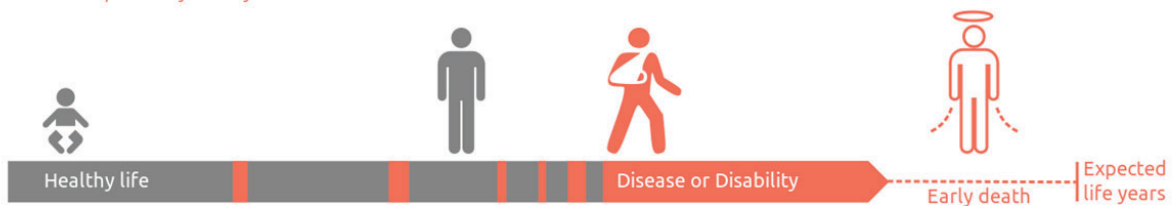
Disability Adjusted Life Years (DALYs)

DALY

Disability Adjusted Life Years is a measure of overall disease burden, expressed as the cumulative number of years lost due to ill-health, disability or early death

$$= \text{YLD} + \text{YLL}$$

Years Lived with Disability + Years of Life Lost



10

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Acceptable harm? (DALYs)

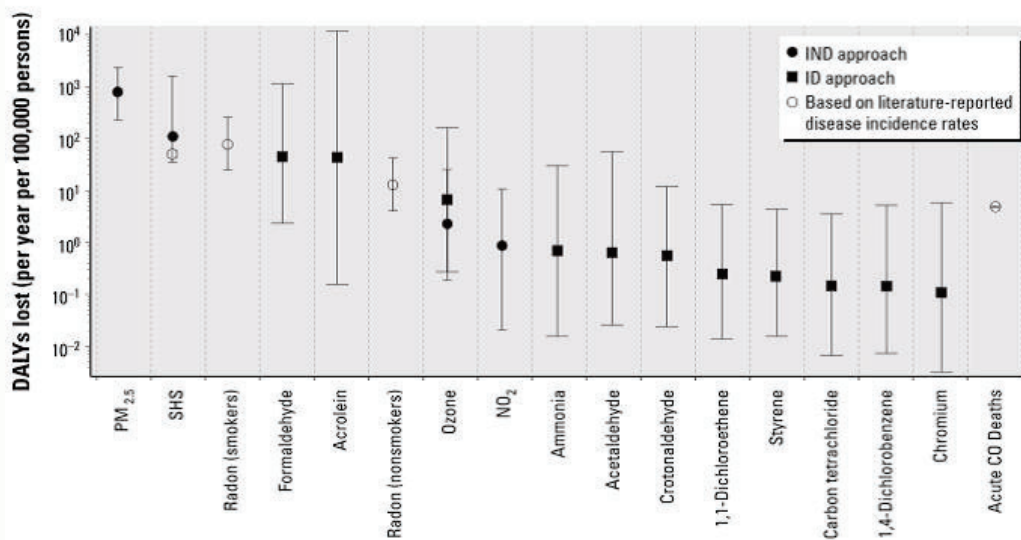
Alcoholism	Smoking	Transport injuries
1,200	2,600	1,000

11

11



Previous work



Logue JM, Price PN, Sherman MH, Singer BC. A Method to Estimate the Chronic Health Impact of Air Pollutants in U.S. Residences. *Environmental Health Perspectives*. 2011;120(2):216-22.

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Chronic harm in houses

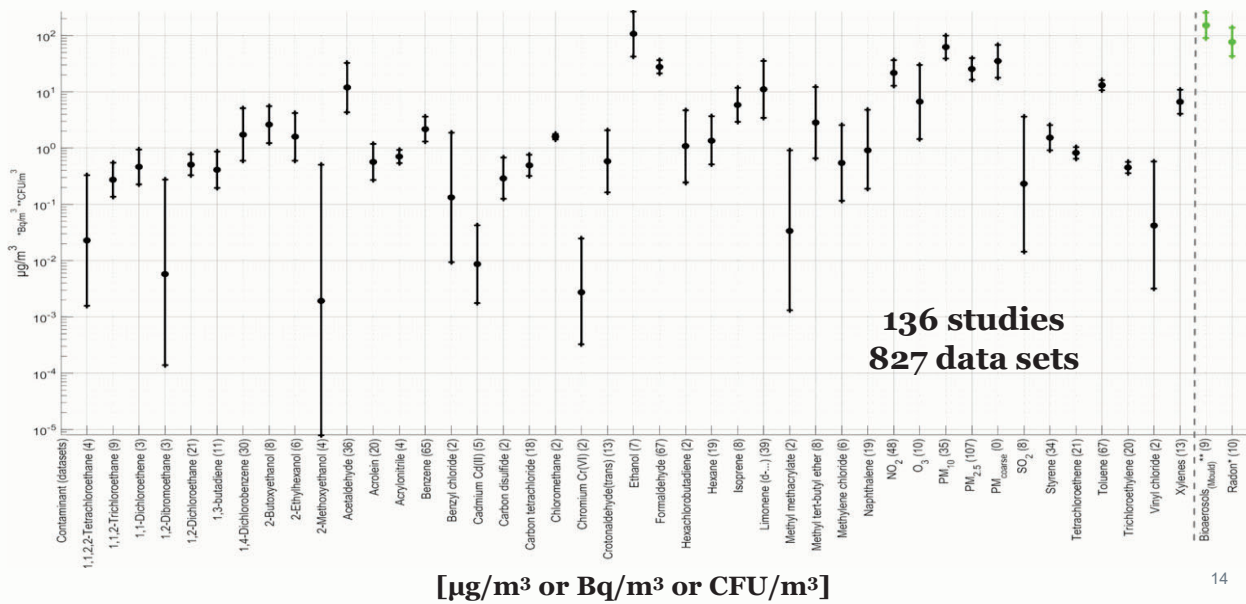


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Concentrations

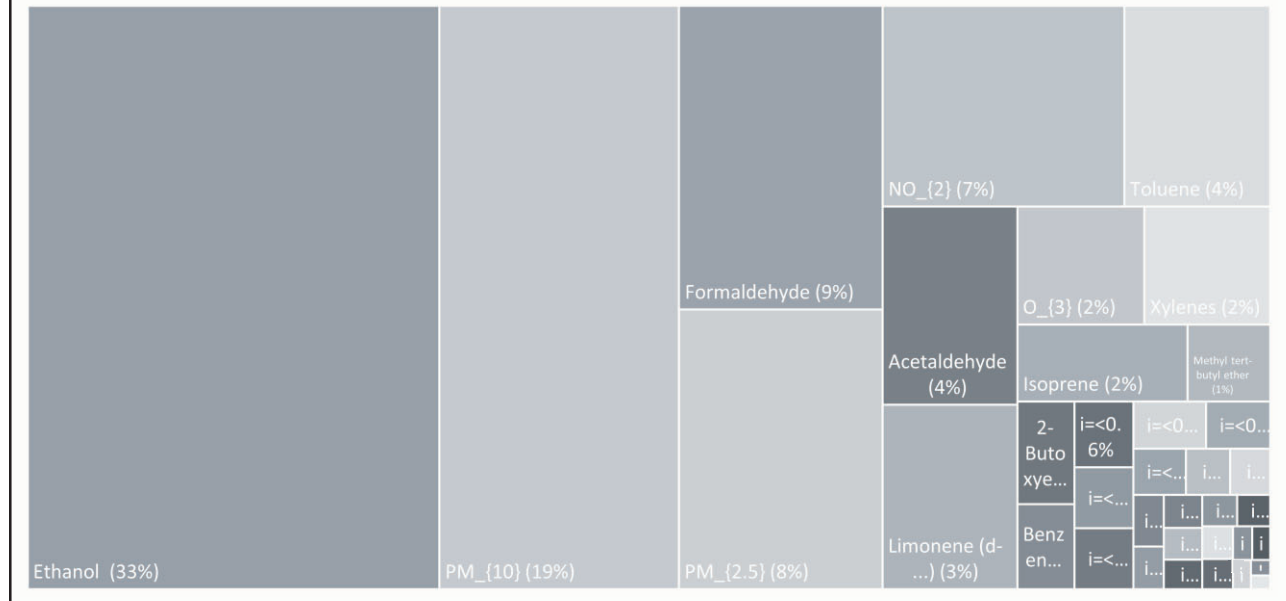


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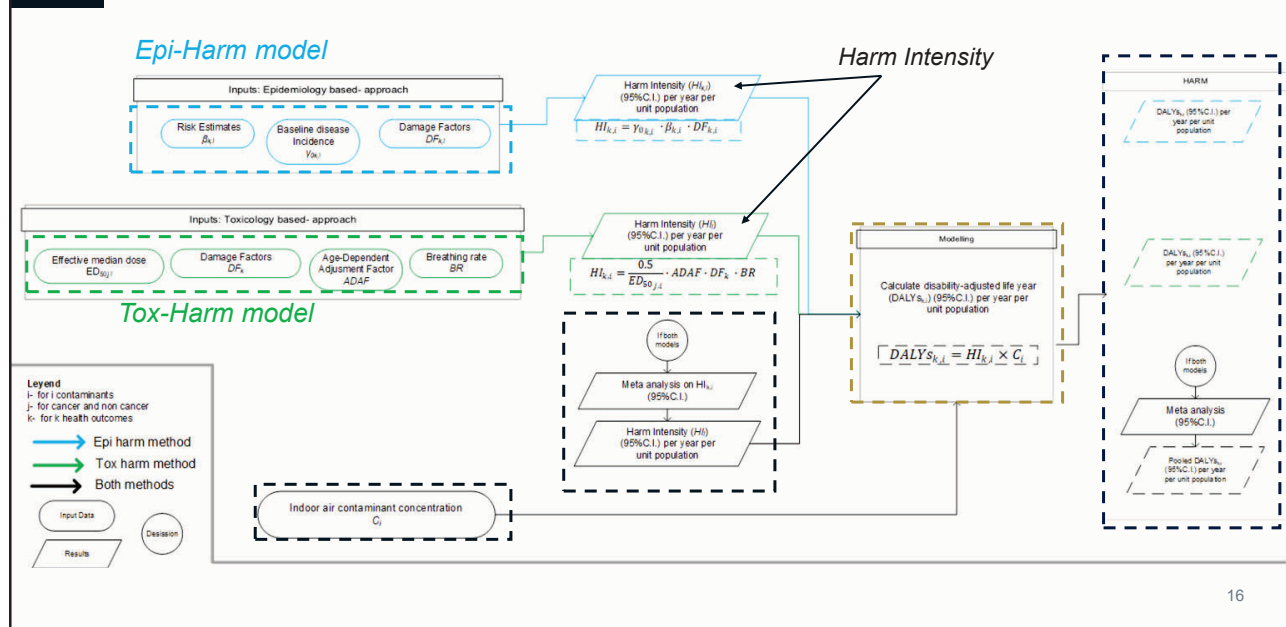
Concentrations



15



Harm model

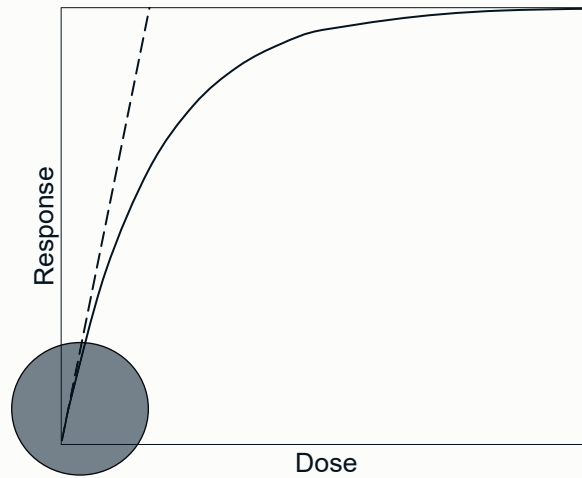


16

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Linearity



17

17



$$Harm = C \times HI$$

$$[\text{DALYs/person/year}] = [\mu\text{g}/\text{m}^3] \times [\text{DALYs}/\mu\text{g}/\text{m}^3/\text{person/year}]$$

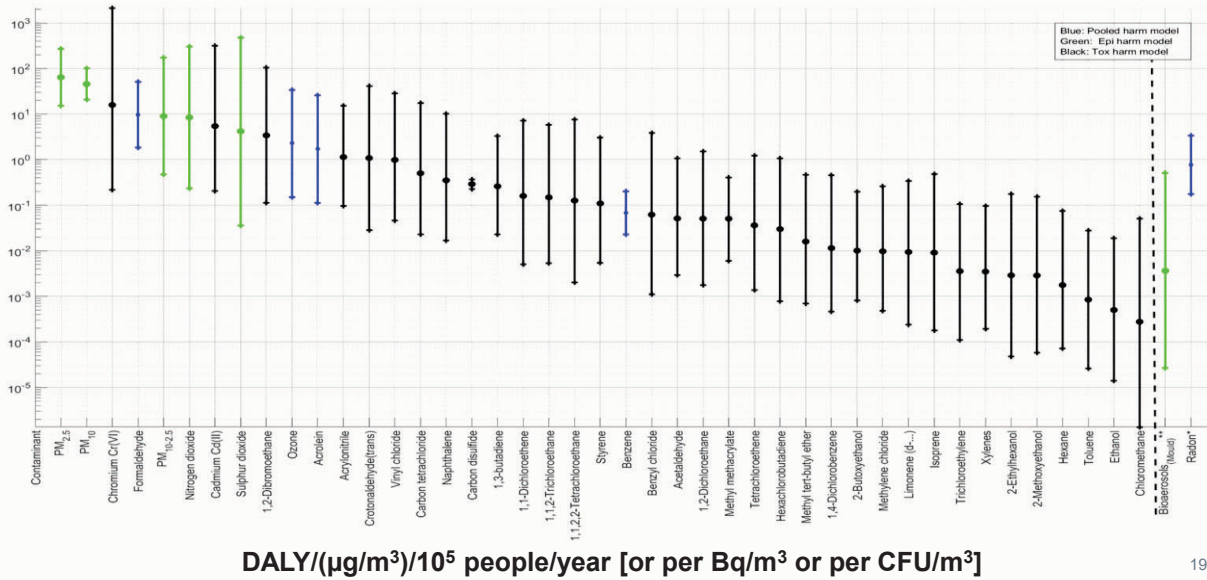
(or per Bq/m³ or per CFU/m³)

18

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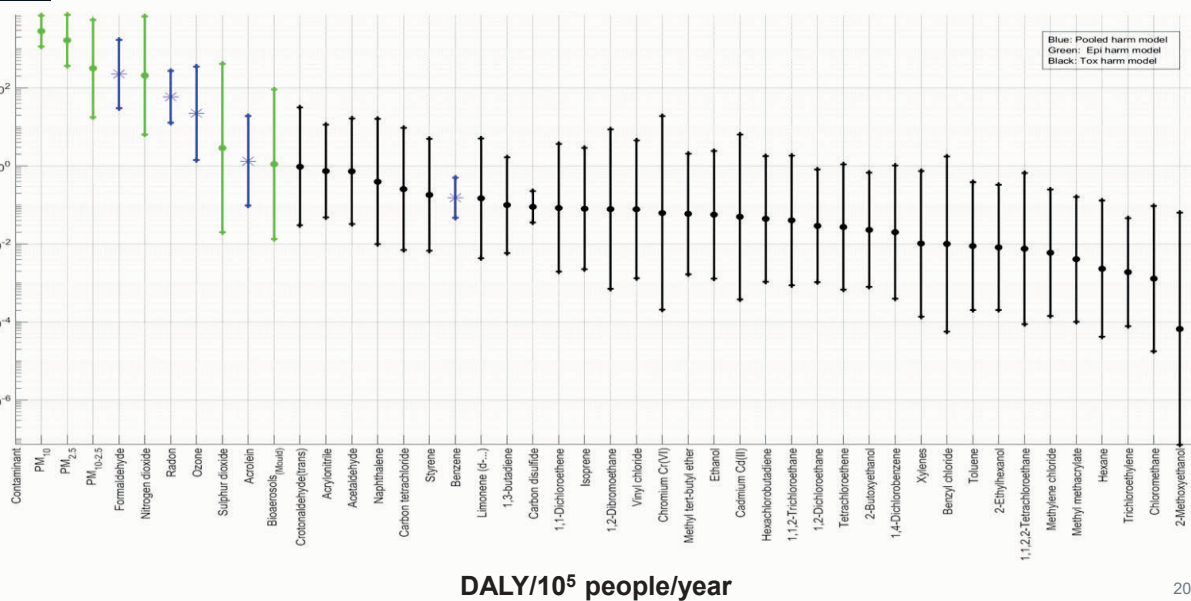
Harm intensity



19



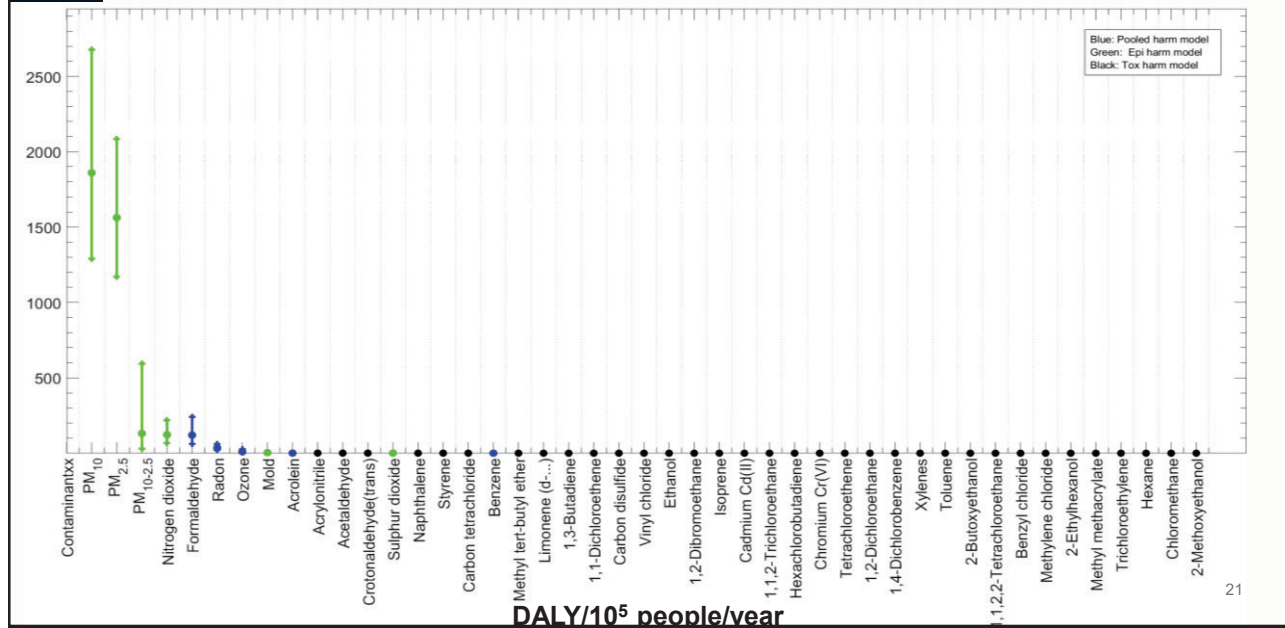
Total harm



20



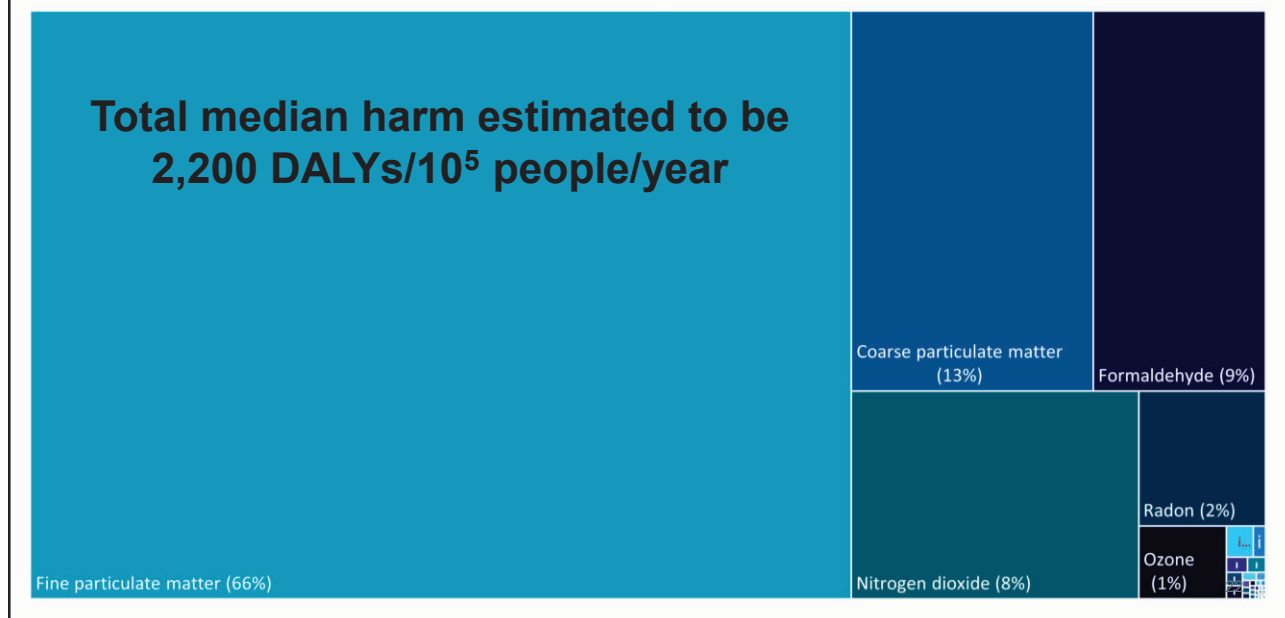
Total harm



21



Total harm



22



Total Harm (DALYs)

Dwelling IAQ	Alcoholism	Smoking	Transport injuries
2,200	1,200	2,600	1,000

23

23



Contaminants of Concern

	Harm (DALYs/10 ⁵ people/year)	Harm Intensity (DALYs/μg.m ⁻³ /10 ⁵ people/year)
PM _{2.5}	1600	60
PM _{10-2.5}	130	3.8
Nitrogen Dioxide (NO ₂)	120	5.7
Formaldehyde (HCHO)	120	4.3
Radon (Rn)	34	0.44
Ozone (O ₃)	10	1.3

24

24



Harm Intensities for CoCs

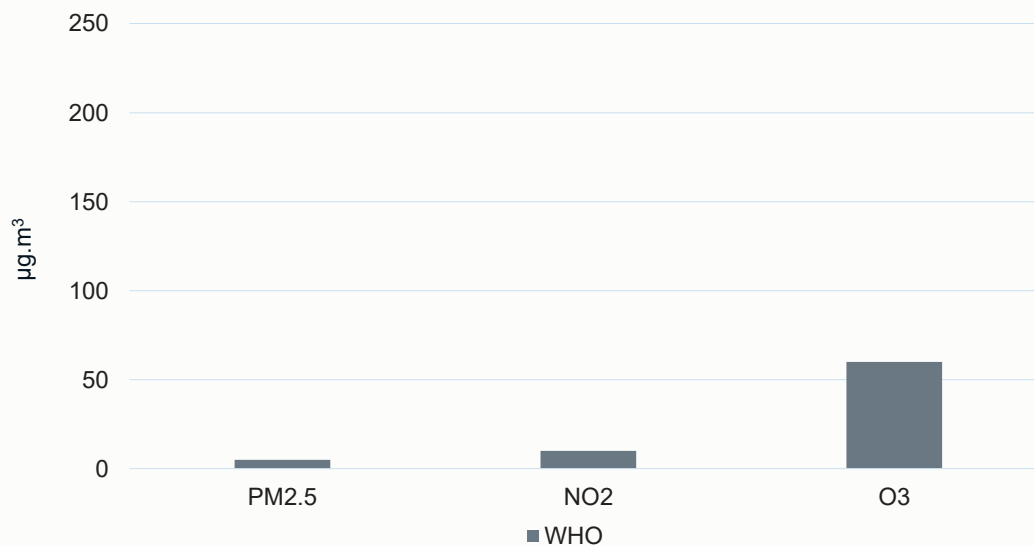
	Harm Intensity (HI) (DALYs/ $\mu\text{g}\cdot\text{m}^{-3}/10^5$ people/year)	HI Limiting Concentration ($\mu\text{g}\cdot\text{m}^{-3}$ or $\text{Bq}\cdot\text{m}^{-3}$)
PM _{2.5}	60	50
PM _{10-2.5}	3.8	25
Formaldehyde (HCHO)	4.3	50
Nitrogen Dioxide (NO ₂)	5.7	240
Ozone (O ₃)	1.3	500
Radon (Rn)	0.44	450

25

25



Harm from WHO threshold values

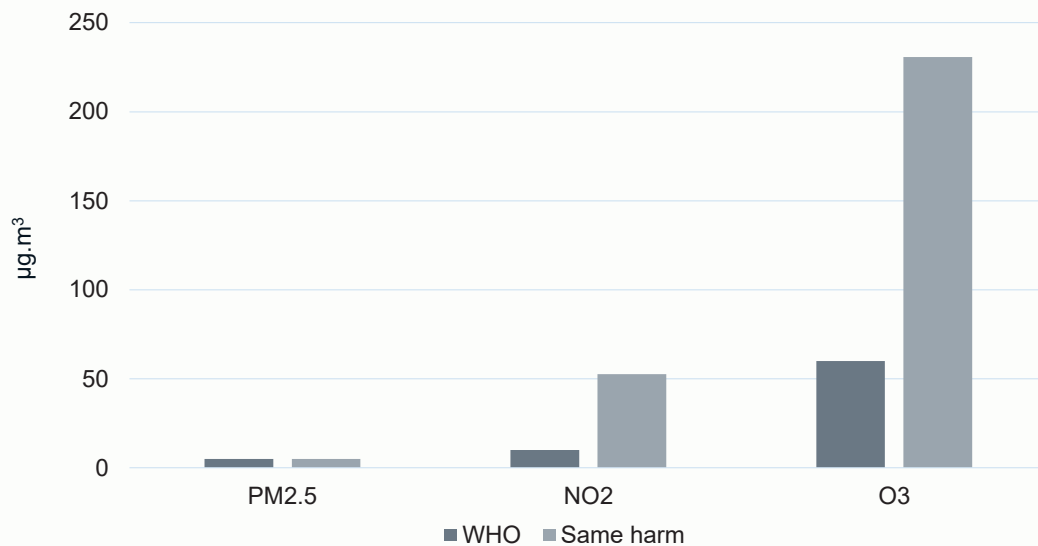


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26



Harm from WHO threshold values



27

27



Harm budget



28

28



Reference scenario

- Singer *et al.* 2020. Indoor air quality in California homes with code-required mechanical ventilation. Indoor air 30(5).
 - N=70
 - All comply with CalEnergyCode
 - PM_{2.5} 5 $\mu\text{g.m}^{-3}$
 - HCHO 23 $\mu\text{g.m}^{-3}$
 - NO₂ 9 $\mu\text{g.m}^{-3}$
- Guideline values used for Rn (100Bq/m³) and for O₃ (40 $\mu\text{g.m}^{-3}$).
- Total harm of 610 DALYs/10⁵ people/year



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Harm (DALYs)

Reference	Dwelling IAQ	Alcoholism	Smoking	Transport injuries
610	2,200	1,200	2,600	1,000

30

30



Acknowledgements



Benjamin Jones
University of Nottingham



Gioberti Morantes Quintana
University of Nottingham



Constanza Molina
Pontifical University of Santiago, Chile



Max Sherman
*University of Nottingham
& Lawrence Berkeley National Laboratory*



University of
Nottingham
UK | CHINA | MALAYSIA



IEA Annex 86



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Summary

1. Developed a harm intensity metric that quantifies the chronic health impact (in DALYs) per unit concentration of an air contaminant.
They apply to any environment.
2. Identified the most harmful indoor air contaminants in dwellings that should be prioritized declaring them *Contaminants of Concern*.
3. Estimated the total harm caused by typical exposures to indoor air contaminants in dwellings.
4. Propose the concept of a ***harm budget*** to define acceptable indoor air quality based on the harm caused by priority contaminants.

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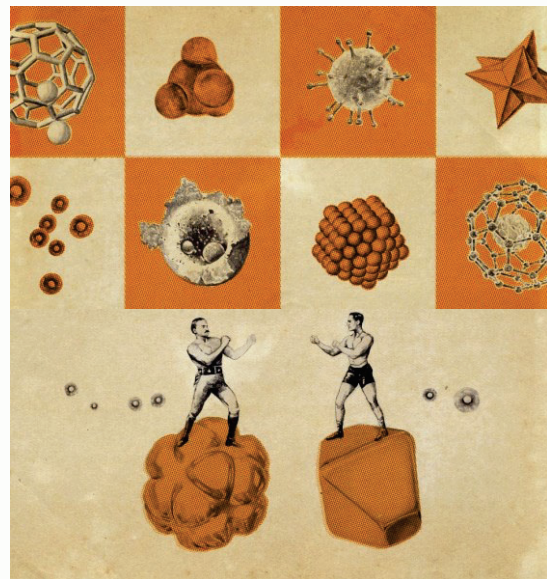
Thank you

33



Equitoxicity

- We assume PM equitoxicity
- PM composition does vary
- Separate indoor/outdoor PM risk estimates are unavailable
- PM size predicts long-term harm
- Indoor PM found to be coated in PAHs and other VOCs
- Would have to be 12x less harmful to be equivalent to PM₁₀, HCHO and NO₂
- Precautionary principle



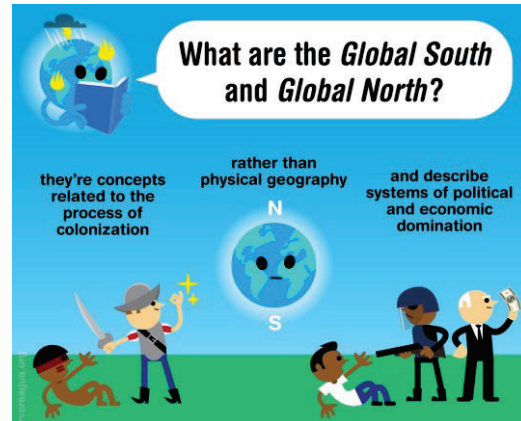
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What do these concentrations represent?

- Indoor concentrations represent the Global North (USA, China, Canada, UK most represented)
- Caution needed for regional comparisons due to lifestyle/location differences
- Include common household activities
- Avoid niche construction types (e.g. Passivhaus)
- Fieldwork essential to reduce uncertainty



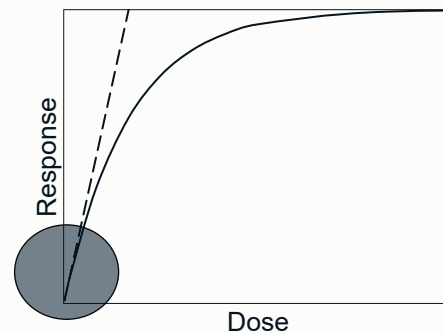
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Linearity

- We don't know if the Poissonian C-R relationship represents these contaminants
- It is good modelling practice to linearise a model, if possible
- It is possible *here* because the concentrations commonly found in dwellings are low enough
- Harm Intensities might be given with upper concentration limits
- We have done an error analysis and this will be in the Annex 86 report



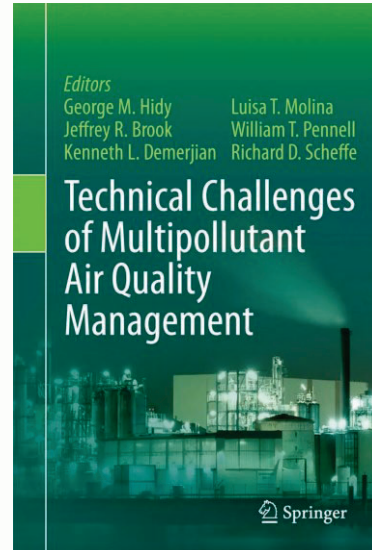
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Synergistic responses

- We do not do this
- It is not possible to do
- Assumes additivity for indoor pollutant effects.
- Additivity simplifies complex interactions, may underestimate/overestimate impacts.
- Future research should explore pollutant interactions for accuracy.



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What about acute effects?

- This data is for chronic harm
- Some contaminants may have significant short-term acute impacts
- Estimate acute harm is a future project

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Where does the health data come from?

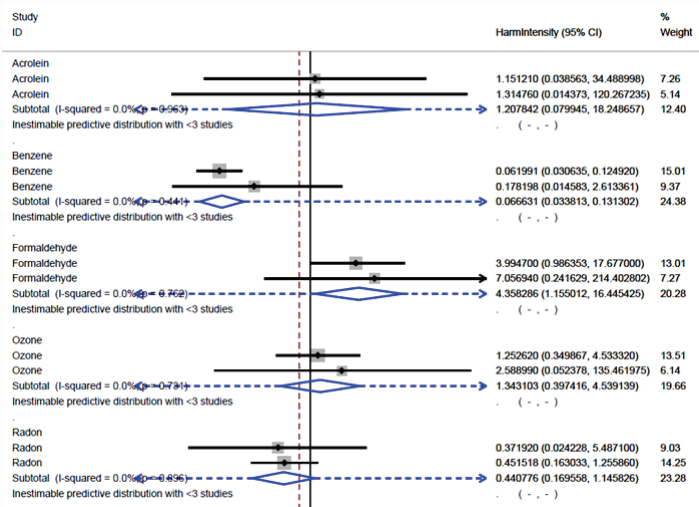
- Toxicological
 - USEtox 2.0
 - Global burden of disease collaborative network for damage factors
 - Standard breathing rate

- Epidemiological
 - Global burden of disease collaborative network (incidence rates, damage factors))
 - Academic literature (for risk estimates)



Merging of epi and tox data

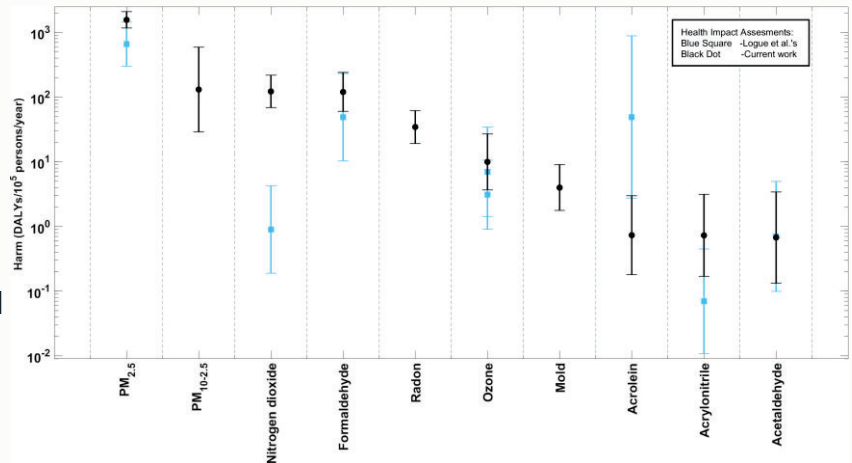
- Central estimates may not align due to methodological differences
- Perfect parity is challenging
- Despite challenges, parameters align





Differences from Logue

- Damage factors from 2019 Global Burden of Disease study.
- Consulted toxicology studies with lower uncertainty
- Health data, like PM2.5, became more robust and precise



41

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My favourite contaminant isn't on the list 😞

There is either

- insufficient data to determine a harm intensity

OR

- It isn't harmful in the concentrations found in dwellings



People aren't harmed by the contaminant they aren't exposed to....

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42

Human exposure against airborne pathogens in an office environment

Risto Kosonen, Sami Lestinen and Simo Kilpeläinen

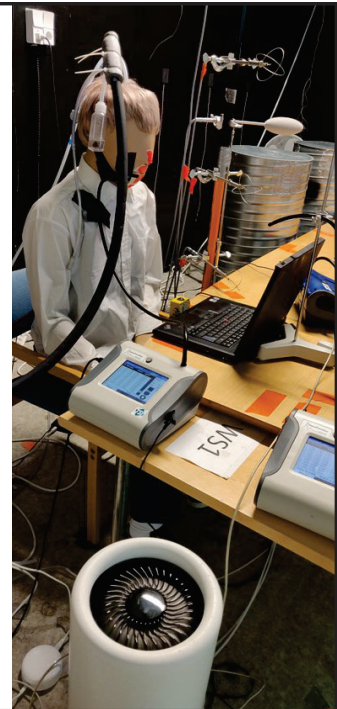
Aalto University, School of Engineering, Department of Mechanical Engineering, Espoo, Finland



1

Motivation

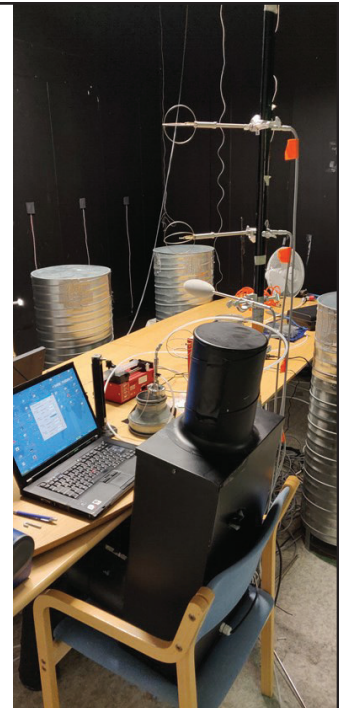
- **Exposure to airborne pathogens** is considerable risk in office environments.
- During **COVID-19 pandemic**, the office workers have been instructed to maintain safety distance, hand hygiene and face masks.
- **Face masks** might worsen working performance and thinking (**breathing resistance, CO₂ rebreathing and decreasing O₂**).
- **Alternative or additional protection methods** are necessary.
- **Remotely working** may not be possible all the time.



2

Objectives

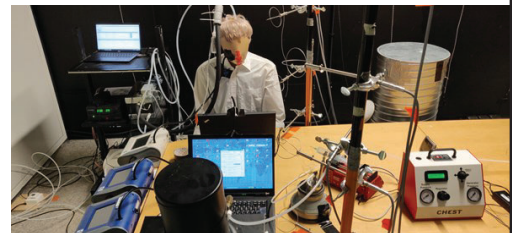
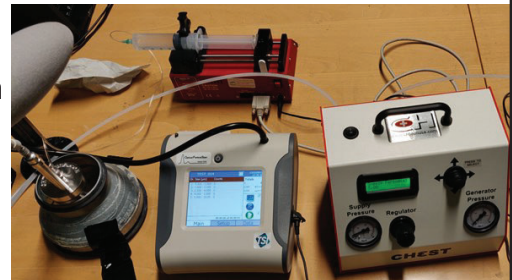
- **Investigating** aerosol transmission with different protection methods (**sick-healthy, opposite places, distance 1.2 m**).
- **Understanding** potential of each protection method.
- **Comparing** filtering and structural protection methods.
- **Providing** new knowledge for occupational health and safety purposes.



3

Methods – infected thermal dummy

- **Exhalation simulator** was used as an infection source.
- **Exhalator** was built by combining the exhalation simulator and a thermal dummy.
- **Exhaling** was set at 6 L/min (**continuous**)
- **Paraffin oil** was aerosolized in the Blaustein atomizer (**0.6 mL/h, syringe pump**).
- **Aerosol distribution** near source had similarities with earlier studies*



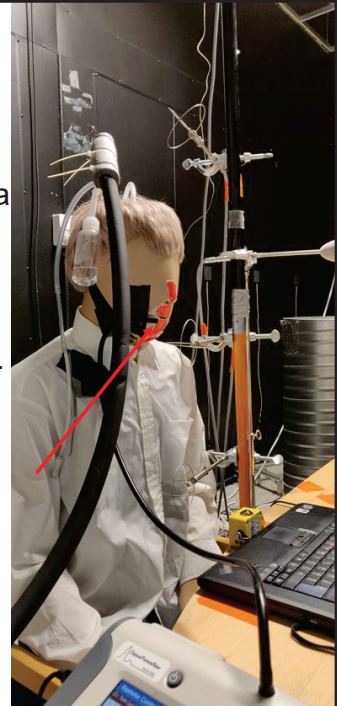
*Morawska et al. 2009. Size distribution and sites of origin of droplets expelled from the human respiratory tract during expiratory activities. Journal of aerosol science, 40(3), 256-269.

*Johnson et al. 2011. Modality of human expired aerosol size distributions [Article]. Journal of aerosol science, 42(12), 839-851.

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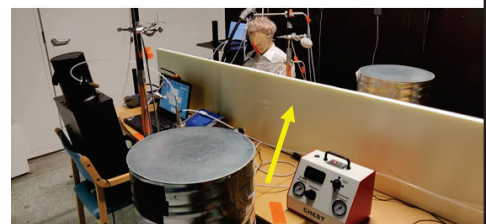
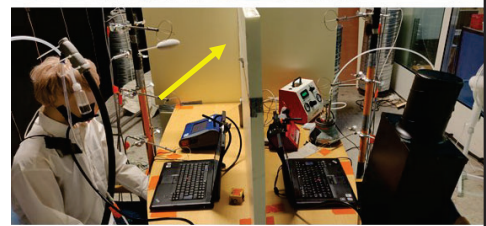
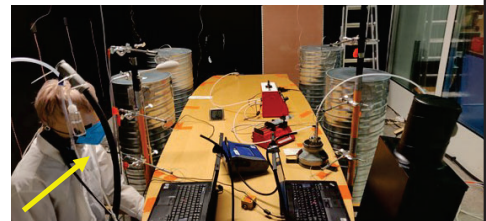
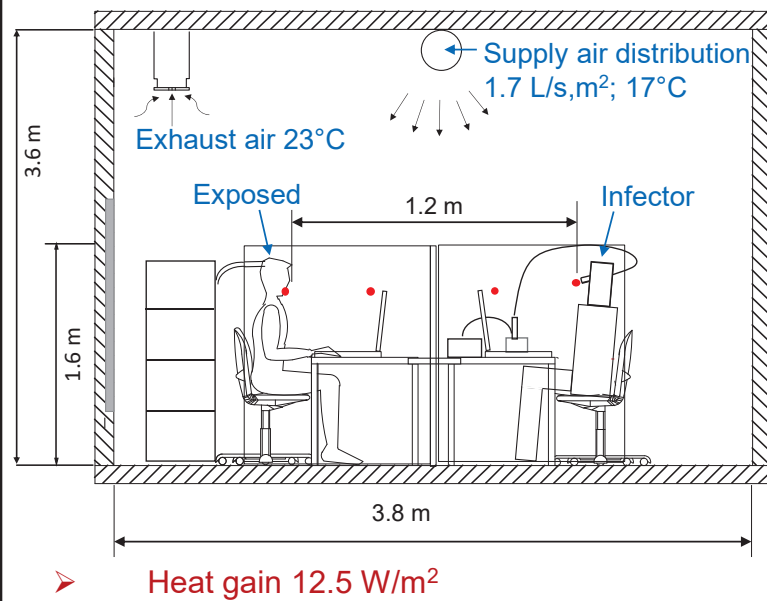
Methods – exposed thermal manikin

- Seated **breathing thermal manikin**.
- Body segments were heated under ‘**comfort mode**’ where a target temperature in each segment was constant.
- Breathing was set at **6 L/min**.
- Breathing cycle: **2.5 s** (in) **1 s** (break) **2.5 s** (out) **1 s** (break).
- Exhalation air temperature was set at **35°C**.
- Relative humidity of exhaled airflow was **85%**.
- Exhalation nose jets were directed **45°** downwards.
- Concentration measurement was **2 cm** beside the mouth.



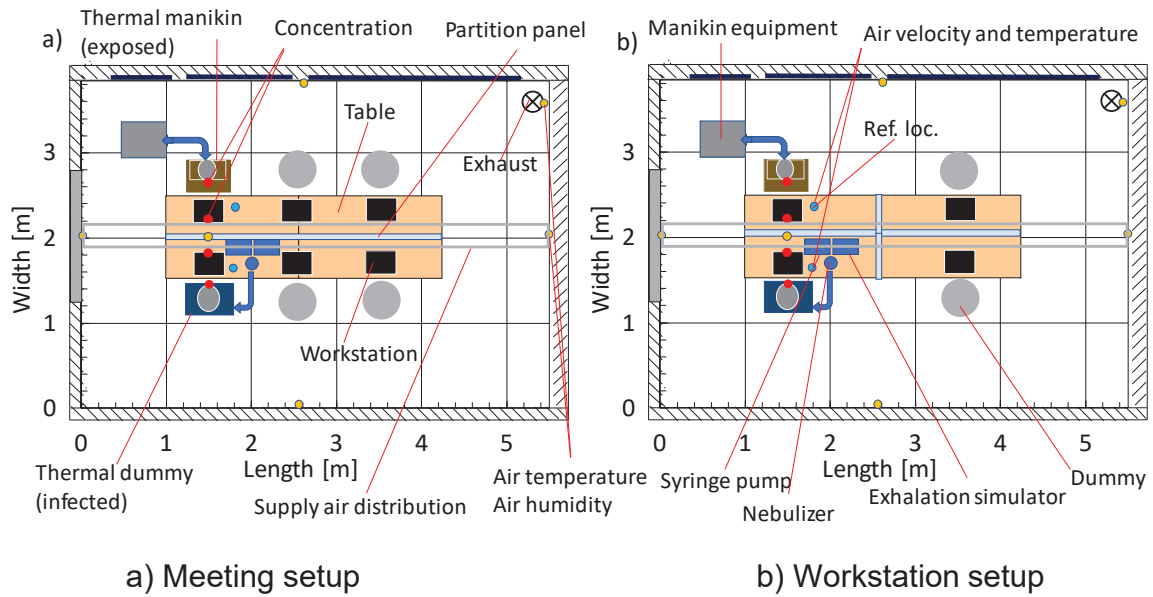
5

Side-view of test chamber



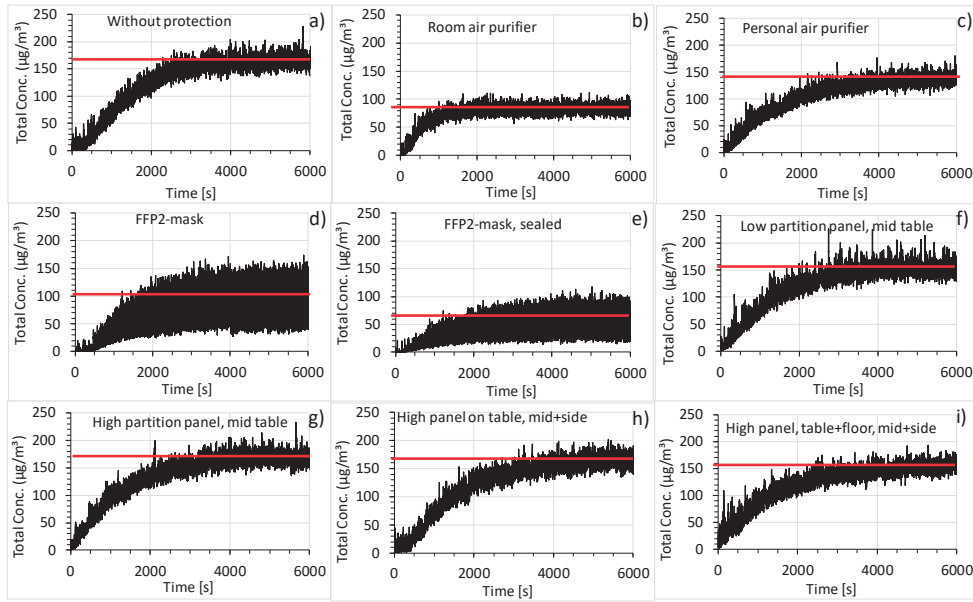
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Top-view of test chamber



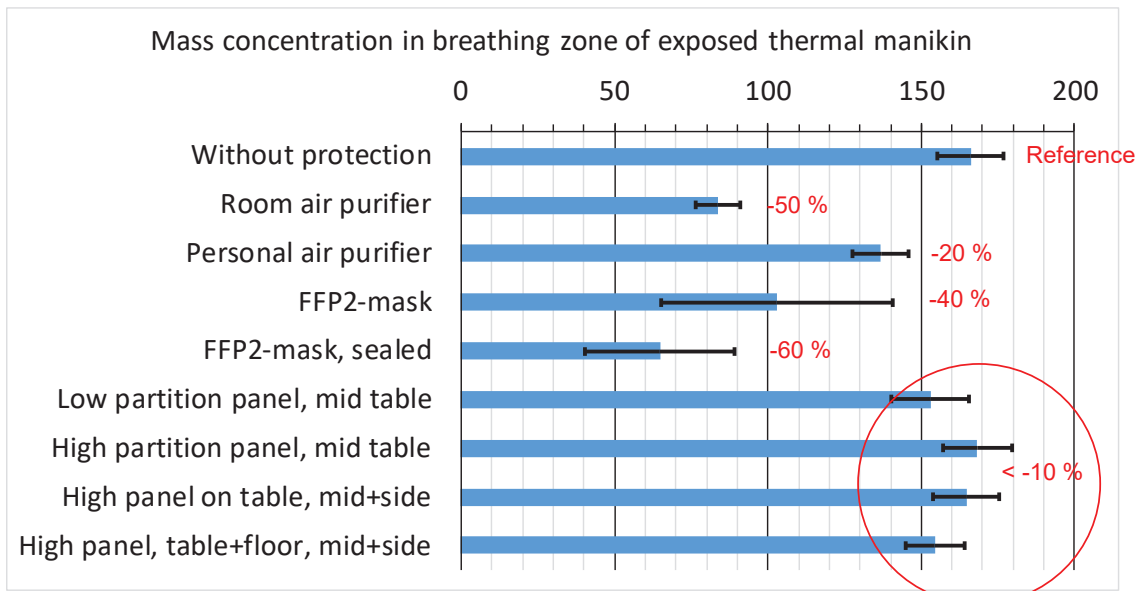
7

Results – temporal concentration characteristics



8

Results – averaged concentration at steady conditions (1000 values)



9

Conclusions

- Room air purifier reduced the average concentration level by **50%**.
- Personal air purifier reduced the average by **20%**.
- FFP2-mask had **40%** decrease on the concentration.
- Partition panels had **negligible** effects on the concentration.
- Room air purifier and FFP2-masks seem effective against exhaled aerosols in well-mixed conditions.

10



Workshop: Airborne cross infection and engineering solutions

Discussion on minimum ventilation rates for infection control

Yuguo Li 李玉國
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Faculty of Architecture
The University of Hong Kong



Thank you

Mr hard-working and smart students/post-docs (since 2003), and collaborators

HKU Mech Eng: Zhang Nan, Wang Qun, Cheng Pan, Jia Wei, Edwin Dung, Jack Chan, Zhao Pengcheng, Li Ao and many others

Former team: Qian Hua, Hang Jian and Xiao Shenglan, Liu Li, Wei Jianjian and Lei Hao

HKU/CUHK Medicine: Hui-Ling Yen, Ben Cowling, Gabriel Leung, Malik Peiris, KY Yuen, Vincent Cheng, David Lung, David Hui

CDCs: Kang Min, Ding Zhen, Hu Shixiong and many colleagues

HKSAR: Kenneth Leung, Lin Zhang, Niu Jianlei, Jimmy Fung and Alexis Lau

To RGC, HMRF, NSFC, HKU and WHO for supporting us in studying environment control of infection since 2003.

Two questions

Nature of dilution, and relationship with exposure/infection

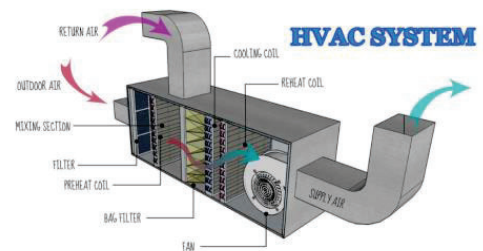
Q1: Why infection need a different standard?

Risk assessment, heterogeneity, technologies

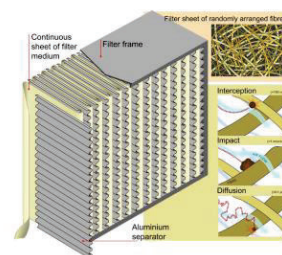
Q2: What are suitable criteria for the required dilution?

Infectious aerosols (IEs) differ from CO₂ and PM2.5

- *One Emission*
 - Everyone exhales CO₂, and multiple sources for PM2.5, but **only one individual exhales IEs in most settings**
 - IEs are only emitted when source person is present.
- *Removal by quadruplet 四胞胎*
 - CO₂ by ventilation q_v and absorption/adsorption
 - PM2.5 also by settling q_s and filtration $\eta_f q_f$, but not absorption/adsorption
 - IEs also by deactivation q_r



Internet images

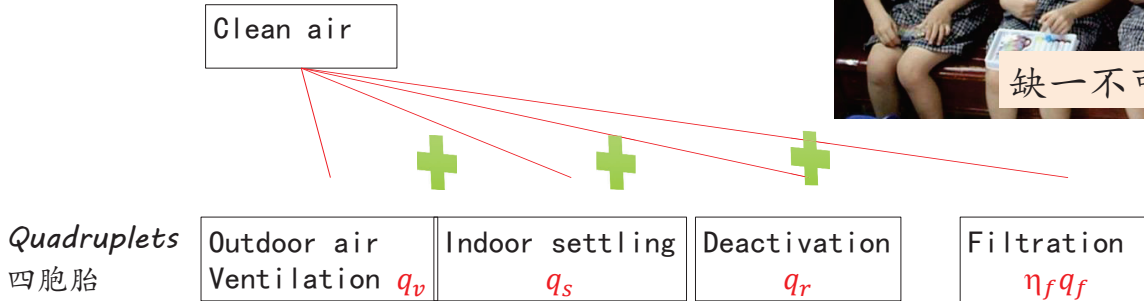


We have quadruplets of clean air 洁净空气四胞胎

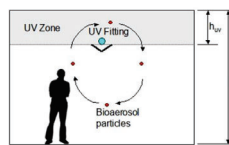
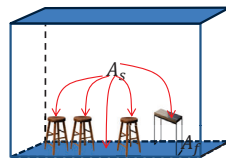
$$q_c = q_v + q_s + q_r + \eta_f q_f = \sum_{j=1}^4 q_{cj}$$



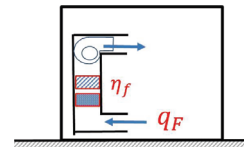
Internet image



Inter-room air flows can also introduce "clean air" for infectious aerosols in multi-zone spaces



Noakes et al



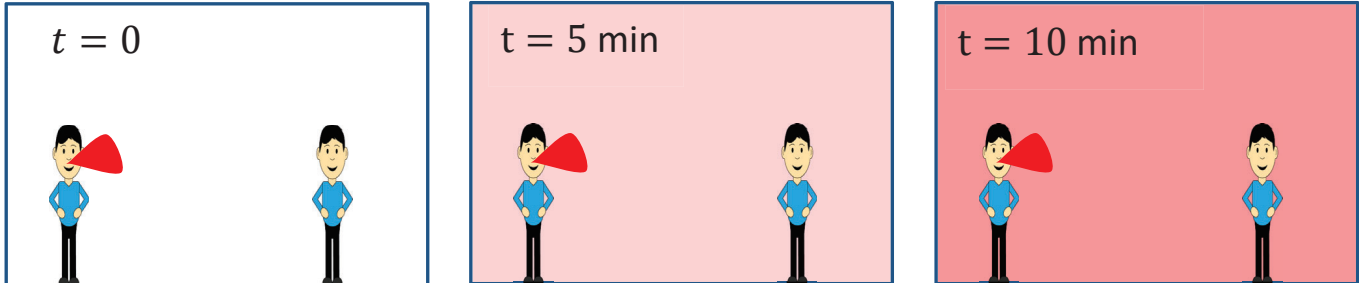
Examples: six outbreak venues

- Spacious space (courtroom and church hall): significant contribution from natural decay and settling.
- Crowded space (bus 1, 2): negligible contribution from natural decay and settling

	Parameters	courtroom	church hall	bus 1	bus 2	restaurant 1	restaurant2
outbreak and venue	attack rate (%)	33.3	86.7	15.2	11.8	10.2	28.9
	no. of susceptible	9	60	46	17	88	76
	exposure time (min)	180	150	200	60	72.4	57
	air volume per person (m ³)	16.7	13.5	1.3	1.3	4.9	5.2
air flow rate (L/s per person)	ventilation	1.8	2.6	1.8	3.4	0.9	1.7
	filtration	0.0	0.0	0.0	0.0	5.5	0.0
	deactivation	2.9	2.4	0.2	0.2	0.9	0.9
	settling	1.4	1.1	0.1	0.1	0.4	0.4
	clean-air flow rate	6.1	6.1	2.1	3.7	7.7	3.1

Clean-air still does not fully characterize the room air dilution ability

Spaciousness means dilution



Li et al. Threshold dilution of airborne infection in buildings. To be submitted

Quadruplets of clean air + triplets of non- q_c factors



triplets of non- q_c factors
三胞胎

dilution air q_d

$$q_d = E_z \times \frac{1}{1-c_t} \times \frac{1}{\eta_{ml}\eta_{ms}} \times q_c$$



clean air q_c

spatial distribution

temporal distribution

mask wearing

$$q_c = \sum_{j=1}^4 q_{cj}$$

quadruplets of clean air
四胞胎



outdoor air ventilation q_v

gravity settling q_s

deactivation q_r

filtration $\eta_f q_f$

Inter-room air flows can also introduce "clean air" for infectious aerosols in multi-zone spaces

Dilution-air flow rate should be determined in outbreak investigations

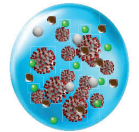
- Spaciousness can introduce significant dilution ability
- A crowded space with 1.5 m³/p introduces negligible dilution

	Parameters	courtroom	church hall	bus 1	bus 2	restaurant 1	restaurant2
outbreak and venue	attack rate (%)	33.3	86.7	15.2	11.8	10.2	28.9
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	settling	1.4	1.1	0.1	0.1	0.4	0.4
	clean-air flow rate	6.1	6.1	2.1	3.7	7.7	3.1
	dilution air per person	8.1	8.1	2.2	4.1	9.0	5.4

9

Li et al. Threshold dilution of airborne infection in buildings. To be submitted

A generalized Wells-Riley equation: individual heterogeneity, spatial non-uniformity, temporal variation, and multi-virion aerosols



$$\bar{p}_i = \frac{1}{N_\sigma} \sum_{i=1}^{N_\sigma} \left(1 - e^{-\sum_{n=1}^{\infty} [1 - (1 - r_i)^n] \int_{t_{1,i}}^{t_{2,i}} q_{in,i}(t) \sum_{j=1}^J c_{D,j}(x,t) \frac{\lambda_j^n e^{-\lambda_j}}{n!} dt} \right)$$

$$\bar{c}_{D,j} = \frac{1}{q_{d,j}} \sum_{k=1}^{N_I} L_{D,j,k}$$

$$\frac{N_I}{N_\sigma} = 1 - e^{-\frac{N_I Q}{q_D} \bar{q}_{in} \bar{\Delta} t}$$

Achieved (effective) dilution-air flow rate $q_{d,a}$

$N_\sigma, \bar{\Delta} t$ human behavior
 \bar{q}_{in} physiology
 Q virology
 N_I epidemiology
 q_D building environment

$$q_{d,a} = \frac{q_D}{N_\sigma} = \frac{E_z q_c}{1 - C_t}$$

$$C_t = \frac{1 - e^{-n\Delta t}}{n\Delta t}$$

Dilution air is a unified concept for removal of infectious aerosols

- Steady and uniform Wells-Riley equation can be

used. $p_i = \frac{N_t}{N_\sigma} = 1 - e^{-\frac{N_I Q}{q_D} \bar{q}_{in} \bar{\Delta} t}$

- With an **acceptable risk level (threshold)**, the threshold dilution q_D or q_d can be determined if quantum emission is known.

How to determine the infectious quantum emission of the population?

11

Two questions

Nature of dilution, and relationship with exposure/infection

Q1: Why infection need a different standard?

Risk assessment, heterogeneity, technologies

Q2: What are suitable criteria for the required dilution?

What risk?

- no infection (everyone in a room is safe),
- no epidemic (someone might be infected, but most people are safe),
- no pandemic?

Should different settings differ in dilution requirement?



43rd AIVC conference, 4-5 October 2023, Copenhagen
NS: Airborne cross infection and engineering solutions

Mitigation of airborne transmission of respiratory viruses by ventilation - past, present and future



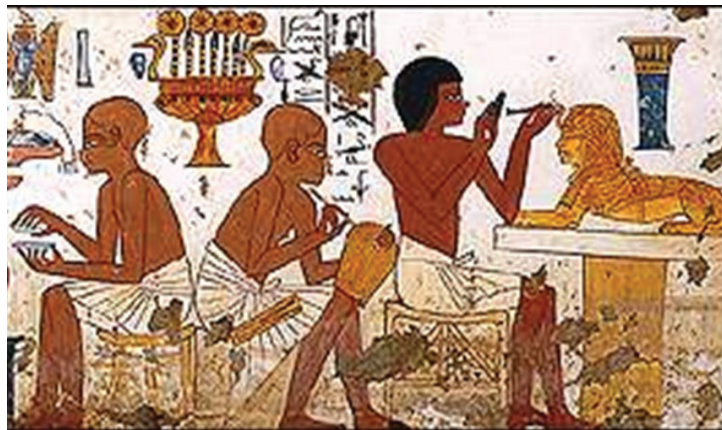
Arsen K. Melikov

ISIAQ Fellow, ASHRAE Fellow, Life member

International Centre for Indoor Environment and Energy, DTU Sustain, Technical University of Denmark
akme@dtu.dk

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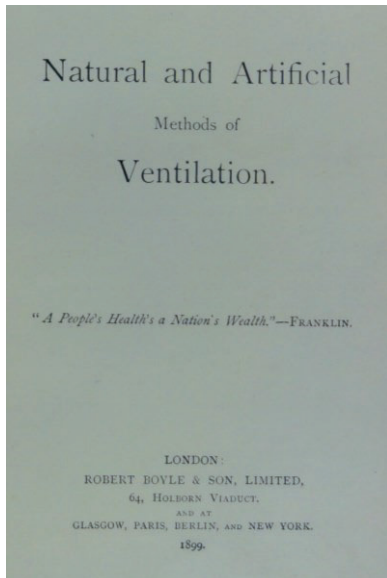
Ventilation and Airborne Transmission: In the Past



Ancient Egyptians used natural ventilation to remove dust and to reduce respiratory diseases of stone carvers working indoors.

2

Ventilation and Airborne Transmission: In the Past



Chapter X.

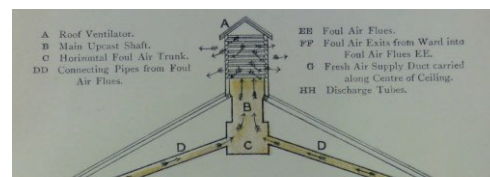
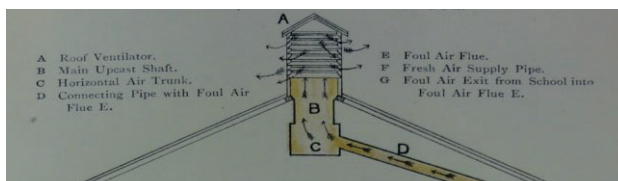
How Infection is Spread

"The report on the influenza epidemic presented to Parliament by the Local Government Board indicates the extreme importance of **proper ventilation** – especially in schools – which is pronounced to be **the only real safeguard against that disease.**"

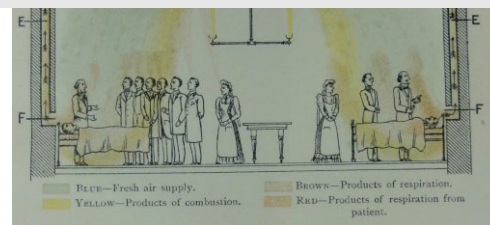
3

Ventilation and Airborne Transmission: In the Past

Book: Natural and Artificial Methods of Ventilation
Robert Boyle & Son Limited, London 1899



Clean (disinfected) ventilation air distribution is important!



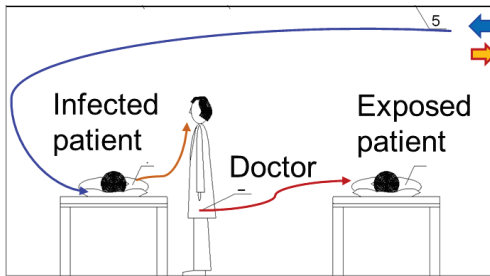
4

Ventilation and Airborne Transmission: At Present

REHVA & ASHRAE guidelines: Increase ventilation rate!

↳ May generate directional flow and may increase transmission!

➤ Bolashikov, Melikov et al. 2012, HVAC&R Res.; Pantelic & Tham, 2013, HVAC&R Res.,



Air distribution:

- **reduces** concentration of infectious agents and
- **spreads** infectious agents

Increase of ventilation rate leads to problems:

- Expensive: rebuilding of existing systems/huge new systems
- High velocities: draught, resuspension of particles/viruses
- Energy consumption is increased

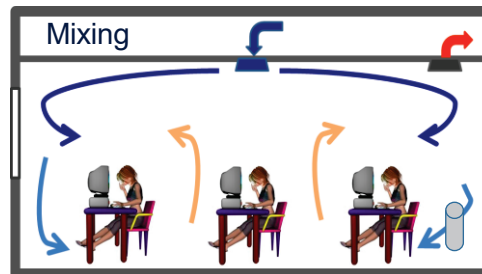
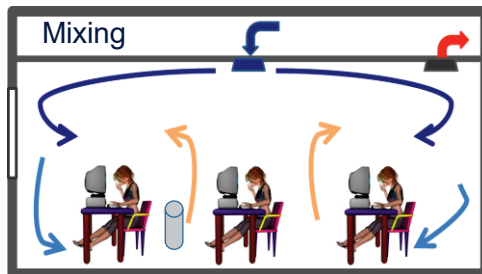
5

Reduction of Airborne Transmission by Air Cleaners



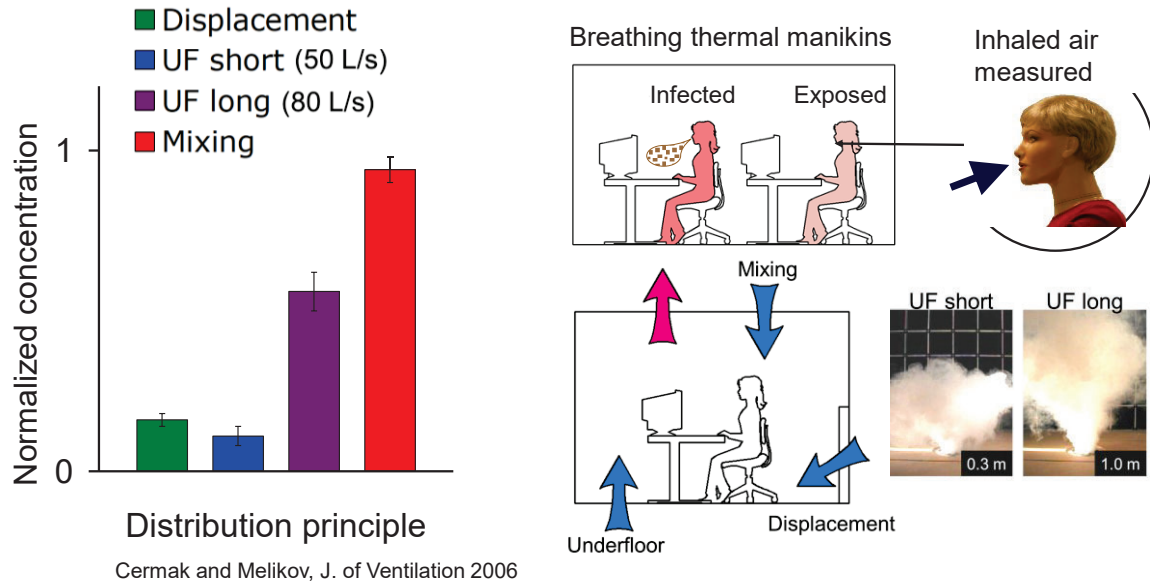
Performance of air cleaners depends:

- Design of air cleaner
- Capacity of air cleaner & generated flow
- Positioning in space
- Background air distribution
- Location of infected occupant
- Occupants' activity



6

Airborne Transmission: Importance of Air Distribution



7

Ventilation and Airborne Transmission: In the Future

Past and present ventilation:

- Reduces long-range airborne transmission
- Distancing & face masks needed for reduction of short range transmission
- Present ventilation requirements lead to increase of energy use

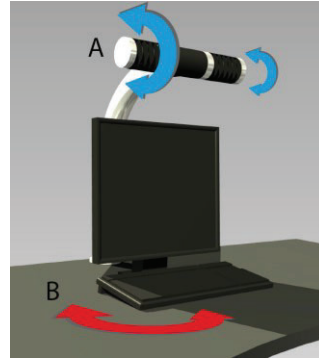
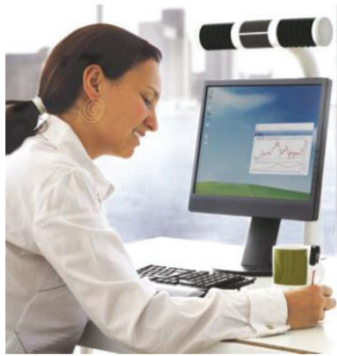


Future ventilation:

- Focus on reduction of short range transmission
- Reduced ventilation rate, small HVAC systems, energy savings

8

Advanced Air Distribution: Personalised Ventilation



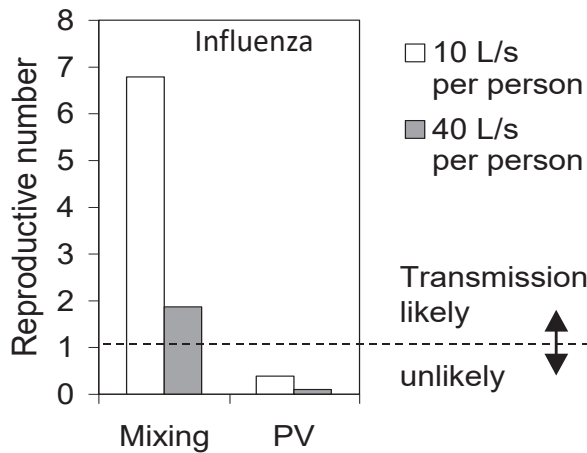
Melikov, Indoor Air J. 2004

Curtesy to "Exhausto" (Denmark)

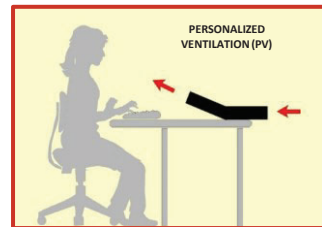
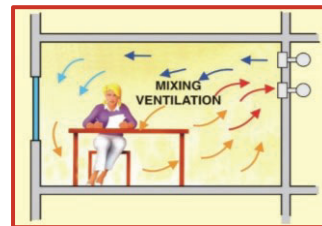
9

Personalized Ventilation vs. Mixing Ventilation

R_{A0} - reproductive number of secondary infections that arise when a single infectious case is introduced into a population where everyone is susceptible
 [Rudnick and Milton, 2003]



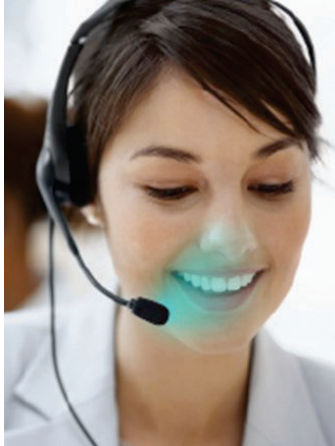
➤ Cermak & Melikov, HVAC&R 2007



10

Ventilation & Air Cleaning

Headset Incorporated Personal Ventilation



Mode 1: Clean air supply

Mode 2: Exhaled air removal



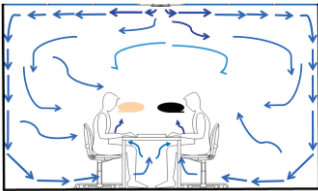
Inhaled clean air: 90% Exhaled air captured: 80%

Bolashikov et al. HB 2003; Zhu et al. Indoor Air 2008; Bolashikov et al. S&T for the Built Environment 2015

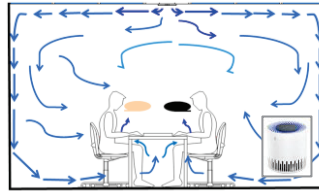
11

Reduction of Airborne Transmission: Approaches

Ventilation



Ventilation+ Air Cleaner



Clean air supply



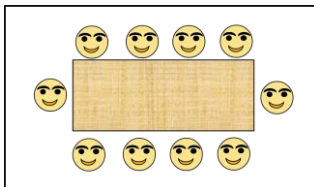
Bolashikov et al. 2003; Zhu et al. 2008

Exhale air removal

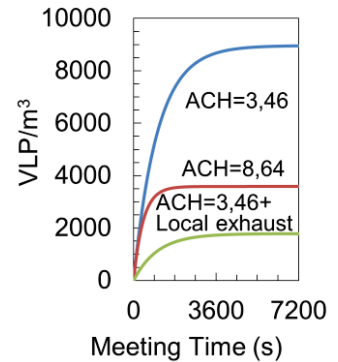


Bolashikov et al. 2015 Melikov, BAE, 2020

Comparison:



- Meeting room, 10 occupants
- 20 m², 50 m³, Low Polluting Building
- Category III: 48L/s; Category I:120 L/s
- One infected occupant
- 2850 virus laden particles/exhalation (VLP)
- Calculations for 2 hours meeting time
- Local exhaust/supply efficiency: 80%
- Air cleaner efficiency: 100%

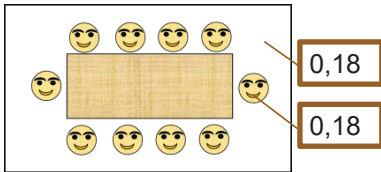


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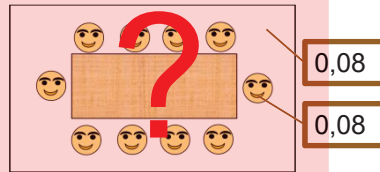
Reduction of Airborne Transmission

iF (intake fraction) = mass intake of infected air/ mass exhaled infected air (%) - 2 h meeting (Bennett et al. 2002)

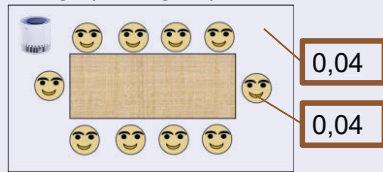
Mixing ventilation (48 L/s)



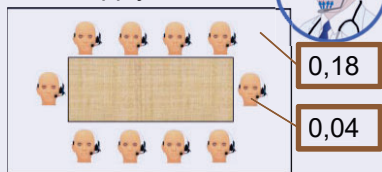
Mixing ventilation (120 L/s)



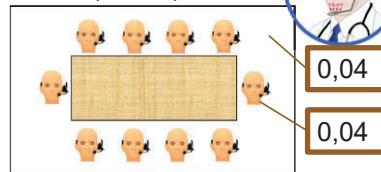
Mixing + Air cleaner
48 L/s + 192 L/s



Mixing ventilation (48 L/s)
+ Local supply



Mixing + Local exhaust
(48 L/s)

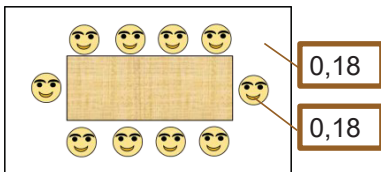


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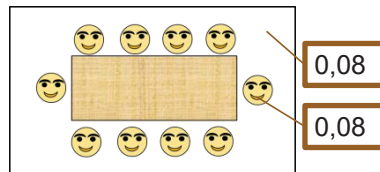
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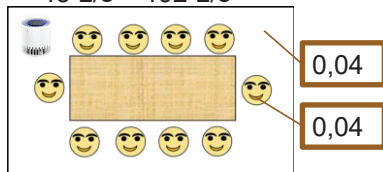
Mixing ventilation (48 L/s)



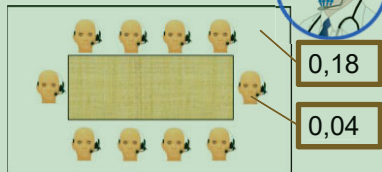
Mixing ventilation (120 L/s)



Mixing + Air cleaner
48 L/s + 192 L/s



Mixing ventilation (48 L/s)
+ Local supply



Mixing + Local exhaust
(48 L/s)



14

Reduction of Airborne Transmission: Future Ventilation Design

Future solutions

- Increase system size?
 - Increase volume of spaces?
 - Increase of ventilation rate?
- Not feasible**
- Increased cost
 - Poor operation
 - Increased energy consumption

Focus on

- Source control (exhaled air removed close to face)
 - Reduction (not increase!) of ventilation rate
- Clean air supply to the breathing zone
 - Advanced ventilation
 - Air disinfecting clothing

G. Settels, Penstate Univ.



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43rd AIVC conference, 4-5 October 2023, Copenhagen
NS: Airborne cross infection and engineering solutions

Focus on the Air Distribution and the Occupant!
THERE IS A LOT TO BE DONE!

Thank you

Arsen K. Melikov

ASHRAE Fellow, Life member

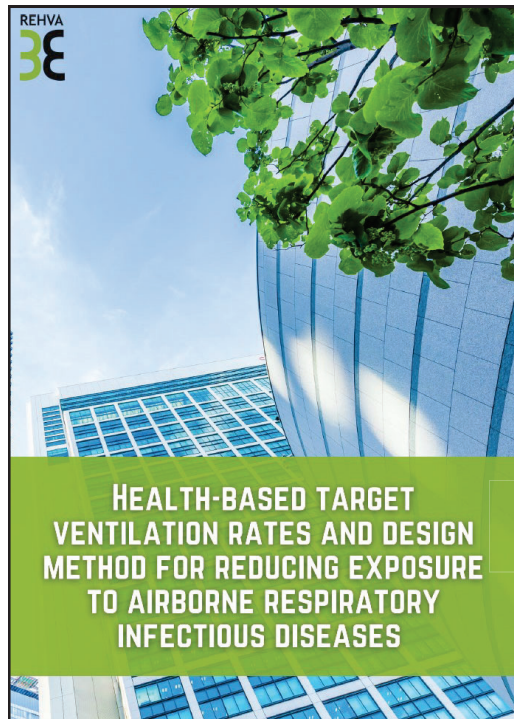
International Centre for Indoor Environment and Energy, DTU Sustain, Technical University of Denmark
akme@dtu.dk

16

POINT SOURCE VENTILATION EFFECTIVENESS IN INFECTION RISK-BASED VENTILATION DESIGN

Jarek Kurnitski
 Tallinn University of Technology, Aalto University,
 REHVA Technology and Research Committee

1



Target outdoor air ventilation rates Q (L/s) calculated using the number of persons in room N (-) and the room volume V (m³)

Space category	Ventilation rate, L/s
Classroom	$Q = 10(N-1) - 0.24V$
Office	$Q = 23(N-1) - 0.24V$
Assembly hall	$Q = 30(N-1) - 0.24V$
Meeting room	$Q = 40(N-1) - 0.24V$
Restaurant	$Q = 40(N-1) - 0.24V$
Gym	$Q = 70(N-1) - 0.24V$

Design ventilation rate supplied by the ventilation system:

$$Q_s = \frac{Q}{\epsilon_b}$$

ϵ_b , point source ventilation effectiveness for the breathing zone (-)
 (contaminant removal effectiveness, ventilation factor)

<https://www.rehva.eu/activities/post-covid-ventilation>

2

2

INFECTION RISK MODELING

- Standard airborne disease transmission Wells-Riley model calibrated to COVID-19 with the correct source strength (quanta emission rates)
- Viral load emitted is expressed in terms of the quanta emission rate (quanta/h). One quantum is defined as the dose of airborne droplet nuclei required to cause infection in 63% of susceptible persons
- The probability of infection (p) is related to the number of quanta inhaled (n):

$$p = \frac{N_c}{N_s} = 1 - e^{-n} = 1 - e^{-\frac{IqQ_bD}{Q}} = 1 - e^{-CQ_bD}$$

p	probability of infection for a susceptible person (-)
N_c	number of disease cases
N_s	number of susceptible persons in the room
I	number of infectious persons (-)
n	number of quanta inhaled (quanta)
q	quanta emission rate per infectious person (quanta/(h pers))
Q_b	volumetric breathing rate of an occupant (m ³ /h)
Q	outdoor air ventilation rate for the breathing zone (m ³ /h)
D	duration of the occupancy (h)
C	average quanta concentration in the room (quanta/m ³) ³

QUANTA CONCENTRATION C UNDER STEADY STATE

Pollutant/quanta mass balance

Q_s Q_s
 $C_0=0$ C_e

$$Iq = C_e \lambda_v V + C \lambda_{rest} V$$

C_i C_{i+1} C C_n

C_e	quanta concentration in the extract air (quanta/m ³)
C	average quanta concentration in the breathing zone (quanta/m ³)
C_0	quanta concentration in the supply air (quanta/m ³)
V	volume of the room (m ³)
λ_v	outdoor air change rate, i.e., removal rate due to ventilation (1/h)
λ_{rest}	other removal mechanisms than ventilation (1/h)

Iq

Point source ventilation effectiveness is defined:

$$\varepsilon_b = \frac{C_e - C_0}{C - C_0}$$

ventilation rate supplied by the ventilation system $Q_s = \lambda_v V$ (m³/h)

$$C = \frac{Iq}{\varepsilon_b Q_s + \lambda_{rest} V}$$

$$p = 1 - e^{-\frac{IqQ_b D}{\varepsilon_b Q_s + \lambda_{rest} V}}$$

INFECTION RISK CONTROL CONCEPT BASED ON R AND PRE-SYMPTOMATIC PERIOD

- Considering that N_s susceptible persons are exposed to C , the **event reproduction number R** (new disease cases per infectious person) can be calculated

$$R = \frac{pN_s}{I}$$

- R based on pre-symptomatic period of 2.5 days: $9/22.5=0.4$ in offices and $2/22.5=0.09$ in meeting rooms (Kurnitski et al. 2023 <https://doi.org/10.1016/j.enbuild.2023.113386>)
- For given R value, ventilation rate for the breathing zone Q can be solved from equation of p because:

$$Q = \varepsilon_b Q_s$$

Q target ventilation rate for the breathing zone = outdoor air ventilation rate at fully mixing

AVERAGE CONCENTRATION C VS. LOCAL C_i

- Probability of infection can be calculated at each measured location i . Pollutant/quantum mass balance applied for each measurement point with concentration C_i :

$$Iq = C_e \lambda_v V + C_i \lambda_{rest} V$$

- Local air quality index is defined:

$$\varepsilon_{P,i} = \frac{C_e - C_0}{C_i - C_0}$$

- Substituting C_e and taking into account that ventilation rate supplied by the ventilation system $Q_s = \lambda_v V$ one obtains:

$$C_i = \frac{Iq}{\varepsilon_{P,i} Q_s + \lambda_{rest} V}$$

$$p_i = 1 - e^{-\frac{Iq Q_b D}{\varepsilon_{P,i} Q_s + \lambda_{rest} V}}$$

R IS THE SUM OF FRACTIONS OF THE EVENT REPRODUCTION NUMBER R_i

- Considering that $N_{s,i}$ susceptible persons are exposed to C_i at location i , the fraction of the event reproduction number R_i (new disease cases per infectious person) can be calculated

$$R_i = \frac{p_i N_{s,i}}{I}$$

- The event reproduction number forms from the exposure of all susceptible persons

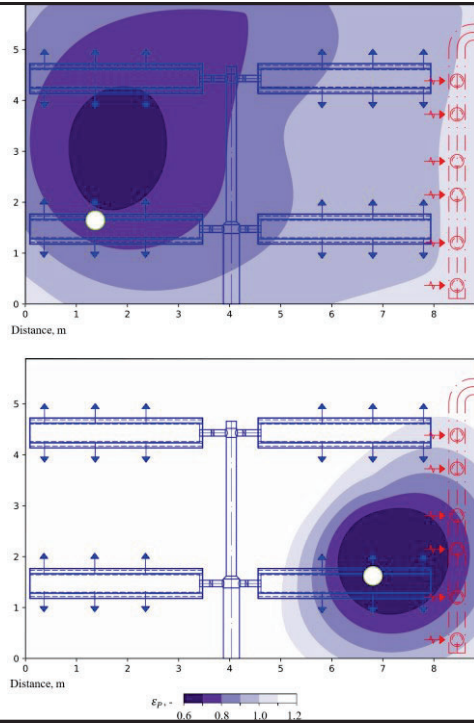
$$R = \sum_i R_i$$

- For given R value, ventilation rate supplied by the ventilation system Q_s can be iteratively solved. Finally:

$$\varepsilon_b = \frac{Q}{Q_s}$$

RESULTS

- Examples of tracer gas measurements
- Meeting room of 52.5 m² with active chilled beams and 3.0 L/(s m²) ventilation
- Local air quality index values with left and right locations of point source
- Concentrations/ventilation effectiveness depends on the source location

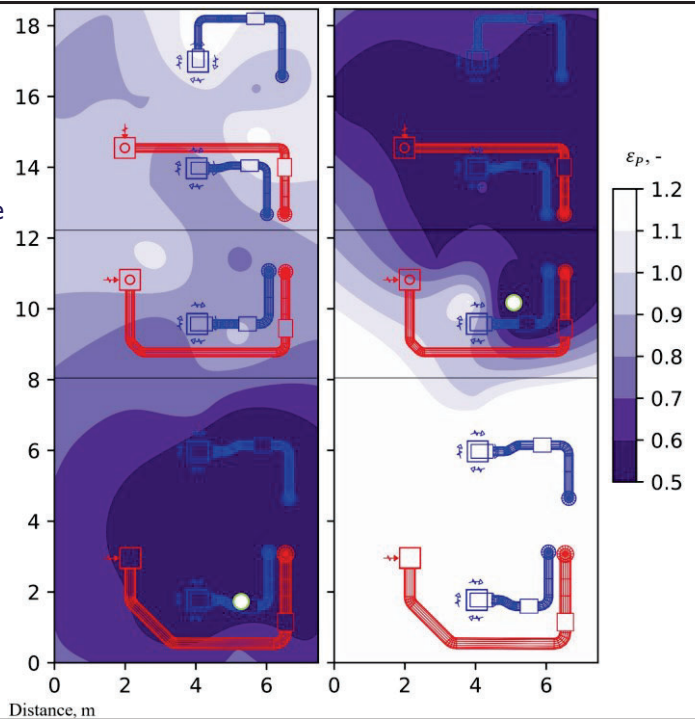


9

9

RESULTS

- Local air quality index values with two locations of point source in the large teaching space of 129.5 m² with 4 L/(s m²)
- The larger room the larger differences



10

RESULTS

- No difference in small rooms
- Up to 26% difference in large rooms with large concentration differences
- Calculation from average concentration provides conservative result

Room	Measurement	ε_b	ε_b , avg	Q_s , L/s	Q_s , avg, L/s
Classroom 30.5 m ² , 13 persons	Meas. No 1	0.99	0.99	70	70
	Meas. No 2	1.89	1.88		
	Average	1.44	1.43		
	%		-0.4%		
Teaching space 129.5 m ² , 50 persons	Meas. No 1	0.77	0.76	513	532
	Meas. No 2	0.81	0.77		
	Average	0.79	0.76		
	%		-3.6%		
Gym 173.5 m ² , 12 persons	Meas. No 1	0.56	0.53	548	627
	Meas. No 2	1.80	1.53		
	Average	1.18	1.03		
	%		-12.6%		
School gym 217.5 m ² , 25 pers.	Meas. No 1	0.74	0.52	2695	3401
	Meas. No 2	0.36	0.35		
	Average	0.55	0.44		
	%		-20.8%		
Meeting room 52.5 m ² , 12 persons	Meas. No 1	0.86	0.86	404	412
	Meas. No 2	1.18	1.14		
	Average	1.02	1.00		
	%		-2.0%		
Open plan office 173 m ² , 17 pers.	Meas. No 1	0.50	0.34	406	511
	Meas. No 2	0.74	0.64		
	Average	0.62	0.49		
	%		-20.5%		
				2695	3401
					26.2%
				406	511
					25.8%

CONCLUSIONS

- Point source ventilation effectiveness was calculated from average and local tracer gas concentrations
- Calculation from average concentration is straightforward, but provides conservative results (= higher ventilation rate to be supplied)
- Calculation from local concentrations and individual risk of probabilities needs iteration but provides more accurate results in spaces with large concentration differences
- Max difference 26% in the airflow rate (school gym and open plan office)
- Infection risk based ventilation design is proposed to be addressed in the ongoing revision of EN 16798-1

Acoustic method for measurement of airtightness – Field testing on three different existing office buildings in Germany

Björn Schiricke, Benedikt Kölsch
Solar Research – German Aerospace Center (DLR)



1

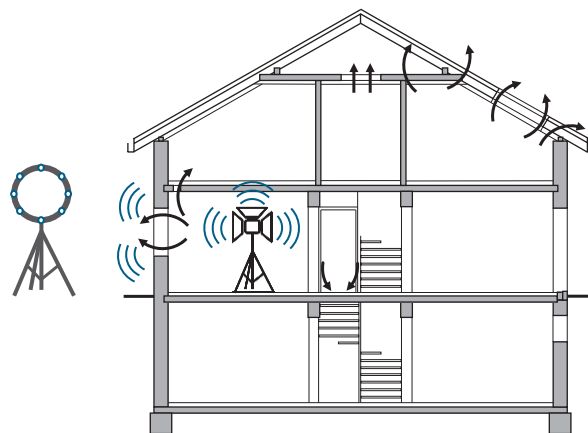
bjoern.schiricke@dlr.de, AIVC Copenhagen, 04.10.2023

1

Acoustic Approach



- **Objective**
Identify leak locations in building envelopes quickly
- **Assumption**
Sound takes predominantly the same paths as air in fan pressurization method
- **Equipment**
 - Speaker
 - Microphone array
- **Advantages**
 - Independent from pressures and temperatures
 - Scanning of large areas possible
- **Previous testing**
 - Successful on known leaks



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2

Field testing on three different office buildings

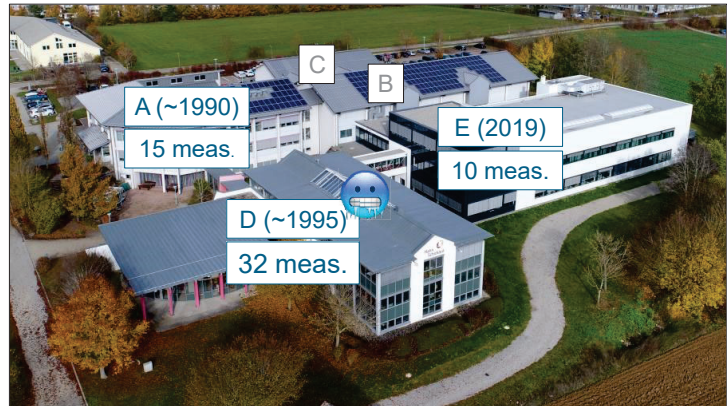


Testing this method...

- Large building complex
- Different building types
- Unknown leaks

Measurements evaluated

- 57 measurements (in 37 rooms)
- Smoke sticks in 3 rooms for validation



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3

Measurement setup



Speakers inside

High frequency speakers
15 - 120 kHz

Mid and low frequency speakers
0.05 - 16 kHz

White noise signal
85 dB for 4 s



Microphone array outside

48 microphones
0.2 - 60 kHz

Video camera
1920 × 1080 px

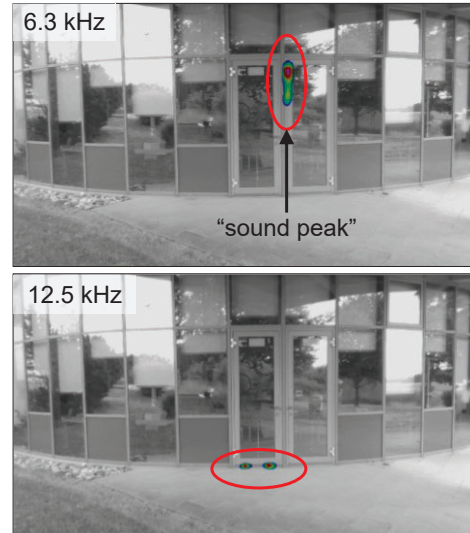


4

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4

Leak detection on facades



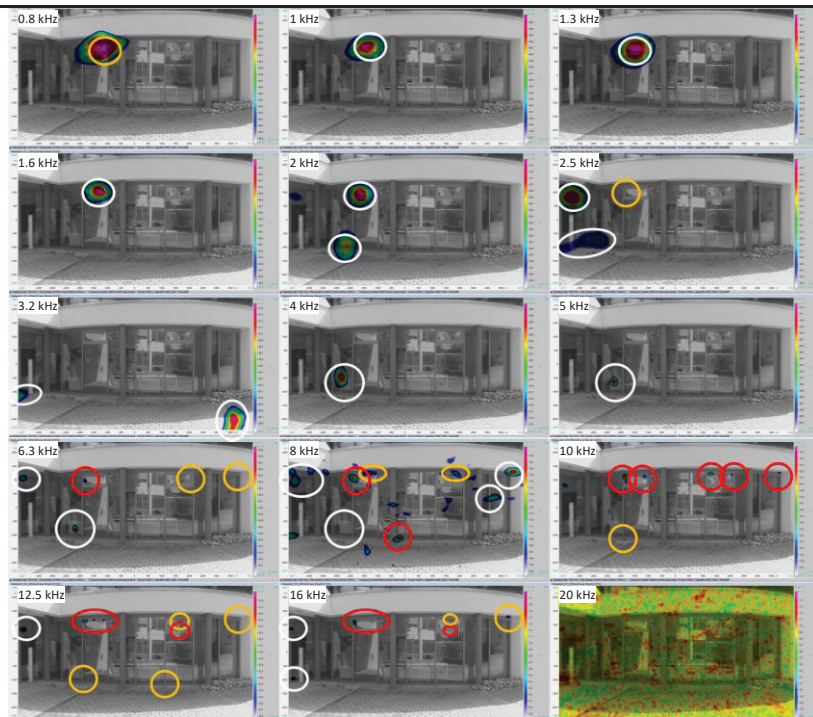
- Analysed in 16 third-octave frequency bands 0.8 – 25 kHz
- Different potential leaks are found in different frequency bands

5

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5

Visualisation of potential leaks

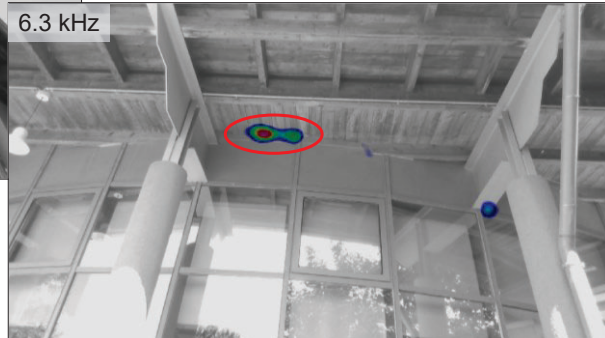


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6

Examples



Transition from the wall to the roof

7

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7

Examples – confirmed by visual inspection



Cover protrudes and the window seal is faulty

8

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8

Examples – confirmed by visual inspection



6.3 kHz



Cracks in the plaster inside and outside

9

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Examples – confirmed by visual inspection



6.3 kHz



Mould

10

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10

Examples – confirmed by smoke stick



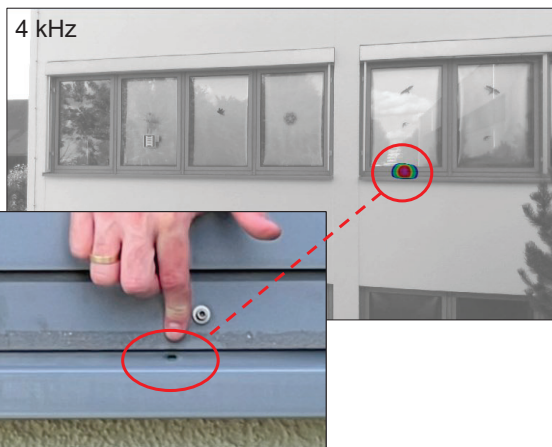
Hole in window seal

11

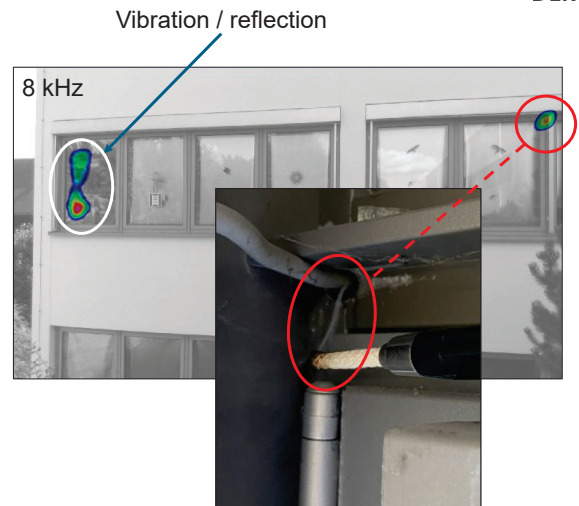
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11

Examples – confirmed by smoke stick



Hole in window frame



Cable fairlead to the blind

12

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12

Categorisation of the acoustic signals



Colour Code	Acoustic Assessment Score	Evaluation of acoustic signals	Description of subjective criteria
○	0	very unlikely leakage	Peak of signal is at implausible location (e.g. on a window pane or facade panel; or outside the area under consideration)
●	1	unlikely leakage	Some indications of leakage
●	2	likely leakage	Peak of signal at different plausible locations
●	3	very likely leakage	Peak of signal at plausible location

Room		Third-octave frequency bands in kHz														Multi Frequency Assessment Score		
Name	note	0.8	1	1.3	1.6	2	2.5	3.2	4	5	6.3	8	10	12.5	16	20	25	
E-Büro 2	2.floor (OG)	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	14
E-Büro 1	2.floor (OG)	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	20
E-Büro	2.floor (EG)	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	5
E-Büro	2.floor (EG)	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	8
E-Bespr.	1.floor (UG)	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	9
E-Büro 1	1.floor (UG)	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	21
E-Büro	1.floor (UG)	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	15
E-Büro	1.floor (UG)	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	24
E-Aufenth.	2.floor (EG)	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	10
E-Büro	1.floor (UG)	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	13

The more ● or even ● → the higher the acoustic indications of leaks

Results



Room		Third-octave frequency bands in kHz														Multi Frequency Assessment Score		
Name	note	0.8	1	1.3	1.6	2	2.5	3.2	4	5	6.3	8	10	12.5	16	20	25	
A-101	SW	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	12
A-101	SW - detail	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	15
A-103	NW	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	9
A-103	NW	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	16
A-201/1	detail	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	3
A-201/2	detail	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	13
A-201/3	detail	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	15
A-201/3	NW	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	16
A-202	SW rep. 1	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	12
A-202	SW rep. 2	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	19
A-202	SW rep. 3	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	16
A-102	SW rep. 3	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	8

Room		Third-octave frequency bands in kHz														Multi Frequency Assessment Score		
Name	note	0.8	1	1.3	1.6	2	2.5	3.2	4	5	6.3	8	10	12.5	16	20	25	
D-112	S up	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	12
D-112	SSW	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	15
D-112	SW	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	16
D-112	SW	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	16
D-112	NW	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	16
D-112	NW	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	16
Corridor	N	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	19
Corridor	N detail	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	19
D-109	W	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	14
D-108	S	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	18
D-107	W	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	11
D-106	S	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	11
D-106	E	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	16
D-103	E	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	19
D-211	S	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	20
D-210	S	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	18
D-209	S	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	11
D-208	W	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	20
D-207	S	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	17
D-206	S	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	13
D-206	E	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	18
D-205	S	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	19
D-204	S	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	19
D-203	S	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	20
D-202	S	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	17
D-201	S	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	19
Bridge	N	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	18
Bridge	N detail	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	21

The more ● or even ● → the higher the acoustic indications of leaks

→ Acoustic method managed to discern differences among the buildings

Summary and Outlook



Achievements

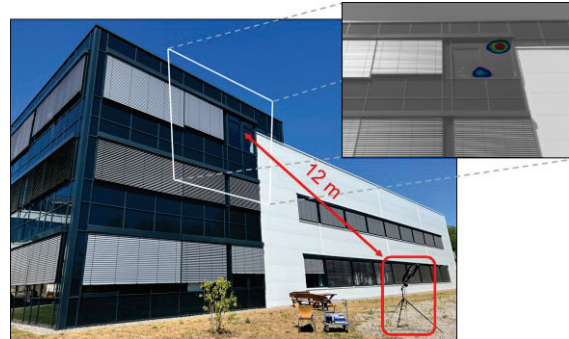
- Significant number of potential leaks localized
- Scanning of large areas possible
- Large distances were demonstrated (3. floor)
- Managed to discern differences among the three buildings

→ **Successful demonstration of acoustic approach to leak detection**

Outlook

- Automation of data analysis
- Compact visualisation of potential leaks
- Laboratory test stand
 - Systematic investigation of various leakage setups
 - Infer the type and size of leaks (e.g. out of spectral data)

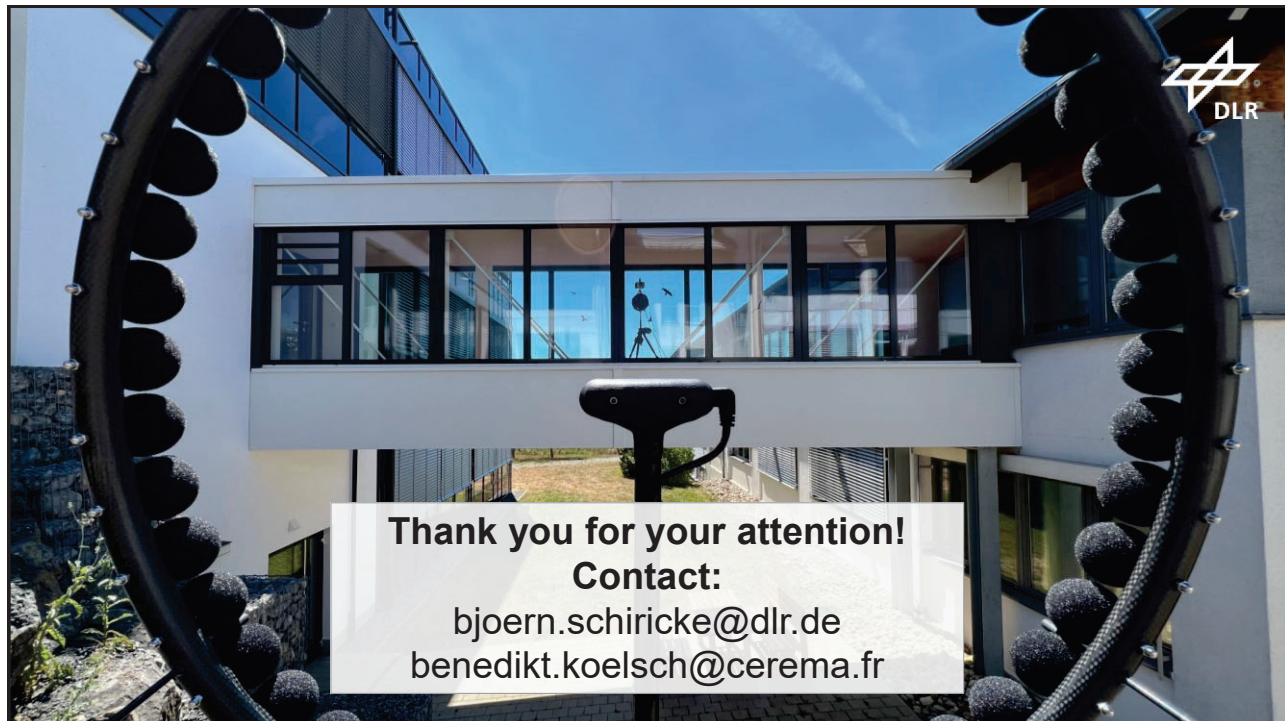
→ **New research projects ongoing**



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15



Thank you for your attention!

Contact:

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Imprint



Topic: Acoustic method for measurement of airtightness – field testing on three different existing office buildings in Germany

Date: 2023-10-04

Author: Björn Schiricke

Institute: Solar Research

Image credits: All images “DLR (CC BY-NC-ND 3.0)” unless otherwise stated

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Examples – confirmed by **visual inspection**



Tilt window and hole in steel structure

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Examples – confirmed by visual inspection



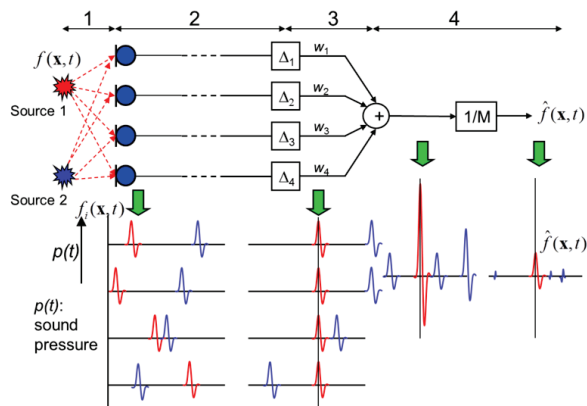
Window seals are damaged

19

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19

Delay and Sum Beamforming



1. Sound arrives at microphones.
2. Signals have phase differences → Offset can be calculated by sound velocity, distances between microphones and to the sound source.
3. Depending on the focus point, signal is shifted by the delay time difference.
4. All signals are added up and normalized with number of microphones.

20

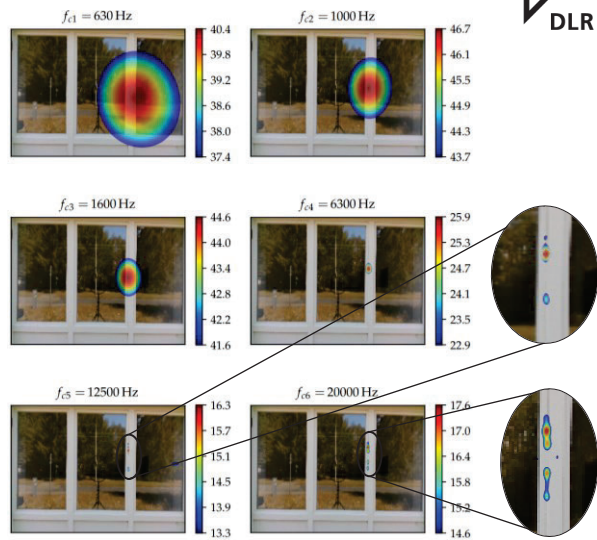
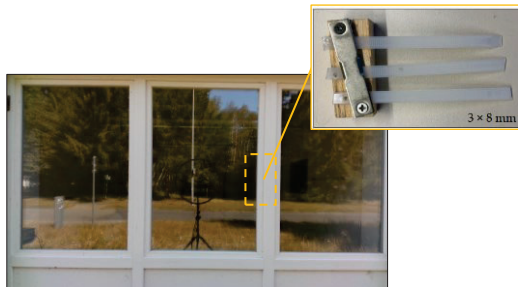
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Background information

Previous testing *

- Single window frame
- With known leaks



* Kölsch et al: "Detection of Air Leakage in Building Envelopes using Microphone Arrays", AIVC 2021 in Athens

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Pulse tests in highly airtight Passivhaus standard buildings

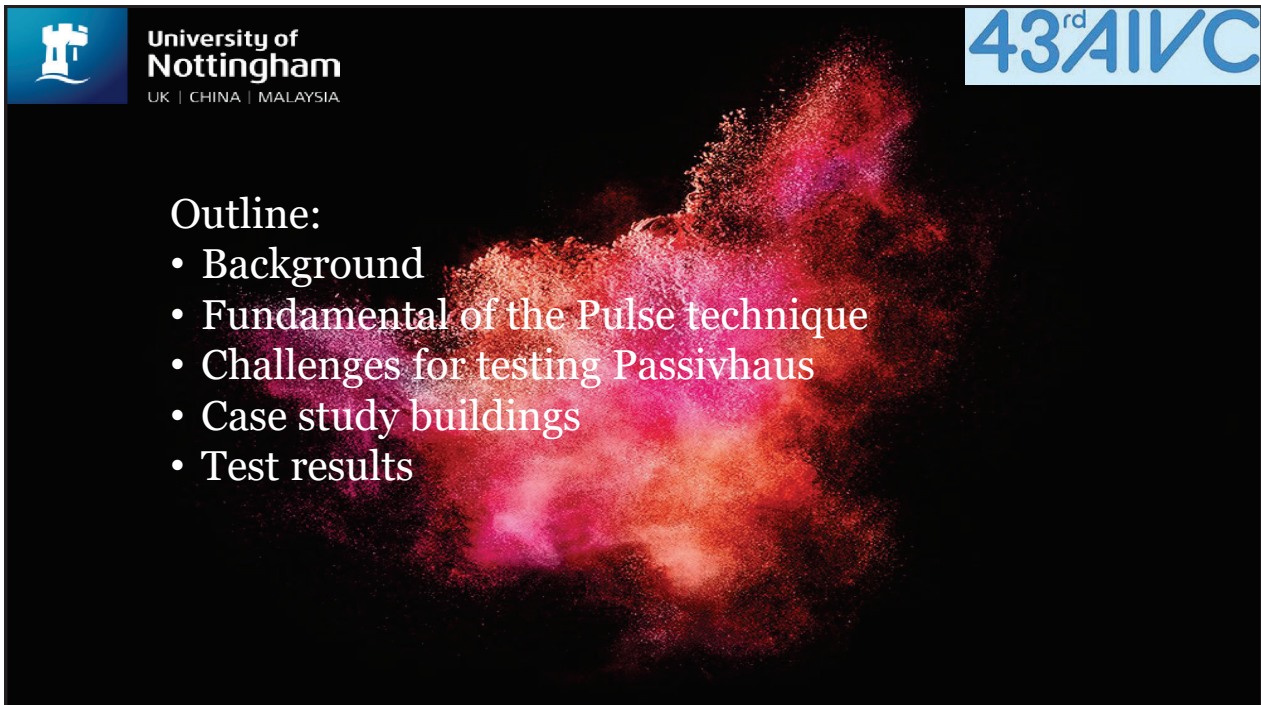

Xiaofeng (Ken) Zheng

Buildings, Energy and Environment Research Group
Faculty of Engineering, University of Nottingham, United Kingdom

1



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Outline:

- Background
- Fundamental of the Pulse technique
- Challenges for testing Passivhaus
- Case study buildings
- Test results

2

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Consequence of building air leakage

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Air Leakage Consequences

```

    graph TD
      AL[Air Leakage] --> TC[Thermal Comfort]
      AL --> AQ[Air Quality]
      AL --> MD[Moisture Damage]
      AL --> HS[HVAC Sizing]
      AL --> CL[Convective Loops]
      
      TC --> IE[Indoor Environment]
      AQ --> IE
      
      MD --> D[Durability]
      
      HS --> EE[Energy Efficiency]
      CL --> EE
      
      CL --> WW[Wind Washing]
      WW --> EE
      
      EE -.-> H[Leaky, Old, Modern, New, Low, Passive]
  
```

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BUILT2SPEC Tools for the 21st Century Construction Worksite

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Conventional steady fan pressurisation method

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Flow meter

Pressure gauge

Door fan

Building

Pressure tap

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The pulse technique is implemented by releasing a measured amount of compressed air from an air tank over short period of time (1.5 s) to the test space and monitor the pressure response in the space.

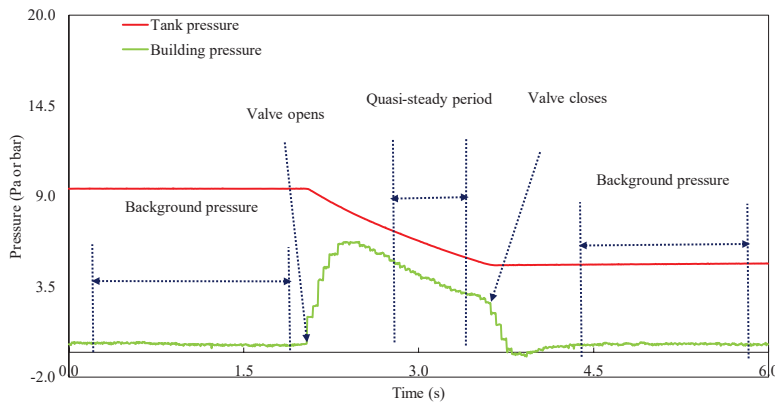


Tank unit

- Compressor motor
- Pressure transducer
- Air tank (10 bar)
- Solenoid valve



Control unit



Steady $Q_{device} = Q_{building}$



Pulse $Q_{compressibility}$



$Q_{device} = Q_{building} + Q_{compressibility}$

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	Year	Funding source	Nottingham academics/researchers	External involvement	
Piston Technique	2001	Internal funding	Etheridge & Carey	-	
	2002-2006	EPSRC Industrial Case Award (PhD)	Etheridge & Cooper	Airflow Developments Ltd	
	2010	Swedish Energy Agency	Cooper & Etheridge	University of Gavle, Sweden	
	2010-2011	EPSRC First Grant	Cooper & Zu	Letters of support from industry	
	2013-2015	Innovate UK (TSB) consortium - Scaling Up Retrofit	Cooper, Riffat, Gillott & Zheng	NEF, Elmhurst, EPS Ltd, Air & Gas Ltd, & ANDtr	
Nozzle Technique	2014-2018	EU Funding (Horizon 2020) – Commissioning of buildings	Cooper, Wood & Zheng	'Built2Spec' consortium of 20 partners in 8 EU countries	
	2017-2018	Innovate UK: High Integrity enclosure testing	Wood & Zheng	Build Test Solutions Ltd (lead)	
	2017-2019	BEIS -PULSE: Rapid Verification and Validation of Air Infiltration in New and Existing Buildings	Wood, Gillott, Zheng & Pasos	Build Test Solutions Ltd (lead)	
	2019-2021	Internal funding: wind impact, Passivhaus and non-residential buildings	Wood, Zheng & Hsu	Build Test Solutions Ltd	

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University of Nottingham UK CHINA MALAYSIA		Background of this study		43 rd AIVC	
 Ministry of Housing, Communities & Local Government	The Future Homes Standard 2019 Consultation on changes to Part L (conservation of fuel and power) and Part F (ventilation) of the Building Regulations for new dwellings				
	Introducing an alternative to the blower door test 5.11. Currently, airtightness is commonly tested using the blower door method. To provide an alternative method of airtightness testing, we are seeking views on introducing the Pulse test as an approved airtightness testing methodology. The Pulse test dynamically measures building air leakage directly at low pressure. 5.12. The Pulse test is performed at a pressure differential of 4Pa as opposed to 50Pa, which is more representative of conditions that properties are likely to experience. 5.13. A constant conversion factor has been identified to convert measurements performed at a pressure differential of 4Pa to 50Pa in SAP. 5.14. The effectiveness of the test in very airtight dwellings has yet to be demonstrated. Therefore, we want to seek views on introducing the Pulse test as an approved method of airtightness testing for new dwellings with a designed airtightness of between 1.5 m ³ /m ² .h and the maximum allowable airtightness value in Approved Document volume 1. This is the range that a 58.5l Pulse unit has been demonstrated to perform at in the field trial, which can be accessed via the link below.				

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Limited to: $>1.5 \text{ m}^3/\text{h}/\text{m}^2$ @50 Pa

11 properties have been Pulse tested between October 2019 and January 2020

@50 Pa

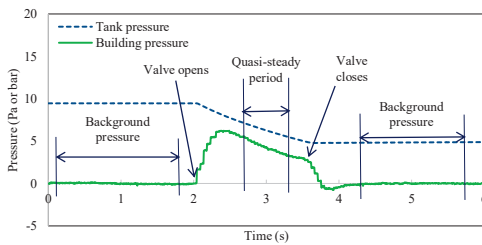
0.29 $\text{m}^3/\text{h}/\text{m}^2$ to 1.19 $\text{m}^3/\text{h}/\text{m}^2$

0.48 ACH to 1.27 ACH

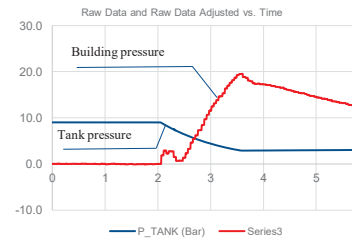
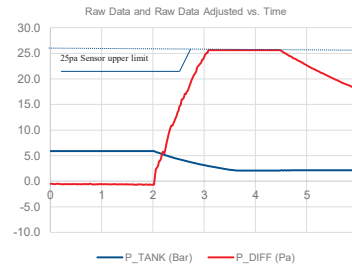
@4 Pa

0.05 $\text{m}^3/\text{h}/\text{m}^2$ to 0.31 $\text{m}^3/\text{h}/\text{m}^2$

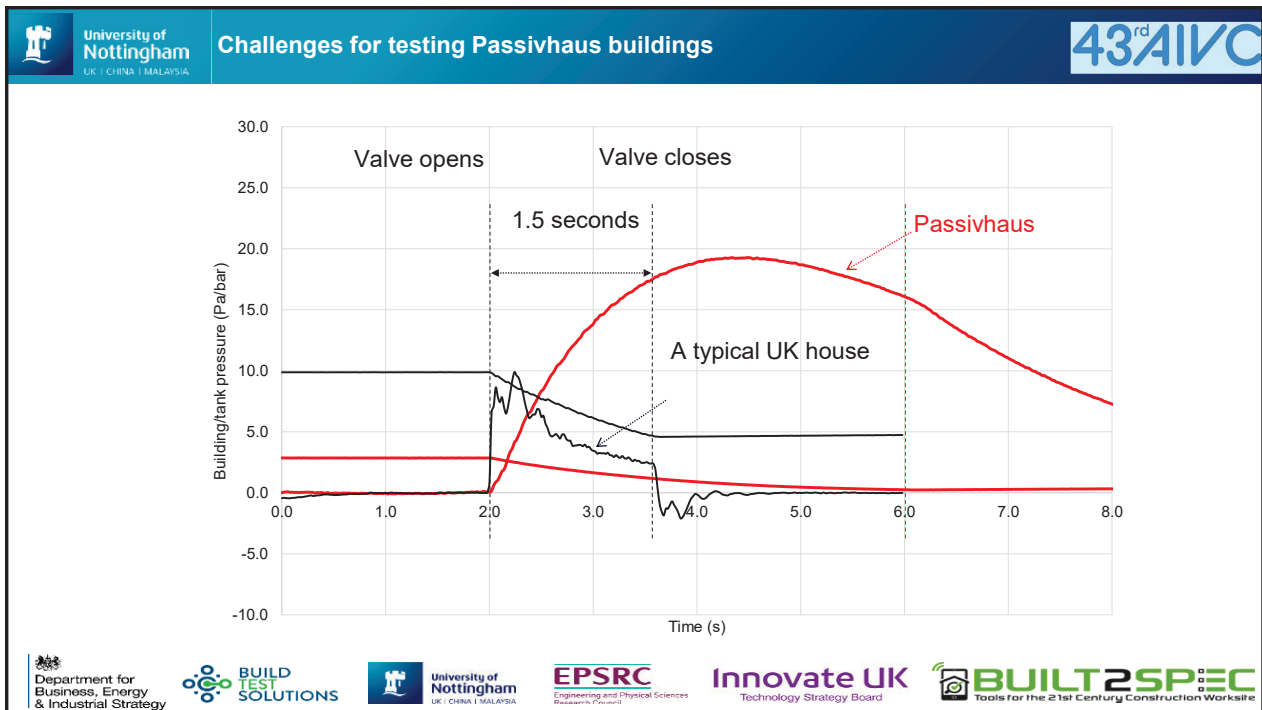
0.07 ACH to 0.33 ACH



Good Pulse shape



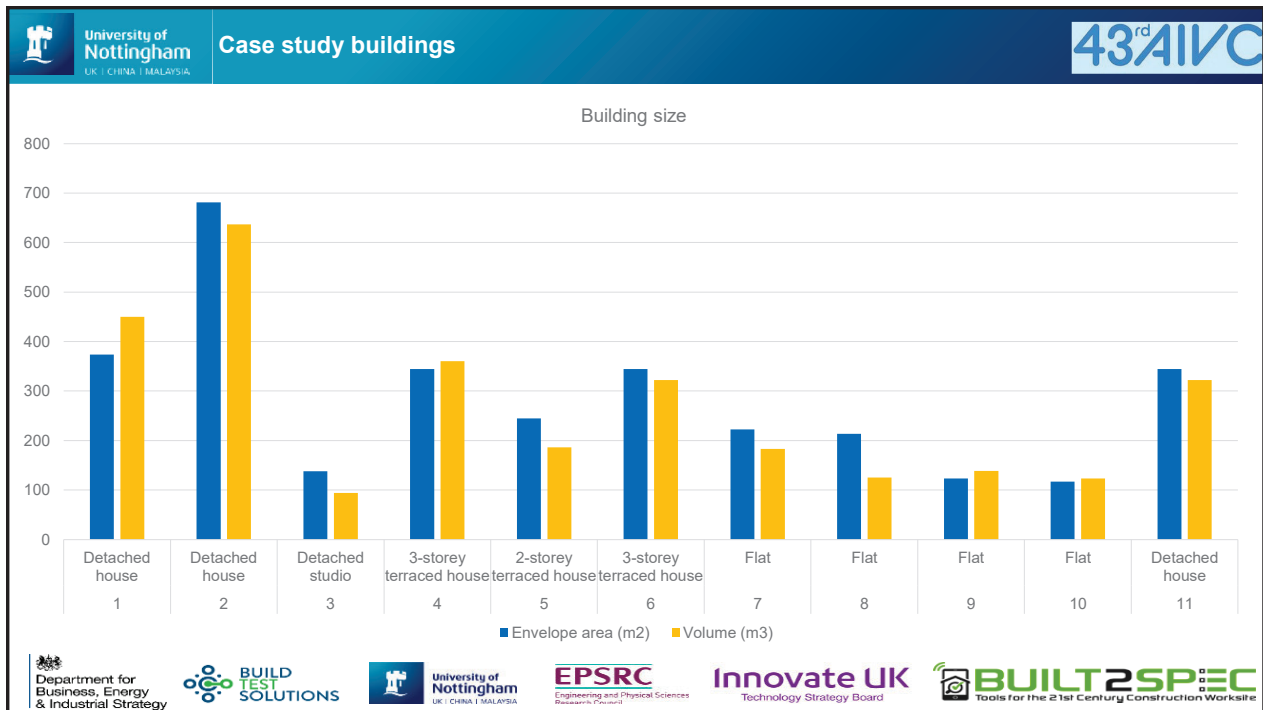
Incomplete or unusable Pulse shapes



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Challenges in minimising the difference in building preparation

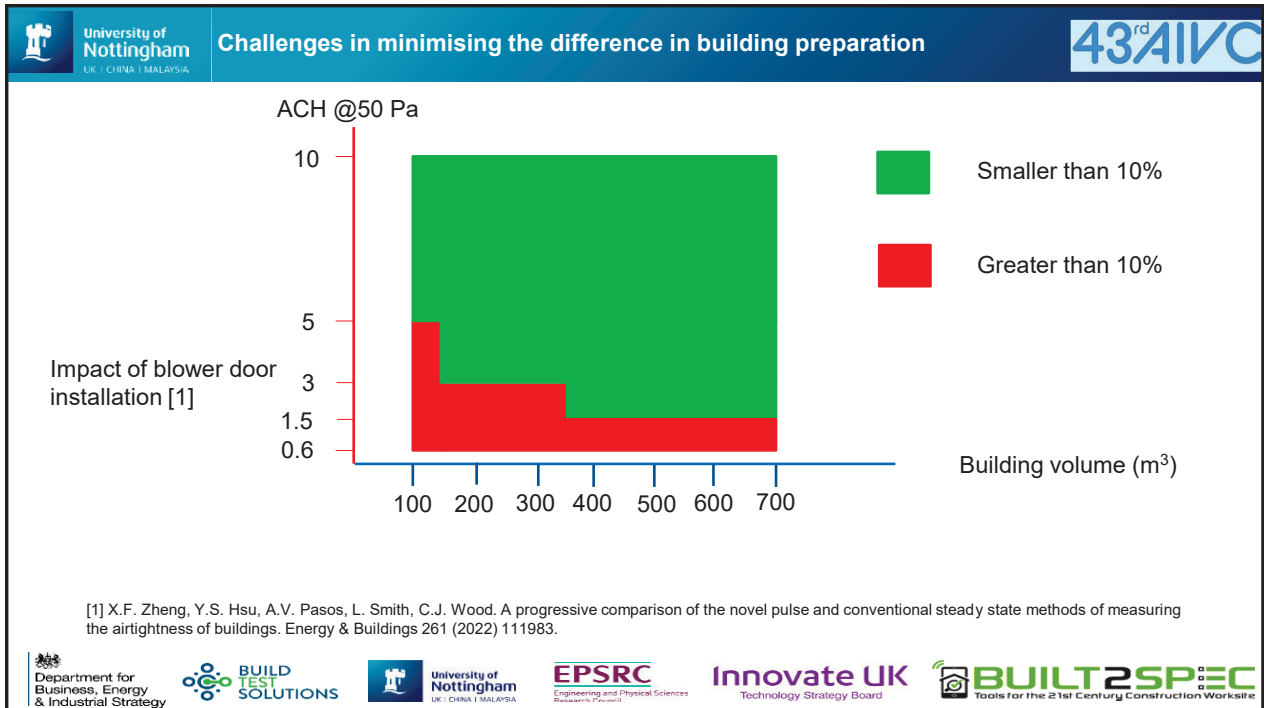
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The installation of both methods can lead to a difference in the measured building envelope due to the different way of installation:

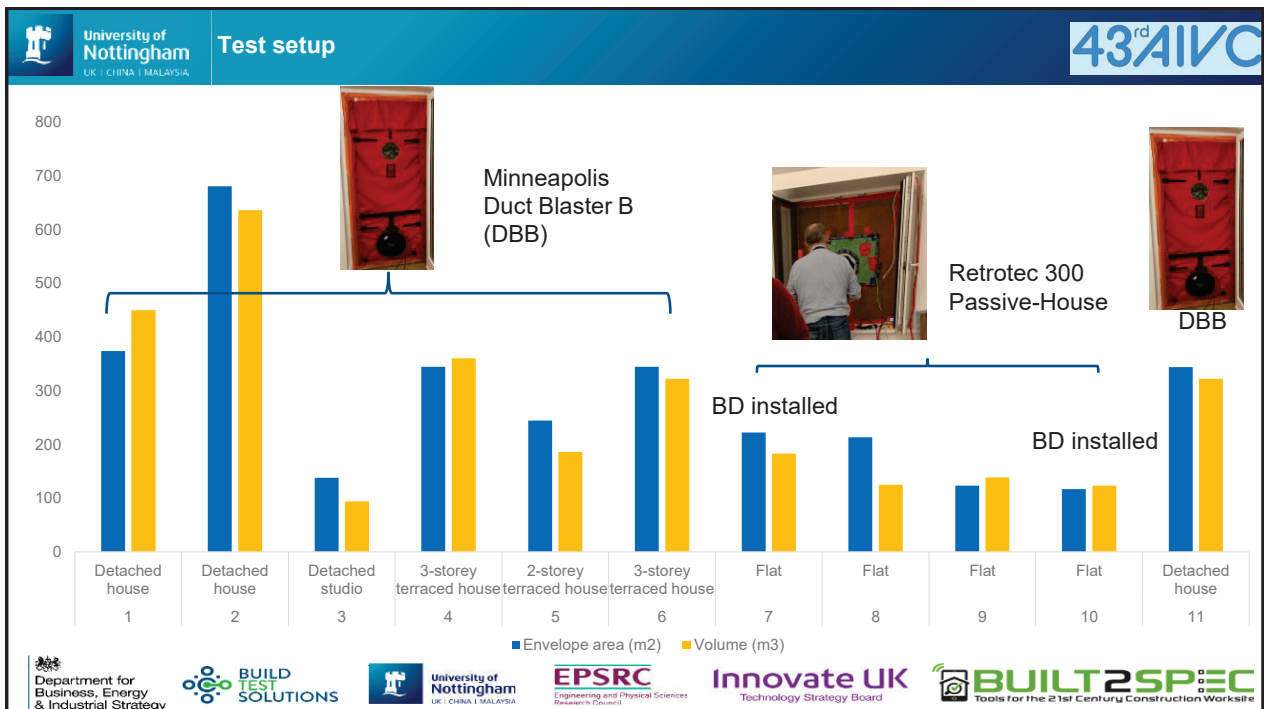
- Difference in the fenestration opening
- This difference is more obvious when the opening is in an irregular shape, the frame is angled, or the door has lots of fittings in the frame.

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Building preparation

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Setup and testing

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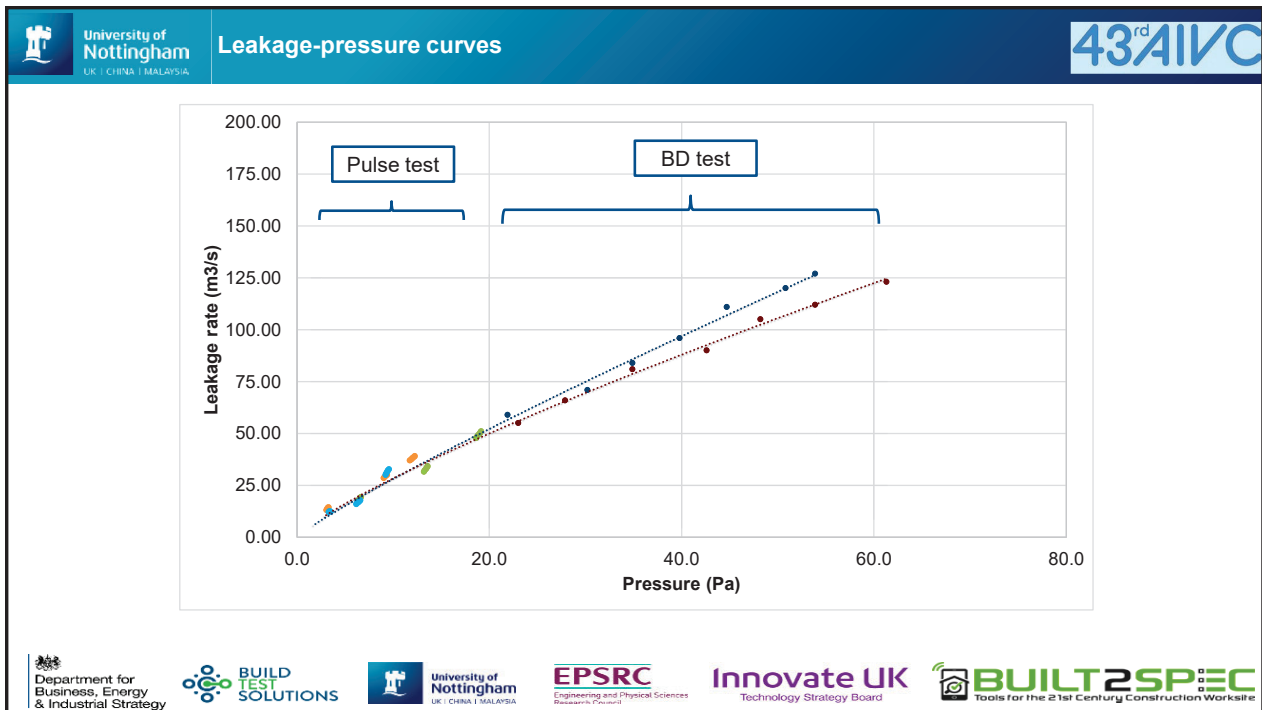
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Test results at 4 Pa

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Property ID	N4 (BDT)	N4 (Pulse)	N4 Difference	N4 Percentage Difference
001	0.11	0.09	0.024	27%
002	0.14	0.13	0.004	3%
003	0.11	0.10	0.01	10%
004	0.13	0.20	-0.066	33%
005	0.05	0.06	0.004	7%
006	0.12	0.14	-0.014	10%
007	0.08	0.09	-0.008	9%
008	0.04	0.05	0.004	7%
009	0.11	0.11	0.002	2%
010	0.11	0.11	0.000	0%
011	0.36	0.31	0.053	17%

■ DBB by Zheng

■ 300 PH by PJ

Department for Business, Energy & Industrial Strategy

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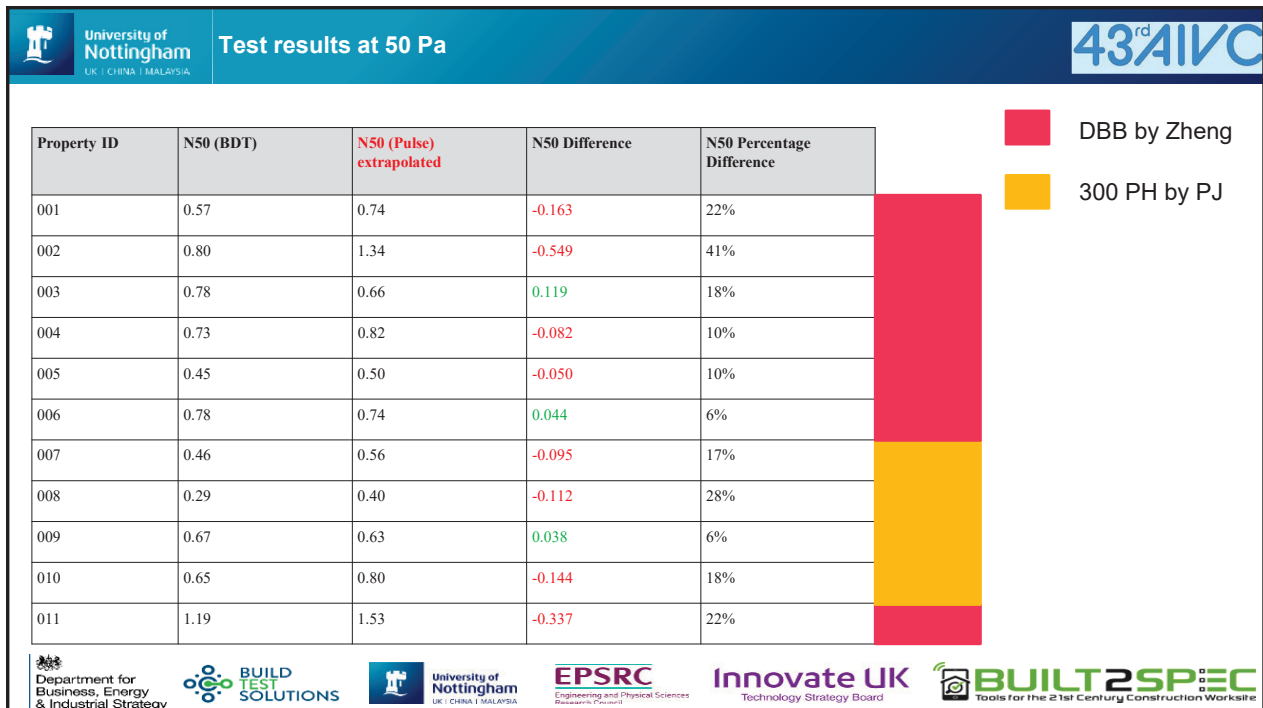
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University of Nottingham **Findings** 43rd AIVC

- The Pulse technique can measure the airtightness of Passivhaus standard building, with an extended valve opening time.
- It can be challenging to achieve a like-for-like comparison between two methods due to the different way of testing.
- With the difference in the building envelope condition minimised, they provide comparable test results.

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Many thanks!

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Correlation analysis between ACH50 and Air permeability considering the floor area of a residential buildings

2023. 10. 04

INHA UNIVERSITY

Choi, Su-Ji

Jo, Jae-Hun

(Email: jhjo@inha.ac.kr)

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CONTENTS

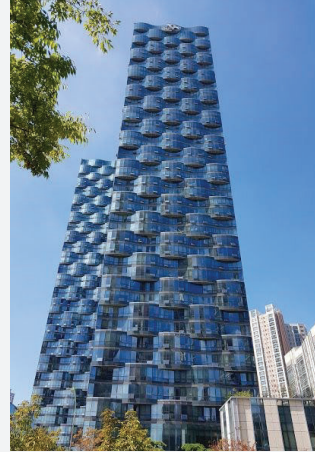
- 1. Introduction**
- 2. Airtightness Expression**
- 3. Field Measurement of Airtightness in Residential Buildings**
- 4. Correlation Analysis between ACH50 and Air Permeability**
- 5. Conclusions**

1

1. Introduction

• 2

- **Typical residential building types in Korea**
 - Residential building types in large Asian cities → Apartment building
 - More than 500 units will be built, consisting of 3 to 7 unit types



<https://www.google.com/search?q=apartment+building+in+asia>

2

1. Introduction

• 3

- **The floor plan of residential building**
 - The floor area is different, but the internal configuration is similar
 - Small type(66m² ~ 95.7m²) / Medium type(99m²~128.7m²) / Large type(132m²~161.7m²)



3

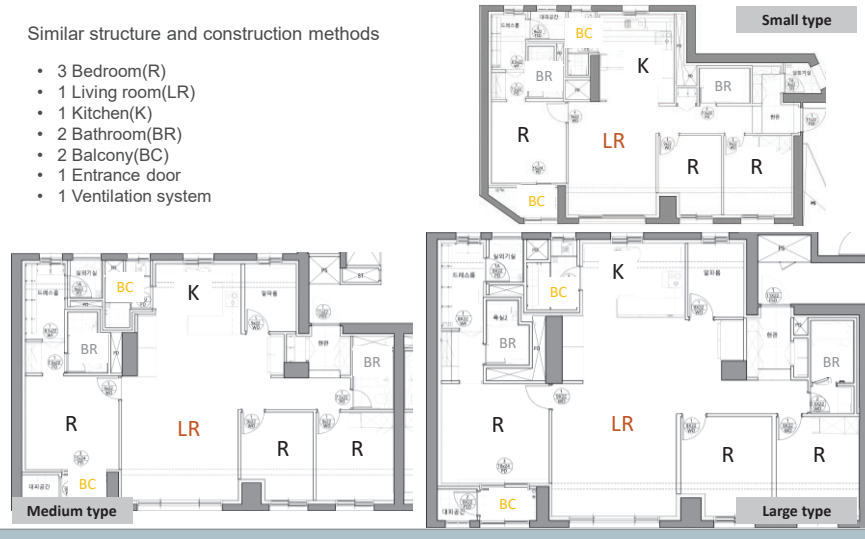
1. Introduction

• 4

▪ The floor plan of residential building

Similar structure and construction methods

- 3 Bedroom(R)
- 1 Living room(LR)
- 1 Kitchen(K)
- 2 Bathroom(BR)
- 2 Balcony(BC)
- 1 Entrance door
- 1 Ventilation system



4

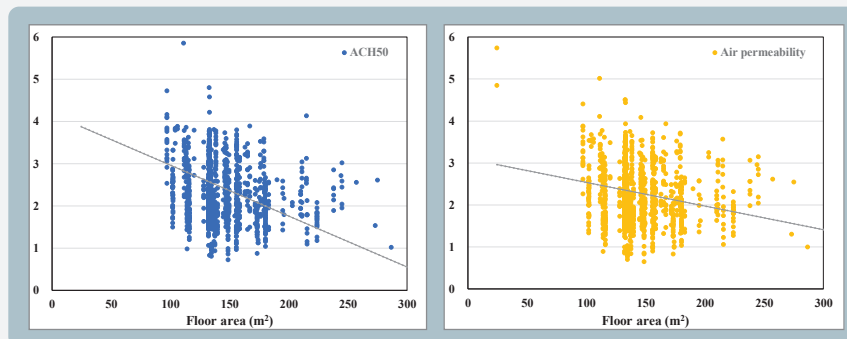
1. Introduction

• 5

▪ ACH50 & air permeability according to the floor area

- The same airtightness techniques and construction details were applied
- Large floor area → ACH50 is tighter than air permeability
- Small floor area → ACH50 is looser than air permeability

→ The potential air leakage sources are similar, but there are many difference in values



5

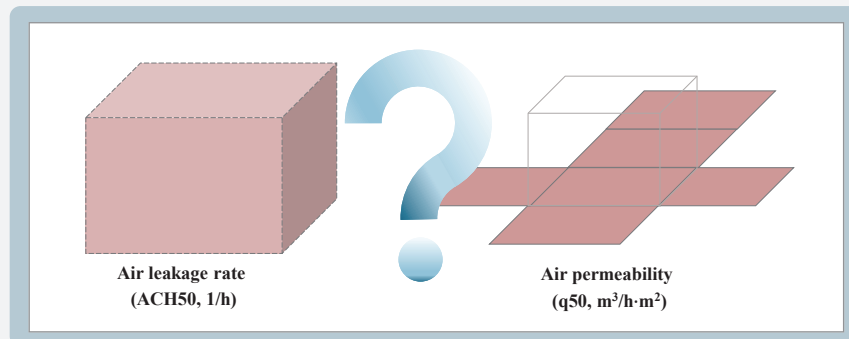
1. Introduction

• 6

▪ Purpose of this study

To properly evaluate the level of airtightness, it is necessary to understand the characteristics of ACH50 and air permeability

- Analyze each characteristic by comparing ACH50 and air permeability
- Analyze the conversion ratio between ACH50 and air permeability



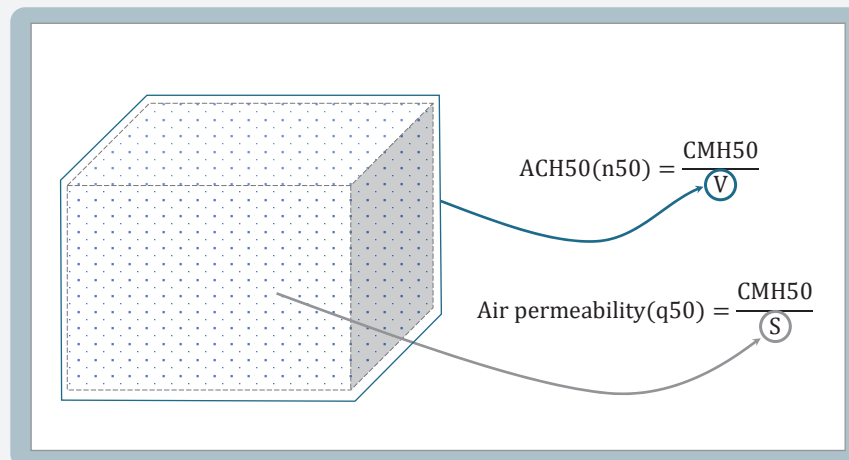
6

2. Airtightness Expression

• 7

▪ ACH50 & Air permeability

- ACH50 : how many times the air in a room is leak out when a pressure of 50 Pa is applied to the building envelope
- Air permeability: The amount of leakage through the building envelope

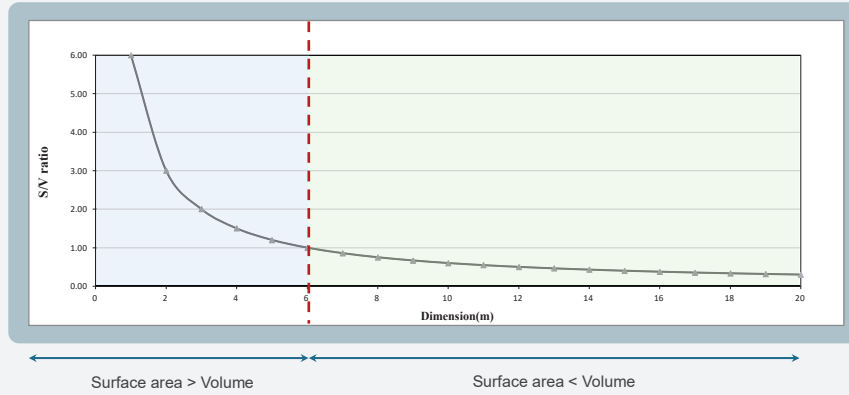


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2. Airtightness Expression

▪ ACH50 & Air permeability

- Growth rate: Volume > Surface area → Differences occur depending on the size of the building
- S/V ratio change
 - Dimension < 6 → Air permeability appears to be tighter than ACH50
 - Dimension > 6 → ACH50 appears to be tighter than air permeability



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3. Field Measurement of Airtightness in Residential Buildings

▪ Field measurement summary

Classification	Small type	Medium type	Large type
Building Use	Multi-unit dwelling		
Construction year	2017		
Construction type	Reinforced concrete structure, Flat slab		
Number of Unit	24	24	24
Floor area(m ²)	101.76~102.21	134.56~137.56	173.18

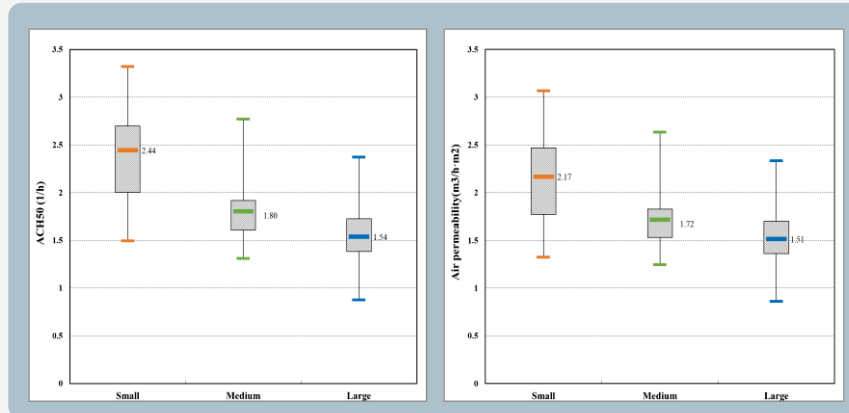
- ✓ Classification by floor area
 - Small type
 - Medium type
 - Large type
- ✓ Blower door test
 - ISO 9972
- ✓ Random sampling
 - 24 units were selected(total 274 units)



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3. Field Measurement of Airtightness in Residential Buildings

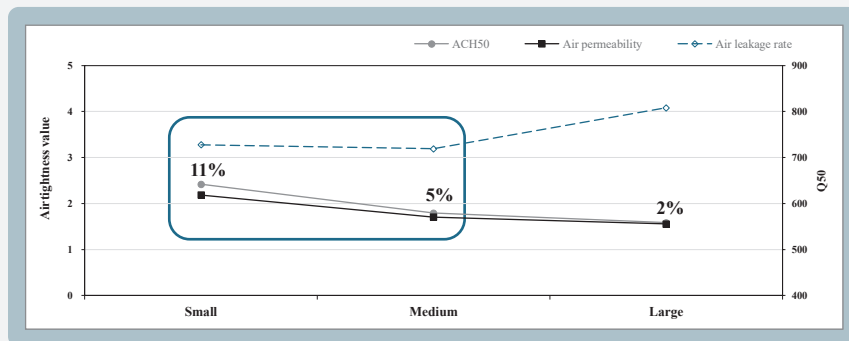
- **Field measurement results**
 - Air permeability is lower than ACH50
 - Airtightness decreases as floor area increases
 - The change rate of ACH50 is larger than air permeability



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3. Field Measurement of Airtightness in Residential Buildings

- **Airtightness according to the floor area**
 - Comparing the change rate → The difference is especially noticeable for small type
 - Q is similar, but the difference is larger due to the difference in area and volume
 - The rate of change in ACH50 is greater than air permeability
 - Air permeability is more suitable for airtightness evaluate



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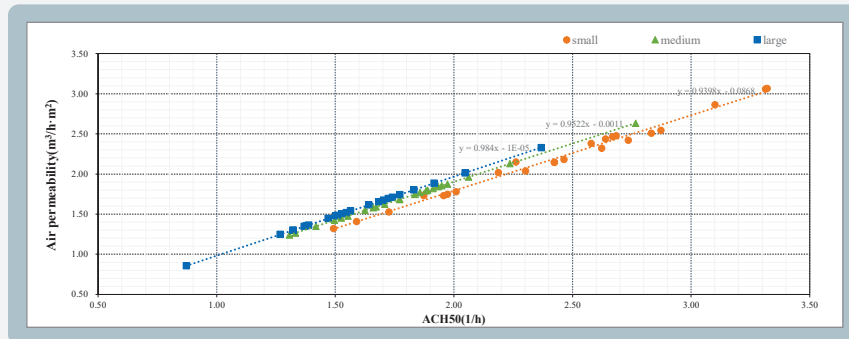
4. Correlation Analysis Between ACH50 and Air Permeability

▪ **Correlation of ACH50 and Air permeability**

- ACH50 and air permeability are linearly related

: Correlation coefficients is close to 1

- Indicates a strong positive correlation
- More difference in small type



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4. Correlation Analysis Between ACH50 and Air Permeability

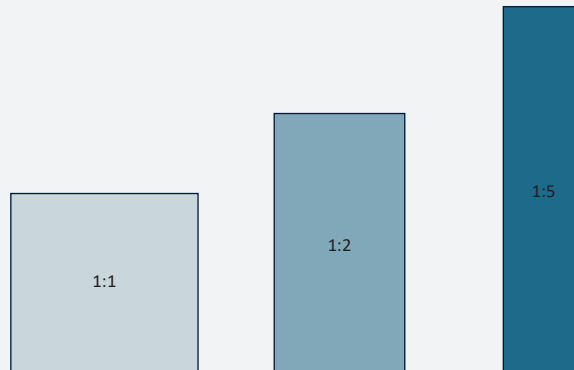
▪ **Analysis the s/v ratio by floor area considering aspect ratio**

- Model summary

: Constant height

: Change only the aspect ratio with the same area

Classification	
Building Use	Multi-unit dwelling
Building height	3
Floor area(m²)	50 ~ 1000
Floor type	1:1, 1:2, 1:5



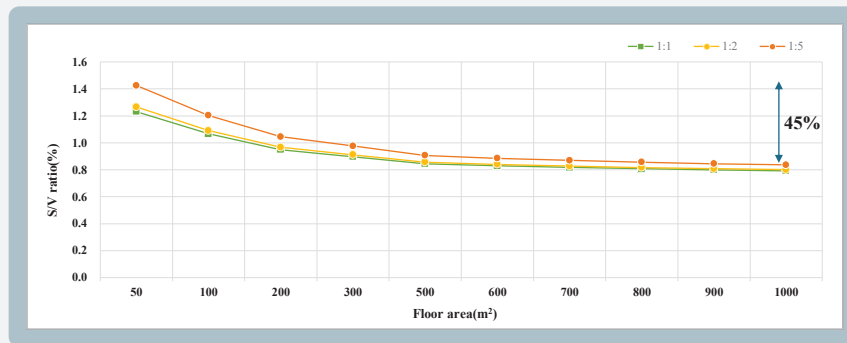
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4. Correlation Analysis Between ACH50 and Air Permeability

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▪ Analysis the s/v ratio by floor area considering aspect ratio

- The s/v ratio ranged
: 50m² : 1.23 ~ 1.43, 1000m² : 0.79 ~ 0.84, 100m² ~ 200m²: 1
- The smaller the difference in aspect ratio, the lower the s/v ratio
- The lower the floor area, the more it is affected by the aspect ratio



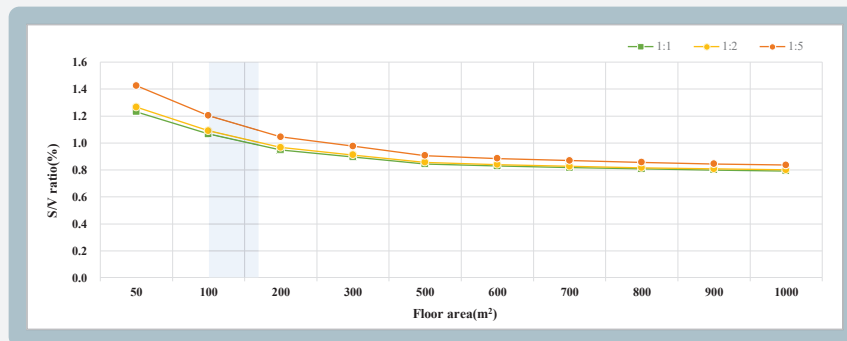
14

4. Correlation Analysis Between ACH50 and Air Permeability

• 15

▪ Analysis the s/v ratio by floor area considering aspect ratio

- In the measured data, the s/v ratio corresponds to 1
 - The data range of floor are is from 101.76 to 173.18
- The residential building does not have a wide range of floor area in Korea
 - The conversion ratio between ACH50 and air permeability is close to 1
- The floor area be <100 m² or >200 m²
 - It is necessary to evaluate the airtightness results considering the s/v ratio by floor area



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5. Conclusions

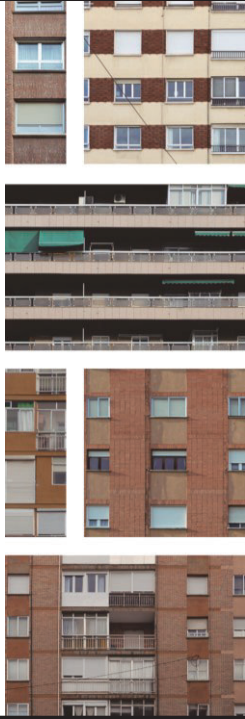
• 16

In this study, the characteristics of ACH50 and air permeability were analyzed to compare the airtightness of buildings of various sizes.

- 1.** The change rate of ACH50 is greater than air permeability and air permeability is more suitable for airtightness evaluation due to its stability
- 2.** The s/v ratio tended to decrease as the floor area increased, the difference in aspect ratio became smaller
- 3.** In Korea, the s/v ratio is close to 1. However, since the s/v ratio varies depending on the floor area, it is necessary to convert the airtightness expression by considering the floor area

→ Although the s/v ratio of Korean residential buildings is close to 1, the same standard should not be applied because of the difference in change

→ Airtightness is related to the flow rate through the envelope, additional consideration of component affecting the air flow rate is required for airtightness evaluation



Airtightness predictive model from measured data of residential buildings in Spain

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Pilar Rodríguez-del-Tío, Dpto. Estadística e Investigación Operativa
Miguel Fernández-Temprano, Dpto. Estadística e Investigación Operativa
Miguel Ángel Padilla-Marcos, GIR Arquitectura & Energía
Alberto Meiss, GIR Arquitectura & Energía

Universidad de Valladolid

43rd AIVC - 11th TightVent & 9th venticool Conference: *Ventilation, IEQ and health in sustainable buildings*
October 4-5, 2023 Aalborg University, Copenhagen, Denmark



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Building and Environment 223 (2022) 109435

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An envelope airtightness predictive model for residential buildings in Spain

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ARTICLE INFO

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Airtightness
Blowerdoor
Dwellings
Database
Statistical analysis

ABSTRACT

The need for airtightness control is a reality given its impact on buildings' energy use and Indoor Air Quality (IAQ). For the past few years, this fact has resulted in energy performance regulations establishment that involves airtightness requirements in many countries in Europe and North America. In this sense, efforts should not only be focused on new buildings, but also existing ones. Considering that around 90% of the built stock in the EU is expected to still be standing in 2050 and that almost 75% of the buildings are energy inefficient, attention must be paid to retrofitting actions.

Airtightness predictive models have become useful in the decision-making process and to estimate input values in energy performance simulation tools. So far, several predictive models have been developed in different countries. However, specific construction systems and practices lead to a lack of consensus regarding the impact of different factors on airtightness performance. Therefore, the applicability of existing models is limited to their specific contents.

This paper presents a predictive model for envelope airtightness, which was developed from a database that contains a fully characterised representative sample of the residential building stock in Spain. A General Linear Model (GLM) was considered to assess significant variables related to the age of the building, typology, building state, construction system, and dimensions. As a result, a predictive model is presented and validated. Overall, even if some limitations were identified, the relevance of the model proposed is warranted from the statistical point of view. The airtightness predictive model presented offers a procedure for airtightness estimation of residential buildings in Spain.

Poza-Casado, Irene, Pilar Rodríguez-del-Tío, Miguel Fernández-Temprano, Miguel-Ángel Padilla-Marcos, and Alberto Meiss. 2022. "An Envelope Airtightness Predictive Model for Residential Buildings in Spain." *Building and Environment* 223(July):109435. doi: 10.1016/j.buildenv.2022.109435



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Airtightness predictive model from measured data of residential buildings in Spain



2



Structure

- Context
- Motivation
- Sample and airtightness
- Statistical model development
- Results
- An application example
- Conclusions and challenges

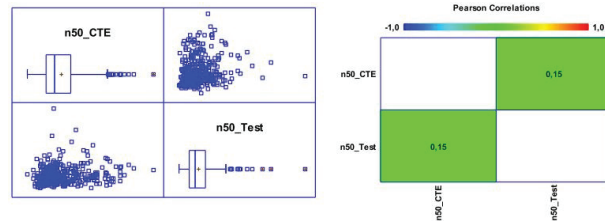


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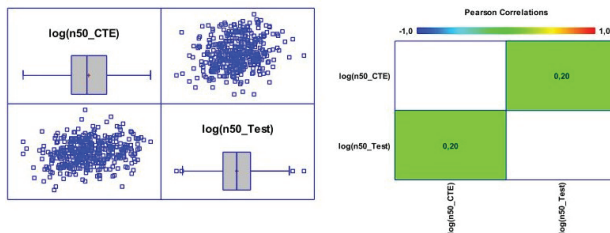


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Context



Correlation analysis between the n50 values obtained from pressurization tests and reference values



Correlation analysis between the logarithms of the n50 values obtained from pressurization tests and reference values

- Inaccurate reference values to prove compliance with building regulations



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Airtightness predictive model from measured data of residential buildings in Spain



4

Motivation

Importance of predictive models:

- Estimate the airtightness from building characteristics
- Evaluate the impact of potential improvements
- Controlling costs and time in the decision-making process
- Estimate input values in EP simulation tools
- Understand the factors that most impact airtightness

airtightness estimation cannot substitute on-site testing



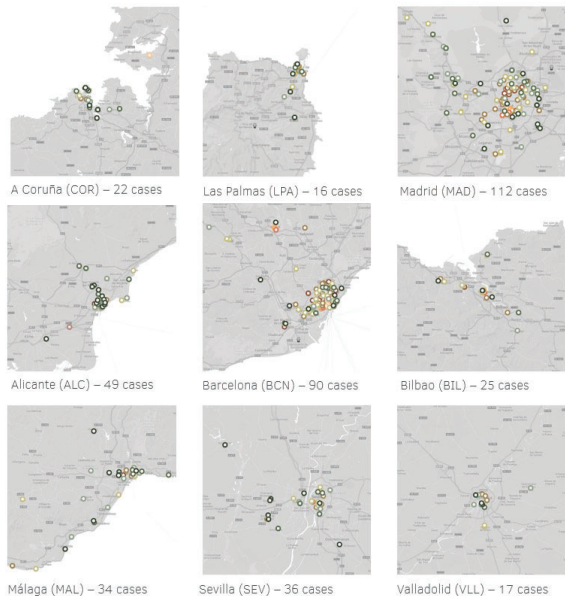
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5

Sample and airtightness

- Dataset: 400 dwellings
 - Representative of the existing built stock
 - Typology, age, climate zone
- Airtightness: Blower Door tests
- Characterization
 - Identification information
 - Configuration
 - Construction of the envelope
 - Building systems



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Airtightness predictive model from measured data of residential buildings in Spain

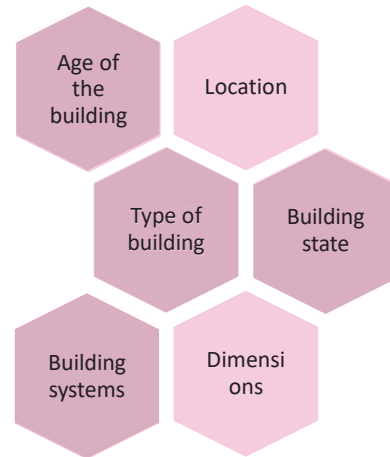


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Statistical model development

- General Linear Model (GLM)
- Qualitative and quantitative variables
- Iterative procedure
- Main effects and interactions

$$Y = \beta_0 + \sum_{i=1}^p \beta_i X_i + \sum_{i < j} \tau_{ij}(X_i X_j) + \varepsilon$$



7

Final variables

Climate zone	Period of construction	Typology	Number of bathrooms	Retrofitting state	Type of ceiling
<ul style="list-style-type: none"> • Winter severity • Summer severity 	<ul style="list-style-type: none"> • Before 1980 • After 1980 	<ul style="list-style-type: none"> • Single family • Multi family 	<ul style="list-style-type: none"> • 0...5 	<ul style="list-style-type: none"> • Original • Retrofitted 	<ul style="list-style-type: none"> • No false ceiling • False ceiling only wet romos • False ceiling
Window permeability	Window material	Shutter position	Heating system	Share of Windows	Share of opaque envelope
<ul style="list-style-type: none"> • Class (UNE-EN 12207) • 0-4 	<ul style="list-style-type: none"> • Aluminium • PVC • Wood • Steel • Wood • Steel 	<ul style="list-style-type: none"> • Non integrated • Internal • External • No shutter 	<ul style="list-style-type: none"> • No heating • Heating units • Underfloor • Ducts 	<ul style="list-style-type: none"> • Quantitative 	<ul style="list-style-type: none"> • Quantitative



8

Results

- 12 main effects and 2 interactions
- Significant variables in line with previously developed models
- Included singularities of the Spanish national built stock
- The model can explain 42.9% of the variability of the response
- Robust model in spite of limitations
 - Influence of supervision and workmanship
 - Size and representativeness of the sample

Table 5: Equation of the final GLM predictive model for airtightness.

Parameter	Coefficient	Parameter	Coefficient
Intercept	0.273	Shutter position. P04	0a
Retrofitting state. Original	0.137**	False ceiling. FC0	-0.313***
Retrofitting state. Retrofitted	0a	False ceiling. FC1	-0.264***
Climate zone. A3	0.346**	False ceiling. FC2	0a
Climate zone. B4	0.545***	Typology. Multifamily	0.412**
Climate zone. C1	0.273	Typology. Single-family	0a
Climate zone. C2	0.630***	Heating system. No heating	0.074
Climate zone. C3	0.053	Heating system. Underfloor heating	-0.041
Climate zone. D2	0.575***	Heating system. Ducts	0.261***
Climate zone. a3	0a	Heating system. Other systems	0.173
Period of construction. Before 1980	-0.329***	Heating system. Heating units	0a
Period of construction. Since 1980	0a	Number of bathrooms. 0	0.610**
Window permeability. Class 0 or 1	0.596***	Number of bathrooms. 1	0.347***
Window permeability. Class 2	0.322***	Number of bathrooms. 2	0.183
Window permeability. Class 3	0.255***	Number of bathrooms. 3	0.090
Window permeability. Class 4	0a	Number of bathrooms. 4 or 5	0a
Window material. Steel	0.071	Share of windows	0.045***
Window material. Aluminium	0.074	Share of opaque envelope	0.003
Window material. Wood	0.298***	Period of construction. Before 1980 * Share of opaque envelope	0.010***
Window material. PVC	0a	Period of construction. After 1980 * Share of opaque envelope	0a
Shutter position. P01	0.195*	Typology. Multifamily * Share of opaque envelope	-0.009**
Shutter position. P02	0.144**	Typology. Single-family * Share of opaque envelope	0a
Shutter position. P03	-0.123		

a. This parameter is set to 0 as it corresponds to the reference class of the variable.
* stands for p-value ≤ 0.1, ** for p-value ≤ 0.05 and *** for p-value ≤ 0.01



An application example

DETERMINACIÓN DE LA HERMETICIDAD EN VIVIENDAS

Edificación cerrada: Planta 9, 96.59 m², B+10, 1966, Text: 24.2 °C / Text: 24.1 °C, Viento: 5 km/h / Racha: 20 km/h (1960 - 1979)

Vivienda expuesta al viento: A CORUÑA, Zona A (Clima Atlántico) - C1 [RE1-2013]

Inmueble: Original vs. Reforma reciente de baños y cocina. Galerías independientes de estancias. Fisuras u otras patologías detectadas en cerramientos.

Cerramiento exterior: F.08 (Exterior), F.08 (REC | RED), RM (C | LH | VL | Interior), P.03 (Persianas)

Particiones interiores	Tabiques de ladrillo	Falso techo	En asco.	Amueblamiento	Vivienda completamente amueblada.
Calefacción	Si: Equipo autónomo (unizona/eléctrico/etc.). Distribución vista con radiadores o similar.		Refrigeración	No	
Ventilación	Natural	Cocina con ventana. Campana extractora a conducto vertical. Baño 1 sin ventilación. Baño 2 con ventana.			
Superficie envolvente	297.92 m ²	Volumen de aire	246.62 m ³	Factor de forma	1.21 m ⁻¹
AExt = 30.93 m ² 10.4%	AEviv = 21.08 m ² 7.1%	AEmed = 34.98 m ² 11.7%	AEcom = 22.43 m ² 7.5%	AF = 189.5 m ² 63.3%	AC = 0 m ² 0.0%
Altura libre media	2.58 m		Altura entre forjados	2.58 m	
Carpinterías	Clase 3 Ac/Ae = 5.9 %		Carpinterías abatibles de aluminio, de hoja simple y persiana integrada.		≤ 9 m ³ /h m ² a 100 Pa
Clase 0	Clase 1	Clase 2	Clase 3	Clase 4	
		Sup = 3.42 m ³ Piez = 12.59 m	Sup = 14.16 m ³ Piez = 45.98 m		

REC: cerramiento exterior con resaca
RED: cerramiento exterior con resaca
RM: cerramiento interior (RM) con un espacio, embalsado y aislado
LH: Malla de ladrillo con resaca o similar
VL: Malla de ladrillo con resaca o similar
C: Malla de ladrillo con resaca o similar
LH: Malla de ladrillo con resaca o similar
VL: Malla de ladrillo con resaca o similar
P: panel abatible con alfiler abatido
Piez: panel abatible con alfiler abatido
AC: Malla de ladrillo con resaca o similar
C: cámara de aire no aislada
CV: cámara de aire ventilada
A1: alfiler abatido
T: tablero a panel interconectado
Y: canal de perfil interconectado
UVA: unidad de vidrio abatido



Climate zone	Period of construction	Typology	Number of bathrooms	Retrofitting state	Type of ceiling
• C1	• Before 1980	• Multi family	• 2	• Original	• No false ceiling
Window permeability	Window material	Shutter position	Heating system	Share of Windows	Share of opaque envelope
• Class 3	• Aluminium	• External	• Heating units	• 5,89%	• 4,48%



Estimated value:

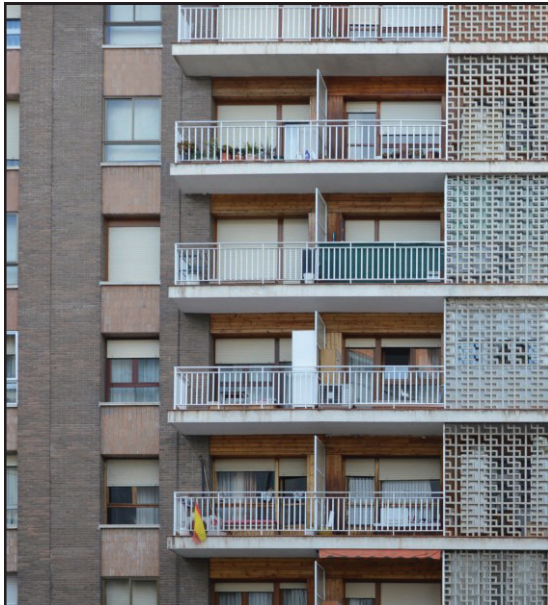
$$n_{50} = 3,081 h^{-1}$$

Parameter	Coefficient
Period of construction. Before 1980 * Share of opaque envelope	0.010***
Period of construction. After 1980 * Share of opaque envelope	0a
Typology. Multifamily * Share of opaque envelope	-0.009**
Typology. Single-family * Share of opaque envelope	0a

$$\log n_{50} = 0,273 + \sum \text{variables} + \left(0,003 \cdot \left(\frac{\sum \text{area}_{\text{opaque env}}}{\sum \text{area}_{\text{total env}}} \cdot 100 \right) \right) + \left(0,045 \cdot \frac{\sum \text{area}_{\text{windows}}}{\sum \text{area}_{\text{total env}}} \cdot 100 \right) + (\sum \text{interactions}) + \varepsilon$$

Parameter	Coefficient	Parameter	Coefficient	Parameter	Coefficient
Intercept	0.273	Period of construction. Before 1980	-0.329***	Shutter position. P04	0a
Retrofitting state. Original	0.137**	Period of construction. Since 1980	0a	False ceiling. FC0	-0.313***
Retrofitting state. Retrofitted	0a	Window permeability. Class 0 or 1	0.596***	False ceiling. FC1	-0.264***
Climate zone. A3	0.346**	Window permeability. Class 2	0.322***	False ceiling. FC2	0a
Climate zone. B4	0.545***	Window permeability. Class 3	0.255***	Typology. Multifamily	0.412**
Climate zone. C1	0.273	Window permeability. Class 4	0a	Typology. Single-family	0a
Climate zone. C2	0.630***	Window material. Steel	0.071	Heating system. No heating	0.074
Climate zone. C3	0.053	Window material. Aluminium	0.074	Heating system. Underfloor heating	-0.041
Climate zone. D2	0.575***	Window material. Wood	0.298***	Heating system. Ducts	0.261***
Climate zone. a3	0a	Window material. PVC	0a	Heating system. Other systems	0.173
Shutter position. P01	0.195*			Heating system. Heating units	0a
Shutter position. P02	0.144**			Number of bathrooms. 0	0.610**
Shutter position. P03	-0.123			Number of bathrooms. 1	0.347***
				Number of bathrooms. 2	0.183
				Number of bathrooms. 3	0.090
				Number of bathrooms. 4 or 5	0a





Conclusions and challenges

- Valuable knowledge
- Added value:
 - Origin of representative data
 - Full characterization of the cases
 - Standardised procedures
 - Both quantitative and qualitative variables and interactions
- Current reference values to prove compliance do not consider important variables



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Airtightness predictive model from measured data of residential buildings in Spain



13

Thank you!

irene.poza@uva.es

*Funding: MOVILIDAD INVESTIGADORES E INVESTIGADORAS UVa-BANCO SANTANDER 2023



43rd AIVC -11th TightVent & 9th venticool Conference: Ventilation, IEQ and health in sustainable buildings
Airtightness predictive model from measured data of residential buildings in Spain



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Bridging The Mechanical / Enclosure Gap AIVC 2023



4-5 October 2023, Conference, Copenhagen- 43rd AIVC conference

1



David de Sola, AIA, LEED BD+C,
Principal, 3iVE



Nathaniel Fanning, CEM
Associate Principal,
Energy and Infrastructure Service Leader,
Fitzmeyer & Tocci

2

LEARNING OBJECTIVES

- Develop an informed understanding of the symbiotic relationship between high-performance exterior enclosures and the corresponding high-performance mechanical systems in a building.
- Enhance understanding of the ways building interventions and metrics can be used to inform mechanical systems.



3

The Issue

Enclosure Design and Evaluation is Designed by Architects and
Constructed mainly of CSI Division 7 and 8 Trades

Mechanical Design is Designed by HVAC Engineers and
constructed by CSI Division 15-23 Trades

4

WE DON'T KNOW EACH OTHER



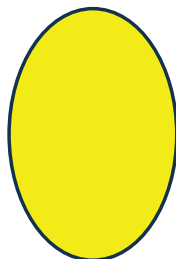
OR EACH OTHER'S WORK

5

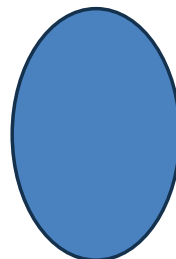
US COMMISSIONING PROCESS

A systematic verification, documentation, applied to all activities during the design, construction, and functional performance testing of the Building Enclosure. It is to confirm that the Owner's Project Requirements (OPR) and the Basis of Design (BoD) are built in accordance with the contract documents.

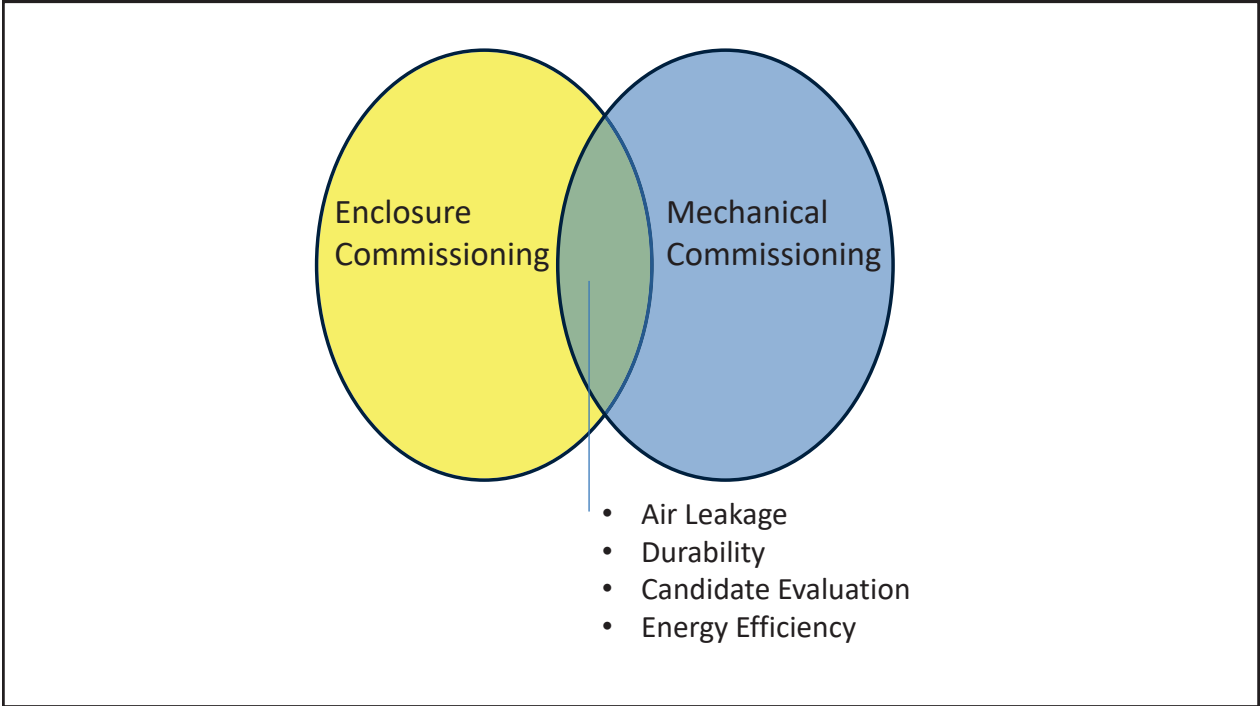
Enclosure
Commissioning



Mechanical
Commissioning



6



7

Why Do Mech Engineers Care about Building Envelope?

Infrastructure Sizing

1. Air Leakage
2. Insulation Values
3. Glazing Values

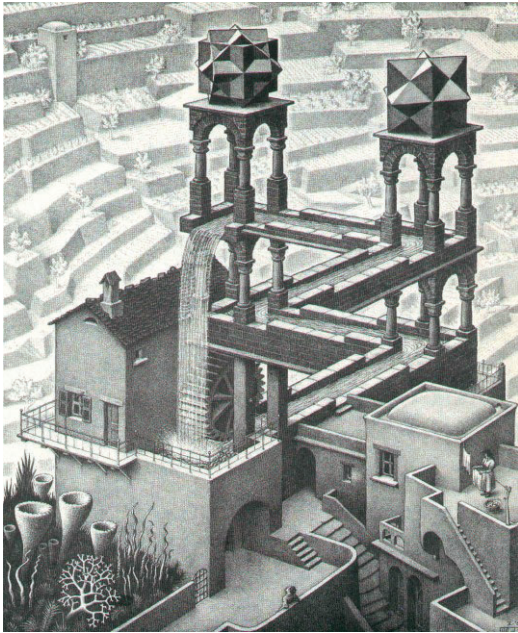
A photograph of a large, light-colored HVAC unit on a rooftop. The unit has several doors and a large vent on the right side. The background shows a cloudy sky and some structural elements of the building.

8

ENCLOSURE THINGS

- Air-Tight Detailing
- Constructability
- Durability

9



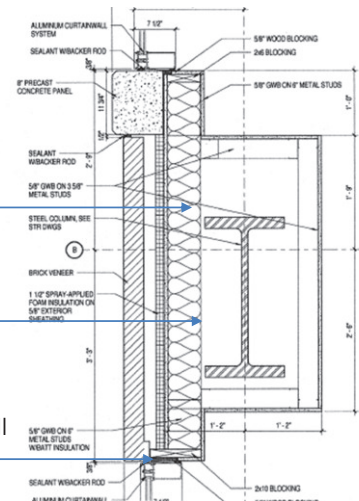
BECx EVALUATION:

Insulation:
Good idea.
BUT...

Is the wall
balanced? Is
there a risk of
condensation?

Can the
insulation be
installed?

Will sealant
work
here? How long will
it last?



10

Mechanical Assessment Findings

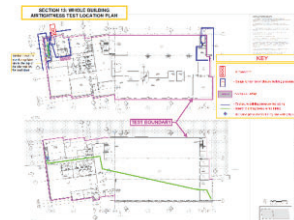
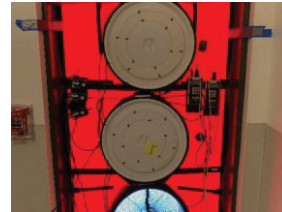
Air Leaks

1. Windows
2. Doors
3. Dampers



11

Whole Building Air Testing



12

How Tight Will the Building Be According to Codes and Sustainability Standards?

Standard	Requirement CFM@75 Pascal/ ft2
2009 IECC International Energy Conservation Code	.55
2012 IECC	.25
R-2000*	.13
LEED IV Multi-Family **	.09
Passivhaus	.05

Table 1: Standards and Requirements for Air Leakage

**Standard for Energy Efficiency in New Construction developed by Natural Resources Canada

**50 Pascal, 2 Point Option, IECC Climate Zone 5-7

•Table References: Genge, C. 2014; USGBC, 2018; USGBC LEED BD+C Homes Air Infiltration 2018

13

How Tight Can We Count On?

Case	Building Type	Measured Air Leakage Rate at 75 PA
1	Elementary School 2018	.06
2	Library 2020	.06
3	University Academic Building 2016	.09
4	Elementary School 2020	1.2*

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Where Do Building Leak?

- Roof/wall intersections
- Bulkhead leakage into attic spaces
- Overhead doors, mainly at base and sides, not between sections
- Masonry Block / floor slab intersections
- Curtain wall / floor slab intersections
- Unsealed walls above ceiling lines
- Doors and windows (both thermally broken and unbroken)
- **Ductwork and pipe penetrations**
- **Underground steam lines**
- **Exhaust and make-up air fans with one-way dampers**

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DURABILITY: WHY WE SHOULD ALL CARE

- Climate change requires it
- Migration of people to less inhabitable locations
- Energy and Environmental impacts of construction



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INSTITUTIONAL DURABILITY

Table 2
Categories of Design Service Life for Buildings
(See Clauses 5.2.3 and 6.2.)

Category	Design service life for building	Examples
Temporary	Up to ten years	<ul style="list-style-type: none">• non-permanent construction buildings, sales offices, bunkhouses• temporary exhibition buildings
Medium life	25 to 49 years	<ul style="list-style-type: none">• most industrial buildings• most parking structures*
Long life	50 to 99 years	<ul style="list-style-type: none">• most residential, commercial, and office buildings• health and educational buildings• parking structures below buildings designed for long life category*
Permanent	Minimum period, 100 years	<ul style="list-style-type: none">• monumental buildings (eg, national museums, art galleries, archives)• heritage buildings

CSA 478 19
Guideline on
Durability in
Buildings

17

THE CASE FOR DURABILITY TO OWNERS

- Exchange of resources for function over time
- Accreditation for Sustainable Program
- Impacts of Economic Investment in Equipment / Years in Service



18

DURABILITY: SOUNDS GOOD. HOW DO I GET IT?

- Start With Robust Materials
- Deploy Them Appropriately
- Understand their interaction with their neighbors
- Understand how details are constructed



19

ENCLOSURE DURABILITY CONSIDERATIONS

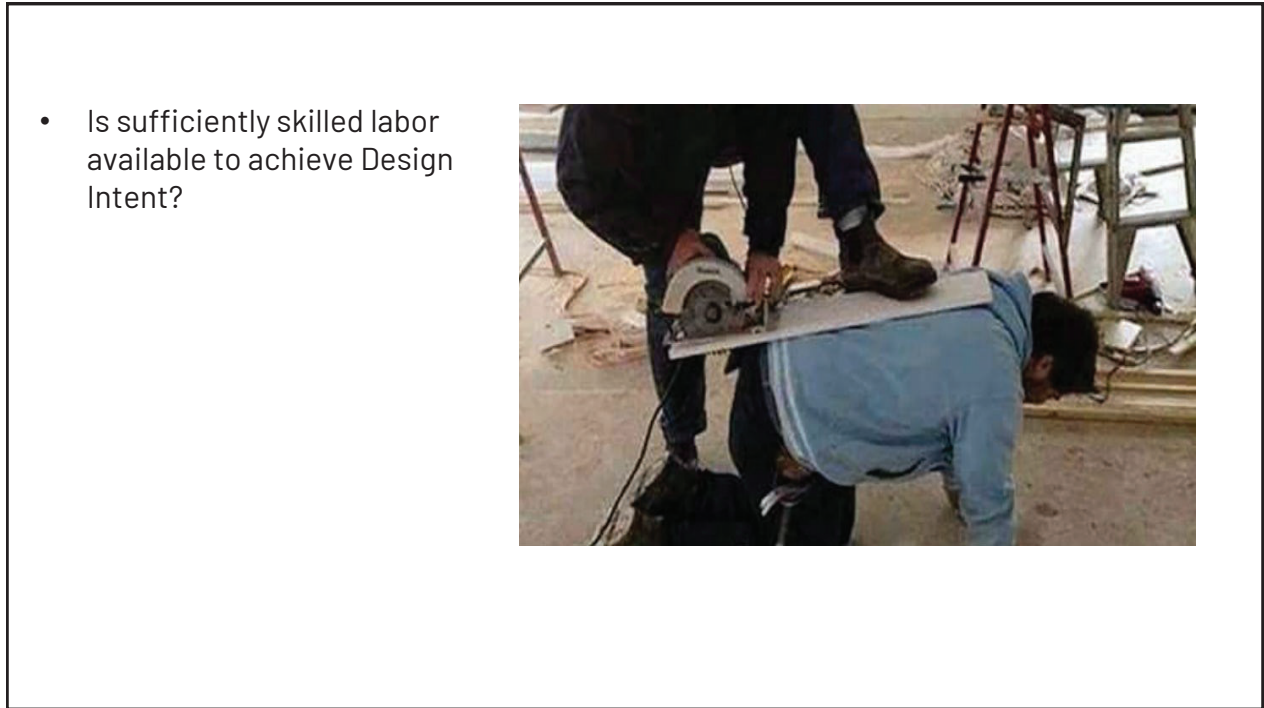
- Does material work for climate?



20



21



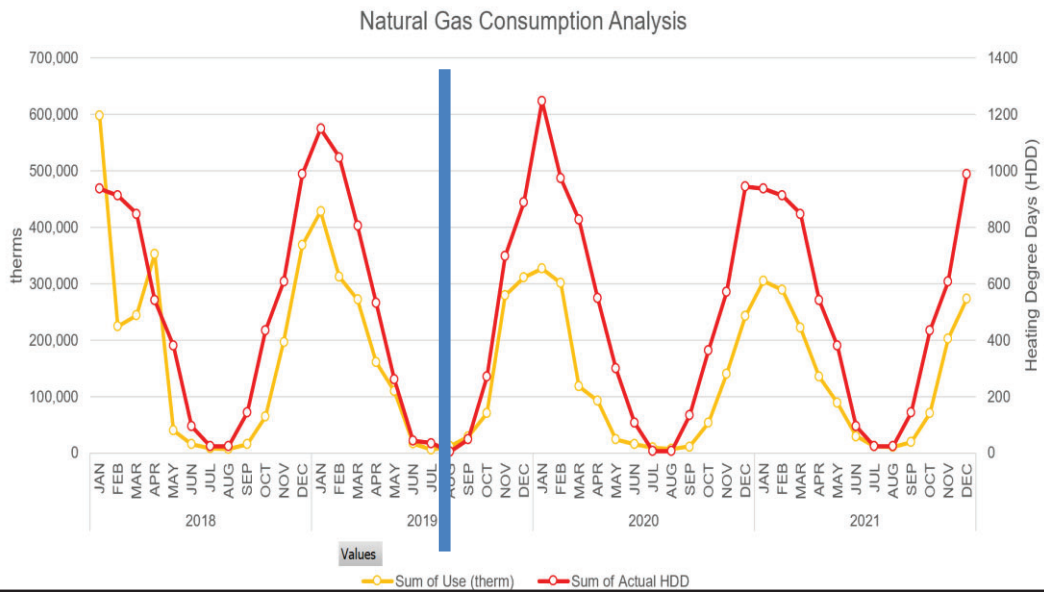
22

- Are the materials sufficiently durable for their placement?



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Mechanical and Building Envelope Upgrade Results



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DISCUSSION



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THANK YOU

3iVE



David de Sola, AIA, LEED BD+C
Principal, 3iVE



Nathaniel Fanning, CEM
Associate Principal,
Energy and Infrastructure
Service Leader,
Fitzmeyer & Tocci

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43rd AIVC
11th TightVent & 9th venticool Conference
Ventilation, EQ and health in sustainable buildings

October 4-5 2023
Copenhagen Denmark

Topical Session 1C: Summer comfort and energy efficiency in hot periods: interest of mixed mode cooling and need of occupant feedback

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Topical Session 1C:

Windows and ceiling fans occupant behaviour model coupling methodology with building energy models

A tropical case study

Maareva PAYET *, LEU Réunion / La Reunion University - PIMENT, France
Maxime BOULINGUEZ**, LEU Réunion / La Réunion University - PIMENT, France

*mp@leureunion.fr
** maxime.boulinguez@univ-reunion.fr

1

DEFINITION OF HUMID TROPICAL CLIMATE

FEATURES and ISSUES

KTC

Ar Aw As BW BS Cs Cw Cf Do Dc E Ft Fi

- Average monthly temperature > 18°C
- Rainfall threshold > 60mm over several months

The surface area of Aw subclass territories has increased since 1965-1994
(Belda et al. 2014)

50% of the population in the intertropical zone forecasted by 2050
(University f James Cook , Australia, 2014)

75% of the population in urban areas
(Rodrigues et al., 2019)

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2

MIXED MODE BUILDINGS

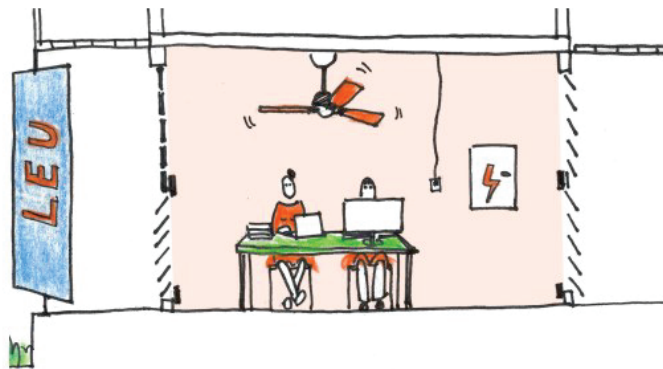


DEFINITIONS

Passive cooling systems -> Openings
Low-energy cooling systems -> Ceiling Fans
High-energy mechanical systems -> HVAC

Zoned Mixte Mode Building (Brager, 2006)

Different controls (Raja, 2014)

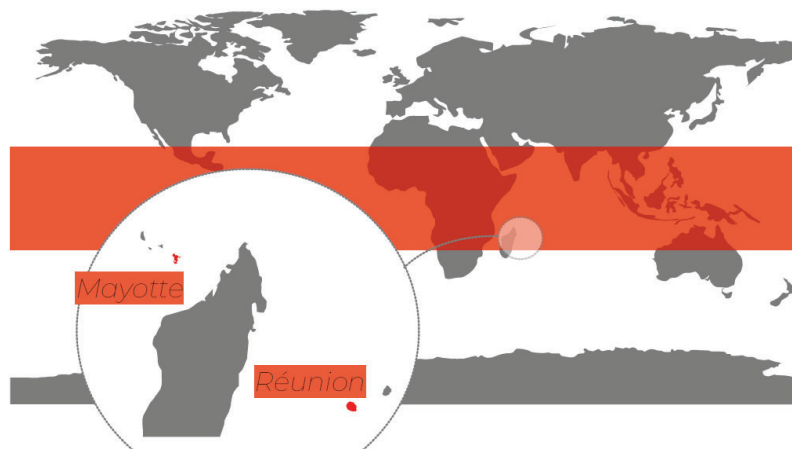


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CASE STUDY IN REUNION ISLAND



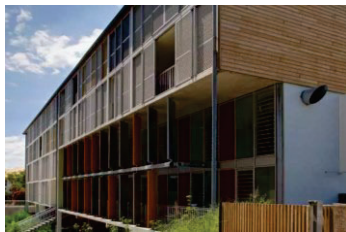
ILET DU CENTRE OFFICE BUILDING



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CASE STUDY IN REUNION ISLAND

ILET DU CENTRE OFFICE BUILDING OVERVIEW



310 m²
Two open plan floors (NV)
Singles offices (NV or AC)
Meeting rooms (AC)
IT room (AC).

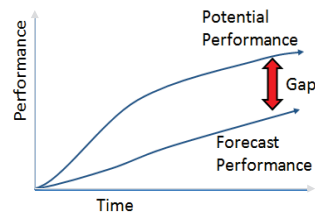


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5

USER BEHAVIOUR IN MIXED MODE BUILDINGS

RESEARCH QUESTIONS



How to assess user behavior in mixed-mode buildings operating with ventilation ?

How to estimate the presence of users on openings and ceiling fans during the design phase ?

- 1- Model behaviours based on measured data
- 2- Integrate these behavioural models into a building model



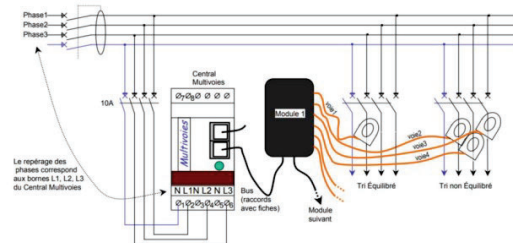
Topical Session 1C: Summer comfort and energy efficiency in hot periods: interest of mixed mode cooling and need of occupant feedback
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CASE STUDY IN REUNION ISLAND



ILET DU CENTRE OFFICE BUILDING FIELD MEASUREMENTS



- 37 **position sensors** NODON (ENOCEAN)
- Irregular timestamp
- 2 states [**0 ou 1**]

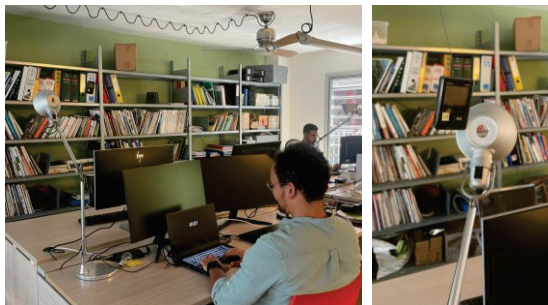
- Energy meters OMEGAWATT
- 1 min timestamp
- Ceiling fan power [W] and offices plug [W]



CASE STUDY IN REUNION ISLAND



ILET DU CENTRE OFFICE BUILDING FIELD MEASUREMENTS



- 9 TESTO 174H temp/rh sensors
- Regular timestamp
- Air Temperature (+/- 0.5°C) et relative humidity (+/- 3 %HR)

- Meteorological station
(Less than 1km far from site)

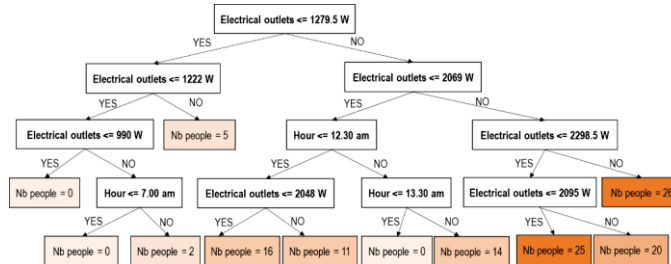


OCCUPATION AND BEHAVIOURS MODELS

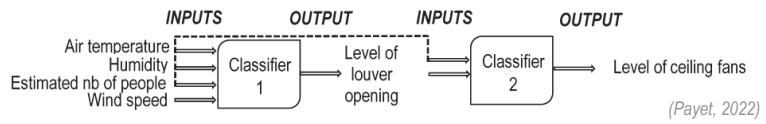


CLASSIFICATION METHODS

Occupation model
(Decision Tree)



Ceiling fans and openings models
(Random forest)



Payet, M., David, M., Lauret, P., Amayri, M., Ploix, S., Garde, F., 2022. Modelling of occupant behaviour in non-residential mixed-mode buildings: The distinctive features of tropical climates. Energy and Buildings 259, 111895. <https://doi.org/10.1016/j.enbuild.2022.111895>



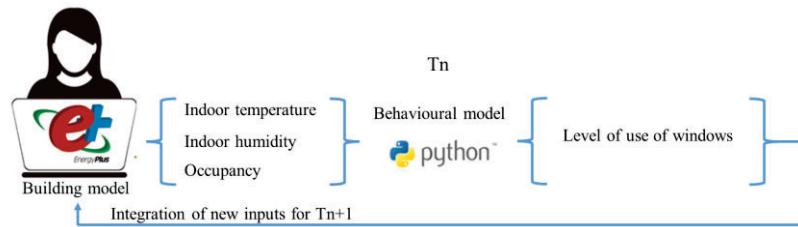
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IMPLEMENTING BEHAVIOURAL MODELS IN ENERGYPLUS



PYTHON PLUGIN METHOD



Method	Ease of implementation	Flexibility
Direct modelling	++++	+
Code customization	++	++
Customization of Core code	+	+++
Co-simulation	++	++++
Python plugin	+++	++++



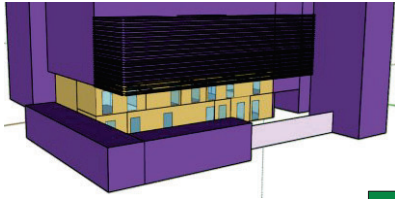
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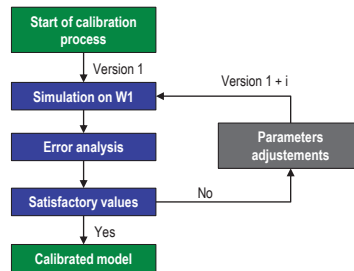
BUILDING ENERGY MODEL



VALIDATION (based on indoor conditions)



Week simulated	Step	From	To	Season	Internal heat gains from users
W1	Calibration	20/12/2020	27/12/2020	Summer	No
W2	Validation	01/12/2020	08/12/2020	Summer	Yes
W3	Validation	06/07/2020	13/07/2020	Winter	Yes
W4	Validation	05/10/2020	12/10/2020	Mid-season	Yes



Validation results	Standards
NMBEh ≤ 5.3 %	NMBEh ≤ 10 %
CV(RMSE)h ≤ 6.6 %	CV(RMSE)h ≤ 30 %
MADh ≤ 2.8 °C	(Baba, 2022)
MBEh ≤ 1.2 °C	(Baba, 2022)

- ASHARE Guideline 14 (Measurement and Energy Demand)
- Int. Measurement & Verification protocol recommended by French Energy Agency ADEME
- Mean Bias Error & Mean Absolute Deviation



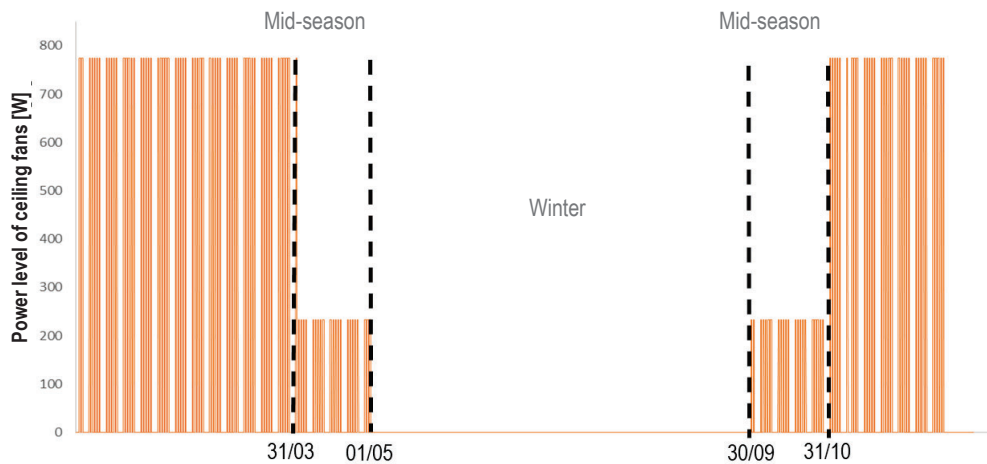
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CONVENTIONAL DESIGN OFFICE MODEL FOR CEILING FAN USE



BASELINE METHOD



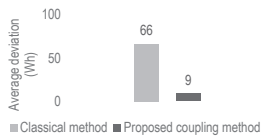
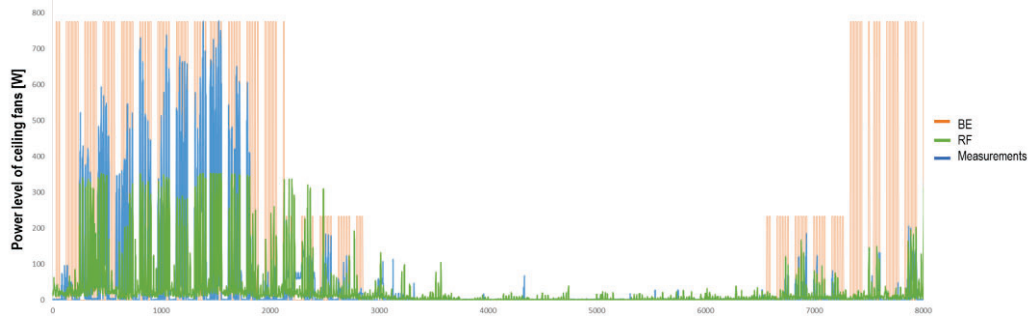
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COMPARISON WITH OUR RESULTS



PROPOSED COUPLING METHOD NOTED in GREEN / baseline orange



Skill Score: assessing the value of a new model compared with a reference model

SS Typical design office model / Coupling method proposed = 63 %



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TO CONCLUDE



LIMITS OF THE PRESENT WORK

- Lack of generalisation capabilities
- Only NV + CF has been modeled so far (no AC+CF)
- Better estimate Ceiling Fans use but still need to improve related energy use for each predicted class

PERSPECTIVES

- Extend field measurement studies to other building types and user categories to better teach models
- Add a level of complexity for mixed-mode cooled building with AC
- Investigate new way to estimate class energy use (seasonal class / monthly class, add model input parameter(s))



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in sustainable buildings

4-5
2023

Topical Session 1C: Summer comfort and energy efficiency in hot periods: interest of mixed mode cooling and need of occupant feedback


www.aivc2023.conference.org

Thank you for your attention

Further readings:

Payet, M., 2022. *Simulation du comportement des usagers dans les bâtiments tertiaires à faible consommation énergétique, en zone tropicale (phdthesis)*. University of la Reunion.

Payet, M., David, M., Lauret, P., Amayri, M., Ploix, S., Garde, F., 2022. *Modelling of occupant behaviour in non-residential mixed-mode buildings: The distinctive features of tropical climates*. Energy and Buildings 259, 111895.
<https://doi.org/10.1016/j.enbuild.2022.111895>



An innovative approach to better understand hot discomfort, based on the measurement of global human responses, including physiological and sensory indicators

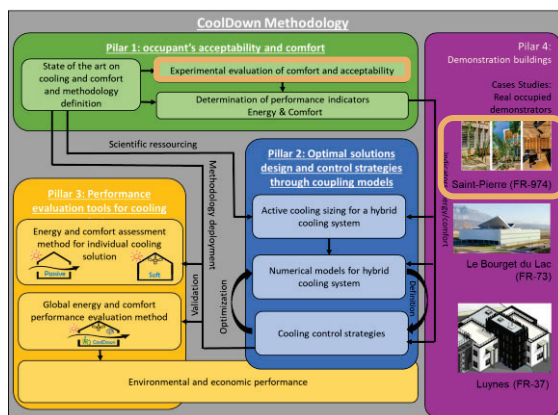
Application to end-users of mixed mode cooled buildings under tropical climate conditions

43rd AIVC -11th TightVent & 9th venticool Conference, October 4-5, 2023, Aalborg Univ., Copenhagen, Dk

Gwénaëlle Haese, Maxime Boulinguez, Pierre Bernaud, Anthony Couzinet

1

Context and objectives



3 main objectives in the task « Experimental evaluation of comfort and acceptability » of the CoolDown project

- 1 Measure physiological response directly on humans to better understand the origin of thermal discomfort.
- 2 Identify the link between office climatic conditions, perceived comfort and human physiological response.
- 3 Optimize and experimentally validate predictions from comfort models.

2

Buildings



Saint-Pierre, La Réunion
Average annual daytime temperatures [7 a.m. to 6 p.m.] > 23°C

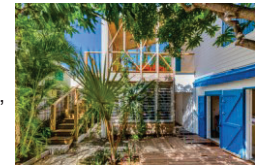
Hot period, expected daily temperature amplitude of 24 to 34°C with a relative humidity > 75%



“Ilet du Centre” building (IDC): large double floor open space office building built in 2008.
> Bioclimatic design primarily based on natural cross ventilation with louvres openings and double protection facades acting as fixed shadow devices (described by Payet et al. (2022)).

“CoArchitectes” (COA): first floor of an old basic concrete residential house on the city’s seaside.

- > Recently renewed and extended as an office building.
- > No natural cross ventilation in all spaces, 2/3 of the building with single-sided openings, therefore equipped with AC units and ceiling fans.
- > Limited solar impact thanks to a second floor and very dense nearby vegetation.



Subjects



21 volunteers

11 ♀
10 ♂

25 to 52 (mean age: 35.2 ± 9.1)



IDC building:
SU1-8



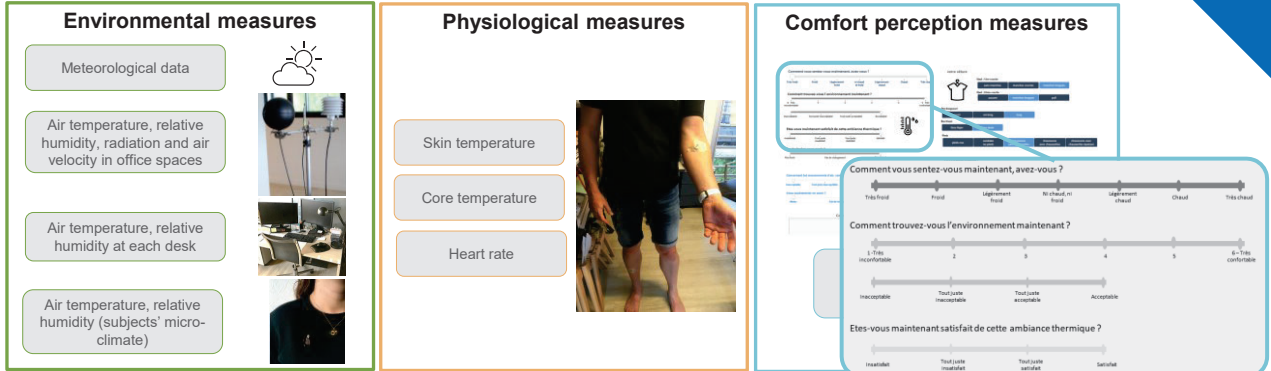
COA building:
SU9-21

Architects, engineers and landscape designers
Sensitized to the AEC industry’s environmental impact and having a basic-to-good understanding of comfort components and their impact on energy use and carbon emission in buildings



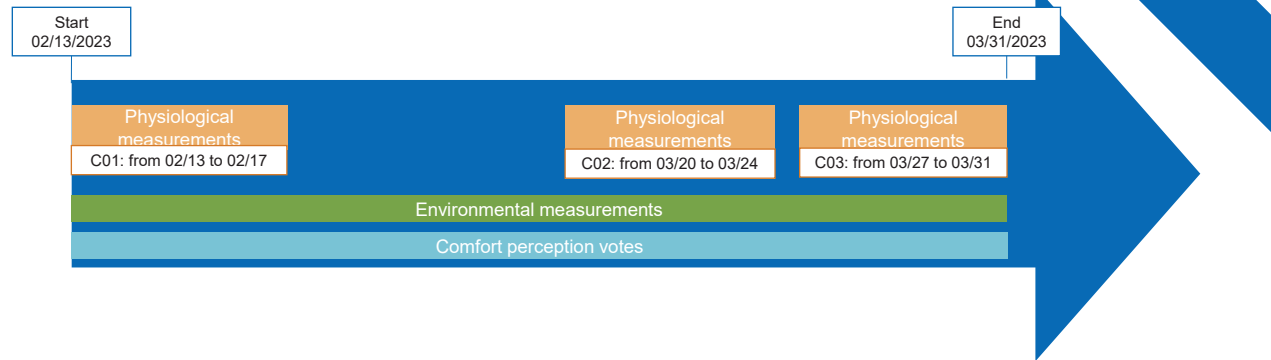
Methodology

Innovative approach
Combined measures on 21 subjects in their offices



5

Experimental campaigns



6

Data acquisition

Questionnaire

- Sensory questionnaires on clothing, activity, thermal sensation, perception of thermal comfort, level of acceptability and satisfaction
- 1 to 6 times per day

Smart watch

- Heart rate recorded by a connected watch,
- Acquisition of beats per minute in one-second time steps

Thermo-buttons

- Skin temperatures measured using thermo-buttons attached to the tibia, torso and forearm with biomedical dressings
- Measured every 5 minutes

Hygro-buttons

- Temperature and relative humidity close to the participant measured by a hygro-button attached to clothes
- At 5-minute time intervals (pin)

T°/RH box

- Temperature and relative humidity from the recording of the PULSE comfort box
- Measured at 10-minute intervals.

Measurement station

- Temperature, relative humidity, radiation, air velocity in the office measured by comfort measurements stations
- At 15-second time intervals

Mereen database

- From French weather stations located at Saint-Pierre, La Réunion
- Data at hourly intervals

Data collection and quality

Synchronization of the data from the different sources of measurements

→ 935 lines, corresponding to the **935 questionnaires** obtained from the 21 participants over the 3 campaigns.

Subject	Date	Time	Questionnaire						Thermo-buttons				Smart watch	Mereen database	Measurement station				Hygro-buttons		T°/RH box				
			Q1	Q2	Q3	Q4	Q5	Q6	Temperature - shin	Temperature - forearm	Temperature - chest	Temperature - core	Heart rate	Mean skin temperature	Meteorological data	Temperature - station	Relative humidity - station	Air velocity - station	Radiation - station	Temperature - pin	Radiation - pin	Relative humidity - pin	Temperature - box	Relative humidity - box	
SU1	02/13/2023	10:00																							
SU21	03/31/2023	18:15																							

0.5°C mean difference between the temperatures acquired by the box and the comfort stations,
 3.6% mean difference for relative humidity → **difference < uncertainty of the sensors.**

- Missing values from the comfort stations completed with the values of the boxes:
 - > 55 missing values remaining after the database cleaning for indoor temperature and indoor relative humidity measurements,
 - > 30 for the skin temperatures.

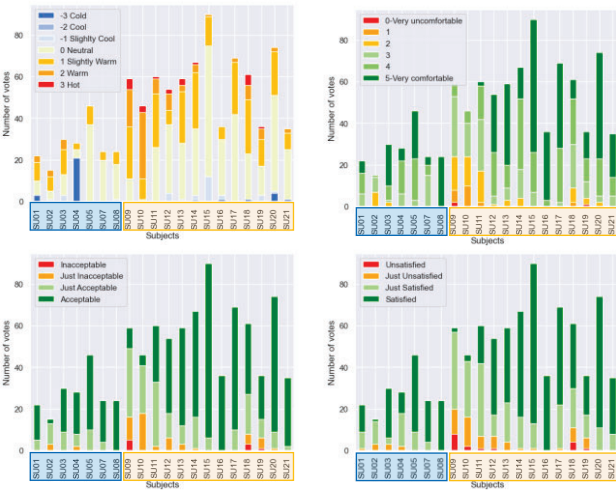
Outdoor and indoor conditions*



	Outdoor conditions	IDC Building	COA building
Mean T (°C)	28.5°C	28.3°C min: 25.4°C max: 29.4°C	29.3°C min: 23.9°C max: 31.3°C
Mean RH (%)	64.7%	59.8% min: 51.5% max: 81.8%	67.1% min: 50% max: 82.5%
Aerulic conditions	Wind from the SE, at a mean speed of 6.5 m/s (min: 0 m/s, max: 13.4 m/s)	Windows opened 85% of the time.	

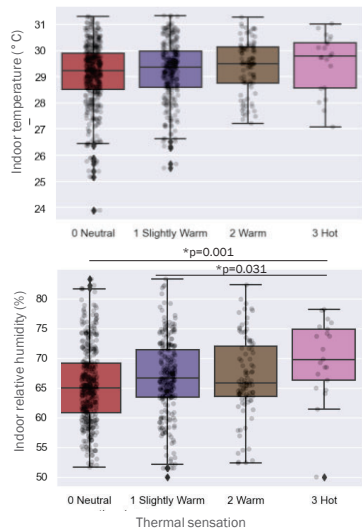
*Conditions seen by the subjects in the office buildings

Comfort perception of the occupants



- > Volunteers were globally satisfied (62% satisfied and 29% just satisfied)
- > With a thermal sensation mainly neutral (50%) or slightly warm (30%)
- > Same analysis for comfort and acceptability.
- > Only COA building occupants felt "hot" and "unsatisfied"

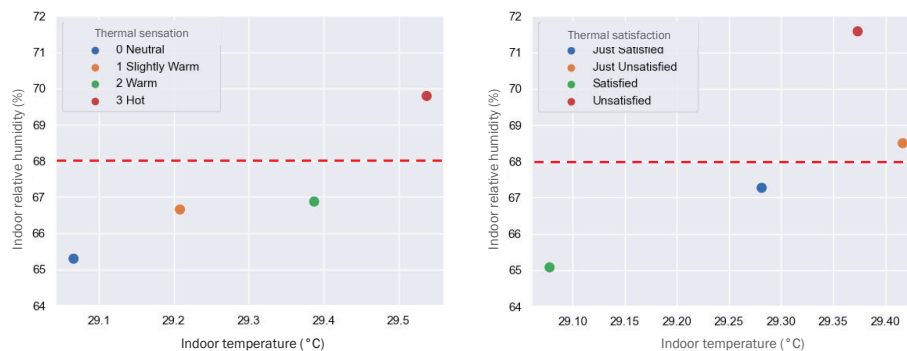
Data crossing



- > The mean indoor temperature for each modality of thermal sensation is similar (about 29.5°C)
- > The indoor temperature is insufficient to explain the differences between the chosen thermal sensation levels
- > On the contrary, the influence of the RH level is very strong on thermal sensation
- > Significant differences are observed between “neutral” and “hot” and between “slightly warm” and “hot” thermal sensations

12

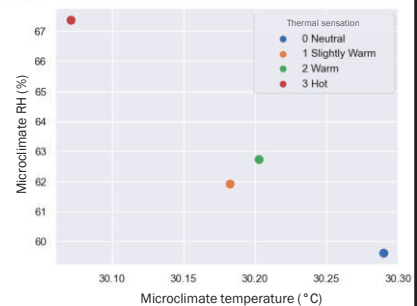
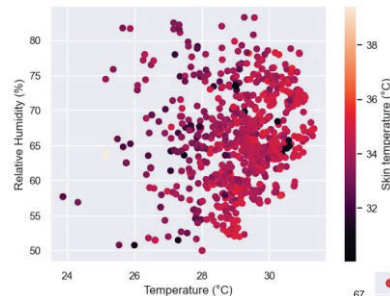
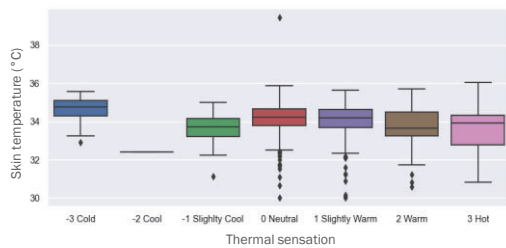
Data crossing



- > The mean indoor temperature for each modality of thermal sensation and satisfaction is almost constant (about 29.3°C)
- > Results confirm that **relative humidity is the main driver** of thermal sensation, comfort acceptability and satisfaction levels, indifferently of the temperature values in the observed range (28-30°C)
- > Hot sensation, as well as discomfort, and unsatisfaction appear at a **threshold of 68% of relative humidity**

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Data crossing



- > Skin temperature seems to decrease as the thermal sensation increases
- > Skin temperature arises with indoor temperatures, but relative humidity plays a role on thermal sensation as seen before

Questions:

- Incomplete measure due to the sensor interaction with the skin: what about the sweating impact ?
- Which impact of sweating evaporation and skin temperature on micro-climate conditions?
- For now, 3-measurement points for skin temperature: more points needed?

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Future works and outlook

- > New measurements campaigns starting November 2023 to gather more data and optimize the protocol and data acquisition and to validate the calibration of comfort models on the same subjects
- > Study of the parametric sensitivity of the comfort models
- > Individualized thermophysiological comfort model calibration for La Reunion's specific population

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 Ventilation, IAQ and health in sustainable buildings

October 4-5 2023
 Copenhagen Denmark

Topical Session 1C: Summer comfort and energy efficiency in hot periods: interest of mixed mode cooling and need of occupant feedback



An IAQ and thermal comfort coach prototype to improve comfort and energy consumption thanks to adequate management of natural ventilation: development and first feedback results


Arnaud JAY, Pierre BERNAUD, Franck ALESSI CEA Liten - INES, FRANCE





1

CONTEXT



- Natural ventilation through open Windows allows to exchange easily 5 - 10 ACH for free
- But might bring some additional energy consumption or incomfort if open at inappropriate time



Thermal Comfort



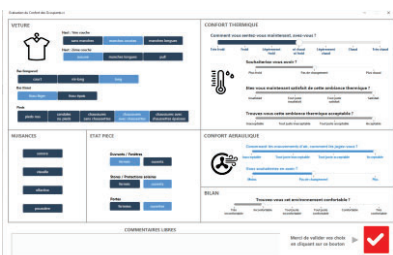
IAQ Energy Consumption

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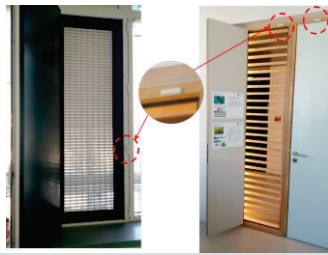
2


Wind'ose genesis




Interface to collect User feedback

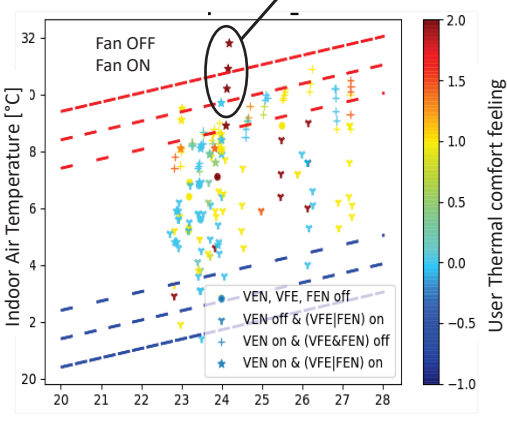
Monitoring of Windows /Doors opening





Windows open:
Inappropriate Behaviors ?





Indoor Air Temperature [°C]

slippery average outside temperature

Legend:

- VEN, VFE, FEN off
- VEN off & (VFE|FEN) on
- VEN on & (VFE&FEN) off
- VEN on & (VFE|FEN) on

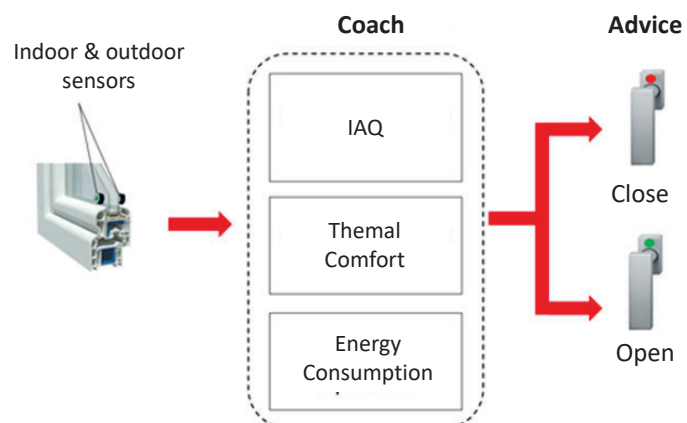
User Thermal comfort feeling

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WIND'OSE OBJECTIVE

- Coaching tool developed to help occupants to know whether it is a good option to open or close their windows.
- Coach objective is to consider the three components:
 - thermal comfort,
 - indoor air quality (IAQ)
 - energy consumption



→ Based on Patent EP3971490 on "Method and system for advising on the opportunity of activating a door in order to improve the thermal comfort and/or the quality of the air"

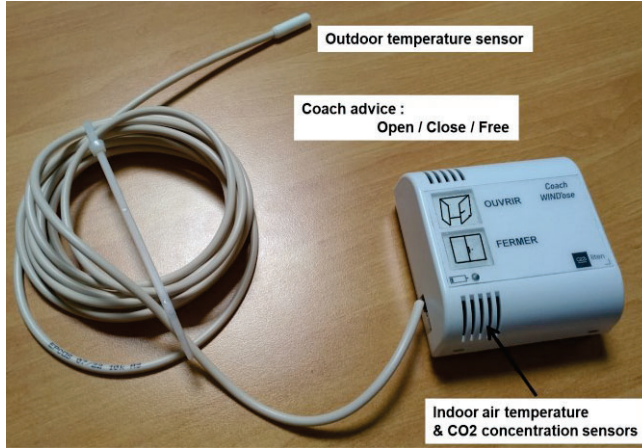
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WIND'ose an e-fAIR prototype

SPECIFICATIONS AND PROTOTYPE V1 -

- 2 Indoor Sensors:
- Indoor air temperature
 - CO₂ concentration
- 1 Outdoor sensor
- Outdoor air temperature



Intuitive message communicate to users

Autonomous (Energy & embedded algorithm) and easy to set up

WIND'OSE ALGORITHM

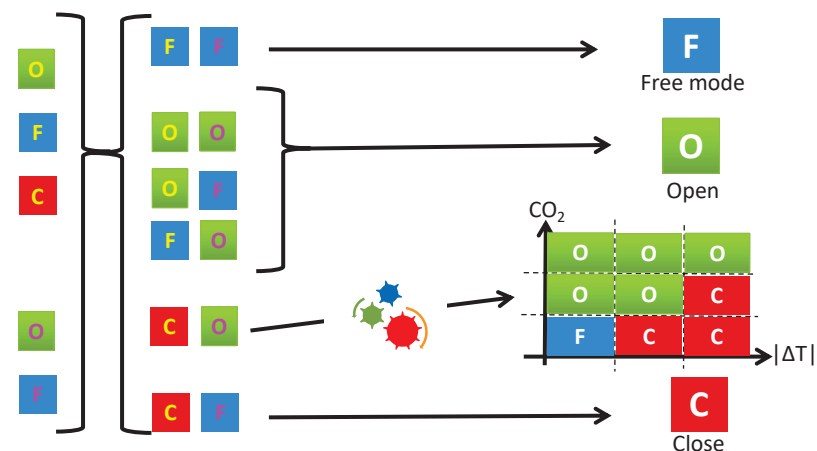
- 1 Separate evaluation of each criteria
- 2 Check consistency between the 2 separate evaluations
- 3 If necessary – Arbitrate between criterias
- 4 Advice push to user

Thermal criteria

Potential depending on Tint-Text + Opportunity based on adaptive comfort

IAQ criteria

Indoor CO₂ concentration compared to a lower threshold

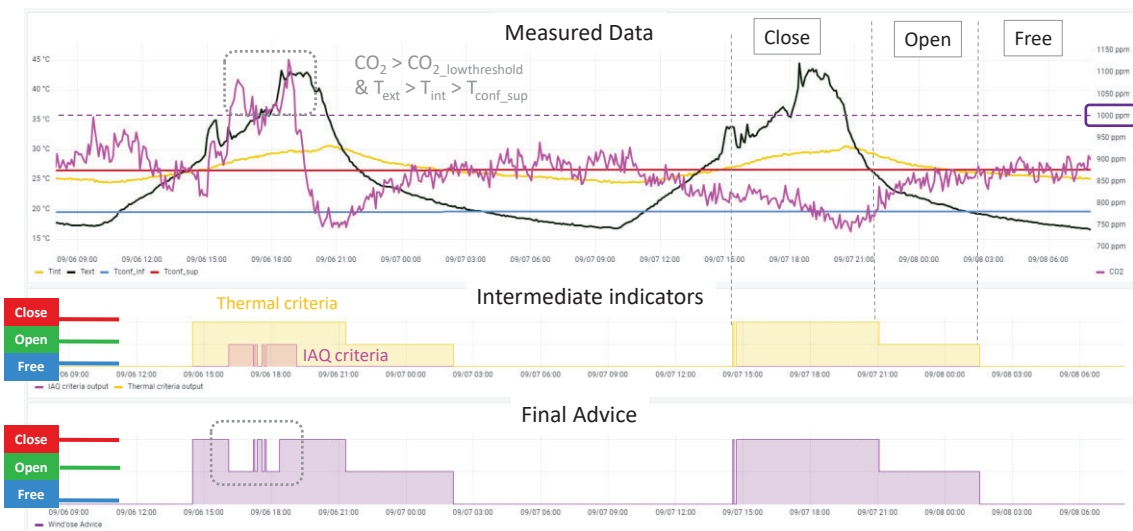


**EXPERIMENTAL CAMPAIGN
SET UP ON A REAL BUILDING.**



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FIRST RESULTS



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CONCLUSION AND PERSPECTIVE

- Wind'ose prototype answer our 1st requirements and specifications
- What's next:
 - Increase period and number of user feedback
 - Improve algorithm, and integrate specificity from
 - Mid-season
 - Winter
 - Enhance battery autonomy or integrate PV cells to get it fully autonomous.
 - Integrate other pollutants sensors.
 - ...



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Topical Session 1C:
Summer comfort and energy efficiency in hot periods:
interest of mixed mode cooling and need of occupant feedback

OPEN DISCUSSION

Arnaud JAY
CEA Liten - INES

Gwénaëlle HAESE
CSTB

Maxime BOULINGUEZ
LEU Réunion / La Réunion Univ. - PIMENT

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CEA Liten - INES



Gwénaëlle HAESE
CSTB



Maxime BOULINGUEZ
LEU Réunion / La Réunion Univ. - PIMENT




















1

TOPICAL SESSION CONTENT

- **Towards an alternative cooling: Optimisation of the successive use of the cooling systems from passive to active - Development of design and control strategies of the hybrid cooling.**
 - Arnaud JAY, CEA Liten - INES, FRANCE
 - Gwénaëlle HAESE, CSTB, FRANCE
 - Maxime BOULINGUEZ, La Réunion University - PIMENT, France



- **An innovative approach to better understand hot discomfort, based on the measurement of global human responses, including physiological and sensory indicators - application to end users of mixed mode cooled buildings under tropical climate conditions.**
 - Gwénaëlle HAESE, CSTB, FRANCE
 - Maxime BOULINGUEZ, La Réunion University - PIMENT, France
- **Windows and ceiling fan occupant behaviour model coupling methodology with building energy models, a tropical case study.**
 - Maxime BOULINGUEZ, La Réunion University - PIMENT, France
- **An IAQ and thermal comfort coach prototype to improve comfort and energy consumption thanks to adequate management of natural ventilation: development and first feedback results.**
 - Arnaud JAY, CEA Liten - INES, FRANCE
- **Open discussions**

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Towards an alternative cooling:

Optimisation of the successive use of the cooling systems from passive to active - Development of design and control strategies of the hybrid cooling

Arnaud JAY, CEA Liten - INES, FRANCE
 Gwénaëlle HAESE, CSTB, FRANCE
 Maxime BOULINGUEZ, Université La Réunion - PIMENT, France
 Virginie CHANTEPIE, Bluetek, France
 Pierre CONSTANT-BERAUD, Bluetek, France

anr The project *CoolDown* is funded by ANR French Research Agency under the number ANR-22-CE22-0014-












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CONTEXT

Global Warming



Increase of
→ Energy consumption for Space Cooling

→ Increase of Health problem in hot period


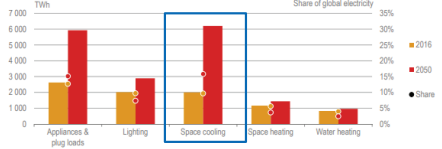
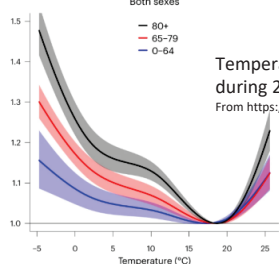


Figure 3.8 • Building electricity demand by end-use application in the Baseline Scenario




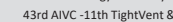

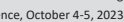


From « The future of Cooling », IEA report 2018
https://iea.blob.core.windows.net/assets/0bb45525-277f-4c9c-8d0c-9c0c5e7d525/The_Future_of_Cooling.pdf

Both sexes



Temperature-related risk of death during 2015–2019
 From <https://doi.org/10.1038/s41591-023-02419-z>

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CONTEXT

Cooling energy Consumption – Baseline scenario

Efficient Cooling scenario

Additional Savings Potential

From « The future of Cooling », IEA report 2018
https://iea.blob.core.windows.net/assets/0bb45525-277f-4c9c-8d0c-9d0c5e7d525/The_Future_of_Cooling.pdf

- Active cooling can be optimized AND
- some alternative solutions exist

Natural ventilation

Insulation

Solar Control

Thermal inertia

Fans (ceiling / desk)

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COOLDOWN OBJECTIVES

Define tools and methodology to optimise the successive and combined use of passive, soft and active solutions to maximize comfort for occupants while minimising energy consumption in summer, hot seasons or heat waves to face the climate change impact

→ Focus on existing office buildings.

Passive

Soft

Active

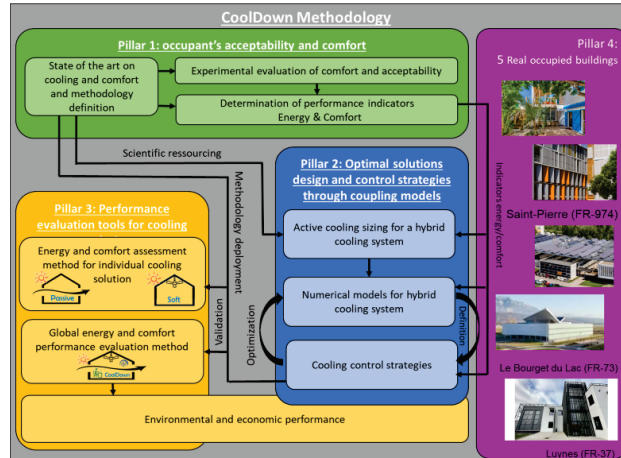
Iso comfort

Energy use for cooling systems

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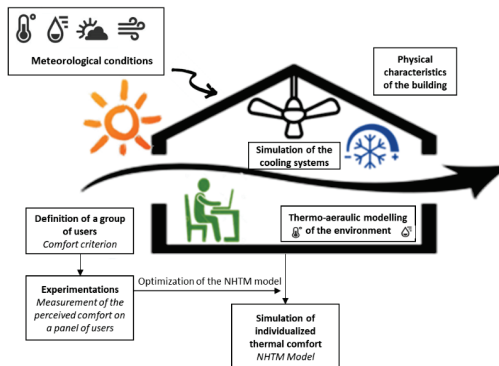
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PROJECT SCOPE AND METHODOLOGY



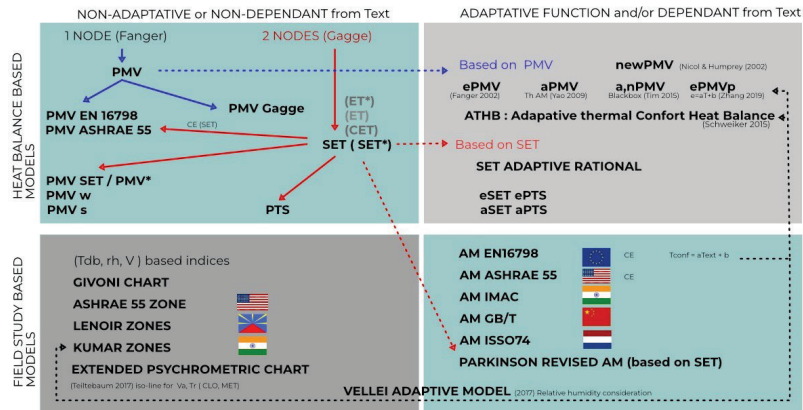
COOLDOWN PILLAR#1: Occupant acceptability and comfort Methodology

- Get feedback of occupants acceptability regarding the passive, soft and active cooling solutions through surveys and monitoring
- Improve thermal comfort models thanks to collected data
- Value this knowledge to defined indicators & control strategies



COOLDOWN PILLAR#1: Occupant acceptability and comfort

First Results – Comfort models mapping

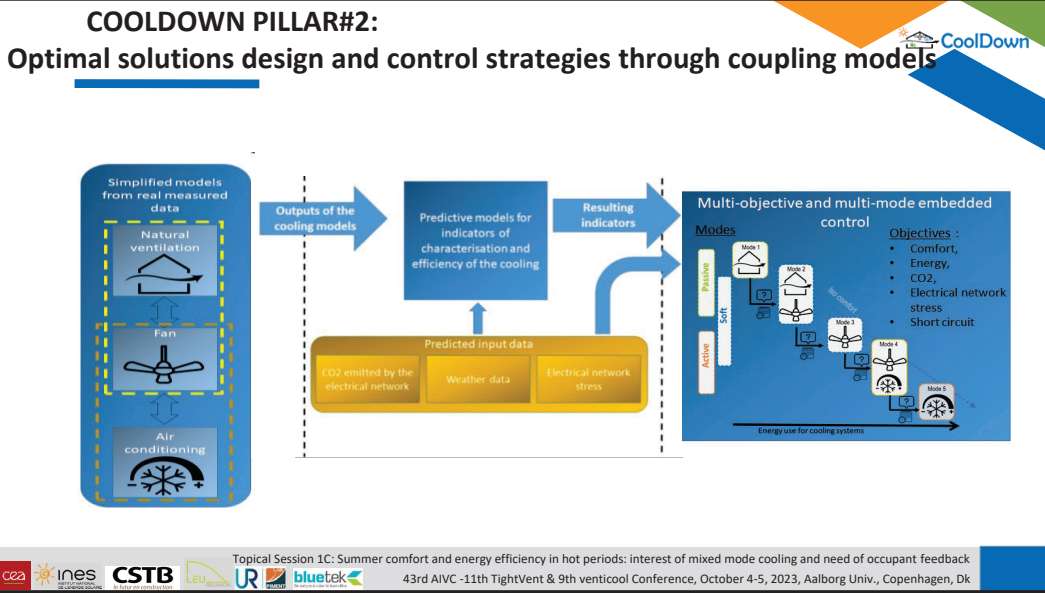


Adapted from Boulinguez & al, IBPSA France 2022, <https://cea.hal.science/cea-03988262>

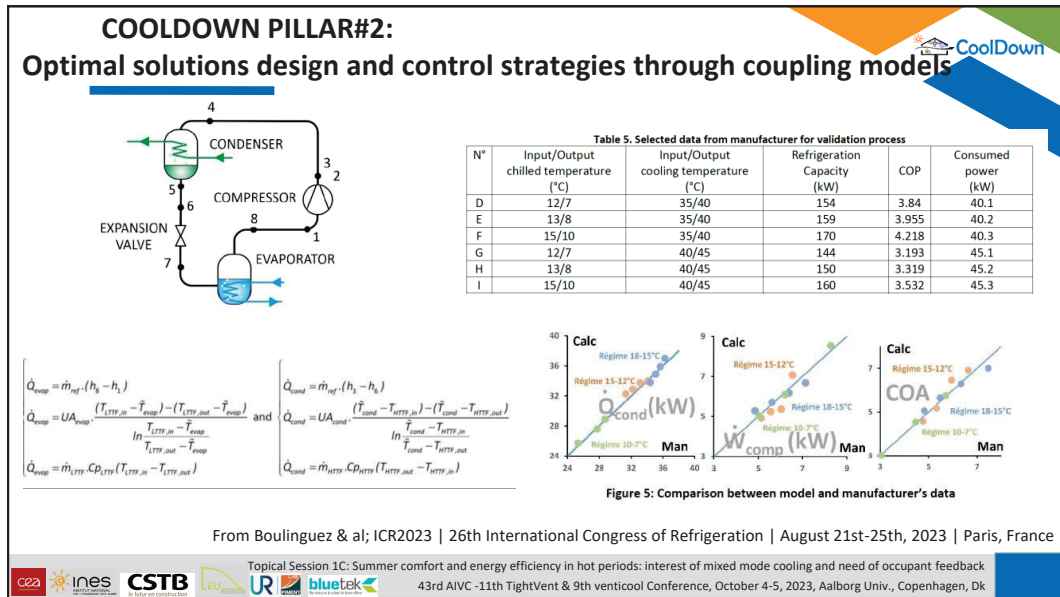
COOLDOWN PILLAR#1: Occupant acceptability and comfort

First Results – PULSE campaign @LaReunion Demo#1 & #2





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COOLDOWN PILLAR#3:
Performance evaluation tools for cooling solutions

Consider both **Comfort & Energy** in **Performance assessment methods & Indicators**

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COOLDOWN PILLAR#3:
Performance evaluation tools for cooling solutions


	Indicators	M&V protocol
Natural ventilation	ACH	Tracer Gaz
Fans (ceiling, desks)		
Active cooling		
Solar gain control	Asol (Solar Aperture)	
Insulation	HTC,	CO-heating, Qub, Pstar, SEREINE, ...
Thermal inertia		


Task just started - Any input welcome

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COOLDOWN PILLAR#4: Real occupied offices building







5 real occupied buildings monitored to:


- Generate database of users feedback & physical data
- Test methods and control strategies












France metropole Reunion Island







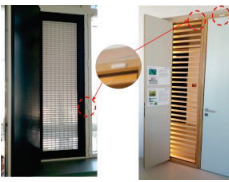







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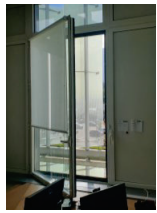
15

COOLDOWN PILLAR#4: Real occupied office buildings


First results



Window louvers to Outdoor



Windows to outdoor



Desk Fan

Fenêtres: fréquence à l'état "ON"







% of opening time

Fenêtres: fréquence à l'état "ON"

% of opening time

Fenêtres: fréquence à l'état "ON"

% of time ON

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COOLDOWN PILLAR#4: Real occupied office buildings

First results



Clothing

Thermal Comfort : sensation, preference, satisfaction, acceptance

Potential annoyance

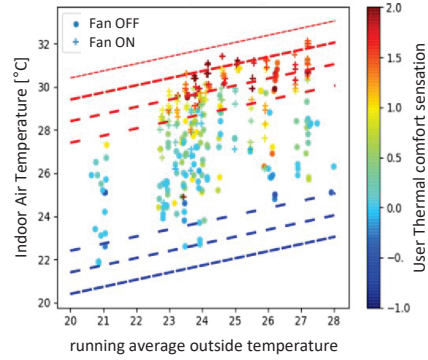
Office Status : Windows, fans, ...

Air movement acceptability and preference

Global Comfort

Interface to gather user feedback

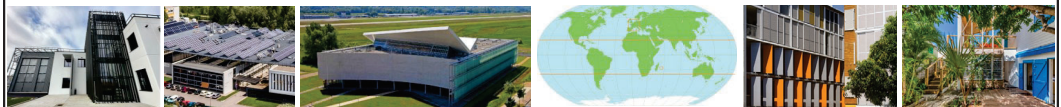
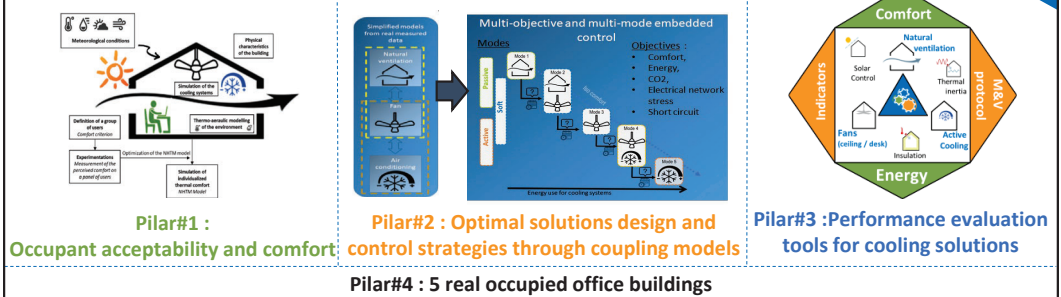
User Comfort on Adaptative comfort chart




Conclusion and perspective



Let's continue the CoolDown adventure : So many challenges to tackle !






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23/24

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CoolDown

The project *CoolDown* is funded by ANR
French Research Agency
under the number ANR-22-CE22-0014-06




THANK YOU

-
QUESTIONS ?











AIVC2023, Copenhagen

IEA-EBC Annex 78 Supplementing Ventilation with Gas-phase Air Cleaning, Implementation and Energy Implications

Bjarne W. Olesen & Pawel Wargocki
ICIEE,
DTU.SUSTAIN
Technical University of Denmark

October 4 2023

Annex 78 Project Period

~~July 1, 2018 Preparation period~~
~~July 1, 2019 Activity period~~
July 1, 2023 Reporting period
July 15, 2024 Finished

ORGANISATION

- Subtask A: Energy benefits using gas phase air cleaning
- Subtask B: How to partly substitute ventilation by air cleaning
- Subtask C: Selection and testing standards for air cleaners
- Subtask D: Performance modelling and long term field validation of gas phase air cleaning technologies

Ventilation
Information
Paper
n° 42

April 2021

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B-1060 Brussels – Belgium
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International Energy Agency's
Energy in Buildings and Communities
Programme



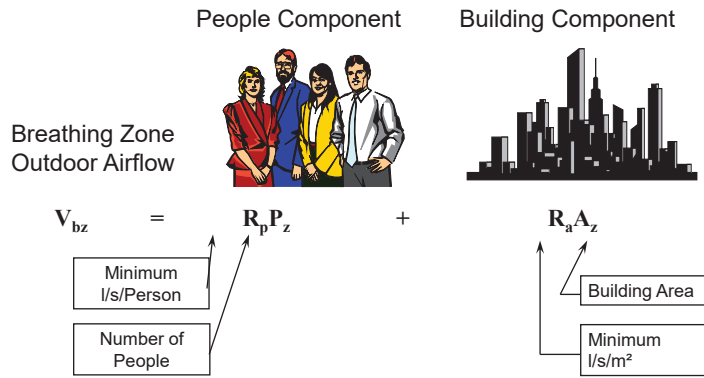
Air Infiltration and Ventilation Centre

The Concept for Substituting Ventilation by Gas Phase Air Cleaning

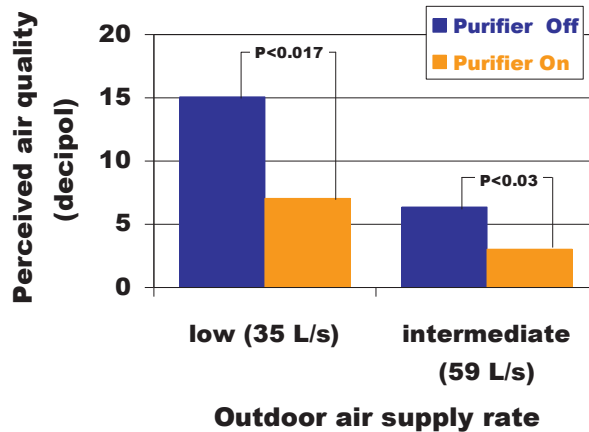
Bjarne W. Olesen, DTU, Denmark
Chandra Sekhar, National University of
Singapore, Singapore
Pawel Wargocki, DTU, Denmark

The concept for calculation of design ventilation rate

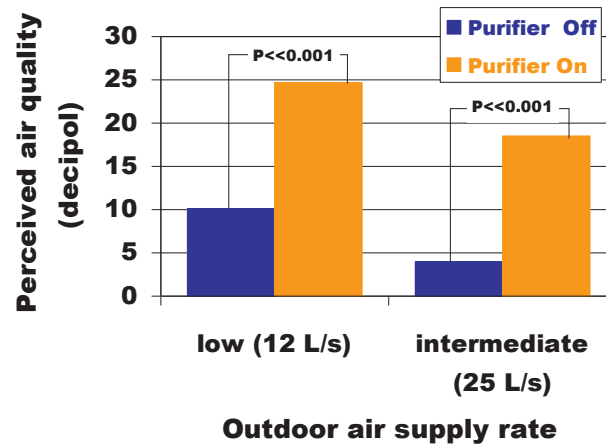
ISO CEN ASHRAE



Results: Bldg mat, PCs, filters

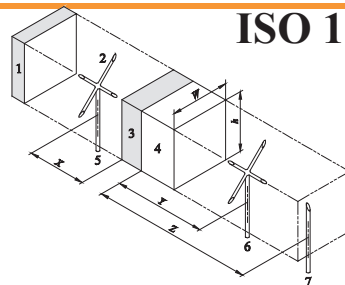


Results: Human bioeffluents



TESTING OF GAS PHASE AIR CLEANERS

ISO 10121-2:2014



- Key
- 1 diffuser and Δp device
 - 2 sampling points – should be of “fork” type or similar with multiple inlet points to make a compounded sample over the whole cross section
 - 3 GPACD under test
 - 4 GPACD section of test duct
 - 5 upstream sampling point for T_D , RH_D , p_D and C_D at X mm before the GPACD
 - 6 Downstream sampling point for T_D , RH_D , p_D and C_D at Z mm after the GPACD
 - 7 Q , air flow rate sampling point at Z mm after the GPACD
- W internal width of the test duct along the GPACD section, 3+4
 h internal height of the test duct along the GPACD section, 3+4

Figure 1 — Normative section of test stand showing ducting, measurement parameters and sampling points

Air Cleaning Efficiency

$$\epsilon_{\text{clean}} = 100(C_U - C_D)/C_D$$

where:

- ϵ_{clean} is the air cleaning efficiency
- C_U is the gas concentration before air cleaner
- C_D is the gas concentration after air cleaner.

ISO/TC 146/SC 6

Date: 2023-09

ISO/FDIS 16000-44:2023 (E)

ISO/TC 146/SC 6/WG 25

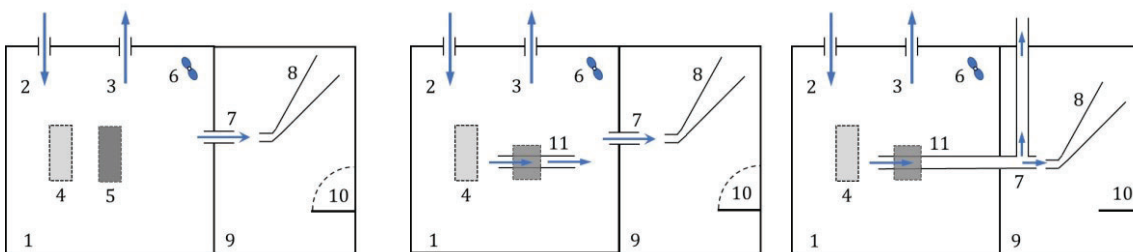
Secretariat: DIN

Indoor air — Part 44: Test method for measuring perceived indoor air quality for use in testing the performance of gas-phase air cleaners.

Air intérieur — Partie 44: Méthode d'essai pour mesurer la qualité perçue de l'air intérieur en vue de tester les performances des épurateurs d'air en phase gazeuse

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Position of Air Cleaner ISO 16000-44



a) A test room for a standalone air cleaner

b) A test room for a duct air cleaner

c) A test room for a duct air cleaner (single-pass condition)

- 1 test chamber
- 2 clean and temperature/humidity conditioned air supply inlet
- 3 exhaust outlet
- 4 emission source
- 5 An air cleaner
- 6 mixing fan

- 7 tube or duct
- 8 sniffing device, complying with relevant specifications and requirements of ISO 16000-28
- 9 front/anterior space in which human panel enter
- 10 Doors where panel enters
- 11 in duct air cleaner

1

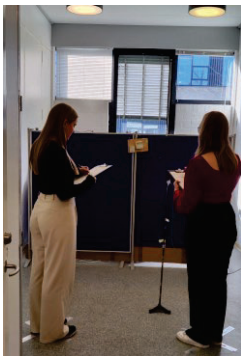
10

Testing of gas-phase air cleaners for improving perceived indoor air quality (PWI 23743) ISOTC142WG8

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■ Sensory evaluations

➤ Type of exposure

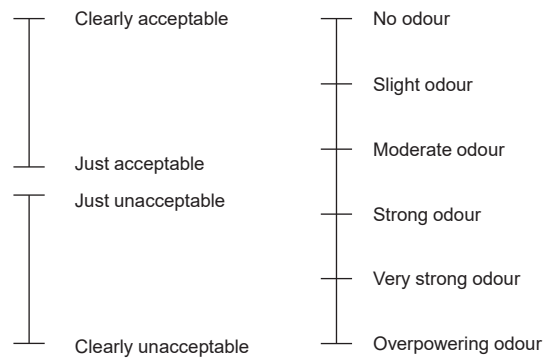


Whole-body exposure

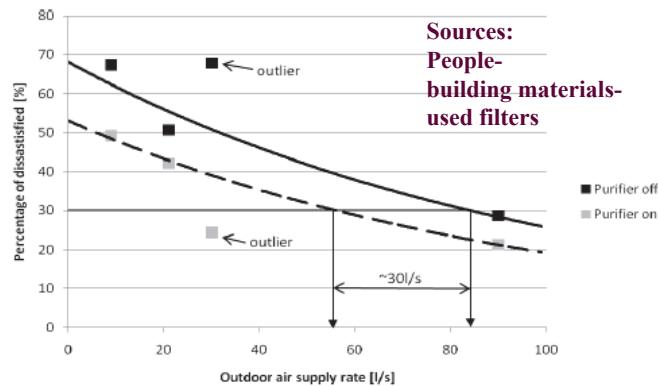


Facial exposure

➤ Scale



Effect of air cleaning on perceived Air Quality



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Deliverables

- A: A method for predicting the energy performance of gas-phase air cleaning technologies and the possible reduction of energy use for ventilation.
 - Lead by Dragos-Ion Bogatu (DTU) and Sasan Sadrizadeh (KTH)
- B: A validated procedure for supplementing (partly substituting) required ventilation rates with gas-phase air cleaning.
 - Lead by Bjarne W. Olesen and Pawel Wargocki
- C: A test method for air cleaning technologies that besides chemical measurements include perceived air quality as a measure of performance. [SEP]
 - Lead by Pawel Wargocki, Lei Fang and Bjarne Olesen (DTU)
- D: A report on the long-term performance of gas-phase air cleaning technology
 - Lead by Karel Kabele
- E: Models for predicting the performance of gas-phase air cleaning
 - Lead by Jianshun Zhang
- F: A report on Gas Phase Air Cleaning Technologies. Publish with AIVC
 - Lead by Alireza (Fang Lei, Jinhan Mo)

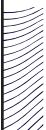
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Air cleaner as an alternative to increased ventilation rates in buildings:

A simulation study for an office

Alireza Afshari



Air cleaner as an alternative to increased ventilation rates in buildings: A simulation study for an office



Alireza Afshari
Alessandro Maccarini
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Department of the Built Environment (BUILD)
Aalborg University Copenhagen
Denmark





Abstract

3

This study analyses the feasibility of utilizing advanced air cleaner technology for air purification in:

- a system-based filter (recirculating ventilation system),
- a room-based filter (local recirculation in each room),
- a beam-based filter (recirculation in an active chilled beam).

The results show that choosing the appropriate air cleaner can significantly impact energy performance and improvement of indoor air quality.



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Background

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Air pollution poses significant risks to human health, as it comprises a combination of gaseous and particulate contaminants.

This exposure has been linked to adverse effects on the immune [1], respiratory, and cardiovascular systems [2-4], as well as an increased risk of lung cancer [5] and premature mortality [3].

Short-term symptoms of exposure to poor indoor air quality include headaches, eye, nose, and throat irritation, fatigue [6], and asthma [7], which can lead to decreased productivity and increased workplace absenteeism [8].



4

Building model

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The building model was chosen to be representative of a typical office room located on the middle floor of a high-rise building. The room has a heated area of 16 m² and a volume of 48 m³.

All the surfaces are considered adiabatic (thermally isolated), except for the south-oriented façade (wall), where ambient boundary conditions are applied.

This facade also includes a window of 6 m². The facade has two parts, an opaque element, and a glass element, with U-values of 0.3 W/m²K and 1.5 W/m²K, respectively.

Shading devices are installed outside the window, which can block 50% of incoming radiation when direct solar radiation on the south facade is higher than 150 W/m². Shadings from nearby obstacles are not considered.



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Building model

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Hourly resolution profiles for occupancy, appliances, and lighting were used to represent user behaviour and internal heat gains.

The profiles were generated based on different user behaviours for weekdays and weekends.

The peak heat gain was assumed during working hours on weekdays and was set 20 W/m².

The natural infiltration rate was assumed to be constant and the air change per hour was set to 0.2 ACH.



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Building model

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The filter was modelled to reduce the concentration of pollutants by 80%.

CO₂ was used for the evaluation of indoor air quality.

During working hours, an indoor production of 9.1E-6 kg/s of CO₂ was assumed, while an outdoor concentration of 400 ppm was used as a reference concentration.

An ideal space heating and cooling system was modelled in order to keep the indoor air temperature within the range of 20-24°C equal for all systems simulated.



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System models

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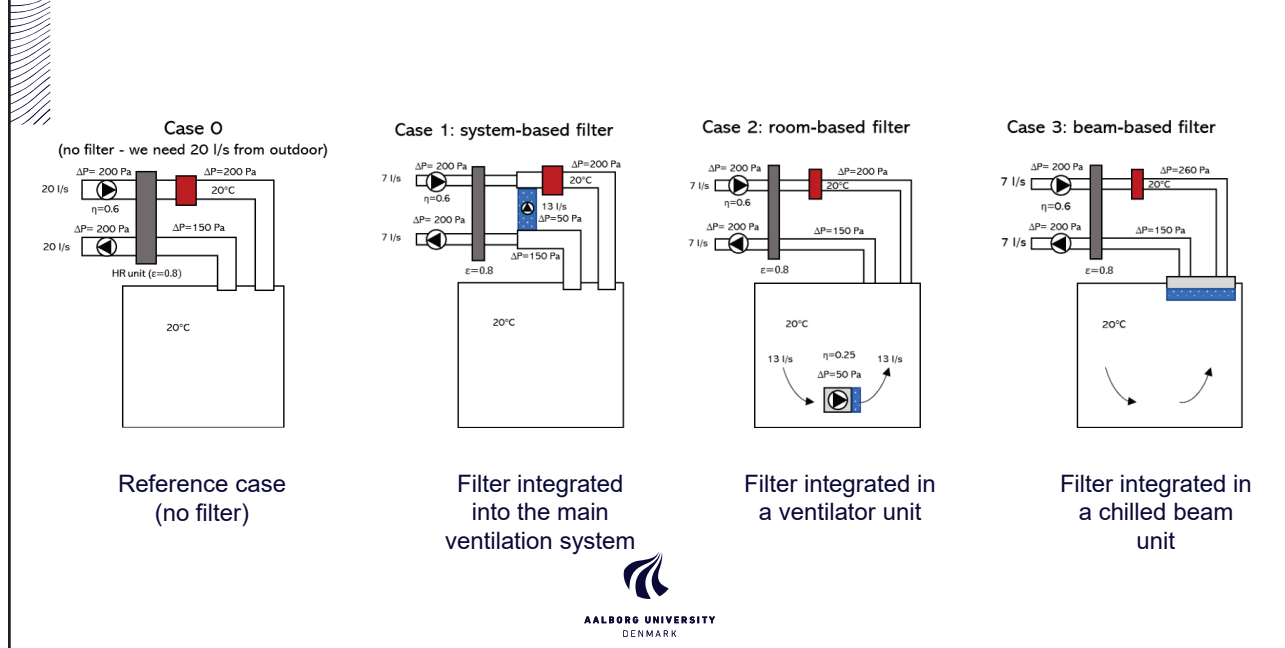
Four different HVAC system configurations were implemented, namely:

- Reference case,
- System-based filter,
- Room-based filter,
- Beam-based filter.



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Reducing the energy use of ventilation systems by including filters



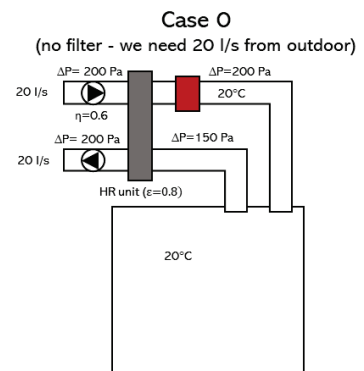
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The reference case:

Represents a typical office ventilation system consisting of a heat recovery unit, a heating coil, and supply and return fans. The heat recovery unit was modelled as a rotary heat exchanger where the speed of rotations (effectiveness) was regulated according to the actual needs in terms of heat transfer between supply and return air streams.

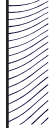
The maximum effectiveness was set to 0.8. An outdoor air flow rate of 20 L/s is delivered to the office room by the supply fan.

The office room receives an outdoor air flow rate of 20 L/s from the supply fan, which has an efficiency of 0.6. The ducts were set to have pressure drops of 150 Pa, whereas the heat recovery unit was set to have a pressure drop of 200 Pa. The heating coil, which maintains the supply air temperature at 20°C, was assumed to have a pressure drop of 50 Pa.



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System-based filter

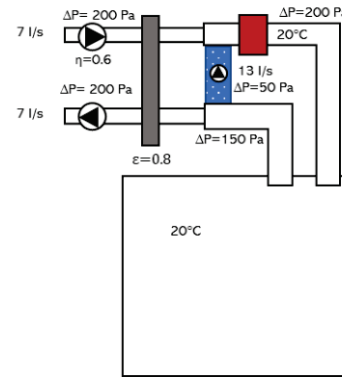
introduces a filter at the system level. This allows for lower outdoor air flow rates by filtering return air from the room.

For this case, an outdoor air flow of 7 l/s was assumed, with a 50 Pa pressure drop set for the filter.

It's worth noting that this approach allows for smaller ducts to be installed in correspondence to the heat recovery unit. The filter was modelled accordingly.



Case 1: system-based filter



Room-based filter:

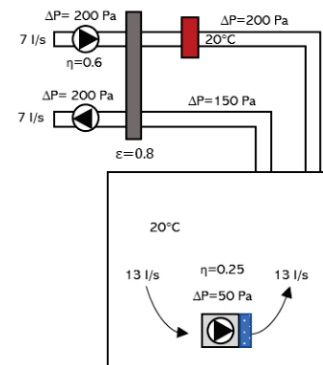
introduces a filter at room level.

In this case, the air is filtered within the room using a small fan.

It's worth noting that, such fans are generally less efficient, with a typical value of 0.25 assumed in this study. A pressure drop of 50 Pa was set for the filter.



Case 2: room-based filter



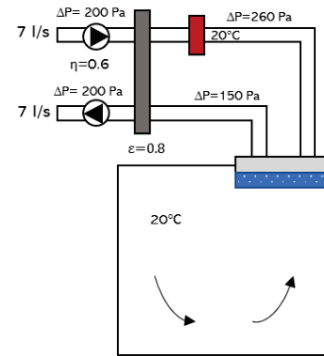
Beam-based filter:

introduces a filter incorporated into a chilled beam unit

In this case, the induced room air is filtered by a device integrated into the beam unit.

To induce air through the beam unit, the pressure drop in the duct system was increased to 210 Pa.

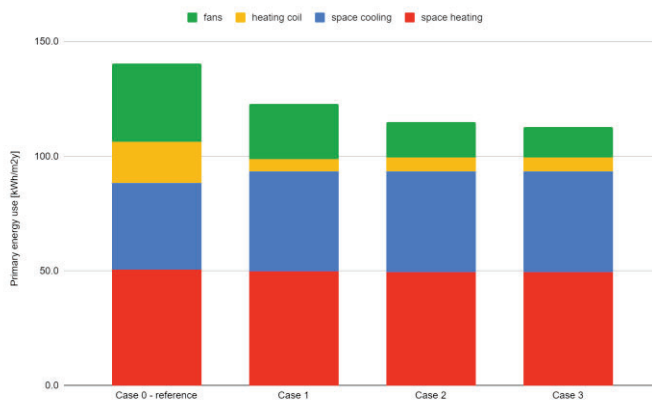
Case 3: beam-based filter



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Reducing the energy use of ventilation systems by including filters



Case 3 (beam-based filter) lead to 24% of total primary energy savings compared to the reference case

14



Conclusions

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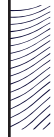
The room-based and beam-based filter systems have similar energy demands for space heating, while the reference case has a slightly higher energy demand due to the lack of air recirculation.

For space cooling, the room-based and beam-based filter systems have the highest energy demand, while the reference case has the lowest energy demand due to a higher supply of outdoor air.

The system using filters has the lowest energy demand for the heating coil, whereas the reference case has the highest demand.



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Conclusions

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For fans, the beam-based filter has the lowest energy demand due to efficient recirculation of air through induction, while the reference case has the highest due to higher air flow through the air handling unit.

Integrating a filter in the active chilled beam unit results in primary energy savings of approximately 26% compared to the reference case.



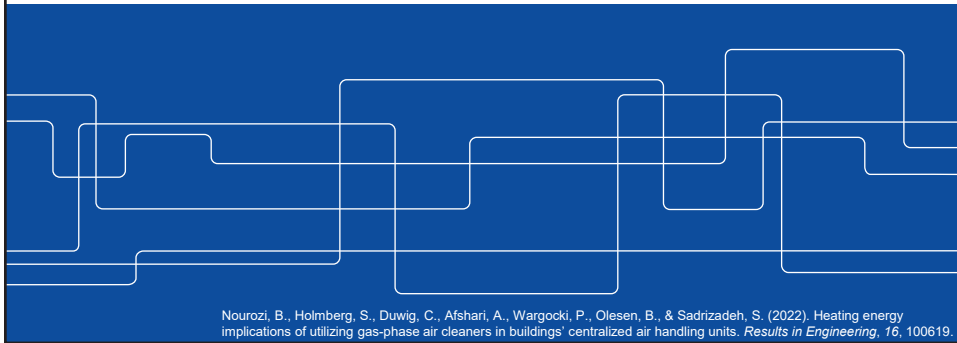
16



Energy implication of using gas-phase air cleaners in residential & office buildings

Sasan Sadrizadeh

PhD, Professor
KTH & MDU



Nourozi, B., Holmberg, S., Duwig, C., Afshari, A., Wargocki, P., Olesen, B., & Sadrizadeh, S. (2022). Heating energy implications of utilizing gas-phase air cleaners in buildings' centralized air handling units. *Results in Engineering*, 16, 100619.

1



Introduction and background

- **Ventilation** systems are important for maintaining a **healthy and comfortable indoor** environment.
- In cold climates, ventilation systems contribute to **approximately 30% of building heat losses**.
- **Indoor emissions** and **outdoor pollutants** affect **indoor air quality** and need to be controlled.
- **Gas-phase air cleaning** as an extension to the ventilation can help **maintain acceptable indoor air quality**, yet **reduce energy use**.

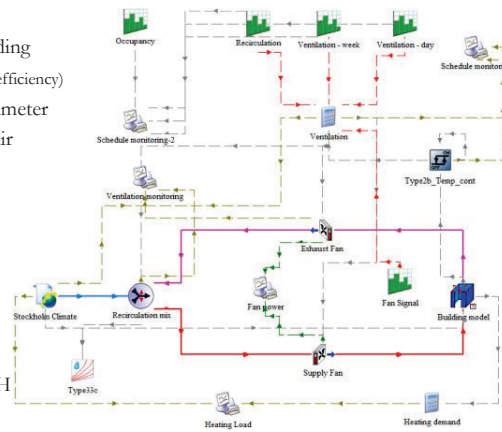
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Investigated parameters

- Heating demand of a ventilated building
- Indoor TVOC level (with 60% capturing efficiency)
- Indoor CO₂ level as a monitoring parameter
- Possibility of air recirculation when air cleaner is integrated.

Simulation case

- Newly constructed or renovated buildings
- Older buildings without heat recovery ventilation
- Residential and office cases with various ACH



Energy simulation using TRNSYS

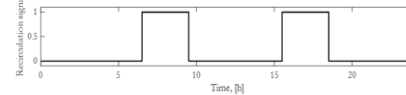
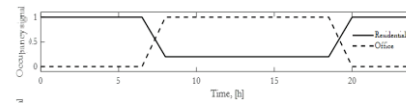
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Simulation cases study in Stockholm climate equipped with centralized air handling unit (2000 m² vent. area)

- Residential building
 - 0.45 ACH
 - Occupancy schedule



- Office building
 - 2.1 ACH
 - Occupancy schedule
 - Ventilation schedule



4



Air pollutant	TVOC			CO ₂	
	Outdoor	Occupants	Interior furnishing	Outdoor	Occupants
Source					
Value	$\mu\text{gr.m}^{-3}$	$\text{mgr.h}^{-1}\text{.person}^{-1}$	$\mu\text{gr.m}^{-3}\text{h}^{-1}$	mgr.m^{-3}	$\text{gr.h}^{-1}\text{.person}^{-1}$
	110	6.3	120	720	120

Indoor and outdoor emission rates

Location	Reference	TVOC concentration $\mu\text{g.m}^{-3}$
Europe	Report EUR 14449 EN. 1992	Comfort range < 300 Multifactorial exposure range < 3000 Discomfort range < 25000 Toxic range > 25000
Finland	Finnish Society of IAQ and Climate. 2000	Individual indoor climate < 200 Good indoor climate < 300 Satisfactory indoor climate < 600 Hygienically safe < 1000
Germany	Federal Environment Agency of Germany	Hygienically noticeable < 3000 Hygienically alarming < 10000 Hygienically unacceptable > 10000
Germany	Seifert B.	300

Guideline values for indoor TVOC concentration

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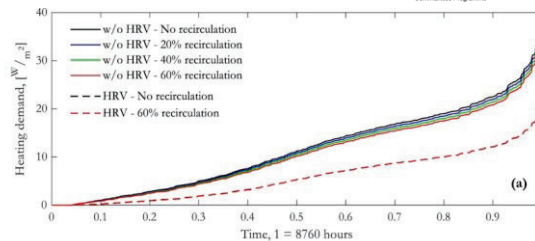


Residential building (0.45 ACH)



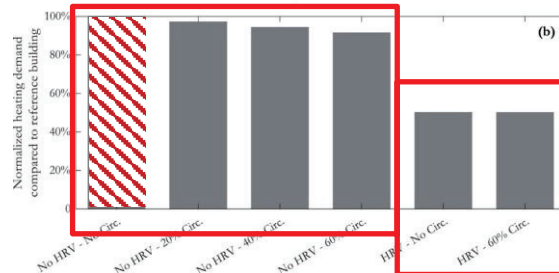
Ventilation with heat recovery:

- The recirculation effect on heating demand is negligible!
- Air cleaner implementation might not be that effective!



Ventilation without heat recovery:

- The recirculation effect on heating demand is small!
- Air cleaner implementation might reduce building heating demand!



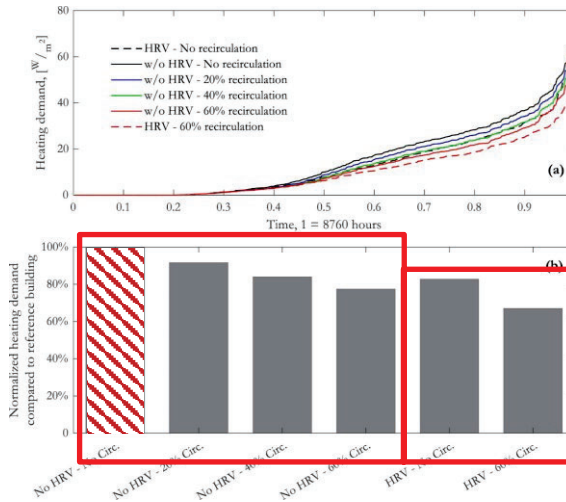
HRV: Heat Recovery Ventilation

6

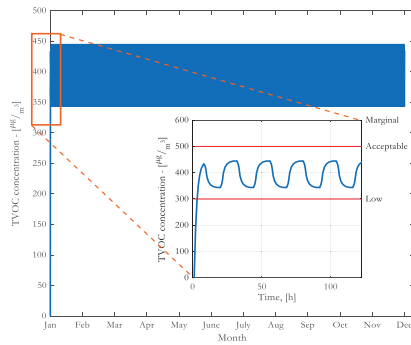
Ventilation with heat recovery:

- The recirculation effect on heating demand is notable compared to the residential buildings!
- This is the case for both with and without heat recovery!
- Air cleaner implementation is effective!

Thus ACH is an important parameter that needs to be considered.

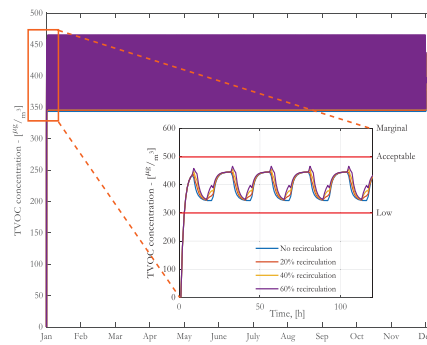


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Without air recirculation

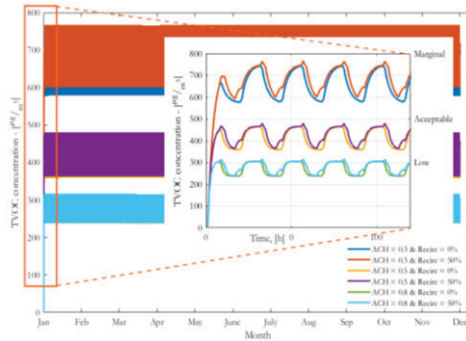
TVOC concentration is within the acceptable range



With air recirculation (and air cleaner)

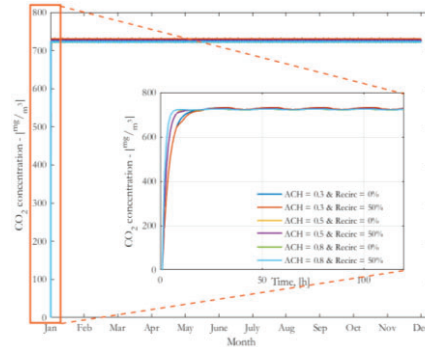
Recirculation does not result in increased TVOC level

8



TVOC concentration with 0 and 50% air recirculation

- High ACH (>0.5) maintains TVOC concentration within an acceptable range, regardless of recirculation level
- Thus, adding air cleaner and recirculation is beneficial to reduce building heating demand



CO₂ concentration with 0 and 50% air recirculation

Recirculation % and ACH do not change CO₂ level since the main CO₂ source is the outdoor air.

Conclusion:

- This study examines the effect of **gas-phase air cleaners on building heating demand**.
- The study also explores indoor **concentrations of TVOC and CO₂** when gas-phase air cleaners are used.
- Different parameters were also discussed, such as **ACH, air recirculation, ventilation, and occupancy schedule** on indoor TVOC and CO₂ levels.
- Increasing recirculation rate **reduced heating demand** in the office building more than in residential.
- 60% recirculation rate reduced heating demand by **9% in residential** and **24% in the office building**.
- Integrating gas-phase air cleaner and increasing recirculation rate during rush hours of mornings and evenings kept TVOC and CO₂ concentrations acceptable.
- Indoor CO₂ concentration value was affected less than TVOC's by increasing the recirculation rate.
- Higher ACH minimizes the impact of recirculation rate on TVOC and CO₂ levels.



Thank you!



1



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Gas phase air cleaning effects on ventilation energy use and indicators for energy performance

2

Outline

1. Introduction to gas-phase air cleaning
2. Clean Air Delivery Ratio (CADR)
3. Energy use
4. Clean Air Efficiency (CAE)
5. Conclusion

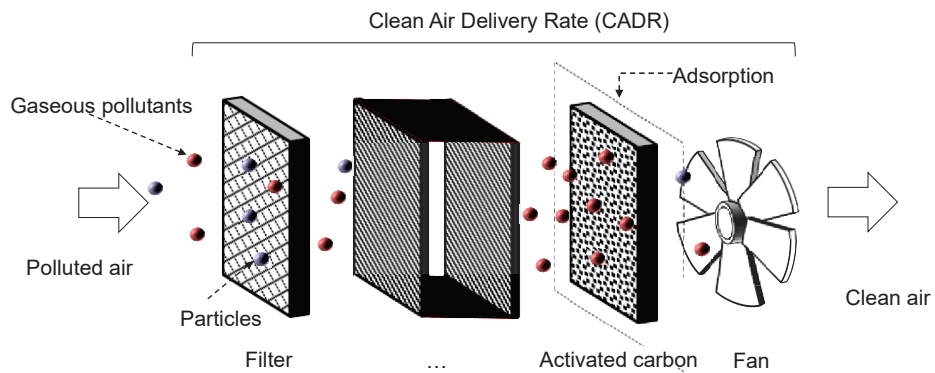
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Gas-phase air cleaning (GPAC)

- Equipped with an activated carbon filter
- Removes gaseous pollutants (e.g. benzene, toluene, xylene) & odour
- Cannot remove all pollutants

4

Gas-phase air cleaning (GPAC)



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Gas-phase air cleaning (GPAC)

- **Reduce occupant dissatisfaction** (same outdoor air flow rate)
- **Reduce outdoor air flow rate** (same occupant satisfaction)

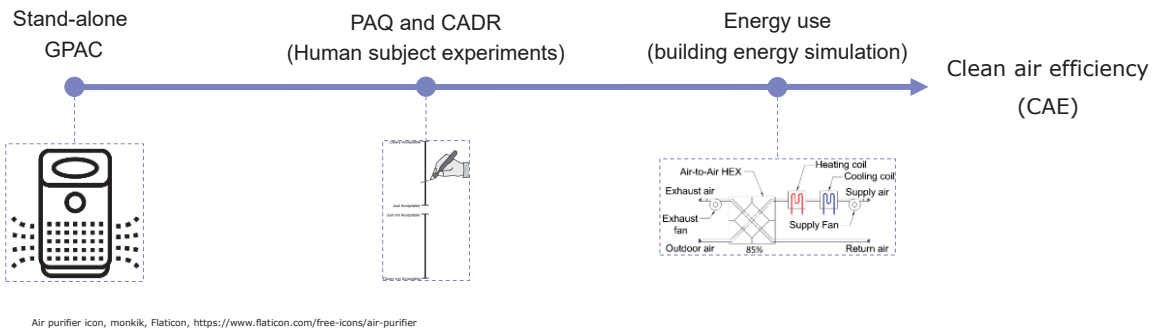
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Methods

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Framework

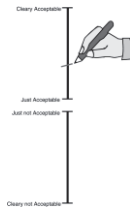
- Assess the air cleaner's effect on energy use
- Find effect relative to other methods



8

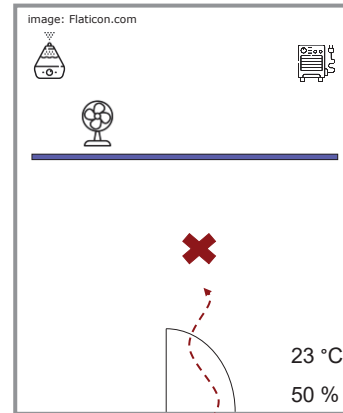
PAQ & CADR

- Human subject experiments



- Estimate difference in q for which the same percentage dissatisfied is achieved

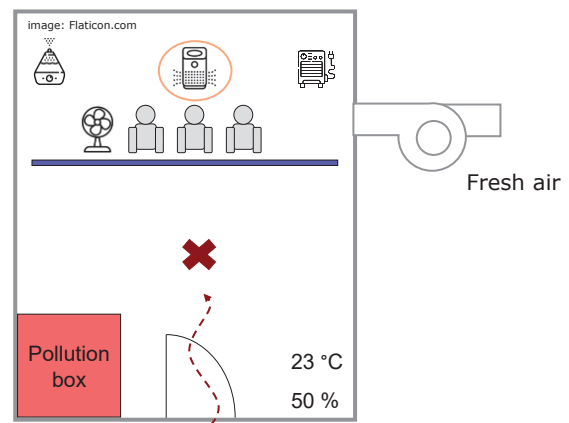
$$CADR = \frac{q - q_{GPAC}}{q} \cdot 100 \text{ [%]}$$



37 subjects (23M + 14F, 25 y)

PAQ & CADR

- Pollution:
 - Building emissions
 - Building emissions & bio-effluents
- Conditions:
 - Without GPAC (REF)
 - With GPAC
- Outdoor air supply:
 - 2.5 / 4.0 / 7.0 / 10 L/s per person



37 subjects (23M + 14F, 25 y)

Building energy simulation

- Determining **energy use**, GPAC **not physically implemented**.
- Implementation by **adjusting air flow rate**:

$$q_{tot} = n \cdot q_p + A_R \cdot q_b$$

- CADR_s:

- **0%, 30%, 50%** of q_{tot} :

$$q = (1 - CADR) \cdot q_{tot}$$

- **50%** of q_b :

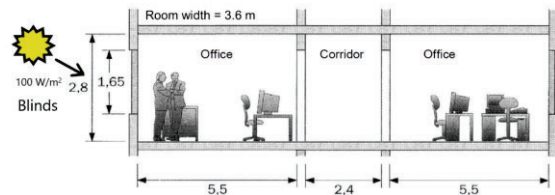
$$q = n \cdot q_p + (1 - CADR) \cdot A_R \cdot q_b$$

EN 16798-1:2019, Category II (20% diss.)

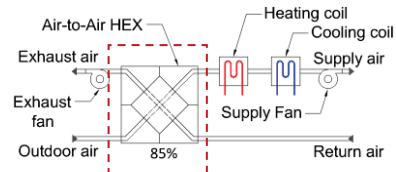
11

Building energy simulation

- Copenhagen, DK
- $20\text{ }^\circ\text{C} < T_{OP,SP} < 26\text{ }^\circ\text{C}$
- Ventilation: CAV
- Scenarios
 - with and without HEX
 - 2x pollution levels (VLP and LP)



Source: Olesen and Dossi, 2004



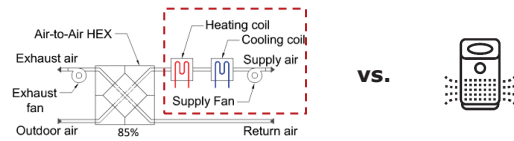
Bogatu et al. "Gas-Phase Air Cleaning Effects on Ventilation Energy Use and the Implications of CO₂ Concentration as an IAQ Indicator for Ventilation Control.", Proceedings of Building Simulation 2021, 2021.

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Clean Air Efficiency (CAE)

Indicator for comparing the efficiency of the AHU and stand-alone air cleaner

$$CAE = \frac{CADR}{Energy\ use} \quad [L/s\ per\ kWh]$$



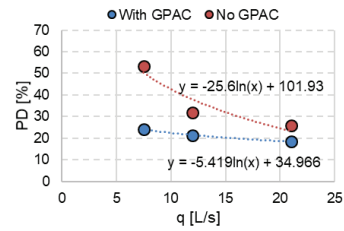
Amount of air, **CADR in L/s**, and energy use for **heating, cooling, and AUX** (transporting air) or **GPAC [kWh]**

image: Flaticon.com

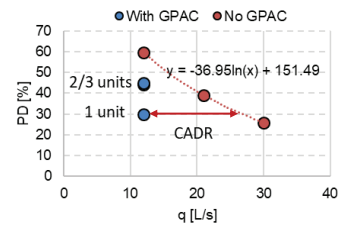
Results

PAQ & CADR

- With GPAC, dissatisfaction rate **reduced** for the same outdoor air flow rate
- With GPAC, outdoor air flow rate can be **reduced** for the same PD
- Increasing n_{GPAC} did not improve PAQ
- CADR:
 - a) 50% (12 L/s)
 - b) 30% (9 L/s)



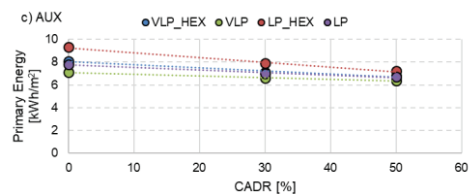
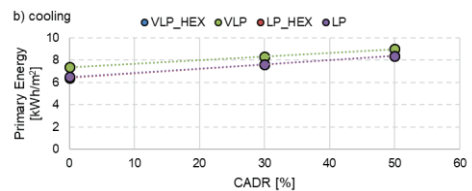
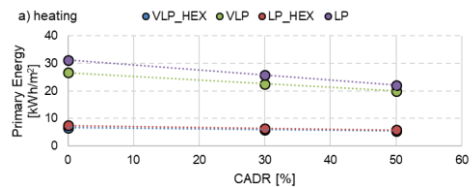
a) building emissions only



b) bio-effluents and building emissions

Energy use

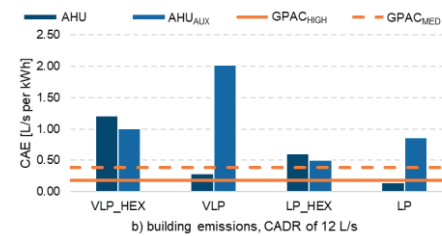
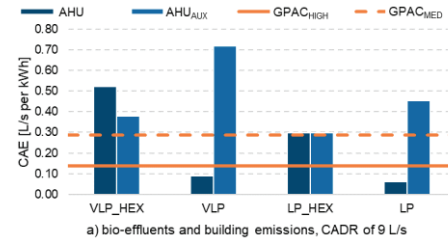
- Linear relationship as a function of CADR
- Heating and AUX decreased and cooling increased as CADR increased
- Heating decreased but AUX increased when the AHU was equipped with a HEX



Bogatu et al. "Gas-Phase Air Cleaning Effects on Ventilation Energy Use and the Implications of CO2 Concentration as an IAQ Indicator for Ventilation Control.", Proceedings of Building Simulation 2021, 2021.

Clean Air Efficiency (CAE)

- If the GPAC is compared only to AHU_{AUX} , the GPAC is never efficient
- If a HEX is included the GPAC is not efficient
- Higher savings can be achieved if GPAC can be operated at a setting lower than HIGH (22 W)



Bogatu et al. "Gas-Phase Air Cleaning Effects on Ventilation Energy Use and the Implications of CO₂ Concentration as an IAQ Indicator for Ventilation Control.", Proceedings of Building Simulation 2021, 2021.

Conclusions

- CAE can be used to compare different solutions for providing clean air into the space.
- In Copenhagen, DK (high heating load), GPAC was competitive only if the AHU was not equipped with a HEX.
- GPAC more efficient when it removed both bio-effluents and building emissions.
- GPAC can be used to either improve IAQ or reduce air flow rate



Contact: Dragos-loan Bogatu
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project No.: 64018-0599

Special thanks to:

- Kanta Amada
- Pawel Wargocki
- Ongun B. Kazanci
- Zhengchai Hu
- Lei Fang
- Bjarne W. Olesen



INTRODUCTION TO THE PROJECT OF ISO 9972 REVISION

Topical Session, 43rd AIVC Conference, Copenhagen (DK) – 04.10.2023

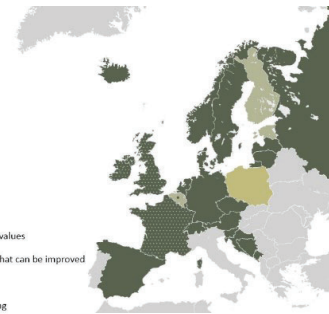
Valérie Leprince

1

AIRTIGHTNESS REGULATIONS IN EUROPE

- Increasing number of tests performed in Europe
- Testing → important part in national energy regulations
- Test is used for :
 - **Measuring air leakage** in buildings to fulfill energy performance standards
 - **Comparing relative airtightness** of buildings
 - **Determining reduction** or air permeability after implementation of improvements

- No regulations
- Airtightness recommended values
- Airtightness default values that can be improved
- Airtightness requirements
- Mandatory systematic testing



Poza-Casado et al. (2020)

2

ISO 9972: FAN PRESURIZATION METHOD

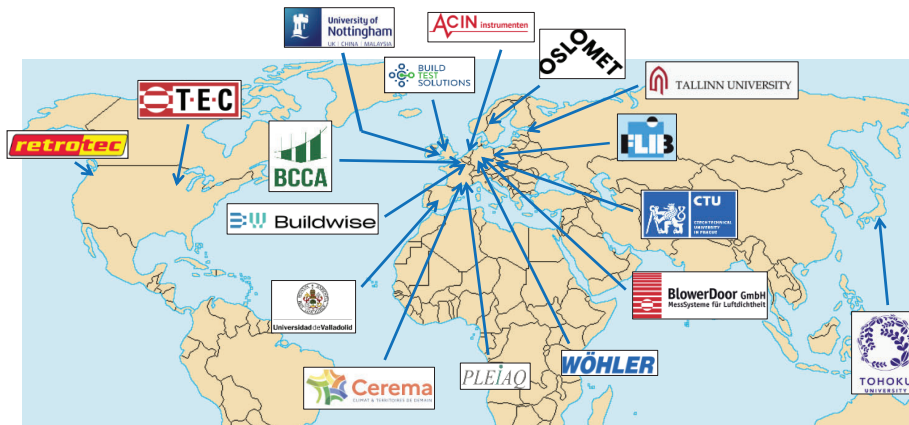
- Describes measurement procedure and calculation methods for determining airtightness
- To obtain comparable and credible results, it needs to be
 - **Reliable** and valid for different kinds of buildings
 - **Reproducible** under challenging environmental conditions
 - **Applicable** in any conditions
 - **Consistent** with other standards
- Recent scientific works + more experience in field testing → **need to improve parts of ISO 9972!**



EUROPEAN STANDARD NORME EUROPÉENNE EUROPÄISCHE NORM	EN ISO 9972 September 2015
ICS: 91.120.10	Supersedes EN 13829:2005
English Version	
Thermal performance of buildings — Determination of air permeability of buildings — Fan pressurization method (ISO 9972:2015)	
Performance thermique des bâtiments — Détermination de la perméabilité à l'air des bâtiments — Méthode de pressurisation par ventilateur (ISO 9972:2015)	
Wärmetechnisches Verhalten von Gebäuden — Bestimmung der Luftdurchlässigkeit von Gebäuden — Differenzdruckverfahren (ISO 9972:2015)	

Revision of ISO 9972

WORKING GROUP AFFILIATIONS



Revision of ISO 9972

REASONS BEHIND A NECESSARY REVISION

Limitations on measurement **reliability**

- Building preparation
- Wind speed and temperature measurements
- Placement of external pressure taps
- Duration of pressure/airflow measurements
- Induced pressure differences
- Type of regression

Work has not started yet
Proposal finished
Work started – more research needed
Work on proposal has started
Work has not started yet
Work on proposal has started

REASONS BEHIND A NECESSARY REVISION

Limitations on measurement **validity**

- Airflow corrections
- Calculation of building volume and area
- Limits on zero-flow pressure measurements
- Knowledge of uncertainty
 - Errors due to measurement instruments, measurement protocol and analysis
 - Errors arising from physical model assumptions

Proposal finished
Work started – more research needed
Work on proposal has started
Proposal finished
Work has not started yet

REASONS BEHIND A NECESSARY REVISION

Limitations on measurement **validity**

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- Calculation of building volume and area → Bassam Moujalled
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REASONS BEHIND A NECESSARY REVISION

Limitations on measurement **validity**

- Airflow corrections
- Calculation of building volume and area
- Limits on zero-flow pressure measurements → Benedikt Kölsch
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REASONS BEHIND A NECESSARY REVISION

Limitations on measurement **validity**

- Airflow corrections
- Calculation of building volume and area
- Limits on zero-flow pressure measurements
- Knowledge of uncertainty
- Errors due to measurement instruments, measurement protocol and analysis
- Errors arising from physical model assumptions

→ Martin Prignon

OUTLOOK

Mid-2023:
Proposition of new versions for issues ready to revise

End 2023:
Proposition of new versions with conducted research

From 2024:
Revision on ISO level

On 2023 September 28th
ISO TC163, SC1 Committee agreed to launch the revision of ISO 9972!

Resolution 347 (2/2023, Atlanta 2023-09-28)
ISO/TC 163/SC 1 agrees to create an Ad-Hoc group "Air tightness of buildings" to develop the preliminary work item ISO 9972, *Thermal performance of buildings — Determination of air permeability of buildings — Fan pressurization method*. Mr. Utsunomiya is appointed as the Ad-Hoc group leader.
P-members approving: 12
P-members disapproving: 0
P-members abstains: 0
The decision was *unanimous*



Uncertainties in fan pressurization measurement

Integration of envelope pressure inhomogeneity and autocorrelation aspects



Dr. Ir. Martin Prignon,
Buildwise
martin.prignon@buildwise.be

43rd AIVC conference "Ventilation, IEQ and health in sustainable buildings"

1

- Context
- Novelty 1: Autocorrelation
- Novelty 2: Inhomogeneity
- Analysis of global impact
- Conclusion

2

- Context
- Novelty 1: Autocorrelation
- Novelty 2: Inhomogeneity
- Analysis of global impact
- Conclusion

Uncertainty estimation is crucial for airtightness testing

*“without such an indication, measurement results **cannot be compared**, either among themselves or with reference values given in a specification or **standard**”*

(JCGM, 2008)

Airtightness:

- High importance (standards)
- Often overlooked in research and practice
- Literature shows large gap between estimated and real uncertainties

Objective:

Improve the uncertainty estimation by integrating two aspects in the process

Quick reminder: fan pressurisation test protocol



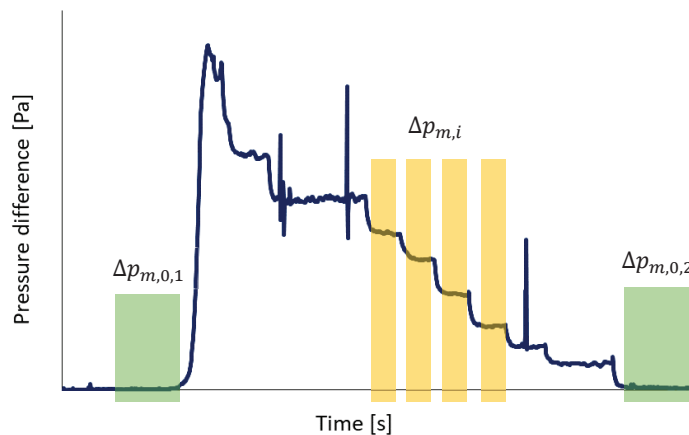
- 1 Measurement of $\Delta p_{m,i,k}$; $q_{m,i,k}$; T_i and T_e
- 2 Determination of Δp_i ; q_i couples
- 3 Evaluation of the linear model fitting x_i ; y_i couples with $x_i = \ln(\Delta p_i)$ and $y_i = \ln(q_i)$
- 4 Determination of building characteristics: n ; $\ln(c)$; q_{50} ; n_{50} ; ELA_4 ; etc.

Quick reminder of the five steps to determine Δp_i

- 1 Test: measurement of $\Delta p_{m,i,k}$
- 2 Zero-flow pressure:

$$\Delta p_{0m} = \frac{\Delta p_{0m,1} + \Delta p_{0m,2}}{2}$$
- 3 At each pressure station:

$$\Delta p_{m,i} = \frac{1}{N} * \sum_{k=1}^N \Delta p_{m,i,k}$$
- 4 $\Delta p_i = \Delta p_{m,i} - \Delta p_{0m}$

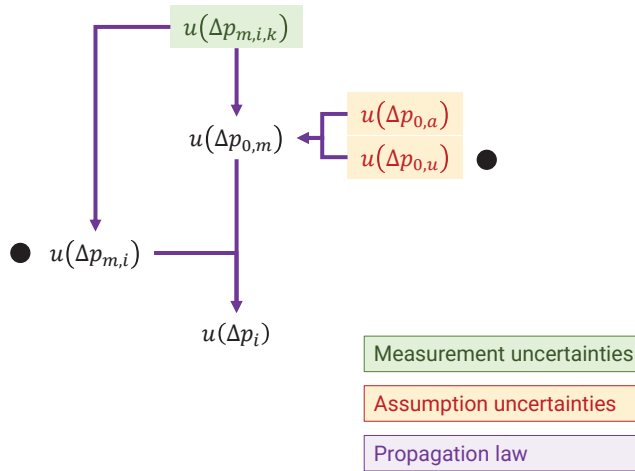


Two novelties in the uncertainty estimation process

- 1 Test: measurement of $\Delta p_{m,i,k}$
- 2 Zero-flow pressure:

$$\Delta p_{0m} = \frac{\Delta p_{0m,1} + \Delta p_{0m,2}}{2}$$
- 3 At each pressure station:

$$\Delta p_{m,i} = \frac{1}{N} * \sum_{k=1}^N \Delta p_{m,i,k}$$
- 4 $\Delta p_i = \Delta p_{m,i} - \Delta p_{0m}$



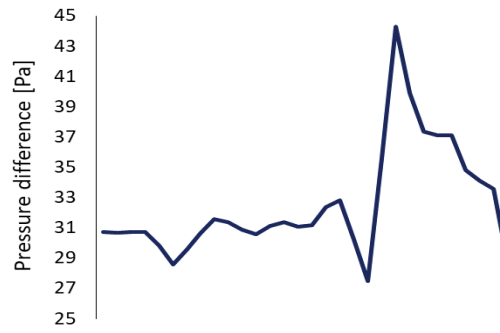
7

- Context
- Novelty 1: Autocorrelation
- Novelty 2: Inhomogeneity
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8

Pressure difference measurements seem autocorrelated

Autocorrelation is when a measurement made at a time t depends on a series of measurements made before this one.

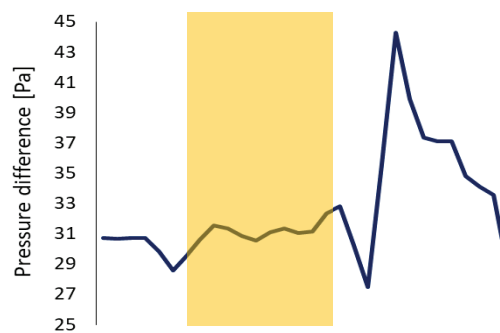


Autocorrelation has an impact on uncertainty calculation

$$\Delta p_{i,m} = \frac{1}{N} * \sum_{k=1}^N \Delta p_{i,m,k}$$

$$u(\Delta p_{i,m}) = \frac{u(\Delta p_{i,m,k})}{\gamma}$$

- $\gamma = 1$ Fully correlated (today)
- $\gamma = N$ Fully uncorrelated (here $N = 30$)
- $\gamma = N_{eff}$ Autocorrelated (suggestion)



Repeatability study to determine the level of autocorrelation (N_{eff}) in pressure measurements

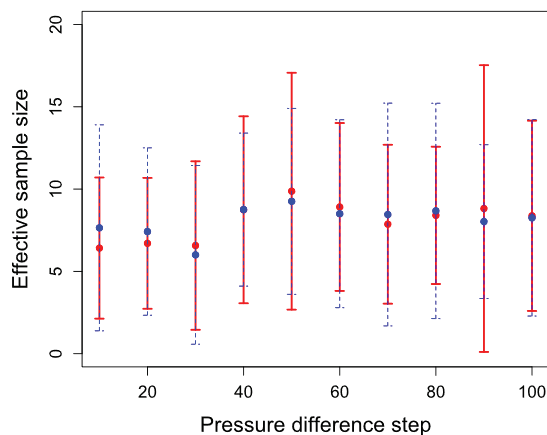
Repeatability study

Total: 30 tests
Case: Appartement in Brussels
Prignon et al., 2020

Calculation of N_{eff}

- For each test
- For each mode (P+ and P-)
- At each pressure station (Δp_i)

Results show consequent autocorrelation of the data



➤ No clear difference between

- P+ and P-
- Different Δp_i

➤ $\mu(N_{eff}) = 8.1$ and $\sigma(N_{eff}) = 3.0$

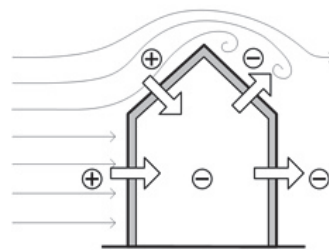
- Context
- Novelty 1: Autocorrelation
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- Conclusion

Inhomogeneity of pressure difference along the building envelope induces some uncertainty

Distribution of airflows among leaks depends on Δp distribution.

→ Inhomogeneity represents a source of uncertainty

In this study inhomogeneity is assumed coming only from wind pressure (low-rise building)



Inhomogeneity is included as an assumption uncertainty for the zero-flow pressure

Temperature inhomogeneity for climatic chambers,

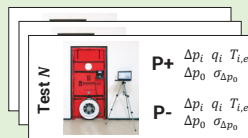
$$u(T) = \frac{\Delta T_{max}}{\sqrt{3}}$$

For wind-derived inhomogeneity,

$$u(\Delta p_{0,u}) = \frac{|c_{p,max} - c_{p,min}| * v_w^2 * \rho}{2 * \sqrt{3}}$$

Repeatability study to quantify uncertainty assumption related to inhomogeneity

Repeatability study



Total: 30 tests
Case: Appartement in Brussels
Prignon et al., 2020

→ Calculation of $u(\Delta p_{0,u})$ for each test
Comparison with $u(\Delta p_{0,a})$

Uncertainty source	Average [Pa]	95% CI [Pa]
$u(\Delta p_{0,a})$	1.1	[0.0 ; 2.1]
$u(\Delta p_{0,u})$	1.6	[0.0 ; 4.2]

- Same order of magnitude than $u(\Delta p_{0,a})$
- Large impact at low pressure station

- Context
- Novelty 1: Autocorrelation
- Novelty 2: Inhomogeneity
- Analysis of global impact
- Conclusion

Repeatability study to study the impact of two novelties on the whole uncertainty estimation process

Repeatability study

Total: 30 tests
Case: Appartement in Brussels
Prignon et al., 2020

Full post-processing up to the determination of q_{50}

	$u(\Delta p_{m,i,k})$	$u(\Delta p_{0,a})$	$u(\Delta p_{0,u})$	N_{eff}
Case 1	•			
Case 2	•	•		
Case 3	•	•	•	
Case 4	•	•	•	•

Results in brief

- Considering inhomogeneity without autocorrelation leads to an overestimation of calculated uncertainties for n and $\ln(c)$, but a good estimation for q_{50}
- Considering inhomogeneity and autocorrelation reduces uncertainty and improves uncertainty estimation of n and $\ln(c)$

- Context
- Novelty 1: Autocorrelation
- Novelty 2: Inhomogeneity
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- Conclusion

Interesting results, but investigation still important






Conclusion

Relevant additions, but still not enough to explain the observed uncertainty

Further work

- Confirm the trends
- Adapt for high-rise building (stack effect)
- Further quantification of precision errors
- Add biases in the evaluation process

**Thank you for
your attention**

STATISTICAL ANALYSIS OF THE CORRELATIONS BETWEEN BUILDINGS AIR PERMEABILITY INDICATORS

Bassam Moujalled, Bnedikt Kölsch, Adeline Mélois, Valérie Leprince

Cerema & Université Savoie Mont Blanc / LOCIE, France

October 4, 2023

1

CONTEXT OF BUILDING AIR TIGHTNESS DATABASES

Development of building air tightness tests in several European countries thanks to the mandatory requirements



- **Almost all newly built residential buildings are tested in France, Belgium and UK**

Creation of building airtightness databases in these countries

- **Databases with more than 500,000 tests in France and more than 1,000,000 tests in UK**

Comparing building performance across different countries is challenging

- **Various indicators used to measure the air permeability of buildings**

43rd AIVC Conference | October 4th 2023

2

OBJECTIVES

Comparison of the air permeability indicator between different countries

- Identify the indicators and their calculation methods in different countries

Establish default conversion equations between the main indicators based on the statistical analysis of the French database:

- Analysis of geometric parameters
- Correlations between air permeability indicators

3

AIR PERMEABILITY INDICATORS

Air permeability indicators in ISO 9972:

- **Specific air leakage rate $q_{E\Delta P}$ [$m^3 \cdot h^{-1} \cdot m^{-2}$]**: airflow rate $q_{\Delta P}$ [$m^3 \cdot h^{-1}$] at a specific pressure difference ΔP [Pa] divided by the building envelope area A_E [m^2]
- **Air change rate $n_{\Delta P}$ in [h^{-1}]**: airflow rate $q_{\Delta P}$ [$m^3 \cdot h^{-1}$] at a specific pressure difference ΔP [Pa] divided by the building volume V [m^3]
- **Air permeability rate $q_{F\Delta P}$ [$m^3 \cdot h^{-1} \cdot m^{-2}$]**: airflow rate $q_{\Delta P}$ [$m^3 \cdot h^{-1}$] at a specific pressure difference ΔP [Pa] divided by the net floor area A_F [m^2]

4

COMPARISON OF AIR PERMEABILITY INDICATORS

The definition of air permeability indicator differs according on the **reference pressure difference** and the **geometric parameter** used in each country

Country	Indicator	Definition	Calculation
France	$Q_{4Pa-surf}$ [m ³ .h ⁻¹ .m ⁻²]	Specific air leakage rate at 4 Pa divided by heat loss area excluding the basement floor	q_4 : air leakage rate at 4 Pa [m ³ .h ⁻¹] A_{TBAT} : thermal envelope area excluding the basement floor [m²]
Germany	n_{L50} [h ⁻¹]	Air change rate at 50 Pa	q_{50} : air leakage rate at 50 Pa [m ³ .h ⁻¹] V_i : internal air volume [m³]
Belgium	V_{50} [m ³ .h ⁻¹ .m ⁻²]	Specific air leakage rate at 50 Pa divided by heat loss area	q_{50} : air leakage rate at 50 Pa [m ³ .h ⁻¹] A_{test} : thermal envelope area [m²]
UK	AP_{50} [m ³ .h ⁻¹ .m ⁻²]	Specific air leakage rate at 50 Pa divided by the internal envelope area	Q_{50} : air leakage rate at 50 Pa [m ³ .h ⁻¹] A_e : envelope area [m²]
Netherlands	q_{v10} [m ³ .s ⁻¹]	Volumetric air flow at 10 Pa	q_{v10} : volumetric air flow at 10 Pa [m ³ .h ⁻¹]

5

COMPARISON OF AIR PERMEABILITY INDICATORS

Envelope area calculation

- **ISO 9972**: building envelope area including floor areas and junctions of internal walls (internal dimensions)
- **France**: Thermal loss envelope area A_{TE} in contact with the exterior environment/adjacent unheated spaces/ground, excluding lower floor area (internal dimensions)
- **Belgium**: Thermal loss envelope area A_{TE} in contact with the exterior environment/adjacent unheated spaces/ground (external dimensions)

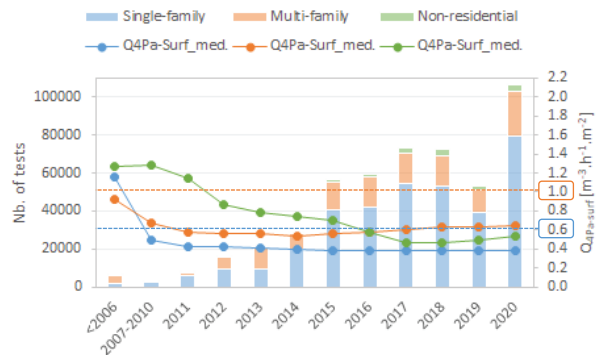
Building volume calculation

- **ISO 9972**: internal building volume without subtracting the volume of walls, floors, cavities or furniture
- **Germany**: net room volume as a product from the net room area and the middle of the room height (heated or cooled spaces)

6

DATA IN THE FRENCH DATABASE

- About 570,000 measurements in the French database
- A set of 406,717 tests is selected for the statistical analysis
 - 324,859 tests in Single-family houses (SF)
 - 71,255 tests in Multi-Family apartments (MF)
 - 10,603 tests in non-residential buildings (NR)
- Air permeability levels :
 - SF : Median $Q_{4Pa-surf} \sim 0.4 \text{ m}^3 \cdot \text{h}^{-1} \cdot \text{m}^{-2}$ ($n_{50} \sim 1.7 \text{ h}^{-1}$)
 - MF : Median $Q_{4Pa-surf} \sim 0.6 \text{ m}^3 \cdot \text{h}^{-1} \cdot \text{m}^{-2}$ ($n_{50} \sim 1.4 \text{ h}^{-1}$)
 - NR : median $Q_{4Pa-surf} \sim 0.5 \text{ m}^3 \cdot \text{h}^{-1} \cdot \text{m}^{-2}$ ($n_{50} \sim 1.9 \text{ h}^{-1}$)

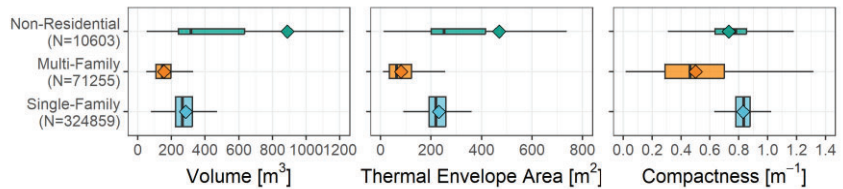


7

ANALYSIS OF GEOMETRIC PARAMETERS

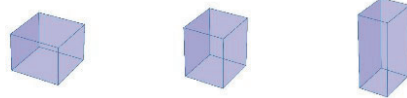
Geometric parameters

- Thermal loss envelope area except for lower floor A_{TE}
- Internal volume V
- Compactness factor = A_{TE}/V
 - *The lower this factor, the more compact the building*



Compactness < 0.8 m⁻¹ Compactness = 0.80 m⁻¹ Compactness = 0.85 m⁻¹ Compactness = 1.00 m⁻¹

Houses with one or more adjacent walls



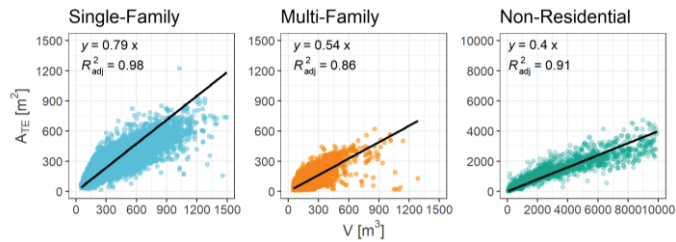
8

ANALYSIS OF GEOMETRIC PARAMETERS

Strong linear correlations are observed between A_{TE} and V

The slopes of the linear regression are 0.79, 0.54 and 0.4 for SF, MF, and NR

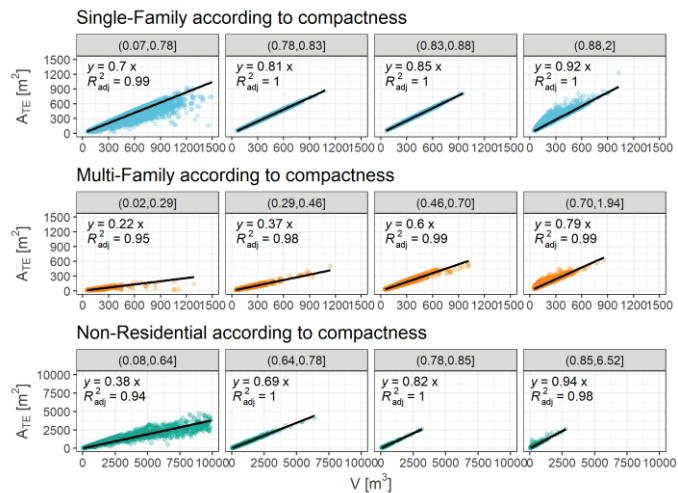
- Average Compactness
- Buildings with lower or higher values of the compactness factor deviate significantly from the regression line



ANALYSIS OF GEOMETRIC PARAMETERS

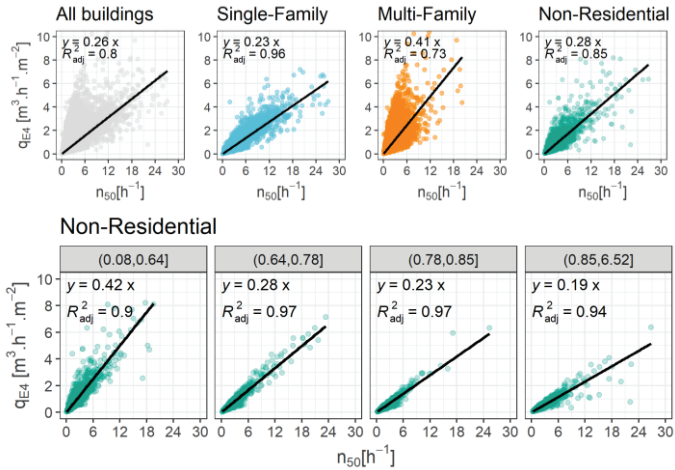
Correlations between the thermal envelope area and the internal volume according to the compactness

- Four classes of compactness corresponding to the quartiles of the compactness factor
- Very strong linear correlations
- Compactness factor appears to be thus a relevant geometric parameter for characterising the relationship between the air permeability indicators



CORRELATIONS BETWEEN AIR PERMEABILITY INDICATORS

- Correlations between five indicators (specific air leakage rates q_{E4} / q_{E10} / q_{E50} and air change rates n_{10} / n_{50}) have been calculated
- Correlation between specific leakage rate q_{E4} and air flow rate n_{50} depends on:
 - the building type
 - The building geometry: The more compact the building (i.e., the smaller the compactness factor), the greater the slope of the regression line



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CORRELATIONS BETWEEN AIR PERMEABILITY INDICATORS

- Correlations between q_{E4} and n_{50} depending on building type and building compactness

Correlation	Building type	Correlation depending on building type and building compactness			Correlation depending on building type		General correlation	
		Compact.	Reg. coef.	Conf. Int. 95%	Reg. coef.	Conf. Int. 95%	Reg. coef.	Conf. Int. 95%
$q_{E4} =$ Coef * n_{50}	Single-Family houses	(0.07,0.78]	0.264 ($r^2=0.965$)	[0.264,0.265]	0.228 ($r^2=0.964$)	[0.228,0.228]	0.26 ($r^2=0.801$)	[0.26,0.261]
		(0.78,0.83]	0.23 ($r^2=0.983$)	[0.23,0.231]				
		(0.83,0.88]	0.219 ($r^2=0.984$)	[0.218,0.219]				
		(0.88,2]	0.2 ($r^2=0.977$)	[0.2,0.2]				
	Multi-Family apartments	(0.02,0.29]	0.91 ($r^2=0.929$)	[0.906,0.914]	0.409 ($r^2=0.728$)	[0.407,0.41]		
		(0.29,0.46]	0.517 ($r^2=0.958$)	[0.516,0.519]				
		(0.46,0.70]	0.33 ($r^2=0.966$)	[0.329,0.331]				
		(0.70,1.94]	0.237 ($r^2=0.964$)	[0.236,0.238]				
	Non-Residential buildings	(0.08,0.64]	0.418 ($r^2=0.9$)	[0.412,0.423]	0.284 ($r^2=0.851$)	[0.282,0.287]		
		(0.64,0.78]	0.276 ($r^2=0.971$)	[0.274,0.278]				
		(0.78,0.85]	0.233 ($r^2=0.973$)	[0.231,0.234]				
		(0.85,6.52]	0.192 ($r^2=0.936$)	[0.19,0.194]				

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CONCLUSIONS

- Various air permeability indicators used across different countries depending on the specific application and nation's regulation
- most common indicators:
 - *Specific air leakage rate per envelope area*
 - *Air change rate*
- Reference pressure differences:
 - *50 Pa in the majority of countries*
 - *Lower values in some countries (e.g. 4 Pa in France, 10 Pa in Netherlands)*
- Calculation of geometric parameters specific for each country
 - *Building envelope area vs. heat loss area (excluding or not some specific area)*
 - *Internal dimensions vs. external dimensions*
 - *Volume with OR without subtracting the volume of walls, floors, cavities or furniture*

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CONCLUSIONS

- The correlations between five indicators (specific air leakage rates q_{E4} / q_{E10} / q_{E50} and air change rates n_{10} / n_{50}) have been calculated according to the building type and compactness
- Strong linear correlations between indicators (correlation coefficients between 0.80 and 0.99)
- Correlations between specific leakage rate and air change rate depend on the building type and geometry
 - *The more compact the building (i.e., the smaller the compactness factor), the greater the slope of the regression line*
 - *If the geometry data is missing, a general correlation can be used with a higher estimation error*
- More analysis are needed with other databases to compare the geometry parameters and make cross analysis between countries

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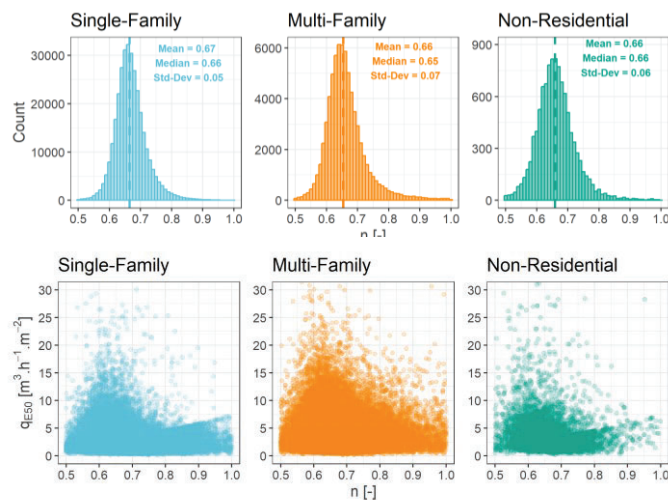
B. Moujalled, B. Kölsch, A. Mélois, V. Leprince, Quantitative correlation between buildings air permeability indicators: Statistical analyses of over 400,000 measurements, *Energy and Buildings*. 298 (2023) 113566. <https://doi.org/10.1016/j.enbuild.2023.113566>

Thank you!

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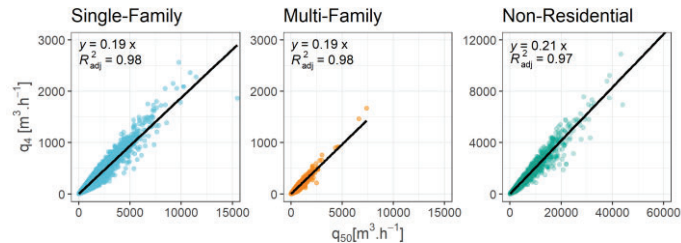
ANALYSIS OF FLOW EXPONENT

- Median and mean of the pressure exponent around 0.66 and a standard deviation of around 0.06
- Whatever the type of building or its geometry
- No correlation between leakage rate and exponent n value
- No tendency of large orifice or specific leaks shape for buildings with a low airtightness level
- No parameter that could influence the pressure exponent has been identified



CORRELATIONS BETWEEN AIRFLOW RATES

- Correlations of the airflow rates at 4 Pa and 10 Pa with the flow rate at 50 Pa
- Very strong linear correlations
- In case of a one-point airtightness test at 50 Pa, these correlations help to estimate the air leakage at 4 Pa or 10 Pa and thus calculate other air permeability indicators, such as q_{E4}



Building type	$q_4 = \text{Coef} * q_{50}$		$q_{10} = \text{Coef} * q_{50}$	
	Reg. Coef.	Conf. Int. 95%	Reg. Coef.	Conf. Int. 95%
Single-Family houses	0.187 ($r^2=0.982$)	[0.187,0.186]	0.343 ($r^2=0.992$)	[0.343, 0.343]
Multi-Family apartments	0.193 ($r^2=0.979$)	[0.193,0.194]	0.350 ($r^2=0.991$)	[0.350, 0.350]
Non-Residential buildings	0.208 ($r^2=0.972$)	[0.207,0.208]	0.366 ($r^2=0.988$)	[0.365,0.367]

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CONCLUSIONS

Statistical analysis of **over 400,000 measurements** from the French database

- The thermal envelope area in the French database excludes the lower floor area !

Compactness factor is a relevant geometric parameter for characterising the relationship between the air permeability indicators

- Around 0.8 m^{-1} for a typical single-family house and non-residential buildings
- Lower values for multi-family apartments, especially for apartments on an intermediate level with only one or two facades (0.2 m^{-1})

18



FLOW ERROR ANALYSIS: ISO 9972 & ZERO-FLOW CONSTRAINTS

A comprehensive study on the influence of steady wind and stack effects

Topical Session, 43rd AIVC Conference, Copenhagen (DK) – 04.10.2023

Benedikt Kölsch, Valérie Leprince, Adeline Mélois, Bassam Moujalled

1

ISO 9972: FAN PRESURIZATION METHOD

EUROPEAN STANDARD
NORME EUROPÉENNE
EUROPÄISCHE NORM

EN ISO 9972

September 2015

ICS: 91.120.10

Supersedes EN 13829:2000

English Version

Thermal performance of buildings —
Determination of air permeability of buildings —
Fan pressurization method
(ISO 9972:2015)

Performance thermique des bâtiments —
Détermination de la perméabilité à l'air des bâtiments —
Méthode de pressurisation par ventilateur
(ISO 9972:2015)

Wärmetechnisches Verhalten von Gebäuden —
Bestimmung der Luftdurchlässigkeit von Gebäuden —
Differenzdruckverfahren
(ISO 9972:2015)

2

ISO CONSTRAINTS

Zero-Flow Pressure Differences

- Δp_0 = Pressure difference between inside and outside when a building is not artificially pressurized
- ISO 9972 defines constraints to limit the influence of wind + temperatures
- Constraints for valid measurements:
 1. $|\Delta p_0| < 5 \text{ Pa}$
 2. Lowest $\Delta p_{st} = 10 \text{ Pa}$ or $5 \times |\Delta p_0|$



3

ISO CONSTRAINTS EXCLUDE MANY BUILDINGS FROM BEING TESTED!

Understanding airflow errors and effectiveness of ISO constraints:

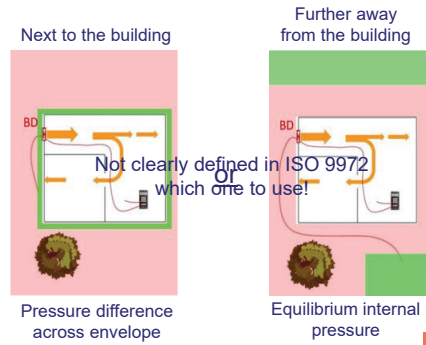
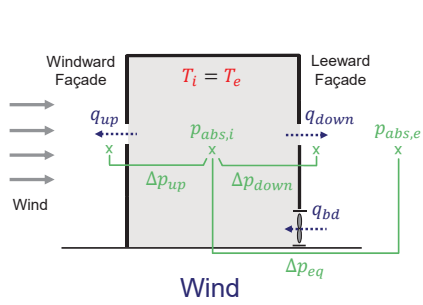
- What is the **error of the measured airflow** induced by Δp_0 ?
- Which **parameters influence Δp_0** in detail?
- Does **pressure tap position** have an influence on Δp_0 ?
- Are there **alternative constraints** that could be applied when the ones in ISO 9972 are impossible to reach (e.g., for windy or high-rise buildings)?



4

OUR APPROACH

Wind model – pressure tap positions



Pressure difference across envelope

Equilibrium internal pressure

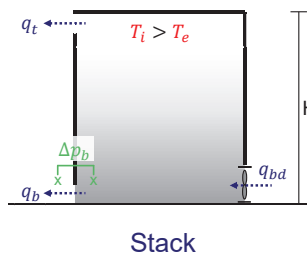
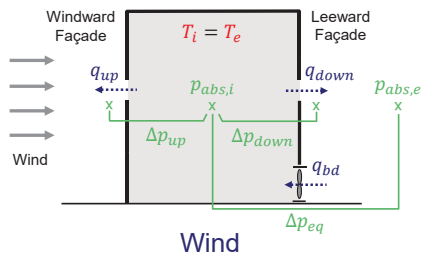
ISO 9972 & Zero-flow pressure

5

5

OUR APPROACH

Wind and stack effect models



Stack

ISO 9972 & Zero-flow pressure

6

6

ISO 9972 AIRFLOW ERROR

Calculation procedure and assumptions in ISO 9972:

Correction of the measured pressure station with Δp_0 :

$$q_{est} = C_{est} (\Delta p_{st} - \Delta p_0)^n$$

Contrary to the assumption in ISO 9972:

$$(C_{up} + C_{down}) (\Delta p_{st} - \Delta p_0)^n \neq C_{up} \Delta p_{up}^n + C_{down} \Delta p_{down}^n$$

$$C_{real} (\Delta p_{st} - \Delta p_0)^n \neq C_{est} (\Delta p_{st} - \Delta p_0)^n$$

Airflow error: $\frac{\delta q}{q} = \frac{C_{est} - C_{real}}{C_{real}}$



7

UNDERSTANDING ZERO-FLOW PRESSURE

Upwind measurement position of Δp_0

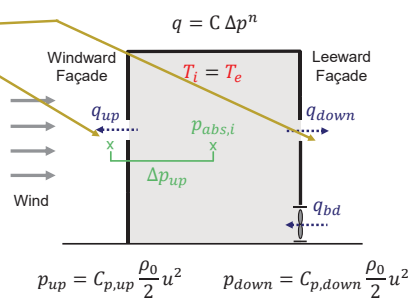
$$\Delta p_{0,up} = - \frac{p_{up} - p_{down}}{1 + 1/z_w^{1/n}}$$

$$z_w = \frac{C_{down}}{C_{up}}$$

$$q = C \Delta p^n$$

$$p_{up} = C_{p,up} \frac{\rho_0}{2} u^2 \quad p_{down} = C_{p,down} \frac{\rho_0}{2} u^2$$

- Always **negative**
- Depends on:
 - Upwind pressure
 - Downwind pressure
 - Pressure exponent
 - Leakage distribution



8

UNDERSTANDING ZERO-FLOW PRESSURE

Upwind measurement position of Δp_0

$$\Delta p_{0,up} = -\frac{p_{up} - p_{down}}{1 + 1/z_w^{1/n}}$$

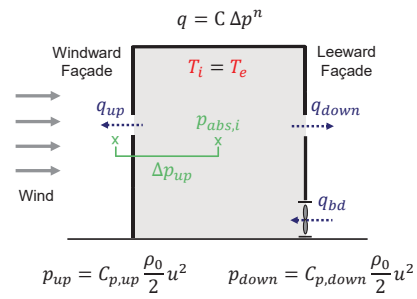
$$z_w = \frac{C_{down}}{C_{up}}$$

$$x_w = \frac{p_{up} - p_{down}}{\Delta p_{st}}$$

Airflow error:

$$\frac{\delta q}{q_{w,up}} = \frac{C_{est,w,up} - C_{real,w}}{C_{real,w}}$$

$$= \frac{1 + z_w(1 + x_w)^n}{(1 + z_w) \left(1 + \frac{x_w}{1 + 1/z_w^{1/n}}\right)^n} - 1$$



UNDERSTANDING ZERO-FLOW PRESSURE

Downwind measurement position of Δp_0

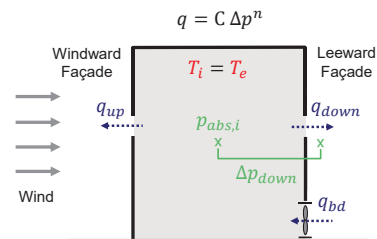
$$\Delta p_{0,down} = \frac{p_{up} - p_{down}}{1 + z_w^{1/n}}$$

$$\Delta p_{0,up} = -\frac{p_{up} - p_{down}}{1 + 1/z_w^{1/n}}$$

➤ Always positive

Airflow error:

$$\frac{\delta q}{q_{w,down}} = \frac{z_w + (1 - x_w)^n}{(1 + z_w) \left(1 - \frac{x_w}{1 + z_w^{1/n}}\right)^n} - 1$$

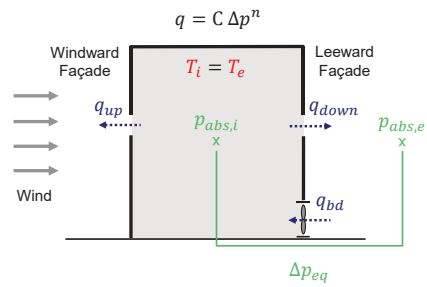


UNDERSTANDING ZERO-FLOW PRESSURE

Equilibrium measurement position of Δp_0

$$\Delta p_{0,w,eq} = \frac{p_{up} + z_w^{1/n} p_{down}}{1 + z_w^{1/n}} \quad z_w = \frac{C_{down}}{C_{up}}$$

- Can be **positive** or **negative** depending on the **leakage distribution!**



11

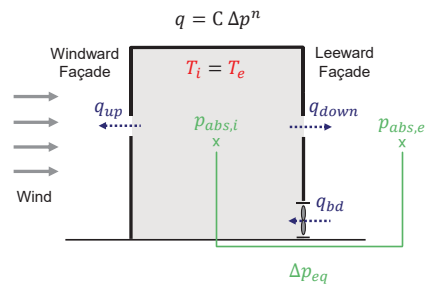
UNDERSTANDING ZERO-FLOW PRESSURE

Equilibrium measurement position of Δp_0

$$\Delta p_{0,w,eq} = \frac{p_{up} + z_w^{1/n} p_{down}}{1 + z_w^{1/n}}$$

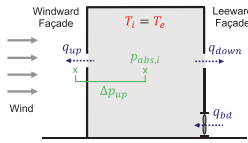
Airflow error:

$$\frac{\delta q}{q_{w,eq}} = \frac{(1 - x_{w,up})^n + z_w(1 - x_{w,down})^n}{(1 + z_w) \left(1 - \frac{x_{w,up} + z_w^{1/n} x_{w,down}}{1 + z_w^{1/n}}\right)} - 1$$



12

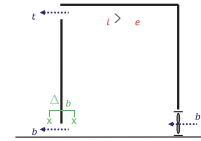
UNDERSTANDING ZERO-FLOW PRESSURE



Wind

$$\frac{\delta q}{q} = \frac{1 + z(1+x)^n}{(1+z) \left(1 + \frac{x}{1 + 1/z^{1/n}} \right)^n - 1}$$

$$\Delta p_{zf} = -\frac{p_{force}}{1 + 1/z^{1/n}}$$



Stack

$$p_{force,w,up} = p_{up} - p_{down}$$

$$x_w = \frac{p_{force,w}}{\Delta p_{st}}$$

$$z_w = \frac{C_{down}}{C_{up}}$$

$$p_{force,s} = p_s \approx \frac{\rho_0}{T_0} g \Delta T H$$

$$x_s = \frac{p_{force,s}}{\Delta p_{st}}$$

$$z_s = \frac{C_t}{C_b}$$

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SIMULATION OF VARIOUS BUILDING SCENARIOS

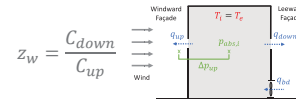
Range of parameters

Input variable	Min. value	Max. value	Distribution characteristics
n	0.5	0.9	Linear distribution in steps of 0.1
$\Delta p_{st,pres/depres}$	± 10 Pa	± 100 Pa	Linear distribution in steps of 10 Pa
u	0 m/s	10 m/s	Linear distribution in steps of 0.1 m/s
z	1/99	99	Logarithmic distribution with 100 values
ΔT	1 K	20 K	Linear distribution in steps of 1 K (always $T_i > T_e$)
H	4 m	100 m	Linear distribution in steps of 1 m
$C_{p,down}$	-	-	[-0.3, -0.5, -0.7]
$C_{p,up}$	-	-	[0.05, 0.25, 0.5]

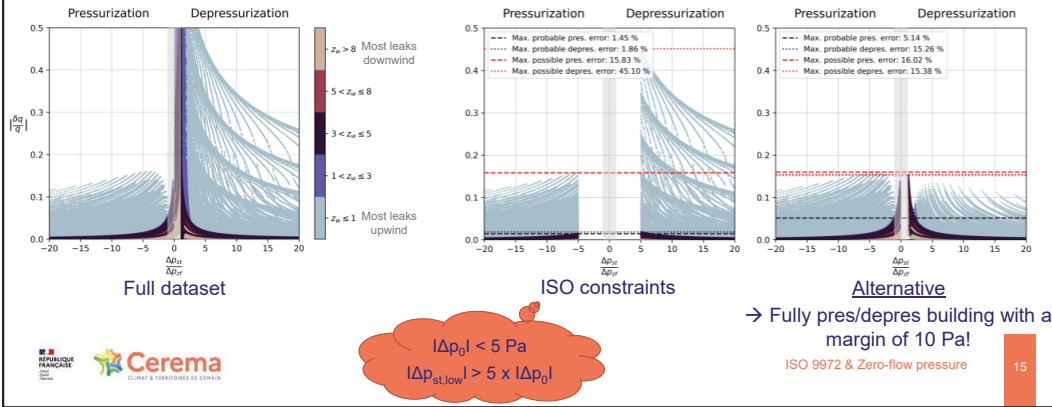
3 million wind and 7 million stack pressure scenarios!

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KEY FINDINGS

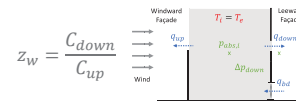


Upwind measurement position of Δp_0

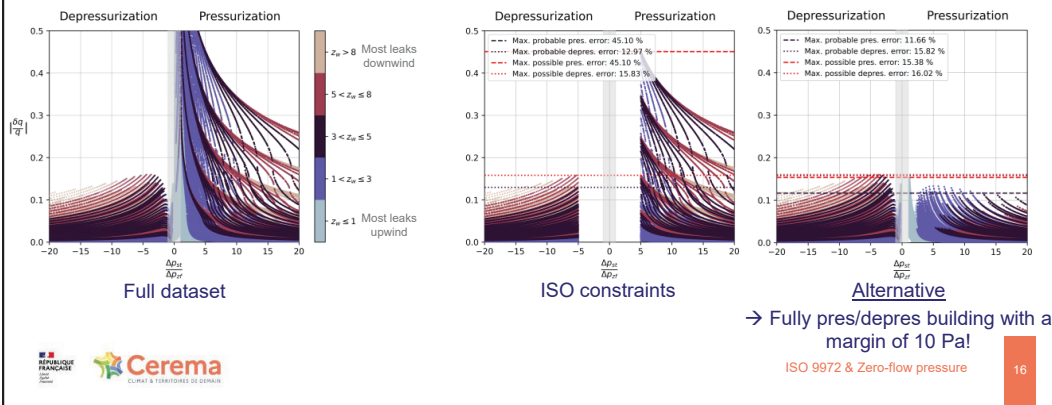


15

KEY FINDINGS

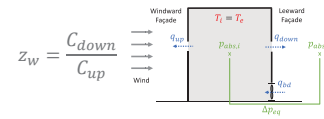


Downwind measurement position of Δp_0

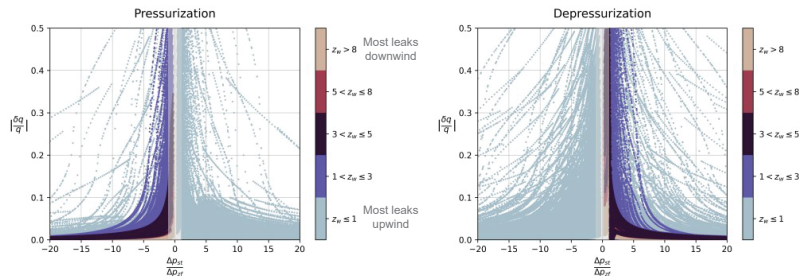


16

KEY FINDINGS

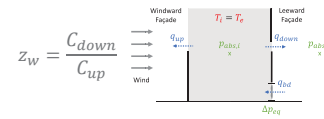


Equilibrium measurement position of Δp_0 – full dataset

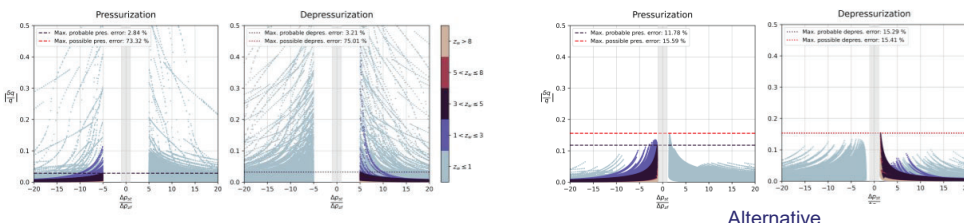


17

KEY FINDINGS



Equilibrium measurement position of Δp_0



ISO constraints

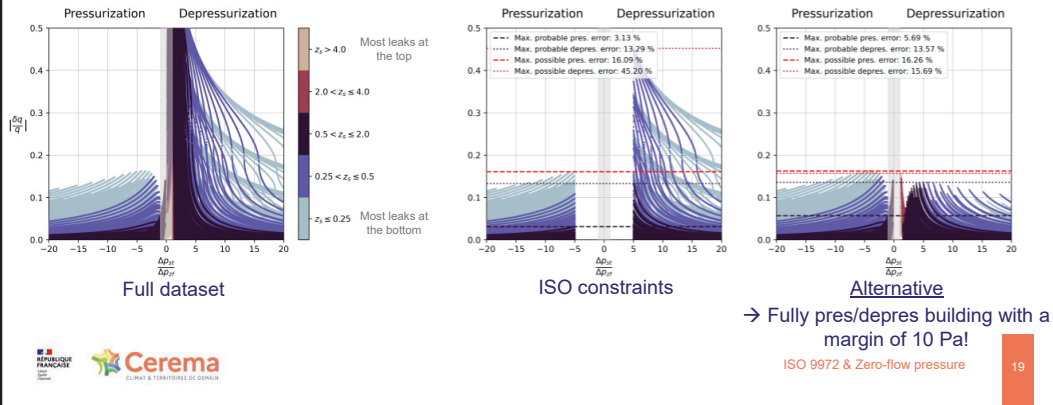
Alternative
→ Fully pres/depres building with a margin of 10 Pa!

18

KEY FINDINGS

$$z_w = \frac{C_t}{C_d} H$$

Stack measurement position of Δp_0



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SUMMARY AND CONCLUSION

- ISO 9972 constraints reduce significant flow errors
- High-rise buildings or buildings in windy locations are often impossible to test

20

SUMMARY AND CONCLUSION

Alternative: Pressurizing or depressurizing the entire building with a margin of 10 Pa would → same range of error!

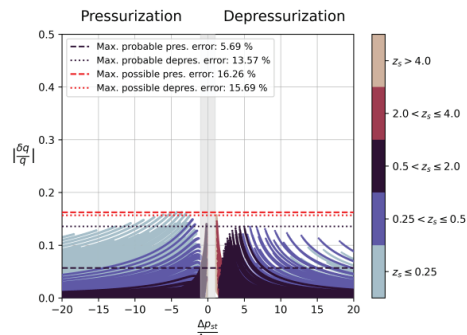
Max. <u>probable</u> error:	Pressurization				Depressurization			
	Downwind	Upwind	Equilibirum	Stack	Downwind	Upwind	Equilibirum	Stack
ISO constraint	45%	1%	3%	3%	13%	2%	3%	13%
New constraint	12%	5%	12%	6%	16%	15%	15%	14%

Max. <u>possible</u> error:	Pressurization				Depressurization			
	Downwind	Upwind	Equilibirum	Stack	Downwind	Upwind	Equilibirum	Stack
ISO constraint	45%	16%	73%	16%	16%	45%	75%	45%
New constraint	15%	16%	16%	16%	16%	15%	15%	16%

21

QUESTIONS?

Benedikt Kölsch
Benedikt.koelsch@cerema.fr



22

Which design parameters impact the resilience to overheating in a typical apartment building?

Abantika Sengupta^{1,2}, Jef Kerckaert¹, Marijke Steeman², and Hilde Breesch¹

1 Department of Civil Engineering
KU Leuven, Belgium



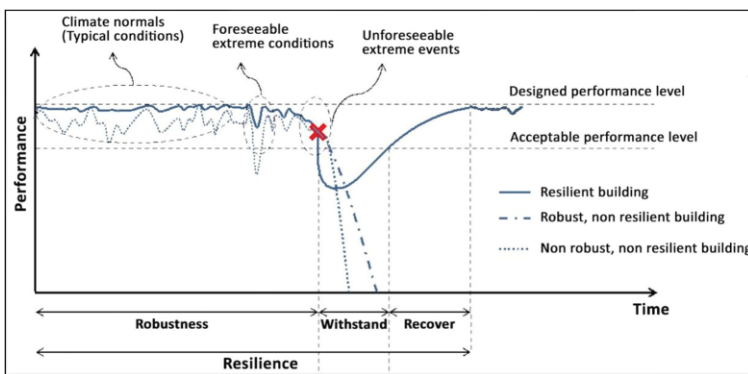
2 Department of Architecture and Urban Planning
Ghent University, Belgium



Problem Statement



IEA EBC Annex 80 - Resilient Cooling of Buildings



Resilience =
**withstand disruptions +
maintain capacity to adapt
+ learn and transform''**

**Resilience against what?
-Extreme Overheating events : Heatwaves**

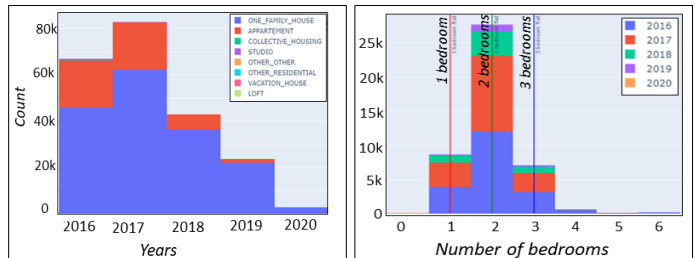
Research objective

To evaluate:
impact of building design parameters
 +
implementation of solar shading
 +
passive cooling strategy (NNV)

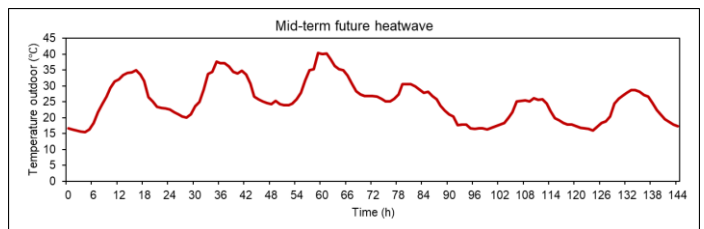
➔ **Thermal resilience to overheating during a heatwave** in a typical Belgian apartment

Annex 80 weather data task force

(a) Typical meteorological year (TMY) 2010s
 (b) 6 days long heatwave



VEKA data from 2016 to 2020 (new buildings) showing the typology of buildings, number of bedrooms and gross area of each apartment



6 days heatwave (29th June-4th July) mid-term future 2050s

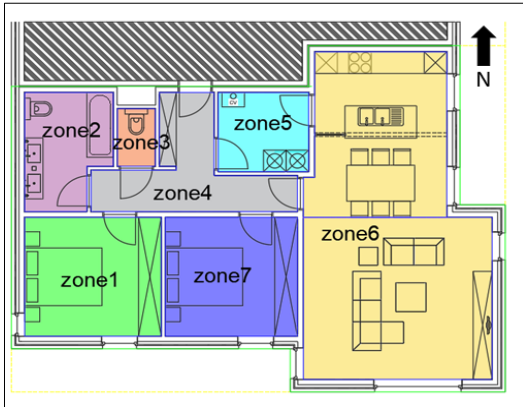
Overview





Research Methodology

Case study: 2-bedroom flat



- External wall u-value: 0.24 W/m²K
- Heavy thermal mass
- South oriented
- Double glazed windows (u-value: 1.00 W/m²K, g-value: 0.56)
- WWR 18%
- internal gains (people and equipment) calculated according to EN 16798-1
- **External solar shading : Radiation > 250 W/m².**

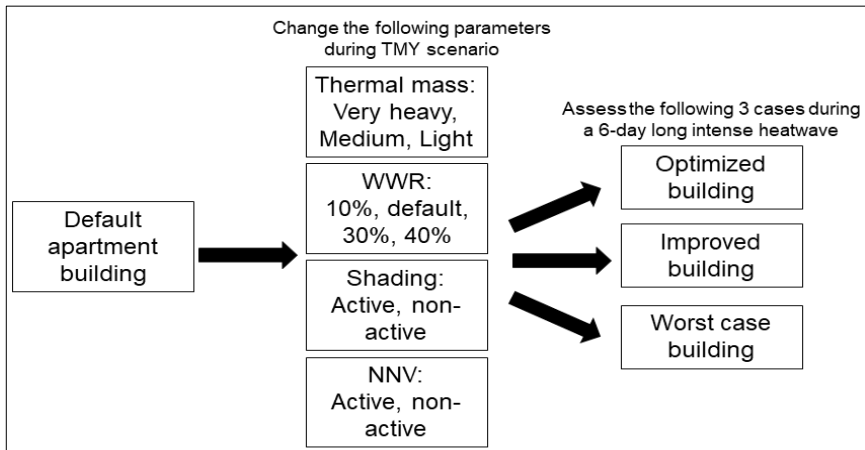
Natural night ventilation (NNV) is implemented:

- Effective area of = 2.7% of the gross floor area.
- April-September :Between 10 pm to 6 am
- T room >22°C
- T room > T outdoor +2°C
- T outdoor > 12°C
- RH room < 70%
- wind velocity < 10 m/s
- There is no rainfall

Thermal zone	Area (m ²)
Zone 1 (Bedroom 1)	11.9
Zone 2 (Washroom)	2.5
Zone 3 (WC)	5.5
Zone 4 (Corridor)	9.0
Zone 5 (Utility room)	5.3
Zone 6 (Living+ kitchen)	38.9
Zone 7 (Bedroom 2)	11.6

5

5



Key performance indicators

Thermal comfort: % of occupied hours above threshold

Heat stress : % of occupied hours above SET 28-degree threshold

Adapted in ASHRAE 55-2017

- ✓ Equivalent dry bulb temperature of an isothermal environment + 50% RH + clothing standard
- ✓ same heat stress (skin temperature) and thermoregulatory strain (skin wetness) as in the actual test environment.

6

6

Building energy simulations

Open Studio,
EnergyPlus

Base case with TMY
contemporary
weather data

Worst, improved and
optimized scenario

Implement Heatwave
shock on the 6 cases

			With Natural night ventilation		
Scenario No	Thermal mass	WWR	Scenario No	Thermal mass	WWR
A1	Heavy		A19	Heavy	
A2	Medium	10%	A20	Medium	10%
A3	Light		A21	Light	
A4	Heavy		A22	Heavy	
A5	Medium	30%	A23	Medium	30%
A6	Light		A24	Light	
A7	Heavy		A25	Heavy	
A8	Medium	40%	A26	Medium	40%
A9	Light	E1	A27	Light	E4
A10	Heavy	E3	A28	Heavy	E6
A11	Medium	10%	A29	Medium	
A12	Light		A30	Light	
A13	Heavy		A31	Heavy	E5
A14	Medium	E2	A32	Medium	30%
A15	Light		A33	Light	
A16	Heavy		A34	Heavy	
A17	Medium	40%	A35	Medium	40%
A18	Light		A36	Light	

7



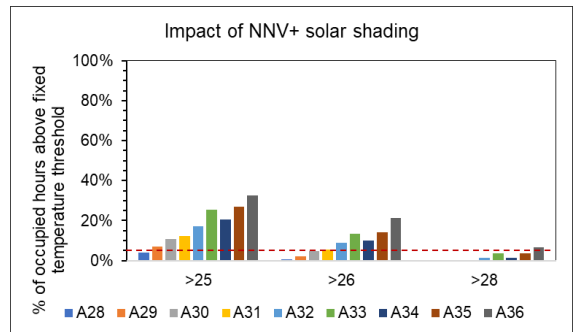
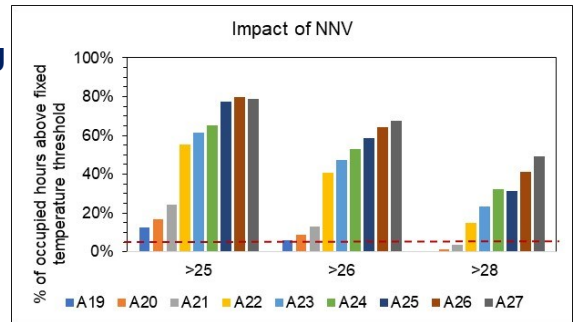
Results

Impact of design and passive cooling

- Thermal Zone 6 (Living-Dining)
 - ➔ Critical zone
- Base case scenario: No passive cooling + No solar shading >5% occupied hours above 26°C
- With NNV + Heavy Thermal mass + low WWR < 5% occupied hours above 26 °C
- With NNV + solar shading + up to 30% WWR < 5% occupied hours above 26 °C

Low thermal mass + high WWR
➔ **Increases overheating**

Solar shading + Natural night ventilation
➔ **Improves thermal resilience**

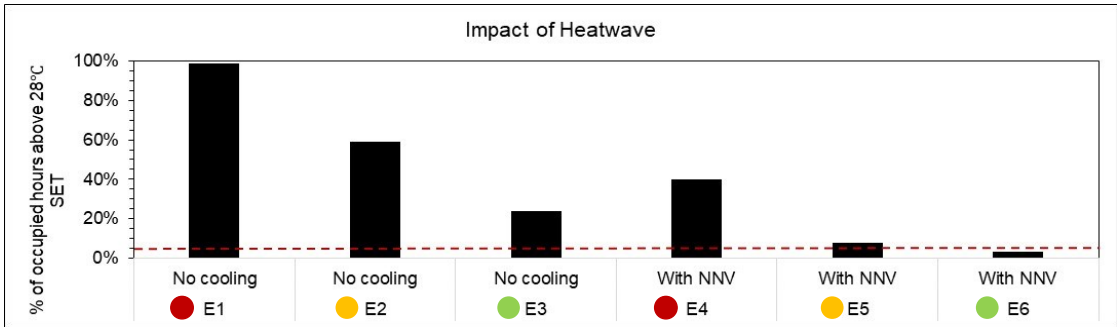


8



Results

Selection of **worst**, **improved** and **optimized** cases



6 days heatwave



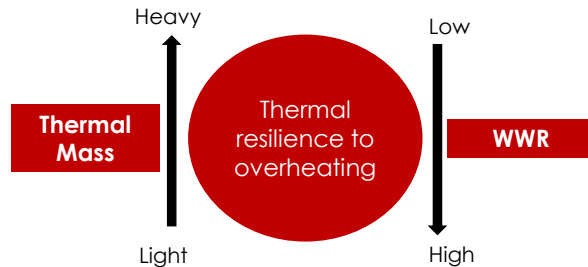
Light Thermal mass + 40% WWR + No shading
95% occupied hours > SET 28 degrees

Heavy thermal mass + 10% WWR + Solar shading + NNV
Less than 5% occupied hours > SET 28 degrees

9



Conclusions



- Higher glazing ratio, high WWR, no shading and with lighter thermal mass ➤ Low thermal resilience
- **WWR** has higher impact than thermal mass
- For buildings with higher WWR and lighter thermal mass, thermal resilience can be improved ➤ **Implement solar shading and passive cooling strategies such as NNV** ^{WWR}
- During heatwave, NNV is not the most thermal resilience solution as the diurnal variations of temperature are limited

10

10

Renewable ventilative cooling? Insights from an Irish perspective

Adam O' Donovan^{1,2},
Theofanis Psomas³, Paul D O'Sullivan^{1,2}

¹MeSSO Research Group, Department of Process, Energy and Transport Engineering, MTU, Cork, Ireland

³MaREI Centre for Energy, Climate, and Marine, Ireland.

²University of Chalmers, Gothenburg, Sweden

43rd AIVC | 11th TightVent & 9th Venticool Conference | Copenhagen, Denmark, October 4-5, 2023

1

Background and context

Motivation for the paper

Personal background

Bachelor of Engineering (Honours) in **Sustainable Energy Engineering**

Passive Cooling in Non-Residential Nearly Zero Energy Buildings: A Thermal Comfort Analysis Based on Measurement, Modelling and Simulation – PhD thesis

42nd AIVC- 10th TightVent & 8th Venticool Conference, 2022

“Is Ventilative Cooling a renewable energy solution and how does it fit into the sustainability agenda?” – Pollet et al. 2022, 42nd AIVC Conference “Ventilation Challenges in a Changing World” Rotterdam, Netherlands

“**Ventilative cooling is, contrary to common renewable energy sources like photo-voltaic etc., a direct renewable energy source.**” – Ventilative Cooling Track Summary, 2022

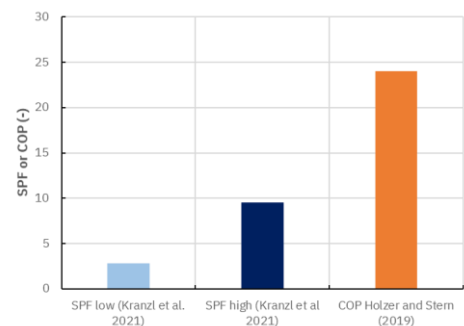


Figure 1: Comparison of SPF values required for renewable status and example from literature

2

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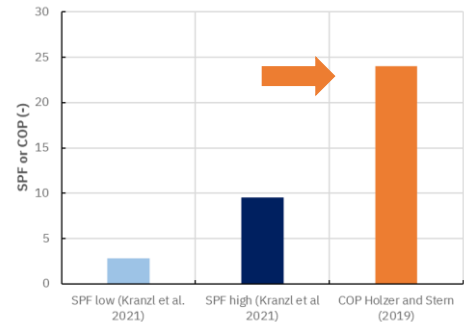


Figure 1: Comparison of SPF values required for renewable status and example from literature

Succeeding Together

www.mtu.ie

3

Background and context

Renewable VC – Previous Work and Challenges



- Limited work focusing directly in the SPF of VC systems (Yan et al. 2022, Holzer and Stern. 2019)
- The scope of renewables should be outside of reversible heat pumps (Krazl et al. 2021)
- VC as a renewable has to overcome:
 1. Cannot be a passive cooling (building insulation, green roof, vegetal wall, shading, thermal mass) – **not attenuation or demand reduction** (section 2.6.2.1)
 2. Cooling without fans or pumps – this **excludes natural ventilation** (section 2.6.2.2)
 3. Has to exclude ventilation for hygienic purposes cooling is **not intentional** (section 2.6.2.3)

Succeeding Together

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4

Background and context

Aim and objectives – Renewable NVC and MVC



A: To present an example of renewable NVC and MVC in a residential building at design-stage where **cooling is intentional (actuated by a control system)** and **hygienic ventilation is excluded** in a northern climate in Ireland (**high potential**).

- 1) Cooling demand assessment (degree hour approach)
- 2) Simplified design stage of supply from MVC and NVC (cooling potential)
- 3) Simplified calculation of SPF for MVC and NVC
- 4) Discussion for residential stock in IE (**cooling demand considered zero** (SEAI, 2022))

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Materials and Methods

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Materials and Methods

Residential Case Study and Weather data



- Retrofitted detached building, inland location in Ireland.
- Limited overheating from previous work by O' Donovan, Psomas and O'Sullivan et al. 2022 (less than 1% of hours > 28°C in living and sleeping spaces).
- Met Éireann historical data and meteorom future data

Table 2: Weather data used for different aspects of the work presented

Location (Name, County)	Elevation (m)	Weather files considered for demand estimates	Weather files used for case study demonstration
Atherry, Galway	40		
Belmullet, Mayo	9		
Shannon Airport, Clare	15	2022,	
Cork Airport, Cork	155	2030 (RCP 2.6), 2030 (RCP 4.5), 2030 (RCP 8.5), 2040 (RCP 2.6), 2040 (RCP 4.5), 2040 (RCP 8.5),	2022, 2050 (RCP 8.5)
Phoenix Park*, Dublin	48		
Valentia, Kerry	24		
Ballyhaise, Cavan	78		
Malin Head, Donegal	20	2.6), 2050 (RCP 4.5),	
Gurteen, Tipperary	75	2050 (RCP 8.5)	
Johnstown Castle, Wexford	62		
Finner, Donegal	33		

*Wind speed and wind direction for Dublin Airport used in the absence of available data

Table 1: Thermo-physical characteristics of case study building used to evaluate renewable NVC and MVC

Variable	Units	Value
Roof U-value	W/m ² K	0.13
Wall U-value	W/m ² K	0.2-0.23
Floor U-value	W/m ² K	0.12-0.13
Window U-value	W/m ² K	1.4
Effective air change rate	h ⁻¹	0.522
Floor area	m ²	182.09
Volume	m ³	491.64
Heat loss co-efficient	W/K	292

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Materials and Methods

Cooling demand and supply calculations (part 1)



$$CD_h = \sum_{h=1}^{h=8760} (T_e - T_b)^+ \quad (1)$$

$$E_{tot,c} = \frac{H \cdot CD_h}{1000} \quad (2)$$

$$Q_b = \frac{1}{3} C_d A_{op} \sqrt{gH \frac{T_i - T_e}{T_i}} \quad (3)$$

$$Q_w = F_R A_{op} U_R \quad (4)$$

$$Q_{nv} = \max(Q_b, Q_w) \quad (5)$$

T_b : (14 to 18.33°C)

H: 292W/K

Floor area: 182m²

C_d : 0.422

$A_{op} = 0.05 \cdot \text{Floor area} \cdot 0.5$

$F_R = 0.025$

$T_i = T_c$ (adaptive)

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Materials and Methods

Cooling supply and SPF calculations (part 2)

$$E_{sup,nv} = \frac{Q_{nv} \cdot \rho_a \cdot C_{p,a} \cdot (T_i - T_e)}{1000} \quad (6)$$

$$Q_{mv} = \frac{ACR_{MV} \cdot V}{3600} \quad (7)$$

$$E_{sup,mv} = \frac{Q_{mv} \cdot \rho_a \cdot C_{p,a} \cdot (T_i - T_e)}{1000} \quad (8)$$

$$SPF_{vc} = \frac{Q_{supply}}{E_{INPUT}} \quad (9)$$

Q_{supply} excludes 0.3l/s/m² for hygienic ventilation

$$ACR_{mv} = 4h^{-1}$$

$$\rho_a = 1.2kg/m^3$$

$$C_{pa} = 1000J/kgK$$

E_{INPUT}

2.4kWh/m²/a – NV
0.1W/(m³/h) – MV

Table 3: Examples of typical energy consumed to operate MVC or NVC systems

Reference	System type	Units	Values reported
(Cho et al. 2021)	Hybrid systems	kWh/m ² /a	0.3 – 2.8
(Agency and Programme 2018)	NVC	kWh/m ² /a	~1.2
(Santos, Hopper, and Kolokotroni 2016)	NVC + phase change materials	kWh/m ² /a	~0.77
(Yan et al. 2022)	NVC	kWh/m ² /a	0.7-1.3
(Holzer and Stern 2019)	MVC	W/(m ³ /s)	<200
(Holzer and Psomas 2018)	MVC	W/(m ³ /h)	0.07 - 0.14

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Results and Discussion

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Results and Discussion

Cooling demand in Ireland (current and future) part 1

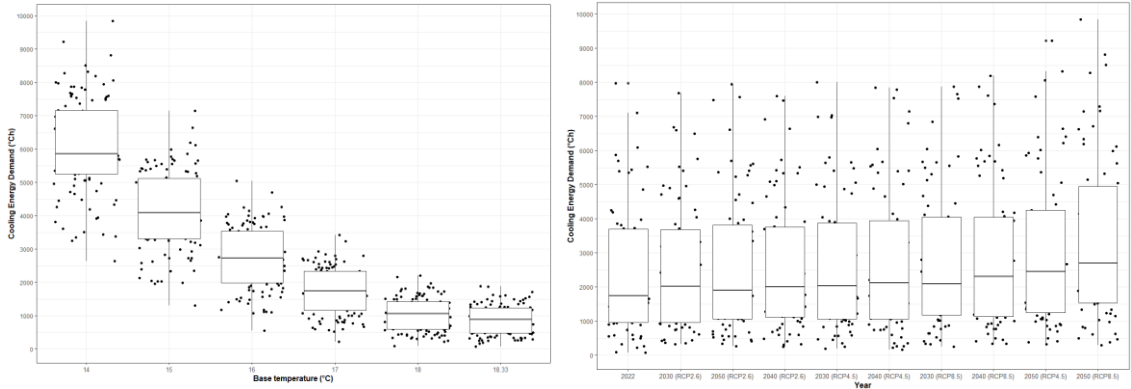


Figure 2: Relationship between cooling degree hours and base temperature and different climate scenarios

Results and Discussion

Cooling demand in Ireland (current and future) part 2

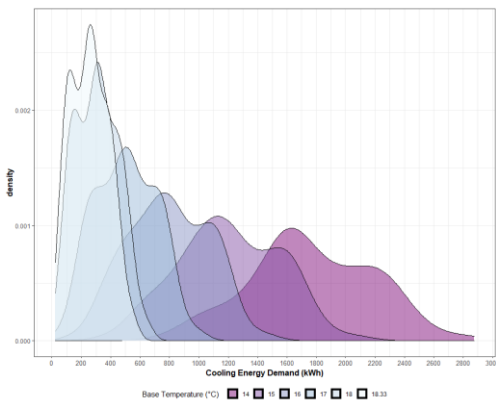


Figure 3: Density plot of estimated cooling energy demand (in kWh) for the selected case study building with respect to different base temperatures

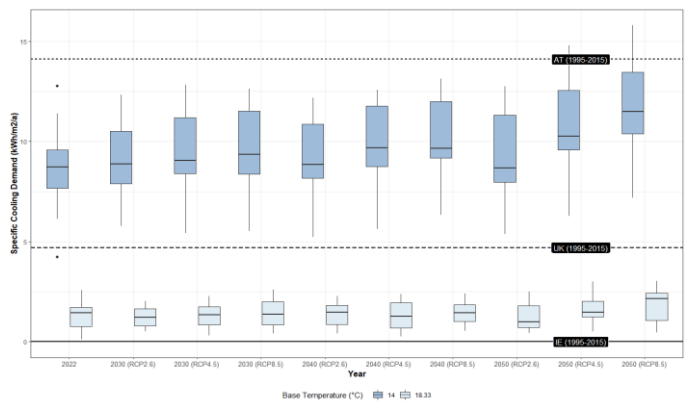


Figure 4: Boxplots of specific cooling demand with respect to climate scenario and different assumptions for base temperature. (Dashed lines indicate different specific cooling demands indicated in the work of (Jakubcionis and Carlsson 2018), for Ireland (IE), United Kingdom (UK), and Austria (AT)).

Results and Discussion

Demand vs Supply

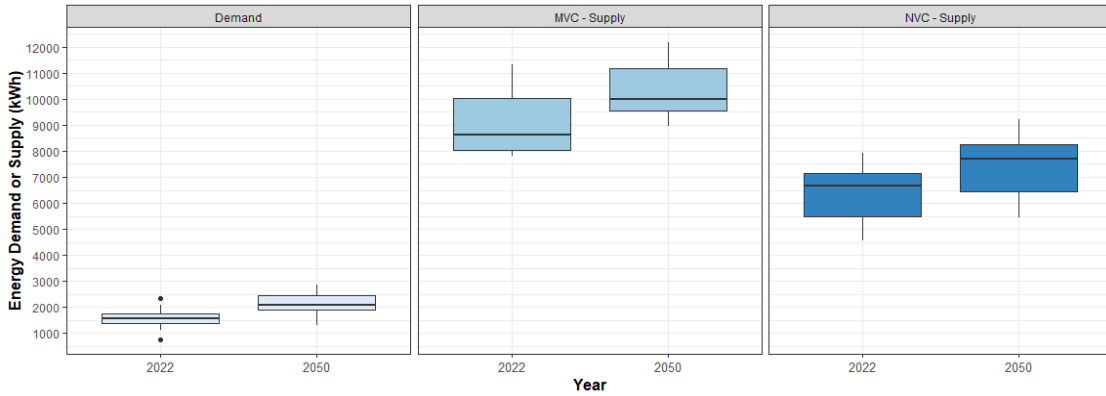


Figure 5: Boxplots of energy demand and supply for cooling with respect to year. (Facet grid represents demand, or supply type for MVC and NVC systems. Note: 2050 refers to RCP 8.5 scenario).

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Results and Discussion

Seasonal Performance Factor

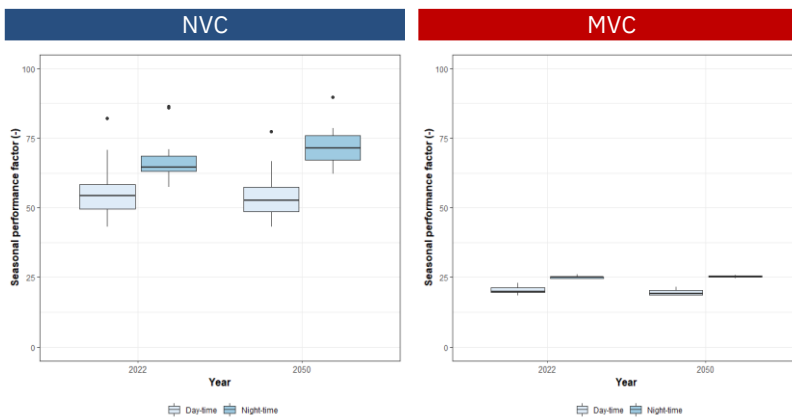


Figure 6: Boxplots of seasonal performance factor with respect to year (Left: SPF values for NVC, Right: SPF values for MVC, colour indicates SPF for day or night-time)

- SPF of **2.6–9.5** is typically required (Kranzl et al. 2021)
- MVC and NVC are **23 and 63** respectively on average
- Capability of generating between **25.1kWh/m²/a to 50.6kWh/m²/a** for NVC, and between **42.9kWh/m²/a to 66.9kWh/m²/a** for MVC.

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Results and Discussion

Seasonal Performance Factor

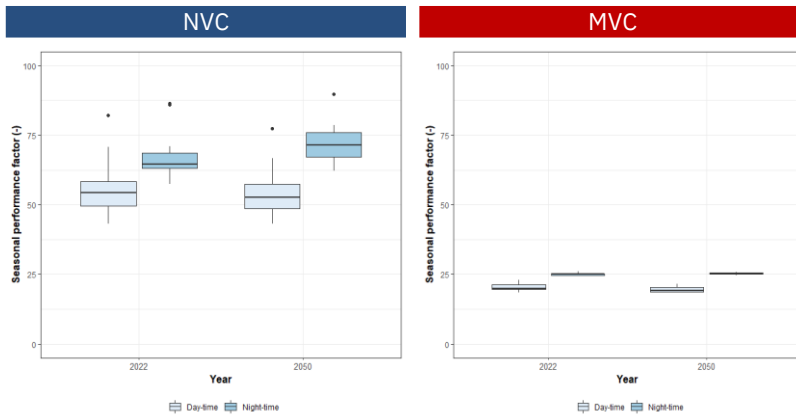


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$$*T_e > 14$$

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Results and Discussion

General discussion related to the Irish residential stock

- **> 2million homes** in Ireland (CSO,2016), **single-sided NVC** the main cooling strategy
- The average floor area is around 111m², with NVC generating between 25.1kWh/m²/a to 50.6kWh/m²/a in renewable cooling.
- **5.6 – 11.3TWh/a** is currently available from NVC in Ireland
- Supply is likely to be outstripping demand by **>3.5 times**

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Conclusion and future work

Conclusion and future work

Renewable cooling in Ireland

• Conclusions

- There is a **cooling demand in Ireland**, but the **NVC supply is currently meeting this demand**.
- Current SPF calculations suggest NVC and MVC systems are **currently renewables that are not officially accounted for**.

• Future work

- Calculations of **SPF for MVC and NVC** in real systems and with dynamic simulation.
- More detail on **cooling base temperatures** for low energy buildings.
- Exploration on a national scale in more detail using specific HLC.



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Munster Technological University

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MECHANICAL ENERGY SYSTEM SIMULATION OPTIMISATION

Thank You, Any Questions?

adam.odonovan@mtu.ie

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Urban context and climate change impact on the thermal performance and ventilation of residential buildings

A case-study in Athens

Maria Kolokotroni, May Zune, Thet Paing Tun
Brunel University London

Ilia Christantoni, and Dimitra Tsakanika
DAEM SA, City of Athens IT Company



This project has received funding from the European Union's Horizon 2020 research and innovation programme under Grant Agreement N° 788576



PRELUDE: Prescient building Operations utilizing Real Time data for ENergy Dynamic Optimization

21 Partners

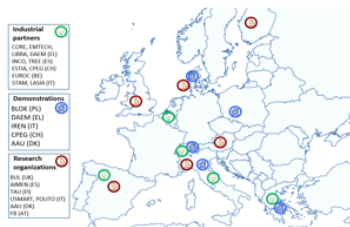
Started in December 2021

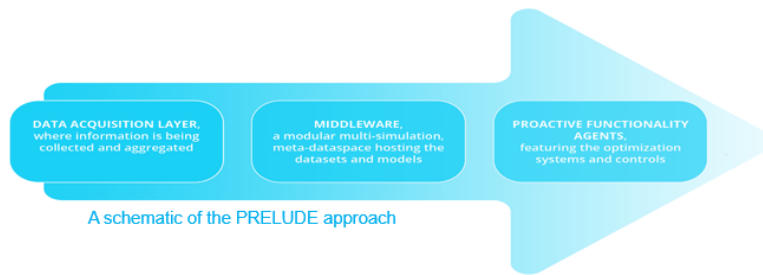
To complete in May 2024

<https://prelude-project.eu/>

8 pilot buildings in Denmark, Poland, Switzerland, Italy and Greece

A testing Living Lab in Austria





The PRELUDE project focusses on assessing the right level of smartness necessary for any given household/building and then providing the optimal tools according to the **needs of the user**.

The system is designed to be versatile and adapt to the engagement, monitoring, and automation level of the building.

- **Passive solutions**, such as natural ventilation and cooling, are prioritized through a free-running strategy.
- **Predictive maintenance** is implemented to reduce costs, emphasizing Renewable Energy Sources.
- **Big data and advanced analytic tools** are used to facilitate flexible building side demand and ease the integration into district heating and electricity grids.
- **Proactive optimization** is to be achieved through data predictive control.

PRELUDE pilot buildings



Geneva



Torino



Athens



Rye

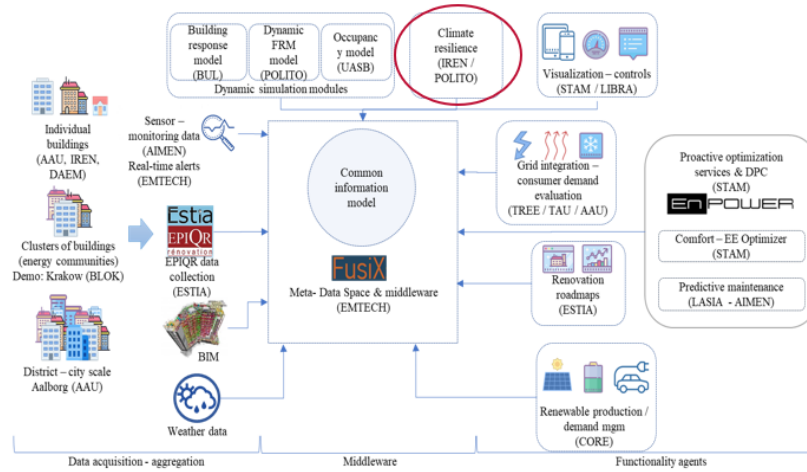
**PRELUDE
Pilot
Buildings**



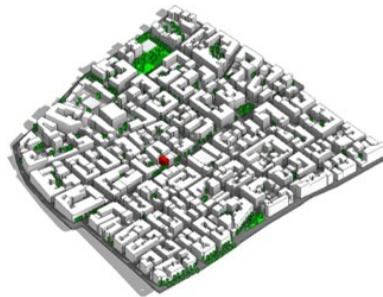
Krakow

Egernsund

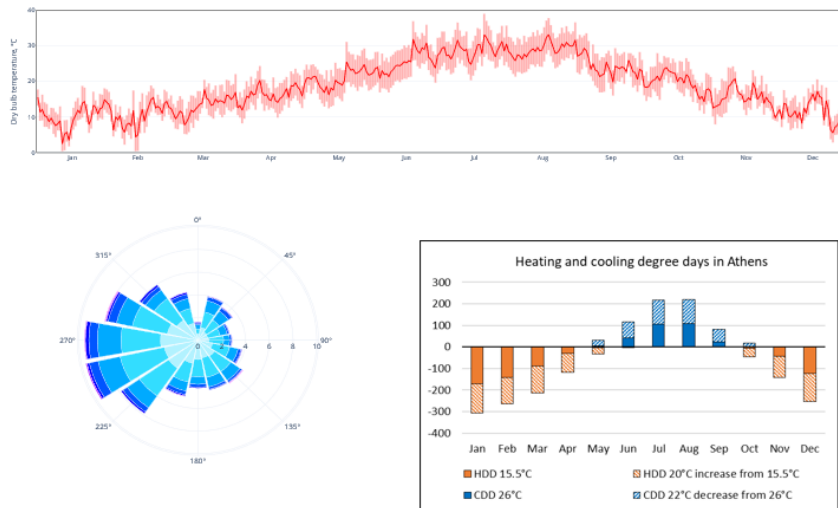
PRELUDE: Prescient building Operations utilizing Real Time data for Energy Dynamic Optimization



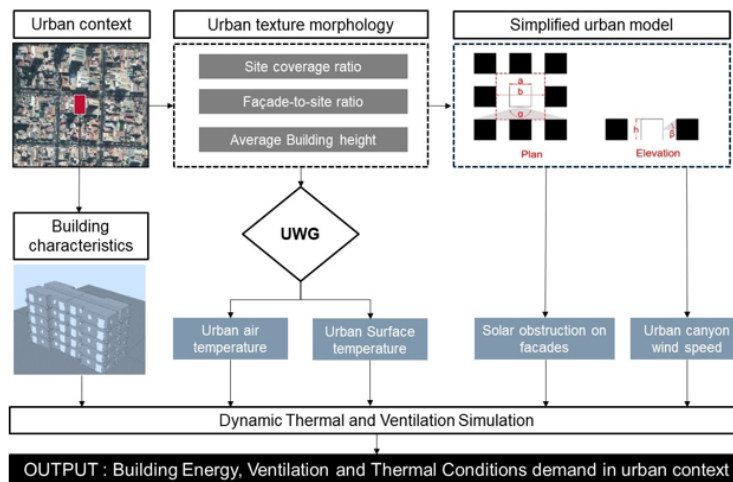
Impact of microclimate on the Athens pilot



Typical weather file – Athens airport



Urban Settings: how to model?



Source: Salvati, A., Palme, M., Chiesa, G., & Kolokotroni, M. (2020). Built form, urban climate and building energy modelling: case-studies in Rome and Antofagasta. *Journal of Building Performance Simulation*, 13(2), 209–225. <https://doi.org/10.1080/19401493.2019.1707876>

Urban weather generator calculations

Autodesk Revit was used to generate the required building information for the UWG program. After the UWG's .xism files and other source files are co-simulated using Matlab, two urban weather files for current and future scenarios were obtained



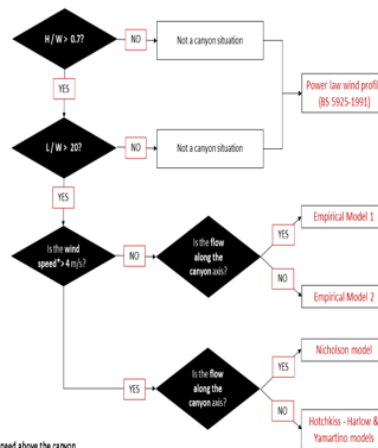
Urban Characteristics	Input data	Vegetation Parameters	Input data
Average Building Height	15.78	Urban Area Veg Coverage	0.0157
Fraction of waste heat into the canyon	1	Urban Area Tree Coverage	0.0245
Building Density	0.473	Veg Start Month	1
Vertical to Horizontal Ratio	1.078	Veg End Month	12
Urban Area Characteristic Length	250	Vegetation Albedo	0.25
Max Dx	62.5	Latent Fraction of Grass	0.5
Road Albedo	0.1	Latent Fraction of Tree	0.5
Pavement Thickness	0.5	Rural Road Vegetation Coverage	0.8
Sensible Anthropogenic Heat (Peak)	20		
Latent Anthropogenic Heat (Peak)	2		

Wind speed and solar obstruction

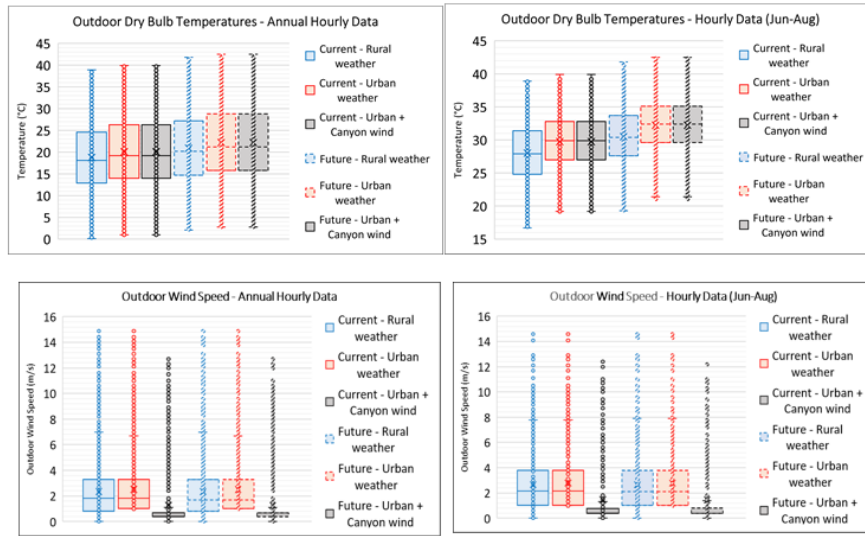
The hourly wind speed was calculated using the algorithms developed from experimental data in Athens (similar canyons). These were carried out under the European project UrbVent almost 20 years ago.

Hourly wind speed values of canyon wind were calculated for the undisturbed wind and wind direction values found in the rural weather files. The urban canyon wind speed values were then replaced with the urban weather files generated from the UWG program.

Overshadowing was calculated from the EnergyPlus program

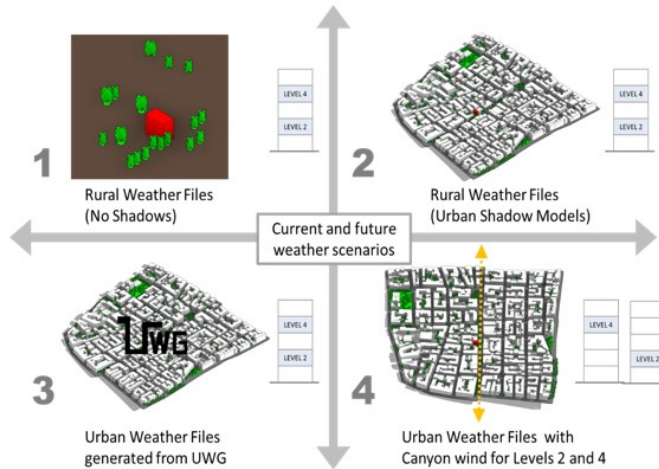


Weather data comparison

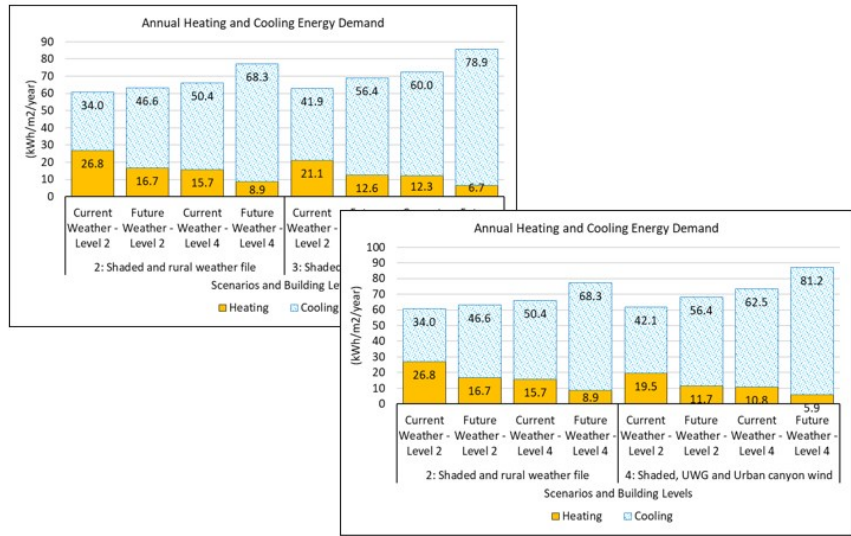


Future weather is RCP8.5 scenario for the year 2050

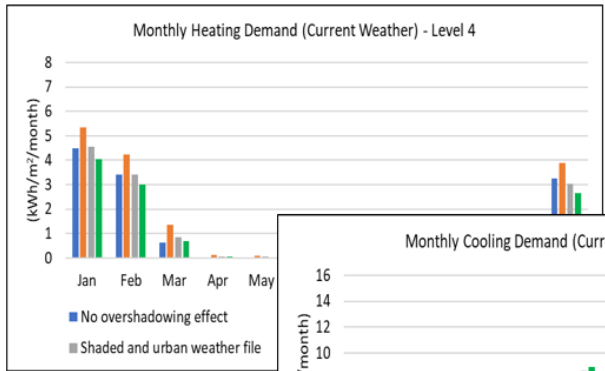
Simulations using EnergyPlus



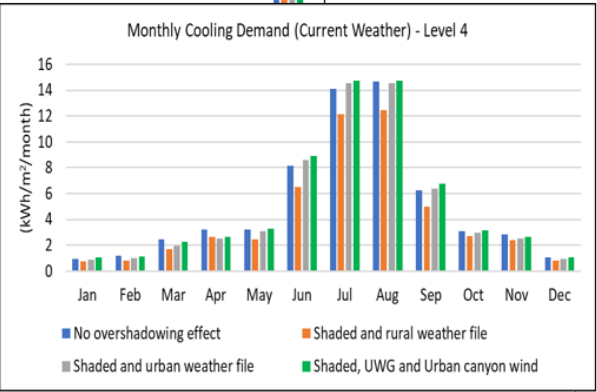
Energy use simulation results



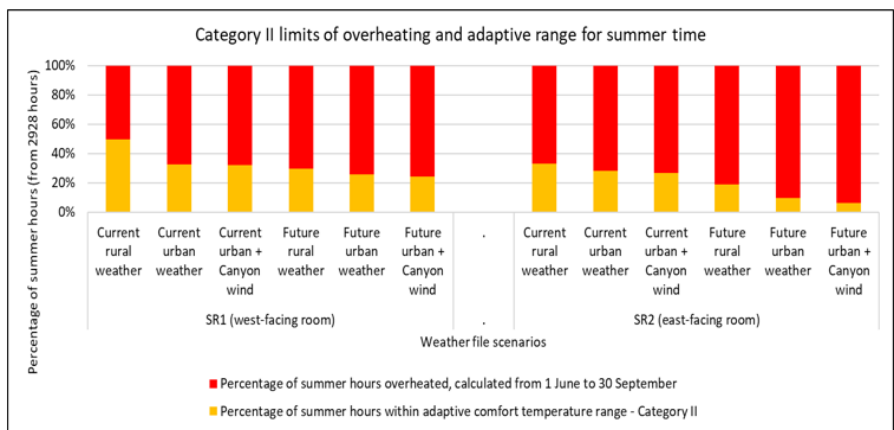
Weather file	Heating		Cooling		Total	
	kWh/m2/year	Change ratio	kWh/m2/year	Change ratio	kWh/m2/year	Change ratio
Overshadowing included in simulations						
Building Level 4						
Current Weather	15.7		50.4		66.1	
Current Urban Weather (UWG)	12.3	0.78	60	1.19	73.1	1.11
Current UWG, Urban canyon Wind	10.8	0.69	62.5	1.24	74.0	1.12
Future Weather	8.9	0.57	68.3	1.36	77.8	1.18
Future Urban Weather (UWG)	6.7	0.43	78.9	1.57	86.0	1.30
Future UWG, Urban canyon Wind	5.9	0.38	81.2	1.61	87.5	1.32
Building Level 2						
Current Weather	26.8		34		60.8	
Current Urban Weather (UWG)	21.1	0.79	41.9	1.23	63.8	1.05
Current UWG, Urban canyon Wind	19.5	0.73	42.1	1.24	62.3	1.03
Future Weather	16.7	0.62	46.6	1.37	63.9	1.05
Future Urban Weather (UWG)	12.6	0.47	56.4	1.66	69.5	1.14
Future UWG, Urban canyon Wind	11.7	0.44	56.4	1.66	68.5	1.13



Click to add text



Overheating assessment

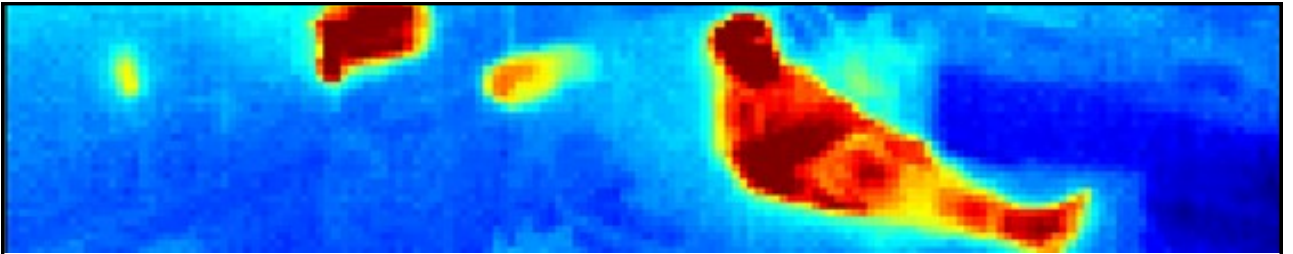


Conclusions

- The confounding effects of urban density, urban textures and exposure to the wind, building design and human activities alter weather characteristics over and around urban areas.
- Weather files were generated from the UWG program and urban canyon wind calculation, and were used to compare how different weather impacts building performance. The overshadowing effects from the surrounding buildings were considered as it is relevant to an urban setting.
- The case study building is located in Athens within an urban canyon with data obtained from the PRELUDE H2020 project.
- Simulations showed (typical weather file compared with fully modified urban weather file)
 - an increase of the cooling demand by 24%. In the future (2050) we will have a 66% increase in cooling demand
 - total energy demand (heating and cooling) increased only 3% for lower floors and 12% for higher floors due to the reduction of heating demand. In the future we will have an increase 32% for higher floors and 13% for lower floors.
 - If the building is free floating an adaptive thermal comfort analysis indicated that only 25% of the summertime will be comfortable in the future in comparison to 50% prediction by current typical weather.
- Therefore, the use of a suitable weather file to include urban external conditions in thermal simulations is essential for more accurate predictions of energy demand and internal avoidance of overheating in free-running buildings.

Thank you





Thermography-based assessment of mean radiant temperature and occupancy in healthcare facilities

Pilot study at Hannover Medical School

Paul Seiwert

EBC | Institute for Energy Efficient
Buildings and Indoor Climate



1

Why are occupancy and mean radiant temperature important information for building control?



Convalescence time and overall medical outcomes in hospitals and other healthcare facilities are **influenced by indoor environmental conditions** [1]



Demand controlled ventilation (DCV) as energy efficient way for providing high IEQ [2]



DCV requires detailed information on **occupancy and indoor environmental parameters**

Pictograms: [5]

2



2

Pilot study on sustainable hospitals - overview

Partners: Chalmers University of Technology
Hannover Medical School
Furbish AB
RWTH Aachen University

Observation period: Mid November – Mid December 2020

Observation space: 5 patients rooms and 1 physicians room in ward for cardiovascular heart diseases at Hannover Medical School

Study design:

- Experimental observation of indoor environment
 - 1 IEQ measurement system (1) per room
 - 2 thermography sensor systems (2) per room
- Patient survey
- Patient diary



Ward building at Hannover Medical School



Exemplary patient room with measurement equipment

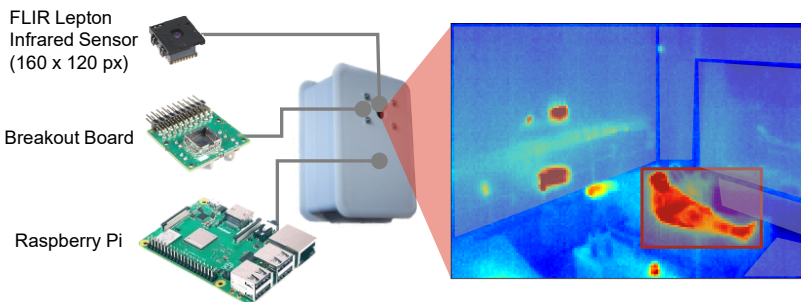
3

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3

Hardware and objectives of thermography based measurement



Mean radiant temperature calculation

Calculation of MRT based on wall, (ceiling) and floor surface temperature and according view factors

Occupancy detection

Machine learning algorithms for detection of occupants based on temperature difference from background

Images: [6]

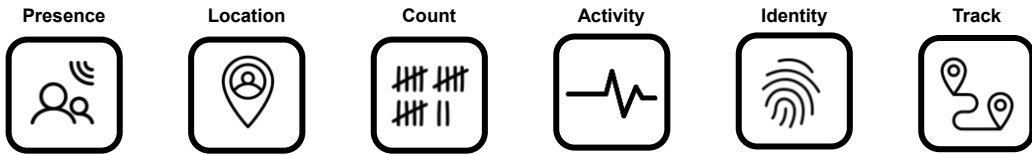
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4

Requirements and properties of image based sensors for occupancy detection



Spatial-temporal properties of occupancy measurement [2]

- Several different occupancy technologies available
 - ≡ e.g. PIR-, CO₂-, Image-, Electromagnetic sensors
 - ≡ Selection can be based on different spatial-temporal properties for detection
- Image based sensors so far have limited applications for occupancy detection
 - ≡ Machine learning → computation intensive
 - ≡ Capable of covering all properties of occupant detection
 - ≡ Privacy considerations are important factor for implementation and acceptance

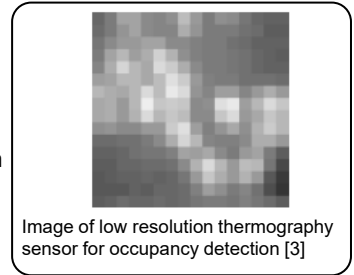


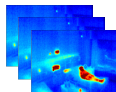
Image of low resolution thermography sensor for occupancy detection [3]

Pictograms: [5]

Data pipeline and training process for machine-learning model

Data basis:

Frames from 10 sensors (2 per room) at 10 second intervals



Preprocessing:

- Normalization
- Redundancy test
- Validity test

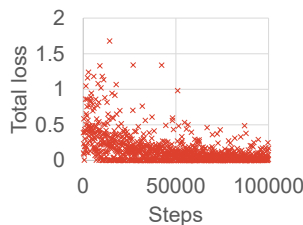
Data pipeline:

- Annotation of occupants in frames with bounding boxes
- 1000 frames (800 training / 200 evaluation)



Network training and evaluation:

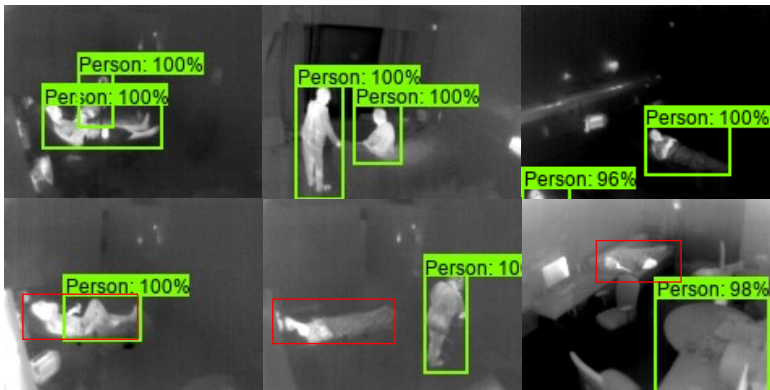
Network: Faster R-CNN Inception ResNet V2 640x640 (transfer learning: pre-trained on visual domain images)



Evaluation metrics:

mAP – mean average precision
IoU – intersection over union

Results of occupancy detection



Exemplary frames with boundary boxes for identified occupants (green) and missing ground truth values (red)

Metric	Value
mAP (overall)	0.47
mAP@IoU=0.5	0.80
mAP@IoU=0.75	0.48

- False Negatives occur frequently when body is partially obstructed (e.g. by blanket)
- Overfitting due to low variation in training data and sensor installation positions
- Further classes (body posture/activity) may reduce variation within classes

7

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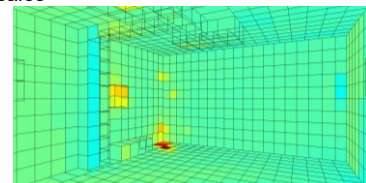
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Summary and outlook

- Acquired data enable machine learning based detection of occupants in hospital rooms
- Presence and count may be evaluated with the proposed methodology
- Light-weight model can be executed on low capacity systems e.g. Raspberry Pi for real time applications
- Accuracy is within typical accuracy of neural networks for object detection
- For further details on mean radiant temperature assessment, please refer to conference paper

■ Outlook

- ≡ Acquisition of further training data with higher variation in perspectives and scales
- ≡ Introduction of additional classes for activity and body posture evaluation
- ≡ Network training exclusively with IR domain data
- ≡ Integration into 3D-mapping of thermal boundary conditions (see [3])
 - = →Location / tracking assessment



8

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8

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References:

- [1] Shajahan, Amreen; Culp, Charles H.; Williamson, Brandon (2019): Effects of indoor environmental parameters related to building heating, ventilation, and air conditioning systems on patients' medical outcomes: A review of scientific research on hospital buildings. In: *Indoor Air* 29 (2), S. 161–176. DOI: 10.1111/ina.12531.
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- [3] Berger, M.; Armitage, A. (2010): Room occupancy measurement using low-resolution infrared cameras. In: *IET Irish Signals and Systems Conference (ISSC 2010)*, IET Irish Signals and Systems Conference (ISSC 2010), Cork, Ireland, 23-24 June 2010: IET, S. 249–254.
- [4] Seiwert, Paul; Schmitt, Lukas; Wesseling, Mark Thomas; Müller, Dirk (2018): Detection of Vertical Air Temperature Distribution by Long- Wave Infrared Thermography. In: *Finnish Society of Indoor Air Quality and Climate (Hg.): Proceedings: Roomvent&Ventilation 2018. Excellent Indoor Climate and High Performing Ventilation*. Unter Mitarbeit von Risto Kosonen, Mervi Ahola und Jarkko Narvanne. Roomvent&Ventilation. Espoo, Finland, 02.06-05.06.2018. Helsinki, Finland: SIY Indoor Air Information Oy.
- [5] Pictograms from Noun project(CC BY 3.0): ventilation by IconMark; clock by Yoyo; Man in Hospital by Gan Khoon Lay; Human by Cuputo; Temperature by Vectors Market; Map by Vectorstall; Count by Adrien Coquet; activity by asnirun al wowo; Identity by Vectorstall; Pin by Vectorstall.
- [6] Images from Flir Inc. and Raspberry Pi Foundation

EBC | Institute for Energy Efficient
Buildings and Indoor Climate

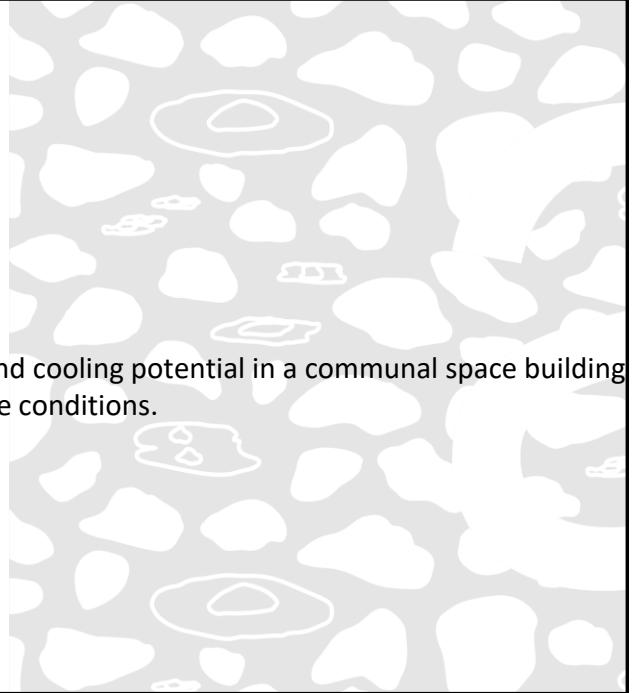


RWTHAACHEN
UNIVERSITY



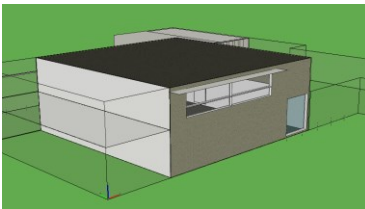
Analyzing natural ventilation and cooling potential in a communal space building in Belgium under future climate conditions.

Shiva Khosravi
Joost Declercq
Delphine Ramon



1

Analyzing natural ventilation and cooling potential in a communal space building in Belgium under future climate conditions.



2

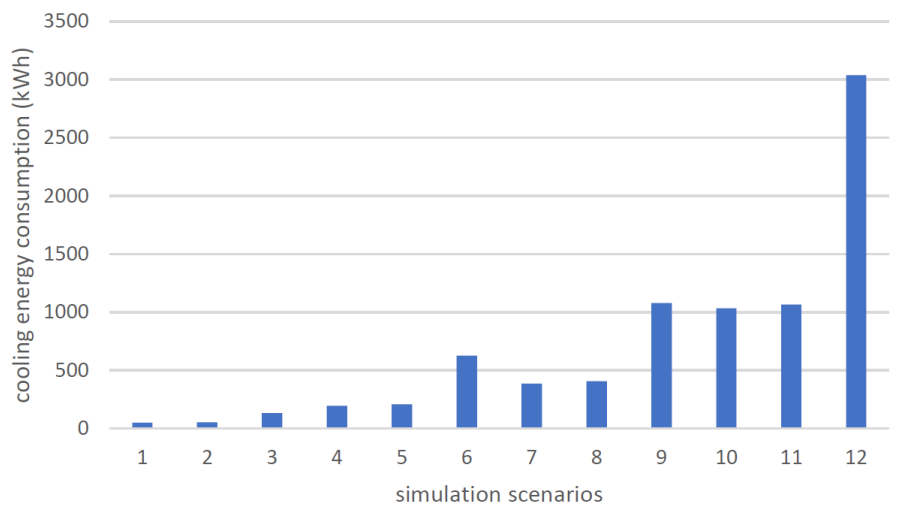


2

scenarios	Type of weather data	Passive cooling strategy	Thermal mass
Scenario 1	TMY current	Natural ventilation	high
Scenario 2	TMY current	Natural ventilation	low
Scenario 3	TMY current	No passive cooling	high
Scenario 4	TMY future	Natural ventilation	high
Scenario 5	TMY future	Natural ventilation	low
Scenario 6	TMY future	No passive cooling	high
Scenario 7	EWY Current	Natural ventilation	high
Scenario 8	EWY Current	Natural ventilation	low
Scenario 9	EWY Current	No passive cooling	high
Scenario 10	EWY future	Natural ventilation	high
Scenario 11	EWY future	Natural ventilation	low
Scenario 12	EWY future	No passive cooling	high

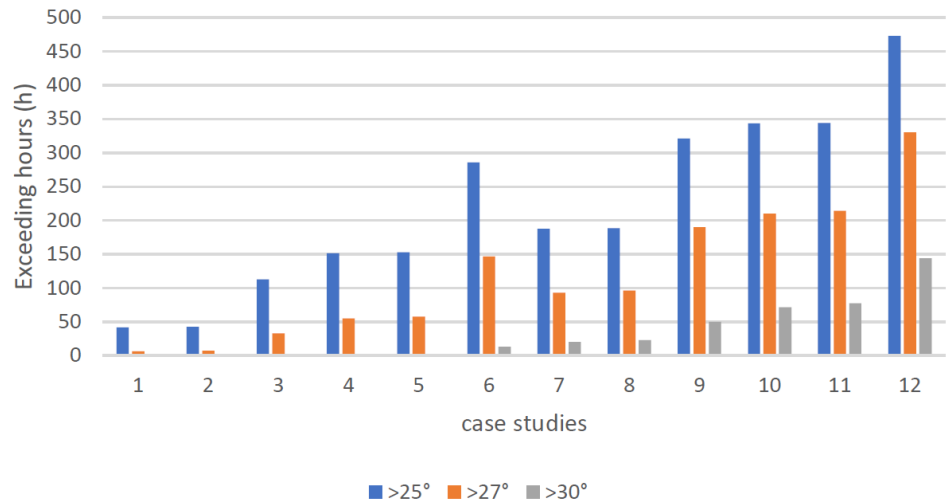
Method

3



Results

4



Results

5

- Results clearly indicate that natural ventilation combined with thermal mass can have a significant influence.
- In anticipated future climate scenarios, the inclusion of mechanical cooling becomes essential to achieve summer comfort under all circumstances. However, the potential energy savings achieved by combining mechanical cooling with natural ventilation are projected to be even higher compared to the present climate.

Conclusion

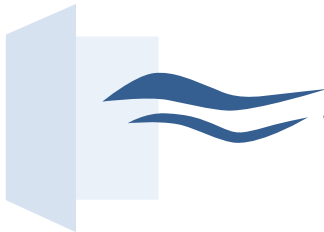
6

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43rd AIVC - 11th TightVent & 9th venticool Conference

In Copenhagen (Denmark) on October 4-5, 2023



A study of indoor environment and window use in French dwellings monitored during a summer with heatwaves

Mathilde Hostein^{1,2}, Bassam Moujalled^{1,3}, Marjorie Musy^{1,3} and Mohamed El Mankibi²

¹ Cerema, BPE Research Team, 46 rue Saint Théobald, 38081 L'Isle d'Abeau, France

² Univ. Lyon, ENTPE, Ecole Centrale de Lyon, CNRS, LTDS, UMR 5513, 3 Rue Maurice Audin, 69518 Vaulx-en-Velin, France

³ Univ. Savoie Mont Blanc, CNRS, LOCIE, UMR 5271, 60 rue du lac Léman, Savoie Technolac, 73376 Le Bourget-du-Lac, France



LOCIE



mathilde.hostein@cerema.fr

1

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Field measurement campaign

- CREATIV research project
 - Thermal comfort
 - Indoor air quality
 - Heatwave



- Lyon & Clermont-Ferrand
- 4 dwellings monitored
- June to September 2022
- Surveys :
 - Logbook
 - Thermal comfort surveys
 - General questionnaire
- Measurements

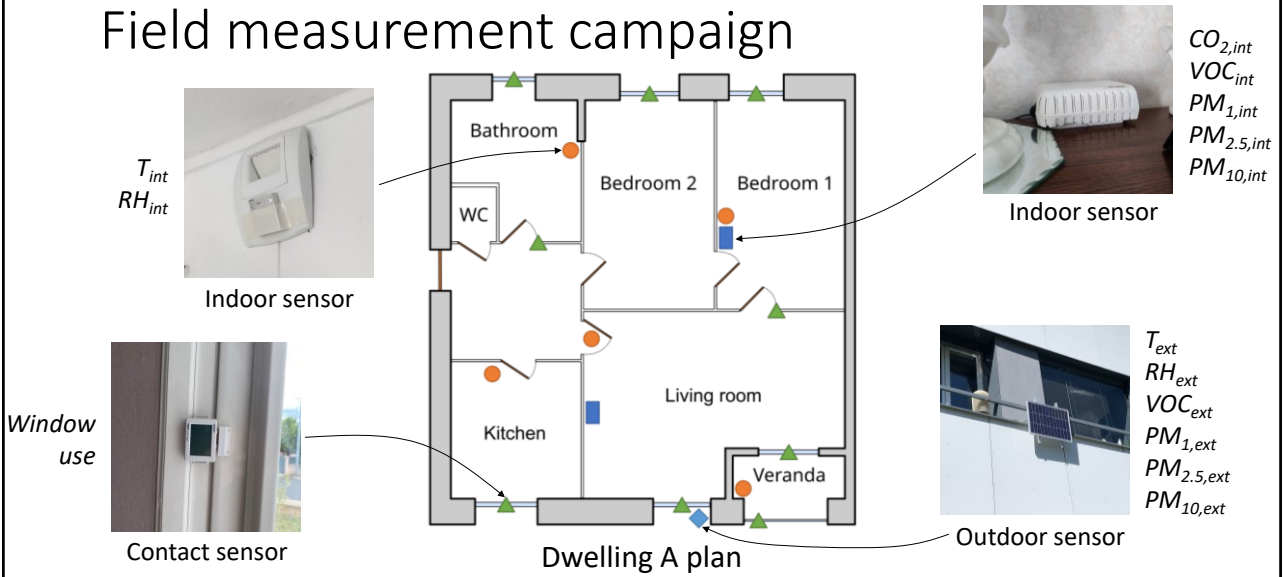
Mathilde Hostein

Short Oral Presentation

2

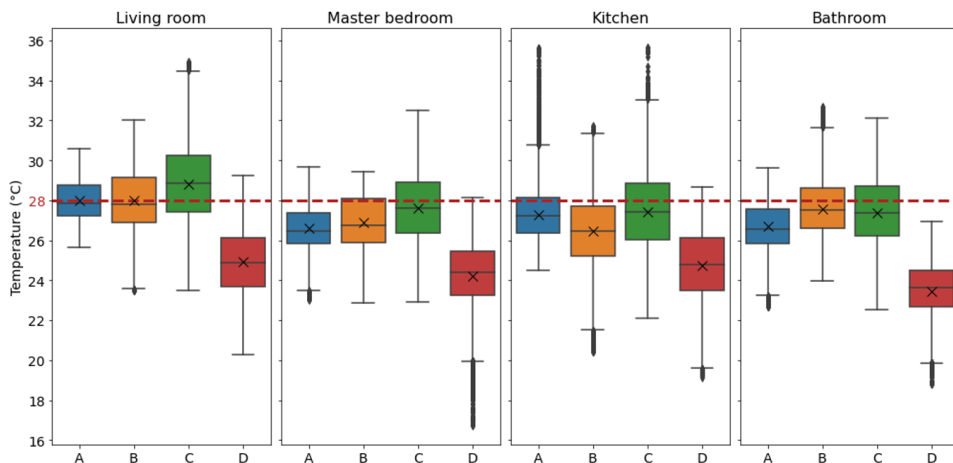
2

Field measurement campaign



3

Indoor thermal environment



Comparison of temperature by room in the different dwellings using boxplot diagrams with means displayed

4

October 2023
43rd AIVC
11th TightVent
& 9th venticool
Conference

A study of indoor environment and window use in French dwellings monitored during a summer with heatwaves
M. Hostein^{1,2}, B. Moujaléd^{3,4}, M. Musy^{1,2}, and M. El Mankibi¹

¹Cerema (Brest), ²UMR 5818 (Lyon), ³UMR 5076 (Lyon), ⁴UMR 5217 (Lyon)

Context
• 3 heatwaves took place during summer 2023 in Lyon and Clermont-Ferrand.
• Occupant behaviour is crucial to ensuring a comfortable indoor environment.

Objective
What are the physical and contextual variables influencing window use in dwellings during summer and heatwaves?

Methodology
A field measurement campaign was conducted in two French cities from June 13 to September 27, 2023. 4 dwellings were monitored, including 1 house and 3 apartments in the same multi-family building. The indoor environment is assessed using hygrothermal and indoor air quality measurements at 10-minute time-steps. Surveys focused on comfort perception and occupant behaviour during summer. Window use is studied using exploratory analysis and a feature selection algorithm.

Window use visualization
The window in the living room B is opened mostly in the evening with some openings in the morning. The openings are short and frequent.

Indoor thermal environment
Thermal environment is perceived to be extremely hot during heatwave days in all apartments (A, B and C) in Lyon. Comparison of T_{int} by room in 4 dwellings using logistic.

Feature selection analysis
Data grouped by room into 16 data sets. Most impacting variables on window use for all data sets: T_{int} , T_{ext} , $CO_{2,int}$, VOC_{int} , $I_{hg_{ext}}$, Time of day.

Conclusion
The 4 households have different window use behaviours.

Acknowledgments
Data were collected as part of the CREATY project funded by ANR (grant agreement n°2000000401).

More during the poster session...

Window use drivers

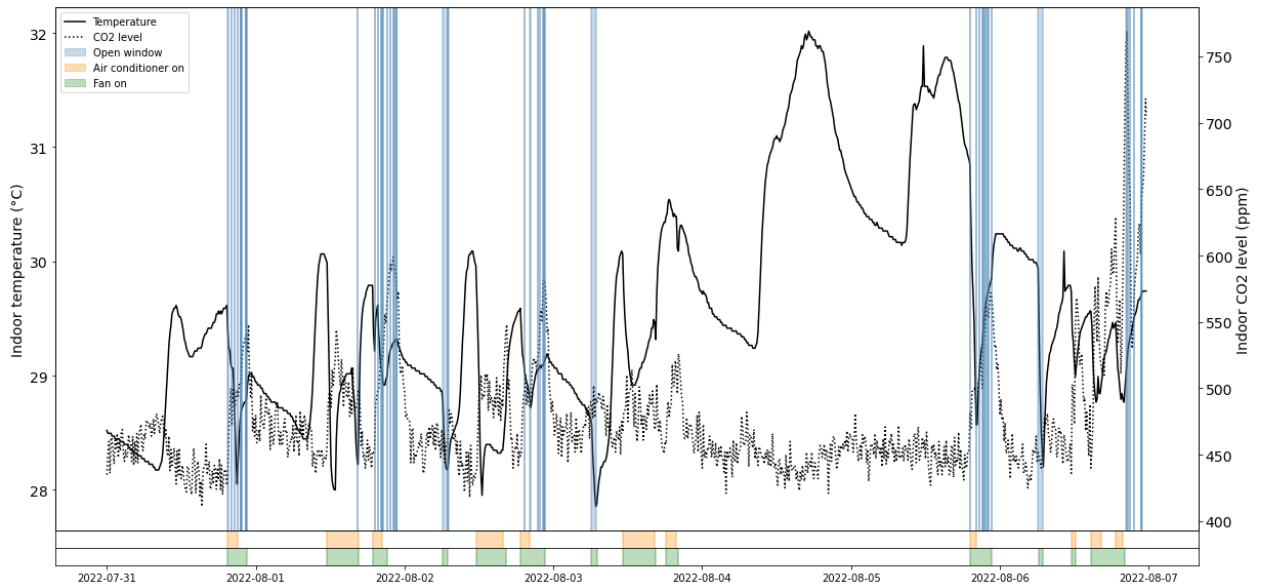
- T_{int} et T_{ext}
- $CO_{2,int}$ et VOC_{int}
- $I_{hg_{ext}}$
- Time of day

5

Thank you !

6

Evolution of the indoor environment with adaptation actions in the living room of dwelling B during the 3rd heatwave



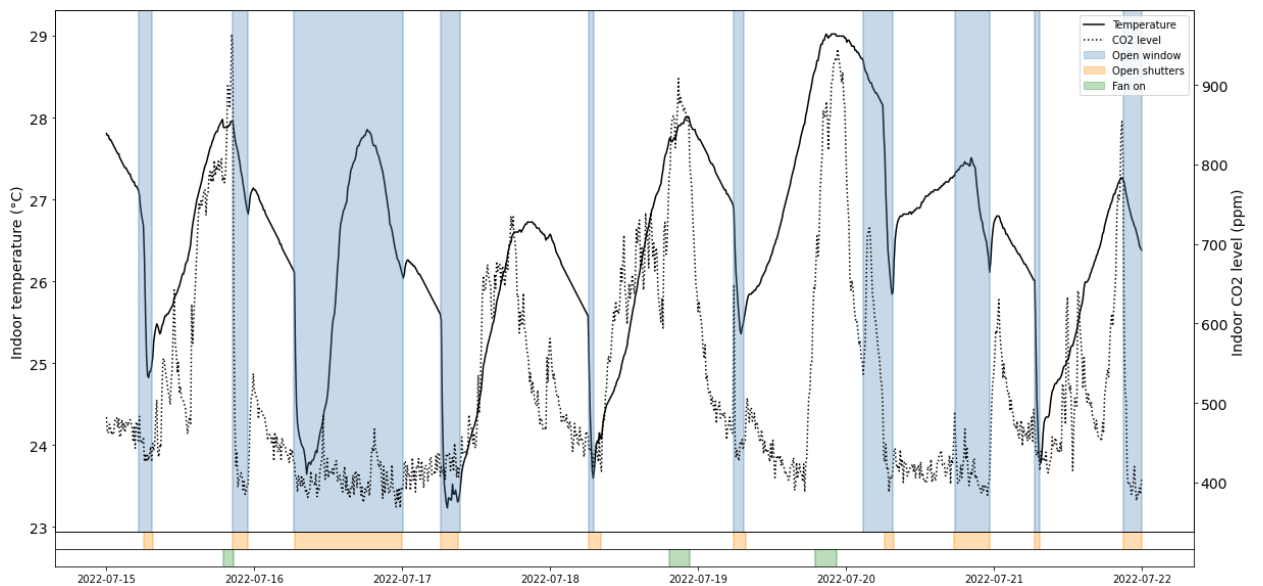
04/10/2023

Appendices

8

8

Evolution of the indoor environment with adaptation actions in the living room of dwelling D during the 2nd heatwave



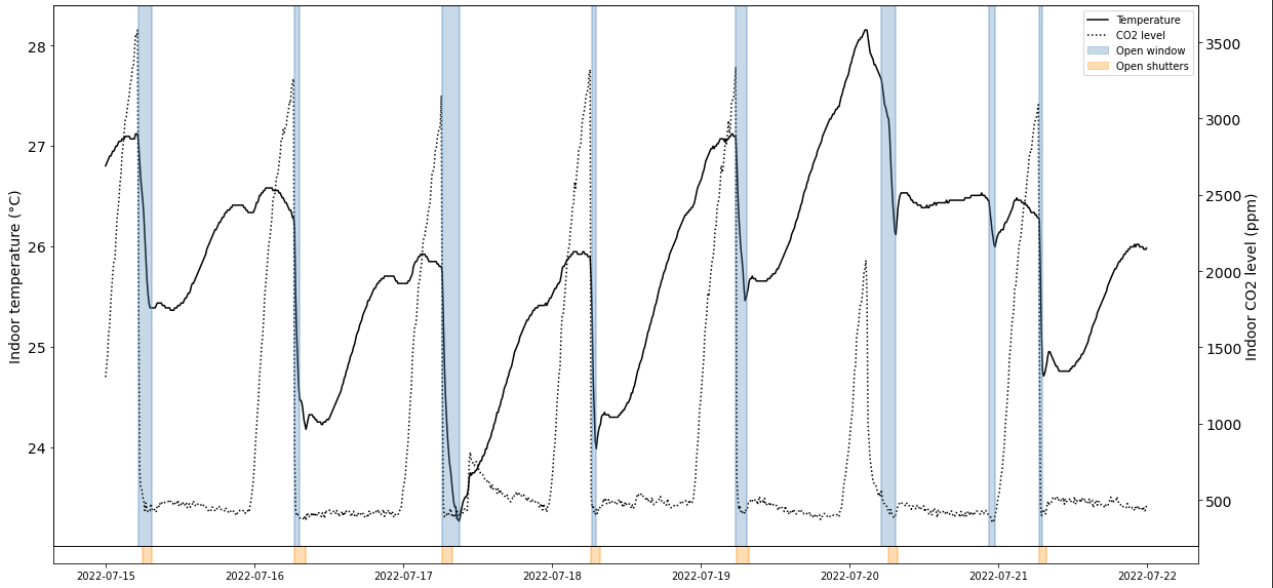
04/10/2023

Appendices

9

9

Evolution of the indoor environment with adaptation actions in the bedroom of dwelling D during the 2nd heatwave



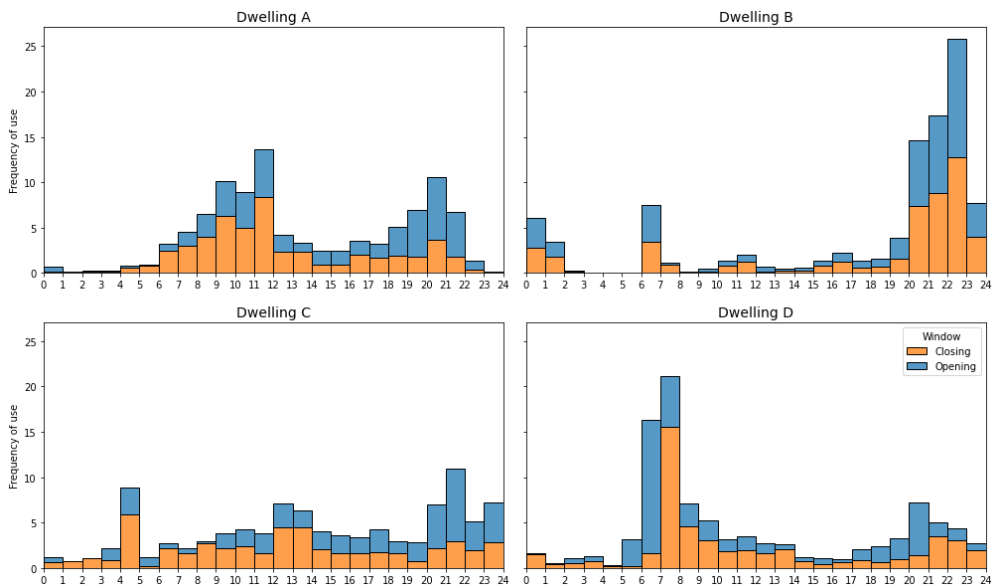
04/10/2023

Appendices

10

10

Windows openings and closings according to the time of day



04/10/2023

Appendices

11

11

Feature selection analysis

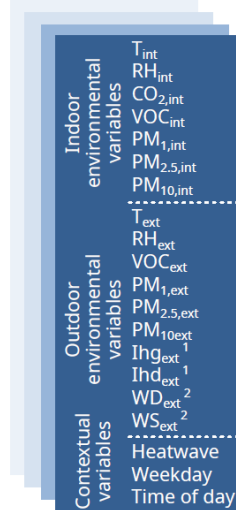
Sensors raw data



DATA PROCESSING

Indoor pollutants
Hygrothermal conditions
Window use
Outdoor variables

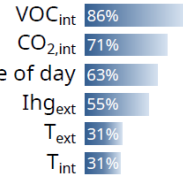
Data grouped by room into 16 data sets



FEATURE SELECTION

VSURF R package based on random forests

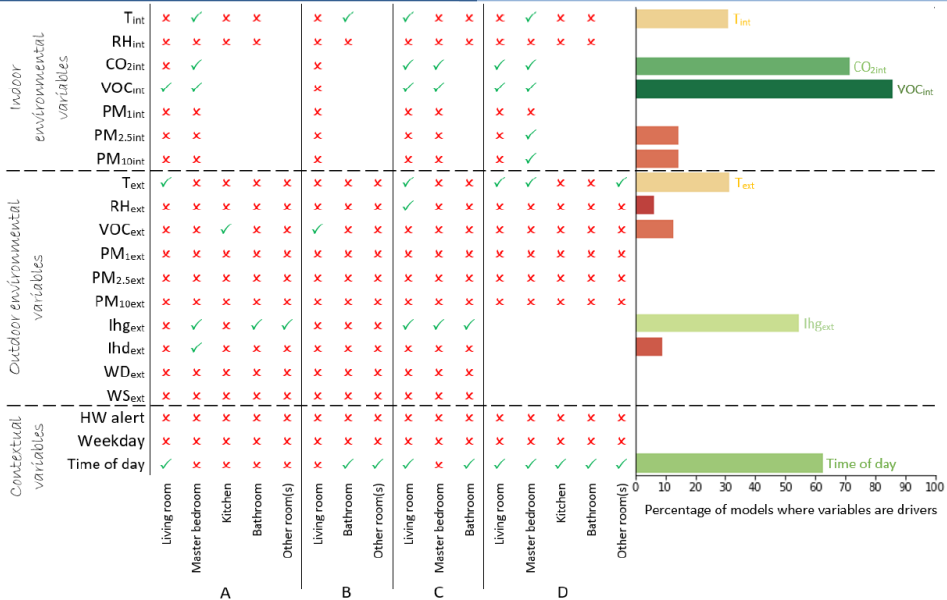
Most impacting variables on window use for all data sets



¹ Global and Diffuse Horizontal Irradiation
² Wind Direction and Speed

12

Feature selection analysis : windows use drivers



13

Importance of thermal stack effect in ventilative cooling concepts for residential buildings

AIVC 2023

04/10/2023

HOME OF OXYGEN

DUCO

1

Research question

Evolutions in residential building sector (e.g. in the Netherlands)

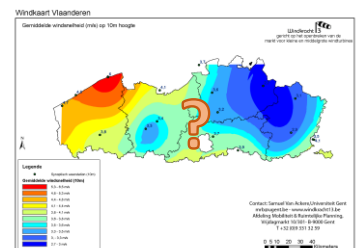
- factory-built (modular) dwellings
- light-weight (sustainable, recyclable) construction
- thoroughly designed

What ventilative cooling concepts to use in residential buildings with light-weight construction?

- That are effective in improving indoor temperature
- That are robust to external conditions

While also:

- being cost-effective
- being burglary-, insect-, weather-proof
- integrating well with other systems
- ...



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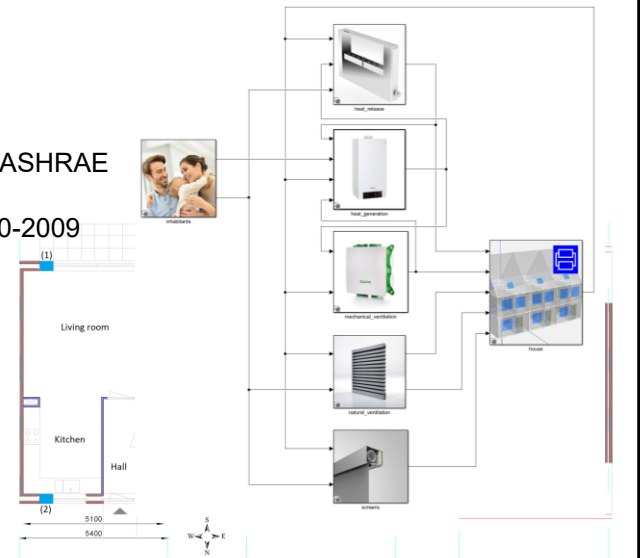
DUCO

2

Model-based simulation setup

- Three-floor terraced house
 - light-weight construction style (CLT)
 - 4 different orientations
 - suburban and urban environment (using ASHRAE correction for wind speed)
- Belgian climate data with extremes from 2000-2009
- Three-person family
- Ventilative cooling:
 - Manual occupant window opening
 - Temperature-controlled
 - single-sided ventilation (1)
 - cross-ventilation (1)(2)
 - thermal stack ventilation (1)(3) (*)

(*) an open staircase is assumed



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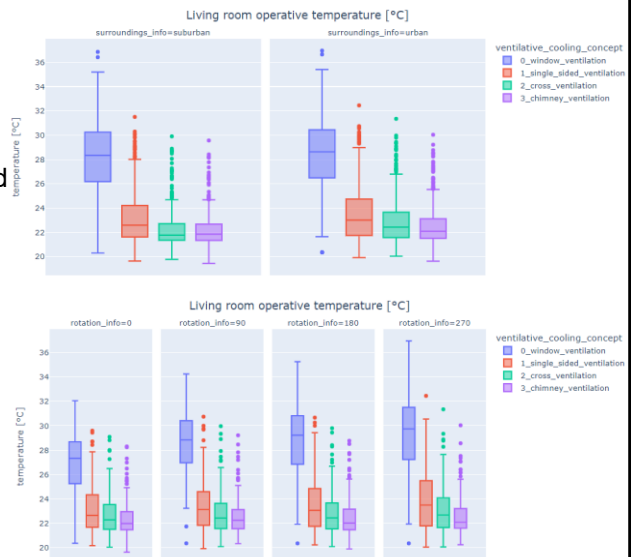
DUCO

3

Results

Robustness to external conditions (May-September)

- Cross-ventilation and single-sided ventilation have higher living room temperatures and spread in urban environments
- Thermal stack ventilation has the lowest living room temperatures and spread independent of (sub)urban environment and building orientation



HOME OF OXYGEN

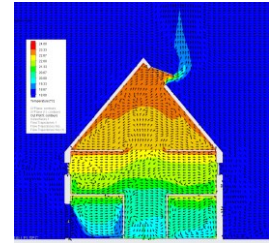
DUCO

4

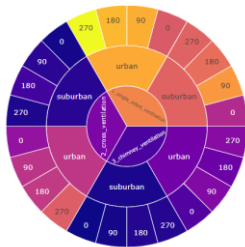
Conclusion

What ventilative cooling concepts to use in residential buildings with light-weight construction?

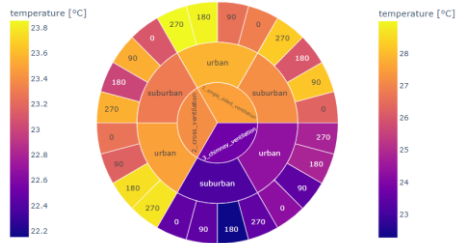
- Thermal stack based ventilation concept:
 - is generally robust to external conditions such as the surroundings and building orientation
 - also reduces bedroom temperatures significantly during hot periods




Average living room operative temperature



Average bedroom 1 operative temperature






Performance 2

Durability of ventilation performance

Performance 2 project - Winter IAQ campaigns in 13 dwellings equipped with Humidity-based DCV systems: analysis of the ventilation performance after 15 years of use

Adeline Mélois, Ambre Marchand Moury, Marc Legree, Juan Rios, Jérémy Depoorter, Nicolas Dufour, Sylvain Rebières, Gaëlle Guyot



1



Performance 2 project (2020 – 2024)

Aim: qualify the durability of smart ventilation systems with humidity-based demand-controlled ventilation, and especially their resiliency regarding long-term use by various tenants



Paris Building
19 instrumented dwellings



Villeurbanne (Lyon) Building
12 instrumented dwellings

monitored since their construction during the 2007-2010 Performance project
→ in situ sensors in the air terminal devices (directly inside the ATD and with circuit board near the ATD)



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2



Performance 2 project (2020 – 2024)

- **Continuous monitoring + 2 winter campaigns on-site:**
 - Preliminary inspection of ventilation systems
 - Measurements in dwellings:
 - ventilation performance
 - comfort parameters (T°, RH)
 - indoor air quality (CO₂, VOCs, Formaldehyde & PM)
 - Interviews of the tenants
 - Outdoor weather stations
- **Laboratory campaign:**
 - evaluation of the air terminal devices' performance before and after cleaning
 - calibration of the sensors and study of the reliability of the indoor air quality sensors
- **Analysis:**
 - assessment of the performance of the ventilation systems regarding indoor air quality, energy input, and their robustness compared to their use by the tenants



3



Analysis - Methodology

1) Durability evaluation : comparison Performance 1 / Performance 2

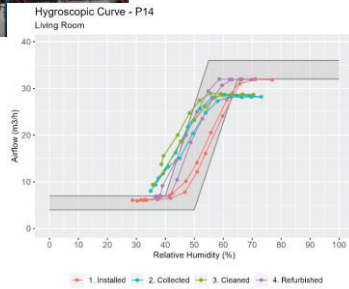
4



Analysis - Methodology

1) Durability evaluation : comparison Performance 1 / Performance 2

- In laboratory



Mélois et al. Durability of humidity-based ventilation components after 13 years of operation in French residential buildings – assessment of components performance in laboratory. *Energy and Buildings*, 2023.

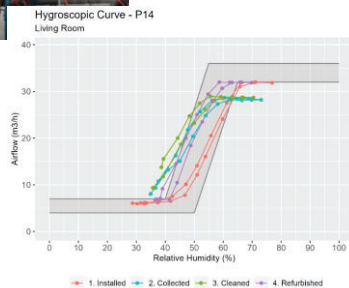
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Analysis - Methodology

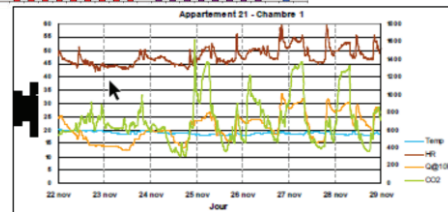
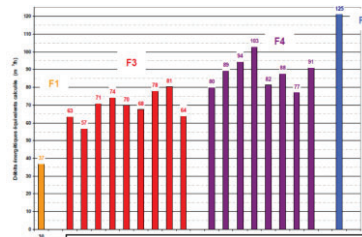
1) Durability evaluation : comparison Performance 1 / Performance 2

- In laboratory



Mélois et al. Durability of humidity-based ventilation components after 13 years of operation in French residential buildings – assessment of components performance in laboratory. *Energy and Buildings*, 2023.

- On-site



6



Analysis- Methodology

2) Performance regarding IAQ: new evaluations

Parameter	Sensors	Limits	Analyses
Formaldehyde	NEMOs	<ul style="list-style-type: none"> Average values for 2h 	?
Total VOC	Embedded sensor	<ul style="list-style-type: none"> Which VOCs? Unit? 24h Auto calibration 	Only dynamic analyses
Light VOC	NEMOs	<ul style="list-style-type: none"> Which VOCs? Unit? High uncertainty 	Only dynamic analyses
PM2.5	Embedded sensor	<ul style="list-style-type: none"> High uncertainty 	Descriptive statistics with precaution
	NEMOs	<ul style="list-style-type: none"> High uncertainty 	Descriptive statistics with precaution
PM10	NEMOs	<ul style="list-style-type: none"> Extrapolated from PM25 	No possible analysis
PM1	NEMOs	<ul style="list-style-type: none"> High uncertainty 	Descriptive statistics with precaution



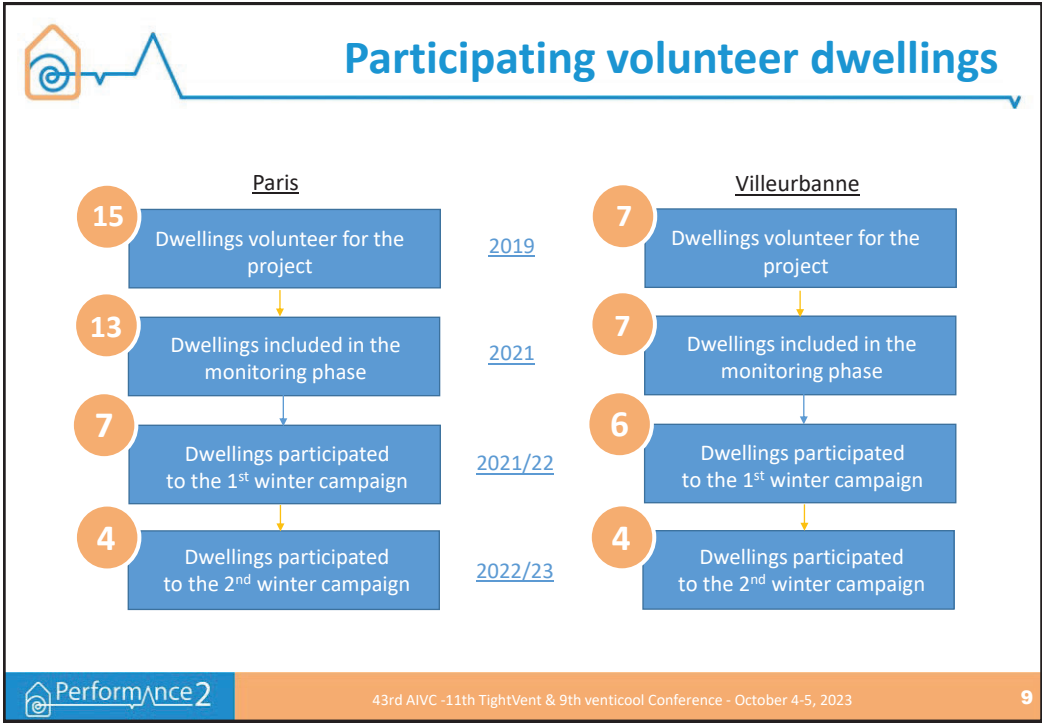
Analysis- Methodology

3) Evolution of ventilation CONDITIONS & NEEDS

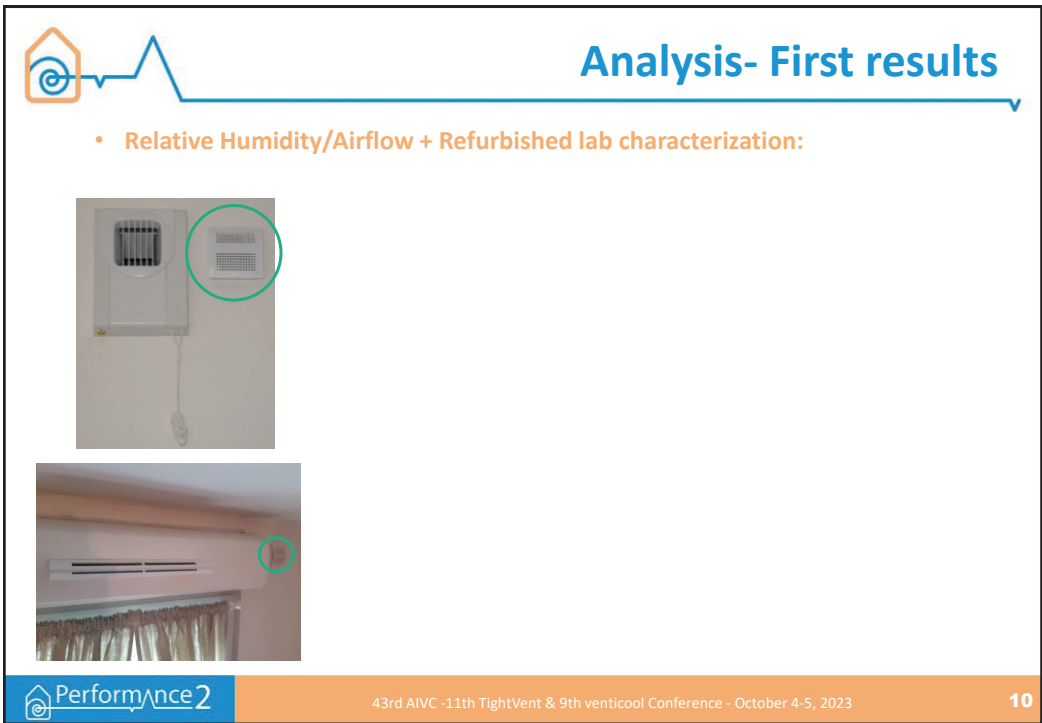
- Diagnosis of the whole ventilation system
- Evaluation of the evolution of occupancy
 - Number of occupants
 - Age
 - Home office
- Evaluation of the airing behaviour
- Evaluation of the internal sources of pollutants



	Bricolage, Loisirs, Bien-être	Cuisine					
		Heure début	Heure fin	Heure début	Heure fin	Heure début	Heure fin
Activité	Parfums d'ambiance						
	Bricolage						
	Peinture						
	Sport						





9




10

Analysis- First results

- Relative Humidity/Airflow + Refurbished lab characterization:

**CO₂, RH, T, PM_{2.5} and VOC +
Shutter Opening and Pressure***




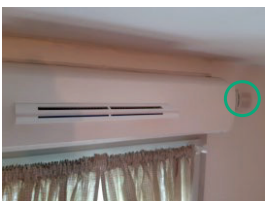
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11

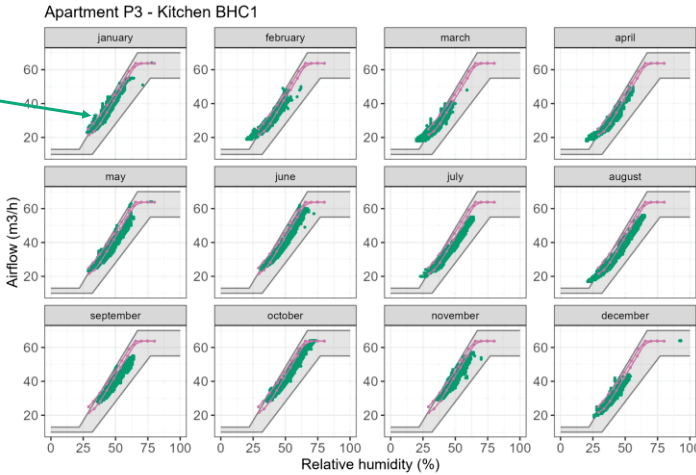
11

Analysis- First results


- Relative Humidity/Airflow + Refurbished lab characterization:

Apartment P3 - Kitchen BHC1



— Monitoring — Refurbished - Lab



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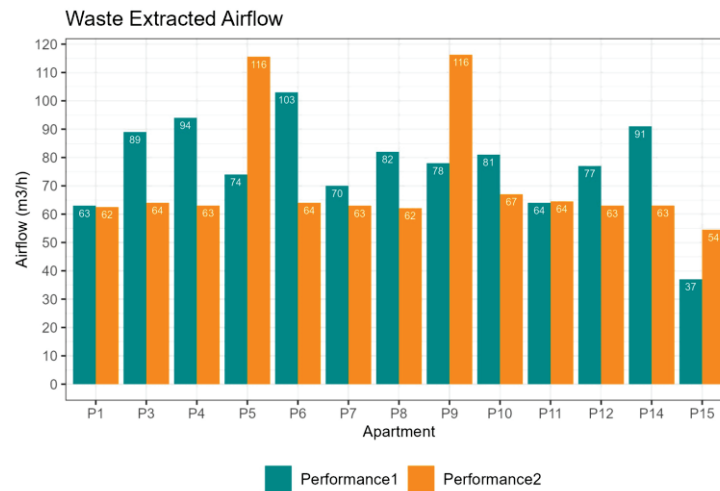
12

12



Analysis- First results

- Comparison Performance 1 et Performance 2 wasted airflow:

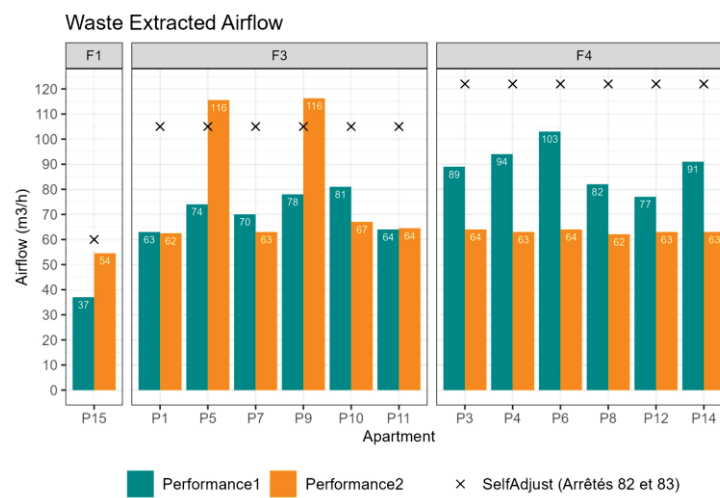


13



Analysis- First results

- Comparison Performance 1 et Performance 2 wasted airflow:

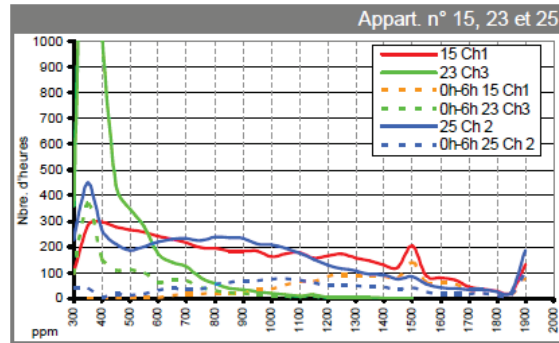


14



Analysis- First results

- Time per average CO2 concentration:

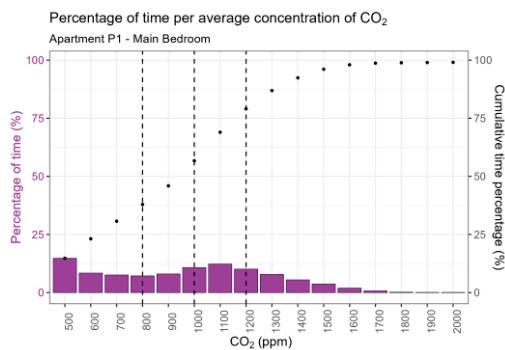


Performance 1 → More spent time at higher average CO2 concentration values when considering only 0h and 6h than for a 24h period



Analysis- First results

- Time per average CO2 concentration:

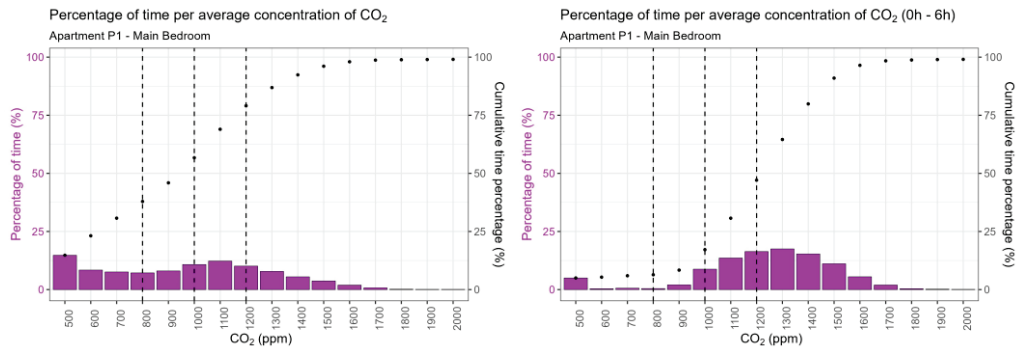


Performance 1 → More spent time at higher average CO2 concentration values when considering only 0h and 6h than for a 24h period



Analysis- First results

- Time per average CO2 concentration:



Performance 1 → More spent time at higher average CO₂ concentration values when considering only 0h and 6h than for a 24h period
Performance 2 → Same tendency

17



Analysis- First results

- Through the detection of humidity (intrinsically human origin), air inlets allow an increase in airflow during increases in CO₂ levels ?

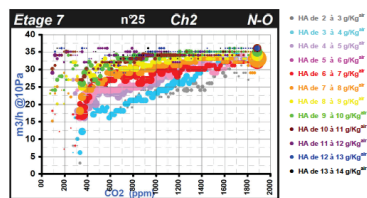


Figure 21 - exemple de l'ouverture moyenne de l'aérateur d'un appartement occupé par 4 enfants sur un an

Performance 1 → General tendency of opening of air inlets with the rise in the CO₂ level

18



Analysis- First results

- Through the detection of humidity (intrinsically human origin), air inlets allow an increase in airflow during increases in CO2 levels ?

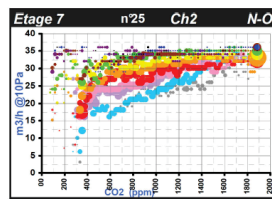
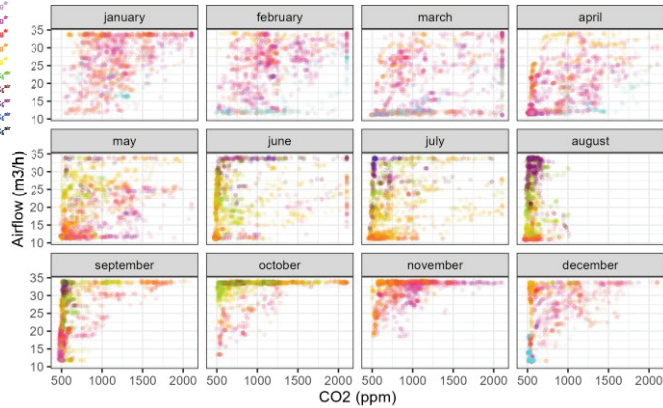


Figure 21: exemple de l'ouverture successive de l'aérateur d'un appartement occupé par 4 résidents sur six ans

Performance 1 → General tendency of opening of air inlets with the rise in the CO2 level

Performance 2 → To verify...

Relationship between Airflow and CO2
Apartment P3 - Main Bedroom

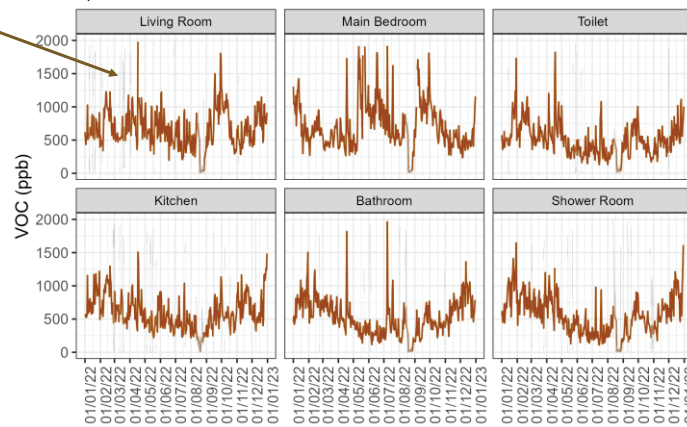


Analysis- First results

- Study of the VOC temporal dynamic evolution: uncertainties on the absolute values after lab characterization



Min, Max and Median value per day - VOC
Apartment P4





Analysis- First results

- Comparison AERECO/NEMO sensors: Particle Matter (PM) 2.5



AERECO sensors →
Installed in all the rooms -
1 record every minute for 2
years



NEMO sensors → 2 IAQ
campaigns of 2 weeks each
installed in the main
chamber and living room -
1 record every 10 minutes

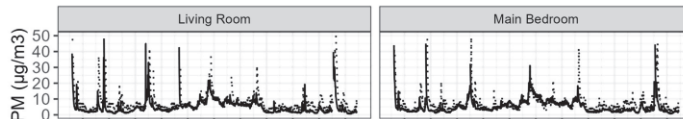


Analysis- First results

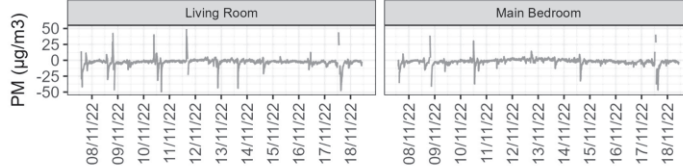
- Comparison AERECO/NEMO sensors: Particle Matter (PM) 2.5




Comparison AERECO/NEMO
Apartment P14 - Campaign II



— AERECO ···· NEMO




— AERECO/NEMO Difference



Next steps

- **Evaluation of the durability from on-site measurements**
 - Comparison with design documents
 - Comparison with Performance 1 project
- **Evaluation of actual performance of ventilation system regarding IAQ and Energy**
 - Indicators for IAQ performance
 - Evaluation of the humidity based system for others pollutants
- **Evaluation of the impact of the occupants**
 - Data collected with interviews
 - Data extrapolated from measurements
- **End of the project: June 2024**
- **Many data: others studies may be performed after Performance 2 project**


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23

23



Thank you for your attention

performance2@cerema.fr














24



Checking and assuring real IAQ and energy performances through demand control and cloud connectivity



Creating healthy spaces

VENTILATION | SUNPROTECTION | OUTDOOR

Ivan Pollet, Kevin Verniers, Steven Delrue

R&D Renson

AIVC Conference, October 4-5, 2023, Copenhagen

1

The Renson story about the company



FACTS & FIGURES

- Belgian family business, since 1909
- Headquarters in Waregem (Belgium)
- Core businesses
 - Ventilation
 - Sun shading
 - Outdoor living solutions
- 1200 employees, 10% employed in R&D
- Annual turnover of over 220 Mio €

MISSION

Creating happiness and well-being
by bringing people a healthy and comfortable
in- & outdoor environment

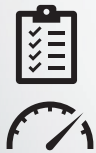
2

Renson Ventilation Strategy (residential)



☐ Ventilate **sustainably** > limitation of **material use & energy use**

- ✓ Minimize **mass & number of components**
- ✓ Maximize **repairability & recyclability**
- ✓ Ventilate **where** and **when** needed on a **sufficient** level
- ✓ Avoid '**overventilation**' except for **ventilative cooling**
- ✓ Improvement **Total Life Cycle**



☐ Enhance **servitization**:

- ✓ **Unburden: support & facilitate practice:** planning/installation/commissioning/service
- ✓ **Visualize + communicate + sensitize** about the invisible **IAQ**

Connectivity highly facilitates

3

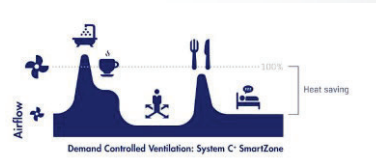
Renson specializations



Control systems: - advanced
- demand driven: RH, CO₂, VOC, T, I_{sol} ...

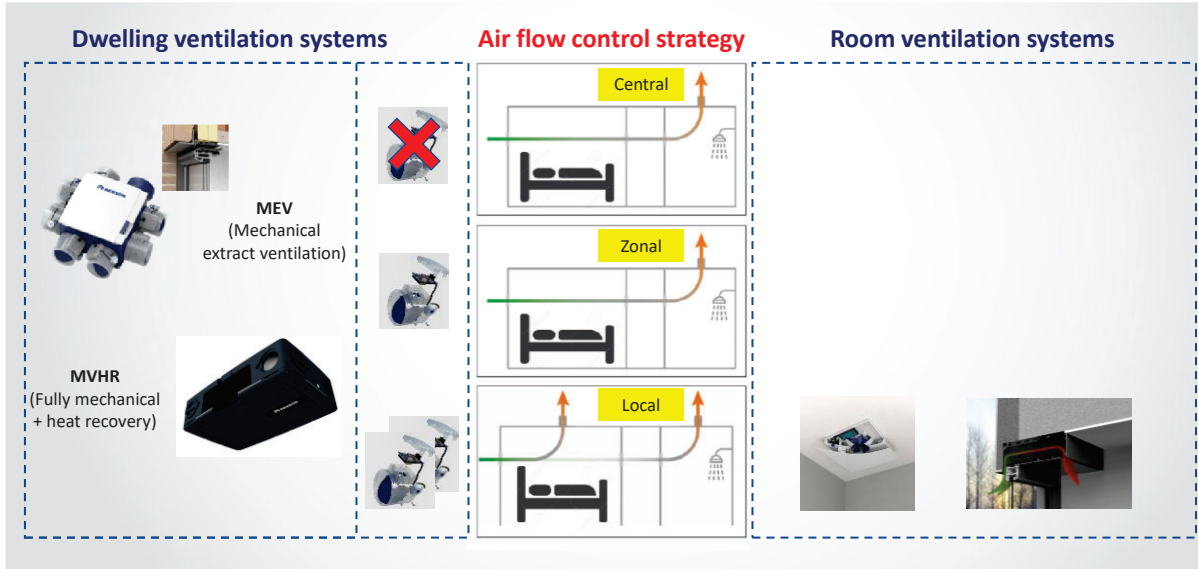
Smart Living

IoT devices: cloud connectivity & Data



4

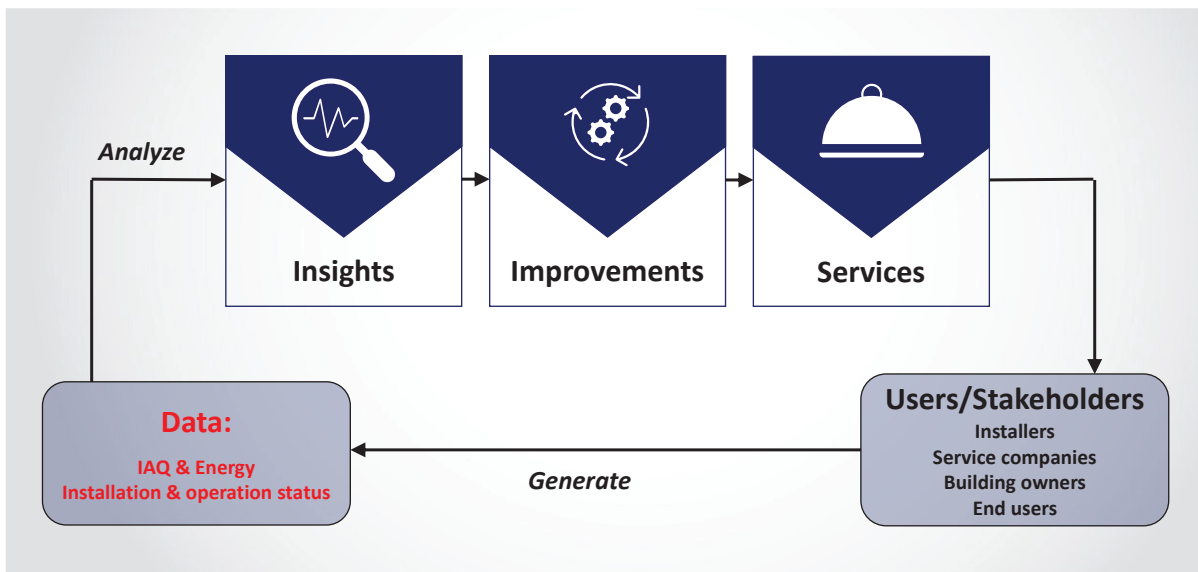
Demand controlled ventilation strategy



5

Big data in construction

Role of data in smart buildings



6

Big data in construction > Servitization

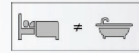
Role of data in smart buildings



Current servitization through digital interactions:

- With residents/installers: via resident/set-up app & web portals
 - **Project preparation & planning** > installation, system calibration & commissioning
 - **Push notifications** > via smartphone for filter warnings and error detection
 - **Visual representation** > real time/historic air quality and ventilation levels
 - **Service management and follow-up** > remote inspection of functioning/maintenance
 - **Automatic software updates**
 - ...

- With smart home/home automation: via API and/or via switching module



Wrong valve detection

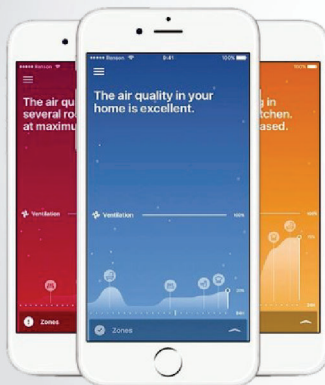
7

Big data in construction > Insights

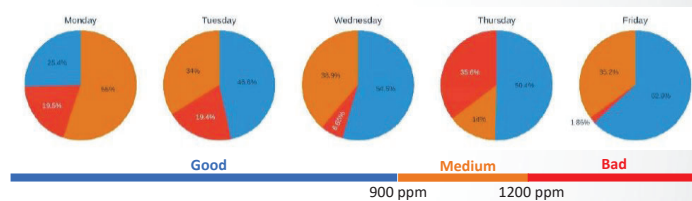
Inform occupants about air quality and energy consumption



Mobile & web applications

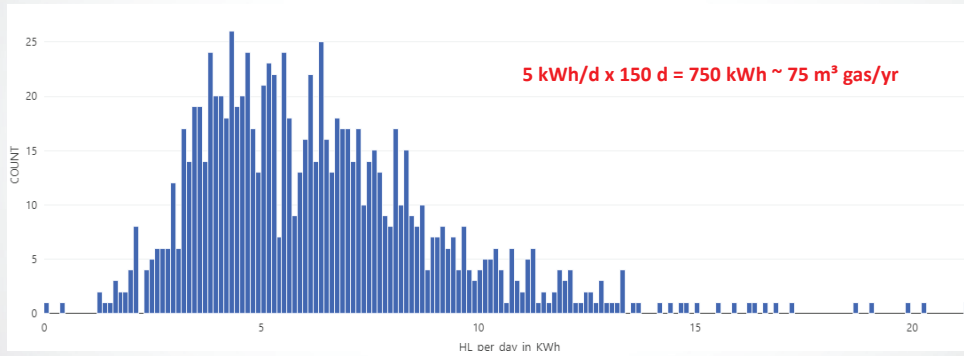


Reporting with statistical summaries



8

Big data in construction > Insights – ventilation losses



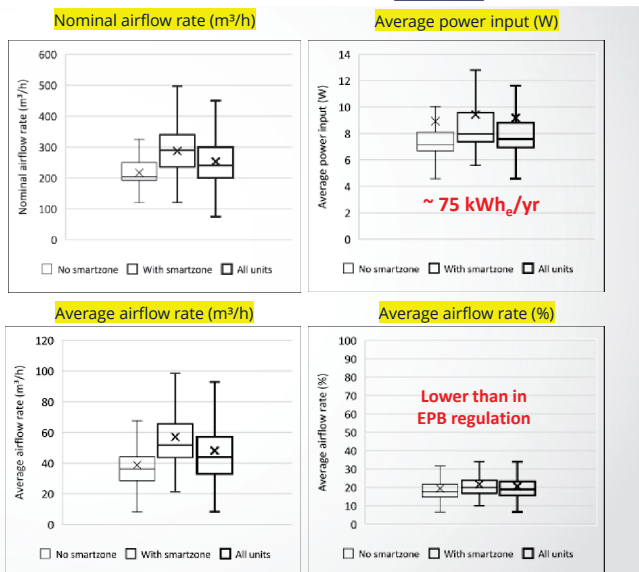
Ventilation heat losses caused by Healthbox 3.0

Big data in construction > Insights – prove our claims



Articles This article is based on a paper presented at the 40th ASHRAE - 4th TightenIt & 4th Ventilation Conference "From energy crisis to sustainable indoor climate" 15-16 October 2018, Ghent, Belgium

Cloud based large-scale performance analysis of a smart residential MEV system



Big data in construction > Insights – prove our claims

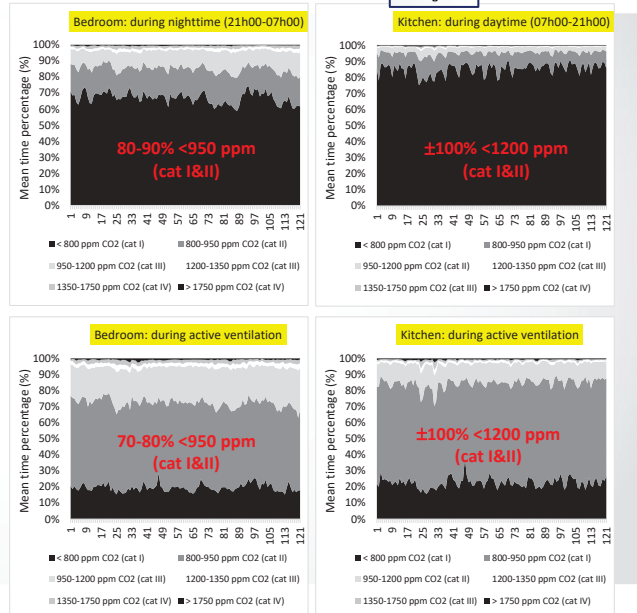


Articles This article is based on a paper presented at the 48th AIVC - 8th Tightest & 6th Ventecol Conference "From energy crisis to sustainable indoor climate" 15-16 October 2016, Ghent, Belgium

Cloud based large-scale performance analysis of a smart residential MEV system

Zonal control

Local control



11

Big data in construction > Insights – design heating systems



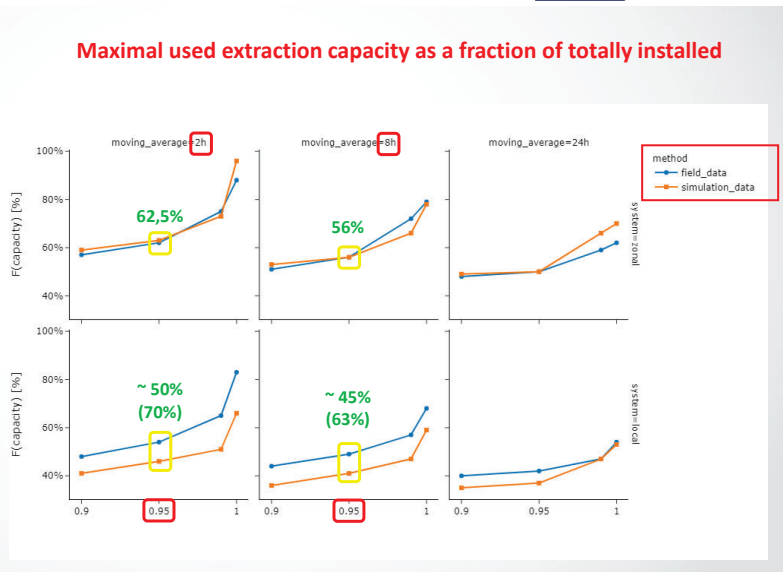
IAQ 2020: Indoor Environmental Quality Performance Approaches

Performance Analysis of the Maximal Used Extract Ventilation Capacity of Dwellings During the Heating Season

Zonal MEV

Local MEV

Large-scale field study
+
Dynamic simulation study



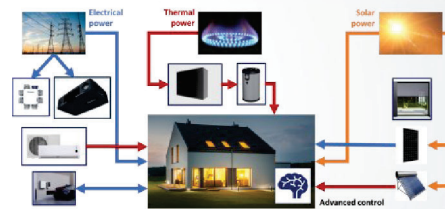
12

Future



Creation of Smart Living concept: HVAC, solar shading, PV, ...

- Cloud connection
- Overall & advanced controls
- Increase/guarantee performance
- Facilitate maintenance & repair



Conditions:

- **Sustainable:** minimize environmental impact by 'collaboration'
- Operate within the **security and privacy rights**
- **Added value** > Cost

13

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Creating healthy spaces

VENTILATION | SUNPROTECTION | OUTDOOR

14



Arbeitsbereich für
Energieeffizientes Bauen



Data driven models for fault detection

Combining thermal and indoor air quality grey box models

Gabriel Rojas, University of Innsbruck
 Romed Jenewein, University of Innsbruck
 Klaus Prenninger, Salzburg University of Applied Sciences
 Johannes Schnitzer, Forschung Burgenland

AIVC Conference, Copenhagen
 Oct. 4th, 2023

AFOM - Automatisierte Fehler & Optimierungsanalyse durch Messwerterfassung







1

Typical faults

	System renovation	Maintenance and repair
	Design	Construction
	Operation	
<ul style="list-style-type: none"> - Over-sizing, under-sizing - Unbalanced water and air system - Inefficient air distribution in rooms 	<ul style="list-style-type: none"> - Damage of building envelope - High resistance of water and air system caused by construction issues - Inefficient room heating and cooling, e.g. air outlet, radiator being covered - Wrong sensor location 	<ul style="list-style-type: none"> - Human wrong behavior - Wrong set point, schedule - Override command - Open windows and doors - Control faults - Controller breaks down - Control algorithm error - Wrong parameters - HVAC equipment and BAS field devices faults - Motors and electric devices fail (fan, pump, heater, etc.) - Dampers/valves stuck, leak, reversed - Sensors fail - Heating and cooling equipment fail (heat pump, boiler, chiller, etc.)

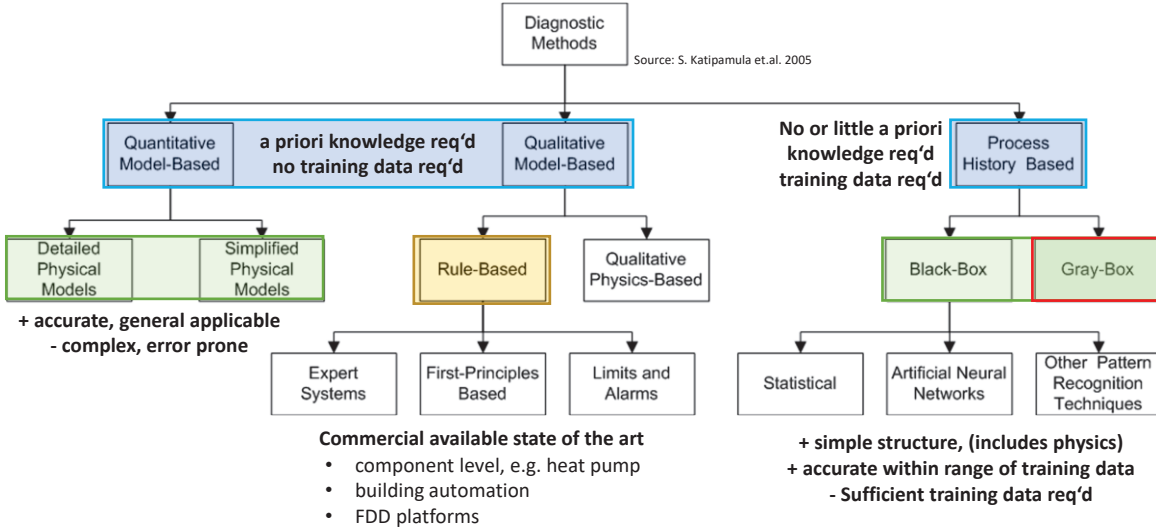
Source: Gao, T. (2020)



Seite 2

2

Fault Detection and Diagnosis (FDD) methods

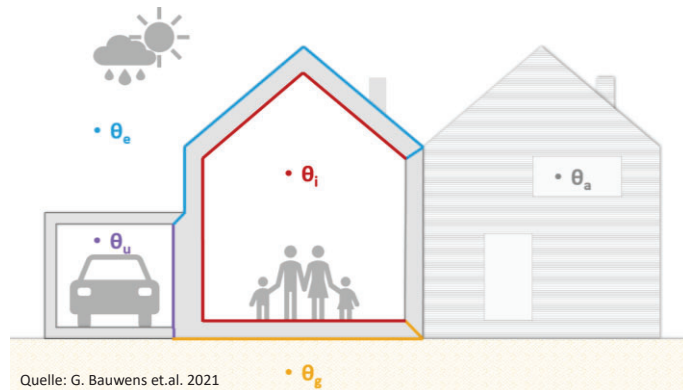


3

Thermal grey box model

Basic idea: energy balance

- Transmission losses
 - Infiltration
 - Solar gains
 - Internal heat gains
 - Ventilation
 - Heating / Cooling
- $\tilde{\Phi}_{tr}$
- Φ_{sol}
 Φ_{int}
 Φ_{vent}
 Φ_{heat}



Stationary $\sum \Phi = 0$

Dynamic $\sum \Phi = C \frac{dT_i}{dt}$

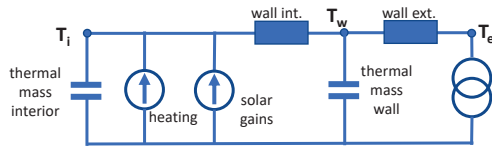
-> Introduce simple model for transmission heat losses

$$\frac{dT_i}{dt} C_i = \Phi_{heat} + \Phi_{sol} + \frac{(T_e - T_i)}{R_{wi}}$$

4

Thermal greybox model using state space model representation

Simplified physical representation



$$\frac{dT_i}{dt} C_i = \dot{Q}_{heat} + \dot{Q}_{sol} + \frac{(T_w - T_i)}{R_{wi}}$$

$$\frac{dT_w}{dt} C_w = \frac{(T_i - T_w)}{R_{wi}} + \frac{(T_e - T_w)}{R_{we}}$$

State space representation (LTI system)

$$\dot{x} = A \cdot x + B \cdot u$$

$$y = C \cdot x + D \cdot u$$

$$x = \begin{pmatrix} T_i \\ T_w \end{pmatrix} \quad u = \begin{pmatrix} T_e \\ \dot{Q}_{heat} \\ \dot{Q}_{sol} \end{pmatrix}$$

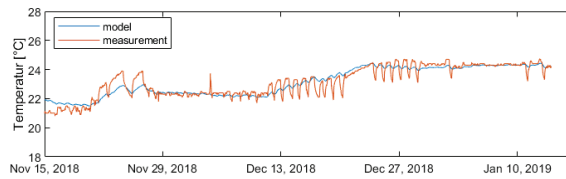
$$A = \begin{bmatrix} -\frac{1}{C_i R_{wi}} & \frac{1}{C_i R_{wi}} \\ \frac{1}{C_w R_{wi}} & -\left(\frac{1}{C_w R_{wi}} + \frac{1}{C_w R_{we}}\right) \end{bmatrix}$$

$$B = \begin{bmatrix} \frac{1}{C_i} & \frac{1}{C_i} & 0 \\ 0 & 0 & \frac{1}{C_w R_{we}} \end{bmatrix}$$

$$C = \begin{bmatrix} 1 & 0 \end{bmatrix}$$

$$D = \begin{bmatrix} 0 & 0 & 0 \end{bmatrix}$$

Use existing parameter identification tools (MATLAB)



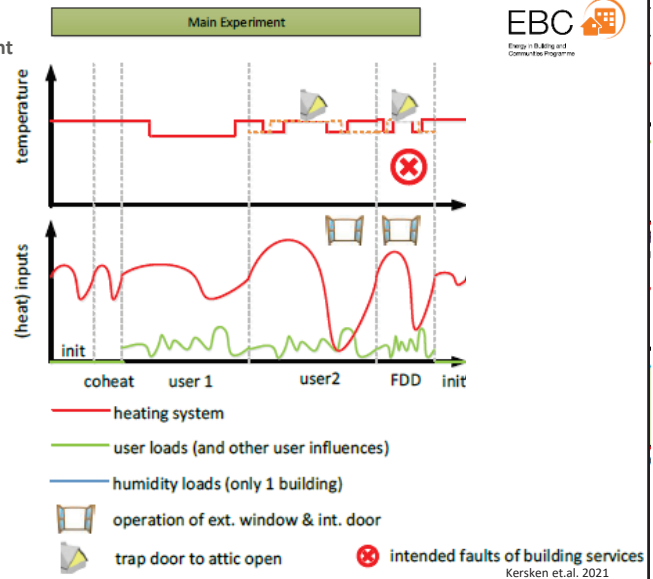
5

Thermal grey box models for FDD

see IEA EBC Annex 71 – Building energy performance assessment based on in-situ measurements

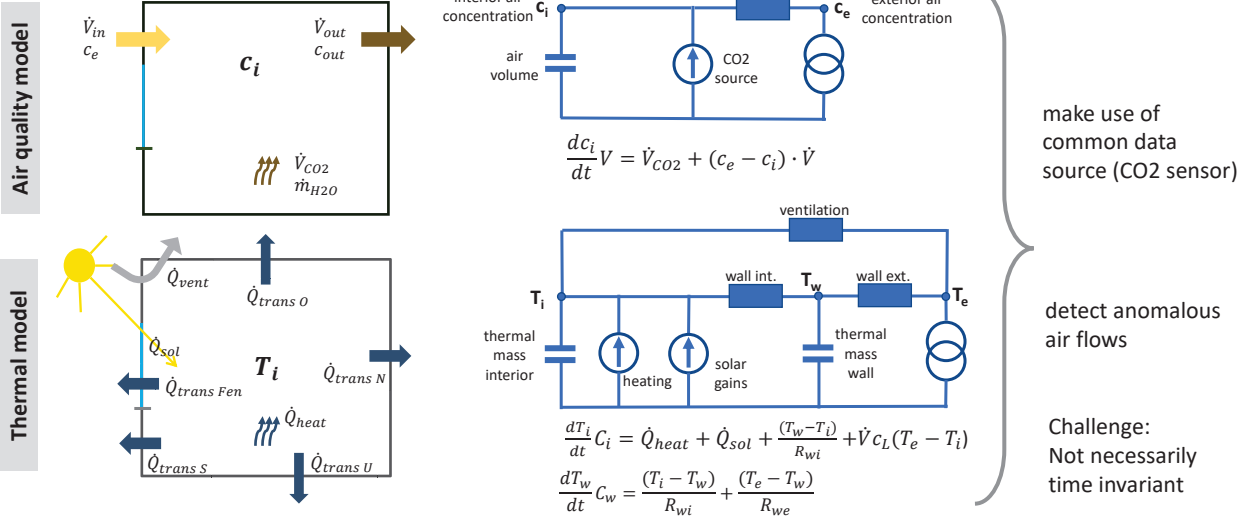


Reynders et al. 2021



6

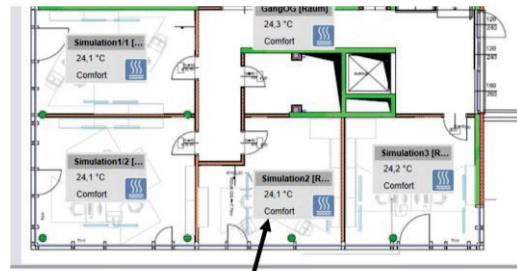
Idea: Combine thermal building model with air quality model



7

Preliminary testing of this IEQ grey-box model for FD

Fault		Start	Duration
Temperature sensor faulty	Sensor outputs +2 K (artificial)	2022-12-01 11:15	60 hrs
CO ₂ sensor faulty	Sensor outputs +200 ppm (artificial)	2022-02-24 11:15	60 hrs
Window left open	Window was left open unintentionally (real occurrence)	2022-09-29 7:40	8 hrs
		2022-09-30 7:40	6 hrs

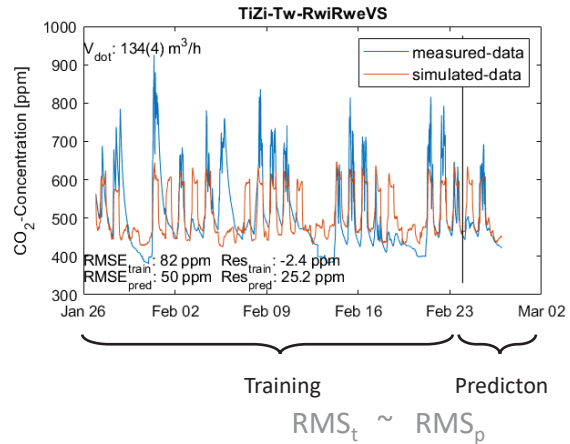
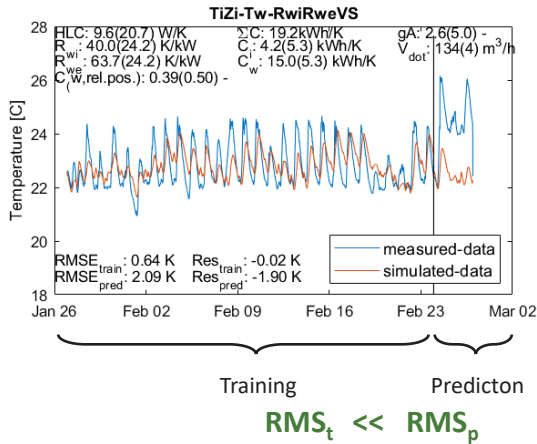


Room „02“

Can this model be used to detect „faults“ (anomalies)?
How does it perform compared to other prediction models?

8

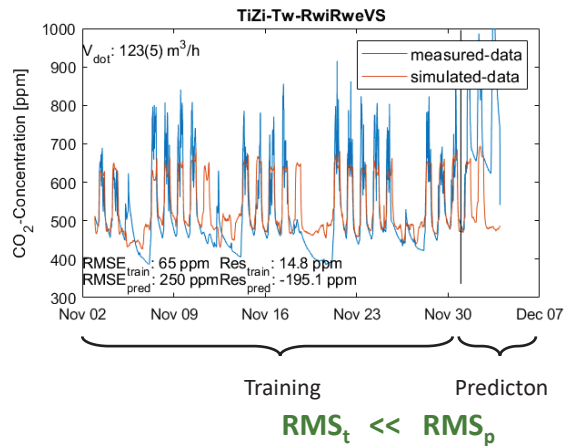
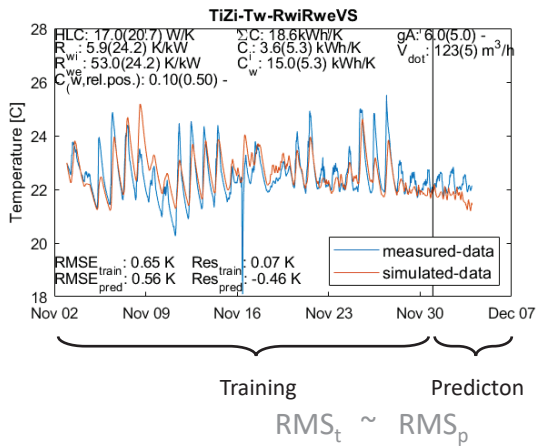
Testing this IEQ grey-box model for FD - preliminary results T-sensor fault: +2K



Simulation error (RMS) depends on „how accurate the occupancy was modelled“

10

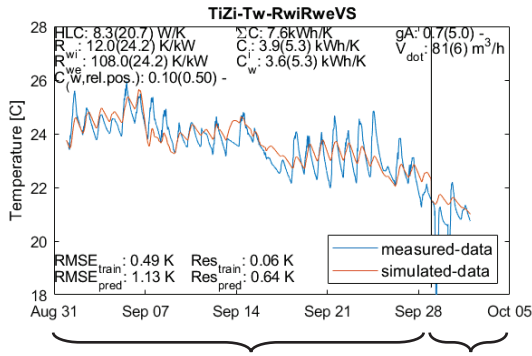
Testing this IEQ grey-box model for FD - preliminary results CO2-sensor fault: +200 ppm



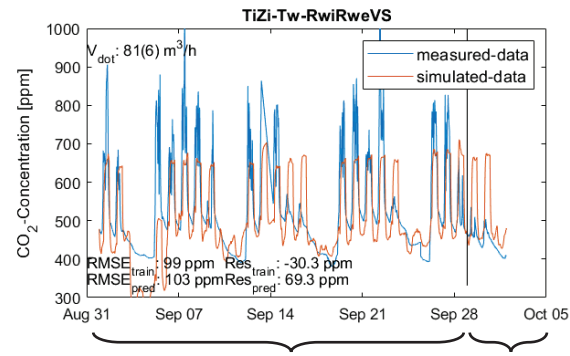
RMS high because of inaccurate occupancy modelling

11

Testing this IEQ grey-box model for FD - preliminary results Window open



Training Prediction
 $RMS_t \ll RMS_p$



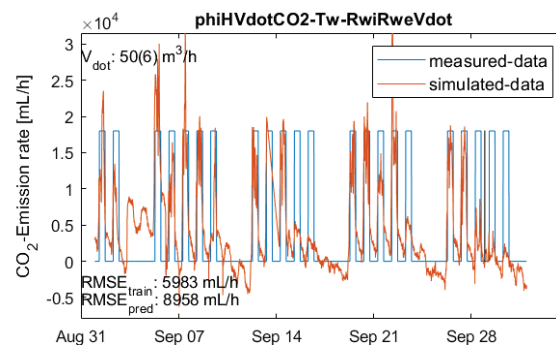
Training Prediction
 $RMS_t \sim RMS_p$

RMS high because of inaccurate occupancy modelling

12

Summary

- Presented & tested idea on how therman grey box model can be extended to include CO2 conc.
- Operational data used to train a physics based model (parameters have physical meaning)
- Models can be used to predict T and CO2 when trying to detect anomalies
- Variations are possible: output energy use, emissionsrate, (i.e. occupancy)...
- Further work is needed:
 - test with more realistic faults
 - improve identification procedure
 - extend model with further variables (e.g. rH)
 - deal with time invariant volume flow



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universität innsbruck
Arbeitsbereich für Energieeffizientes Bauen



TECHNISCHE UNIVERSITÄT WIEN



FH Salzburg
Smart Building



FORSCHUNG Burgenland
RESEARCH & INNOVATION

Austrian National Project
AFOM - Automatisierte Fehler & Optimierungsanalyse durch Messwerterfassung



Thank You!



Bundesministerium
Verkehr, Innovation und Technologie



IEA FORSCHUNGS KOOPERATION

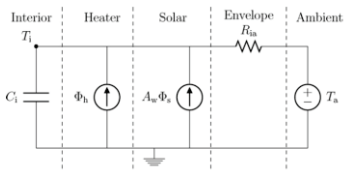



FFG
Forschung wirkt.

www.uibk.ac.at/bauphysik

14

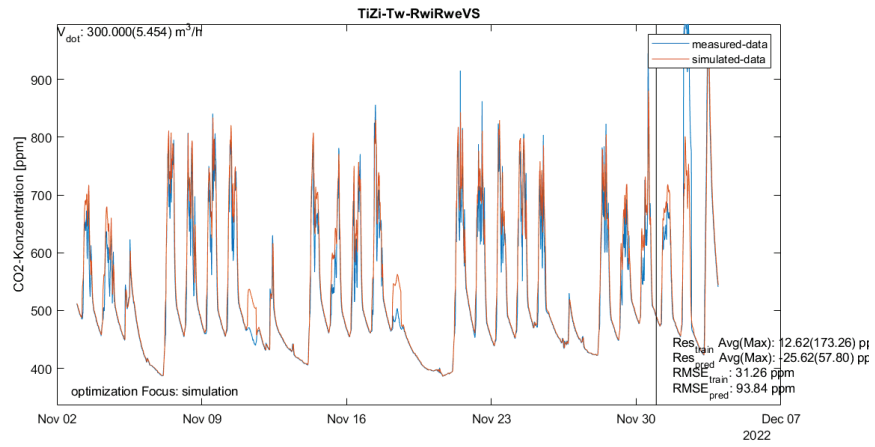
Building models in „different shades of grey“

White-Box	Grey-Box	Black-Box
<p>Physical parameters material, geometry, construction,...</p> <p>physics Input data/BC (weather, ...)</p> <div style="border: 1px solid black; padding: 5px; width: fit-content; margin: 10px auto;"> <p>EnergyPlus TRNSYS IDAICE Modelica ...</p> </div> <p>Predicts all states in any BC</p>	<p>Training data (e.g. temperature, heating)</p> <p>physics simplified Input data/BC (weather, ...)</p>  <p>Predicts certain state(s) within limited BC</p>	<p>Training data (e.g. temperatur or heating)</p> <p>math Input data/BC (weather, ...)</p> <div style="background-color: #333; color: white; padding: 5px; width: fit-content; margin: 10px auto;"> <p>Linear Regr. ARX / ARMAX ANN SVM Random forest ...</p> </div> <p>Predicts certain state(s) within BC strictly limited to training data</p>
<p>← Physical parameter identification Quality control of as-built condition →</p>		<p>← Building behaviour identification Model predictive control, Fault detection and diagnosis →</p>


 Accuracy and use of such models was investigated in <https://www.iea-ebc.org/projects/project?AnnexID=71>

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Model with adaptable volumeflow



Evaluation of supply temperature set-points and airflow imbalance using smart ventilation data

3.10.2023

AIVC 2023 Conference Copenhagen

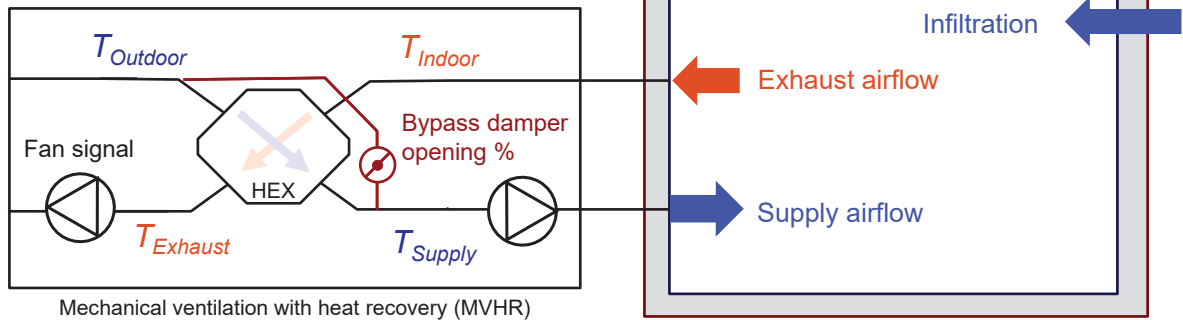
Topical Session - Real performance of (smart) residential ventilation – performance assurance, fault detection, continuous commissioning

Kevin M. Smith, Technical University of Denmark, kevs@dtu.dk

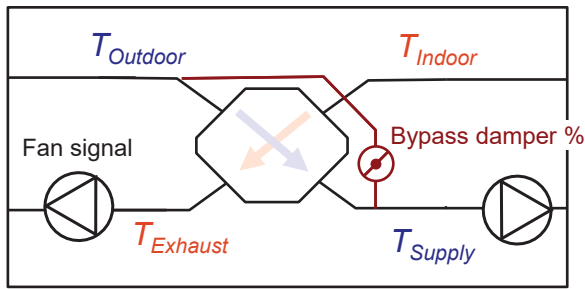
Jakub Kolarik, Technical University of Denmark

1

Common data sources from MVHR units



2

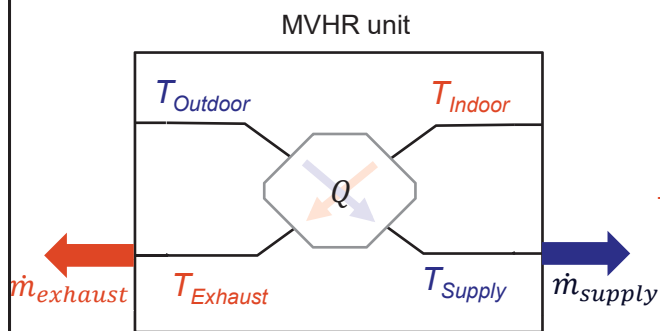


Hypothesis

We use this data to balance airflows and maximise heat recovery

3

Energy balance



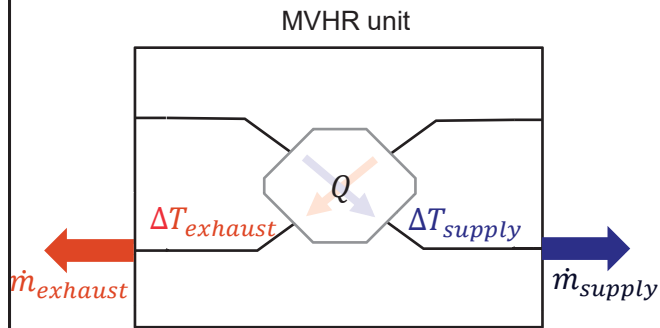
$$Q_{supply} = -Q_{exhaust}$$

$$Q_{supply} = \dot{m}_{supply} c_p (T_{supply} - T_{outdoor})$$

$$-Q_{exhaust} = \dot{m}_{exhaust} c_p (T_{indoor} - T_{exhaust})$$

4

Energy balance



$$Q_{supply} = -Q_{exhaust}$$

$$Q_{supply} = \dot{m}_{supply} c_p \Delta T_{supply}$$

$$-Q_{exhaust} = \dot{m}_{exhaust} c_p \Delta T_{exhaust}$$

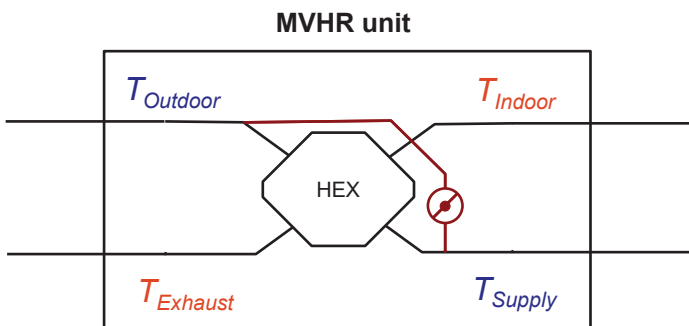
Since, $c_{p_{supply}} \approx c_{p_{exhaust}}$

$$\text{Flow ratio, } Fr = \frac{\dot{m}_{supply}}{\dot{m}_{exhaust}} = \frac{\Delta T_{exhaust}}{\Delta T_{supply}}$$

5

Highest accuracy when...

$$Fr = \frac{\dot{m}_{supply}}{\dot{m}_{exhaust}} = \frac{\Delta T_{exhaust}}{\Delta T_{supply}}$$



Cold weather

- Outdoor < 10°C

No bypass flow

- Damper position ≠ 0

Dry heat exchange

- Exhaust > Indoor dewpoint

6

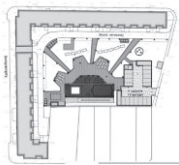
Simple indicator for airflow balance

Flow ratio,
$$Fr = \frac{\dot{m}_{supply}}{\dot{m}_{exhaust}} \approx \frac{\Delta T_{exhaust}}{\Delta T_{supply}}$$

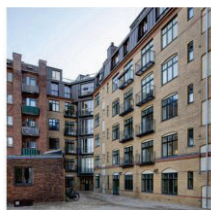
Balanced ventilation in cool climates targeting a slight underpressure

$$0.95 < Fr < 1.0$$

Exploration in two renovation case studies in Denmark



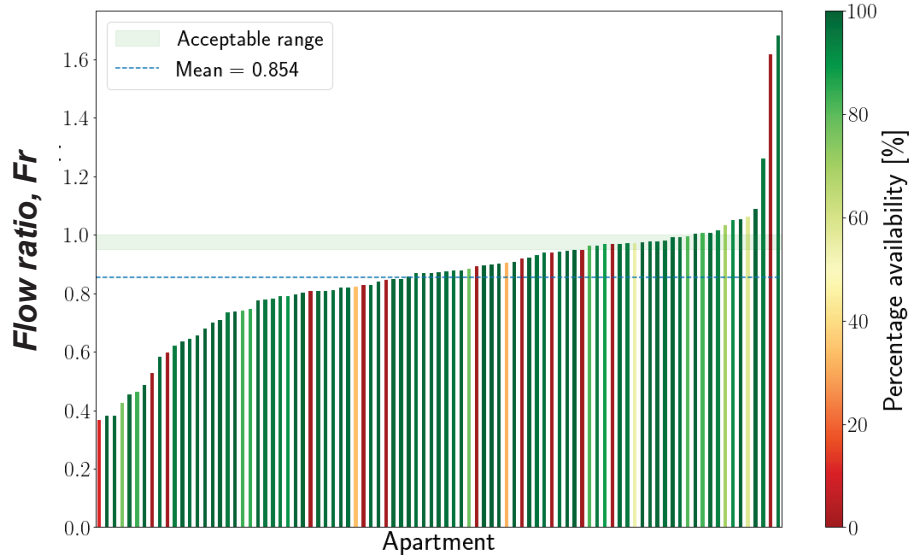
- ~100 public rental apartments
- Expected 50% heat savings



- 42 private rental apartments
- DGNB silver & DK Renovation Class 1

Internet-connected MVHR unit in each apartment

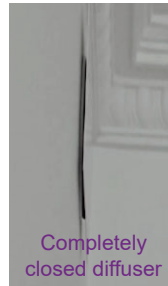
1st case study – Flow ratio in 100 apartments



9

2nd case study

- Observed poor initial commissioning of airflows in 42 apartment MVHR units



- Re-commissioning didn't improve balance of airflows



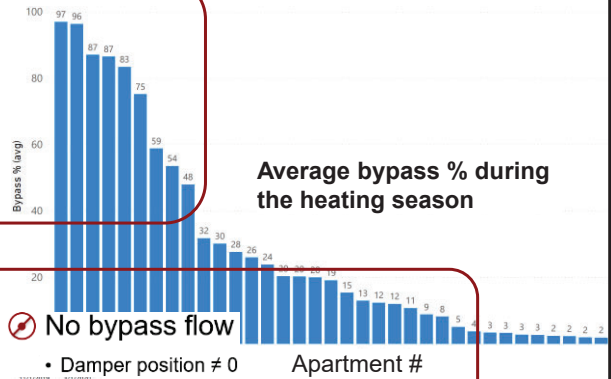
Rum	Luft-mængde [m ³ /h]	Støj dB	Fugt %	EI W
Stue	70			
Køk.	-63			
Bad	-65			
Sove.	48			
Køk.	-52			
Bad	-65		41 %	
Stue	58			
Sove.	42			
Sove.	48			
Køk.	-63			
Bad	-45		49 %	50 W
Stue	48			
Sove.	45			
Køk.	-68			
Bad	-74		40 %	55 W
Stue	58			
Sove.	58			
Køk.	-69			
Bad	-48		45 %	60 W
Stue	65			
Sove.	64			
Køk.	-74			
Bad	-96		41 %	
Sove.	74			
Sove2.	65			
Køk	-76			

10

2nd case study: 42 apartment MVHR units

First check the bypass operation

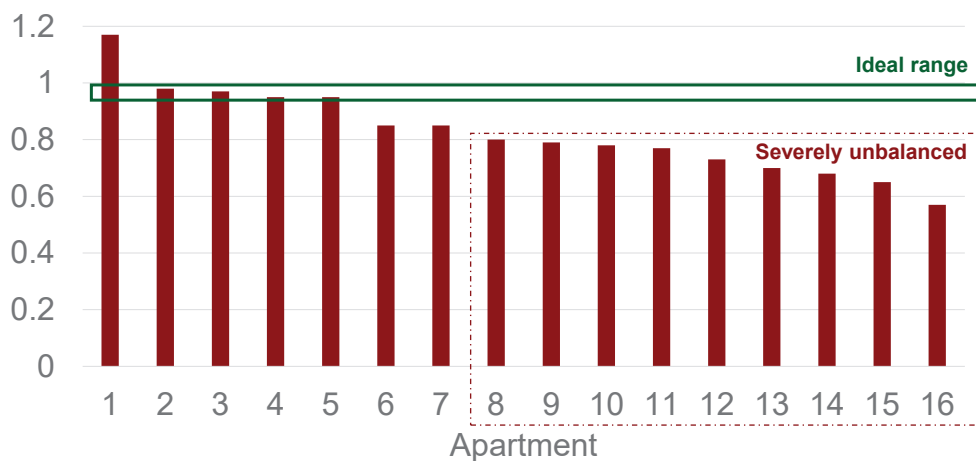
- Power BI dashboard indicated >50% average bypass in many units.
- In one stairwell: The supply-temperature sensor was not mounted in the duct.



- In the apartments with constant partial bypass of the heat exchanger: Need to set a higher supply temperature.

11

Estimated flow ratio, Fr



12

Balancing airflows by adjusting fan speeds

For a given fan, based on the fan affinity laws:

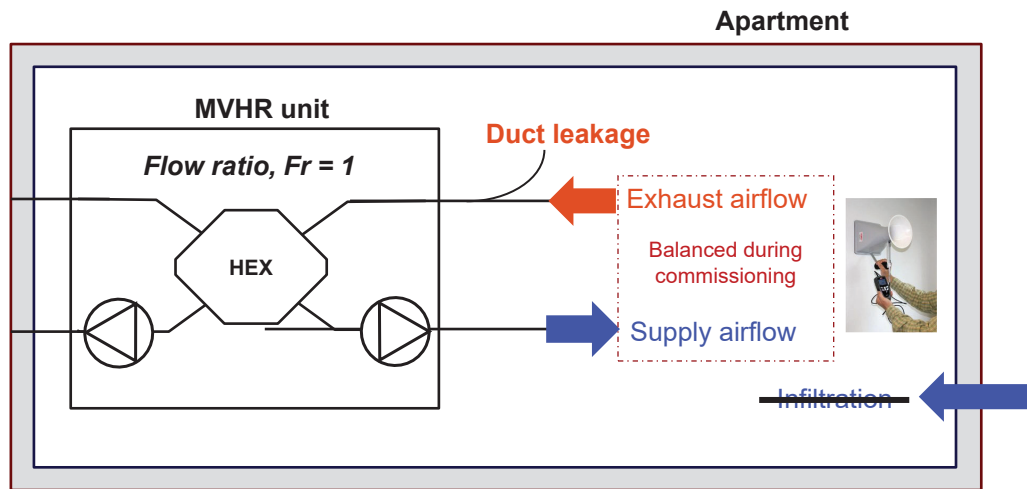
$$\text{airflow} \propto \text{fan speed} \propto \text{fan signal}$$

Adjust: $\left(\frac{0.95}{Fr_{existing}}\right) (fan\ signal)_{supply}$ **OR** $\left(\frac{Fr_{existing}}{0.95}\right) (fan\ signal)_{exhaust}$

Results of the balancing method

Anonymised apartment	Balance (Fr) before	Balance (Fr) after scaling
C1	0.72	1
C5	0.79	0.95
C10	0.78	0.93
E0	0.78	0.97
E1	0.81	0.95
E2	0.7	1
E4	0.65	0.95
Average	0.75	0.96

Advantage: Accounts for duct leakage



15

Risks with automatic balancing

- Masking errors/faults (clogged ducts/valves, closed vents)
- Adjusting the wrong airflow.
Which is the more correct airflow – supply or exhaust?

Closed
diffuser



16

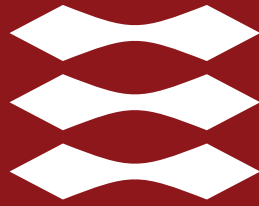
Conclusions

- Typical data from MVHR units can indicate faults and flow imbalances – rarely used in practice
- A scaling method could assist balancing of supply and exhaust – yet to be validated with real heating measurements
- Great care must be taken when automatically balancing to avoid exacerbating issues that demand physical repair

Future work

- Validation
- Clustering analysis
- Fault detection and automatic balancing

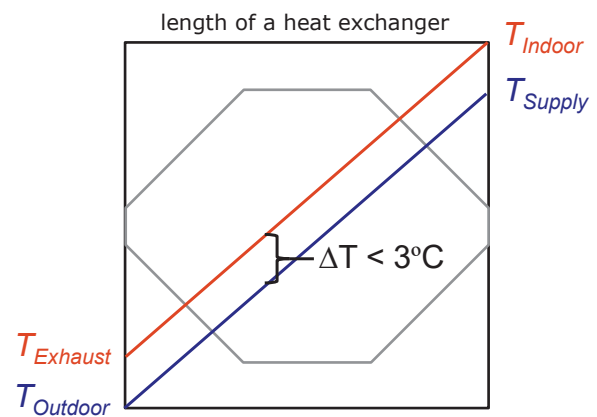
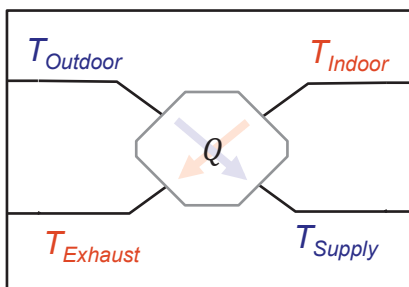
DTU



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Temperatures along the length of a heat exchanger



$$1.00 < \rho_{\text{outdoor}} / \rho_{\text{exhaust}} < 1.02$$

20

Results of data-driven balancing method

Anonymised apartment	Balance (Fr) before	Balance (Fr) after scaling
C1	0.72	1.00
C5	0.79	0.95
C10	0.78	0.93
E0	0.78	0.97
E1	0.81	0.95
E2	0.70	1.00
E4	0.65	0.95
Average	0.75	0.96

AIVC Conference 2023 in Copenhagen

Technologies in balanced ventilation systems to maintain optimal performance in energy and comfort

Bart Cremers
Knowledge Consultant Ventilation technologies



1

Introduction

Observing installed base since 1980's vs. Monitoring performance of currently sold units



Technologies since 2015:

1. Automated fan speed adjustment during commissioning
2. Balance in mass flow rate rather than volume flow rate
3. Automated fan speed correction (FlowControl)
4. Comfort levels instead of values
5. Adaptive comfort technology
6. Modulating bypass technology
7. Season detection
8. Filter change warning

prolonged optimal performance in bringing fresh air, energy savings and comfort

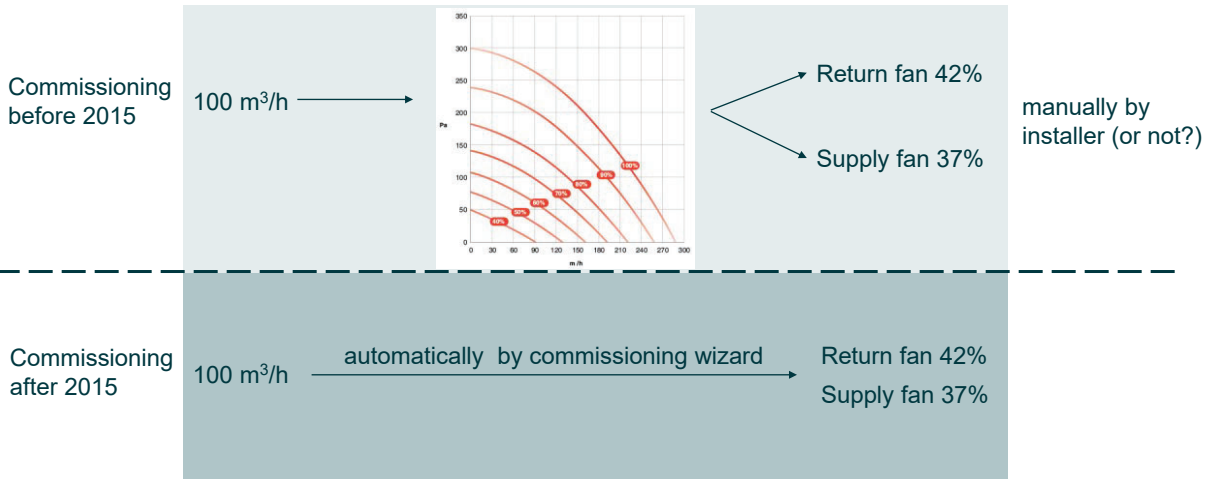
2

always the best climate

2



1. Automated fan speed adjustment during commissioning



3

always the best climate

3



2. Balance in mass flow rate rather than volume flow rate

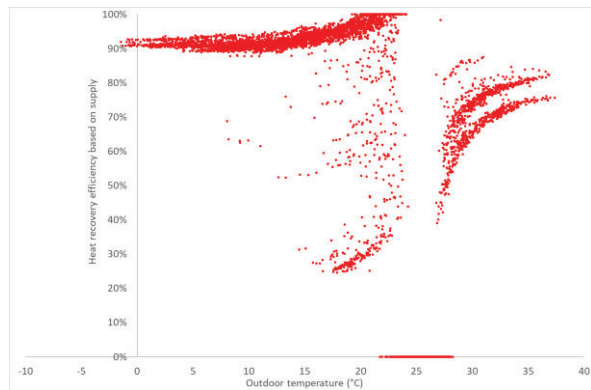
Commissioning before 2015:
 During commissioning of existing units, air volumes are measured, that means that during the year the mass unbalance can easily be 8%(!), giving undesired variations in recovery efficiency

$$\frac{\dot{V}_{supply}}{\dot{V}_{extract}} = 1$$

Commissioning after 2015:

Commissioning is done with volume air flow (m³/h), but unit measures and balances internally in mass air flow (kg/s)

$$\frac{\dot{m}_{supply}}{\dot{m}_{extract}} = 1$$



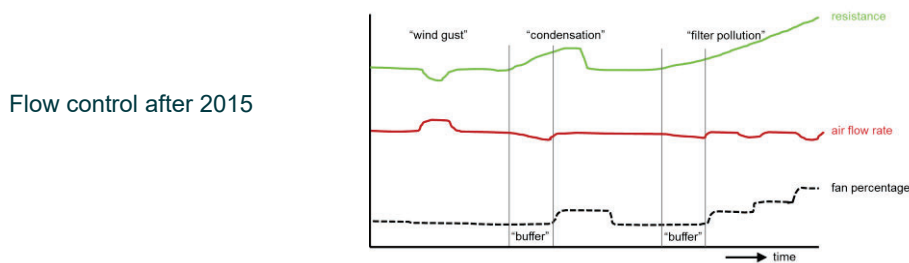
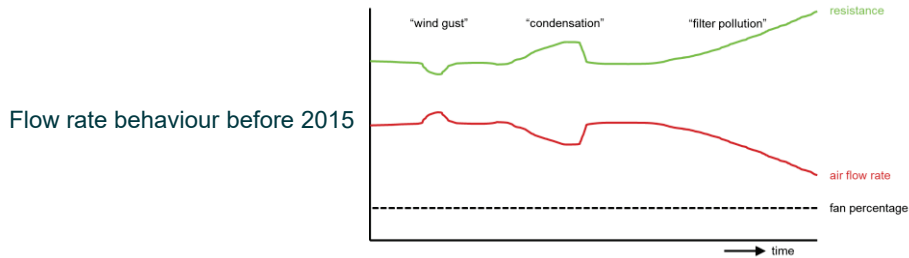
always the best climate

4

4



3. Automated fan speed correction (Flow Control)



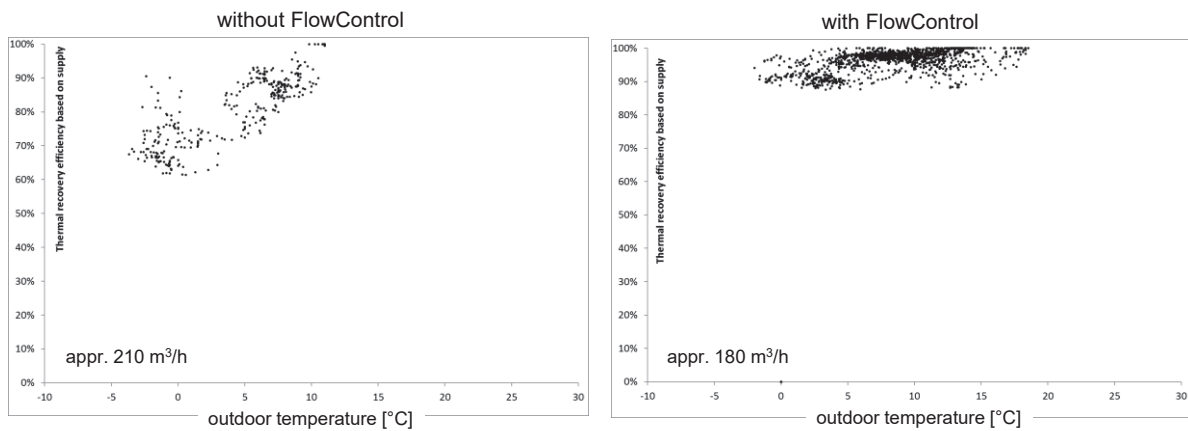
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5



3. Automated fan speed correction (Flow Control)



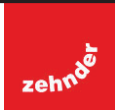
Cremers B.E. and Bakker T., Technologies to overcome effects of condensation in exchangers of ventilation units - analysis of monitored field studies, Proceedings of AIVC Conference 2017 in Nottingham, England

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6

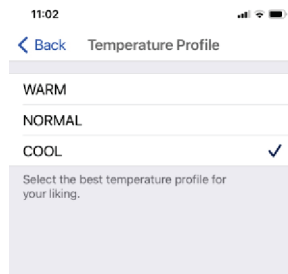
4. Comfort levels instead of values



Before 2015: The comfort temperature is set with a temperature value



After 2015: The comfort is set with a comfort profile



7

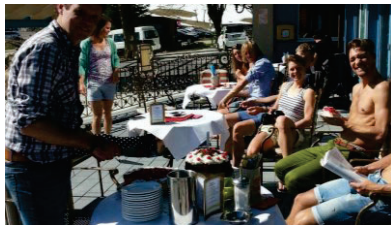
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5. Adaptive comfort technology



Fixed temperature leads to excessive cooling costs, unhealthy indoor temperatures and discomfort during heat waves



≠



≠



Today people feel warm

Today people feel chilly

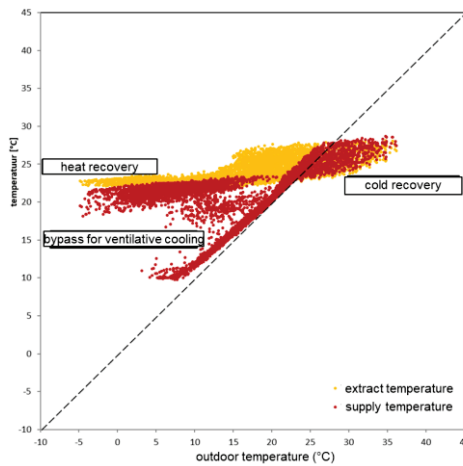
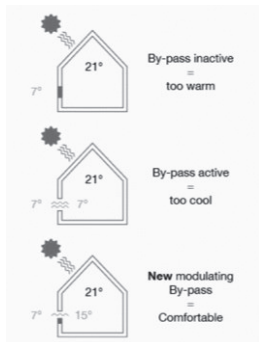
8

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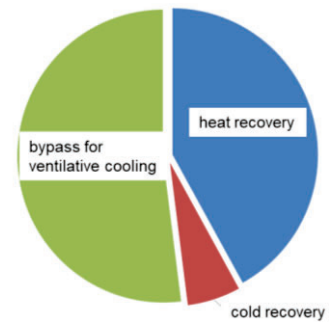
8

6. Modulating bypass technology

Modulating by-pass



Distribution of hours in one year

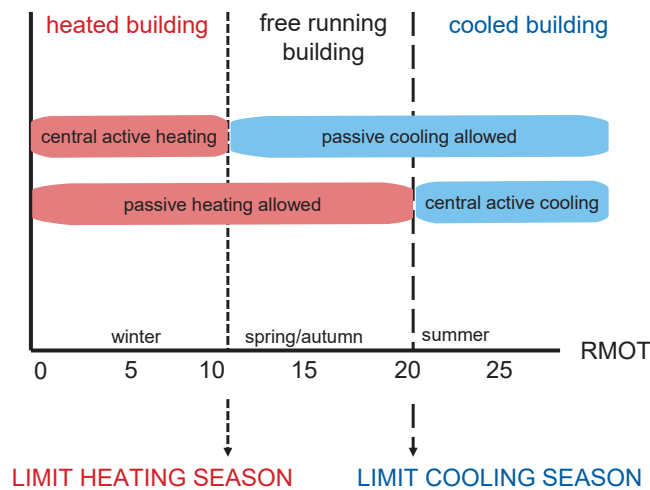


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9

9

7. Season detection



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10

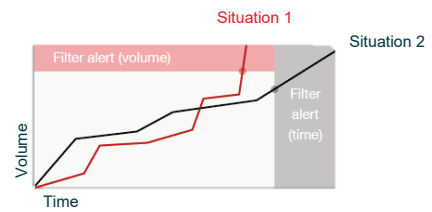
10

Filter

Filter change warning before 2015 based on time only

Filter change warning after 2015 based on time and flow rate

1. Warning signal after half a year
2. Warning signal after amount of m^3 has passed the filter
3. Warning signal when fan speed is much higher than before (resistance guard)
4. Warning signal when fan speed has exceed a limit (noise guard)



Summary

Technologies since 2015:

1. Automated fan speed adjustment during commissioning
2. Balance in mass flow rate rather than volume flow rate
3. Automated fan speed correction (FlowControl)
4. Comfort levels instead of values
5. Adaptive comfort technology
6. Modulating bypass technology
7. Season detection
8. Filter change warning

} prolonged optimal performance in bringing fresh air, energy savings and comfort

Topical Session - Building and ductwork airtightness regulations in various countries

AIVC 2023 CONFERENCE COPENHAGEN
4TH OCTOBER 2023

PLEIAQ
NOLWENN HUREL

VIP series on Building & Ductwork Airtightness

Series of Ventilation Information Papers (VIP) published by the AIVC

- “Building and ductwork airtightness - National trends and requirements”
- Template prepared: **similar structure** for all papers
- Authors found in various countries via the TightVent Airtightness Associations Committee (**TAAC**) and the **AIVC** board members
- Already **8 published papers** (on the AIVC website: <https://www.aivc.org/collection-keys/vip>)
 - Estonia (VIP 45.1)
 - Spain (VIP 45.2)
 - Czech Republic (VIP 45.3)
 - Belgium (VIP 45.4)
 - Latvia (VIP 45.5)
 - France (VIP 45.6)
 - Greece (VIP 45.7)
 - China (VIP 45.8)

VIP content

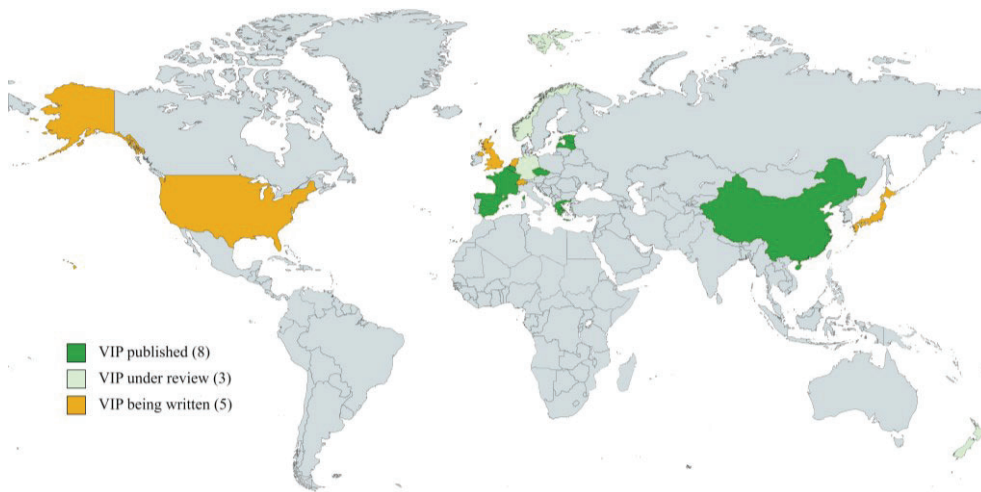
For both BUILDING and DUCTWORK airtightness, it details :

- **national requirements and drivers:** airtightness indicator, requirements in the regulation, energy programs, airtightness justifications, sanctions, etc.;
- if it is included in the **energy calculations** and how;
- the **airtightness test protocol:** qualification for the testers, guidelines, requirements on measuring devices;
- **tests performed:** tested buildings/ductworks, database, evolution with time;
- **guidelines** to build airtight buildings/ductworks.



3

More publications expected (around 16 countries)



4

Topical session program:

Building and ductwork airtightness - National trends and requirements in:

- **Norway** – Tormod Aurlien (NMBU) **12' (+5' for questions)**
- **The Netherlands** – Niek-Jan Bink (ACIN) **12' (+5')**
- **Spain** – Irene Poza Casado (University of Valladolid) **12' (+5')**
- **Latvia** – Andrejs Nitiievskis (IRBEST) **12' (+5')**

Air tightness and its impact on energy consumption in multi-family residential buildings in Montenegro - Esad Tombarević (University of Montenegro) **5' (+5')**

Oct. 3rd 23 TA/-

Ventilation
Information
Paper
n° 45
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International Energy Agency's
Energy in Buildings and Communities
Programme

Final draft Sept12th23



Air Infiltration and Ventilation Centre

**Trends in building
and ductwork
airtightness in
Norway**

Tormod Aurlien, NMBU

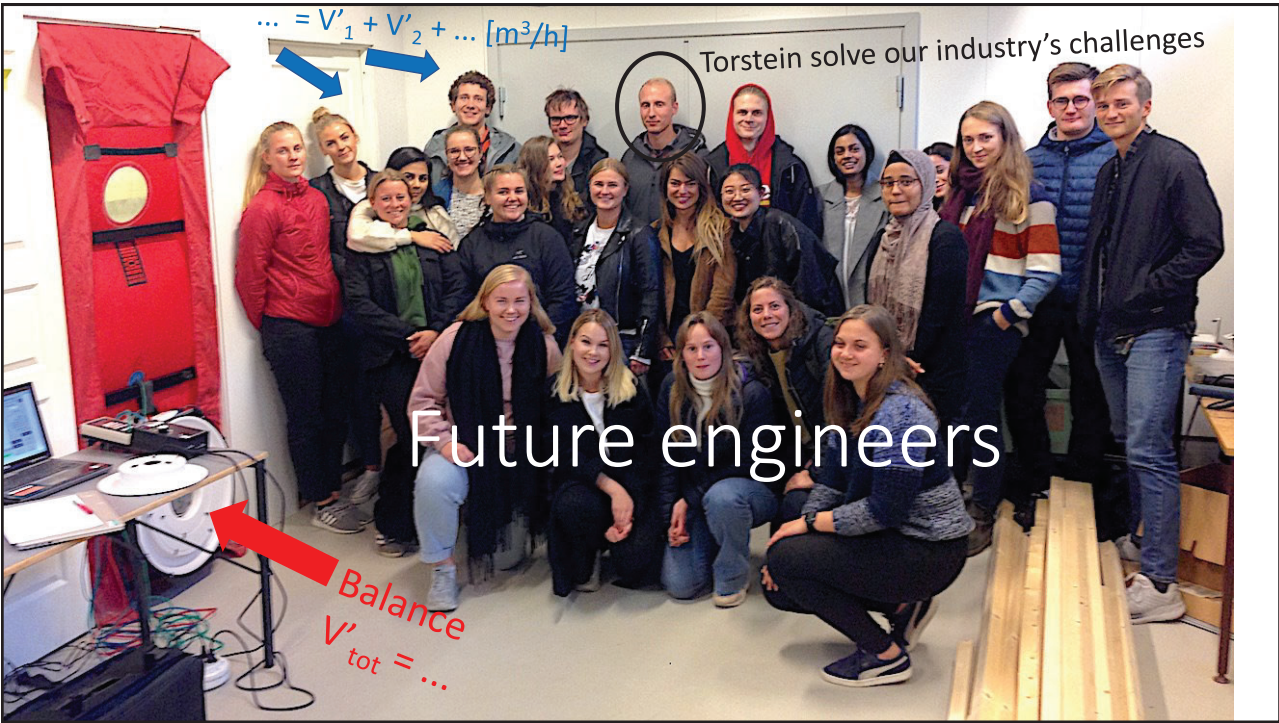
1 **General introduction**
The objective of the paper is to report

flow at 50 Pa divided by the building volume
(internally measured).

Building and ductwork airtightness - National trends and requirements in:

- Norway – Tormod.Aurlien@nmbu.no

1



2

Indicators and requirements

1 General introduction

The objective of the paper is to report

flow at 50 Pa divided by the building volume
(internally measured).

- Norway, being a cold and windy nation, required **wind barriers** in constructions (as part of regulations from the 1920s)
- Indicator: leakage number $n_{50} = V'_{50} / V$ (and when the volume is large, n_{50} becomes small)

Regulations

- for detached houses, gradually in steps from $n_{50} \leq 4$ /h to $n_{50} \leq 0.6$ /h (10 years back)
- $n_{50} \leq 0.6$ /h for all dwellings, if simplified documentation (= "passive-house"-level !)
- all buildings must have airtightness $n_{50} \leq 1.5$ /h (unchanged since 1980 for larger buildings)
- **extensive refurbishment of existing buildings is an important green challenge**

3

Drivers, justifications and sanctions

- We have had low-energy and passive-house labelling
- (the difference from minimum requirements for ordinary new buildings is often not large)
- EU taxonomy, and BREEAM compliance
"...air tightness measurement and thermography that are needed to prove a significant contribution to stopping climate change..."
- **"All new buildings shall be tested"**
(Less than 100% is probably measured in practice..?)
- The penalty of not fulfilling documented airtightness requirements is that no *use permit* should be granted (in theory)

4

EP-calculation

- $n_{inf} = n_{50} * 0,07$ [ach/h] ($V'_{inf} = n_{inf} * V$ [m³/h] ...)
 ↖ James Bond-number 😊
- more nuanced when making energy *budget*
- Wind-speed etc is not part of calculation according to our national EPC Standard (NS3031)
- n_{50} *requirement levels* are used as default input to the EP-calculation
- EP-calculation is performed long before actual building, and measuring..!
• and costly repairing..?

5



Protocol, Qualification and Equipment

- Tests and protocol often comply with ISO 9972, not always (but then then for good reasons :)
- No official Norwegian qualification scheme
- ← Simplified methods: depressurization at Flow 1, 2 and 3 (and corresponding Δp_1 , Δp_2 , and Δp_3)
- ↖ Discover leaks in wind barrier, in *early windtight stage*: easy to repair !

6

Concluding remarks

- Builders avoid measuring single flats, and instead measure the whole building, with all doors to the stairway open ☹️
(A large volume gives a large number below the fractional line, and a “nice” n_{50} -value)
- The air tightness quality may be very unevenly distributed, and (systematic) errors should not be camouflaged!
- *Internal* building air leakages should also receive increased focus.
 - Important factors when trying to control unnecessary energy misuse and reduced thermal comfort from wind and thermal stack effects
 - Internal airtightness reduces spread of infection, smell, fire gases, noise etc.
- Normalising on external surface area has some advantages (Swedish approach)
- Have we lost our Norwegian airtightness momentum..?

7

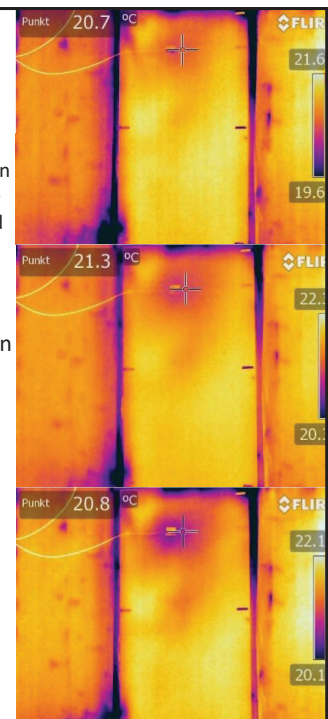
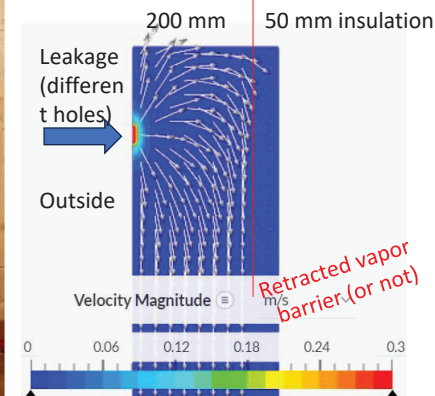
Torstein Hagen: Locating air leakages in wind barriers by thermography of insulated building structures, NMBU, Ås, 2022

Extract from Master thesis (in Norwegian)



It takes very long time with $\Delta p = -50 \text{ Pa}$ before an outside hole is detectable by the camera on the inside, especially with the vapour barrier placed in this way.

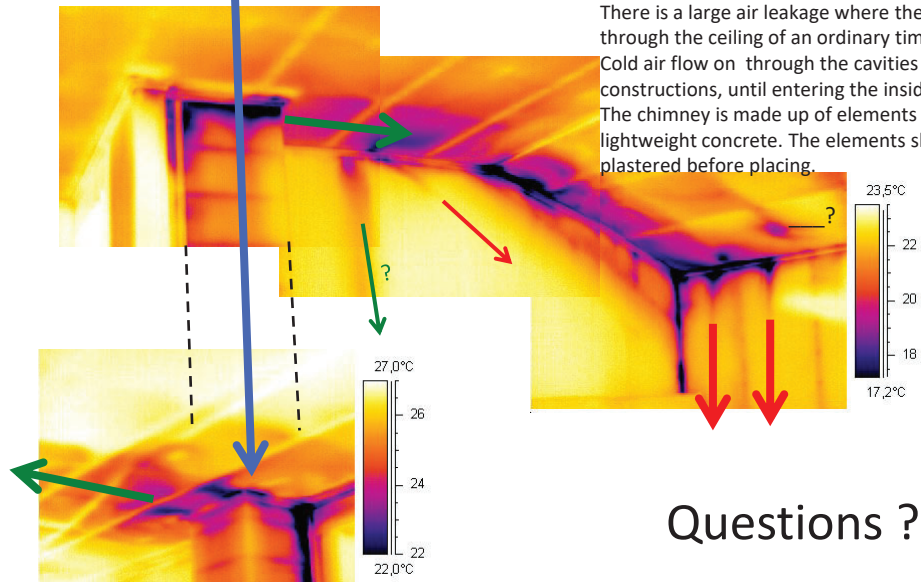
Depressurisation in *early windtight stage*, is a better strategy to ensure airtight wind barrier.



8

Locating and analyzing flow path

a continuously important path to progress



There is a large air leakage where the chimney passes through the ceiling of an ordinary timber frame house. Cold air flow on through the cavities of the constructions, until entering the inside. The chimney is made up of elements of very porous lightweight concrete. The elements should have been plastered before placing.

Part of 5 Termo in TBA210 © Tormod Aurlien

Questions ?

9

3 Ductwork airtightness

Air tightness of ventilation ducts is probably not an important topic in Norway, the way we build: We use spiro ducts with gaskets and the duct system is usually within the heat-insulated building construction. Small leakages may more be a topic related to internal vent adjustment and to a very small degree influencing energy or air quality.

10

Building and ductwork airtightness in the Netherlands: national trends and requirements

Niek-Jan Bink, Rob Dam, and Marcus Lightfoot

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Contents

1. Legislation
2. Building airtightness in the Netherlands
3. Duct airtightness in the Netherlands

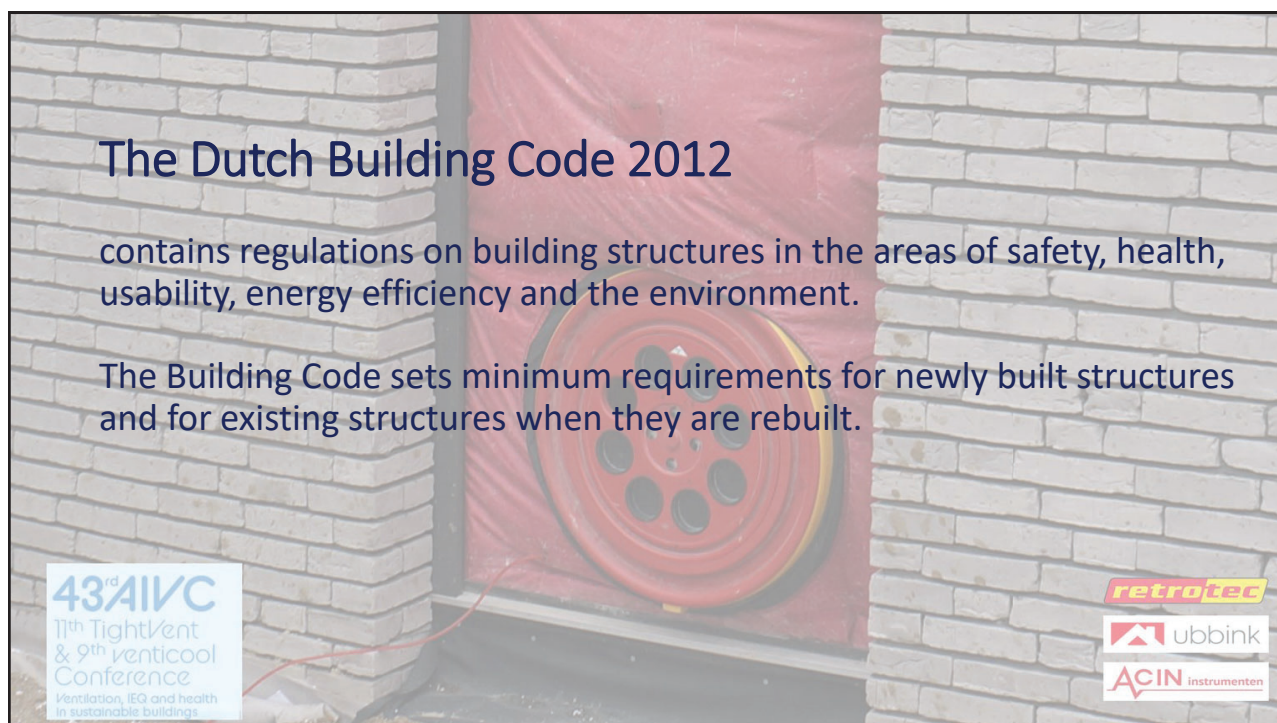
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3



4

Building Works Environment Decree

The Building Decree 2012 becomes the Building Works Environment Decree (Bbl). With the introduction of the Environment Act expected on January 1, 2024, the current Building Decree 2012 will expire and the technical building regulations will be included in the Building Works Environment Decree, or Bbl for short.

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Tools to meet Building Code regulations

To comply with the requirements of the 2012 Building Code, you can use:

- **NEN standards.**
- Recognized quality declarations.
- Equivalent solutions.
- Dutch practice guidelines (NPRs).
- **Dutch Technical Agreements (NTAs).**

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NEN standards

NEN standards are Dutch agreements that market parties make among themselves on a voluntary basis concerning the quality and safety of their products, services and processes.

As a neutral body, NEN makes an inventory of which standards (including NEN standards) are needed.

NEN standards are generally applicable standards, they are not generally binding regulations and therefore need not be published.

43rd NEN-EN: 1:1 copy of European standard

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NTA

NTAs are guidelines that deal with the practical elaboration of a standard from the Buildings Decree.

NTA 8800:2023 is the method for determining the energy performance of buildings, which can be used, among other things, to demonstrate compliance with BENG requirements. The intention is to designate NTA 8800:2023 in the building regulations as of July 1, 2023. NTA 8800 is freely available.

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Building Quality Assurance Act (Wkb)

The Wkb, aiming at enhancing construction quality and transparency was originally accepted by the Senate in May 2019. Implementation was postponed to July 1, 2023, and then again to January 1, 2024 for new constructions and July 1, 2024 for renovations.

The law's impact remains a topic of discussion, particularly concerning private quality assurance and construction liability.

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Building Quality Assurance Act (Wkb)

Key features include:

- **The introduction of Private Quality Assurance.**
This law shifts the responsibility for supervising the quality of construction works from the municipality to private quality assurance companies.
- **Strengthening the position of the building consumer.**
Builders become more clearly responsible for the quality of their construction works as a result of the law. Upon delivery, the builder must demonstrate that construction has been carried out in accordance with building regulations
- **Lower regulatory burden on municipalities.**
As supervision shifts to private parties, the municipality will be primarily concerned with the supervision of spatial planning and less with the technical aspects of construction.
- **Amendment in the Civil Code:**
clarifying the contractor's liability for hidden defects.

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q_{v10}

The most common airtightness indicator in the Netherlands is the q_{v10} , the leakage rate in $\text{dm}^3/\text{s}@10\text{Pa}$.

Normalizing by the

- useable floor area : $\text{dm}^3/\text{s}/\text{m}^2$ (m/s! >> confusing)
- building volume: $\text{dm}^3/\text{s}/\text{m}^3$

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Airtightness requirements

The Building Decree demands based on building volume

$< 500 \text{ m}^3$: $q_{v10} = 0.2 \text{ m}^3/\text{s}$ (200 l/s) as q_{v10} value

$> 500 \text{ m}^3$: $q_{v10, kar} = q_{v10} (500/Vb)$

The q_{v10} should be measured as per the NEN 2686 standard (which mainly follows the ISO-9972)

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$< 500 \text{ m}^3$: $q_{v10} = 0.2 \text{ m}^3/\text{s}$ (200 l/s) as q_{v10} value

$> 500 \text{ m}^3$: $q_{v10, kar} = q_{v10} * (500/Vb)$

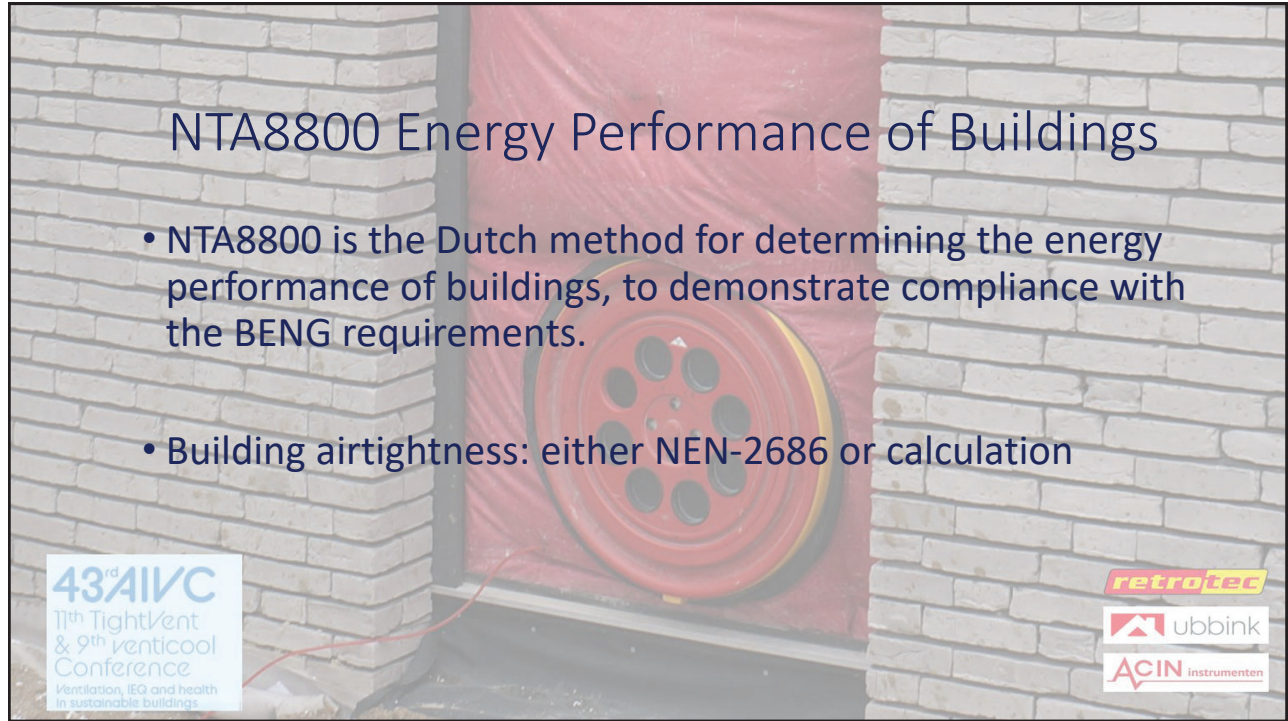
V (m3)	qv (m3/s)	ACH	qv10, kar (m3/s)
200	0,2	3,6	0,2
500	0,2	1,4	0,2
2000	0,8	1,4	0,2

$0,8 * 500 / 2000$

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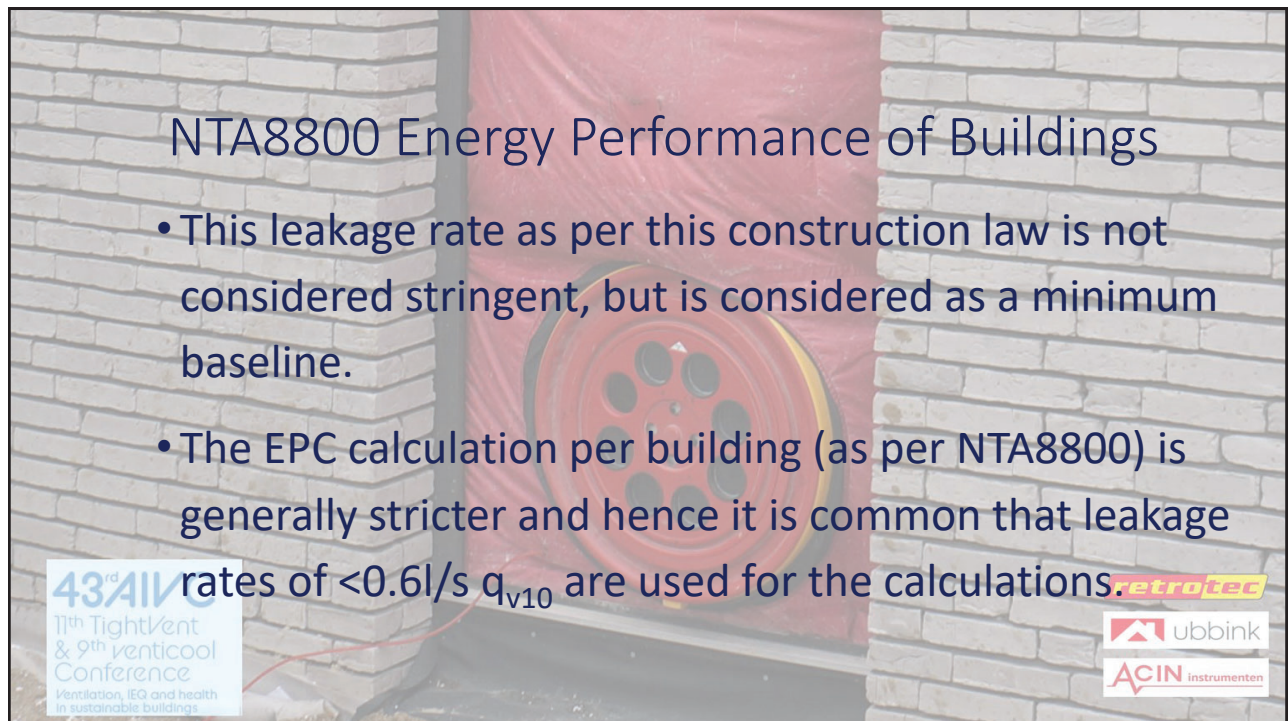
NTA8800 Energy Performance of Buildings

- NTA8800 is the Dutch method for determining the energy performance of buildings, to demonstrate compliance with the BENG requirements.
- Building airtightness: either NEN-2686 or calculation

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NTA8800 Energy Performance of Buildings

- This leakage rate as per this construction law is not considered stringent, but is considered as a minimum baseline.
- The EPC calculation per building (as per NTA8800) is generally stricter and hence it is common that leakage rates of $<0.6 \text{ l/s } q_{v10}$ are used for the calculations.

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NTA8800 Energy Performance of Buildings

In the NTA8800 [3] (used for the EPC calculation) there is a specific reference to a correction factor (as part of a formula) for the leakage rate based on the year of construction:

The formula used is $q_{v10,ref} = f_{type} \times f_y \times q_{v10,calc}$ where f_{type} is a correction factor for the building type and f_y is the correction factor for the age of the building.

Year of construction/renovation	Correction factor f_y
<1970	3,0
1970 - <1980	2,5
1980 - <1990	2,0
1990 - <2000	1,5
2000 - < 2010	1,0
>2010	0,7

Building Airtightness

- The only actual requirement is for buildings to comply to the building code, and for the tests to be performed as per the relevant standard.
- no mandatory qualification scheme for airtightness testers.
- Local initiatives have been started with some of them well on their way.
- At the moment building airtightness testing is not considered as a regulated profession

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Duct Airtightness

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Ductwork Airtightness in The Netherlands

It are mainly the manufacturers of ductwork that realize how important airtightness is. Poor airtightness can significantly reduce the energy performance of a building

Manufacturers of duct work are developing their own programs and are cooperating to test and certify their products



21

No specific regulation in the Netherlands that directly addresses ductwork airtightness, there are several NEN standards

The NTA8800 regulation, which came into force in 2021, focuses on the energy performance of buildings



22

The LUKA Quality Guide provides airtightness requirements for air ducts and fittings based on NEN-EN standards. It uses airtightness classes for ductwork components and fittings, which are defined in European Standard EN 12237 for circular ductworks, EN 1507 for rectangular ductworks, and EN 17192 for non-metallic ductworks.

Quality Guide Air duct systems 2022-06

View the most recent version of the Quality Guide at handboek.luka.nl

2.11.3 Classes of airtightness

The permissible amount of air leakage is related to airtightness classes, for which a test pressure applies, which is taken from NEN-EN 1507 and 12237. LUKA members test the airtightness only at overpressure. The following classes are used internationally:

Density class		Air leakage limit (f max)	
Future	Current		(in m3/hm2)
ATC 5	A	0.027	. pt 0.65 . 10-3
ATC 4	B	0.009	. pt 0.65 . 10-3
ATC 3	C	0.003	. pt 0.65 . 10-3
ATC 2	D	0.001	. pt 0.65 . 10-3

If not indicated otherwise in the specifications, Luka uses class C as standard requirement for airtightness. By measuring it can be determined whether the investigated section of the duct complies with the requirement set. In practice, after measuring with an appropriate testing device,

NTA8800 Energy Performance of Buildings

Focus is on thermal loss, not fan energy use or sound production

Impact of ductwork leakage on the fan energy use and sound production of central mechanical ventilation units in houses

Valérie Leprince^{1*}, Marcus Lightfoot², Jelmer de Jong³

¹ PLEIAQ
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² Ubbink - Centrotherm
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*Corresponding author: valerie.leprince@pleiaq.net

³ Brink Climate Systems
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To validate the airtightness value of ducts, the LUKA handbook recommends:

- Systematic Testing: testing the whole system (advisable)
- Sampling: specific sections of the ductwork can be tested focusing on complex areas like joins or points where the duct changes direction.
- visual inspections.
- in some circumstances, testing might only be undertaken if mandated by the building owners or another responsible entity.

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LUKA ensures the quality control of ductwork airtightness through a combination of internal policies, adherence to international standards, regular inspections by TÜV Rheinland Nederland B.V., and the provision of training and skill development opportunities for staff.



- The LUKA Quality Guide sets forth airtightness requirements for air ducts and fittings based on NEN-EN standards.
- The Netherlands does not have specific energy programs or subsidies directly targeting ductwork airtightness. It is only indirectly addressed via energy performance regulations and building codes.
- The LUKA handbook has established protocols for checking and ensuring the airtightness of ducts and takes action against non-compliant members.



Duct Airtightness in the Netherlands

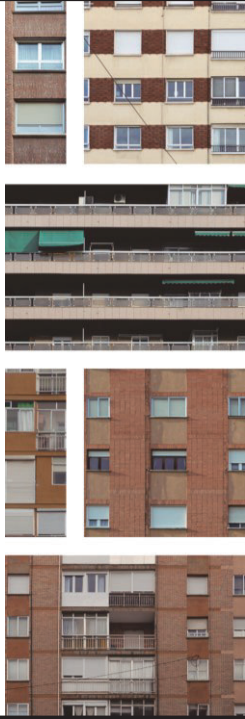
- The only actual requirement is for buildings to comply to the building code, and for the tests to be performed as per the relevant standard.
- Residential duct airtightness testing does not exist
- All initiatives come from duct manufacturers

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BUILDING AND DUCTWORK AIRTIGHTNESS IN SPAIN

NATIONAL TRENDS AND REQUIREMENTS

Irene Poza Casado, Universidad de Valladolid

43rd AIVC -11th TightVent & 9th venticool Conference: *Ventilation, IEQ and health in sustainable buildings*
October 4-5, 2023 Aalborg University, Copenhagen, Denmark



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**Ventilation
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Trends in building and ductwork airtightness in Spain

Timo Hoek, airtest, Spain
Irene Poza-Casado, UVa, Spain
Sergio Melgosa, eBuilding, Spain

1 General Introduction

The Spanish residential stock is, on average, 45 years old and is in the lower part of the energy efficiency ranking, with an average valuation of 'E'. The market is recovering so vigorously from the Covid-19 hit that some voices in the sector warn that we are already facing the beginning of a new real estate "boom" that will continue until 2023. According to several projections, the number of dwellings in Spain could increase by 1 103 761 (5 064) between 2020 and 2035

2 Building airtightness

2.1 Introduction

Building airtightness has not traditionally been a major priority in the Spanish construction industry. Because most dwellings did not have any controlled ventilation systems, air infiltration has been a supplemental source of air renewal together with window airing, that contributed to indoor space air renewal (2).

VIP 45.2: Trends in building and ductwork airtightness in Spain

<https://www.aivc.org/resource/vip-452-trends-building-and-ductwork-airtightness-spain>



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Building and ductwork airtightness in Spain: national trends and requirements – Irene Poza-Casado



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Structure

- Context
- Building airtightness requirements
 - Window airtightness
 - Envelope airtightness
 - Building airtightness justification
- Ductwork airtightness requirements
- Conclusions and challenges



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Context

- No traditional awareness of airtightness
 - Infiltration as a source of air renewal
- Energy reduction for a decarbonised system: nZEB
 - Heat transmission:
 - Improvement of construction materials. Traditional systems still prevail
 - Increased proportion of air infiltration losses (up to 25% of the heating demand)
- Controlled ventilation strategies since 2006

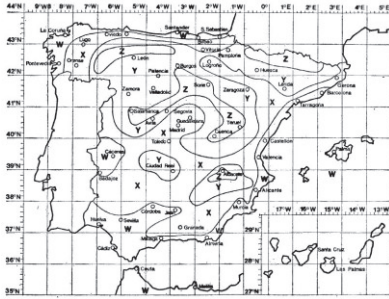


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Building airtightness requirements



- Window permeability regulation since 1975 (RD 1490/1975)
- 2020: whole building airtightness limitation - nZEB

R. O. del E.—Núm. 185 11 julio 1975 18003

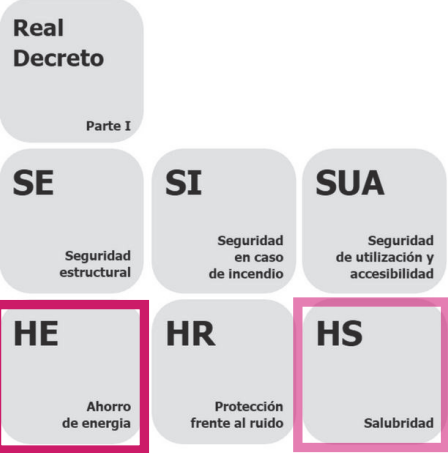
No se decretan unas para las Islas Canarias por sus especiales condiciones climáticas.

Artículo primero.
La obligación a utilizar en edificios emplazados en las zonas climáticas W y X tendrá como máximo un grado de permeabilidad, en m³ de aire/m² de fachada.
En las zonas climáticas Y y Z tendrá, como máximo, un grado de permeabilidad, en m³ de aire/m².

Artículo segundo.
A. Construcción de celos, depósitos estructurales e impermeabilización de techos.
Cuando la superficie de pérdida sea superior a 1 m², el volumen del aislamiento será como mínimo de 20 mm. En el caso de edificios excavados, e intercomunicados de un sótano más superficie de pérdida inferior a 2 m², el espesor será como mínimo de 30 mm.
Artículo tercero.
Para el mejor aprovechamiento de las aportaciones gratuitas



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Windows airtightness



Maximum airtightness values of windows per climate zone in winter ($m^3/h \cdot m^2$) at a pressure difference of 100 Pa

Zone α	Zone A	Zone B	Zone C	Zone D	Zone E
≤ 27	≤ 27	≤ 27	≤ 9	≤ 9	≤ 9

where: q_{100} is the reference air permeability at a pressure difference of 100 Pa [$m^3/h \cdot m^2$]. Note: according to UNE-EN 12207, the permeability limit values correspond to Class 2 ($\leq 27 m^3/h \cdot m^2$) and Class 3 ($\leq 9 m^3/h \cdot m^2$). If a window has a rolling shutter, its permeability value should also include it. Climate zones A, B, C, D and E refer to Continental Spain. Zone α refers to the Canary Islands.



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Envelope airtightness



- New residential buildings $>120m^2$
- Based on compacity (Volume/Area)

Maximum n_{50} [h^{-1}] values at a pressure difference of 50 Pa

Compacity V/A [m^3/m^2]	n_{50}
$V/A \leq 2$	6
$V/A \geq 4$	3

where: n_{50} is the air change rate at 50 Pa [h^{-1}]; V is the internal volume of a building or part of a building [m^3]; A_{ET} is the sum of areas of the thermal building envelope with heat exchange with the outdoor air. Therefore, internal partitions and the envelope area in contact with other adjacent spaces or buildings are excluded [m^2]. Note: the limit permeability values for intermediate V/A values can be obtained by interpolation.



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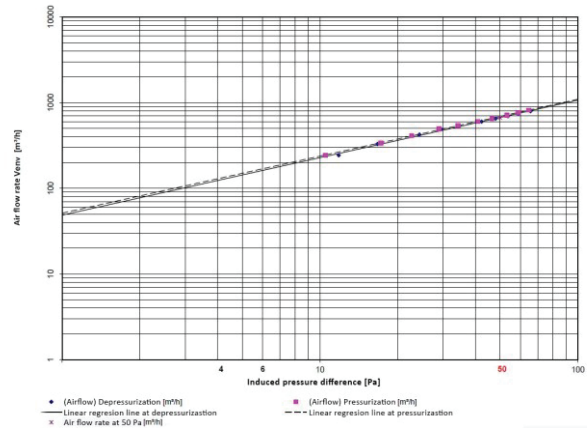
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Building airtightness justification

OPTION A

BlowerDoor test

- ISO 9972, Method 2
- No other specification
- No qualification scheme for testers
- No airtightness database
- No quality control system



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Building airtightness justification

OPTION B

Analytically

$$n_{50} = 0.629 \cdot \frac{C_0 \cdot A_0 + C_h \cdot A_h}{V}$$

- This is the widespread approach
- Input values of the energy performance calculation

where:

n_{50} is the calculated air change rate at 50 Pa [h^{-1}]

V is the internal volume [m^3]

C_0 is the airflow coefficient of the opaque part of the thermal envelope at a reference pressure of 100 Pa [$m^3/h m^2$]. Reference values:

- New or existing buildings with improved airtightness, $C_0 = 16 m^3/h m^2$
- Existing buildings $C_0 = 29 m^3/h m^2$

A_0 is the sum of areas of the opaque thermal building envelope [m^2]

C_h is the permeability of doors and windows in the thermal building envelope at a reference pressure of 100 Pa [$m^3/h m^2$]

A_h is the sum of the area of the doors and windows of the thermal building envelope [m^2]. The thermal building envelope consists of the building parts with heat exchange with the outdoor air. Therefore, **internal partitions in contact with adjacent indoor spaces or buildings are excluded.**



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Analytical model validation

- Airtightness test results were compared to calculated values
- **Lack of linear association** between the values of the CTE model and the test values

I. Poza-Casado et al.

Building and Environment 223 (2022) 109435

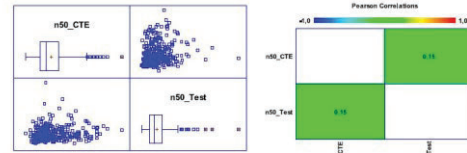


Fig. 1. Correlation analysis between the n_{50} values obtained from pressurization tests and those computed using the CTE model.

Poza-Casado, I., Rodríguez-del-Tío, P., Fernández-Temprano, M., Padilla-Marcos, M.-Á., & Meiss, A. (2022). An envelope airtightness predictive model for residential buildings in Spain. *Building and Environment*, 223(July), 109435. <https://doi.org/10.1016/j.buildenv.2022.109435>

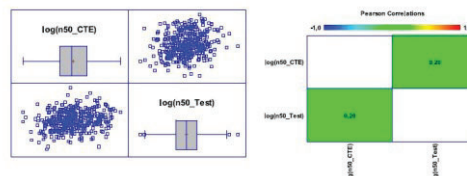


Fig. 2. Correlation analysis between the logarithms of the n_{50} values obtained in the pressurization tests and those computed using the CTE model.



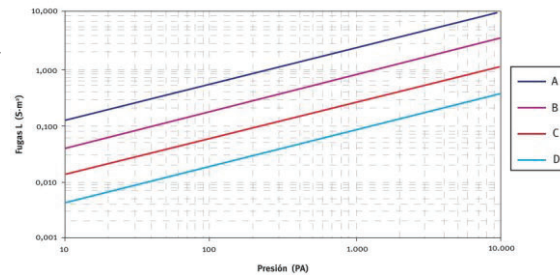
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Ductwork airtightness requirements

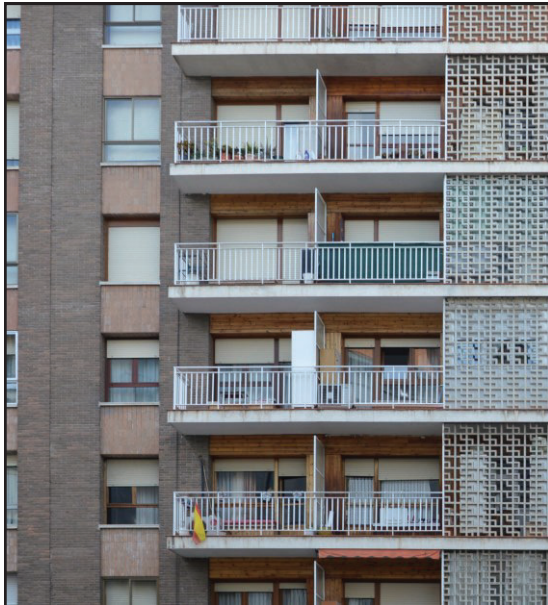
- Residential buildings:
 - CTE – DB HS3 (2007)
 - “Ducts must be airtight for its designed pressure”
 - “Airtightness must be checked every 5 years”
- Other buildings:
 - *Reglamento de Instalaciones térmicas de los edificios* (RITE) – 2007
 - At least Class B
 - Tests are required (UNE-EN 12599:01)
 - No awareness



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Conclusions and challenges

- Raised awareness for the past few years
- Positive progress towards energy-efficient buildings



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Conclusions and challenges

- Gaps:
 - Application only for a small share of buildings
 - Limit values not too stringent
 - Unrealistic analytical approach – not mandatory testing
 - No airtightness database: no data on airtightness improvement
 - Qualification frame for testers
 - No quality control system
 - No inspections



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Thank you!

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*Funding: MOVILIDAD INVESTIGADORES E INVESTIGADORAS UVA-BANCO SANTANDER 2023



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Building and ductwork airtightness in Latvia: national trends and requirements

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OCTOBER 4-5, 2023

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ANDREJS NITIJEVSKIS
VLADISLAVS KEVISS
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Introduction – Building market in Latvia

- **Population:** 1,9 million
- **Residential buildings:** about 3000 building permits delivered / year
(2/3 for single dwellings; 1/3 for multifamily)
- **Non-residential buildings:** about 45% of the construction activity
- **Total investment :** 1.2 billion €



Per cent variation of investment in real terms
investment Min. € fixed prices

Sectors	2019a	2016	2017	2018	2019a
Building	1,232	-2.5	12.1	19.9	9.7
1.1. Housebuilding	287	2.1	-10.1	32.2	6.0
1.1.1. New	174	10.6	13.7	3.6	7.1
1.1.2. R&M	113	-8.6	-46.6	126.0	4.3
1.2. Non residential (c)	945	-4.1	20.5	16.4	10.8
1.2.1. Private	NA	NA	NA	NA	NA
1.2.2. Public	NA	NA	NA	NA	NA

a: estimate - b: forecast - c: incl. R&M

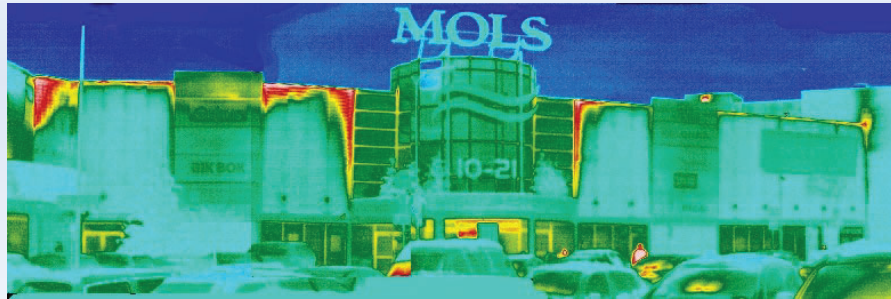
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Building airtightness



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Introduction on building airtightness

- **2010:** requirement of blower door tests for buildings renovated with EU funds
→ interest in building airtightness starting
- **2015:** Latvian Construction Standard (LBN 002-01) on thermal insulation and airtightness became stricter
- **2021:** government recommendation to provide airtightness tests for the commissioning of all public buildings > 5000 m³



Source: Retrotec

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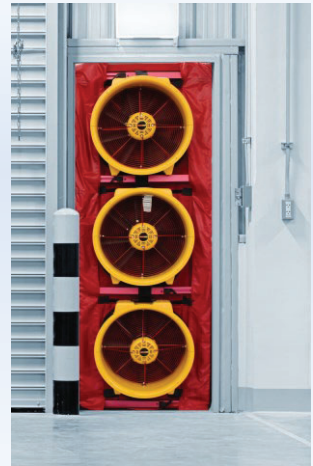
Introduction on building airtightness

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- **2021:** government recommendation to provide airtightness tests for the commissioning of all public buildings > 5000 m³

Airtightness indicator: q_{50} (m³/(h.m²))

→ Leakage flowrate at 50 Pa divided by the total envelope area (incl. floors)

n_{50} (h⁻¹) also used for some project



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Requirements in the regulation

From 2001 to 2015:

- $q_{50} \leq 3,0$ m³/(h.m²) for **dwelling, hospitals, kindergartens, homes for the elderly**
- $q_{50} \leq 4,0$ m³/(h.m²) for other **public buildings**
- $q_{50} \leq 6,0$ m³/(h.m²) for **industrial buildings**
- Mechanical ventilation for buildings with $q_{50} < 3$ m³/(h.m²)

Since 2015:

- $q_{50} \leq 3,0$ m³/(h.m²) for buildings with **natural** ventilation (airing);
- $q_{50} \leq 2,0$ m³/(h.m²) for buildings with **mechanical** ventilation;
- $q_{50} \leq 1,5$ m³/(h.m²) for buildings with mechanical ventilation equipped with a **heat recovery system**;
- $q_{50} \leq 4,0$ m³/(h.m²) for **industrial buildings**
- ~~Mechanical ventilation for buildings with $q_{50} < 3$ m³/(h.m²)~~

Update in 2019: same requirements regarding airtightness



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Airtightness test

So there are **mandatory airtightness requirements** for all new buildings...

... but **no mandatory testing!**

- **No sanctions** in case a building does not comply with the requirements
- Only the owner/developer or construction regulator can initiate and write requirements to **perform airtightness tests for new projects**
(Requirements usually described initially in the **project documentation**)



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Incentives for building airtightness

- **2021:** government recommendation to provide airtightness tests for the commissioning of all public buildings > 5000 m³
- **Since January 2022:** the city of Riga (capital) gives a **90% discount on property taxes** for:
 - **newly built detached houses classified as NZEB**
(for a period of 5 years from the commissioning of the building)
 - **apartment in multi-family buildings for which the insulation of all facades has been performed after its commissioning**
(as energy retrofit reaching better than class C)
Discount applied throughout the period of validity of the energy certificate, with a maximum of 10 years.

90%

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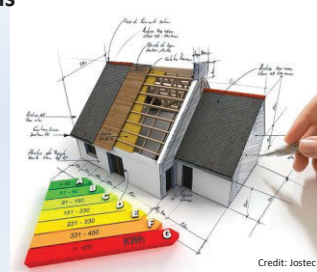
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Building airtightness in the EP calculation

- Airtightness is an **input of the Energy Performance (EP) calculations**
- **Default values:** requirements provided in the Latvian construction standard

- $q_{50} \leq 3,0 \text{ m}^3/(\text{h}\cdot\text{m}^2)$ for buildings with **natural** ventilation (airing);
- $q_{50} \leq 2,0 \text{ m}^3/(\text{h}\cdot\text{m}^2)$ for buildings with **mechanical** ventilation;
- $q_{50} \leq 1,5 \text{ m}^3/(\text{h}\cdot\text{m}^2)$ for buildings with mechanical ventilation equipped with a **heat recovery system**;
- $q_{50} \leq 4,0 \text{ m}^3/(\text{h}\cdot\text{m}^2)$ for **industrial buildings**



- Possible to use **lower air permeability values** if a test is performed
- the rather favourable default values are **not encouraging airtightness testing**

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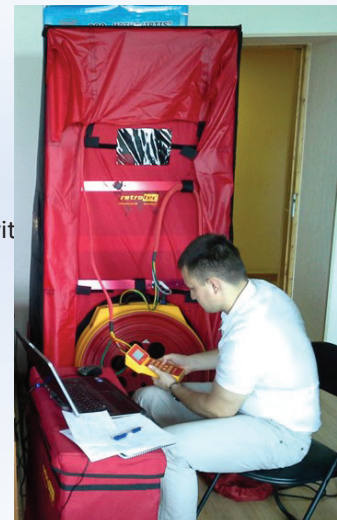
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Building airtightness test protocol

- **No national qualification scheme** for airtightness testers
Currently there are approximately:
 - **8 persons** qualified by the manufacturer program **Retrotec**;
 - **2 persons** qualified by the Air Tightness Testing & Measurement Association (**ATTMA**)Only some of them are testing building airtightness as their main activity
- **No national guidelines** to perform the airtightness test
Tests should be performed in accordance with **EN 9972:2016 Method 2**: “by closing all the windows, doors, hatches in the building”



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Building airtightness tests performed

Estimation of the percentage of buildings tested (no official data available):



- **70-80%** of **public** buildings (new or renovated)



- **5-10%** of **industrial** buildings;



- **5-15%** of **dwellings** (single-family houses and multi-apartment buildings)

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Conclusion

- Personal experience of **>2000** tests proved the reasonableness of the **$q_{50} \leq 1,5 \text{ m}^3/(\text{h}\cdot\text{m}^2)$** as a to be a motivating tool for quality of an building envelope
- e.g. Lithuania - **$n_{50} \leq 0.6 \text{ h}^{-1}$** - **too strong for small buildings but too weak for large building**
- Mandatory **airtightness requirements** but **no mandatory tests** → still very few buildings tested
- Currently: airtightness **stimulated by the taxes reduction** for NZEB in Riga

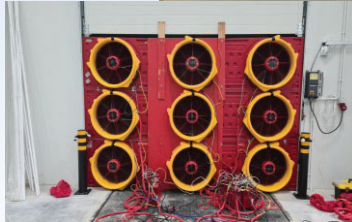
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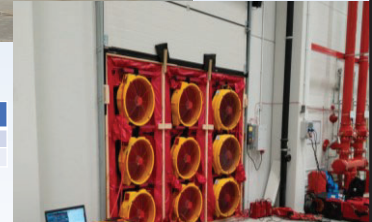
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Test of Large Building V=1.75 million m³



Symbol	Unit	Test Results
q_{ESD}	Specific leakage rate at 50 Pa, q_{ESD} [m ³ /h.m ²]	0.88
n_{50}	Air change rate at 50 Pa, h ⁻¹	0.14



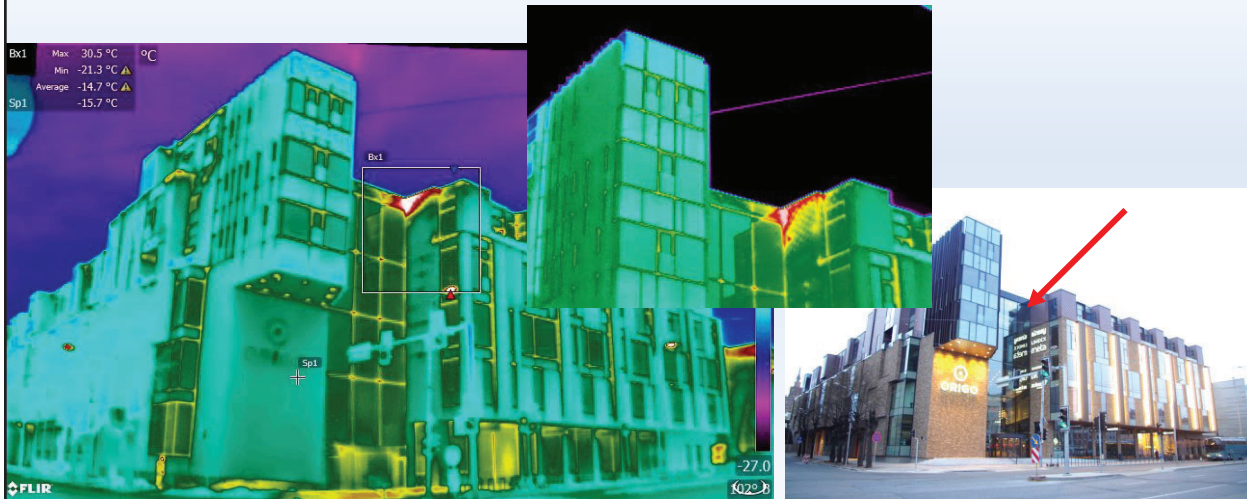
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Blower Door + Thermography

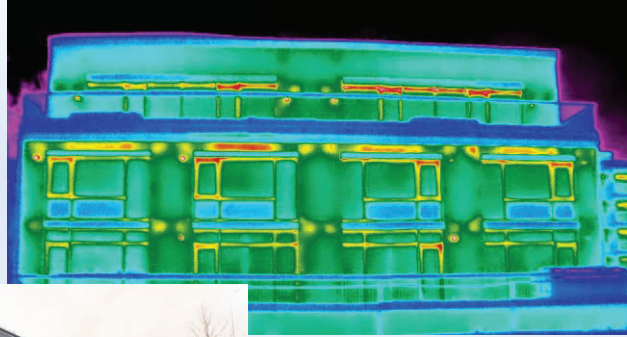
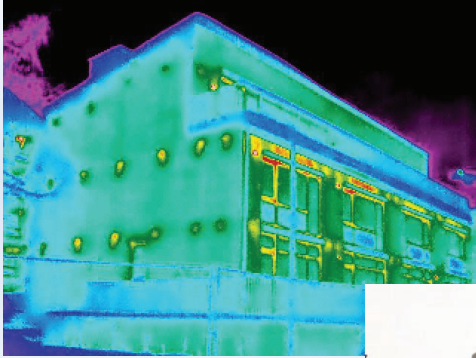


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Ductwork airtightness

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Ventilation ductwork airtightness

- **Not really taken into account** so far
 - No national regulations/guidelines
 - No requirements on airtightness levels
- **Reference document:** European standard LVS EN 12237
- Only **rare cases** in which **customers initiate a ductwork airtightness test**
- **No progress foreseen** in the next years



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Ventilation, IEQ and health in sustainable
buildings

Sesion 3B – Topical Session: Building and ductwork airtightness regulations in various countries

Air tightness and its impact on energy consumption in multi-family residential buildings in Montenegro

Esad Tombarević, Igor Vušanović, Miloš Krivokapić
University of Montenegro, Faculty of Mechanical Engineering
e-mail: esad.tombarevic@ucg.ac.me



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Building and ductwork airtightness in Montenegro



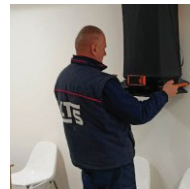
Building envelope airtightness

- Montenegro has warm climate and proper attention has not been paid to air tightness of buildings
- Rulebook on minimum energy efficiency requirements for buildings (Official Gazette, No. 23/2013)
- Airtightness requirement in Montenegro
 - ~~without mechanical ventilation $ACH_{50}=3\text{ h}^{-1}$~~
 - with mechanical ventilation $ACH_{50}=1.5\text{ h}^{-1}$
- MEST EN ISO 9972:2017 is adopted
- Blower door tests are not mandatory (performed only on several buildings for which the investor aimed for the LEED certificate)

		$< 10\text{ m}^3/\text{h}\cdot\text{m}^2 @ 50\text{ Pa}$		
	$< 1500\text{ m}^3: n_{50}$	$< 3\text{ l/h}$		$< 1.5\text{ l/h}$
	$> 1500\text{ m}^3: q_{50}$	$< 4.5\text{ m}^3/\text{h}\cdot\text{m}^2$		$< 2.5\text{ m}^3/\text{h}\cdot\text{m}^2$
	n_{50}	4.5 l/h	1.5 l/h	1 l/h
		1.5 l/h	0.6 l/h	
			3 l/h	1.5 l/h
	$q_{50} \leq 7\text{ m}^3/\text{h}\cdot\text{m}^2$			
	q_{4Pa_surf}	$0.6\text{ m}^3/\text{h}\cdot\text{m}^2$	$1\text{ m}^3/\text{h}\cdot\text{m}^2$	
		$3\text{ m}^3/\text{h}\cdot\text{m}^2$	$2\text{ m}^3/\text{h}\cdot\text{m}^2$	$1.5\text{ m}^3/\text{h}\cdot\text{m}^2$

Ductwork airtightness

- MEST EN 1507:2012 and MEST EN 12237:2012 are adopted
- There is only one company which has tests for leakage of air ducts in the scope of its accreditation (according to ISO 17020)
- Tests are not mandatory (performed only on several buildings for which the investor aimed for the LEED certificate)



2

- Building's airtightness significantly affects the building's energy consumption due to air infiltration
- National software for calculation of the energy performance of buildings takes into account infiltration
- Energy auditors are at a loss as to which values to use in calculations.

The screenshot shows the MEEC software interface with several red boxes highlighting key areas:

- Top right:** Infiltration settings for 'Location of building' (Moderately open), 'Condition of building' (Open), and 'Very sheltered'.
- Middle left:** Infiltration settings for 'Location of building' (Moderately open) and 'Condition of building' (Windows and façade walls in normal condition).
- Bottom left:** A table titled 'Building location' with columns for 'Open', 'Moderately open', and 'Very sheltered'. It includes a formula for infiltration based on a blower-door test: $n = \frac{n_{50}}{15} \cdot C_{loc}$.
- Bottom right:** Infiltration settings for 'Condition of building' (Windows and façade walls in bad condition), 'Location of building' (Windows and façade walls in normal condition), 'Infiltration based on Blower-door test', and 'Condition of building' (Windows and façade walls in normal condition).

A cartoon character is shown thinking about ACH₅₀.

3

Results of performed blower door tests



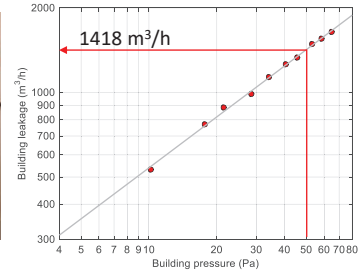
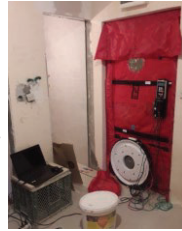
No.	Floor area A (m ²)	Volume V (m ³)	Year of construction	Air leakage rate \dot{V}_{50} (m ³ /h)	Air change rate n_{50} (h ⁻¹)	Specific leakage rate W_{50} (m ³ /h/m ²)	Effective leakage area ELA (cm ²)	Building leakage curve		Windows type
								Air leakage coefficient C_L (m ³ /h/Pa ⁿ)	Air flow exponent n	
1	53	127	2006	1001	7.87	18.89	228.8	90.6	0.614	Wood
2	43	114	2011	679	5.95	15.78	148.5	57.4	0.631	Aluminium
3	58	162	2012	711	4.38	12.26	150.1	56.9	0.646	PVC
4	68	190	1963	1831	9.63	26.92	453.3	187.6	0.582	Wood
5	68	190	1963	1225	6.45	18.02	280.2	111.0	0.614	Wood/PVC
6	68	190	1963	305	1.60	4.48	63.1	23.6	0.654	PVC
7	85	227	1986	1418	6.25	16.69	334.5	134.8	0.602	Wood
8	85	227	1986	174	0.77	2.05	37.5	14.4	0.637	PVC

4

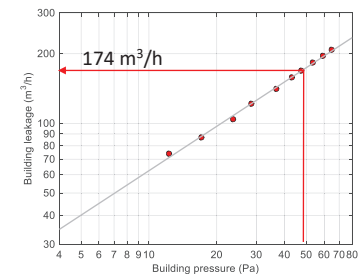
Impact of windows replacement



- external walls are made of cast-in-place reinforced concrete
- Old windows: wooden, double-glazed, no rubber seals, lack of maintenance
- New windows: UPVC windows, 5 chamber profile, 70 mm standard installation depth, two seals, double glazed low e-glass, all with roller shutters



Before - Wood

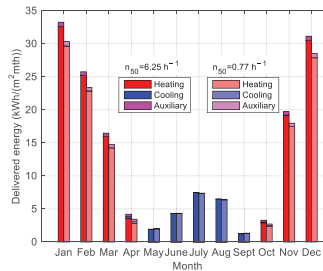
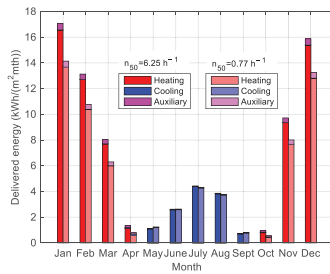


After - PVC

Value	Before	After
Air leakage rate, V_{50} (m^3/h)	1418	174
Air change rate, n_{50} ($1/h$)	6.25	0.77
Air permeability, q_{50} ($m^3/h/m^2$)	11.53	1.41
Specific leakage rate, w_{50} ($m^3/h/m^2$)	16.69	2.05
Effective leakage area, ELA (cm^2)	334.5	37.5
Building Leakage Curve		
Air leakage coefficient, c_L ($m^3/h/Pa^n$)	134.8	14.4
Air flow exponent, n	0.602	0.637

5

Climate zone I (design day $-6\text{ }^\circ\text{C}$)



2.3 kWh/m²year for each ACH

Climate Zone	Building Envelope	n_{50} (h^{-1})	Delivered Energy (kWh/m ² ·year)			
			Heating	Cooling	Auxiliary	Total
I	Existing thermal envelope	6.25	129.84	21.78	3.82	155.44
		0.77	117.32	21.73	3.58	142.63
	Improved thermal envelope	6.25	63.60	12.67	2.68	78.95
		0.77	51.57	12.67	2.37	66.61

Significant reduction

Negligible reduction

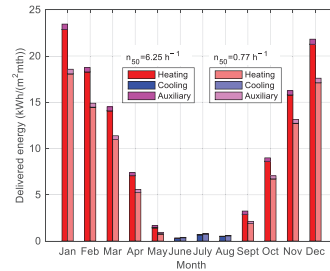
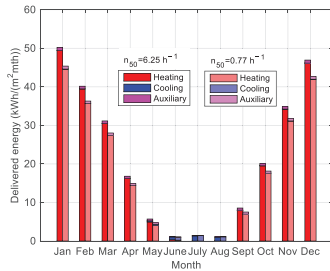
49.2% reduction

8.2% reduction

15.6% reduction

6

Climate zone III (design day temperature $-18\text{ }^{\circ}\text{C}$)



4.6 kWh/m²·year
for each ACH

Climate Zone	Building Envelope	n_{50} (h ⁻¹)	Delivered Energy (kWh/m ² ·year)				
			Heating	Cooling	Auxiliary	Total	
III	Existing thermal envelope	6.25	248.67	3.50	6.04	258.21	9.6% reduction
		0.77	223.85	3.68	5.73	233.26	
	Improved thermal envelope	6.25	112.25	1.49	3.92	117.66	20.9% reduction
		0.77	87.95	1.75	3.41	93.11	

Significant reduction Negligible reduction 54.5% reduction

7

Conclusions

- The problem of airtightness is ignored in Montenegro
- Windows play a crucial role in maintaining the overall air tightness of a building's envelope
- Windows replacement proves to be an effective measure that improves the airtightness almost to the standard of a passive house
- By increasing airtightness through windows replacement, energy consumption for heating is significantly reduced, while the reduction in energy consumption for cooling is practically negligible
- In relative terms, the reduction in energy consumption due to the increase in airtightness is more pronounced in colder climates and when the thermal envelope is improved
- Energy consumption increases linearly with increasing air leakage rate (from 2.3 kWh/m²·year in Climate zone I to 4.6 kWh/m²·year in Climate zone III for a unit reduction of ACH₅₀)

8

Thank you for attention!

IEA EBC Annex 80 - Resilient Cooling

Patryk Czarnecki
Institute of Building Research & Innovation
Vienna, Austria



Institute of
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2023-10-04

1

1

Topical Session Resilient Cooling of Buildings meets Resilient Cooling in Cities



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2

Agenda

- Presentation of Technology Profiles
- Resilient Cooling Design Guidelines with Vincenzo Corrado
- Resilient Cooling in Cities – New Annex?
- Interactive Part including discussion with Hilde Breesch

3

3

Objective and Target Audience - Technology Profiles

What?

- Structured Information about the Essential Resilient Cooling Technologies

Who?

-

4

4

Objective and Target Audience - Technology Profiles

What?

- Structured Information about the Essential Resilient Cooling Technologies

Who?

- Planners, Property Owners and Investors, Interested Public

5

5

1. Reducing Heat Loads to People and Indoor Environments

- 1.1. Solar Shading Technologies
- 1.2. Cool Envelope Materials
- 1.3. Glazing Technologies
- 1.4. Ventilated Façades
- 1.5. Green Roofs and Green Façades

6

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2. Removing Heat from Indoor Environments (Production, Emission and Combined)

- 2.1. Ventilative Cooling
- 2.2. Thermal Mass Utilization
- 2.3. Evaporative Cooling
- 2.4. Sky Radiative Cooling
- 2.5. Compression Refrigeration
- 2.6. Adsorption Chillers
- 2.7. Natural Heat Sinks
- 2.8. Radiant Cooling

7

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3. Increasing Personal Comfort Apart from Space Cooling

- 3.1. Comfort Ventilation and Elevated Air Movement
- 3.2. Micro-cooling and Personal Comfort Control

8

8

4. Removing Latent Heat from Indoor Environments

4.1. Dehumidification

9

9

Outcome

- 130 pages document with key and concentrated information
- 16 chapters; 1 for each technology
- Already internally reviewed

10

10

Chapter Structure

- Description
 - Physical principles, function, characteristic applications, relevant subtypes
 - “Abstract” of the full Technology Profile.
- Key Technical Properties
 - Technical properties, system design indicators, qualitative numbers or ranges
 - Characterization of properties which are relevant for designing/purchasing

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Chapter Structure

- Performance and Application
 - Information on the expected effect on whole building performance, especially about qualities of resilience
 - Exemplary quantitative outputs from simulation studies and qualitative statements backed with sources and literature
 - Information about the range of possible applications as well as about limitations including climate dependency and building types
 - Compatibility and incompatibility with other technologies.
 - Availability and expected developments

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Chapter Structure

- Further Reading
 - Links to further information such as other Annex 80 content, websites of relevant interest groups, industry associations, rating associations and others.
 - Specific manufacturers were not included

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Qualities of Resilience

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Qualities of Resilience

- Absorptive capacity
 - The degree to which a system is able to absorb the impacts of disruptive events and minimize their consequences with little effort.
- Adaptive capacity
 - The ability to adjust undesirable situations by undergoing some changes. The system can learn from the event, evaluate the system performance and modify its configurations, and make it more flexible to future disruptions

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15

Qualities of Resilience

- Restorative capacity
 - The ability to return to normal or improved operation.
- Recovery speed
 - The speed of the recovery process. Recovery may be accelerated if absorption activities are well implemented and the system can quickly mobilize and effectively use all the resources at its disposal.

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Simulation Data

- Outcomes from simulations are presented in tables
- Stated as a percentage of reduction

Table 2: Reductions in daily thermal stress in a heat wave without air conditioning, annual HVAC energy use, and annual HVAC carbon emission after application of cool envelope materials to a single family home in Los Angeles, California circa 2050 [9].

KPI	Baseline	Reduction from cool roof ^a	Reduction from cool wall ^b	Reduction from cool roof + cool wall ^c
Daily heat stress ^d [°C.h]	19	3%	7%	10%
Annual HVAC electricity need intensity ^e [kWh/m ²]	33	6%	14%	20%
Annual HVAC heating need intensity ^f [kWh/m ²]	27	-1%	-4%	-6%
Annual HVAC primary energy intensity ^g [kWh/m ²]	98	4%	8%	12%
Annual HVAC carbon emission intensity ^h [kg CO ₂ e/m ²]	15.1	3%	6%	9%

^a Daily degree hours of exceedance against a standard effective temperature (SET) of 30 °C during a heatwave without AC.

^b Annual HVAC electricity need per unit conditioned floor area

^c Annual HVAC heating need per unit conditioned floor area

^d Annual HVAC primary energy usage per unit conditioned floor area based on 2021 eGRID database for California-average primary energy factors of 2.05 for electricity and 1.09 for gas [10]

^e Annual HVAC carbon emission per unit conditioned floor area based on 2021 eGRID database for California-average CO₂ emission factors of 272 g/kWh electricity and 225 g/kWh [10]

^f Roof solar reflectance raised to 0.60 (bright-white asphalt shingle) from 0.10 (conventional asphalt shingle)

^g Wall solar reflectance raised to 0.60 (white paint) from 0.25 (conventional medium-lightness color paint)

Data from: S.H. Lee, R. Levinson, Cool envelope benefits in future typical weather and heatwave conditions for single-family homes in Los Angeles [accepted paper], in: Sixth International Conference on Countermeasures to Urban Heat Islands, RMIT University, Melbourne, Australia, 2023. <https://doi.org/10.20357/87DK6T>.

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Exemplary Chapter

[Exemplary TPS.pdf](#)

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Availability

Soon available at:

- IEA EBC Annex 80 Website

<https://annex80.iea-ebc.org/publications>

- Venticool Annex 80 Website

<https://venticool.eu/information-on-annex-80/annex-80-home/>

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Resilient Cooling Design Guidelines



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Preparing a new Annex-Proposal (?)

Resilient Cooling in Cities

Passive Climate Mitigation: Green, Blue and “White” infrastructures.

Hybrid Cooling: Hybrid Cooling solutions in time and space including latent cooling.

Heat sinks on urban scale: Seasonal storage in soil and water.
Deep Radiative Cooling.

Urban grids: Thermal & electric networks.



Cheonggyecheon stream revitalization project.
Downtown Seoul, South Korea, 2003 – 2005
Source: <https://www.ser-rrc.org/project/south-korea-restoration-of-the-cheonggyecheon-river-in-downtown-seoul/> (04.10.2022), foto credits: City of Seoul.

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Preparing a new Annex-Proposal (?)

If you are interested in joining this new Annex, please contact us!

Peter Holzer, peter.holzer@building-research.at

Philipp Stern, philipp.stern@building-research.at

Patryk Czarnecki, patryk.czarnecki@building-research.at

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Interactive Part

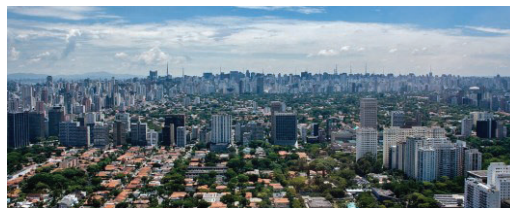
- Please divide in Groups of 3 to 5 people
- Each group receives one case building
- Work on case building for 20 minutes
- Presentation and discussion afterwards (15 minutes)

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Case Building 1 - São Paulo

- Hot and Humid Climate (ASHARE Climate Zone 2A)
- 15 story residential building with small business on ground floor

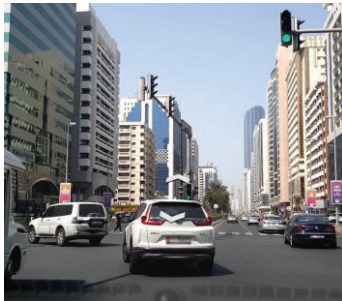


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Case Building 2 - Abu Dhabi

- Extremely Hot and Dry Climate (ASHARE Climate Zone 0B)
- 12 story residential building with small business on ground floor



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Case Building 3 - Los Angeles

- Warm and Dry Climate (ASHARE Climate Zone 3B)
- 5 story residential building with small business on ground floor

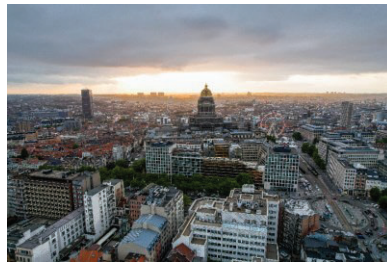


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Case Building 4 - Brussels

- Mixed and Humid Climate (ASHARE Climate Zone 4A)
- 9 story residential building with small business on ground floor



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Interactive Part

You are about to perform a retrofit to make the case building more resilient against heat waves and power outages.

- Are there any measures concerning the building envelop you would implement? If so, which and why?
- Are there any measures concerning the building interior and operation you would implement? If so, which and why?
- Are there any effective measures concerning the surrounding building exterior you would implement? If so, which and why?
- If you could impose any other measures (technical, political, societal, etc.) to improve the resilience against heat waves and power outages in general in urban surroundings, which would that be?

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Interactive Part – Abu Dhabi

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Interactive Part – Sao Paolo

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Interactive Part – Los Angeles

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Interactive Part – Brussels

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Contacts



Annex 80 at VENTICOOL
<https://venticool.eu/information-on-annex-80/annex-80-home/>

Annex 80 at IEA EBC
<https://annex80.iea-ebc.org/>

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Resilient Cooling Design Guidelines

Vincenzo Corrado

Politecnico di Torino – Department of Energy

October 2023



1

1

Introduction

- Within the IEA EBC Annex 80 project, key challenges, opportunities and framework associated with resilient cooling design have been identified and innovative concepts (solutions and strategies) that can help address these challenges have been explored.
- An extensive analysis of **different technologies, practices, and building designs** has been conducted to determine the most effective solutions for improving the resilience of buildings.
- To this regards, **disruptive events** and shocks such as **heatwaves** and **power outages** are the main points of interest.
- **The drafting of guidelines dedicated to resilient cooling design in buildings will help drive innovation, promote collaboration among different experts, and promote public awareness and engagement around the issue.**

2

2

Scope

The **Resilient Cooling Design Guidelines**:

- will describe **resilient cooling solutions** allowing for the reduction of overheating risks as well as the cost-effective, energy-efficient and low-carbon coverage of cooling demands;
- will integrate **state-of-the-art knowledge** based on the broad consensus of professionals ;
- will address the problem of **heat waves** and guide designers, building professionals, and owner representatives on the latest developments of **low-impact cooling technologies** in regard to their choice and sizing;
- will pursue the mission of improving health, comfort, safety, and energy efficiency in buildings and communities.

The main question to be answered is:

- **“How to design a resilient cooling building?”**

3

3

Objectives of the Guidelines

- 1) To develop **resilience cooling definitions** for buildings;
- 2) To propose well-established or new **metrics and key performance indicators** for the assessment of the design and solutions;
- 3) To examine and promote **simulation tools and methods** for evaluation of the design and concepts;
- 4) To define **inputs, weather files, building data, operational data** and others, for proper assessment of the performance;
- 5) To describe **technological profiles** of sustainable and energy-efficient cooling solutions and strategies;
- 6) To integrate them into **a demonstration case study**.

4

4

- The Resilient Cooling Design Guidelines are proposed as a **European Guidebook** to be published by REHVA through a joint task force with IEA EBC Annex 80.
- REHVA is the **Federation of European Heating, Ventilation and Air Conditioning associations**. It acts as an umbrella organization that represent over 120,000 HVAC designers, building services engineers, technicians and experts across 26 European Countries.
- REHVA's mission to help **the exchange of technical information** and knowledge transfer, and to disseminate the latest technology developments to foster high-quality European engineering practice.
- The most important outcome of this work are the **REHVA European Guidebooks**, written by REHVA Task forces, which are international groups of REHVA experts cumulating latest European knowledge about various topics in our field. REHVA experts have elaborated 31 guidebooks.

5

5

Target audience

- The target audience of this guidebook is **practitioners in the building design and architectural firms, and building services sector**, including designers, manufacturers, facility managers, and contractors.
- The market segmentation of the target audience indicates a **wide reach of potential readers and beneficiaries**.
- Geographically the report caters to EU countries but has a broad scope that can make it a highly used guide **beyond Europe**. The guide was written to cover the **eight climatic zones** defined by ASHRAE 169 and significantly contribute to professional building and energy experts in hot climates.

6

6

Potential use of the Guidelines

- The user profile of the target audience includes **consulting engineers, firms and national authorities, and building owners and tenants, policymakers, government officers, and building services institutions.**
- The guide is developed to address **small and mid-size facilities, including residential and commercial buildings.**
- Also, the guide is not only useful for **new construction** but also for **existing buildings** in regard to their operation, management, and maintenance.

7

7

Market of the Guidelines

- This guideline addresses a **priority topic**, which is reflected in the whole **legislative framework of the European Union** that assign an important role to buildings for climate mitigation and climate adaptation. The legislative framework of the European Union (EU) reflects the important role of buildings for climate change through several relevant policies.
- The potential customers of the Guidelines include both professionals and cooling technology companies.
- With reference to the European continent only, they can be listed over 500,000 building designers (source ACE) and more than 120,000 HVAC designers and building services engineers (source REHVA).
- As regards cooling technology companies, among the other, about 117,000 people work in the European heat pump industry (source EHPA), more than 450,000 people are employed in the solar shading industry (source ES-SO).

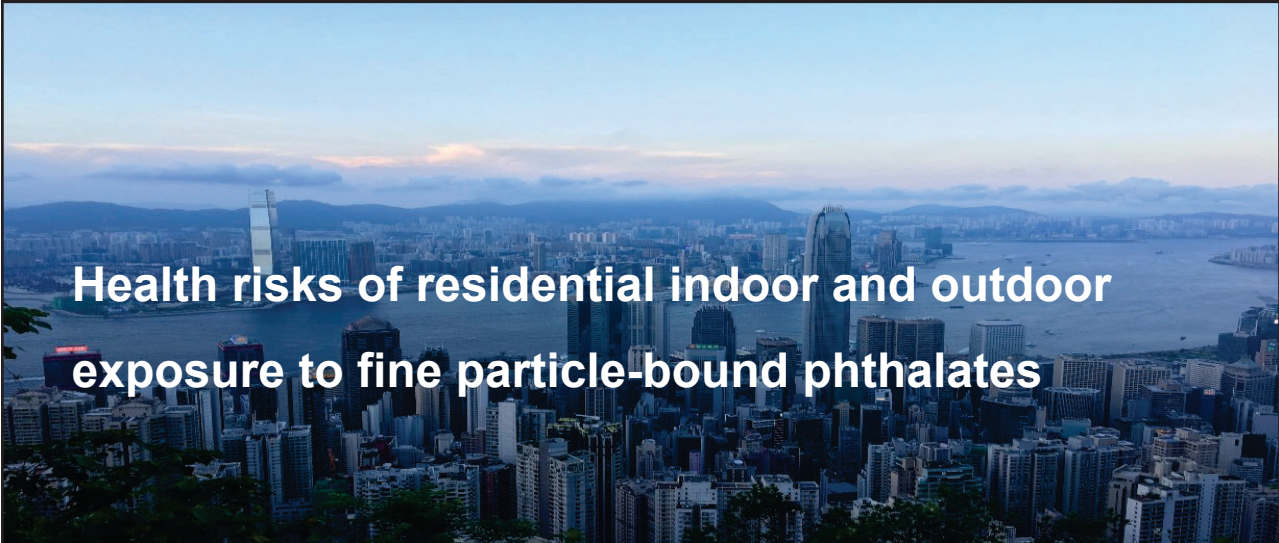
8

8

Table of contents

1. Introduction to resilient cooling (general context, design framework)
2. Definition of resilience and disruptions
3. Resilient cooling solutions (strategies, technologies, components)
4. Key performance indicators for evaluation of resilience in buildings
5. Building performance assessment methods and tools (whole building, specific resilient cooling technologies, simulation tools, calibration and validation)
6. Weather data (present vs future climate data, heat waves, urban effects)
7. Occupancy patterns (internal conditions, heat gains, bldg. operation)
8. Building data (description of geometry, constructions, technical bldg. systems)
9. Developed case study 1 & 2

10



UCD School of Architecture, Planning and Environmental Policy
UCD Spatial Dynamics Lab (SDL)

Dr. Jiayao CHEN, Prof. Francesco PILLA

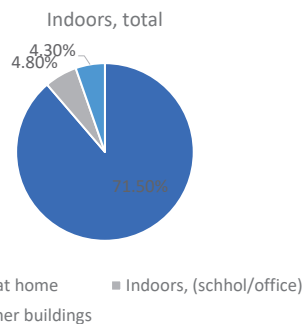
jiayao.chen@ucd.ie; francesco.pilla@ucd.ie

4 October 2023, Conference,
Copenhagen- 43rd AIVC conference

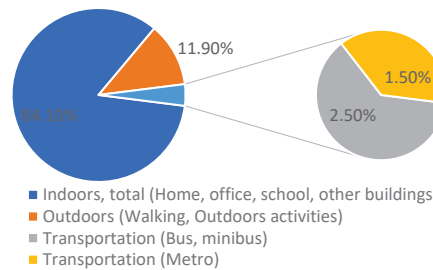
1

Identify the problem

- The negative impacts of phthalates (PAEs) on human health have raised global concerns due to their widespread use in polyvinyl chloride (PVC) products and consumer products (such as commodities, medical products, cosmetics, and personal care products) and in households (building materials, furnishing, household goods) .
- The International Agency for Research on Cancer (IARC) has classified DEHP and BBP as possible human carcinogens (Group B2 and Group C) based on experimental animal studies.



Daily activity patterns in the adult population of Hong Kong (Survey in 2014-2015, N = 229)



2

Methods/Solutions

• Study design

- Indoor monitoring for PM_{2.5} was conducted in 26 adult subjects (ages 18–63 years) in Hong Kong (Fig. 1).
- Outdoor PM_{2.5} samples were collected concurrently at fixed sites in the same district as residential indoors.

• Sampling and instruments

- Mini-Volume air sampler was placed at ~1.5 m above the ground in the participants' living room and operated at a flow rate of 5 (±0.25) L/min for 24 h. Two to four samples were obtained from each household, leading to 63 sets of indoor PM_{2.5} samples, with 63 sets of outdoor PM_{2.5} samples collected.

• Analytical methods

- A punch (0.526–2.630 cm²) of the exposed quartz filter was subjected to particle-phase organic compound determination employing the thermal desorption–gas chromatography/mass spectrometer (TD–GC/MS) method.

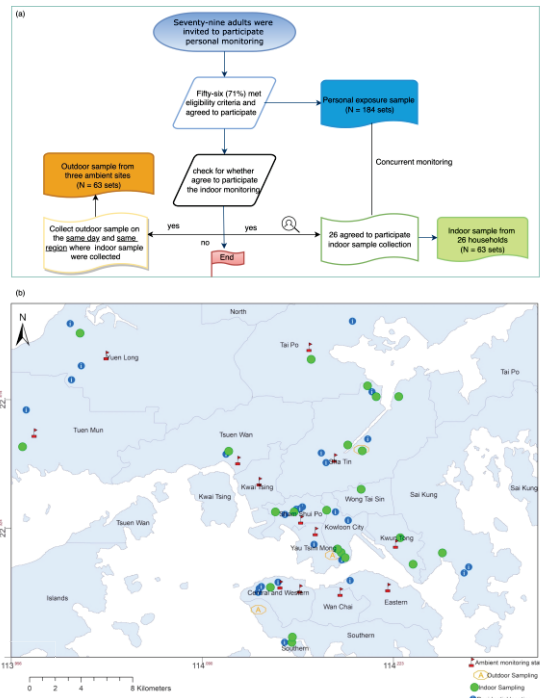
• Statistical analysis

- PCA for source identification
- Health risk assessment:



$$EC_i = \frac{C_i \times ET \times EF \times ED}{AT} \quad (Eq. 1)$$

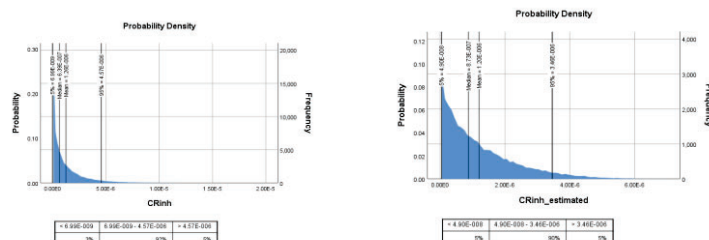
$$DI_{i,inh} = \frac{EC_i \times IR_{inh} \times EF \times ED}{BW \times T \times CF} \quad (Eq. 2)$$



3

Results & Discussion

- DEHP [di(2-ethylhexyl) phthalate] was the most abundant PAE congener (80.3%–85.0%).
- Low-molecular-weight (LMW) PAEs (i.e., DMP, DEP) tended to be present in the gas phase.
- Strong correlations for DEHP with DnBP ($r_s = 0.88$; $p < 0.01$) (e.g., household products, plasticizers), BBP ($r_s = 0.83$; $p < 0.01$) (PVC flooring), and DnOP ($r_s = 0.87$; $p < 0.01$) (polymer products) were shown in residential indoor.
- The highest average I/O ratio was shown for DnBP (4.8), suggesting indoor DnBP source.
- Outdoor monitoring at fixed sites could not capture indoor origin pollutants.
- The inhalation cancer risks attributable to measured and estimated personal exposure to DEHP exceeded the U.S. EPA's benchmark (1×10^{-6}).



4

Thank you!



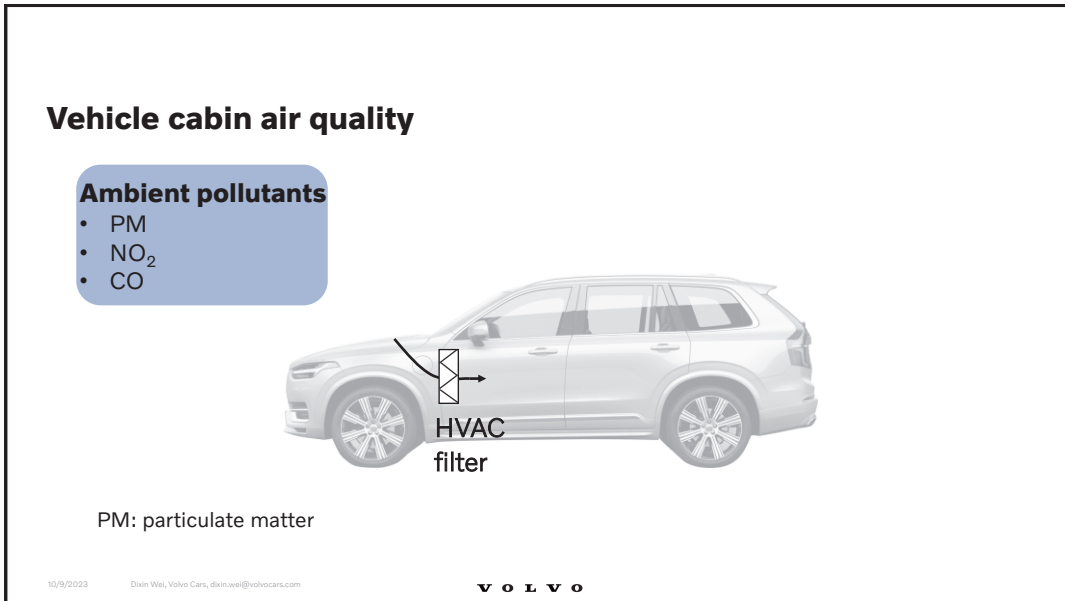
Acknowledgements: This study was supported by the National Natural Science Foundation of China (Grant No. 41907181). Full article: <https://www.mdpi.com/1660-4601/19/20/13425>



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1



2

Vehicle Measurements

HVAC air inlet

Pre-filter



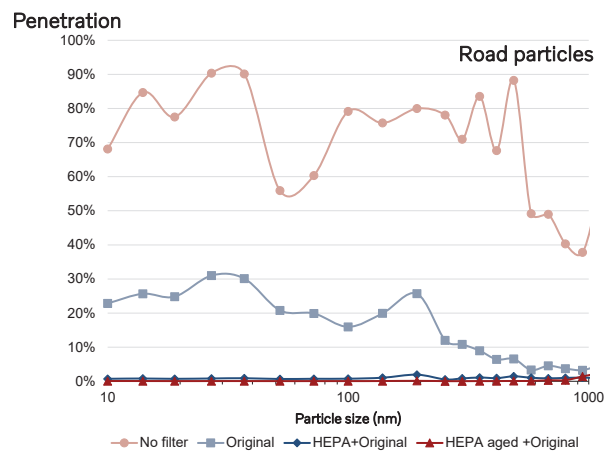
Front storage in BEV

10/9/2023 Dixin Wei, Volvo Cars, dixin.wei@volvocars.com

VOLVO

3

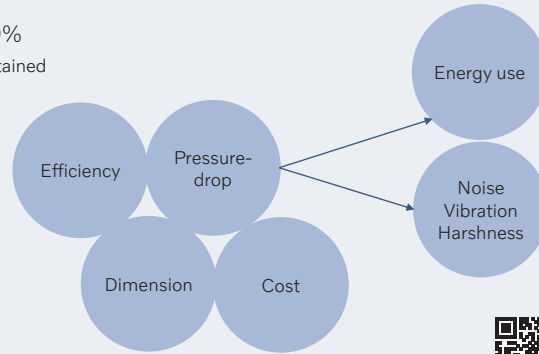
Results- Vehicle Measurements



4

Conclusions

- EPA, HEPA filter -> efficiency above 99%
- Aged HEPA filter -> performance maintained
- Balance in practical application



Scan for more info

Experimental study of an innovative wet scrubber concept in regards to particle filtration and pressure loss

Nhat Nguyen, Martin Kremer, Hendrik Fuhrmann, Philipp Ostmann, Dirk Müller

RWTH Aachen University, E.ON Energy Research Center, Institute for Energy Efficient Buildings and Indoor Climate, Aachen, Germany

Nhat Nguyen

EBC | Institute for Energy Efficient
Buildings and Indoor Climate

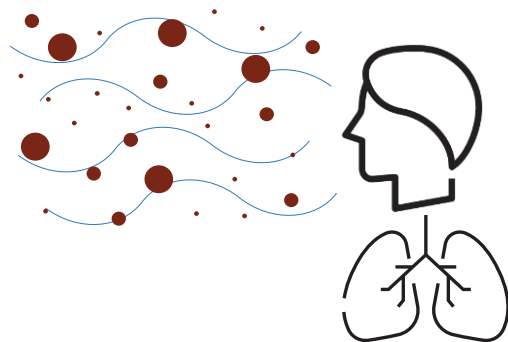


1

Motivation

Health issues due to particulate pollutants and VOCs

- Fine and coarse particles penetrate deep into the respiratory tract
 - ≡ Increasing mortality & morbidity [1]
 - ≡ Removal of particles to reduce risk to human health
- Wet scrubbers can filter both solid and liquid pollutants
 - ≡ High energy demand
- Increasing removal efficiencies for new concepts
 - ≡ Shift focus to energy efficiency
- ➔ Investigating wet scrubber concept with low energy demand and resulting particle removal efficiencies



[1] World Health Organization (2016). Ambient air pollution: a global assessment of exposure and burden of disease.

2

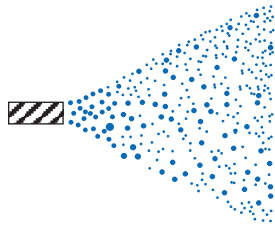
Experimental study of an innovative wet scrubber concept in regards to particle filtration and pressure loss | Nhat Nguyen
Institute for Energy Efficient Buildings and Indoor Climate | AIVC 2023 | 05.10.2023



2

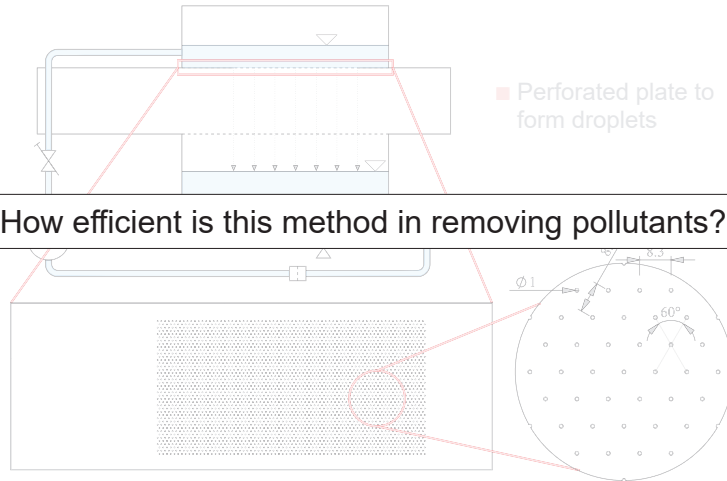
Different wet scrubber concept Perforated plate instead of atomization

Conventional Wet Scrubbers



- Atomization of injected water
 - ≡ High energy demand
 - ≡ High pressure losses
- High removal efficiencies

New Wet Scrubber Concept



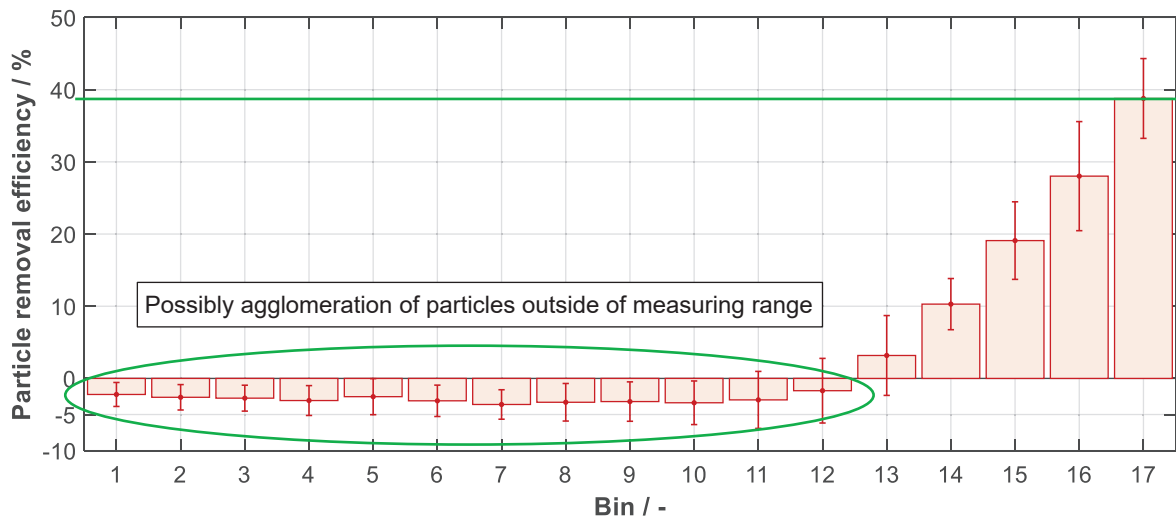
3

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3

Results: Particle Removal Efficiency



4

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4

Conclusions and Outlook

Conclusions:

- Overall low particle removal efficiencies due to low number of droplets
- Investigated wet scrubber concept has low pressure losses

Outlook:

- Future studies: Investigation of negative particle removal efficiencies
- Future studies: Increasing particle removal by modifying dripping behavior and droplet size



5

Experimental study of an innovative wet scrubber concept in regards to particle filtration and pressure loss | Nhat Nguyen
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5

Thank You For Your Attention!

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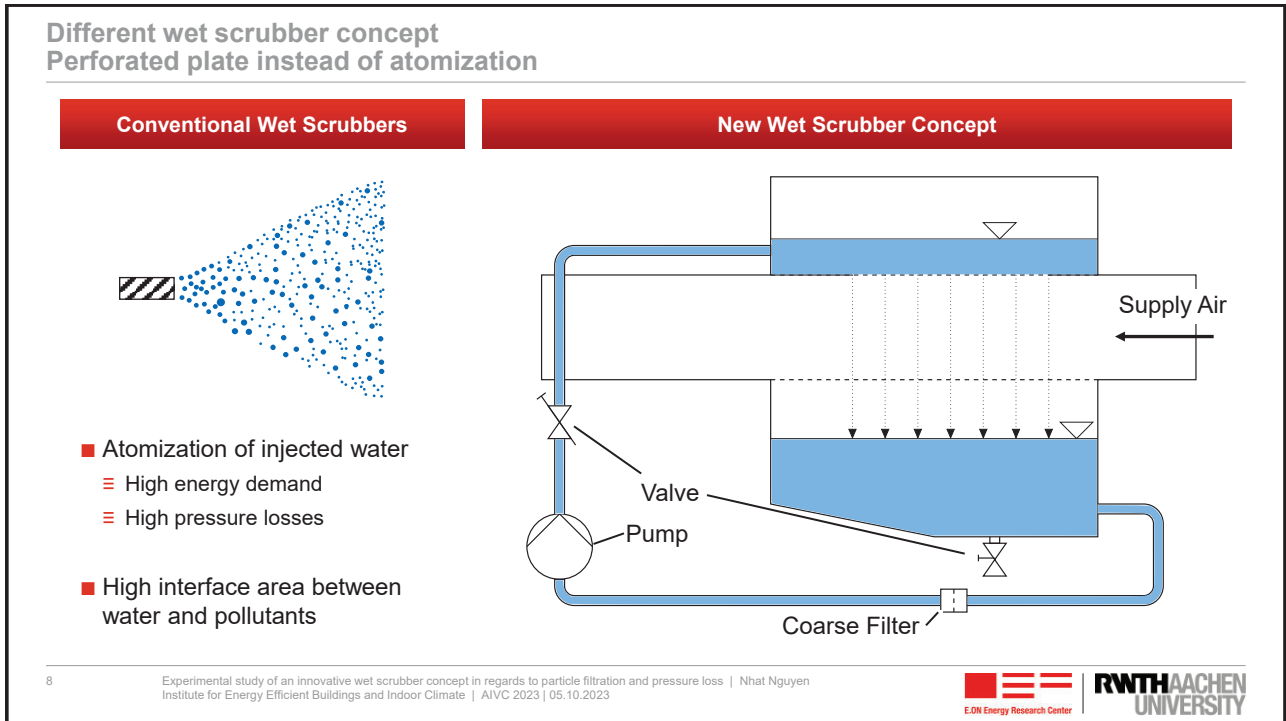
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6

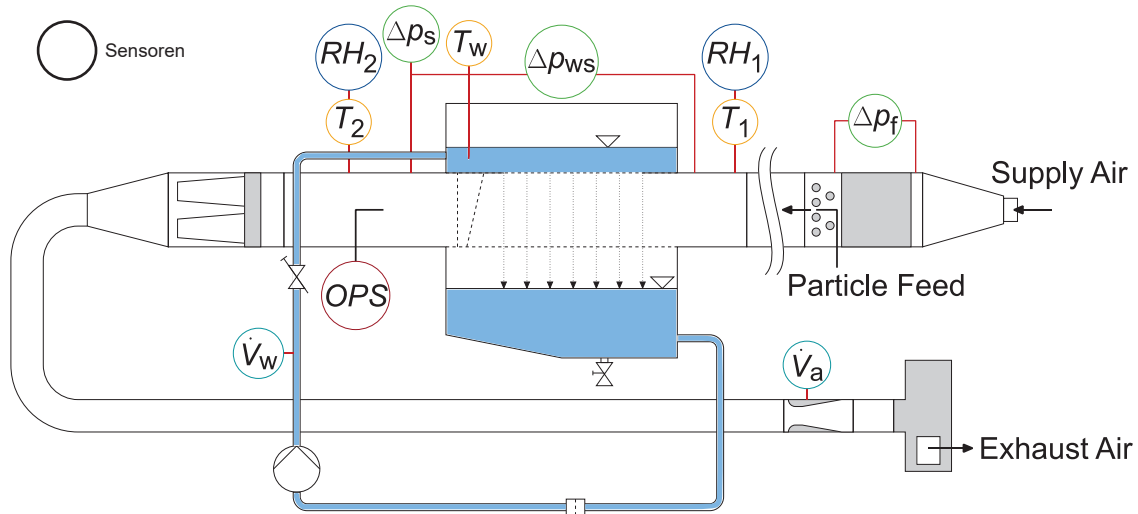


7



8

Schematic of test bench



9

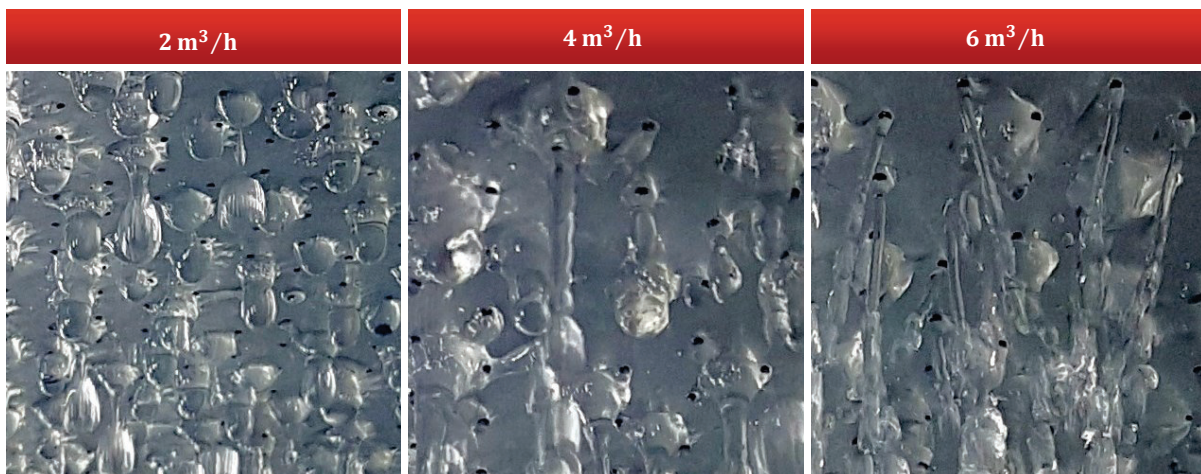
Experimental study of an innovative wet scrubber concept in regards to particle filtration and pressure loss | Nhat Nguyen
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9

Droplet Behavior



- Droplets larger than holes of perforated plate due to surface wetting → water accumulation before dripping

10

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10



UC | Chile

An evaluation of CO₂ emission rates by Chilean school children

Nicolás Carrasco¹, Constanza Molina^{*1}, and Benjamin Jones²

[*cdmolina@uc.cl](mailto:cdmolina@uc.cl)

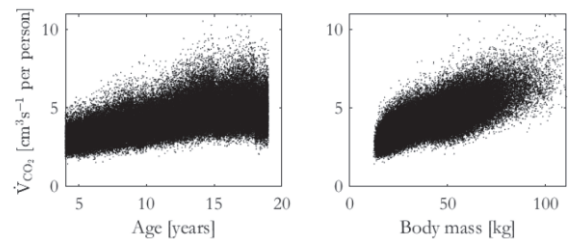
¹ Pontificia Universidad Católica de Chile ² Department of Architecture and Built Environment, University of Nottingham, England

AIVC2023

1

Introduction

- Carbon dioxide (CO₂) levels are used to estimate ventilation rates in occupied spaces.
- Recent research showed children's CO₂ emission rates vary significantly between ages 4-19 due to body mass changes.
- One study derived US child **emission rates of 3.1-5.1 ppm m³ s⁻¹ per person** using government weight data.
- *As US child body mass may differ from other countries, emission rates based on US data may not apply elsewhere*



Are the expected CO₂ concentrations in Chilean schools different from those in US schools?

2

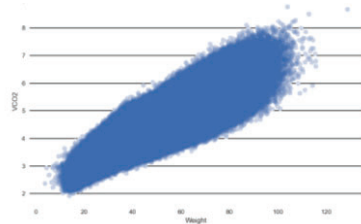
Method

Multiple simulation of classrooms of **29 students** from 5 to 18 years of age, until a normal distribution is obtained for their means



$$\dot{V}_{CO_2} = BMR \cdot M \cdot RQ \cdot VO_2 \cdot T \cdot P^{-1}$$

$$Q_o = \dot{V}_{CO_2} (C_{i,ss} - C_o)^{-1} = \dot{V}_{CO_2} C_{e,ss}^{-1}$$



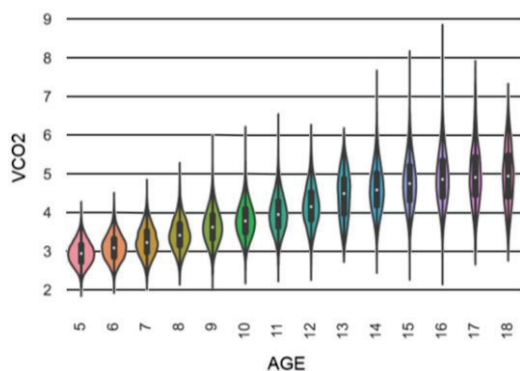
Body mass, BMR, is the most important input in emission rates.

Qo for US and Chilean classrooms following **ASHRAE Standard 62.1** adds per capita airflow and 0.6L/s per unit floor area, and the **ASHRAE Standard 241** focuses on controlling infectious aerosols with 20L/s per person.

Then, compared both Chile and US samples to see if there are significant differences in emission rates and $C_{e,ss}^{-1}$.

3

Results and conclusions



Age	Median \dot{V}_{CO_2}		Excess concentration, $C_{e,ss}$ ppm			
	USA	Chile	ASHRAE 62.1		ASHRAE 241	
	USA	Chile	USA	Chile	USA	Chile
5	3.1	3.0	420	490	160	150
6	3.3	3.1	440	510	160	160
7	3.4	3.2	460	540	170	160
8	3.6	3.4	490	570	180	170
9	3.9	3.6	580	600	200	180
10	3.9	3.8	580	630	200	190
11	4.1	4.0	610	660	210	200
12	4.3	4.2	640	700	220	210
13	4.5	4.4	670	740	230	220
14	4.8	4.6	720	760	240	230
15	5.0	4.8	750	790	250	240
16	4.9	4.9	730	810	250	240
17	5.0	5.0	750	830	250	250
18	5.1	5.0	760	830	260	250

There is very little difference between countries for the same standard

4

Thank you

Nicolás Carrasco¹, Constanza Molina^{*1}, and Benjamin Jones²

¹ Pontificia Universidad Católica de Chile ² Department of Architecture and Built Environment, University of Nottingham, England

* Corresponding author: Constanza Molina cdmolina@uc.cl



Part of this work and presentation was financed by ECCUC / PUC University



The Effects of Bedroom Mechanical Ventilation on Health and Sleep Quality

Speaker: Jeong Won Kim
12210197@dankook.ac.kr

Jeong Won Kim, Dankook University
Sun Ho Kim, Dankook University
Yong Kyu Biak, Seoil University
Hyeun Jun Moon, Dankook University

1

Background

▪ Healthy Indoor air quality

- Humans typically spend about one-third of their total time sleeping
- Pursuit healthy indoor air quality(IAQ) as improve the quality of life
- Adverse health effects
 - Poor sleep: energy restoration, physical and mental recovery, maintaining bodily functions, etc.
 - CO₂: Dyspnea, psychological and physical fatigue, etc.



▪ Comfort sleep environment

- Ensuring good indoor air quality is important for maintaining the quality of sleep.
- However, occupants may not always consciously implement planned ventilation in their bedrooms.

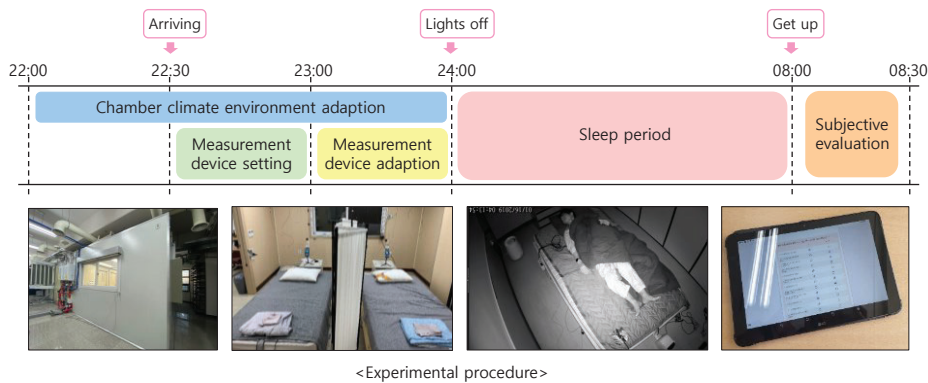
 **This study aims to analyse the difference in sleep efficiency based on the operation of bedroom ventilation systems in the intermediate season.**

2

Method

Experimental design

- 2-hour indoor temperature acclimation for each experimental condition.
- Elimination of any disruptive factors during sleep for each experimental condition from midnight (00:00) to 8:00 AM (total 8 hours).
- Conducting a sleep satisfaction survey.



3

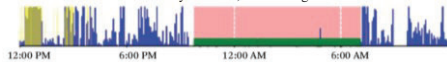
Method

Sleep assessment methodology

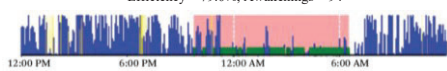
- Quantifying sleep efficiency using the wrist-worn Actigraph device equipped with a 3-axis accelerometer sensor.
- Utilizing the previously developed survey items from the Subjective sleep quality assessment (Zilli et al., 2009).



Efficiency = 98.0%, Awakenings = 7



Efficiency = 79.0%, Awakenings = 94



<Sleep efficiency(Actigraph)>

Questions	Score
1. How was your sleep like last night? (Calmness of sleep)	1-5
2. How easy was it to fall asleep last night? (Ease of falling asleep)	1-5
3. How easy was it to wake up this morning? (Ease of awakening)	1-5
4. Did you feel refreshed after waking? (Freshness after awakening)	1-5
5. Are you satisfied with your last night's sleep? (Satisfaction about sleep)	1-5
6. Do you think you get enough sleep? (Sufficient sleep)	Yes(1) or No(0)



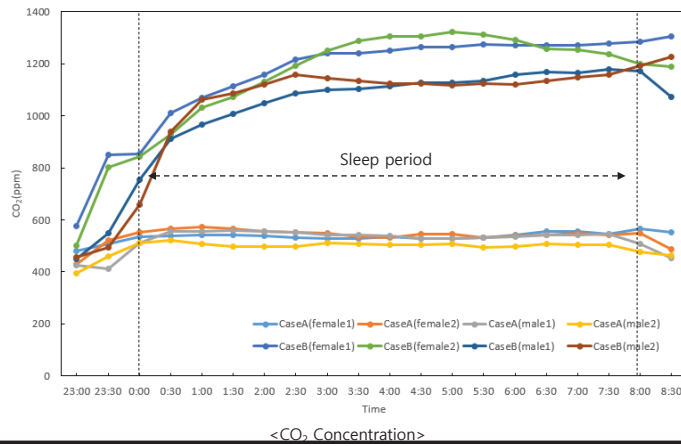
<Subjective Sleep Quality Assessment survey>

4

Result

▪ Sleep environment

- Case A: Ventilation On(2.78ACH), Case B: Ventilation Off
- Case A : CO₂ ranged from 492 to 537 ppm
- Case B : CO₂ ranged from 1,019 to 1,152 ppm

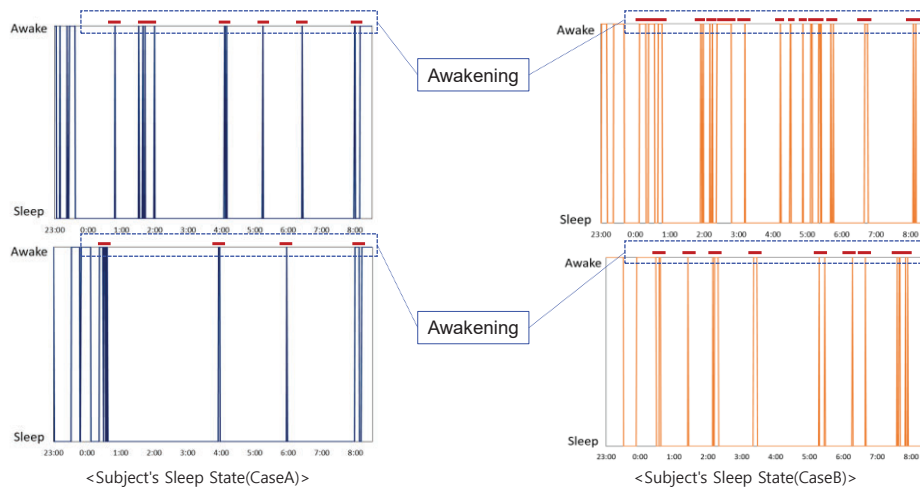


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Result

▪ Awakenings during sleep

- Decreased by 66% in awakening
 - Case A averages 17.5, Case B average 52.3

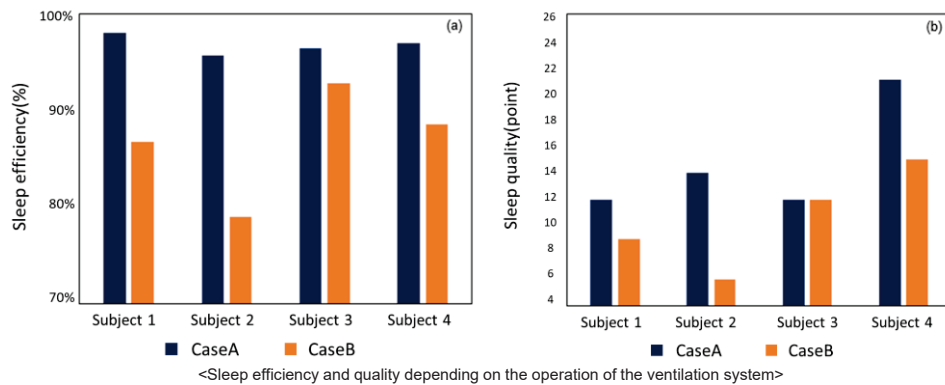


6

Result

▪ Comparison of ventilation and without ventilation

- Increased by 17.5% in sleep efficiency
 - Case A average 97%, Case B average 86%
- Increased by 57% in sleep quality
 - Case A average 15 points, Case B average 10 points

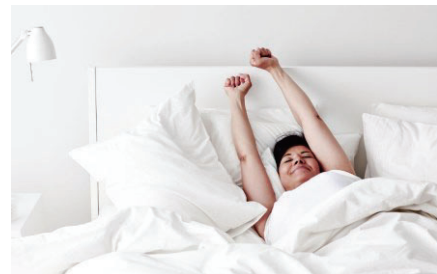


7

Conclusions

▪ Sleep experiment considering intermediate season ventilation system

- Reduced awakenings during sleep, leading to improved objective sleep efficiency
- Increase occupant's sleep comfort



▪ Further research

- Need to expand the number of participants and conduct experiments with various combinations of sleep parameters.

8

Thank you for your attention!

Speaker: Jeong Won Kim

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Jeong Won Kim, Dankook University

Sun Ho Kim, Dankook University

Yong Kyu Biak, Seoil University

Hyeun Jun Moon, Dankook University

Analysis of PM_{2.5} indoor-outdoor ratio in lobby floor according to configurations of entrance

2023. 10. 04

INHA UNIVERSITY
Park, So-Yi
Jo, Jae-Hun
(Email: jhjo@inha.ac.kr)

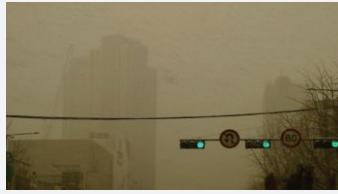
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CONTENTS

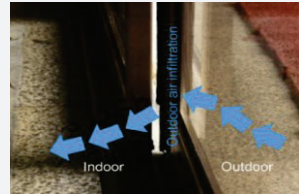
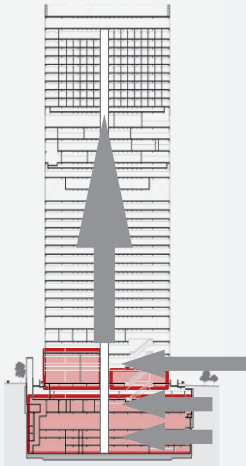
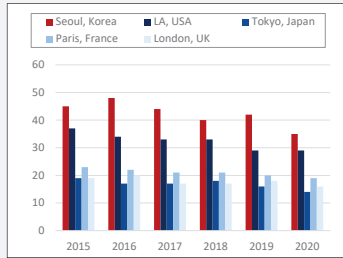
- 1. Introduction**
- 2. Approach**
- 3. Impact of entrance door on PM_{2.5} IO ratio**
- 4. Conclusions**

1

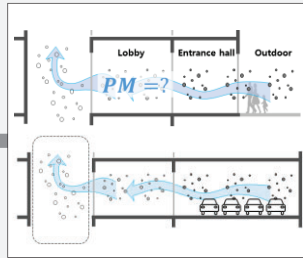
1. Introduction



High PM concentrations in outdoor ambient



Pollutant Transfer

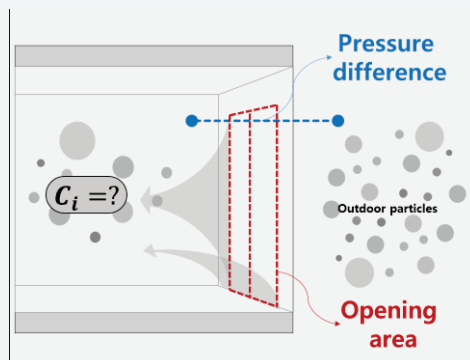


- Outdoor PM_{2.5} has a continuous and significant effect on the indoor environment.
- It is necessary to evaluate the impact of entrance doors on indoor PM_{2.5} to implement appropriate particle control measures.

2

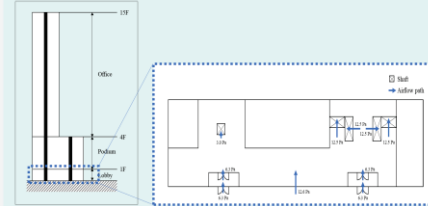
2. Approach

- Penetration of PM through entrance doors



PM_{2.5} I/O ratio for lobby floors was evaluated according to the operation type and configuration of entrance doors

- Pressure difference: airflow analysis



- Opening area: Case study

- I. Operation type : leakage area
- II. Operation time difference : occupancy traffic

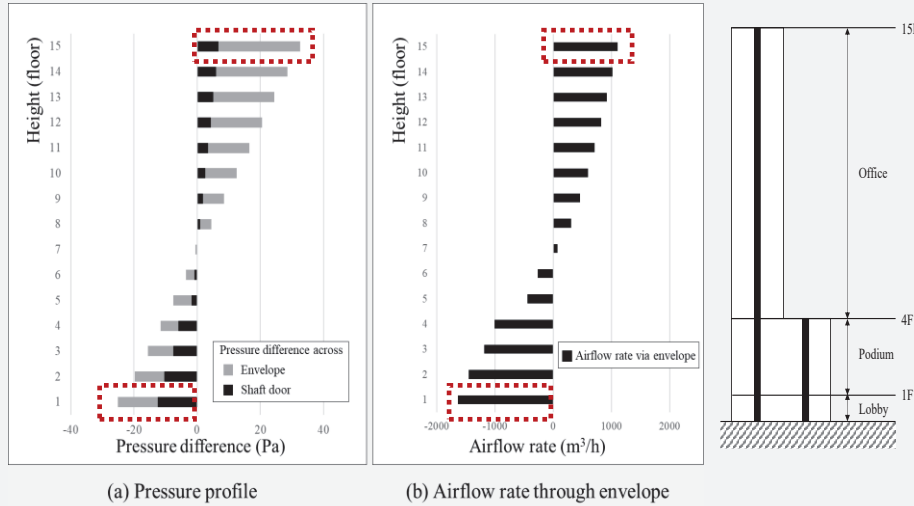
Parameter	Case II			
	Time difference of door operation			
Door type	Step (leakage)			
Air leakage area	150 cm ² @5Pa			
Opening area	1.8 m ²			
	0 s	1 s	2 s	3 s
	Step (stationary)	Operating (moving)		
	150 cm ² @5Pa	600 cm ² @5Pa		

3

3. Impact of entrance door on PM2.5 IO ratio

• **Airflow analysis for model building**

- Lobby level (first floor) had a higher airflow rate (1,634 m³/h) than the top floor (1,102 m³/h) due to entrance doors and a larger envelope area.

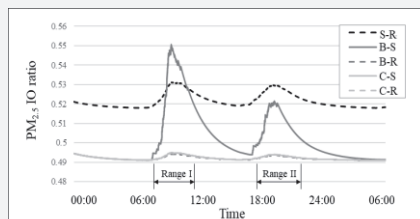


4

3. Impact of entrance door on PM2.5 IO ratio

• **Case study**

I. Operation type (Air leakage area)

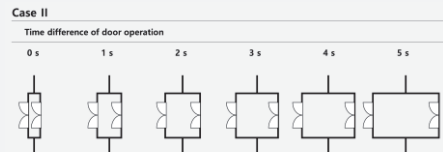
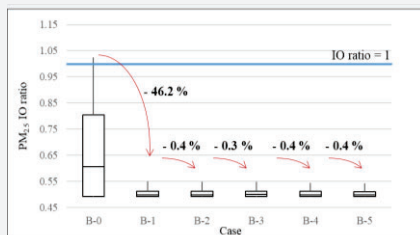


Parameter	Case I
Door type	Stop (close)
Air leakage area	150 cfm@50Pa
Opening area	Operating (open) 1.8 m ²
	Stop (stationary) 150 cfm@50Pa
	Operating (revolving) 600 cfm@50Pa

Plat type	Box type	Combo
outside	Double	C-S
inside	B-S	C-S
P-S	B-S	C-S

Vestibules can reduce PM concentrations, but consideration needs to be given to occupant traffic and door operation.

II. Operation time difference



Even with a vestibule strategy, depending on the architectural design (door spacing), it can result in a similar performance to a single-type operation.

5

4. Conclusions

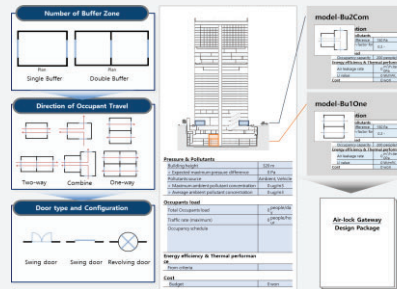
• 6

The lobby floor's I/O ratio to outdoor PM2.5 concentrations was examined with a case study.

1. Lobby space's penetrated PM2.5 varies due to differing airflow rates from various door operations.
2. A single-type entrance without a vestibule connects directly to outdoor ambient, potentially exposing it to PM2.5 levels equal to or higher than outdoor concentrations.
3. Entrances with vestibules can experience high concentrations if there's no difference in door operating times.

Limitation and Future Work

- Depicting the entire building, door geometry and components are simplified, requiring detailed airflow analysis.
- Development of entrance systems that include architectural design and operational systems to ensure indoor air quality.



6

• 7

Thank you

7

Proposal of an effort-benefit diagram to compare unit and room air-change rates applied to a literature review

Sven Auerswald
43rd AIVC & 11th tightvent & 9th venticool conference
Copenhagen, 4th of October
www.ise.fraunhofer.de

1

Agenda

1. Air schange efficiency - absolute
2. Air change efficiency - relative
3. Effort-Benefit-Diagram air exchange - Applied for a literature review

2

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2

Air change efficiency absolute

- Efficiency ratio for the air exchange for the whole space j ventilated
- Valid interval: 0 ... 1
 - Complete ventilation short-cut: 0
 - Ideal mixed ventilation: 0.5
 - Plug flow condition: 1

$$\epsilon_j^a = \frac{\tau_j}{\langle \bar{\tau}_j \rangle} = \frac{1}{2} \frac{\tau_j}{\langle \bar{\alpha}_j \rangle} = \frac{1}{1 + \mu_2^*(\tau_j)} = \frac{1}{2} \frac{\langle \bar{n}_j \rangle}{n_j} = \frac{\langle \bar{n}_{e,j} \rangle}{n_j}$$

ϵ_j^a	Absolute air change efficiency of a system j	[-]
τ_j	Nominal time constant of the system j as a characteristic statistical measure for the time air spends at least within that system	[h]
$\langle \bar{\tau}_j \rangle$	Average residence time of air within the system j or air age in the exhaust plane of that system	[h]
$\langle \bar{\alpha}_j \rangle$	Average air age within the system j	[h]
$\mu_2^*(\tau_j)$	Dimensionless second order central moment or dimensionless variance of the statistical distribution of residence times outside the system	[-]
$\langle \bar{n}_j \rangle$	Average room air change rate	[h ⁻¹]
n_j	Nominal air change rate of a system j	[h ⁻¹]
$\langle \bar{n}_{e,j} \rangle$	Average room air change rate in the exhaust plane of the system j	[h ⁻¹]

3

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3

Air change efficiency relative

- Efficiency ratio for the air exchange for a **subsystem i** inside the **space j** ventilated.
 - A subzone can be the breathing level.
- Valid interval: 0 ... 1
 - ventilation short-cut: 0
 - mixed ventilation: 0.5
 - Plug flow condition: 1

$$\langle \epsilon_j^a \rangle_i = \frac{\tau_j}{\langle \bar{\tau}_j \rangle_i} = \frac{1}{2} \frac{\tau_j}{\langle \bar{\alpha}_j \rangle_i} = \frac{1}{2} \frac{\langle \bar{n}_j \rangle_i}{n_j} = \frac{\langle \bar{n}_{e,j} \rangle_i}{n_j}$$

$\langle \epsilon_j^a \rangle_i$	Relative air change efficiency of a subsystem i in j	[-]
$\langle \bar{\tau}_j \rangle_i$	Average residence time of air within the subsystem i or air age in the exhaust plane of that subsystem	[h]
$\langle \bar{\alpha}_j \rangle_i$	Average air age within the subsystem i in j	[h]
$\langle \bar{n}_j \rangle_i$	Average room air change rate in the subsystem i	[h ⁻¹]
n_j	Nominal air change rate of a system j	[h ⁻¹]
$\langle \bar{n}_{e,j} \rangle_i$	Average room air change rate in the exhaust plane of the subsystem i	[h ⁻¹]



4

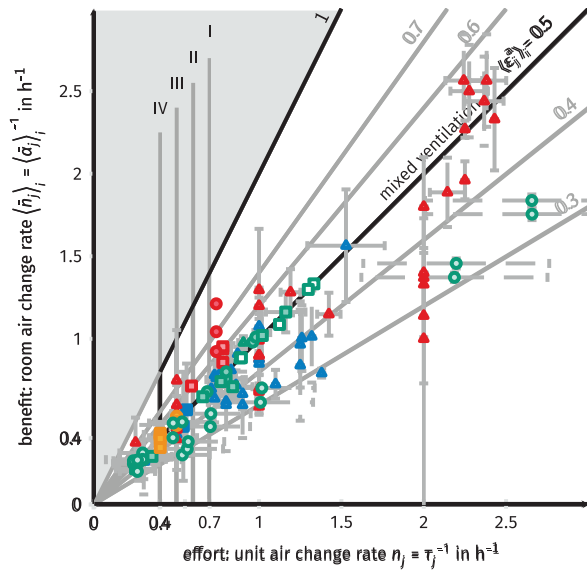
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4

Effort-Benefit-Diagram air exchange Applied for a literature review

Vertical lines represent the indoor environmental quality categories from EN 16798

-  red = test facilities
-  orange = simulations residential
-  green = measurement residential
-  blue = measurement non-residential
-  triangle = centralized continuous
-  rectangel = decentralized continuous
-  round = decentralized alternating



5

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5





Vielen Dank für Ihre
Aufmerksamkeit

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6



Experimental Investigation of Indoor Air Quality in an Open Office Environment

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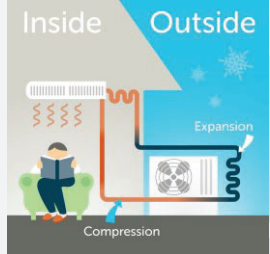
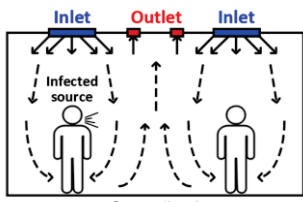
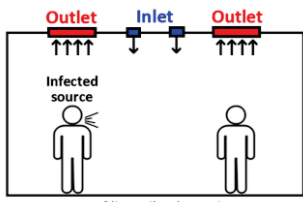


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1

INTRODUCTION

- IAQ measurements
- Different ventilation strategy
- Heat pump application in a cold climate

Inputs for Energy Balance

Metabolic Rate
Clothing Insulation
Air Temperature
Air Velocity
Mean Radiant Temperature
Relative Humidity

PMV Index

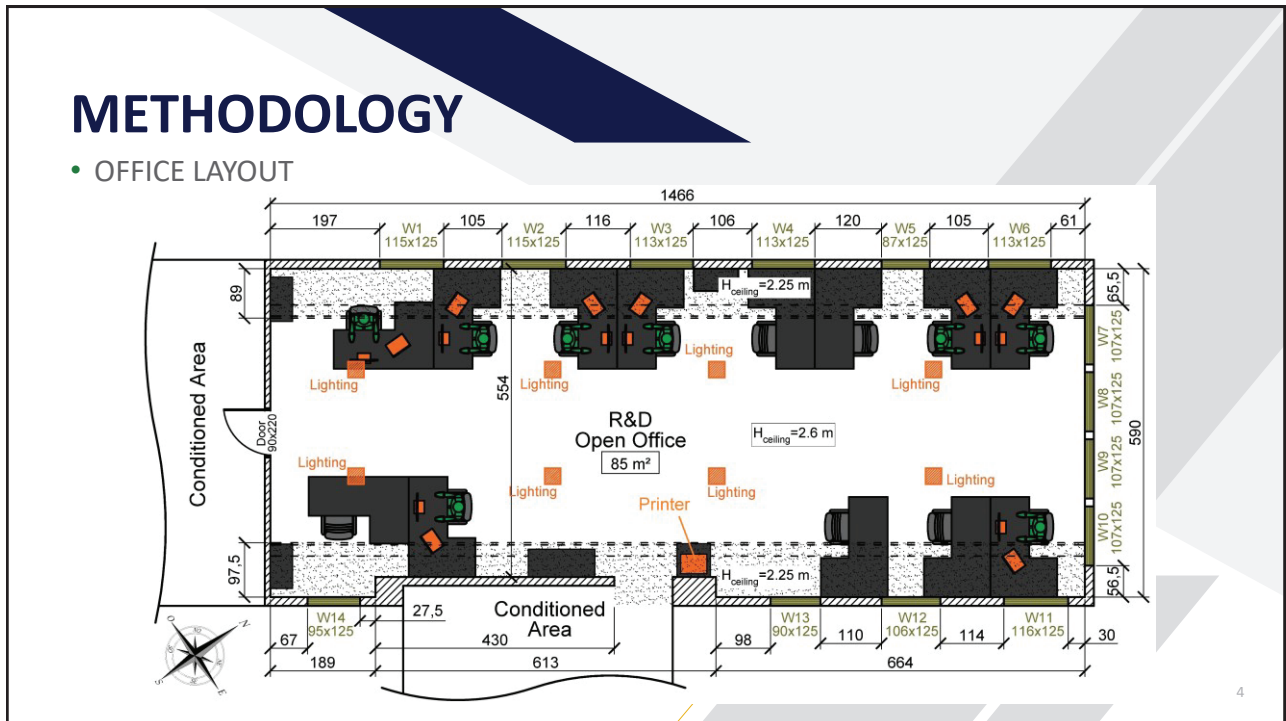
+3	Hot
+2	Warm
+1	Slightly Warm
0	Neutral (Comfort)
-1	Slightly Cool
-2	Cool
-3	Cold

Storage = Production - Loss

2



3



4

METHODOLOGY

DESIGN CALCULATIONS

Design Conditions		Outdoor Air Rate		Cooling – Heating Load Calculations				
Parameter	Value	Parameter	Value	Load Source	Load Details	Cooling		Heating
						Sensible	Latent	Sensible
^a Latitude	40.079 N	^b q _{occ}	2.5 L/s.person	Q _{windows}	18 m ²	2733 W	-	1497 W
^a Longitude	32.566 E	N _{occ}	13 person	Q _{walls}	48 m ²	80 W	-	369 W
^a Elevation	843 m	^b q _{area}	0.3 L/s.m ²	Q _{roof}	84 m ²	1485 W	-	1910 W
Design Heating		Area	85 m ²	Q _{lighting}	8 lights	422 W	-	-
^a DB	-12°C	q _{req}	58 L/s (208.8 m ³ /h)	Q _{equipment}	8 laptops	500 W	-	-
Setpoint	22°C	^b E _z	0.8		1 printer			
Design Cooling		^b E _v	0.66	Q _{occ}	13 people	975 W	780 W	-
^a DB	31.9°C	^b q _{calc}	396 m ³ /h	Safety	10%	620 W	78 W	378 W
^a WB	16.8°C			(Q _{tot}) _{space}	-	6815 W	858 W	4154 W
Setpoint	24°C	q _{safe}	514 m ³ /h	(Q _{tot}) _{vent}	600 m ³ /h	976 W		6058 W
		q _{design}	600 m ³ /h	Q _{tot}		8649 W		10212 W

^a The data were retrieved from ASHRAE Climatic Design Conditions website (ASHRAE, 2021).

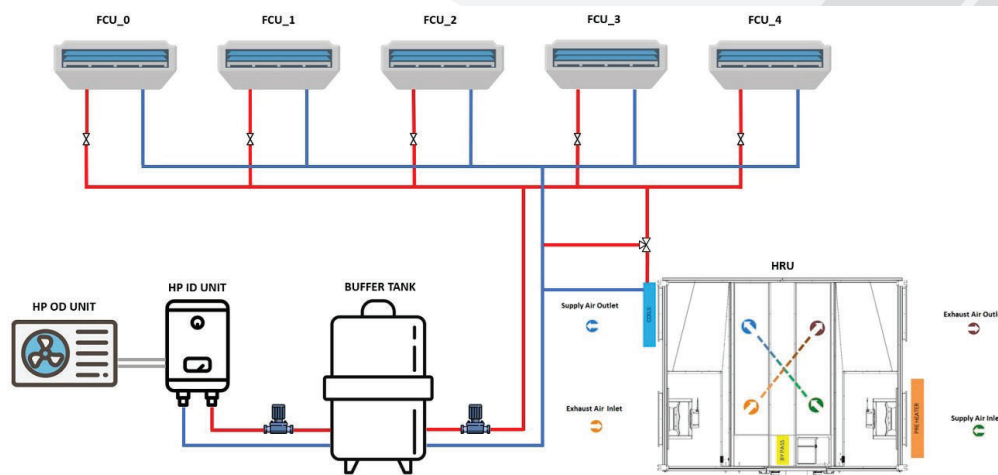
^b Values were retrieved from ANSI/ASHRAE Standard 62.1 "Ventilation for Acceptable Indoor Air Quality" (ANSI/ASHRAE, 2019). Minimum air requirements per each occupant (q_{occ}) and per unit area (q_{area}) were obtained by considering the space as an office, whereas the air distribution effectiveness (E_z) and system ventilation efficiency (E_v) were defined by evaluating the ventilation layout described in Figure 1 and Figure 2. The total amount of required outdoor air rate (q_{calc}) was calculated in accordance with the methodology presented in the standard.

5

5

METHODOLOGY

SCHEMATIC DIAGRAM

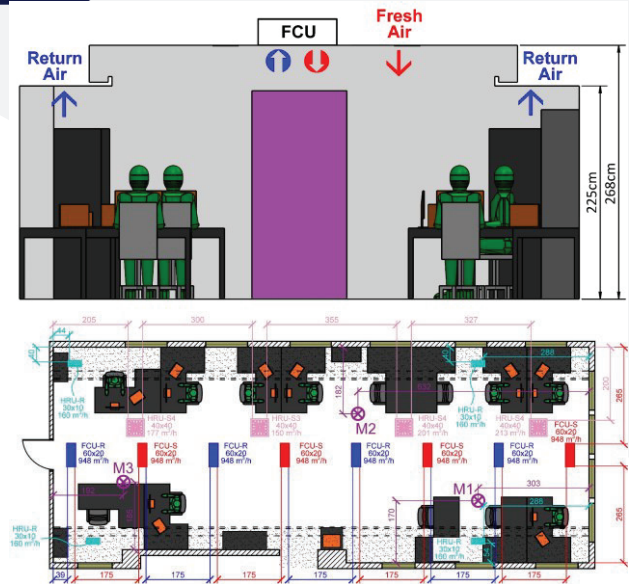
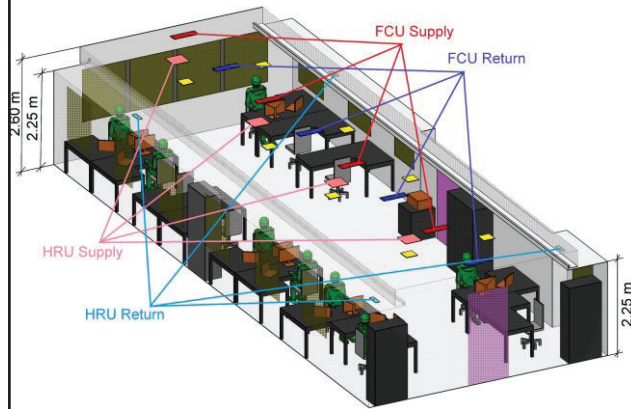


6

6

METHODOLOGY

• HVAC LAYOUT



7

METHODOLOGY

• Device Properties

- Testo 440
- Globe Thermometer
 - Emissivity (ϵ)=0.95
- CO2 Probe
 - 0-10000 \pm 50 ppm
 - 0-50 \pm 0.52°C
 - 5%-95% \pm 5% RH
- Turbulence Probe
 - 0-5 \pm 0.03 m/s
 - 700-1100 \pm 0.1 hPa
- PMV-PPD calculated

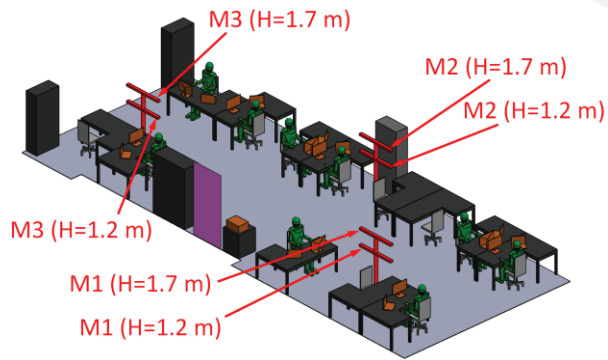


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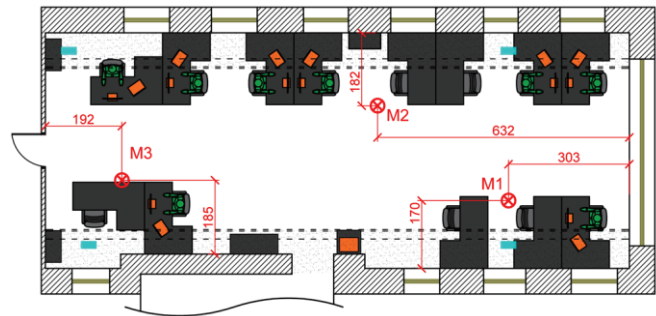
8

METHODOLOGY

- Measurement Locations



Measurement Day	Measurement Location	Measurement Height	Start-Stop Time for HRU
Day 1	M1	1.1 m	10:30 – 18:00
Day 2	M1	1.7 m	09:00 – 18:00
Day 3	M2	1.1 m	08:00 – 18:00
Day 4	M2	1.7 m	08:00 – 18:00
Day 5	M3	1.1 m	08:00 – 18:00
Day 6	M3	1.7 m	11:30 – 18:00



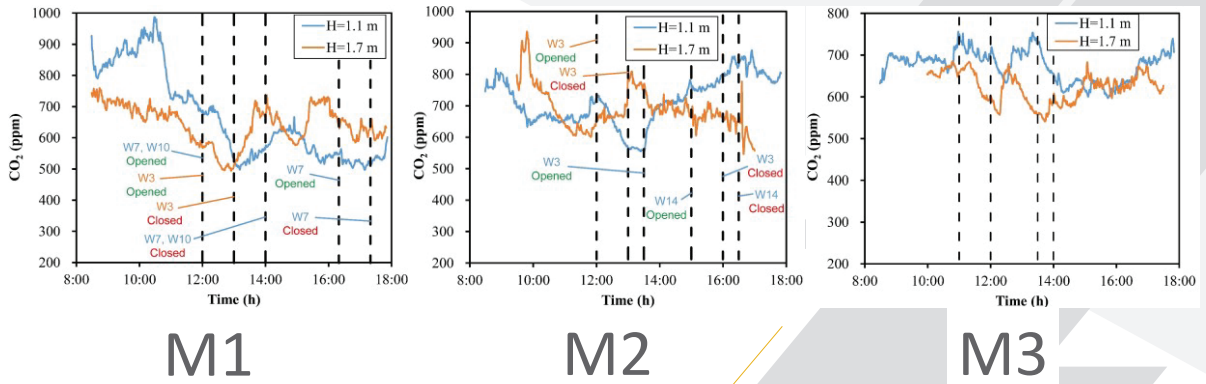
9



10

RESULTS

• CO2 measurements

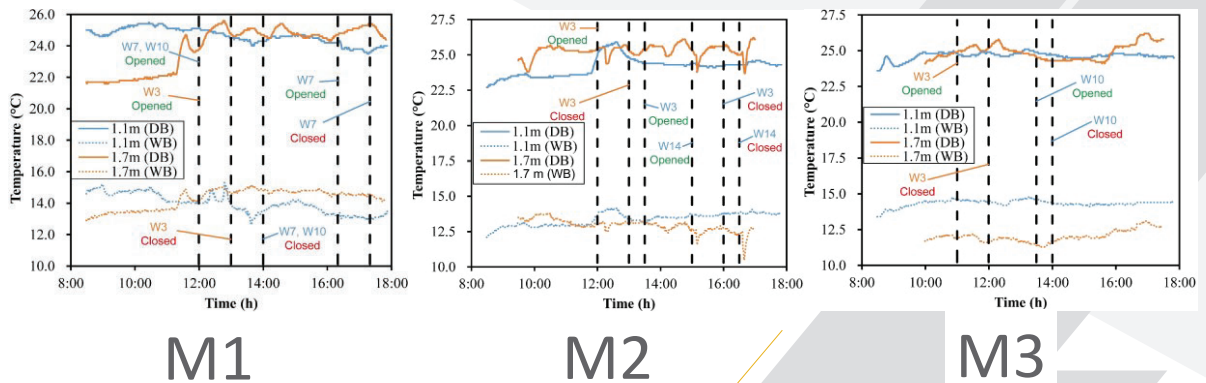


11

11

RESULTS

• Temperature measurements

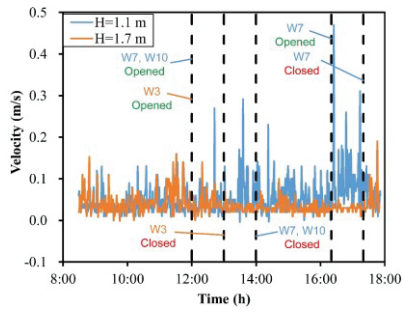


12

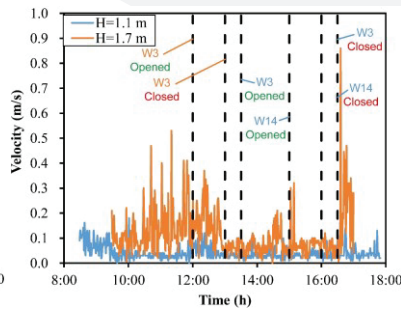
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RESULTS

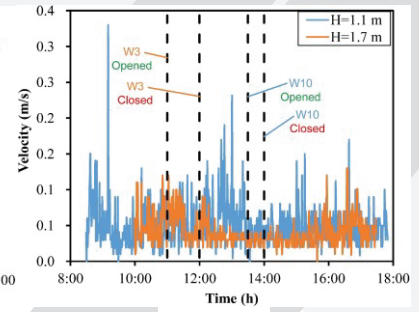
• Velocity measurements



M1



M2



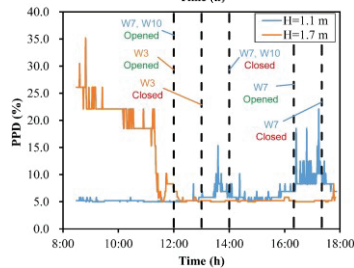
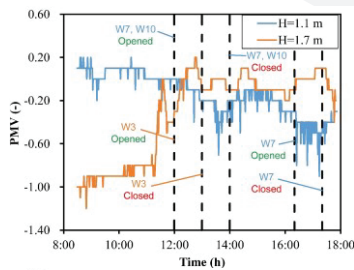
M3

13

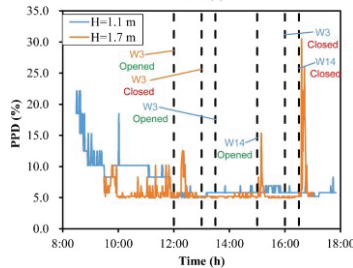
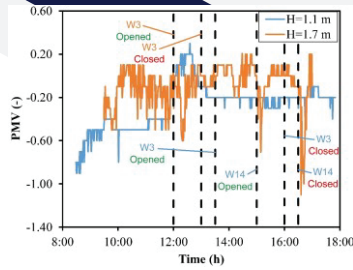
13

RESULTS

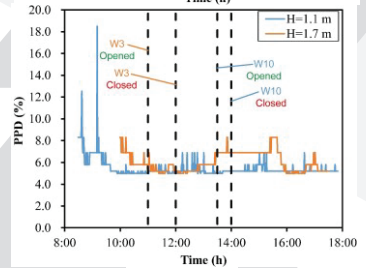
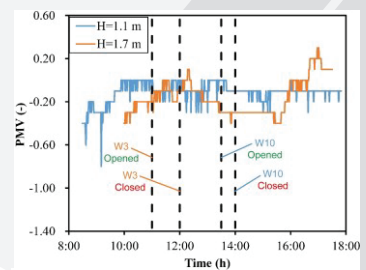
• PMV-PPD measurements



M1



M2



M3

14

14

Conclusions

- The alternative supply-exhaust layout can be applicable during epidemic periods.
- CO₂-based ventilation equipments should be considered for Turkish HVAC&R market from energy efficiency aspect.
- Humidifier option for the HRU unit should be considered.

15

Future Studies

- Evaluation of summer data
- Ventilative cooling
- Particle dispersion

16



Thank You!

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 **GAZI UNIVERSITY**
Faculty of Engineering
Department of Mechanical Engineering

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Hygienic Air Handling Unit Certification Program: the new necessity for a guaranteed indoor air quality

43rd AIVC Conference, October 4-5, 2023, Aalborg University, Copenhagen, Denmark
Ventilation, IEQ and health in sustainable buildings

1

Context



7
millions
Death

Per year according to WHO due the IAQ



420 %

Highest Increase in electricity prices in EU compared to 2021

2

Context



25,1
 $\mu\text{g}/\text{m}^3$

Concentration of PM
2.5 in Cremona (WHO
recommand a
maximum of 5 $\mu\text{g}/\text{m}^3$)



40 %

HVAC part in building
energy consumption

3

Context



20,000
Patient

Surgical site infection
(SSI) cases in EU in
2020



35 %

The part of mechanical
ventilation in HVAC
consumption

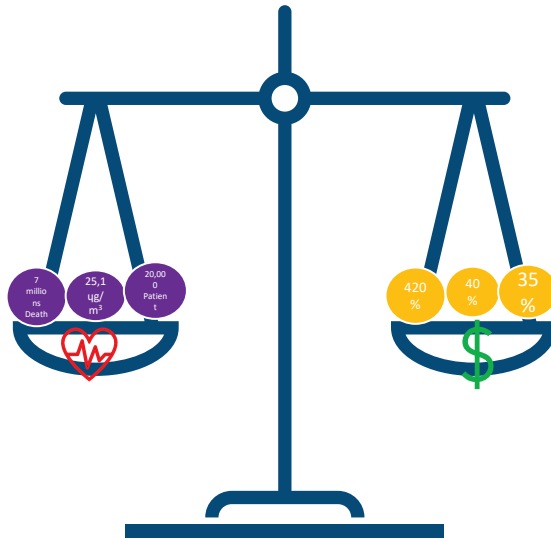
4

Context



10 %

Less filtration
between 2 grades of
filters



18,500 €



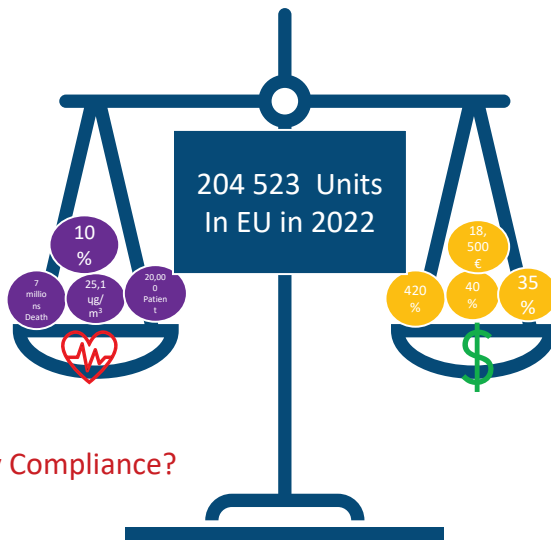
Excessive energy cost
per year for 6 %
deviation in efficiency

5

Context



How To verify Compliance?



6

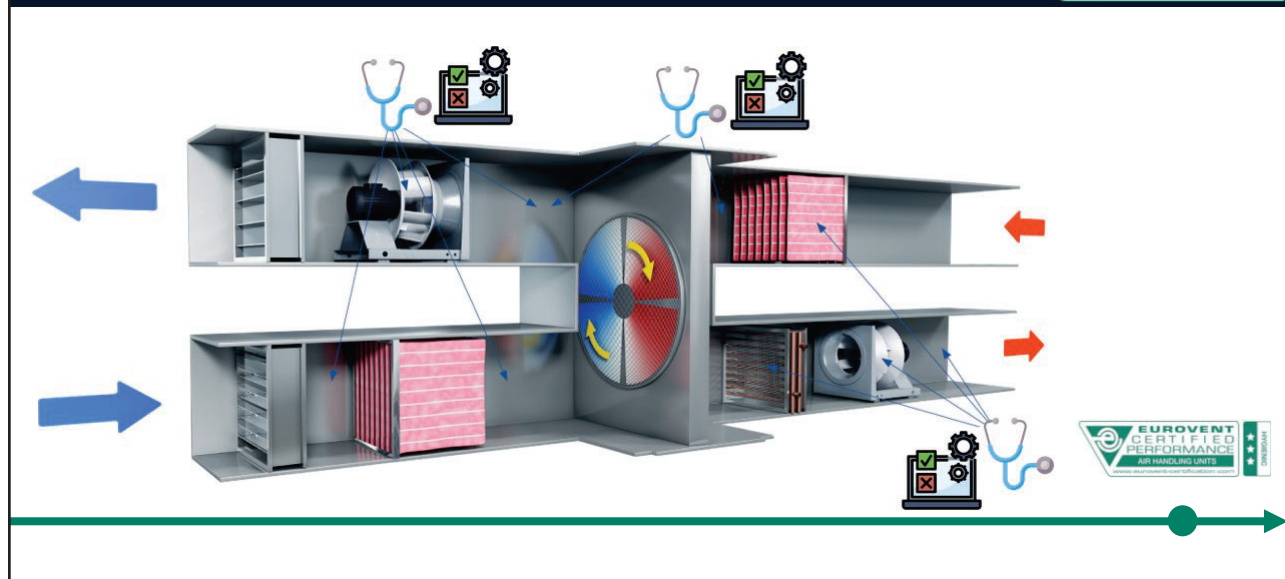
AHU certification program



General	Unit Housing	Air Treatment
<ul style="list-style-type: none"> ✓ Planning ✓ Manufacture ✓ Shipment 	<ul style="list-style-type: none"> ✓ Metallic Materials ✓ Non-Metallic Materials ✓ General AHU Arrangement ✓ Inner Casing Surface ✓ Inspection, Maintenance and Cleaning ✓ Filter Maintenance 	<ul style="list-style-type: none"> ✓ Filter ✓ Cooling and Heating Coil ✓ Humidifier ✓ Dehumidifier ✓ Heat Recovery System ✓ Fans ✓ Silencer

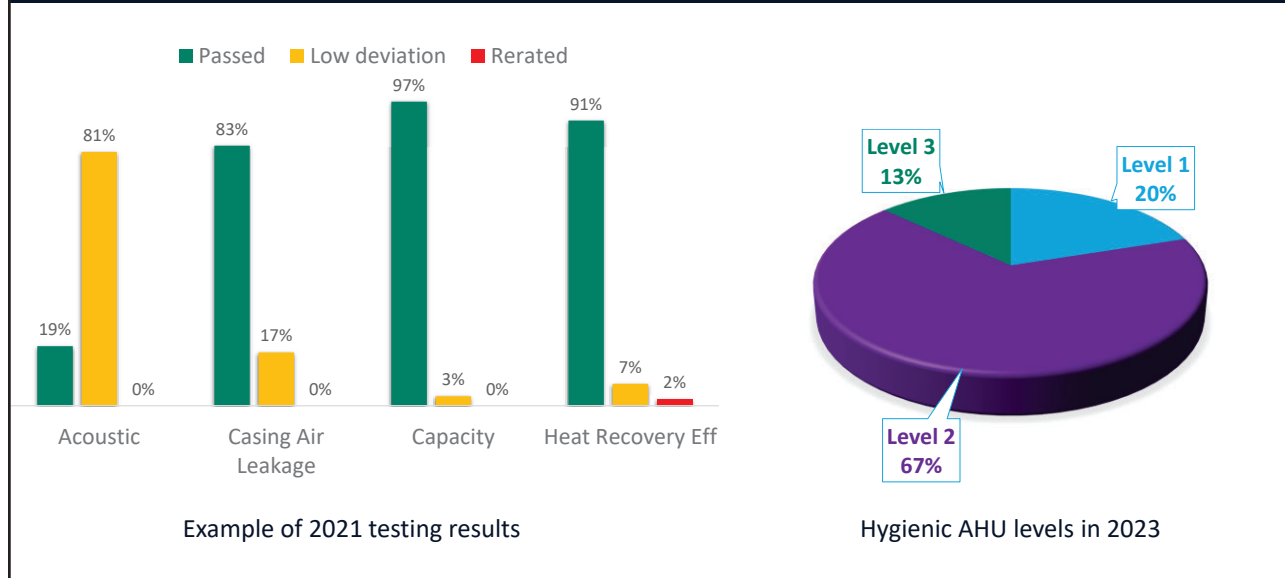
7

AHU certification program



8

2021 Campaign Results



9



THANK YOU

10



Car traffic or emissions from heating sources:

What is responsible for IAQ?

Katarzyna Ratajczak

Faculty of Environmental Engineering and Energy

Maciej Siedlecki

Faculty of Civil and Transport Technology

1



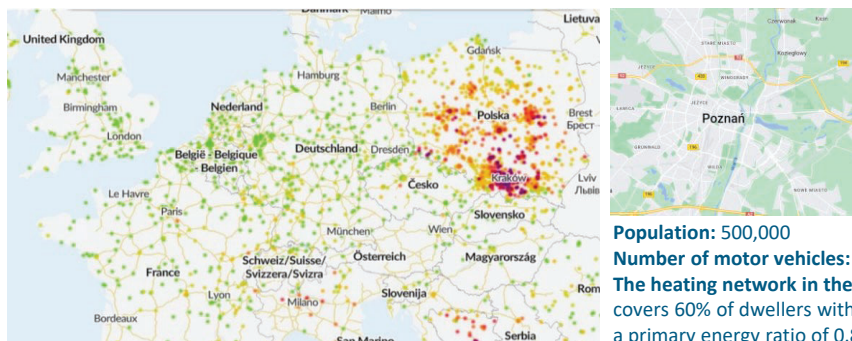
POZNAŃ UNIVERSITY OF TECHNOLOGY

Car traffic or emissions from heating sources:
What is responsible for IAQ?
Katarzyna Ratajczak, Maciej Siedlecki



The objective of short experiment

Assessing the impact of car traffic and heating sources on indoor and outdoor air quality – PM_{2.5} concentration.



Population: 500,000
Number of motor vehicles: 527,500
The heating network in the city: covers 60% of dwellers with a primary energy ratio of 0.8-0.9
Other heating sources: burning solid fuels and gas

Annual PM_{2.5}: 17.7 µg/m³

May-August PM_{2.5}: 10.1 µg/m³

September-April PM_{2.5}: 22.1 µg/m³

2



METHODOLOGY

In measurement points assessing:

- number of vehicles passing the building (TOPO by Vitronic, Poland)
- PM2.5 concentration in the outdoor air (NEMo Outdoor by Ethera, France)
- PM2.5 concentration in indoor air (NEMo XT by Ethera, France)
- IAQ in rooms located in front of the street
- Off-heating season: temperature 13.5-20.7°C

ASSESSMENT CRITERIA

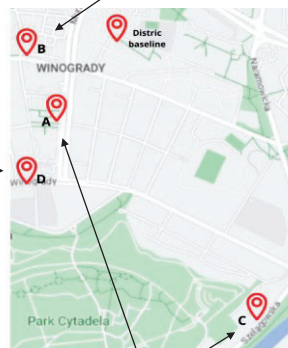
Low PM concentration in the off-heating season → car traffic does not generate high pollution

High PM concentration in the off-heating season → car traffic generates a lot of pollution

3



Main areteria of district

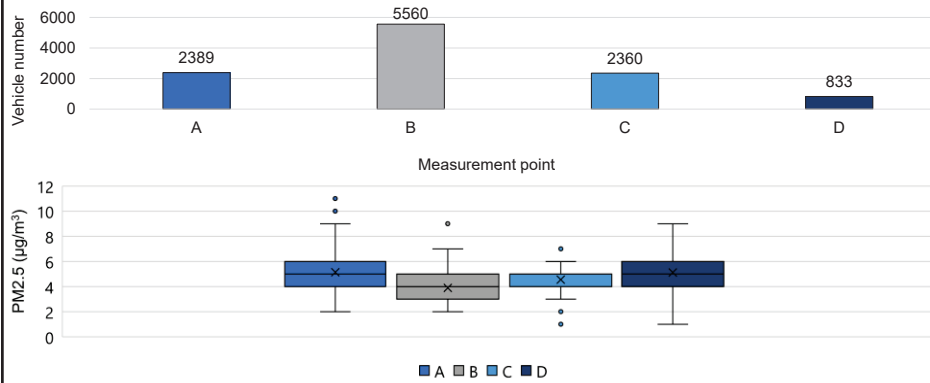


Not busy street

Busy street



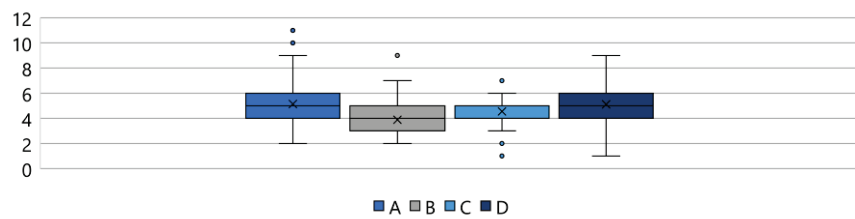
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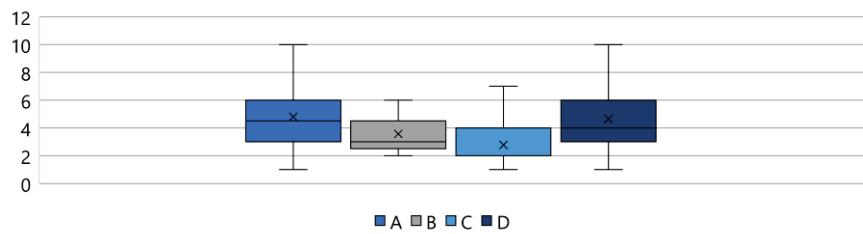
5



Concentration of PM2.5 ($\mu\text{g}/\text{m}^3$) in outdoor air



Concentration of PM2.5 ($\mu\text{g}/\text{m}^3$) in indoor air



6



CONCLUSIONS

- car traffic does not affect the quality of indoor air in rooms
- assuming that car traffic is similar in May and June to the traffic during winter it can be concluded that poor air quality in winter is caused by the fuels used to heat buildings.

LIMITATIONS

- the measurements lasted only one day
- other parameters like wind speed and direction may affect the results
- we took into account only PM2.5 which is not only parameter of AQ
- research concerns Poland, where the use of low-efficiency heat sources that burn fossil fuels is a huge problem
- we are aware that different socioeconomic conditions may occur in different countries

7



Car traffic or emissions from
heating sources:

What is responsible for IAQ?

Katarzyna Ratajczak
Faculty of Environmental Engineering and Energy
Maciej Siedlecki
Faculty of Civil and Transport Technology

8



POZNAN UNIVERSITY OF TECHNOLOGY

Monitoring VOCs' concentrations in a circular biobased residential building using low-cost sensors

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Seppe VERBIEST (seppe.verbiest@student.kuleuven.be)

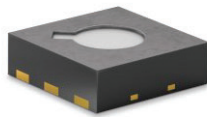
Faculty of Engineering Technology
Ghent Campus

Promotor: Arch. Versele Alexis
Promotor: Prof. dr. ir. Breesch Hilde
Copromotor: Dr. ir. Al-Assaad Douaa

1

Introduction and objective

- Biobased materials
 - Laboratory environments
- TVOC concentrations
- Low-cost sensors

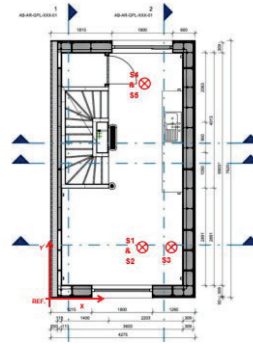


2

2

Methodology

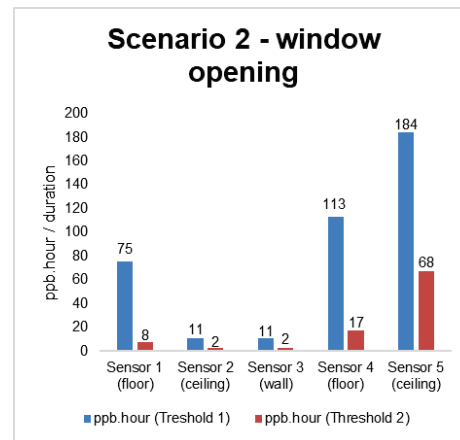
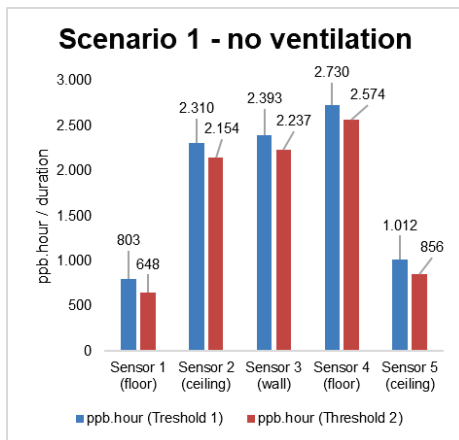
- CBCI Living Lab at Campus Ghent (Belgium)
- Biobased building envelope
- Ventilation mechanical extraction
- Sensor location
- Scenarios



<https://iiv.kuleuven.be/onderzoek/building-physics-and-sustainable-design/research/research-projects-map/cbci>

Results

- TVOC concentration [ppb.hours]



Threshold 1: Guidance value= 66,7 ppb - Threshold 2: Intervention value= 222,2 ppb

Conclusion

- Importance of ventilation
- Impact of opening windows
- Limitations – experience using low-cost sensors
- Future work
- Questions?

References

- **Slide 2:**
- VOC Emission Test Chamber | ESPEC CORP. (z.d.). <https://www.espec.co.jp/english/products/env-test/voc/>
- Smart Sensor Solutions. (z.d.). Sensirion.com. <https://sensirion.com/products/catalog/SGP30>
- Circular bio-based construction industry. (z.d.-b). Building Physics and Sustainable Design. <https://iiw.kuleuven.be/onderzoek/building-physics-and-sustainable-design/research/research-projects-map/cbci>
- **Slide 3:**
- Interreg Programmes Portal: Find programmes, calls and jobs. (2023, 31 mei). Interreg.eu. <https://interreg.eu/>

Smart & Predictive Air Quality Solution

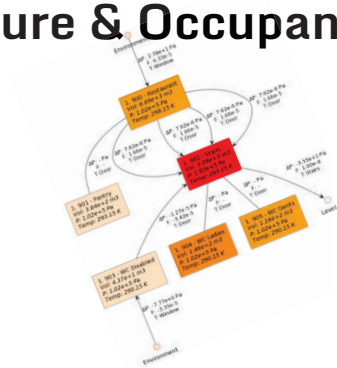
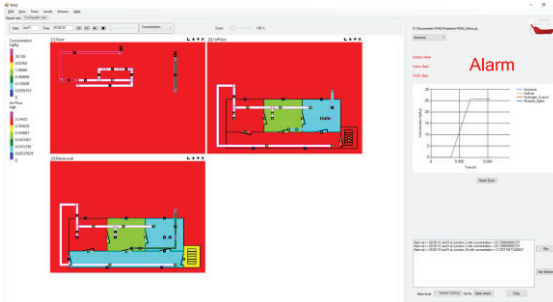


Prometech

Dr. Ir. Paul Brassler – *Scientific Software Developer*
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1

Protecting Critical Infrastructure & Occupants



Why:

Air quality monitoring to safeguard occupants from external and internal **hazards**

- Monitor and regulate indoor **air quality** to ensure safety of occupants
- Protect against external threats like pollutants and viruses
- Optimise HVAC systems layout & operation
- Improve **emergency** procedures & response and evacuation planning

28-Sep-23

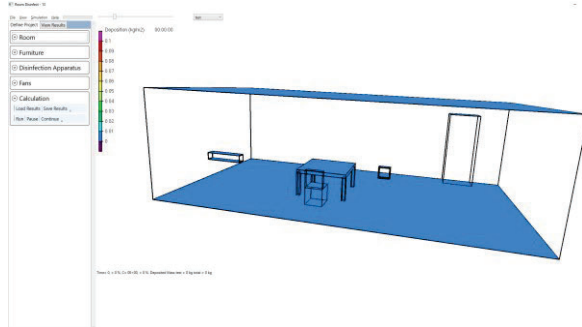
Dr. Ir. Paul Brassler, paulb@prometech.eu

2

Predict Airflow and Hazard Dispersion

How:

- **Predict** indoor airflow and hazard dispersion
- Considers HVAC and **filtering** systems
- Estimates **effects** on occupants
- Automate smart **evacuation** routes



28-Sep-23

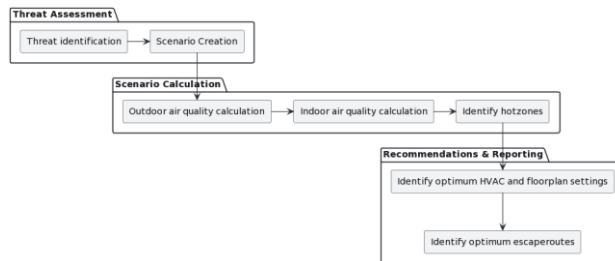
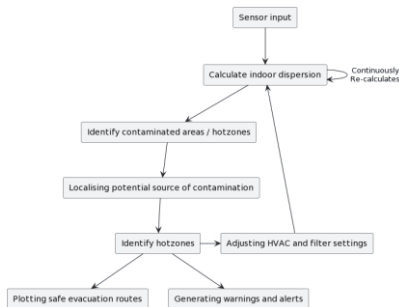
Dr. Ir. Paul Brassler, paulb@prometech.eu

3

HAVAC Suite - Your Air Quality Guardian

When:

- **Design** : Optimize floorplans
- **Real-time** monitoring



What:

- HAVAC software suite utilises specialised **AI**
- Integrate **IAQ sensors** for predictive modelling
- Identify & predict rooms with **low air quality**
- Smart **alerting** and warning system

28-Sep-23

Dr. Ir. Paul Brassler, paulb@prometech.eu

4

ENERGY IMPLICATIONS OF INCREASED VENTILATION IN COMMERCIAL BUILDINGS TO MITIGATE AIRBORNE PATHOGEN TRANSMISSION

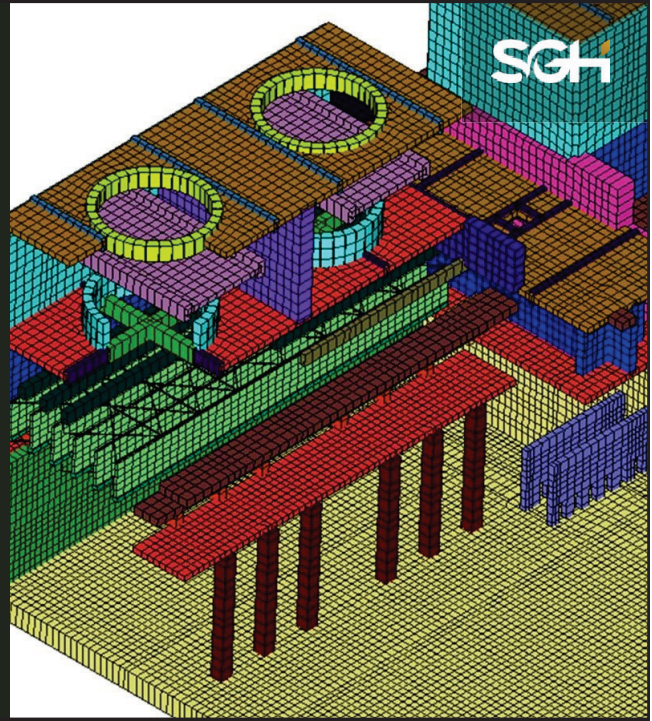
Sean M. O'Brien, P.E.

Senior Principal, Simpson, Gumpertz & Heger Inc., New York, NY USA

David Artigas, P.E.

Senior Project Manager, Simpson, Gumpertz & Heger Inc., New York, NY USA

4 October 2023



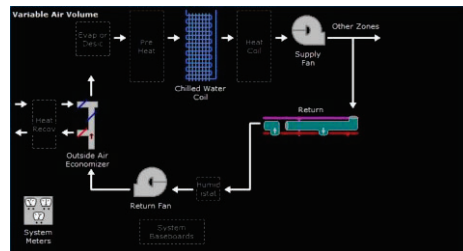
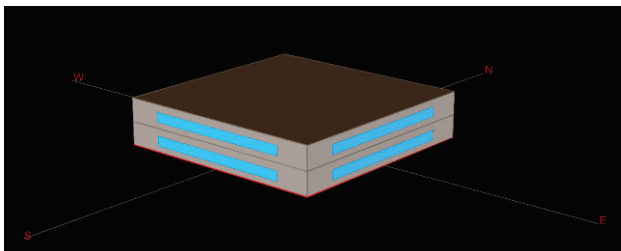
1

OBJECTIVE



Analyze the energy costs due to and feasibility of increased mechanical filtration and ventilation to mitigate airborne pathogen spread in commercial buildings

Locations	Cases (both locations)	Filtration (NYC 2-story Case)	Ventilation
New York, NY, USA	2-story Office Building	None	IECC minimum
Miami, FL, USA	22-story Office Building	MERV 9-12	20%, 50%, and 100% increase over IECC
		MERV 13-15	USA CDC Recommended 5 ACH



2

RESULTS

Filtration

Filtration Level	System Fan Power (kW)	Increase over Baseline	Building Source EUI	Increase over Baseline
None	3.1	-	174	-
MERV 9-12	5.3	71.0%	180	4.0%
MERV 13-15	5.8	87.1%	182	4.7%

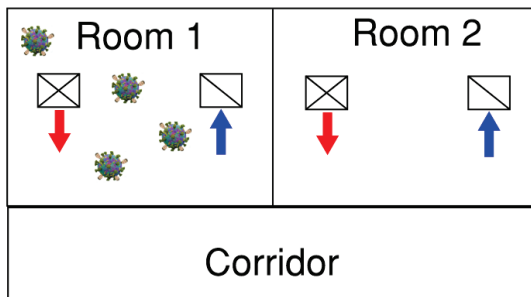
Ventilation

		Heating Energy	Increase	Cooling Energy	Increase	Source EUI	Increase
Ventilation L/s/Per	Low Rise, New York						
	2.38 (IECC)	48.5	-	32.2	-	181.7	-
	4.72 (100% increase)	58.4	20.36%	35.3	9.64%	190.2	4.69%
	39.3 (5 ACH)	507.6	947%	146.9	356%	634.4	249%
	High Rise, New York						
	2.38 (IECC)	305.7	-	379.5	-	178.2	-
	4.72 (100% increase)	426.7	39.60%	411.8	8.49%	186.8	4.78%
	Low Rise, Miami						
	2.38 (IECC)	0	-	84.1	-	235.7	-
	4.72 (100% increase)	0	-	96.3	14.53%	251.7	6.83%
	High Rise, Miami						
	2.38 (IECC)	0	-	853.4	-	225.9	-
4.72 (100% increase)	0	-	982.1	15.08%	241.0	6.70%	

3

NEED TO REBALANCE

Balanced

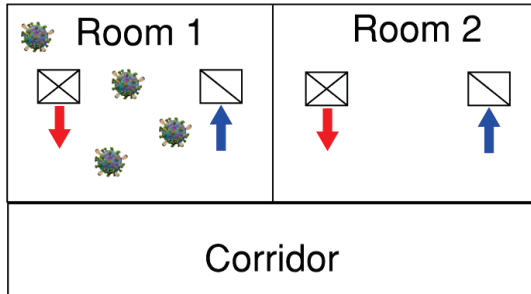


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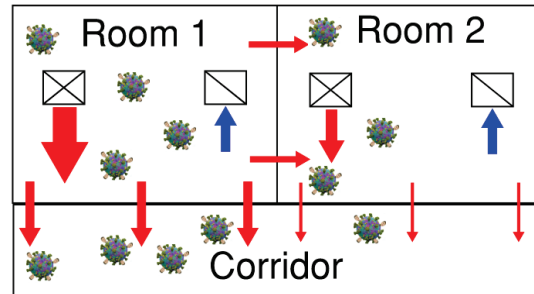
NEED TO REBALANCE

SGH

Balanced



Unbalanced



5

QUESTIONS?

SGH

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6

Reflections on alternative modelling approaches regarding occupants' window operation behaviour

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Department of Architecture, Design and Media Technology,
Aalborg University, Denmark

Ardeshir Mahdavi

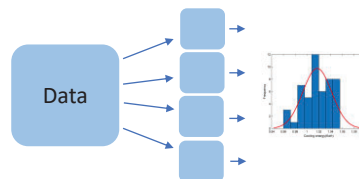
Institute of Building Physics, Services, and Construction
Faculty of Civil Engineering Sciences, TU Graz, Austria

AIVC2023 – 3-5 October 2023 – Copenhagen, Denmark

1

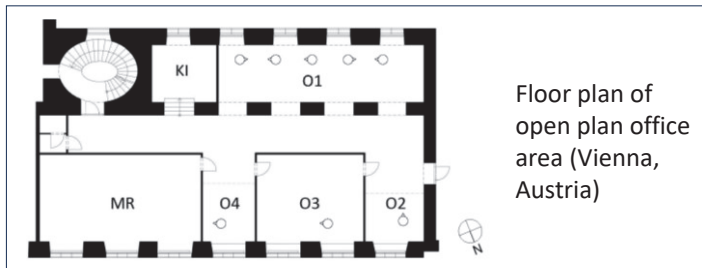
Motivation

- Models derived from empirical data are typically of a black-box type
- Formulating and testing explicit hypotheses regarding factors that could influence occupants' operation of windows



2

Illustrative case study



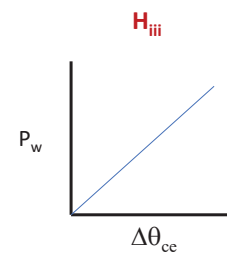
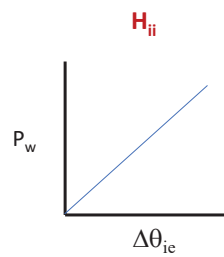
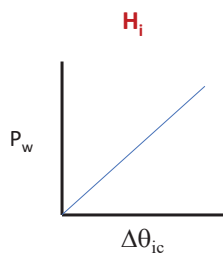
Collected data

- Indoor and outdoor temperatures
- Occupants' presence
- Window operation actions

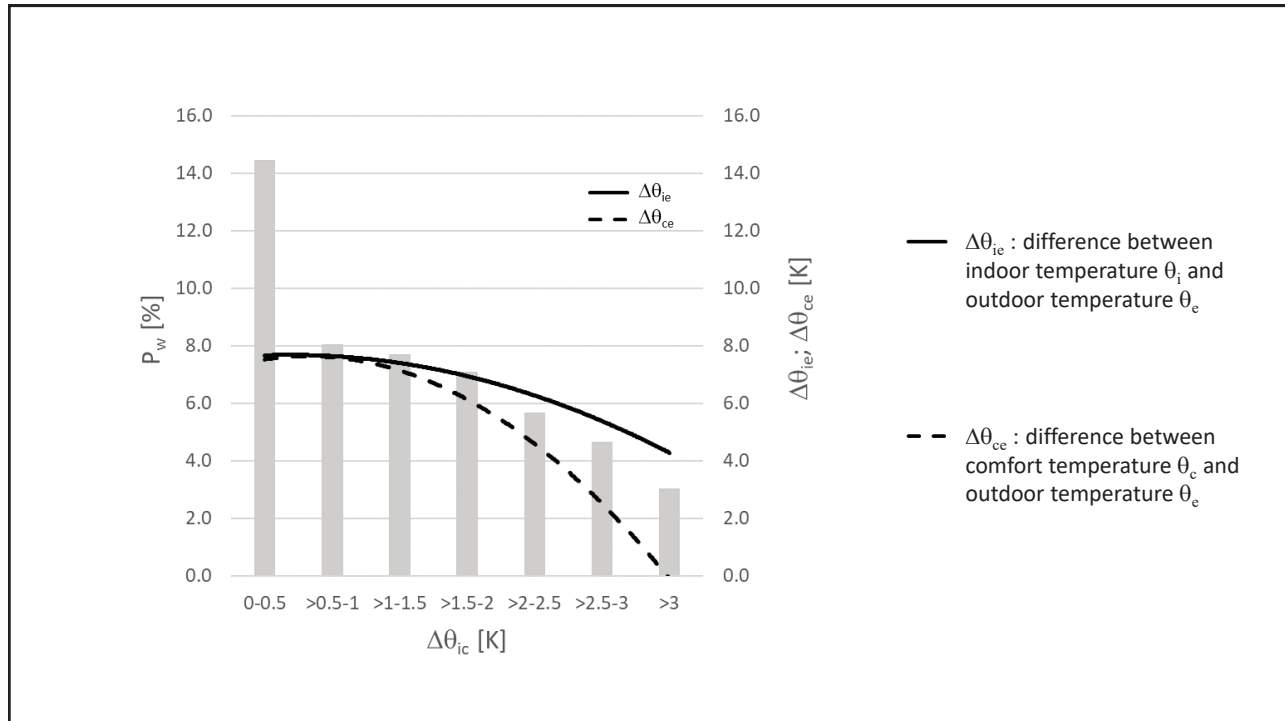
3

Methodology

- Focus on the probability of window operation actions upon occupants' arrival in the office
- Descriptive statistical analysis
- Formulation of 3 qualitative hypotheses:



4



5

Conclusion

- H_i : The magnitude of perceived thermal discomfort upon arrival cannot, on its own, explain the observed window operation tendency
- H_{ii} and H_{iii} are not rejected: Both higher cooling potential of the outdoor air (H_{ii}) $\Delta\theta_{ie}$ and the pre-arrival experience of thermal discomfort (H_{iii}) $\Delta\theta_{ce}$ could have encouraged window opening behaviour immediately after entering the office space

6

Reflections on alternative modelling approaches regarding occupants' window operation behaviour

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Ardeshir Mahdavi

Institute of Building Physics, Services, and Construction
Faculty of Civil Engineering Sciences, TU Graz, Austria

**Thank you
for your
attention!**

Please consult our paper for more detailed information!

Development of air supplied ceiling radiant air conditioning system using the Coanda effect

Satoshi Noguchi
The University of Kitakyushu

Yasuyuki Shiraishi
The University of Kitakyushu

Daishi Inoue
Nikken Sekkei

Hiroaki Tanaka
Nikken Sekkei



Research background and aim

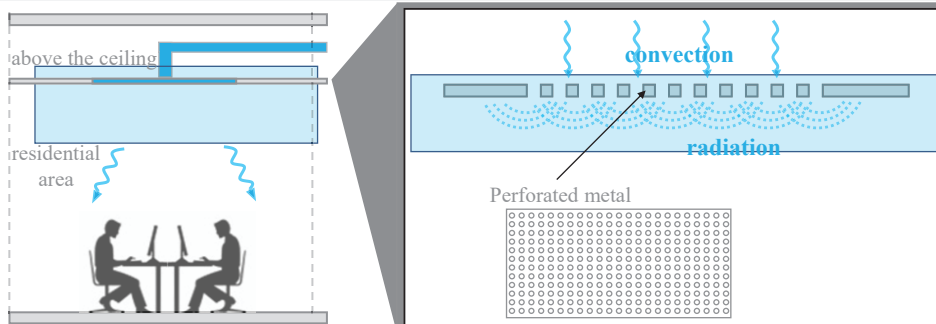
Recently...

ceiling radiant air conditioning systems are increasingly being adopted in office buildings.

In particular ...

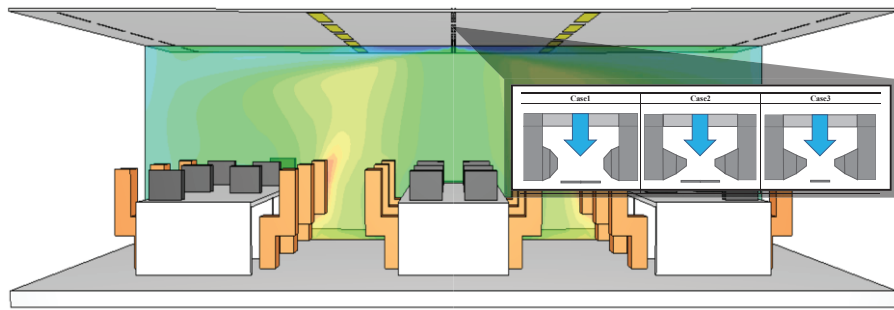
air-supplied ceiling radiant air conditioning systems are expected to become more widespread.

- ✓ no leakage from pipes
- ✓ no condensation on the surfaces of radiant panels.



Research background and aim

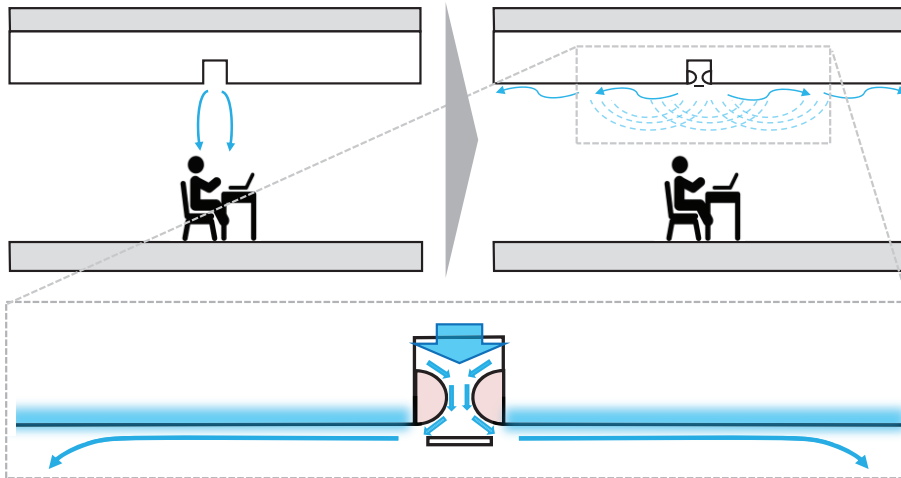
this research aims to further improve **the comfort of and ease of installing and retrofitting** air-supplied ceiling radiant air-conditioning systems by developing **a line-type Coanda air-conditioning system** for installation in office spaces.



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Line type Coanda air conditioning system

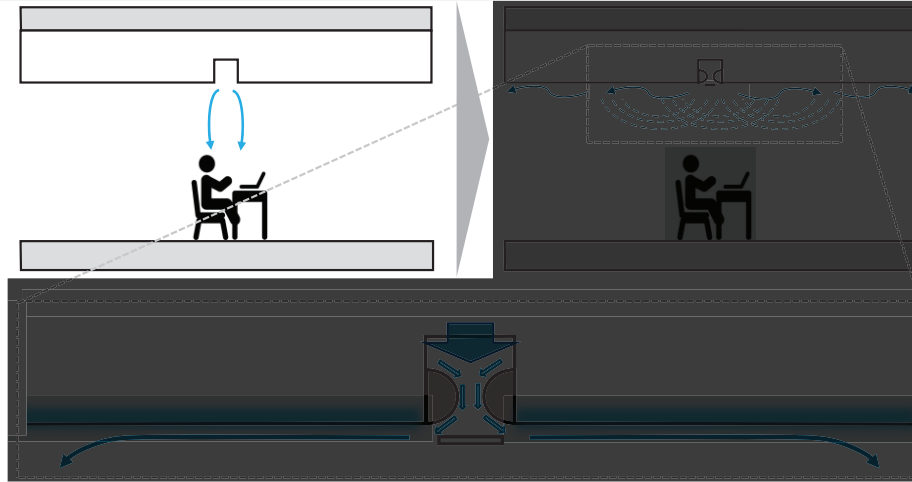


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Line type Coanda air conditioning system

A typical air-conditioning system in an office space tends to cause **discomfort due to drafts**.



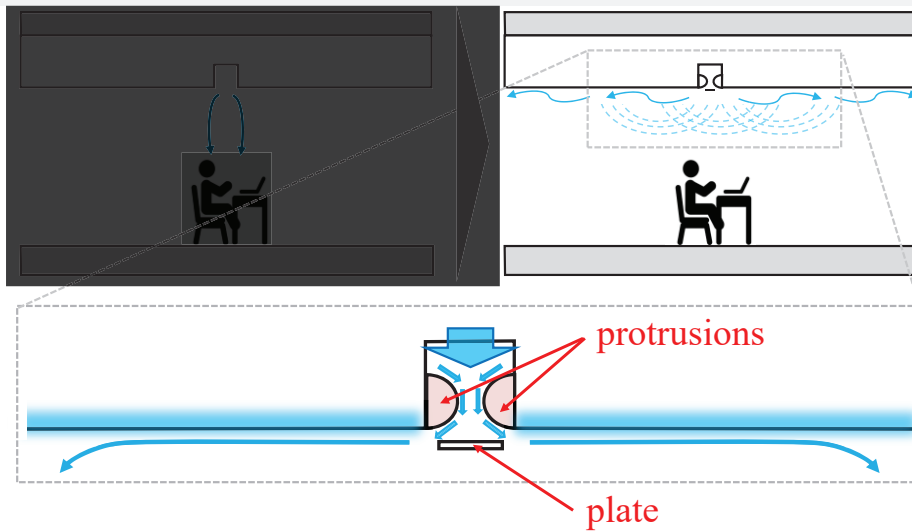
5

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Line type Coanda air conditioning system

in this system, the airflow direction is changed by **installing protrusions** and **plates** at the air outlets and cooling the ceiling surface by the Coanda effect.



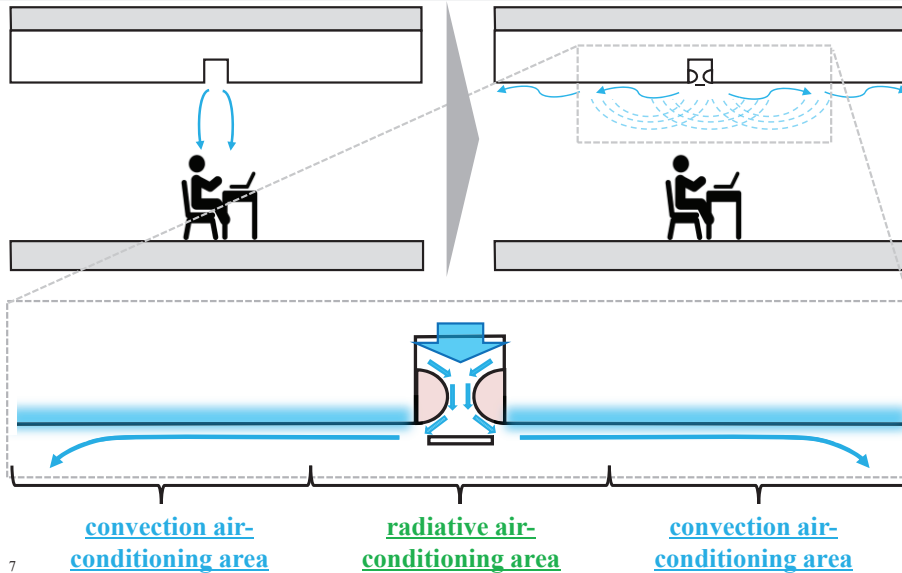
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Line type Coanda air conditioning system

The coldness creates a **radiative air-conditioning area** near the air outlet and a **convection** one away from it, the aim being to improve the comfort of the entire space.



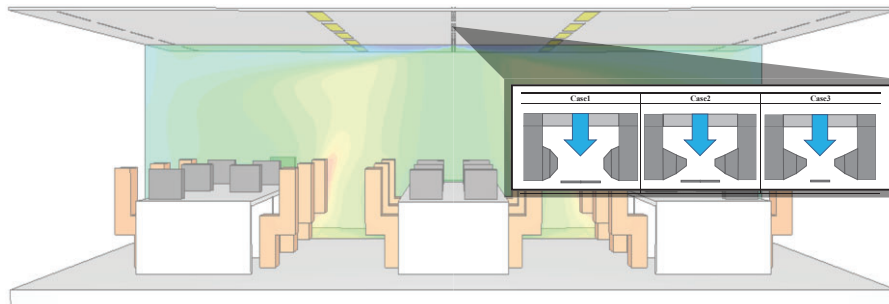
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Overview of our research

this research...

- ① Optimization of outlet shape

» **determine the case that gives the largest separation distance**



8

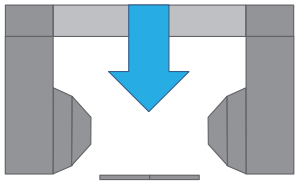
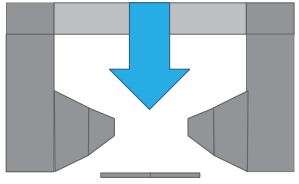
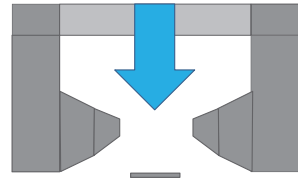
8

Overview of our research

this research...

- ① Optimization of outlet shape

» **determine the case that gives the largest separation distance**

Case1	Case2	Case3
		
Basic Case	A model with extended central projection	A model with half the plate width of Case 2

9

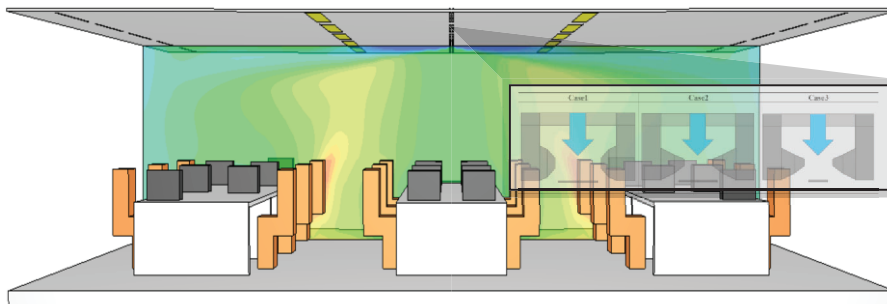
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Overview of our research

this research...

- ② Validation using model office

» **Demonstrating the effectiveness of a line-type Coanda air-conditioning system**



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1. Research background and aim
2. line type Coanda air conditioning system
3. Optimization of outlet shape
4. Validation using model office
5. Conclusion

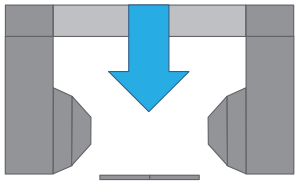
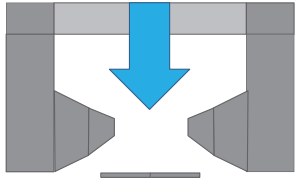
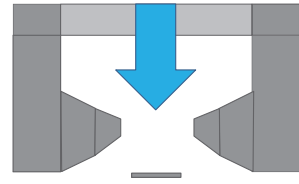
12

3. Optimization of outlet shape

the protrusion shape is optimised in a computational fluid dynamics (CFD) case study...

- ✓ determining the separation from the calculated **streamlines**
- ✓ the distance until the blown airflow separates from the ceiling is calculated from how the distribution of **wind velocity vectors varies along the z axis**

the shape that **gives the largest separation distance is considered the optimal shape.**

Case1	Case2	Case3
		
Basic Case	A model with extended central projection	A model with half the plate width of Case 2

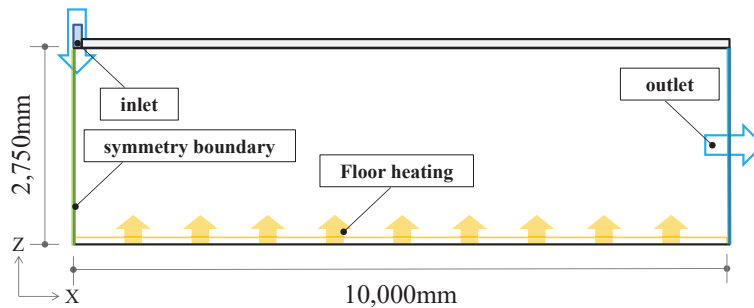
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3. Optimization of outlet shape

■ the analytical model



for improved calculation **accuracy** and **reduced calculation load**...

The analysis is performed on **a two-dimensional model**

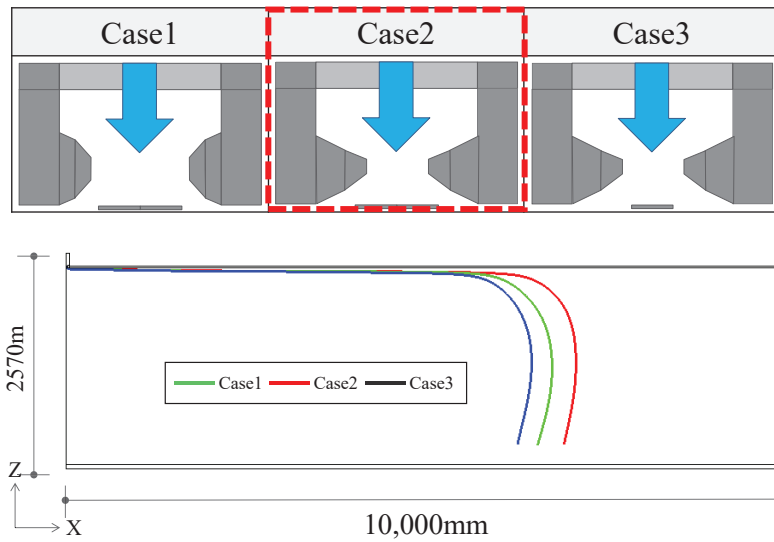
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3. Optimization of outlet shape

■ Results (the streamline analysis)



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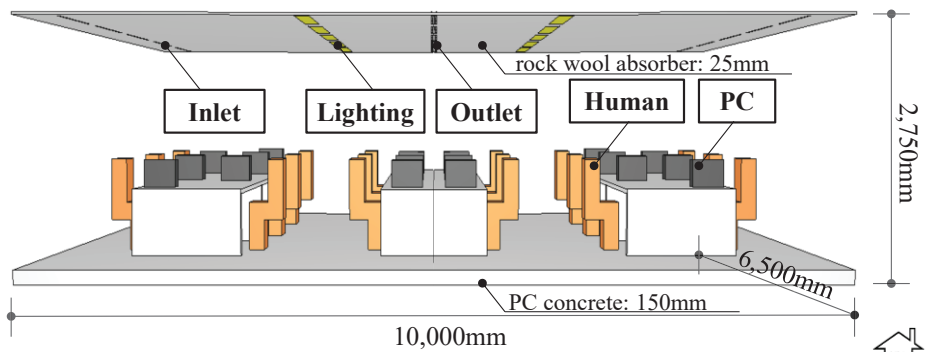
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4. Validation using model office

the temperature, airflow, and radiation fields are analysed by means of steady-state CFD to assess whether the proposed system forms a comfortable thermal environment.

■ analytical model



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4. Validation using model office

■ analysis cases

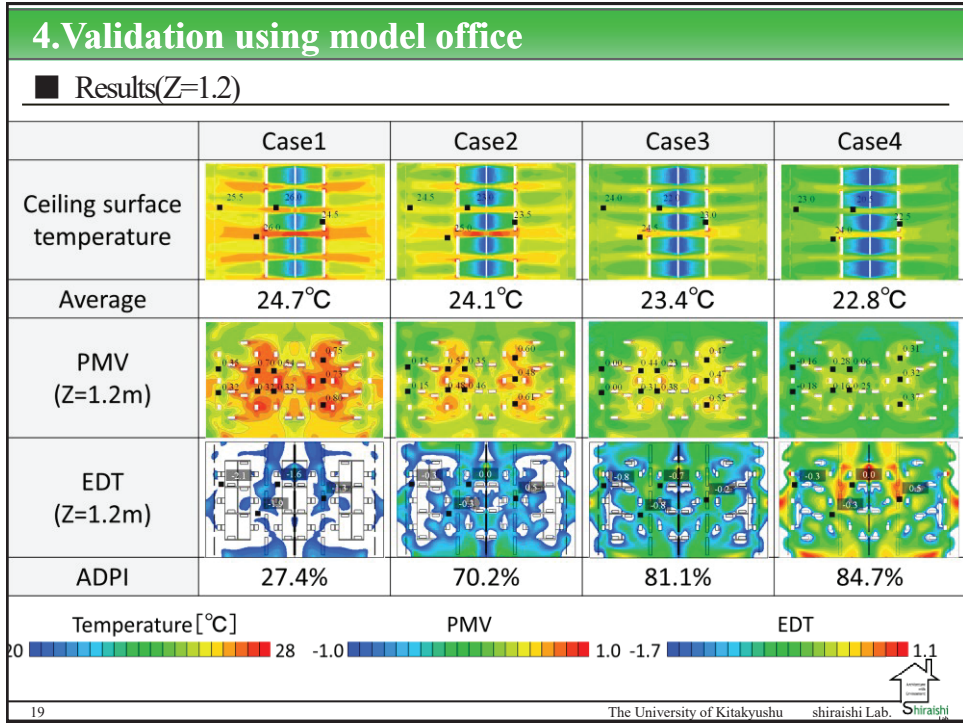
	flow rate[m ² /h]	blowoff temperatures[°C]
Case2-1	190	19.11
Case2-2		18.11
Case2-3		17.11
Case2-4		16.11

To derive the optimal operating conditions for the proposed system, the cases were set up with a fixed air flow rate and different air temperatures at each outlet.

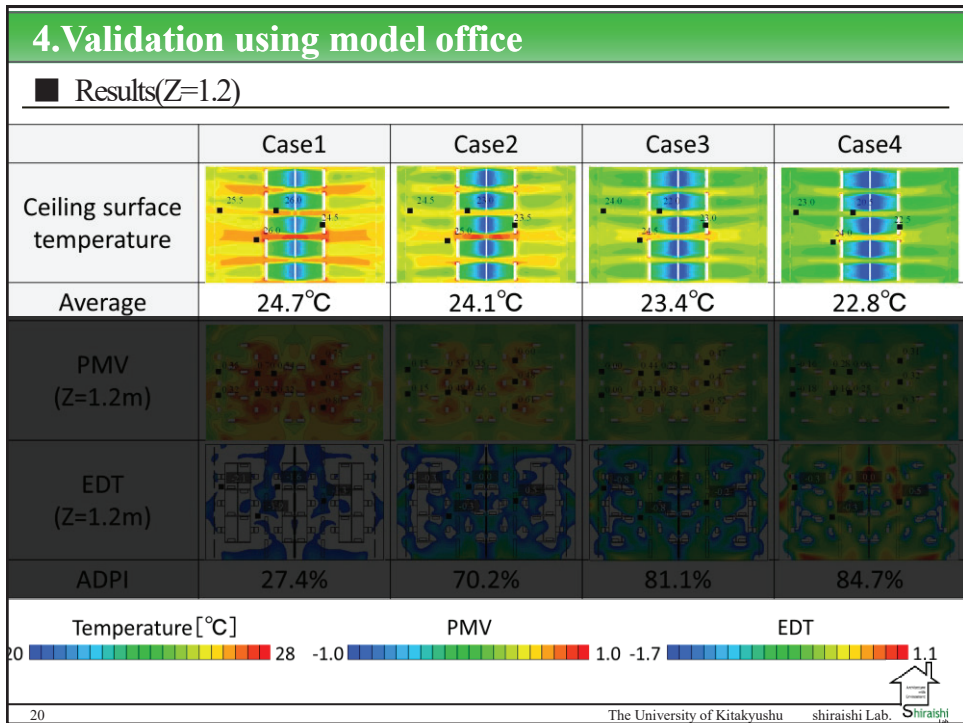
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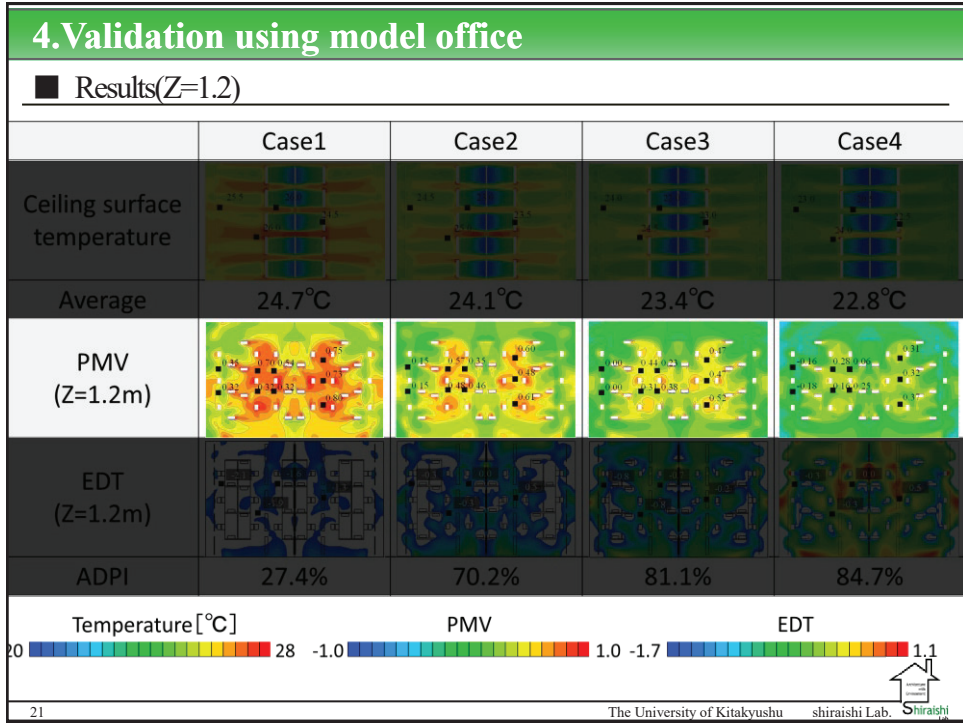
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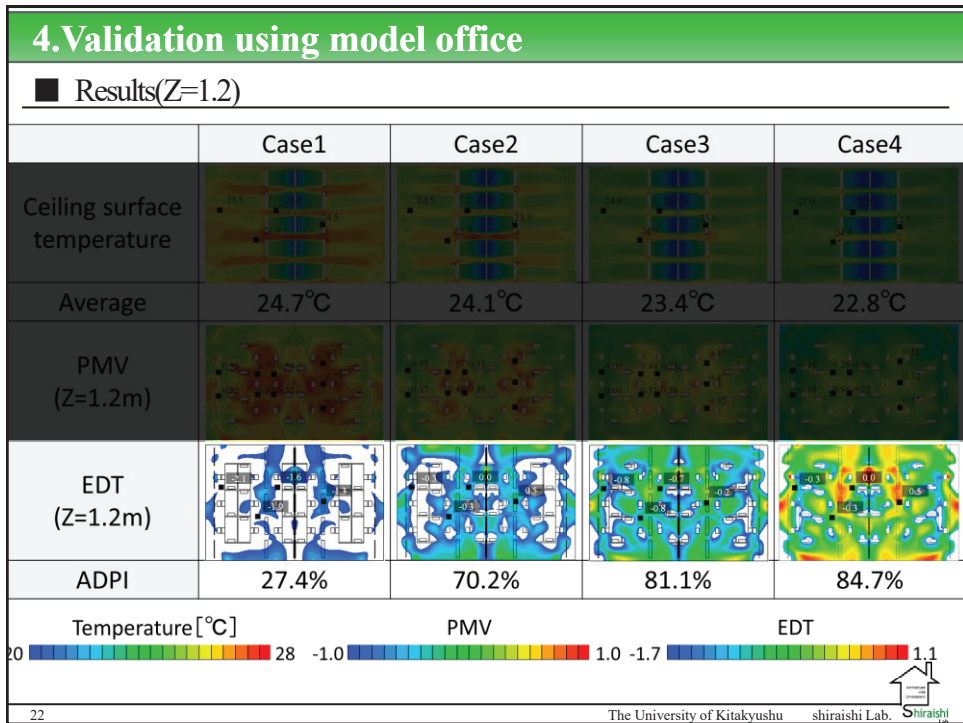
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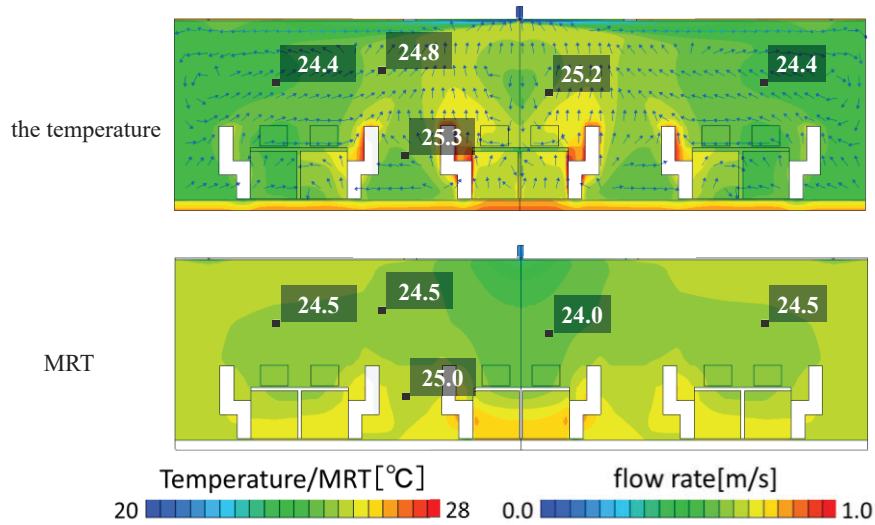
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4. Validation using model office

■ Results (temperature and MRT distributions Y=4.8m)



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5. Conclusion

The followings are the findings of this study...

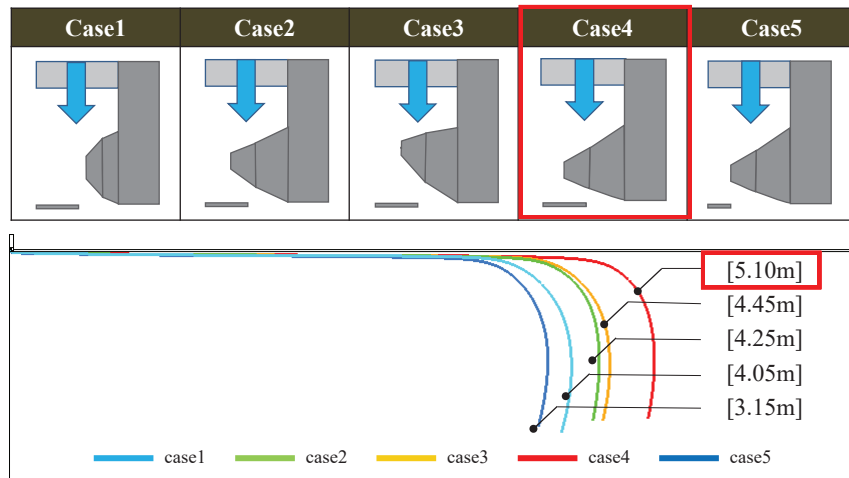
- ✓ We proposed an outlet shape that efficiently cools the ceiling surface.
- ✓ It was confirmed that installing a line-type Coanda air-conditioning system in the model office formed a radiative and a convection air-conditioning area and a comfortable thermal environment with little draft and uneven temperature.

From now on...

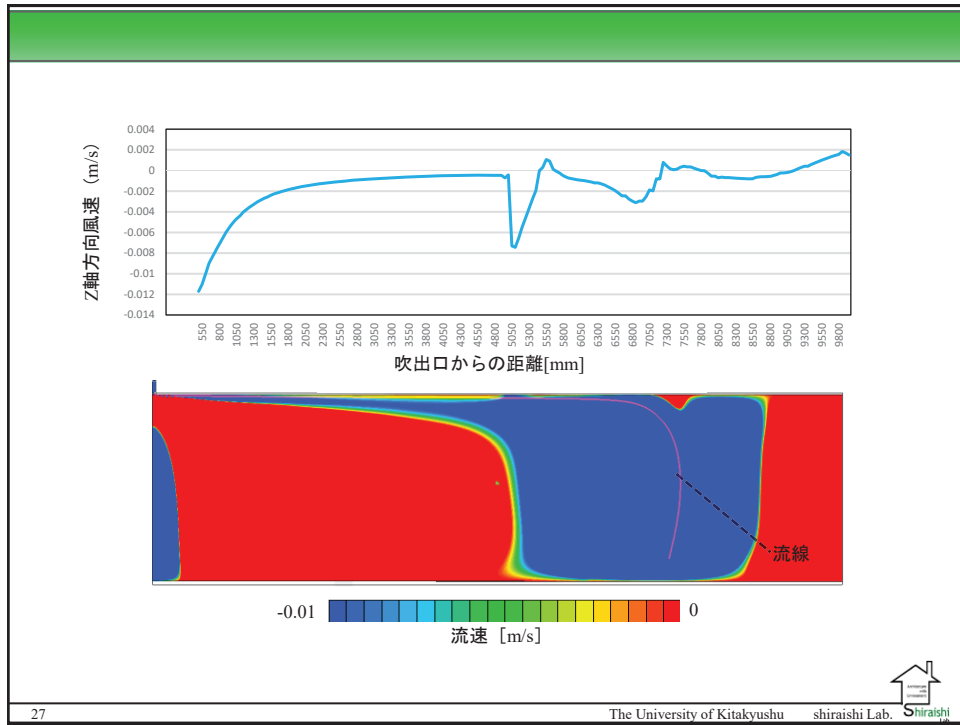
- ✓ Comparison with general air-conditioning systems
- ✓ the proposal of design conditions using Archimedes number
- ✓ confirmation of ventilation performance using Age of air and other indices

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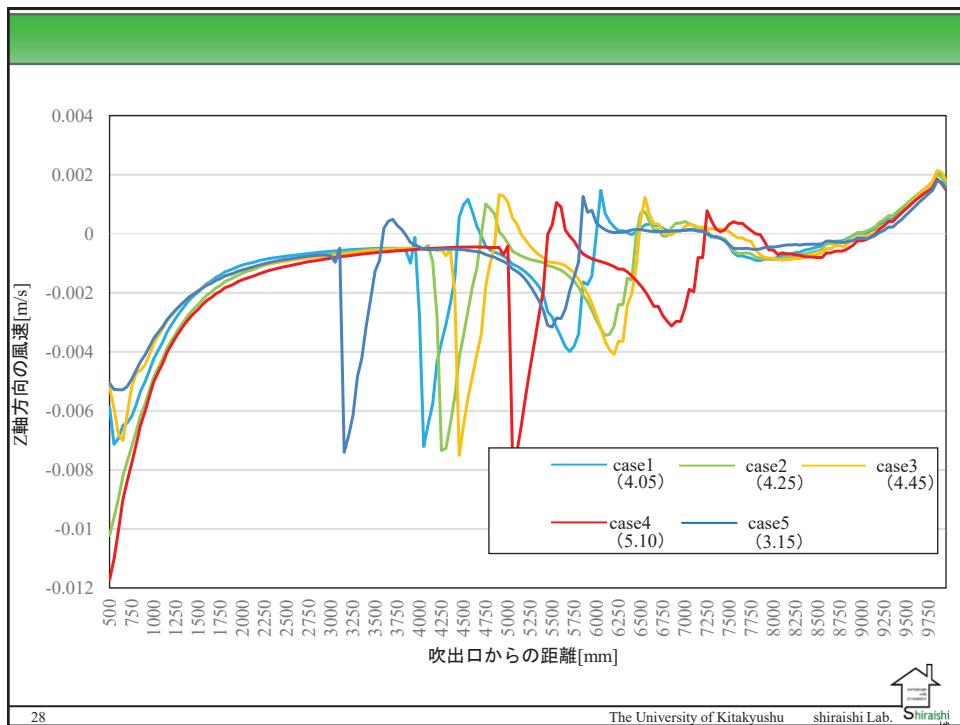
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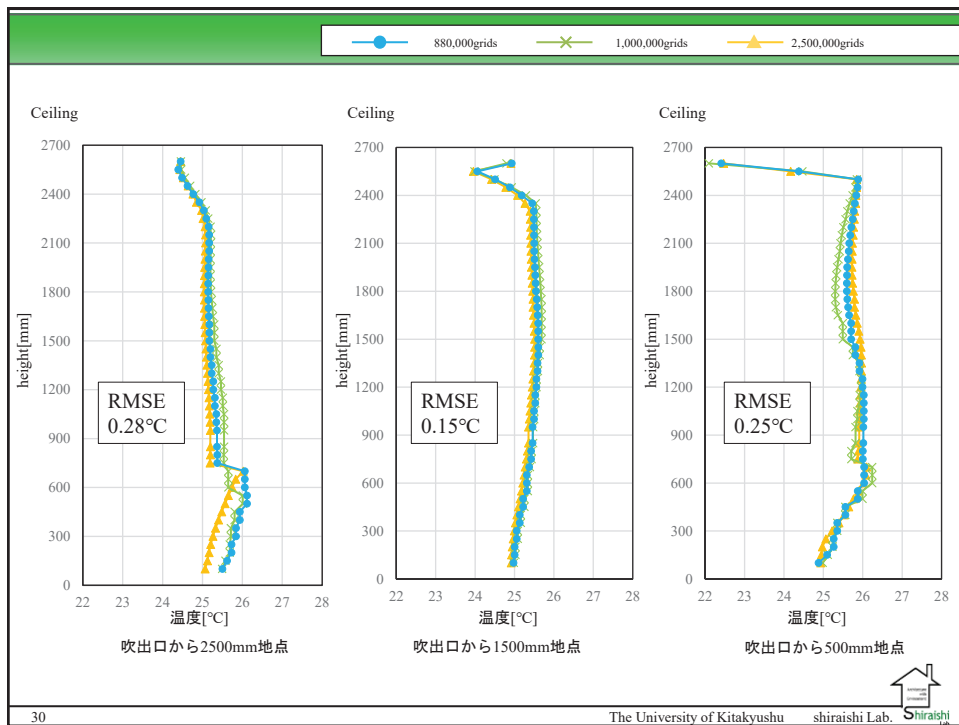
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4. Validation using model office


■ Results(Z=1.2)

	Case1	Case2	Case3	Case4
Ceiling surface temperature				
Average	24.7°C	24.1°C	23.4°C	22.8°C
PMV (Z=1.2m)				
EDT (Z=1.2m)				
ADPI	27.4%	70.2%	81.1%	84.7%
<p>Temperature [°C] PMV EDT </p>				
<p>29 The University of Kitakyushu shiraishi Lab. </p>				

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■ EDT(Effective Draft Temperature)	
Comfort index taking into account temperature differences and draughts in the room.	
EDT=(TEMP-t_c)-7.66*(VECTV-0.15) [°C]	
TEMP:室内の局所温度[°C], t_c:居住域平均温度[°C], V_x:局所風速[m/s]	
■ ADPI (Air Distribution Performance Index)	
Percentage of EDTs in the comfort zone in the room.	
ADPI=n/n' × 100 [%]	
-1.7 < EDT < 1.1 かつ 気流速度 < 0.35m/s …(1) を満たせば座っている在室者の大多数は満足する n:(1)を満たすデータ数 n':総データ数	
31	The University of Kitakyushu shiraishi Lab. 

Wind Tunnel Experiment of Wind-Induced Single-sided Ventilation under Generic Sheltered Urban Area

43rd AIVC 11th TightVent & 9th Venticool conference



Zitao JIANG (Osaka University, Japan)
Tomohiro KOBASHI (Osaka University, Japan)
Toshio YAMANAKA (Osaka University, Japan)
Mats Sandberg (Gavle University, Sweden)

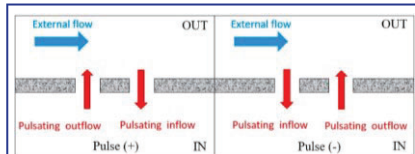
1

1. Introduction

Natural ventilation is an effective strategy to improve indoor air quality without using energy.



Practical problems
The urban morphology influences the ventilation performance of natural ventilation.



$$Q = (\alpha A)_{eff} V \sqrt{\Delta C_p}$$

It is not accurate if ΔC_p is small.

$(\alpha A)_{eff}$: effective opening area [m²]

V : Reference velocity [m/s]

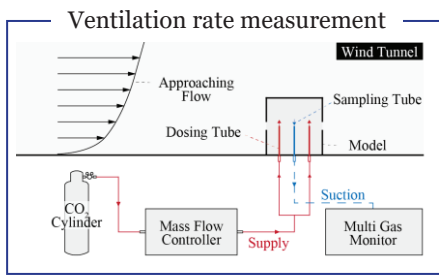
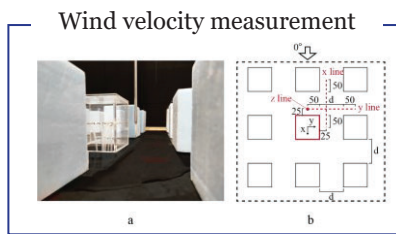
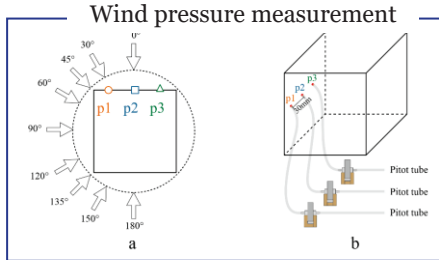
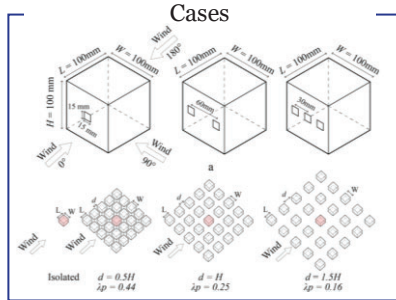
ΔC_p : Mean wind pressure coefficient difference[-]

Theoretical problem
The **Orifice equation**, that uses the mean wind pressure coefficient difference, could not predict ventilation rate well when ΔC_p is small.

2

2.Methods

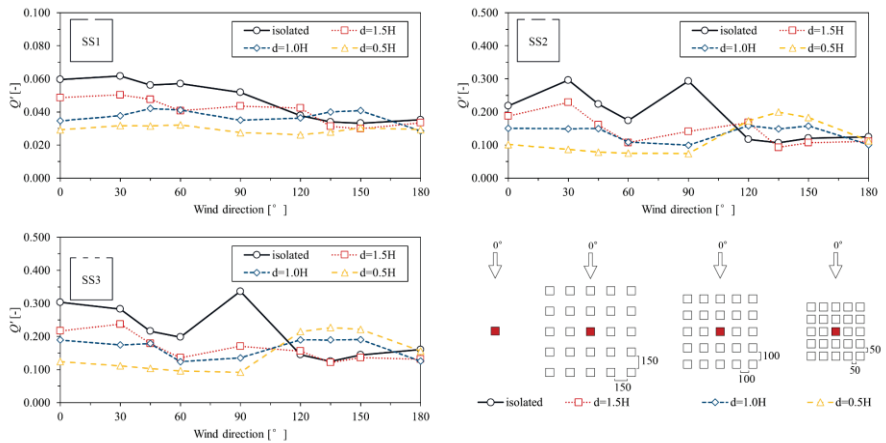
Investigated parameter: wind direction, building density, the number of openings.



3

3.Results & Conclusions

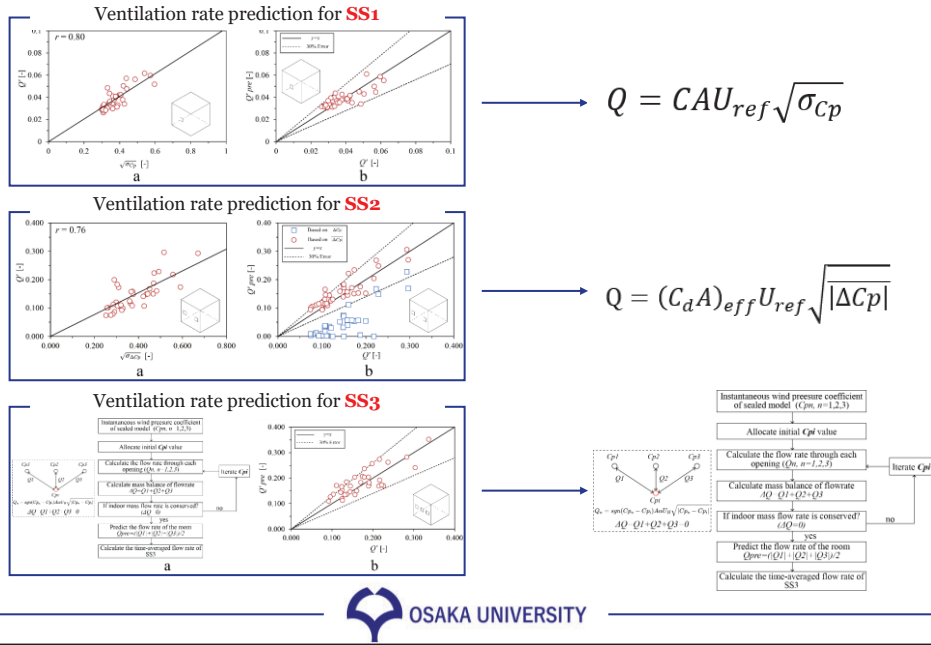
The influence of sheltering effect



Windward openings: Higher building density **reduces** ventilation rate
Leeward opening: Higher building density **increases** ventilation rate

4

3. Results & Conclusions



5

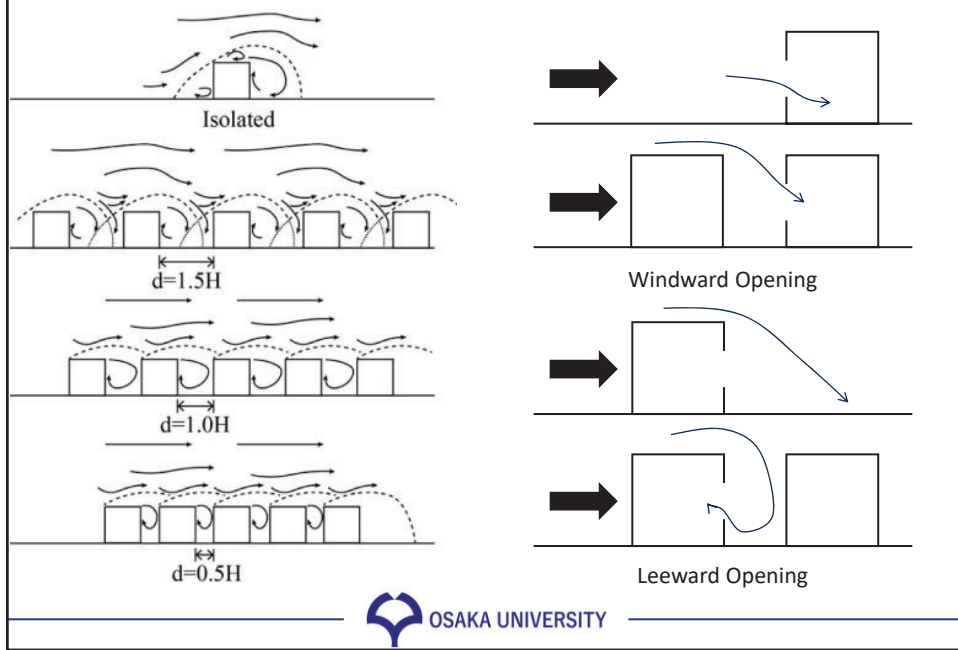


Thank you!

Wind tunnel experiment of wind-induced single-sided ventilation under generic sheltered urban area. Building and Environment, 242, 110615.
 Jiang, Z., Kobayashi, T., Yamanaka, T., Sandberg, M., Choi, N., Kobayashi, N., ... & Toyosawa, K. (2023).

6

Why high building density can sometimes increase ventilation rate?



A study on desiccant system regenerated by waste heat from home-use solid oxide fuel cell cogeneration system

Keita Mizuno
Isamu Ohta

Misawa Homes Institute of Research and Development Co., Ltd.

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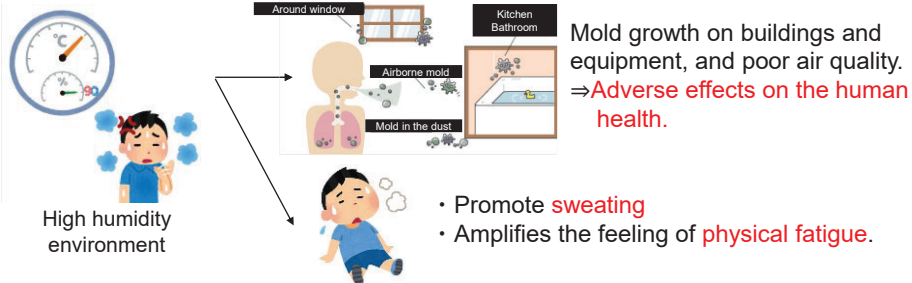
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0

Research background

① The need for indoor humidity control

- High humidity environments during summer : Humidity environment above 60%



- In a low-humidity environment such as winter : Humidity environment below 40%



It is necessary to control indoor humidity appropriately with a desiccant system throughout the year in order to realize an indoor environment that takes human health into consideration.

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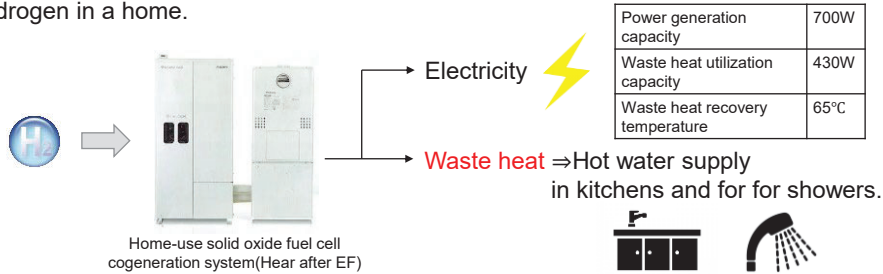
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Research background

② The need to achieve carbon neutral by 2050 (COP21, COP27)

- **Hydrogen utilization** is attracting attention as a way to achieve carbon neutral.
- Building facilities are required to achieve **significant energy savings**.
- A **home-use solid oxide fuel cell cogeneration system** is an equipment that uses hydrogen in a home.



- In summer and in hot and humid regions, the demand for hot water supply is low and unused waste heat is generated.
- If exhaust heat is not utilized and accumulates, EF will shut down.

By using waste heat for dehumidification and humidification, unused waste heat is reduced. In addition to improving overall efficiency of the EF, it also **increases the operating hours and high-load operation (efficient operation) of EF.**

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2

2

Research purpose

By identifying methods to maximize the use of limited waste heat, the goal is to achieve an indoor environment that contributes to energy conservation while also being considerate of human health.

Subject①

The EF waste heat usage by sending **water (about 65°C)**.
⇒ **Clarified the flow rate** that maximizes the amount of the waste heat usage.

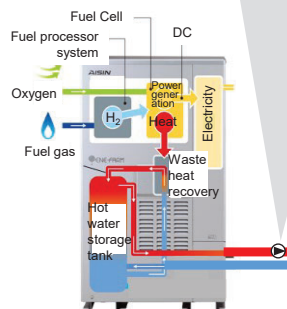


Figure: Home-use solid oxide fuel cell cogeneration system (EF)

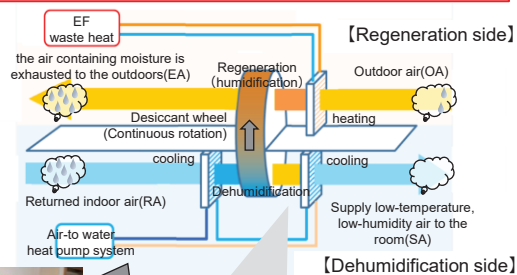


Figure: Desiccant unit

Subject②

The balance of air volume (**OA and RA**) that maximizes the amount of dehumidification was clarified.

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3

3

Research results

Subject①

Clarified the flow rate that maximizes the amount of the waste heat usage.

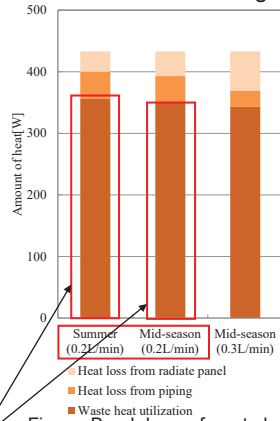
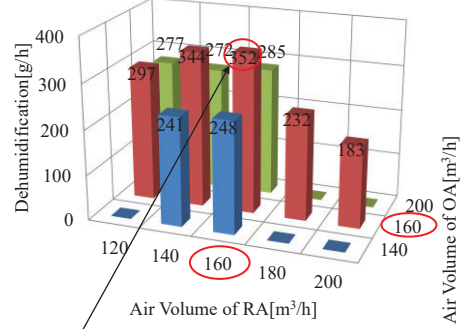


Figure: Breakdown of waste heat

With a flow rate of 0.2L/min, about 80% of the heat emitted from the EF was utilized in the desiccant unit in both the mid-season and summer seasons.

Subject②

The balance of air volume(OA and RA) that maximizes the amount of dehumidification was clarified.

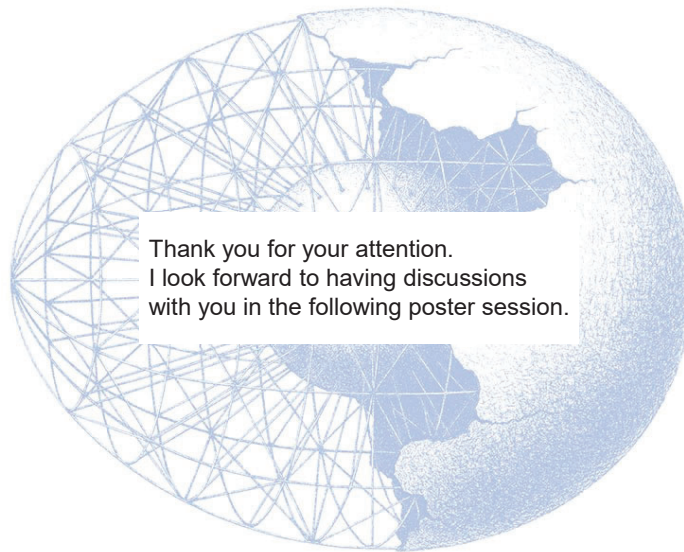


The peak dehumidification amount was at 160 m³/h for both air volumes, with a maximum dehumidification rate of 352 g/h.

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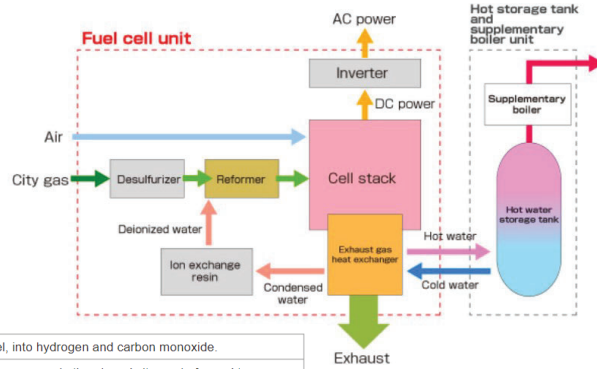


Thank you for your attention.
I look forward to having discussions
with you in the following poster session.

5

Supplementary materials

SOFC consists of two parts, a 'fuel cell unit' and a 'hot storage tank and supplementary boiler unit'. The fuel cell unit generates electricity and simultaneously recovers the exhaust heat as hot water. The hot storage tank and supplementary boiler unit stores the recovered hot water to supply hot water.



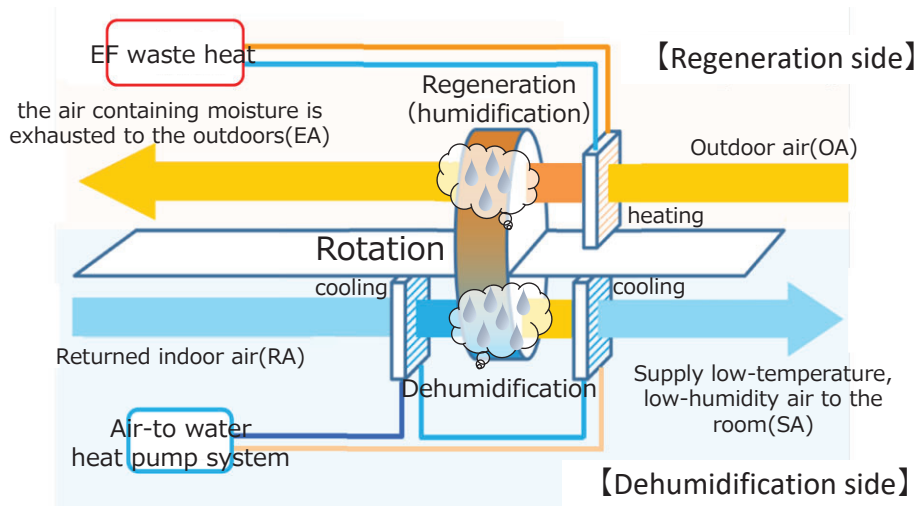
1. Reformer	Converts city gas, fuel, into hydrogen and carbon monoxide.
2. Cell stack	Generates power using oxygen in the air and city gas(reformed to hydrogen and carbon monoxide.)
3. Inverter	Converts power generated in the cell stack into alternating current and supplies it to homes.
4. Exhaust gas heat exchanger	Recovers the heat generated during power generation as hot water.
5. Hot-water storage tank	Tank for storing hot water recovered by the exhaust gas heat exchanger.
6. Supplementary boiler	When there is not enough hot water in the hot water storage tank, the supplementary boiler is used to supply hot water.

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6

6

Supplementary materials-Desiccant unit-



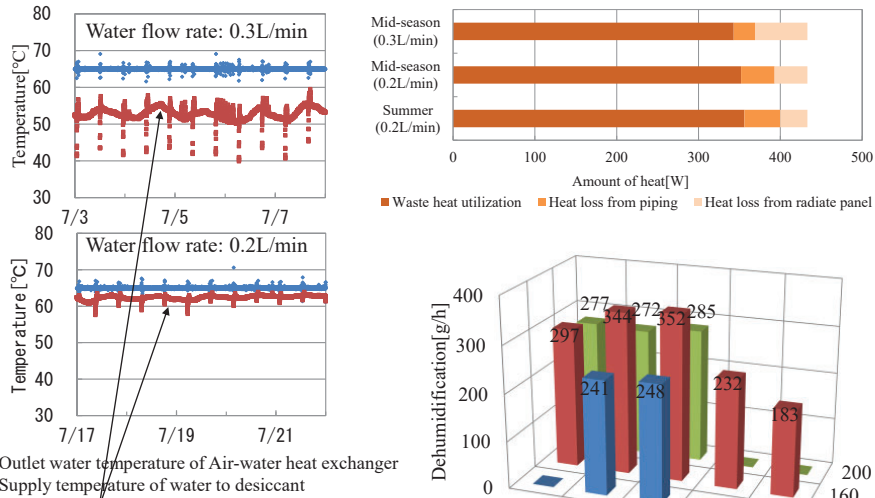
The balance of air volume that maximizes the amount of dehumidification was clarified.

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Supplementary materials



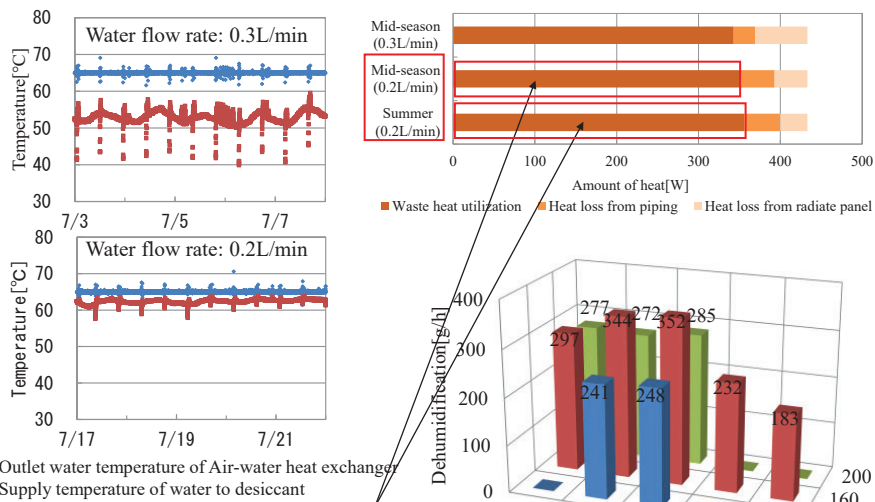
The hot water temperature supplied to the desiccant unit was stable at 62-63°C, approximately 10°C higher when the flow rate was 0.2 L/min.

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Supplementary materials



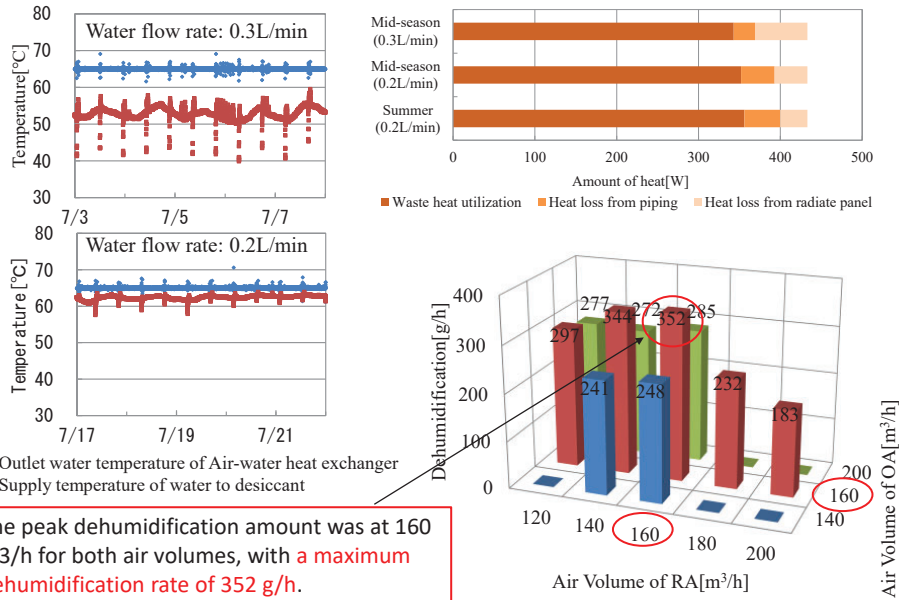
With a flow rate of 0.2L/min, about 80% of the heat emitted from the EF was utilized in the desiccant unit in both the mid-season and summer seasons.

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Supplementary materials

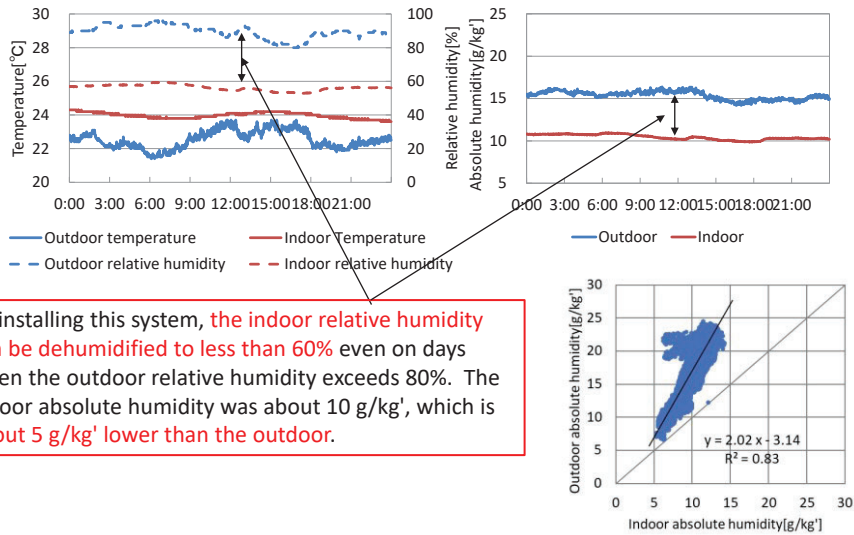


The peak dehumidification amount was at 160 m³/h for both air volumes, with a maximum dehumidification rate of 352 g/h.

Supplementary materials

Okinawa has the harshest humidity environment in Japan.

Figure: Indoor environmental transition during rainy season

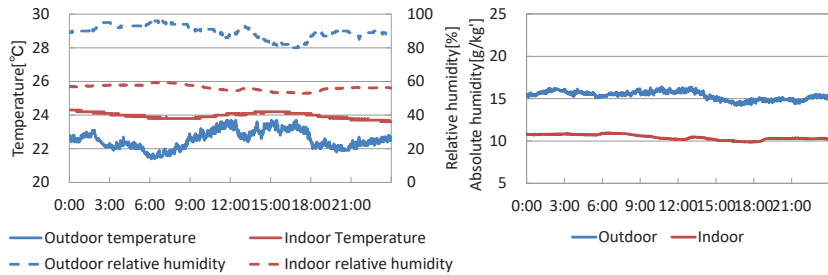


By installing this system, the indoor relative humidity can be dehumidified to less than 60% even on days when the outdoor relative humidity exceeds 80%. The indoor absolute humidity was about 10 g/kg, which is about 5 g/kg lower than the outdoor.

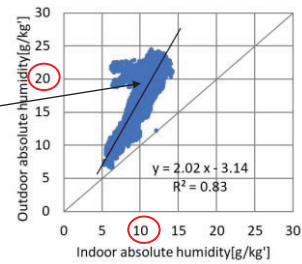
Supplementary materials

Okinawa has the harshest humidity environment in Japan.

Figure: Indoor environmental transition during rainy season



Under the high humidity environment exceeding 20 g/kg', the indoor environment was able to achieve 5 to 10 g/kg' lower than the outdoor air. It can be said that this system has realized a humidity environment favourable to the building and human body throughout the year.

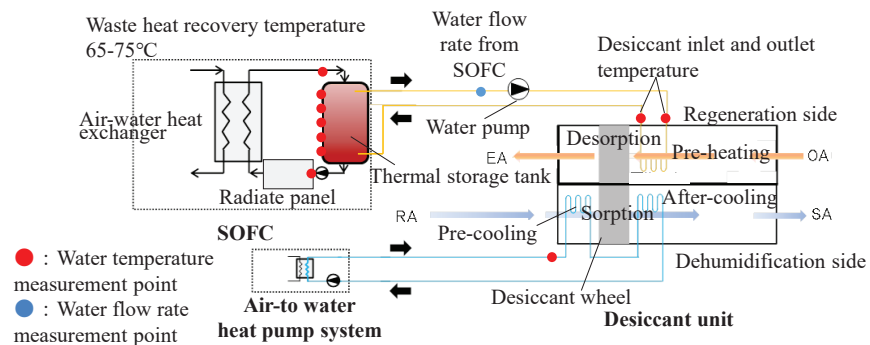


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Supplementary materials



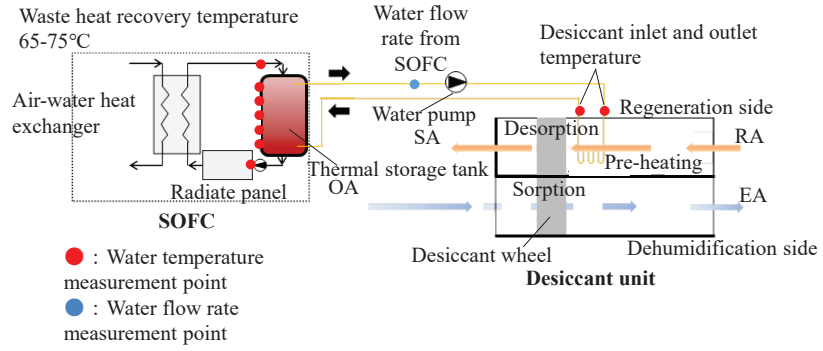
① The waste heat from the EF is recovered by the air-water heat exchanger and stored in the hot water storage tank.

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Supplementary materials



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Supplementary materials

EFデシカントシステムの特徴

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高分子収着剤が低温再生でき、エネ
ファームの排熱が利用できる説明

Method for Evaluating an Air-Conditioning System with Natural Ventilation by Coupled Analysis of a Building Energy Simulation Tool and Computational Fluid Dynamics

YASUNAGA Ryuichi

SHIRAISHI Yasuyuki

The University of Kitakyushu



Shiraishi Lab.



The University of Kitakyushu

1

BACKGROUND

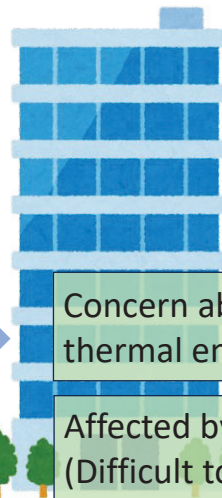
Office building



Outside Comfortable



Natural Vent.



Cooling loads

Concern about uneven indoor thermal environment.

Affected by weather conditions. (Difficult to predict quantitatively.)

2

2

BACKGROUND

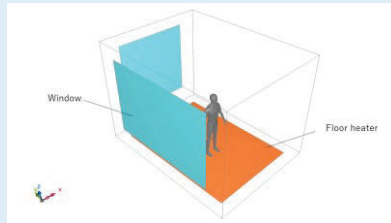
BES (Building Energy Simulation) tool

Comprehensive analysis tool for evaluating the performance of buildings.



Represents the physical quantity of the room as a **SINGLE NODE**

CFD (Computational Fluid Dynamics)



calculate 3D matrix.

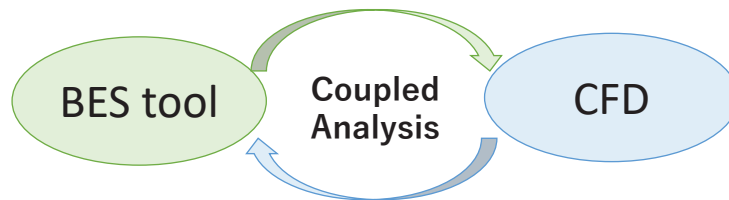
3

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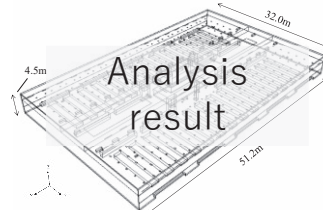
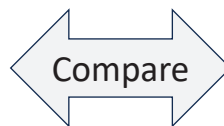
3

BACKGROUND

Propose a method



Verify accuracy

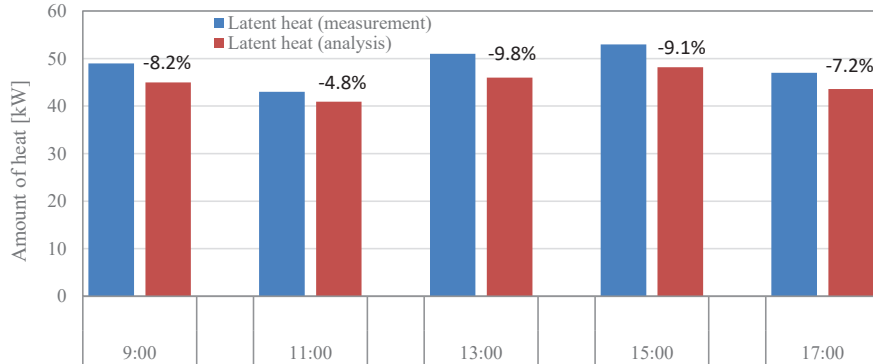


4

The University of Kitakyushu YASUNAGA Ryoichi Shiratshi

4

RESULT – Heat removed by natural vent. -



- Maximum error was under 10 %
- This model has sufficient predictive accuracy

5

The University of Kitakyushu YASUNAGA Ryoichi Shirashi

CONCLUSION

Proposed a method for evaluating an A/C system by coupling a BES tool and CFD.

Verified the prediction accuracy of the model by comparing the simulation results with measurements

Future

- Verification of this analysis model under **unsteady conditions**
- Verify the performance of this building as a case study of the method.


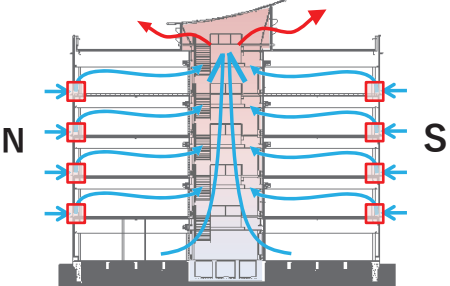
6

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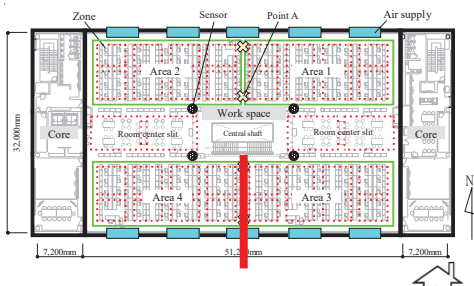
7

OFFICE BUILDING

INFORMATION

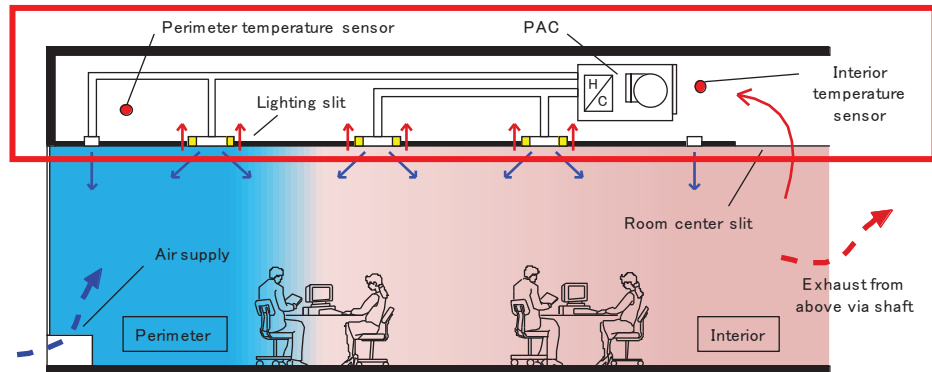
- Japan
- 5 floors
- **BEMS**(building energy management system)



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8

OFFICE BUILDING



9

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MEASUREMENTS

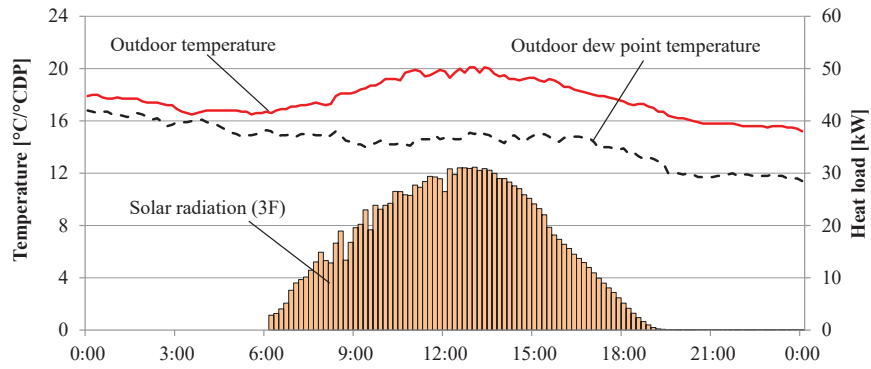


- Measured for May 1st to 31st
- Select May 10th as a representative day

10

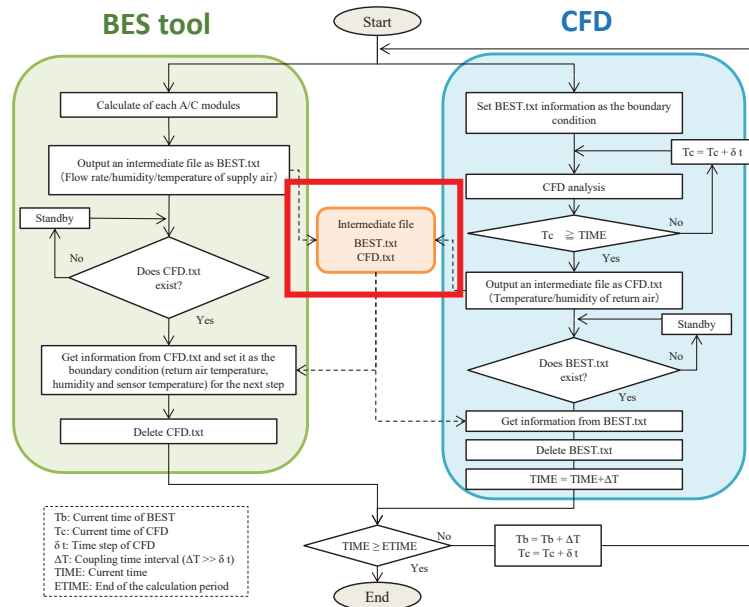
The University of Kitakyushu YASUNAGA Ryoichi Shiratshi

MEASUREMENTS

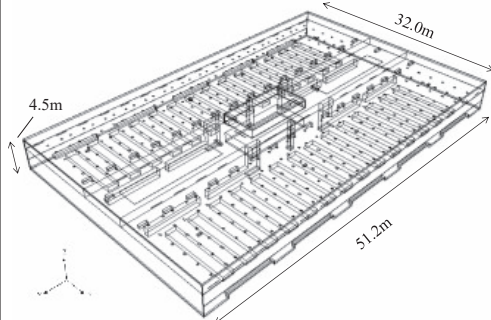


Representative day (May 10)

ANALYSIS

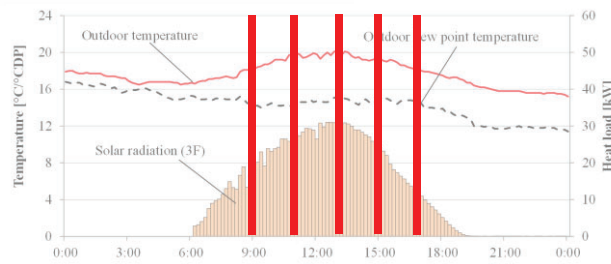


ANALYSIS



Analysis conditions

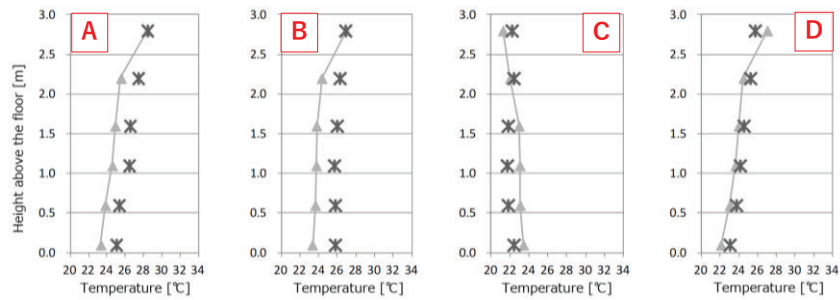
- 3rd floor office room
- Steady-state analyses
- Five times of the representative day



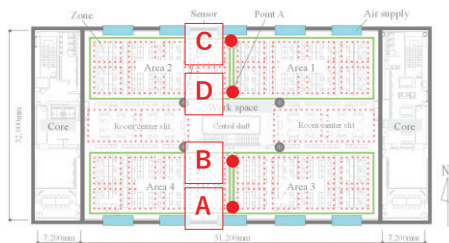
13

13

RESULT – Vertical temperature -



3 p.m.

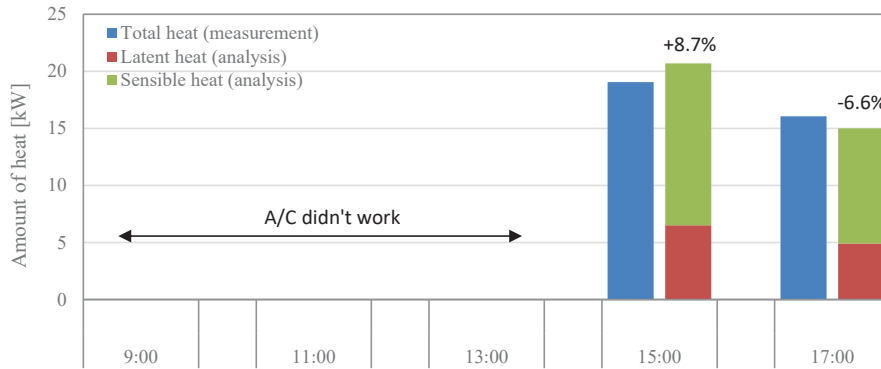


Time	9:00	11:00	13:00	15:00	17:00
RMSE [°C]	0.45	0.66	0.98	0.75	0.97

14

14

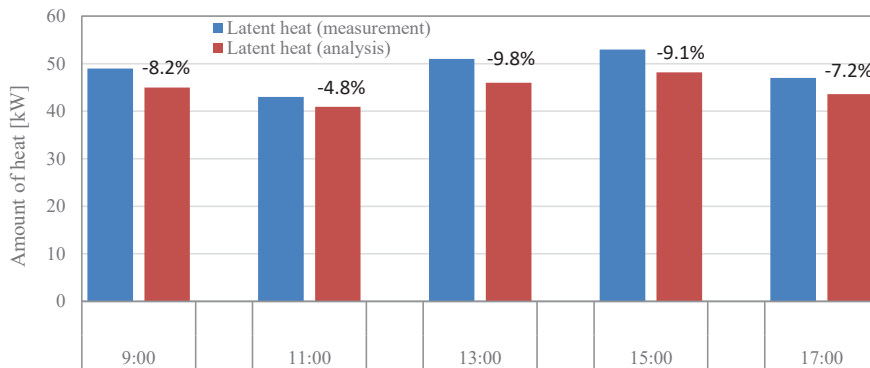
RESULT – A/C heat load -



➤ Maximum error was 8.7 %

15

RESULT – Heat removed by natural vent. -



- Maximum error was under 10 %
- This model has sufficient predictive accuracy

16

CONCLUSION

Proposed a method for evaluating an A/C system by coupling a BES tool and CFD.

Verified the prediction accuracy of the method by comparing the simulation results with measurements

- Verification of this analysis model under **unsteady conditions**
- Verify the performance of this building as a case study of the method.

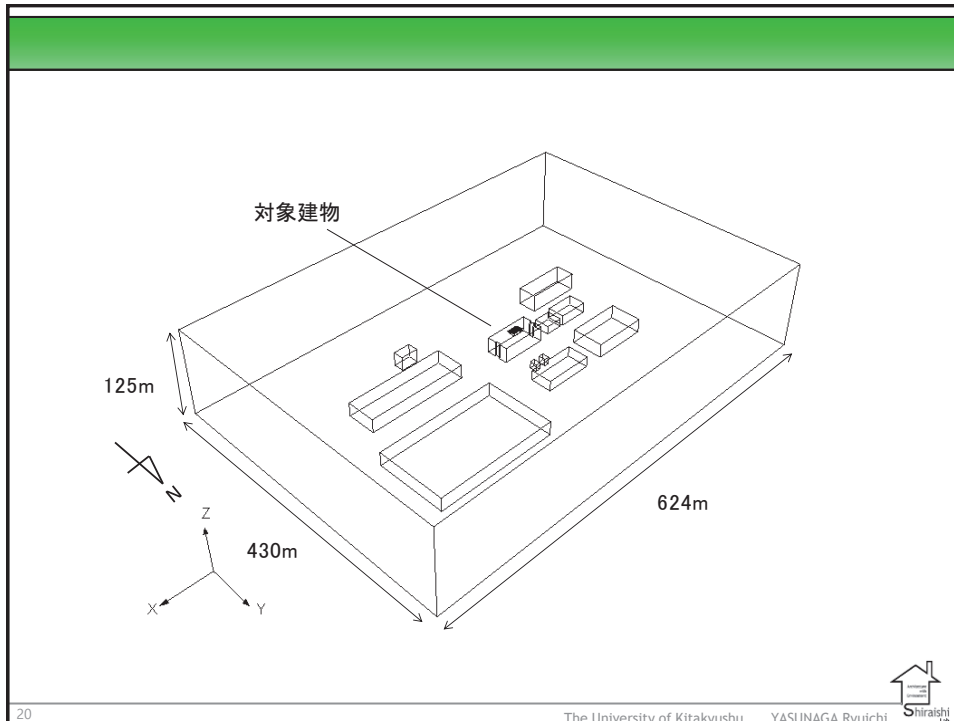


CFD conditions

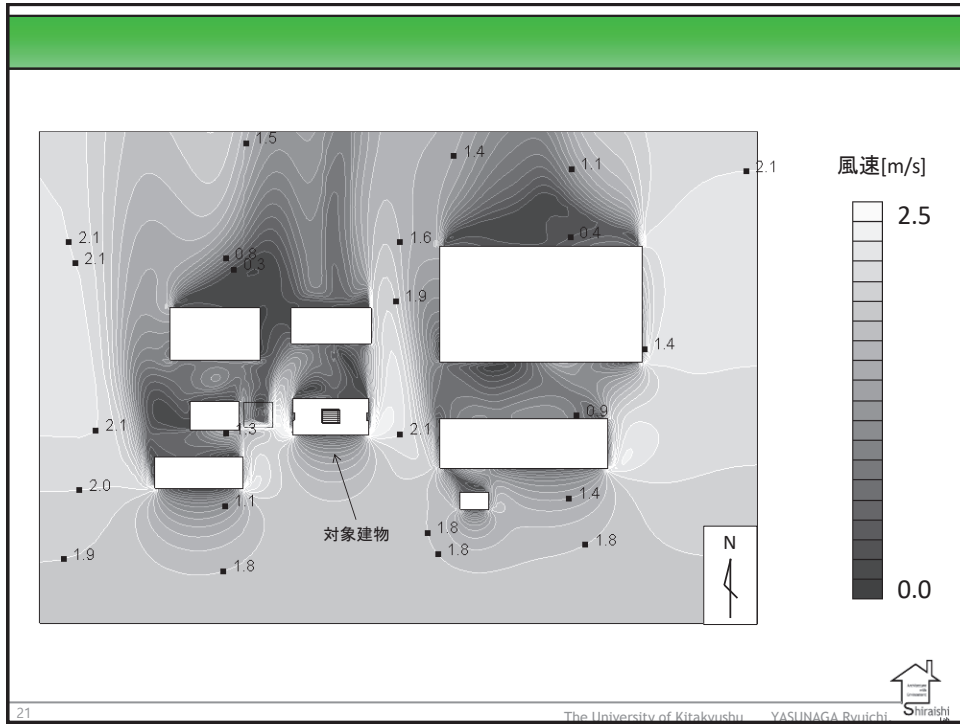
Domain	32.0m(x) × 51.2m(y) × 4.45m(z)	
Mesh	452(x) × 283(y) × 39(z) = 4,988,724	
Mesh for radiation	7,291	
Turbulence model	Standard k-ε model	
Inflow conditions	A/C	Temperature: Proportional control Flow rate: 17.5 m ³ /min per unit $k_{in}=(U_{in}/10)^2$, $\epsilon_{in}=C_{\mu}^{3/4}k_{in}^{3/2}/\ell_{in}$
	Natural ventilation	Temperature: Outdoor temperature of the BEMS data Flow rate: BEMS data $k_{in}=3/2(U_{in} \times 0.05)^2$, $\epsilon_{in}=C_{\mu}^{3/4}k_{in}^{3/2}/\ell_{in}$
Outflow conditions	A/C	Fixed flow
	Natural ventilation	Fixed static pressure
Wall boundary conditions	Temperature: Fixed convection heat transfer coefficient 4.5 W/m ² K Speed: Generalized logarithmic law	
Outside boundary conditions	South/North: Sol-air temperature, Fixed heat transfer coefficient 23W/m ² K East/West: insulation	
Heating conditions	Lighting/Office equipment/Human/Solar radiation: Measured value	

U_{in} : supply wind speed [m/s]; k_{in} : turbulence energy [m²/s²]; ϵ_{in} : dissipation rate of k_{in} [m²/s³]; C_{μ} : model constant (= 0.09); ℓ_{in} : inlet length

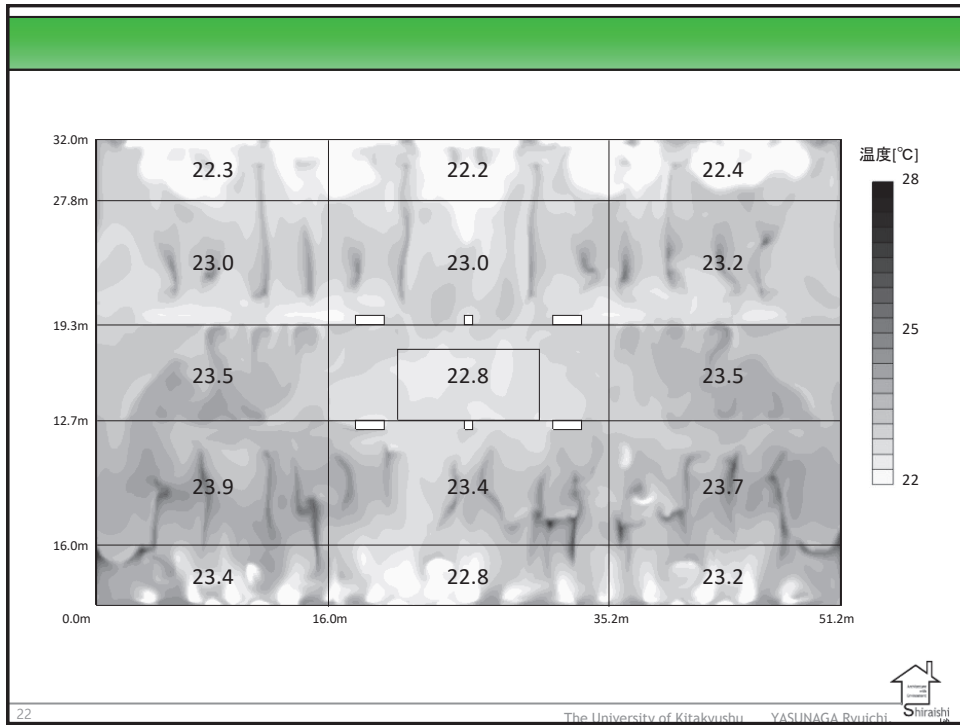
19



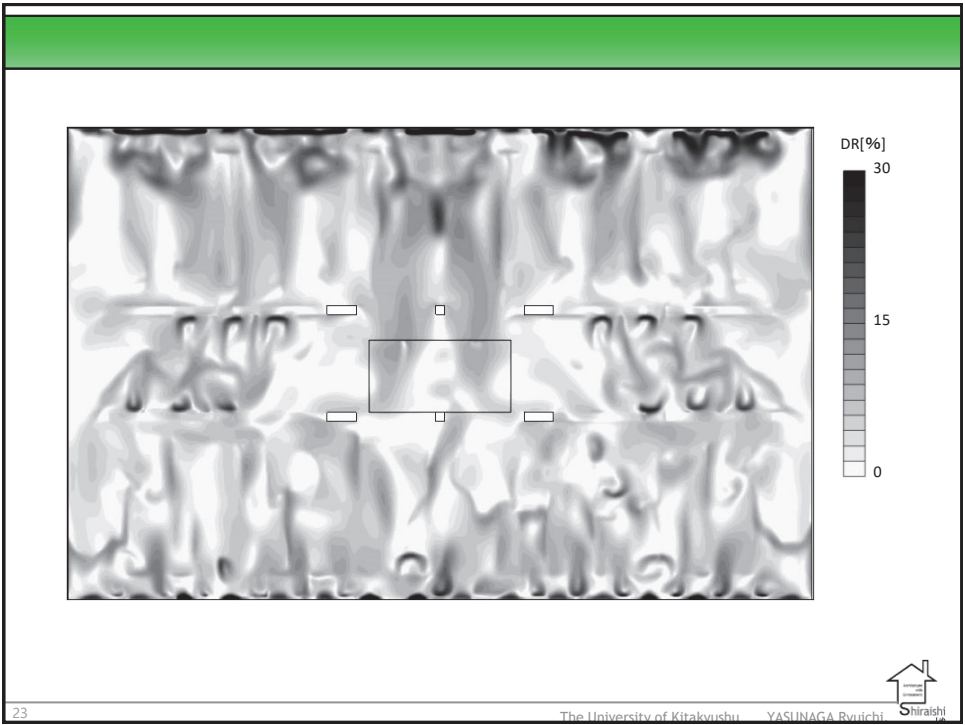
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21



22



Performance comparison of different ventilation strategies in elderly care homes in Belgium



Douaa Al-Assaad, Quinten Carton, Abantika Sengupta, Hilde Breesch
Building Physics & Sustainable Design

1

Context and aim

- Elderly care homes
 - EU field studies
 - Poor IAQ and ventilation
 - Exposure to high levels indoor pollution -> decreased health
 - During pandemic: advise to maximize window opening -> increased energy use & decreased thermal comfort
- Aim
 - Determine window opening strategies to ensure good IAQ throughout year
 - Common room in elderly care home



Quality handbook
"Ventilation in elderly care homes"

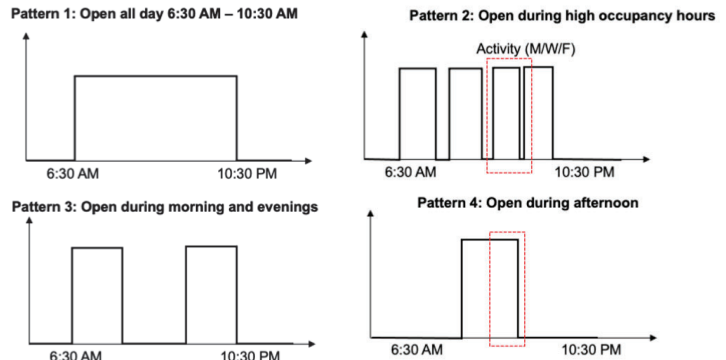
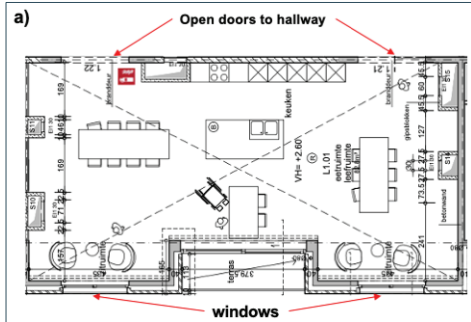
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2

Case study

- Typical common room
 - 13.3 x 6.8 x 2.7 m³
 - 5 to 15 pers

- Ventilation strategies
 - Natural supply & extract (A)
 - Mechanical extract (C) (ACH = 1.5, IDA3)
 - Balanced mechanical (D) (ACH = 1.5, IDA3)
- Window opening patterns



3

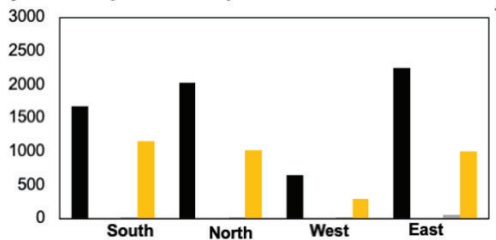
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Results

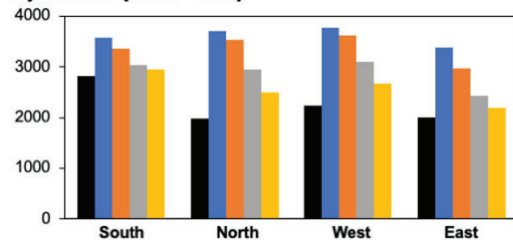
- IAQ heating season (ppm.hours CO₂ > 1000 ppm)

- Energy use heating season (kWh)

System D (ACH = 1.5)



System D (ACH = 1.5)



- NL: max 30.000 ppm.hours/a

4

4

Conclusions

- Guidelines for window opening strategy in common room in elderly care homes

SYSTEM TYPE	SEASON	WINDOW OPENING STRATEGY	PATTERN N°
Natural ventilation	Heating	Open during afternoon	4
	Cooling	Open all day during occupied periods	1
Mechanical extract ventilation (ACH = 1.5)	Heating	Open during morning and evening (south, west, north)	3
		Open during high occupancy periods (east)	2
	Cooling	Open all day during occupied periods	1
Balanced mechanical ventilation (ACH = 1.5)	Heating	Windows can be kept closed	0
	Cooling	Windows can be kept closed	0

Sea Water Air Conditioning (SWAC): A Resilient and Sustainable Cooling Solution for hot and humid climates Energy Performance and Numerical Modeling

Kanhan Sanjivy, Olivier Marc, Franck Lucas

SANJIVY Kanhan – PhD Student - ADEME

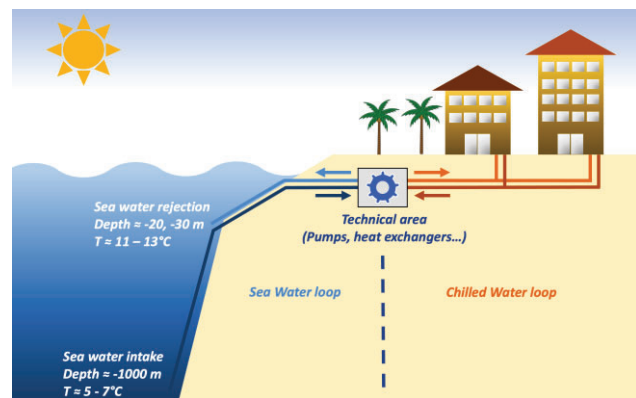


1

Sea Water Air Conditioning (SWAC) process

- Sea Water Air Conditioning (SWAC) uses Deep Ocean Water (DOW) as a cold source to supply a District Cooling (DC):
 - SWAC process efficiency is not limited by Carnot efficiency
 - Seawater temperatures constant throughout the year
 - SWAC doesn't use any refrigerants
- Requirements:
 - High cooling needs close to the sea
 - Great depths near the coast

- SWAC system is divided in three parts:
 - Primary loop (Seawater)**
Drawn around -960 m at 5/6 °C
Discharged around -30 m at 10/11 °C
 - Technical area**
Pumps on both loops with heat exchangers linking them
 - Secondary loop (Chilled water)**
Temperature regime : 7/12 °C



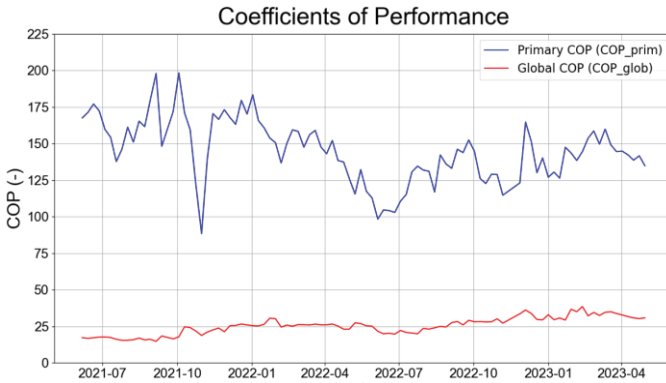
SANJIVY Kanhan – PhD Student - ADEME

04/10/2023

2

2

Energy Performance on operation data



- SWAC process: SCOP = 25,44
- Conventional AC: SCOP = 4

■ Performance indicators

Primary COP : District Cooling excluded
Compared to other centralized systems

Global COP : District Cooling included
Compared to unitary systems

SCOP : Averaged global COP
Consider seasonal variations of performance

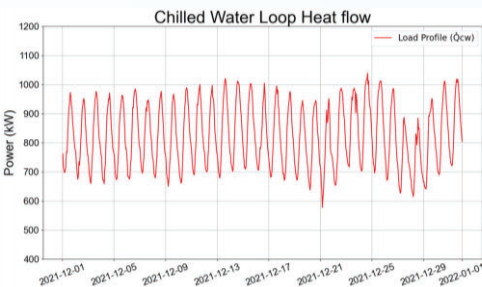
$$COP_{prim} = \frac{Q_{cw}}{W_{sw}}$$

$$COP_{glob} = \frac{Q_{cw}}{W_{sw} + W_{cw}}$$

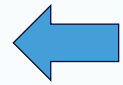
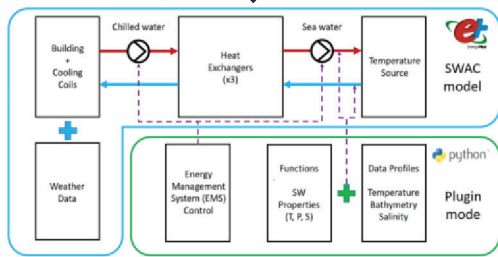
$$SCOP = \frac{1}{T} \int_0^T COP_{glob} \text{ with } T \geq 1 \text{ year}$$

3

Numerical Modeling



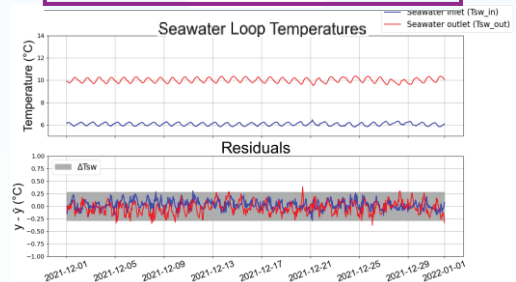
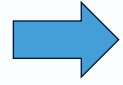
Input replacing building model



Validation method

- Load profile sequence of December 2021 on an hourly timestep input to the model

- Residuals estimation (difference between experimental and predicted value) compared to measurement uncertainties

$$r_i = y_i - \hat{y}_i$$


4

**Sea Water Air Conditioning (SWAC):
A Resilient and Sustainable Cooling Solution for hot and humid climates
Energy Performance and Numerical Modeling**
Kanhav Sanjiv^{1,2}, Olivier Marc^{1,3}, Franck Lucas^{1,3}

Background

- Sea Water Air-Conditioning (SWAC) uses Deep Ocean Water (DOW) as cold source to supply a District Cooling (DC)
- SWAC process efficiency is not limited by Carnot efficiency
- Sea water temperatures constant throughout the year
- SWAC: search for an optimization

Requirements:

- High cooling capacity close to the sea
- Great depths near the coast

Operating process of SWAC system

SWAC system is divided in three parts:

- Primary loop (Desalinated):** Driven around 40°C in at 5/15 °C (biological control) (30°C at 30/12 °C)
- Technical side:** Pumps on both loops with high efficiency linking them
- Secondary loop (DHW cooled):** Temperature regime: 17/12 °C

Objectives

- Present SWAC technology as a resilient cooling solution for tropical climates
- Determine precisely its performance based on years of real operation data
- Perform an experimental validation over a month using measurements on the real SWAC installation
- Develop a design tool model to improve future installations and provide an accurate cost estimation

Material and Methods

Python program for differential equations:

- Control system: flow rate pressure control
- SWAC: $QP = \mu QP^{*}$
- SWAC: $Q = \rho C_p \dot{V} (T_{in} - T_{out})$
- Primary pipeline heat loss: $\dot{Q}_{loss} = U A (T_{pipe} - T_{amb})$

Numerical model of SWAC process

Validation method:

- Load profile: sequence of December 2021 on the SWAC installation input to the model
- Single pump mode validation
- Randomly estimation difference between experimental and predicted values
- Accuracy: level of 0.1-0.2 of temperature measurements

Energy Performance

Cooling demand:

- Q_{cool} = ρ_w C_{p,w} (T_{amb} - T_{ref}) V_{cool}
- Q_{cool} = ρ_w C_{p,w} (T_{amb} - T_{ref}) V_{cool}

Performance indicators:

Primary COP (District Cooling included):

- General for other technical cases: $COP_{DC} = \frac{Q_{cool}}{W_{pumps} + W_{DC}}$
- Comprehensive energy system: $COP_{DC} = \frac{Q_{cool}}{W_{pumps} + W_{DC} + W_{DOW}}$
- General: $COP_{DC} = \frac{Q_{cool}}{W_{pumps} + W_{DC} + W_{DOW} + W_{loss}}$

SWAC primary: $COP_{DC} = 25.44$

Conventional AC: $COP_{DC} = 4$

Results

Experimental Validation

Sea water temperature:

Primary temperature regime:

- Sea temperature: actualized for control loop validation
- Control loop: constant and variable through the cooling period
- Secondary return temperature: around 12°C after the month
- Accuracy: level of 0.1-0.2 of temperature measurements

Deep flow rate:

- Primary flow rate: constant to meet temperature regime of 5/15°C on heat exchanger (distal side)
- Secondary flow rate: constant to meet temperature regime of 17/12°C on heat exchanger (distal side)
- Accuracy: level of 0.1-0.2 for absolute flow rate and 0.1-0.2 for distal side

Conclusion and Future Works

- Integrated system for electricity generation: Digester: Rankine Cycle (ORC) for Ocean Thermal Energy Conversion (OTEC)
- Optimized system for extra cooling needs: Tropical cities: conditioning on seawater discharge (load profile with high demand peak for tertiary building)

1: Université de La Réunion, 2: Institut National de Recherche pour l'Exploitation de la Mer (INM), 3: Institut National de Recherche pour l'Exploitation de la Mer (INM), 4: Institut National de Recherche pour l'Exploitation de la Mer (INM)

Thank you for your attention !

bba building health & comfort
binnenmilieu

The Effects of Lowering Temperature Setpoints on Perceived Thermal Comfort

An experimental case study in office buildings

Beatriz Coutinho
bc-bba@binnenmilieu.nl
Mechanical Engineer Master's project

1

• • •

Methodology

121
Thermal comfort surveys

Thermal comfort
Adaptation mechanisms
Controls
Motivation
Evaluation of the execution of the energy-saving campaign.

Office workers
3 office buildings

- Introduction
- Methodology**
- Results
- Conclusion

2

2

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Results

Occupants feeling disregarded

Complains and lack of motivation

Occupants will be less willing to accept lower temperatures

You can not simply turn the switch

Create resistance toward the application of energy saving measures

Dissatisfied and unproductive employees

Lower motivation levels when energy saving camapaign is not well executed

3

Introduction
Methodology
Results
Conclusion

3

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Conclusion

Occupant Comfort and Preferences

Effective Communication

Lowering Temperature Setpoints

Acceptance of Energy-Saving Measures

Control Over Thermal Environment

Individual Adaptation Strategies

4

Introduction
Methodology
Results
Conclusion

4

Long-term energy performance of dew-point indirect evaporative cooler under the climate change world scenario

 María Jesús Romero-Lara^{1*}, Francisco Comino² and Manuel Ruiz de Adana¹

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²Departamento de Mecánica, Escuela Politécnica Superior, Universidad de Córdoba, Campus de Rabanales, Antigua Carretera Nacional IV, km 396, 14071 Córdoba, Spain

*p42rolam@uco.es; francisco.comino@uco.es; manuel.ruiz@uco.es

Objective

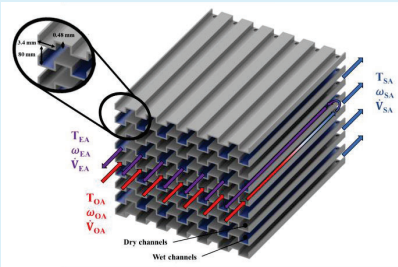
What is the feasibility of using dew-point indirect evaporative coolers under the climate change scenario?

Methodology

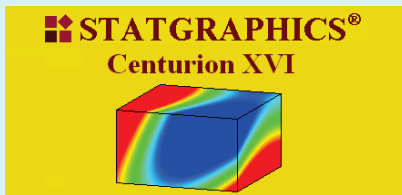


Experimental

1) Tests in DIEC based on DOE statistical technique



2) Empirical model of DIEC



3) Selection of climatic zones



Simulation

4) Climate change scenario (years 1995, 2020, 2050 and 2080)



5) Annual energy simulations



$$SEER = \frac{\sum \dot{Q}_{cooling}}{\sum W} = \frac{Q_{cooling}}{W}$$

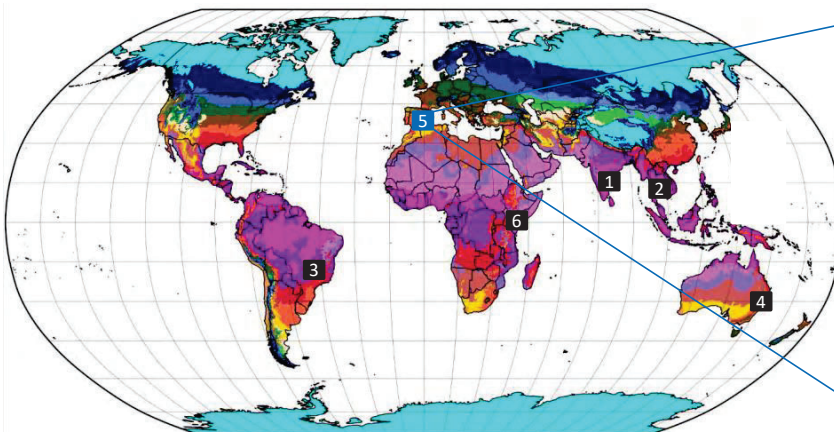
6) Analysis of SEER results



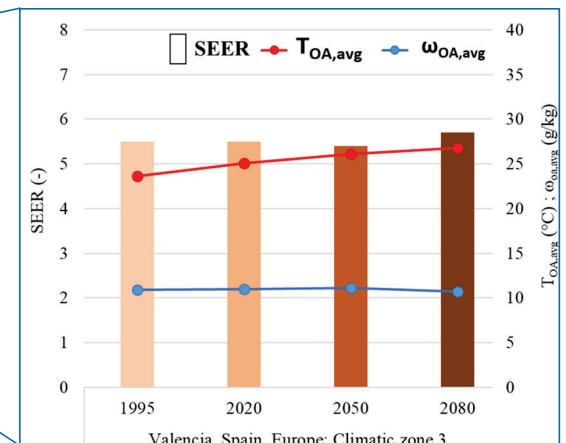
Results and analysis



- Progressive increase in outdoor air temperature (T_{OA}) in the climate change scenario



ANSI/ASHRAE 169–2020 world climate zones map



- High SEER values for DIEC for all cities studied → **above 4.7**
- Outdoor humidity is maintained + outdoor temperature increases → **SEER value increases**

Conclusions

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- **High SEER values for DIEC** were obtained for all cities studied under the climate change scenario, between 4.7 and 6.3.
- DIEC showed higher SEER values for **hot-dry climatic conditions** than for hot-humid climatic conditions.
- The use of **DIEC is feasible** under the climate change world scenario given its **increasing SEER value**.

4/4

Acknowledgments



The authors acknowledge the financial support received by European Union's Horizon 2020 research and innovation programme, through the research project **WEDISTRICT H2020-WIDESPREAD2018-03-857801**

Thanks for your attention!





On the assessment of the pressure coefficient on the mixed ventilation modeling

Pontifical Catholic University of Paraná
(Brazil)
&
University of Savoie Mont-Blanc
(France)

Marcos Batistella
Gaëlle Guyot
Nathan Mendes

October 4 2023, Copenhagen



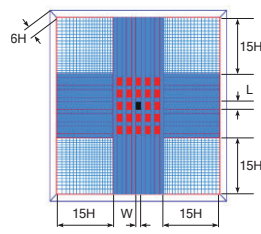
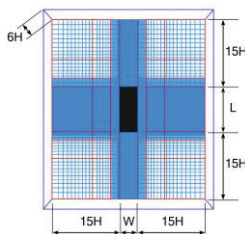
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Validation Study

Isolated

Non-Isolated

CFD



Length scale 1:100

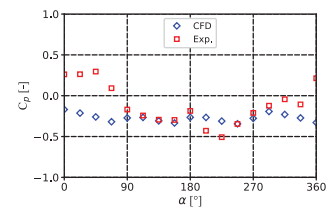
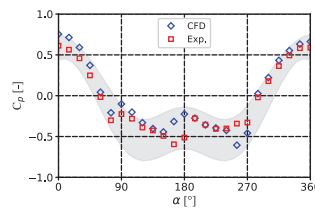
$U_H = 7 \text{ m/s}$

Isolated

Non-Isolated

H = 4m
W = 16m
L = 40m
94×146×46 nodes

H = 12m
W = 16m
L = 40m
AR = 1.0
166×200×30 nodes

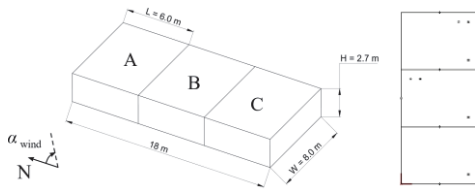


Exp. data from TPU Aerodynamic Database (2023)

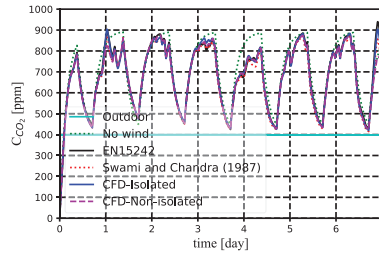
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Test Study

Modified BESTEST MZ320



Zone B



Case	ACH	Er_{ACH}	$\bar{C}_{CO_2}^{Zone A}$	$Er_{C_{CO_2}}^{Zone A}$	$\bar{C}_{CO_2}^{Zone B}$	$Er_{C_{CO_2}}^{Zone B}$	$\bar{C}_{CO_2}^{Zone C}$	$Er_{C_{CO_2}}^{Zone C}$
No wind	0.334	-	59	-	754	-	497	-
EN15242	0.354	+6%	411	+599%	713	-6%	480	-3%
Swami and Chandra (1987)	0.360	+8%	409	+596%	703	-7%	476	-4%
CFD-Isolated	0.357	+7%	412	+602%	708	-6%	478	-4%
CFD-Non-isolated	0.354	+6%	411	+599%	708	-6%	482	-3%

3

Thank you



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venticool
the platform for residential ventilative cooling



Gaëlle Guyot
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Questions?

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4

Construction of operational control rules for an earth-to-air heat exchanger through transfer reinforcement learning

○ Yuki ADACHI

Yasuyuki SHIRAISHI

The University of Kitakyushu

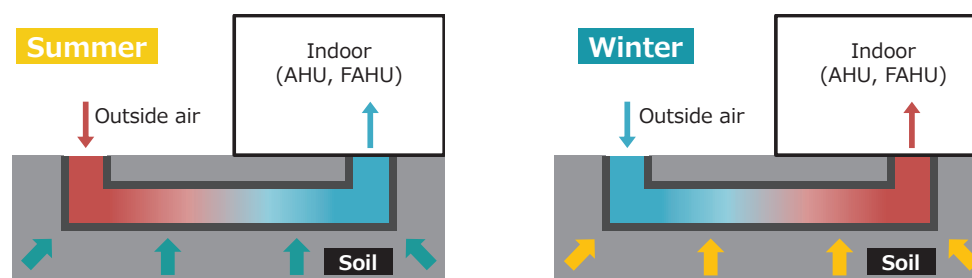
1

INTRODUCTION

Overview of EAHE

Features of EAHE (Earth-to-air heat exchanger) :

- The EAHE utilizes the enormous heat capacity of soil to pre-cool and pre-heat the OA
- The EAHE can reduce the heat load of OA for an AHU or a FAHU by introducing the pre-heated or pre-cooled OA via this system



2

2

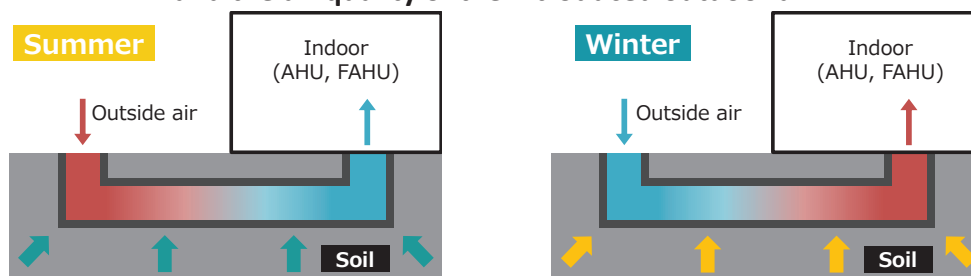
INTRODUCTION

Issues in operation

Common problems for EAHE :

- Risk of air quality contamination of introduced outdoor air
- Dew condensation due to introduction of humid outside air
- Air quality pollution due to growth of mold

It is necessary to establish an optimal operation method that ensures the energy-saving effect of the earth-to-air heat exchange system and the air quality of the introduced outdoor air.



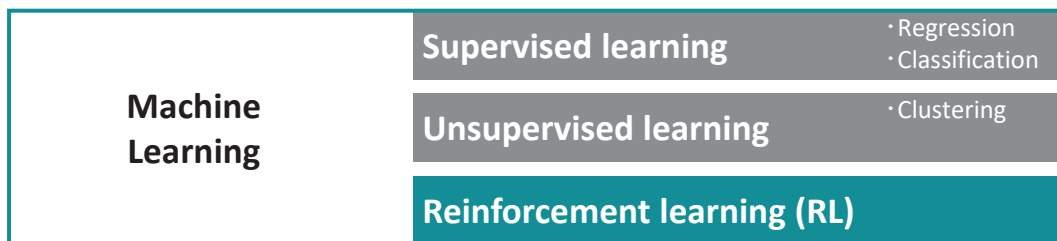
3

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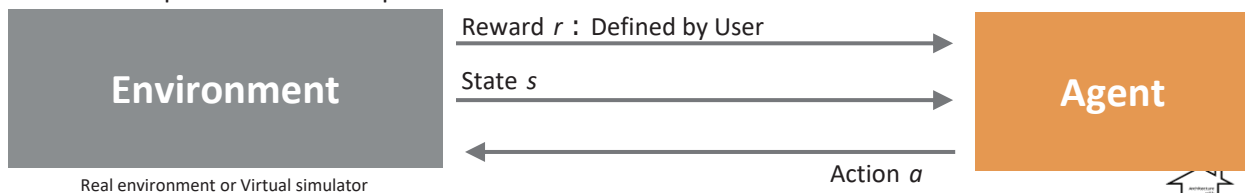
3

INTRODUCTION

Reinforcement learning



- **RL**: Sequential decision-making Goal: Select actions to maximize total future reward
- **Reward**: Indicate how well the Agent's output is doing
- **State**: Information of an environment used to determine next action
- **Action**: Operation to be implemented in an environment



4

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4

INTRODUCTION

Transfer Learning

Reinforcement Learning Issues

Huge number of trials required for learning convergence

CFD as an environment for reinforcement learning → Each analysis takes several hours

→ Simplify the model and reduce the number of meshes → Supports a huge number of trials

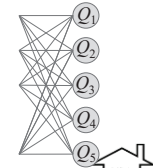
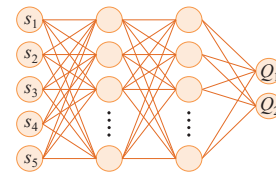
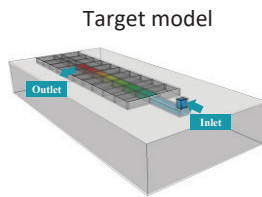
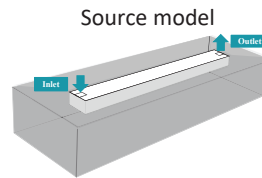
Approach methods for reducing learning time in Reinforcement Learning

Transfer Learning

The transferring agent has
Learned knowledge



Transfer agents reuse



5

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5

INTRODUCTION

Transfer Learning

Reinforcement Learning Issues

Huge number of trials required for learning convergence

CFD as an environment for reinforcement learning → Each analysis takes several hours

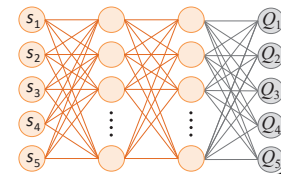
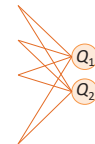
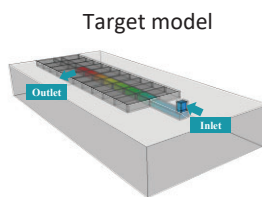
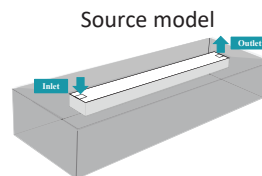
Simplify the model and reduce the number of meshes → Supports a huge number of trials

Approach methods for reducing learning time in Reinforcement Learning

Transfer Learning

Advantage

- ① Can be trained with less data
- ② Reduced learning time
- ③ Improved learning performance



7

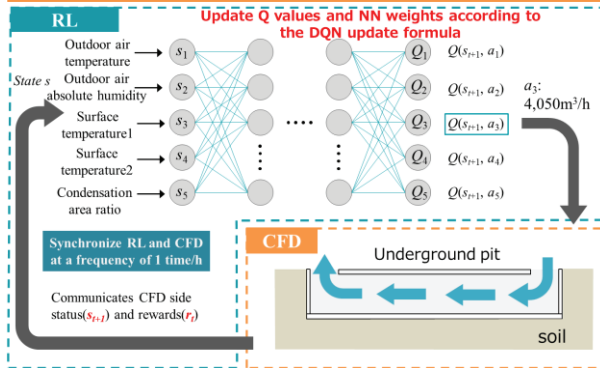
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7

INTRODUCTION

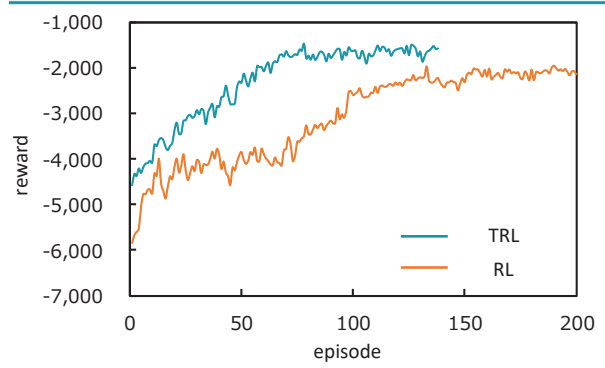
As a study of learning speed-up and learning performance improvement of reinforcement learning

Construction of control laws by reinforcement learning



Coupled Reinforcement Learning and CFD

Combination of transfer learning and reinforcement learning



Learning Progress

Construction of Efficient Operational Control Laws for EAHE Systems Using Reinforcement Learning

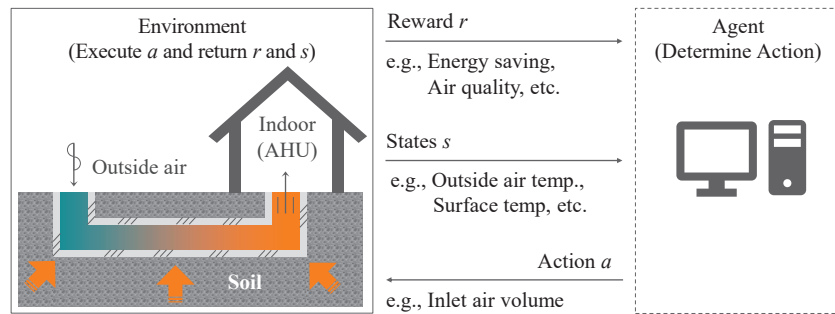
Purpose of this study :

As a study of learning speed-up and learning performance improvement of RL

- Construction of control laws by RL
- Combination of TL and RL



Construction of efficient operational control laws for EAHE systems using RL

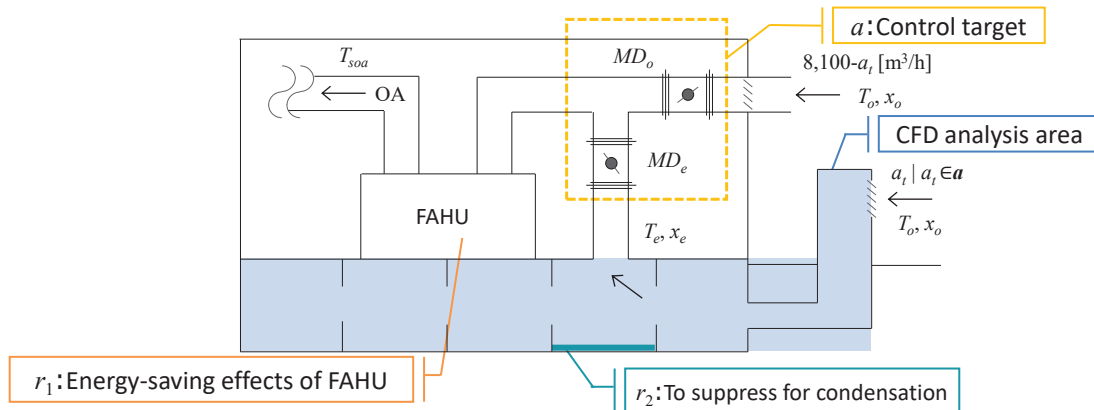


Expected Results in Transfer Reinforcement Learning

- ① Once the control side of the soil heat exchange system is built by reinforcement learning, control models can be built for various targets in a short time.
- ② Direct reuse of control laws, i.e., sufficient control performance may be obtained for practical use without new learning.
- ③ Accelerate learning while improving learning performance

Reinforcement learning control –Method-

- State s : Outside air temperature • absolute humidity , Surface temperature in pit (two points), Condensation area
- Action a : MD_o / MD_e OA damper opening → **Amount of outside air introduced through the system**(5types)
- Reward r : r_1 : **Energy-saving effects of FAHU** , r_2 : **To suppress for condensation**



12

12

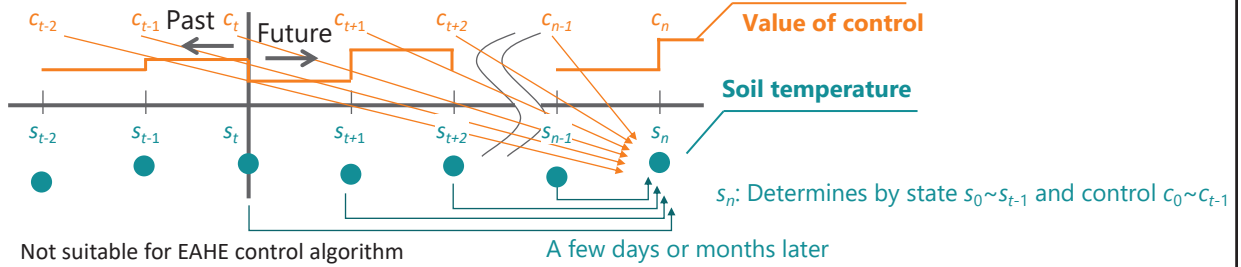
CASE	Source	Target	Subject of transfer
1	2	5	Action
2-1	Straight	L	Shape
2-2	Straight	Corridor	
2-3	Straight	Meandering	
2-4	Straight	Actual tunnel	
3-1	Kitakyusyu	Fukuoka	Weather
3-2	Kitakyusyu	Akita	

13

13

Application of conventional control :

- EAHE uses enormous heat capacity of soil
 - > Sequential operations are affected over a period of days to months
- We need to consider the situation a few months ahead
 - > Real-time prediction of the future is complex in terms of accuracy and calculation time



Ventilation and Thermal Performance Examination of Slot Line Diffuser for Perimeter Usage by CFD Simulation



○Shaoyu Sheng
Toshio Yamanaka
Tomohiro Kobayashi

Specially Appointed Researcher
Professor
Associate professor
@Osaka University, Japan

1

◆ Background

2

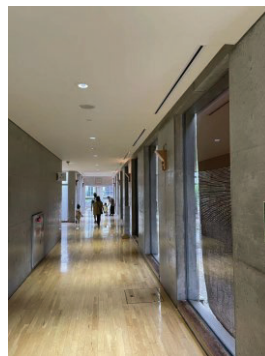
The slot line (linear slot) diffuser



@Office



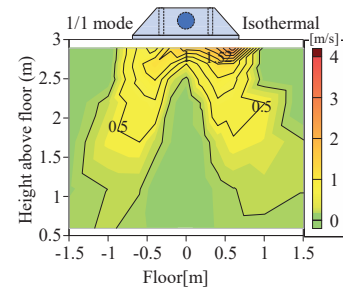
@Ferry boat



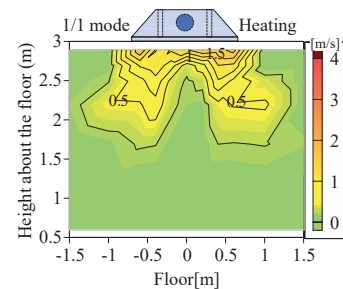
@Hotel

- An air terminal widely used in Japan for near-window application
- Slim outlet area (always 1~2m in long side) and supply air vertically downward
- Can effectively block the heat flow during the summer
- Insufficient throw distance in winter due to buoyancy impact

— Isothermal air supply —



— Heating supply —

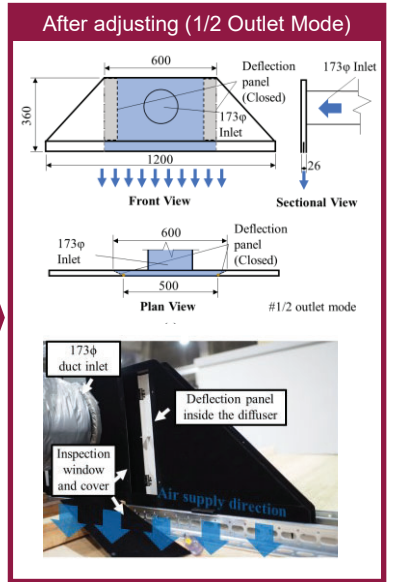
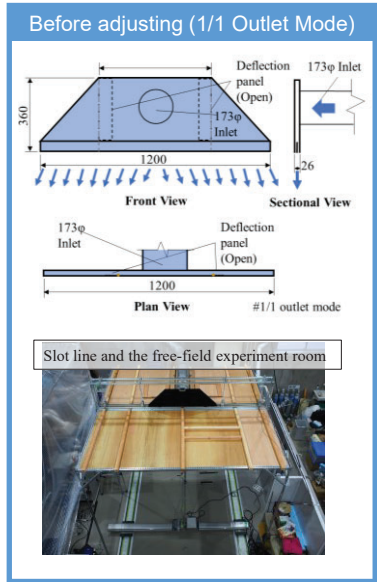


2

◆ Purpose and Method

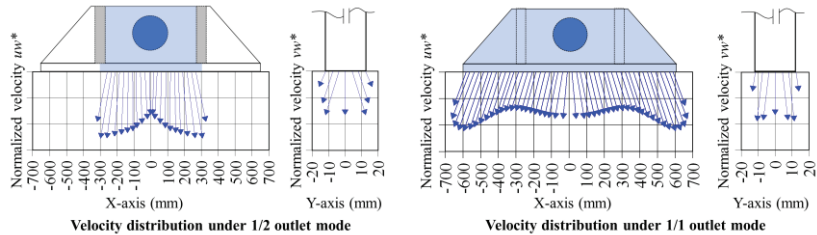
A Novel Slot Line with Adaptive Outlet Area

- Previous studies always emphasized the **slot number** or the **nozzle shape**'s impact
- The **velocity distribution** along the **long side** of the slot line is often **disregarded**
- Deflection panels are putting in the diffuser's chamber to adjust the outlet area in the **longitudinal direction**.
- These panels serve both to **enhance outlet velocity** and **modify airflow** distribution.
- The outlet area can be **adapted** based on either the **supply temperature** or **room temperature**

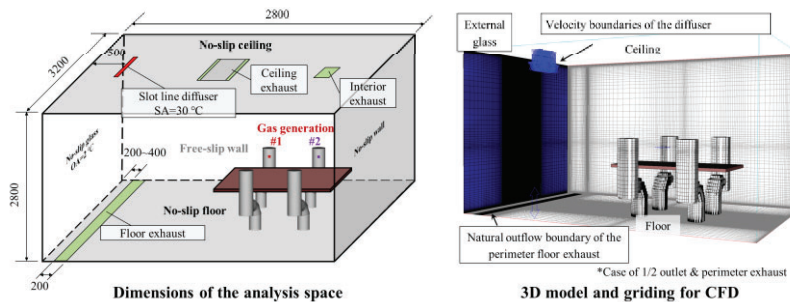


◆ Evaluation by CFD

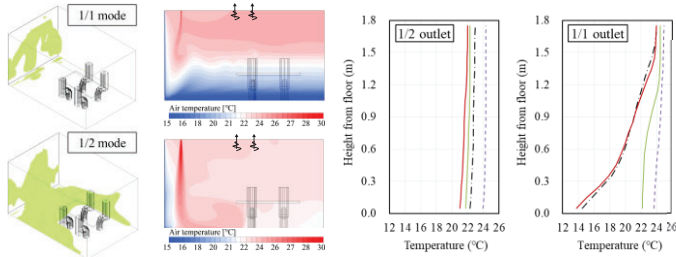
Full-scale experiment in the free field: Proposing and validating a detailed modeling method for replicating outlet airflow



Numerical study by CFD: Evaluating Heating and Ventilation Performance in an Office Space with Large-Area Glazing

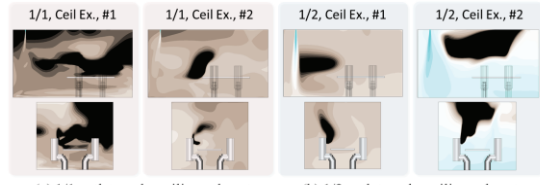


1/2 outlet mode improved the heating and ventilation performance



Velocity and temperature distribution

Velocity and temperature vertical distribution



(a) 1/1 outlet mode, ceiling exhaust

(b) 1/2 outlet mode, ceiling exhaust

Normalized concentration distribution in the vertical section

- The impact of exhaust location has also been examined
- Recommend to use a floor exhaust for extra heating efficiency
- Compared the mean normalized concentration and age of air
- Ceiling displacement ventilation tends to be achieved by combine the 1/2 slot line with a ceiling exhaust



Please refer to our:

- Poster
- Conference paper
- Related research papers

for more details and further examinations

Quantifying the Potential Health Impacts of Unvented Combustion in Homes

Jacob Bueno de Mesquita, PhD
Núria Casquero-Modrego, PhD
Iain Walker, PhD

Lawrence Berkeley National Lab. (LBNL)

10/9/2023

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1

Combustion-Related Contaminants



- **PM_{2.5}** (and **PM₁₀**/ultrafine) ► Generally, the most important contaminant of concern for health impacts
- **NO₂** ► is an irritant to the respiratory tract
- **VOCs** ► health impact minimal compared with PM_{2.5} and NO₂
- **Water vapor and odours** – not directly health-related and not the subject of this paper

This Study:

- **Combustion of Gas** – not oil / wood / propane, etc.
- **NO₂ only** – to disambiguate between gas and electric cooking, PM common to all cooking

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2

What is “Unvented”



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Most combustion equipment has a vent to outside:

- Boilers
- Furnaces
- Water heaters

This study is about unvented combustion:

- Cooking with gas
- Room heaters

3

Methodology: Meta-Analysis

- Review of publications since the year 2000.
- Literature related to:
 - Interventions where gas combustion appliances were replaced.
 - Effective engineering controls were implemented.
 - Measured NO₂ was taken as the main exposure variable.
 - Meta-analysis focused on gas stove effects on cough and wheeze.

Studies that used epidemiologic methods:

▶ Associations between health outcomes and the presence or use of unvented gas combustion appliances and/or measured NO₂ with results presented as effect estimates with confidence intervals.

4

Summary of relevant articles

TOTAL 29 Studies

- 10 meta-analyses
- 12 observational studies
- 7 RCTs or quasi-experimental studies

TOTAL ► 184 effect estimates



5

Types of Studies



- Primarily Acute exposures – monitoring over several years
- RCT and observational studies isolated exposure from unvented appliances
 - Does not include exposure to contaminants from vented sources that enter a building from outdoors.
 - Studies compared home with indoor unvented combustion to homes without, for the same outdoor conditions
- Meta-analyses based on overall NO_2 concentrations and presence of unvented appliances



6

Health Outcomes: 12 categories

1. Mortality
2. Hospital visits
3. Health symptoms
4. Others

- All-cause mortality.
- Cardiovascular mortality.
- Respiratory mortality.
- Healthcare visits or emergency room visits or changes to asthma management.
- Hospitalization or emergency room visits
- Asthma symptoms including wheeze, cough, shortness of breath, chest tightness, respiratory symptoms, nasal symptoms, and difficulty breathing.
- Asthma symptom scores including ordinal scores 0-3 (3 as the most severe and 0 as no symptom)
- Systemic symptoms including poor/fair health, diarrhea, vomiting, ear infection stomach ache, eczema, sensitization, allergies, night-time waking, and steroid use.
- Medication use including asthma preventer and reliever use.
- Nonpharmaceutical interventions including limiting activity.
- Neurological disease development including schizophrenia.
- Absences from school.



Combining results from different studies

- Convert effect estimates into continuous, independent variables.
 - ▶ e.g., effect per 20 ppb increase in NO_2 .
- Convert intervention studies to consider control groups as the numerator in effect estimate ratios.
- Use effect estimates adjusted for confounders: (e.g., age, season, mold exposure, etc.)
- Stratified pooled estimates by:
 - Health outcome
 - Categorical or continuous exposure.
 - Effect estimate type:
 - Odds ratios (OR)
 - Risk ratios (RR)
 - Incident rate ratios (IRR)

Geographical Distribution

Gas Cooking

- The Netherlands

Gas Stove

- USA: New York
- Global

Room Heater

- Australia: Adelaide; New South Wales
- New Zealand
- USA: Connecticut; Massachusetts

Indoor

- USA: Connecticut; Massachusetts; Maryland
- Japan

Ambient (exterior)

- Australia: Adelaide
- Denmark
- USA: Maryland; Massachusetts



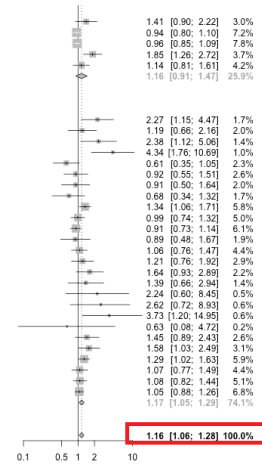
Results

Odds ratios for combustion cooking and/or heating on asthma symptoms

► Overall.... **Gas vs Electric Appliances (both vented AND unvented)**
Associated with 1.16 times the odds of reporting asthma symptoms (95% CI 1.06-1.28)



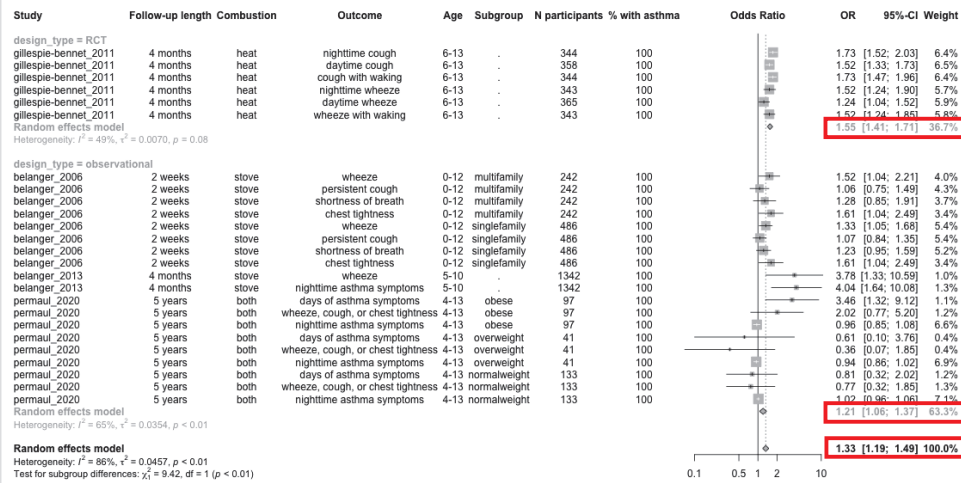
Study	Follow-up length	Combustion	Outcome	Age	Subgroup	N participants	% with asthma	Odds Ratio	OR	95%-CI	Weight
<i>design_type = RCT</i>											
howden-chapman_2008	52 weeks	heat	wheezing attack	6-12	.	349	100	1.41 [0.90; 2.22]	3.0%		
marks_2010	6 weeks	heat	morning wheeze or cough	8-12	.	400	15	0.94 [0.80; 1.10]	7.2%		
marks_2010	6 weeks	heat	evening wheeze or cough	8-12	.	400	15	0.96 [0.85; 1.09]	7.8%		
marks_2010	6 weeks	heat	morning wheeze or cough	8-12	atopy	152	15	1.85 [0.26; 2.72]	3.7%		
marks_2010	6 weeks	heat	evening wheeze or cough	8-12	atopy	152	15	1.14 [0.81; 1.61]	4.2%		
Random effects model Heterogeneity: $I^2 = 70\%$, $\tau^2 = 0.0534$, $p < 0.01$											
<i>design_type = observational</i>											
belanger_2006	2 weeks	stove	wheeze	0-12	multifamily	242	100	2.27 [1.15; 4.47]	1.7%		
belanger_2006	2 weeks	stove	persistent cough	0-12	multifamily	242	100	1.19 [0.66; 2.16]	2.0%		
belanger_2006	2 weeks	stove	shortness of breath	0-12	multifamily	242	100	2.38 [1.12; 5.06]	1.4%		
belanger_2006	2 weeks	stove	chest tightness	0-12	multifamily	242	100	4.34 [1.76; 10.69]	1.0%		
belanger_2006	2 weeks	stove	wheeze	0-12	singlefamily	486	100	0.61 [0.35; 1.05]	2.3%		
belanger_2006	2 weeks	stove	persistent cough	0-12	singlefamily	486	100	0.92 [0.55; 1.51]	2.6%		
belanger_2006	2 weeks	stove	shortness of breath	0-12	singlefamily	486	100	0.91 [0.50; 1.64]	2.0%		
belanger_2006	2 weeks	stove	chest tightness	0-12	singlefamily	486	100	0.68 [0.34; 1.32]	1.7%		
willers_2006	8 years	stove	nasal symptoms	0-8	.	3148	4	1.34 [1.06; 1.71]	5.8%		
willers_2006	8 years	stove	wheeze	0-8	.	3148	4	0.99 [0.74; 1.32]	5.0%		
willers_2006	8 years	stove	transient early wheeze	0-8	.	3148	4	0.91 [0.73; 1.14]	6.1%		
willers_2006	8 years	stove	late onset wheeze	0-8	.	3148	4	0.89 [0.48; 1.67]	1.9%		
willers_2006	8 years	stove	persistent wheeze	0-8	.	3148	4	1.08 [0.76; 1.47]	4.4%		
rice_2020	28 months	both	wheeze or cough	0-16	bpd	244	.	1.21 [0.75; 1.92]	2.9%		
rice_2020	28 months	both	nighttime respiratory symptoms	0-16	bpd	244	.	1.64 [0.93; 2.89]	2.2%		
rice_2020	28 months	both	wheeze or cough	0-16	bpd with resp support	114	.	1.39 [0.66; 2.94]	1.4%		
rice_2020	28 months	both	nighttime respiratory symptoms	0-16	bpd with resp support	114	.	2.24 [0.60; 8.46]	0.5%		
lu_2018	0 weeks	heat	asthma-like symptoms	6-11	.	231	.	2.62 [0.72; 8.93]	0.6%		
lu_2018	0 weeks	both	flu-like symptoms	6-11	.	242	.	3.73 [1.20; 14.95]	0.6%		
paulin_2017	0 weeks	both	daytime asthma symptoms	5-12	.	30	100	0.63 [0.08; 4.72]	0.2%		
boulic_2012	0 weeks	heat	asthma symptoms current	6-12	gas vs electric heat	3874	.	1.45 [0.89; 2.43]	2.6%		
boulic_2012	0 weeks	heat	asthma symptoms current	6-12	kerosene vs electric heat	3874	.	1.58 [1.03; 2.49]	3.1%		
boulic_2012	0 weeks	heat	asthma symptoms current	6-12	unflue vs flue combustion heat	3874	.	1.29 [1.02; 1.63]	5.9%		
boulic_2012	0 weeks	heat	asthma symptoms ever	6-12	gas vs electric heat	3874	.	1.07 [0.77; 1.49]	4.4%		
boulic_2012	0 weeks	heat	asthma symptoms ever	6-12	kerosene vs electric heat	3874	.	1.08 [0.82; 1.44]	5.1%		
boulic_2012	0 weeks	heat	asthma symptoms ever	6-12	unflue vs flue combustion heat	3874	.	1.05 [0.88; 1.26]	6.8%		
Random effects model Heterogeneity: $I^2 = 45\%$, $\tau^2 = 0.0179$, $p < 0.01$											
Random effects model Heterogeneity: $I^2 = 52\%$, $\tau^2 = 0.0271$, $p < 0.01$ Test for subgroup differences: $\chi^2 = 0.00$, $df = 1$ ($p = 0.96$)											



Results – Asthma Symptoms

20 ppb increase in NO₂ = It is an increase in the odds of asthma symptoms by 33%

Asthma Symptoms and Symptom Scores

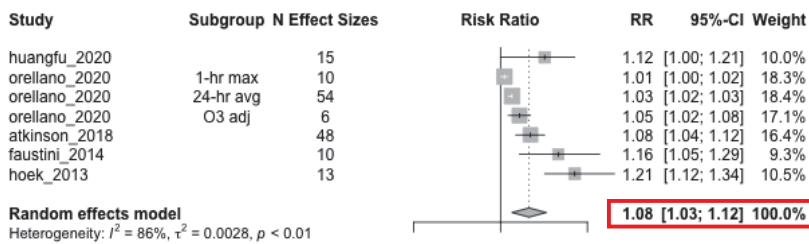


Odds ratios for a 20 ppb increase in average NO₂ exposure linked with combustion cooking and/or unvented heating

11

Results - Death

- Meta-analyses mostly used a continuous NO₂ exposure variable to evaluate risk of mortality, asthma incidence, emergency room visits, and hospitalizations.

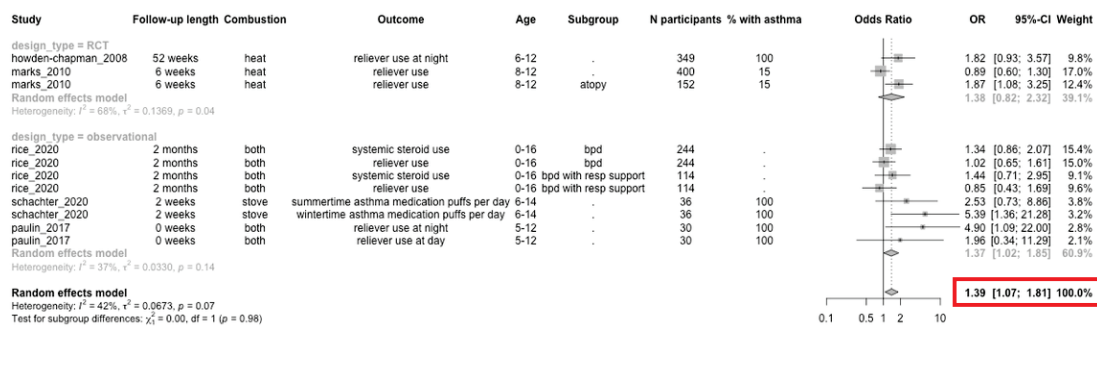


20 ppb increase in NO₂ = 8% increase of DEATH overall

12

Summary

- We found an increased likelihood of using asthma medication across both randomized controlled trials (RCTs) and observational studies, with a combined confidence interval showing 7-81% higher odds when gas appliances were present.



Summary

- Findings** ► Foundation for extrapolating the potential health advantages of broad-scale initiatives aimed at mitigating the effects of unvented combustion.
- Initiatives** ► Enhancing ventilation and source control, transitioning to vented appliances, improving regulations governing device emissions, and quantifying the advantages of electrifying various end-uses.

QUESTIONS...?



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FRENCH PERFORMANCE BASED REGULATION

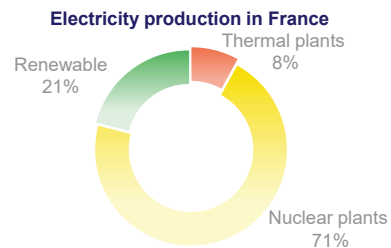
Valérie Leprince

AIVC conference, October 4th and 5th, 2023

1

PERFORMANCE BASED REGULATION AND DECARBONATION

- **New buildings : Electrically heated**
 - Ventilation around 20% of the need
 - **Low impact of ventilation saving on decarbonisation**
- **Existing buildings**
 - More than 50% fossil energy for heating
 - Prescriptive regulation difficult to apply in refurbishment
 - Need for alternative performance-based regulation
- **Embodied energy in ventilation system**
 - Taken into account in new RE2020
 - Performance base regulation promote/allow innovation
 - Just-tech systems



Performance-based approaches



Required IAQ performance indicators to be achieved by the ventilation systems instead of ventilation rates.

2

Prescriptive French regulation

> Arrêté de 1982

3

ARRÊTÉ DE 1982

Art. 1: The air renewal in dwelling is general and permanent at least during the heating season.

Art2: The air renewal system shall include natural or mechanical inlet in main rooms and outlet in utility rooms. The air shall circulate between main and utility rooms

Art 3: The ventilation system shall be able to reach, simultaneously or not the following values:

Number of main rooms in the dwelling	Extract flowrate in m ³ /h				
	Kitchen	Bathroom	Other room with water source	Toilet	
				Only one	Multiple ones
1	75	15	15	15	15
2	90	15	15	15	15
3	105	30	15	15	15
4	120	30	15	30	15
5 or more	135	30	15	30	15

Additional requirements are set for fire safety and interaction with combustion appliance.

Art. 4: The total extract flowrate can be reduced as follow :

	Number of main rooms						
	1	2	3	4	5	6	7
Total minimal flowrate in m ³ /h	35	60	75	90	105	120	135
Minimal flowrate in the kitchen m ³ /h	20	30	45	45	45	45	45

If the ventilation system **automatically control flowrate** to maintain an indoor air quality that is not dangerous for occupant and avoid condensation (except temporarily) the flowrate can be reduced. Provided that the **system has been validated by the ministry** in charge of construction and health. In any case the total extracted flowrate shall at least be:

	Number of main rooms						
	1	2	3	4	5	6	7
Total minimal flowrate in m ³ /h	10	10	15	20	25	30	35

Art.5: air inlet shall be designed to reach extracted flowrates defined at article 3.

4

Objective of the new performance base regulation



FRENCH PERFORMANCE BASED
REGULATION

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NEW REGULATION

The new construction code states that :

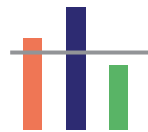
*“Air renewal, shall be such as, in normal condition of use, the **indoor air pollution does not endanger health and security of occupants and that condensation is avoided, except temporarily**”.*

This is respected if the system:

Respects

Arrete de 82 requirements
Art.1 _____
Art.2 _____
Art.3 _____
Art.4 _____
Art.5 _____

Or



Fulfills
Key Performance
Indicators levels*

*(named Résultats minimaux – Minimal results) as defined in a Regulatory text to be published by January 2025.



*Ambitious because **defining KPI for ventilation with minimum is still a matter of research, this is worked on in IEA-EBC Annex 86***



FRENCH PERFORMANCE BASED
REGULATION

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WHAT IS THE PURPOSE OF VENTILATION?

Ventilation is air renewal by purpose-provided means which replaces the air with air coming directly from outdoors. The objective of a ventilation system can be to:

Maintain healthy indoor air
limit indoor-produced pollutant concentration, and/or limit outdoor-produced pollutant concentration,
Ensure olfactive comfort and avoid stuffiness feeling

Improve summer comfort

Regulate humidity level to mitigate the risk of condensation and mold development (building lasting quality),

Etc...

In France the regulation in 1982 has been made to:

- First, **humidity**: avoid condensation,
- Second, **health**: limit indoor produced pollutant concentration and their transfer from utility rooms to main rooms
- Three, **comfort**: limit stuffiness feeling and avoid olfactive discomfort due to the transfer of odors from utility rooms (ex. Kitchen to main rooms)

> **KPI follow this logic**



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REGULATION

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Foreseen new requirements >



• Titre presentation
FRENCH PERFORMANCE-BASED
REGULATION

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PRELIMINARY DECISIONS

Before starting the work on indicators, it has been decided that:



The validation of systems shall be done building's project by building's project

The agreement is valid only for a given ventilation system in a defined architecture and climate.

As some parameters of local climate is difficult to anticipate (such as wind) and may change over the years, parametric study will be done on those parameters

> and not for a ventilation system.

Today system that fulfill Arrêté de 82 (modified in 83) regulation shall have performance in-line with required levels for indicators but not necessarily systematically comply for all kind of dwelling/location etc...

A performance-based regulation shall be more "safe-sided" than a prescriptive one.

The performance will be evaluated on :



the 3 French climate zones



multiple size and configuration



every existing systems

> to help the definition of the required level for key performance indicators



FRENCH PERFORMANCE BASED
REGULATION

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PRELIMINARY DECISIONS

Before starting the work on indicators, it has been decided that:

> The ability of a system in a project to fulfill performance indicators level will be



validated through preliminary simulations



not through on-site measurements

Nevertheless, the ability of indicators to be compared to on-site measurement is a criterion to define them

> Some prescriptive requirements will be kept as a safeguard as everything cannot be planned. The following prescriptive requirements shall be kept:



A general and permanent minimum flowrate (every rooms shall be ventilated)
The foreseen flowrate is twice the second table of Article 4



A non-closable outlet in each utility rooms



A non-closable inlet (or outlet) in each main room



Existing requirements on fire-safety



Maintenance requirements



The system shall not compromise the well-functioning of combustion appliance if any in the dwelling

FRENCH PERFORMANCE BASED
REGULATION

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PRELIMINARY DECISIONS

Before starting the work on indicators, it has been decided that:

- > **The validation of a project will be done by an independent body with a process to be defined**
- > **A system includes a maintenance process and the description of its inspection protocol to check and maintain its performances.**

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DEFINITION OF KPI

The following questions needs to be answered to define performance indicators

On which criteria shall the system be evaluated to determine whether he fulfill the regulation

Indoor air parameters – which ones?
CO₂, humidity, PM_{2.5}, Radon, fictive pollutants, etc.
Energy parameters

Which indicators for each criterion?

Criteria on rooms or occupant's exposition
Cumulated exposure
Maximum exposure
A multi-criteria aggregation, in this case the impact of the weight distribution needs to be clearly explored in order to propose distribution adapted to the IAQ
Else?

Which level of requirements on indicators

Absolut acceptable threshold
Relative performance regarding a reference system ? (theoretical or ideal system)
Else?

> **In the context of the French regulation on ventilation it has been decided not to evaluate performances according the energy performance as it is already included in the energy performance calculation.**

> **In France, to fulfill the objective of ventilation defined in the introduction the chosen criteria regarding indoor quality are the following :**



Humidity



A fictive pollutant P1 emitted continuously in every rooms with an emission rate proportional to the area



A fictive pollutant P2 emitted in the kitchen during cooking events at mid-day and in the evening.

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FORESEEN KPI IN FRANCE



For CO₂

- For each room, the CO₂ concentration (in ppm) below which it remains 67% of occupied time

> This indicator reflects the mean operating conditions

An indicator in ppm is more easily readable than a cumulative exposure in ppm.h

- For each room, the CO₂ concentration (in ppm) below which it remains 95% of the occupied time

> This Indicator reflects pic conditions



For Humidity

- For every room, a maximum percentage of time over 75% of relative humidity during the heating period. The maximum value will depend on the type of room (as surface finishing standards depend on it)
- Number of hour when at least one leak is overpressured and the humidity level is above 75% (for every rooms)



For the fictive pollutant P1 -> permanent emission

- The mean exposure (for the most exposed person)
- The maximum exposure over one hour (moving average, for the most exposed person).

For the fictive pollutants P2 -> Emitted during cooking periods

- The maximum exposure over one hour (or half an hour) (moving average, in the kitchen).
- The mean concentration of pollutant P2 for each bedroom (to avoid transfer)
- For each room but kitchen, the P2 concentration (in ppm) below which it remains 99% of time



FRENCH PERFORMANCE BASED
REGULATION

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CONCLUSION

> France is about to have an alternative to its prescriptive regulation on ventilation developing a performance-based regulation that :

- Open the market to



Smart ventilation

- And systems that will



Maintain or improve the indoor air quality



Limit the energy use and their embodied energy

> This new regulation should help decarbonize the full building stock for two reasons :



- It will give a framework and guidelines for refurbishment and for existing building
- The reduction of the energy use but also on the reduction of buildings' materials impact



Natural, hybrid, low-pressure systems etc...

The objective is to define KPI by the end of 2023 and to publish the regulation by 2025

This regulation should allow the development of low-tech ventilation systems with less impact on embodied energy which is now considered in the French energy performance regulation RE2020.



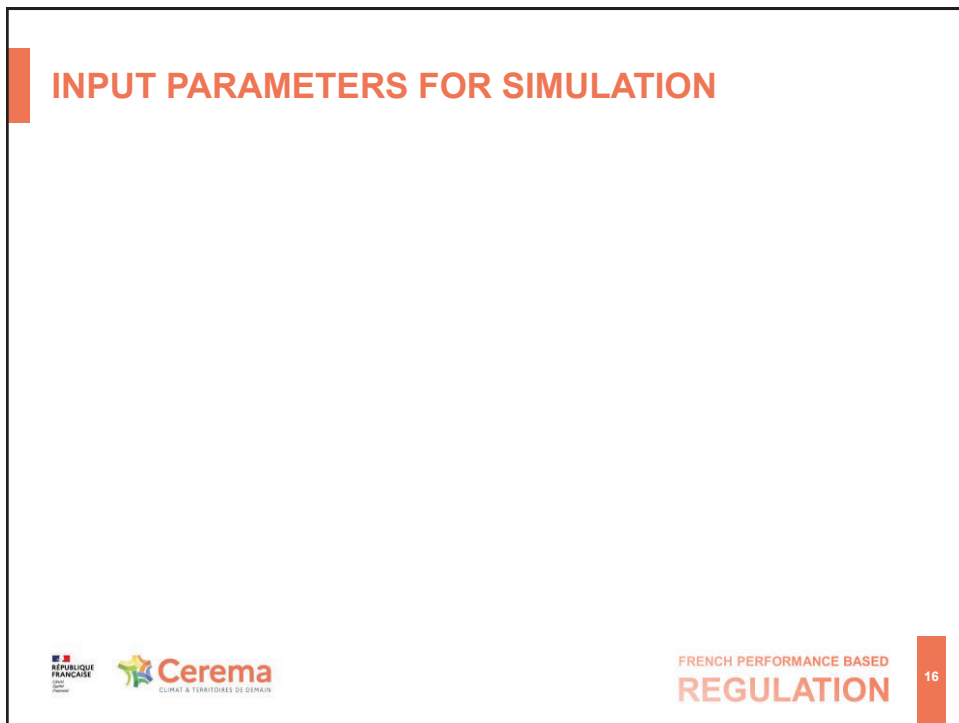
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REGULATION

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REGULATION






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FRENCH PERFORMANCE BASED
REGULATION

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Comparative Analysis Between Indoor Temperatures of Dwellings at Urban Scale During a Typical and Extreme Summers in a Temperate Climate

Ainhoa Arriazu-Ramos, Germán Ramos Ruiz, Juan José Pons Izquierdo, Ana Sánchez-Ostiz Gutiérrez and Aurora Monge-Barrio

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43rd AIVC
11th TightVent
& 9th Venticool
Conference



1

INTRODUCTION

OBJECTIVES

METHODS

RESULTS

CONCLUSIONS

PREDICTIONS...



IPCC → temperature 1.1 °C above 1850-1900 in 2011-2020



Extreme warm temperatures=North Africa

Mega-heatwaves will increase by a factor of 5-10 in the next 40 years



ALREADY HAPPENING...



August 2003 → 50,000 excess deaths

2022 → 110,000 excess deaths (and around 4,500 in Spain)



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RESEARCH OBJECTIVES

This research is focused on...

Quantifying and comparing indoor operative temperatures (IOT) of dwellings in relation to their **building parameters**

The assessment was conducted in **two neighbourhoods of Pamplona** (a city in the north Spain) during a

typical summer (climate series 1980-2010) / **2003 extreme warm summer** and **2022 extreme warm summer with heatwaves**

Considering the **effect of microclimate**

Specific research aims are the following:

1. To quantify the **influence of microclimate** on indoor operative temperatures in dwellings.
2. To compare how **indoor operative temperatures of dwellings are intensified in extreme warm summers** in relation to a typical climate series.
3. To **analyse the influence of different building parameters** (built period, floor level, main orientation, area of windows and number of orientations) on indoor operative temperatures.



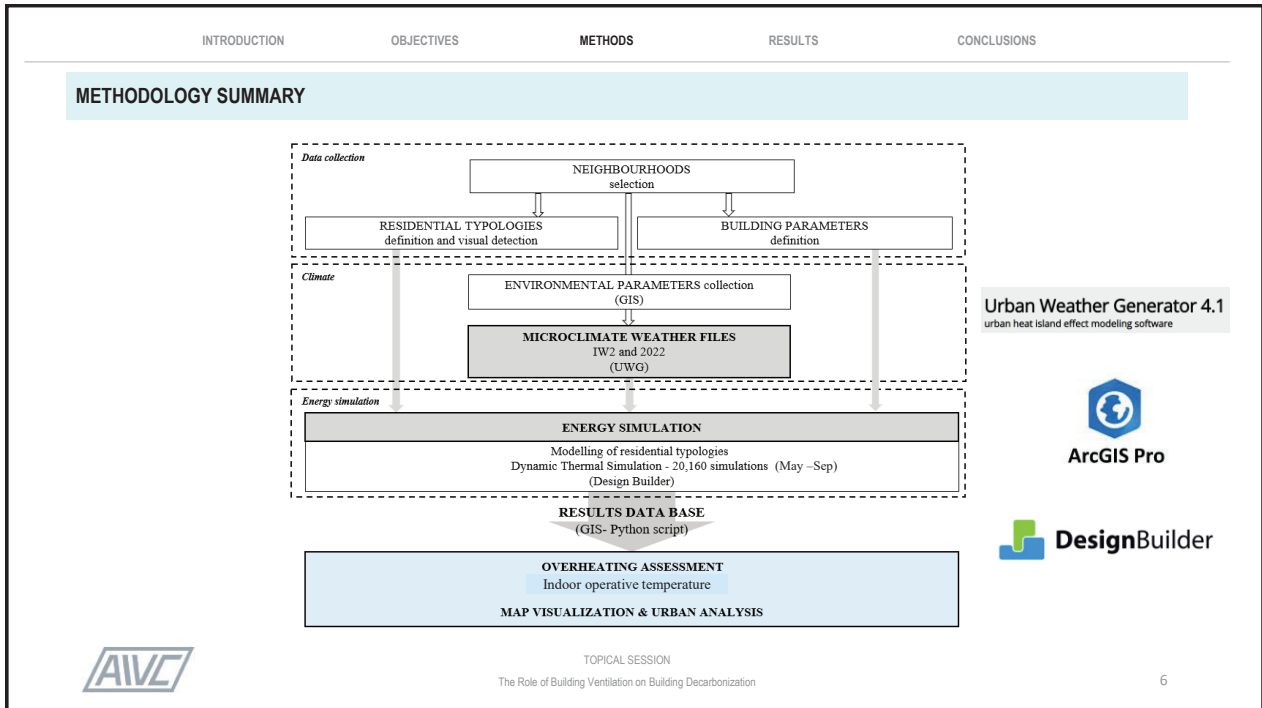
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RESEARCH NOVELTY

- **Simulation with real data of 2003 and 2022 extreme summers with heatwaves**
- **Assess the effect of microclimate on indoor temperatures**
- **The neighbourhood-scale**
- **The simulation results per dwelling**



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INTRODUCTION	OBJECTIVES	METHODS	RESULTS	CONCLUSIONS
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ENERGY PARAMETERS

Energy parameters → built period → energy regulation

N1 was built before 1979 (no energy regulation) when there weren't any energy regulations for buildings.

N2 was built between 1980-2006 (CT-79 period) with the first standard energy regulation in Spain NBE CT-79 which appears after the 1970s energy crisis in other countries.

Infiltration rates were not regulated in Spain until 2019 (with the Spanish Building Code regulation) so the used values are based on previous studies.

There weren't any IAQ regulations in any of the periods and

All dwellings are naturally ventilated.

Used parameters and values for energy simulations

	Built period / Energy regulation	U _{façade / U_{roof} (W/m²K)}	U _{glass / U_{frame} (W/m²K)}	Infiltrations (50Pa)	Solar shading system	Ventilation
ENVELOPE 1	<i>N1</i> No energy regulation	1.39 ^a / 2.9	5.7 / 8.5	7	Blinds with low reflectivity slats ^b	Calculated natural ventilation: Windows free aperture = 15% 1AM- 8AM: 4 ach/h 9AM- 12PM: 0 ach/h Cracks: medium
ENVELOPE 2	<i>N2</i> CT-79	0.73 ^a / 0.65	3.5 / 8.5	7	Blinds with medium reflectivity slats ^b	

^a This value considers the influence of thermal bridges, which worsen the façade transmittance (U) it by 30%.
^b They are considered to be in use (completely down) when solar radiation > 150 w/m²

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INTRODUCTION	OBJECTIVES	METHODS	RESULTS	CONCLUSIONS
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BUILDING TYPOLOGIES

Typologies 11, 12, 13, 14 and 15
→ multi-family buildings grouped in **linear block**:

Typologies 21 and 22
→ multi-family buildings grouped in **H-blocks**

Typologies 31, 32 and 34
→ dwellings in **tower**

Typologies 51
→ single-family dwellings: **attached**

Codes of residential typologies

- 11
- 12
- 13
- 14
- 15
- 21
- 22
- 31
- 32
- 34
- 51

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INTRODUCTION OBJECTIVES METHODS RESULTS CONCLUSIONS

CLIMATE AND MICROCLIMATE

Pamplona : Cfb climate
temperate without dry season, "oceanic" type

Koppen-Geiger classification

Outdoor temperatures (°C)

Climate series 2003 2022

Three weather files were used for energy simulations:

- Climate series (IWEC2-based in climate series 1980-2010 of ASHRAE)
- Two extreme warm summers
 - 2003 → 20 days of heatwave
 - 2022 → 41 days of heatwave

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INTRODUCTION OBJECTIVES METHODS RESULTS CONCLUSIONS

CLIMATE AND MICROCLIMATE

Urban Weather Generator (UWG)

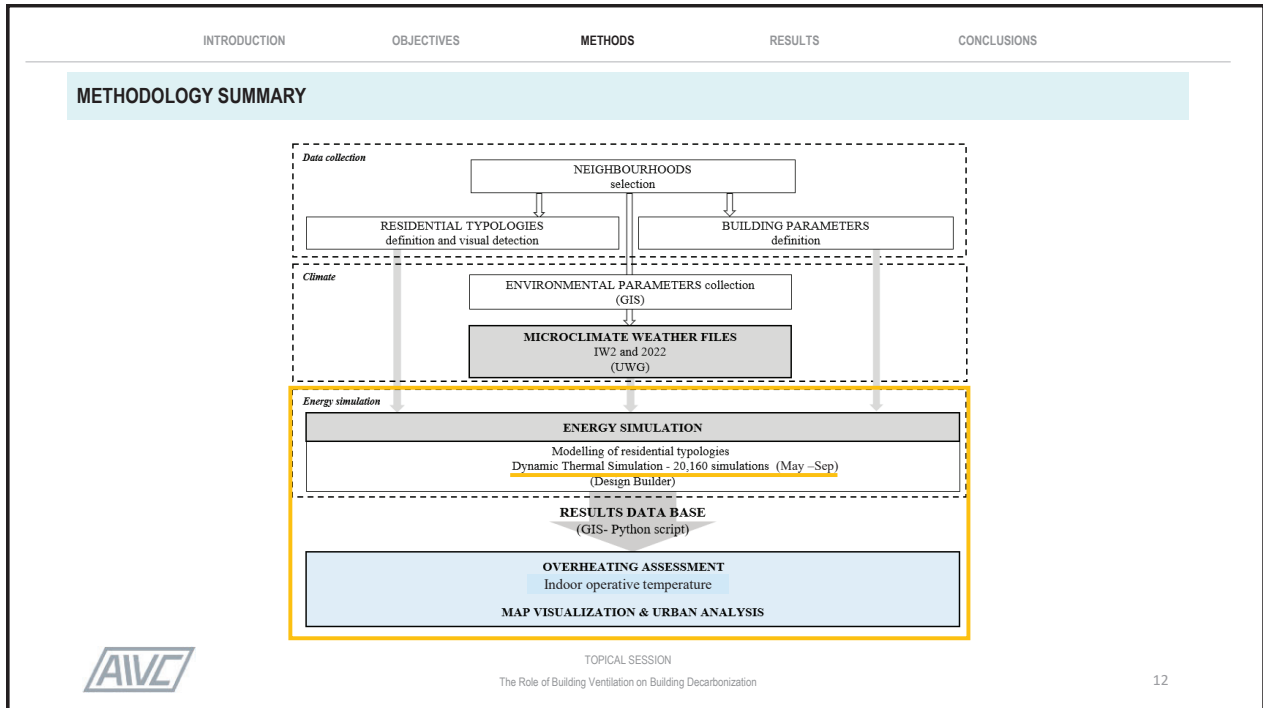
	Site coverage ratio	Facade to site ratio	Road albedo/ Roof albedo/ Wall albedo	Average building height	Urban ground covered in grass/ Urban ground covered in trees	Vegetation albedo	Anthropogenic heat generation (traffic)
	(0-1)	(0-1)	(0-1)	(m)	(0-1)	(0-1)	(w/m ²)
N1	0.34	0.57	0.20/ 0.45/ 0.60	25.45	0.25/ 0.05	0.4	25
N2	0.17	0.18	0.20/ 0.29/ 0.63	10.32	0.47/ 0.02	0.4	25

GIS and Cadastre

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INTRODUCTION OBJECTIVES METHODS **RESULTS** CONCLUSIONS

RESEARCH OBJECTIVE 1

To quantify the influence of microclimate on indoor operative temperatures in dwellings

Differences between mean indoor operative temperatures (°C) considering microclimate or not considering it.

Group	Climate series	2003	2022
N1. base	23.08 (SD. 1.61)	26.01 (SD. 2.82)	26.64 (SD. 2.00)
N1. microclimate	23.22 (SD. 1.45)	26.95 (SD. 3.23)	27.94 (SD. 2.00)
N1. Diff.	0.13 (0.15-0.11) p<0,001	0.93 (0.97-0.89) p<0,001	1.30 (1.32-1.27) p<0,001
N2. base	23.07 (SD. 1.59)	25.90 (SD. 2.73)	26.47 (SD. 1.96)
N2. microclimate	23.40 (SD. 1.26)	26.96 (SD. 3.17)	27.74 (SD. 1.96)
N2. Diff.	0.32 (0.35-0.30) p<0,001	1.06 (1.11-1.01) p<0,001	1.30 (1.33-1.26) p<0,001

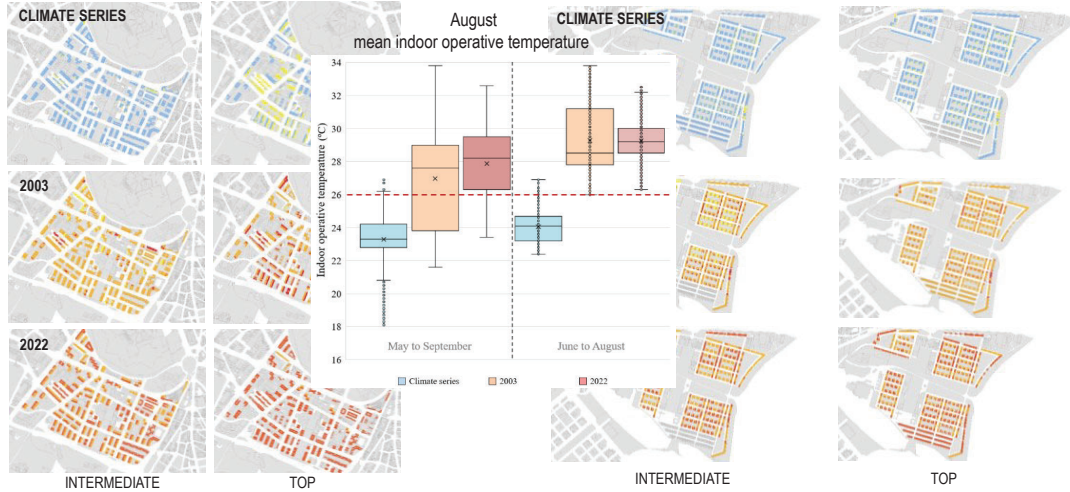
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RESEARCH OBJECTIVE 2

To compare how indoor operative temperatures of dwellings are intensified in extreme warm summers in relation to a typical climate series



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RESEARCH OBJECTIVE 3

To analyse the influence of different building parameters on indoor operative temperatures

Parameters**	Climate series		2003		2022		p value
	Beta Coef.	[95% Conf. Interval]	Beta Coef.	[95% Conf. Interval]	Beta Coef.	[95% Conf. Interval]	
Built period							
No regulation (N1)	0 (Ref.)		0 (Ref.)		0 (Ref.)		
CT-79 (N2)	+0.20	(+0.18 to +0.21)	-0.17	(-0.19 to -0.16)	-0.37	(-0.38 to -0.35)	<0.001
Floor level							
Top floor	0 (Ref.)		0 (Ref.)		0 (Ref.)		
Intermediate floor	+0.31	(+0.29 to +0.32)	-0.20	(-0.22 to -0.18)	-0.40	(-0.42 to -0.37)	<0.001
Orientation							
N/ NE / NW	0 (Ref.)		0 (Ref.)		0 (Ref.)		
S / SW / W	+0.20	(+0.19 to +0.21)	+0.38	(+0.37 to +0.40)	+0.44	(+0.42 to +0.46)	<0.001
E / SE	+0.13	(+0.11 to +0.14)	+0.06	(+0.05 to +0.08)	+0.03	(+0.00 to +0.05)	<0.001
Window area							
≤ 4m ²	0 (Ref.)		0 (Ref.)		0 (Ref.)		
>4m ²	+0.16	(+0.15 to +0.17)	+0.39	(+0.37 to +0.40)	+0.31	(+0.29 to +0.32)	<0.001
N° orientations							
1 orientation	0 (Ref.)						
> 1 orientation	-0.40	(-0.41 to -0.37)	-0.33	(-0.35 to -0.30)	-0.83	(-0.04 to -0.12)	<0.001

multilevel mixed-effects linear regression




TOPICAL SESSION



The Role of Building Ventilation on Building Decarbonization

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INTRODUCTION	OBJECTIVES	METHODS	RESULTS	CONCLUSIONS
Quantifying and comparing indoor operative temperatures (IOT) of dwellings in relation to their building parameters in different climate scenarios and at urban scale				
1. Microclimate effect on indoor operative temperatures in dwellings <div style="border: 1px solid #ADD8E6; padding: 5px; margin-left: 20px;"> Statistically significant ($p < 0.05$) Higher when the summer was warmer → reaching a difference between means of 1.3°C in 2022. </div>				
2. Indoor operative temperatures of dwellings are intensified in extreme warm summers in relation to a typical climate series <div style="border: 1px solid #ADD8E6; padding: 5px; margin-left: 20px;"> 2003 and 2022 showed higher indoor operative temperatures → difference of 4.1°C on average 2003 and 2002 → most dwellings are over 26°C </div>				
3. Influence of built period, floor level, orientation, window area and number of orientations on indoor operative temperatures <div style="border: 1px solid #ADD8E6; padding: 5px; margin-left: 20px;"> Statistically significant ($p < 0.05$) Worst parameters → built before any energy regulation / top floors / S and W façade / single orientation dwelling / windows area > 4m² Intensified in old neighbourhood and in 2003 and 2022 </div>				
		<small>The Role of Building Ventilation on Building Decarbonization</small>		<small>16</small>

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INTRODUCTION	OBJECTIVES	METHODS	RESULTS	CONCLUSIONS
Quantifying and comparing indoor operative temperatures (IOT) of dwellings in relation to their building parameters in different climate scenarios and at urban scale				
				
In temperate climates <ul style="list-style-type: none"> • Assess temperatures through a summer with heatwaves • Considering microclimate • Identify the key building parameters → to adapt dwellings to warming conditions and prevent the risk of overheating within cities 				
		<small>TOPICAL SESSION The Role of Building Ventilation on Building Decarbonization</small>		<small>17</small>

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Thank you for your attention!

Project behind: "*ClimaReady -Adaptation Assessment of Spanish residential buildings*" (2020-2023)

School of architecture - University of Navarra (Spain)

aarriazu@unav.es



AVC **Tight Vent Europe** *venticool* **2023**
BUILDING AND OUTDOOR AIRFLOW PLATFORM the platform for resilient ventilative cooling

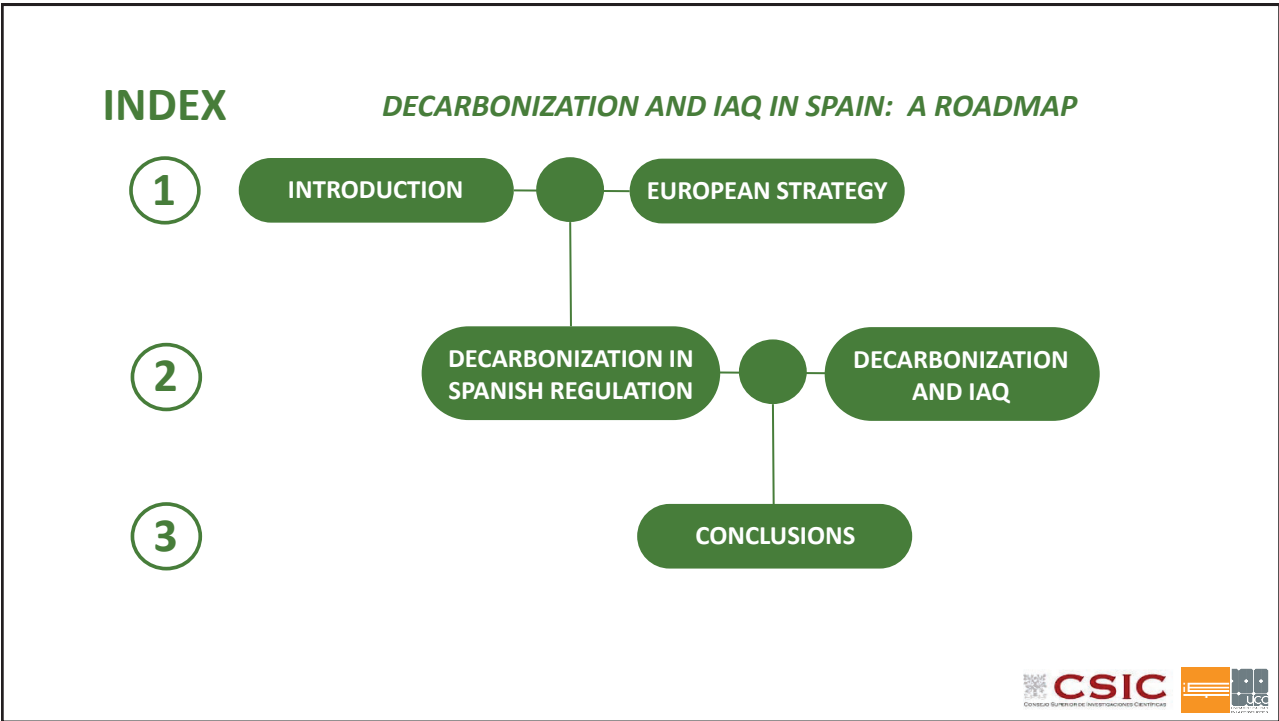
**DECARBONIZATION AND IAQ IN SPAIN:
A ROADMAP**

Marta Sorribes Gil msorribes@ietcc.csic.es
 Rafael Villar Burke pachi@ietcc.csic.es
 Daniel Jiménez González danielj@ietcc.csic.es

CSIC **UCC**
CONSEJO SUPERIOR DE INVESTIGACIONES CIENTÍFICAS CONSEJO SUPERIOR DE INVESTIGACIONES CIENTÍFICAS

**Construction Quality Technical Department (UCC)
Eduardo Torroja Institute for Construction Sciences (IETCC-CSIC)**

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1

INTRODUCTION

THIS IS A CURRENT OVERVIEW OF THE DECARBONIZATION PROCESS UP TO 2050 IN THE EUROPEAN FRAMEWORK, AND ITS TRANSPOSITION INTO SPANISH ENERGY POLICY AND REGULATIONS, WITH A SPECIAL FOCUS ON THOSE ASPECTS RELATED TO BUILDING VENTILATION AND INDOOR AIR QUALITY



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1

EUROPEAN STRATEGY

DECARBONIZATION OF THE EUROPEAN BUILDING STOCK:



NEW BUILDINGS:
ZERO-EMISSION BUILDINGS
BY **2030**



EXISTING BUILDINGS:
ZERO-EMISSION BUILDINGS
BY **2050**

EPBD Article 2:
*A **zero-emission building** is a building with a very high energy efficiency (determined in accordance with Annex 1, which states that it shall be expressed by an indicator of primary energy consumption per unit area per year [kWh/m²]) in which the small amount of energy that is still needed is fully covered by energy from renewable sources generated on-site, from a renewable energy community or from a district heating and cooling system in accordance with the requirements of Annex III*



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DECARBONIZATION IN SPANISH REGULATION

CURRENT REGULATION:

- **DB-HE** (Basic Document on Energy Saving)

$C_{ep,nren}$ CLIMATE ZONE DEPENDING VALUES
 $C_{ep,tot}$ CLIMATE ZONE DEPENDING VALUES

- **EPC** (Energy Performance Certification)

CO_2 emissions in USE PHASE
 $C_{ep,nren}$ CLIMATE ZONE DEPENDING VALUE

- **ERESEE** (Long Term Strategy for Energy Rehabilitation of the building sector in Spain)

NEXT REGULATION:

- **DB-HE** (Basic Document on Energy Saving)

$C_{ep,nren} \approx 0$
 $C_{ep,tot} \lllll$

- **EPC** (Energy Performance Certification)

CO_2 emissions in LIFE CYCLE ASSESMENT
 $C_{ep,tot}$

- **PNRE** (National Plan for Building Rehabilitation)

- RECHARGING OF ELECTRIC VEHICLES
- FOSSIL FUELS BAN BY 2030
- ELECTRIFICATION OF EPB USES AND DECARBONIZATION OF SUPPLY NETWORKS

$k_{exp} = 1$



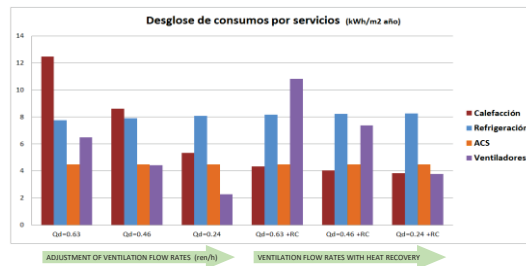
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DECARBONIZATION AND IAQ

VENTILATION AND ENERGY IMPACT

Growing impact of the air exchange with the outside on the building energy performance



FUTURE EVOLUTION

1 ADJUSTMENT OF VENTILATION FLOW RATES

2 IMPROVING THE AIRTIGHTNESS TREATMENT AND THE USE OF HEAT RECOVERY SYSTEMS



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DECARBONIZATION AND IAQ

1

ADJUSTMENT OF VENTILATION FLOW RATES

- DISCREPANCIES IN THE CALCULATION OF VENTILATION FLOW RATES BETWEEN THE ENERGY MODEL AND THE AIR QUALITY MODEL:
 - DIFFERENCES BETWEEN THE VOLUME OF LIVING QUARTERS CONSIDERED
 - USE SCENARIOS:
 - FOR AIR QUALITY: MAXIMUM RATE
 - FOR ENERGY CALCULATION: AVERAGE RATE
- DEMAND-CONTROLLED VENTILATION SYSTEMS:
THEIR USE CAN BE EXPECTED TO PLAY AN IMPORTANT ROLE IN REDUCING ENERGY REQUIREMENTS IN BUILDINGS AND THERE IS A NEED TO CLARIFY HOW THEY SHOULD BE CONSIDERED FOR ENERGY CALCULATION PURPOSES, ESPECIALLY IN LESS COMPLEX BUILDINGS AND WITH SIMPLIFIED TOOLS

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DECARBONIZATION AND IAQ

2

IMPROVING THE AIRTIGHTNESS TREATMENT AND THE USE OF HEAT RECOVERY SYSTEMS

- IS AIRTIGHTNESS OF THERMAL ENVELOPE AN ENERGY REGULATORY STRATEGY IN SPAIN?
 - NOT GENERALLY (MILD CLIMATES)
 - NEED FOR IMPROVEMENT IN SOME CONFLICT POINTS
- FORTHCOMING INCORPORATION IN THE REGULATIONS OF THE NEED TO ADJUST THE PERMEABILITY OF THE BUILDING WHEN USING HEAT RECOVERY UNITS
- OTHER VENTILATION-BASED ENERGY EFFICIENCY STRATEGIES (FREE-COOLING / THERMAL BYPASS)
NEED TO IMPROVE THEIR KNOWLEDGE IN TECHNICIANS AND ENERGY CALCULATION TOOLS
- VENTILATION AND BUILDING RENOVATION:
 - REPLACEMENT OF WINDOWS WITH LOWER AIR PERMEABILITY NEED TO INCREASE AWARENESS OF BEST PRACTICES

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CONCLUSIONS

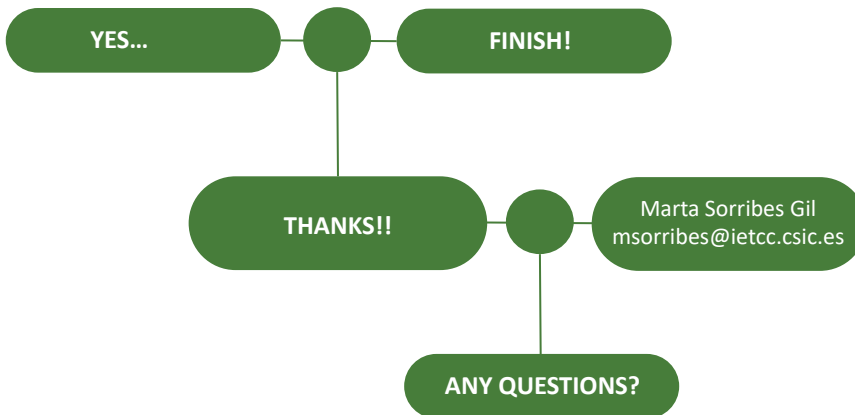
- IN HIGH EFFICIENCY BUILDINGS, CONSUMPTION LINK TO AIR TRANSPORT AND VENTILATION SYSTEMS IS SIGNIFICANT AND COMPARABLE TO THAT OF OTHER EPB SERVICES, AND IS THEREFORE ESSENTIAL TO MAKE PROGRESS ALONG THE FOLLOWING LINES IN ENERGY REGULATIONS:

1 ADJUSTMENT OF THE REQUIRED FLOW RATES

2 IMPROVING THE EFFICIENCY OF EXISTING SYSTEMS AND PROMOTE BEST PRACTICES

3 FACILITATING INCORPORATION OF NEW TECHNOLOGIES WHEN APPROPRIATE

- AIR QUALITY AND VENTILATION TECHNOLOGIES ARE CLOSELY LINKED TO ACHIEVING THE CLIMATE CHANGE MITIGATION OBJETIVE AND MUST BE ADDRESSED IN AN INTEGRATED MANNER, PAYING SPECIAL ATTENTION TO BUILDING RENOVATION



Ventilation behaviour of occupants driven by outdoor temperature: 12 case studies

Copenhagen, October 5th, 2023

Speaker: Sonia García-Ortega

Eduardo Torroja Institute for construction sciences ETcc - CSIC

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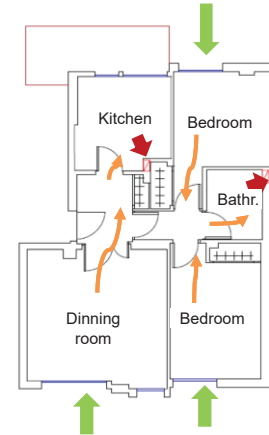
Outline

- Objective
- Current situation
- Dwellings selection and characterization
- Measurements

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Current situation

- The traditional way of ventilating dwellings in Spain was by **natural ventilation** based on the operation of **windows and high levels of infiltration** through the building envelope
- Since the middle of the last century to 2006, the use of **vertical ventilation shafts** in wet rooms of dwellings became widespread and is currently the most common ventilation system in existing dwellings.
- Since 2006, **mechanical or hybrid ventilation** is in force.



3

Objective

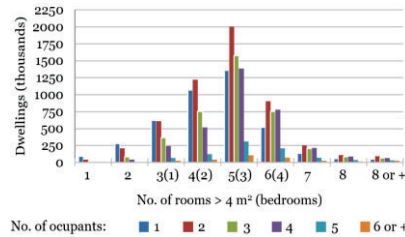
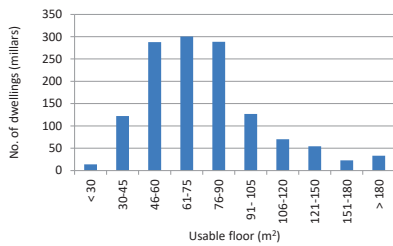
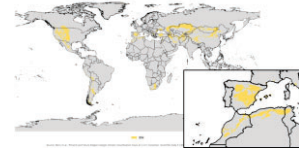
To find relevant parameters affected IAQ in dwellings with natural ventilated systems, focus on the occupant behaviour.



4

Selection of case studies

- Natural ventilation system
- Consolidated urban environment of the city of Madrid (BSk or continental climate)
- Flats in multi-family buildings
- Usable floor area: 30 to 150 m²
- No. of rooms: 1 to 7 (rooms > 4 m², except bathrooms)
- 1 to 4 occupants



5

Case studies characterization

Dwelling code	Year of construction (or main retrofit)	Vertical shafts	Permeability of envelope	Compactness of environment	Compactness of dwelling
01	1986 (2014)	Yes	low	low	medium
02	2000	Yes	high	medium	low
03	1956 (2000)	No	medium	low	higt
04	1960 (2016)	Yes	low	medium	low
05	1987	Yes	high	higt	higt
06	1991	Yes	high	medium	low
07	1970	Yes	medium	medium	higt
08	1963	Yes	medium	medium	low
09	1956	No	medium	higt	medium
10	1982	Yes	high	medium	low
11	1960	Yes	low	medium	medium
12	2011	Yes	Low	higt	medium



6

Monitoring and surveys

75 monitorings (42 rooms belongs to 12 differents flats), 37 in summer and 38 in winter

Period: 8 to 21 days (average of 16 days)



Surveys about occupants actions

Findings - Surveys

Dwelling code	No. of occupants	Occupant's behaviour patterns (general surveys)
01	2	Open windows 5' in the morning
02	4	Open windows in function of outdoor temperature and thermal comfort
03	2	Open windows 5' in the morning; all night in summer
04	4	Open windows for a few minutes in the morning and afternoon. Open the windows a lot in summer
05	2	Open windows 10' - 60' in the morning; all night in summer
06	4	No specific habits
07	2	Open windows in the morning. In leaving room: open window all the day except in winter (in winter only open at night)
08	2	No specific habits
09	1	Open the bathroom window after showering, open other windows when feel it is necessary, usually once a day
10	2	Open windows 10' in the morning; in summer leave a small opening all day, and total open window at night
11	4	Open windows 5-10' in the morning
12	1	No specific habits

60% open windows in the morning 5-10 minutes
50 % open more time in summer or warm weather

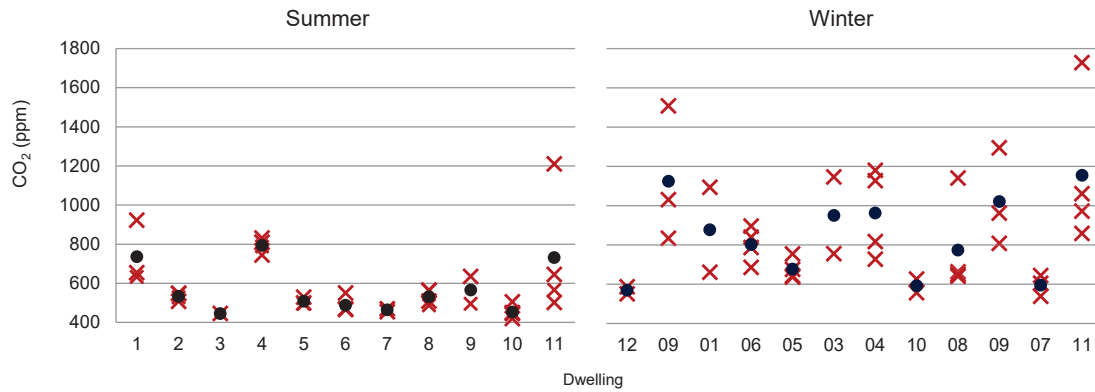
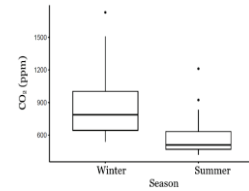
Discrepancies are observed **between** what is expressed in the **surveys** and what is observed during **monitoring**

Monitoring shows:

- **longer ventilation** periods
- in general, **higher ventilation** in warm and summer periods

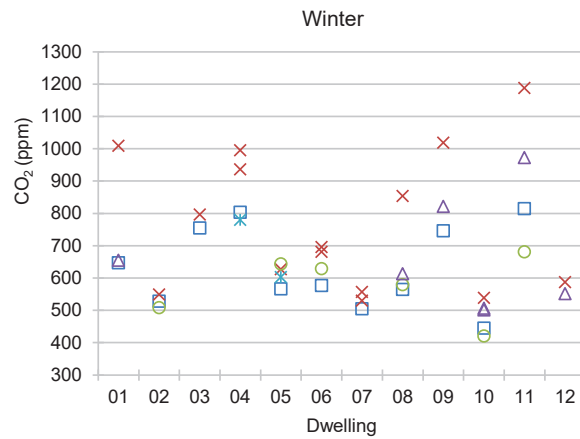
Findings – Season influence

Higher values in winter than in summer



Findings – Season influence

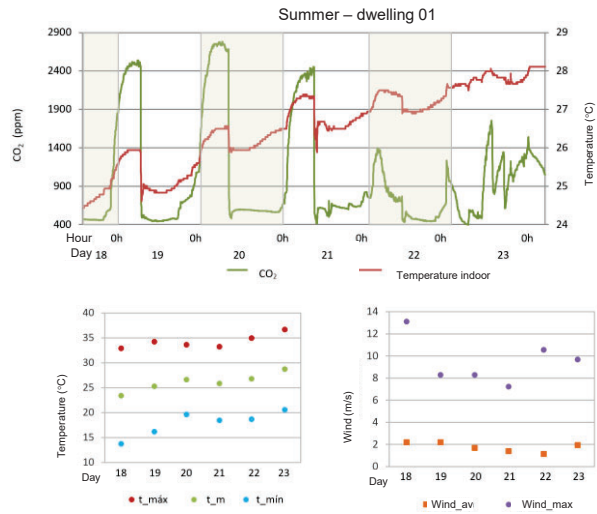
Especially in **bedrooms**



- Living/dining room × Bedroom ○ Kitchen
- △ Bathroom × Other room

Findings - Occupants actions behind season influence

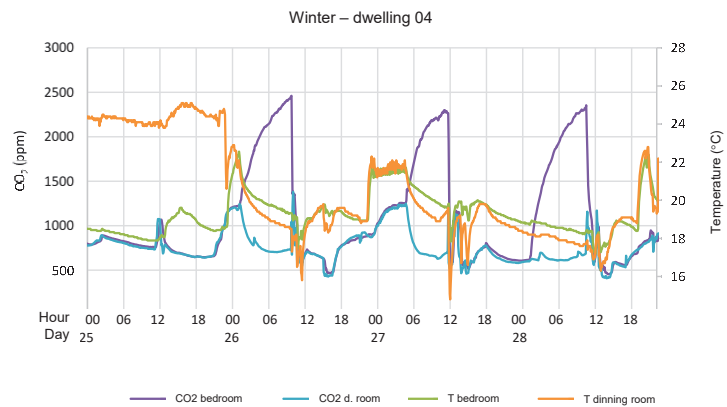
1 - Increasing of ventilation with high outdoor temperature, e.g. opening windows more time



11

Findings - Occupants actions behind season influence

2 - Reduction of volume to diluted concentrations with cold outdoor temperature, e.g. closing indoor doors



Outdoor temperature: minimum: 3°C, medium: 7, maximum: 11

12

Conclusions for the cases studies

- Main driver of occupant behaviour seems to be **thermal comfort rather than IAQ control** itself, at least when expressed as a function of CO₂ concentration
- Occupant behaviour in relation to IAQ could be considered as a **function of outdoor temperature**
- Occupant behaviour is **not easily captured by general surveys**

Future studies

Established **occupant influence** in IAQ based on **outdoor temperature**



Thanks for your attention

Speaker: Garcia_Ortega, Sonia
soniag@ietcc.csic.es

Authors: García-Ortega, Sonia
Linares Alemparte, Pilar

DIGIdat

Indoor air quality in Austrian classrooms

Assessing different ventilation strategies with a citizen science approach

Simon Beck – simon.beck@uibk.ac.at – 5. October 2023

43. AIVC Copenhagen – Topical Session on Indoor Air Quality & Ventilation

Paper Authors: Simon Beck, Gabriel Rojas, Sebastian Goreth, Elena Krois, Christian Hechenberger

Bundesministerium
Bildung, Wissenschaft
und Forschung

oead
Agentur für Bildung
und Internationalisierung

1

DIGIdat

Air Quality in Austrian Schools

15-25
students

55-90 m²
classroom

➔

Fresh air
used up
quickly

➔

Standard window
airing often
insufficient

How is the **situation in existing Austrian schools today?**

What ventilation strategy has what impact?

What role play the **different air pollutants?**

E.g., Austria: 16 out of 18 investigated classroom with **CO₂ concentration too high** (LUKI Studie, Hohenblum et al., 2008)

Slide 2

2

What we did so far...

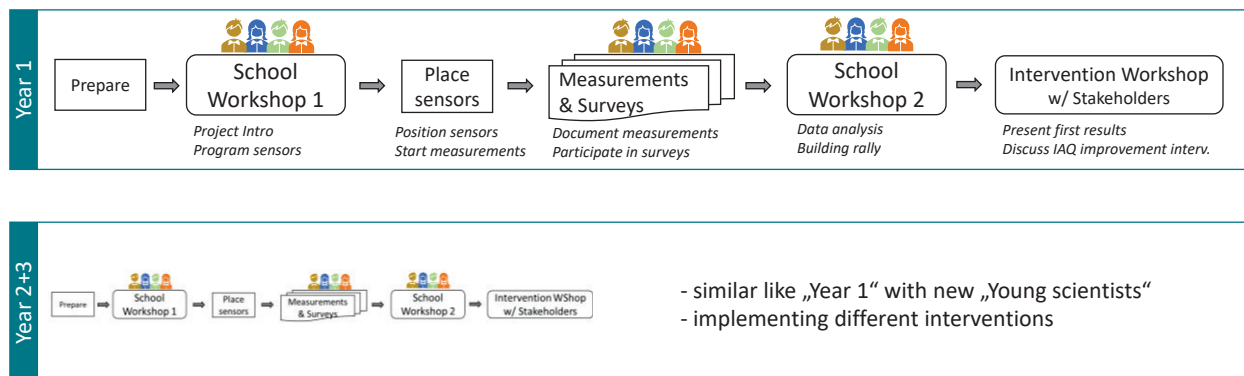
Long term **air quality measurements** in classrooms ...
 ... with help of **children** in schools as **citizen scientists**.

Assessing boundary conditions for the ten participating schools

Data analysis of the measurements in winter/spring
 (preliminary results of **project DIGIdat**)

3

Project concept



4



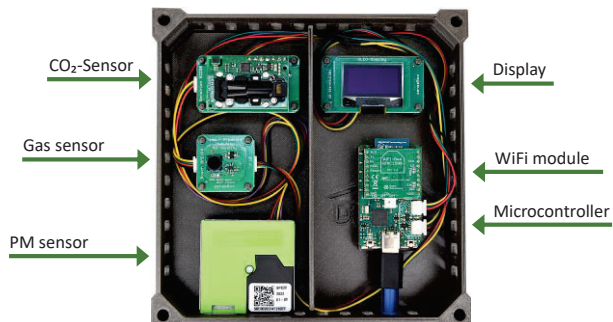
Measurements

5

Low-cost sensor kit



<https://sensebox.de/>



Sensor	Measurement	Method	Range	Accuracy	Source
BME680	b-VOC _{eq} *	Metal Oxide Semiconductor	0.5 – 1000 ppm**	-	(Bosch Sensortec, 2022)
SCD30	CO ₂	Nondispersive Infrared	400 – 10000 ppm	± 30 ppm	(Sensirion, 2020a)
SPS30	PM _{2.5}	Optical (light scattering)	0 – 1000 µg/m ³	± 10 %	(Sensirion, 2020b)

*calculated from correlation of typical VOCs to gas sensor resistance
 **min- and max-output (tested range not available)

Measuring ...

Temperature, Humidity,
 Volatile organic compounds (b-VOC_{eq}),
 Carbon dioxide (CO₂),
 Fine particulate matter (PM₁, PM_{2.5})

6

6

Positioning of sensors

housing 3D-printed or crafted by students

mostly on walls in **height of 1 – 1.5 meters**

in **distance** to doors, windows, sinks and other **sources of direct contamination**

mainly **3 sensors per classroom** -> redundancy and detection of unreliable sensor readings



Results

Measurement period January – July 2023

Citizen Science approach successful



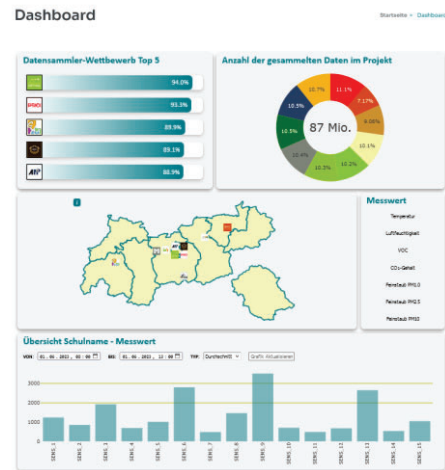
Students are **responsible** for their sensor-kit

Live transmission of measurements:

-> **online database** (openSenseMap.org)

-> DIGIdat **Dashboard** (digidat.at/dashboard/) ->

92.5 %
actually collected /
potentially collectable
(within timeframe)

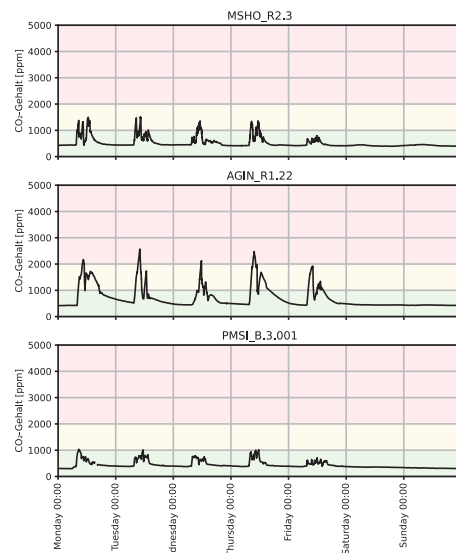
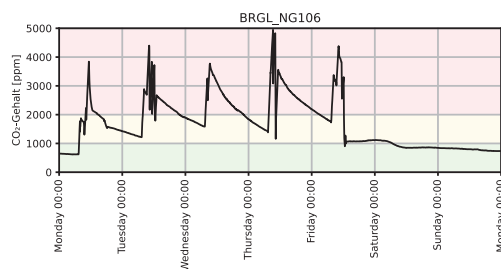


Examples of CO₂ profiles

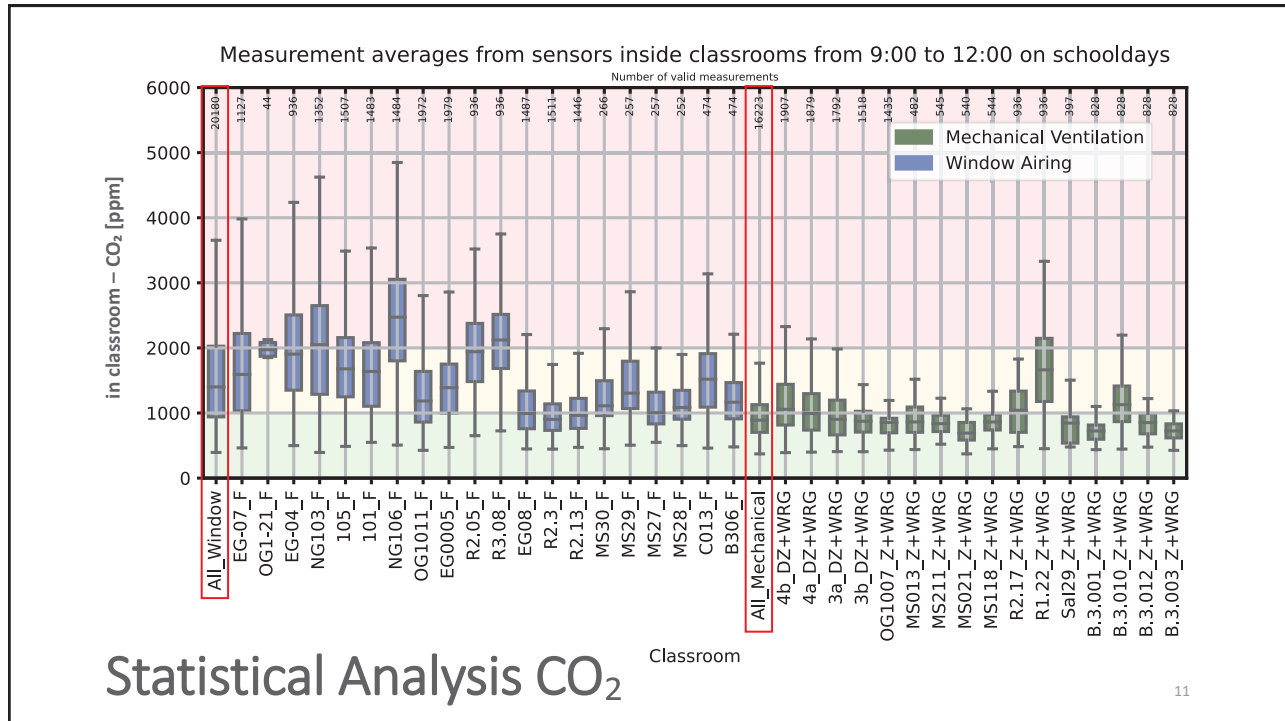
CO₂ as indirect **IAQ** indicator gas

Measuring every **60 seconds**

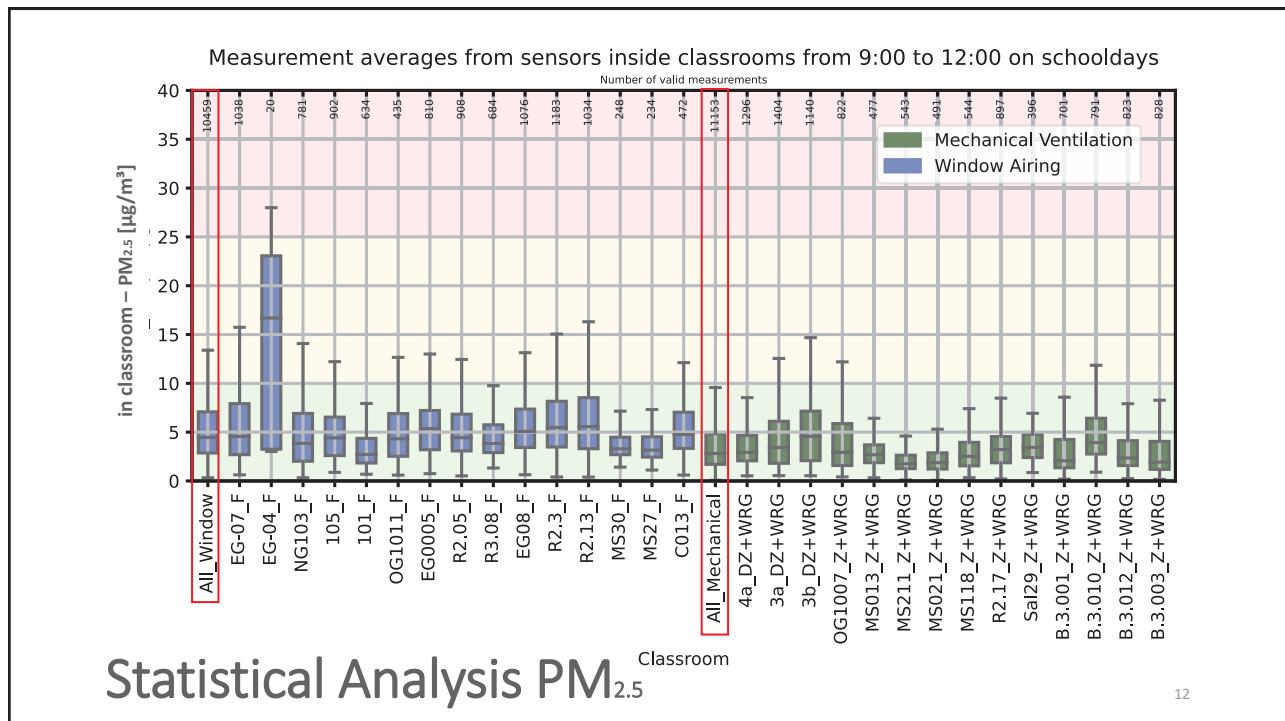
Data handling using Python



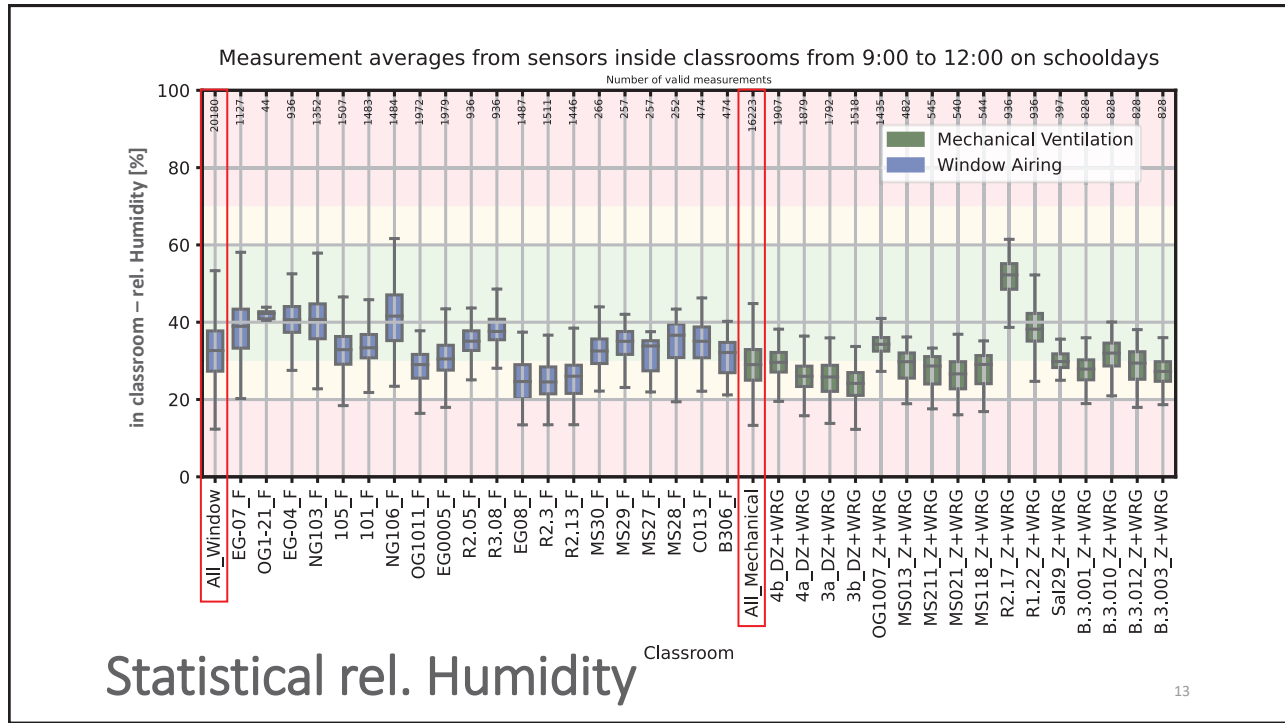
Data from 17.-23. April



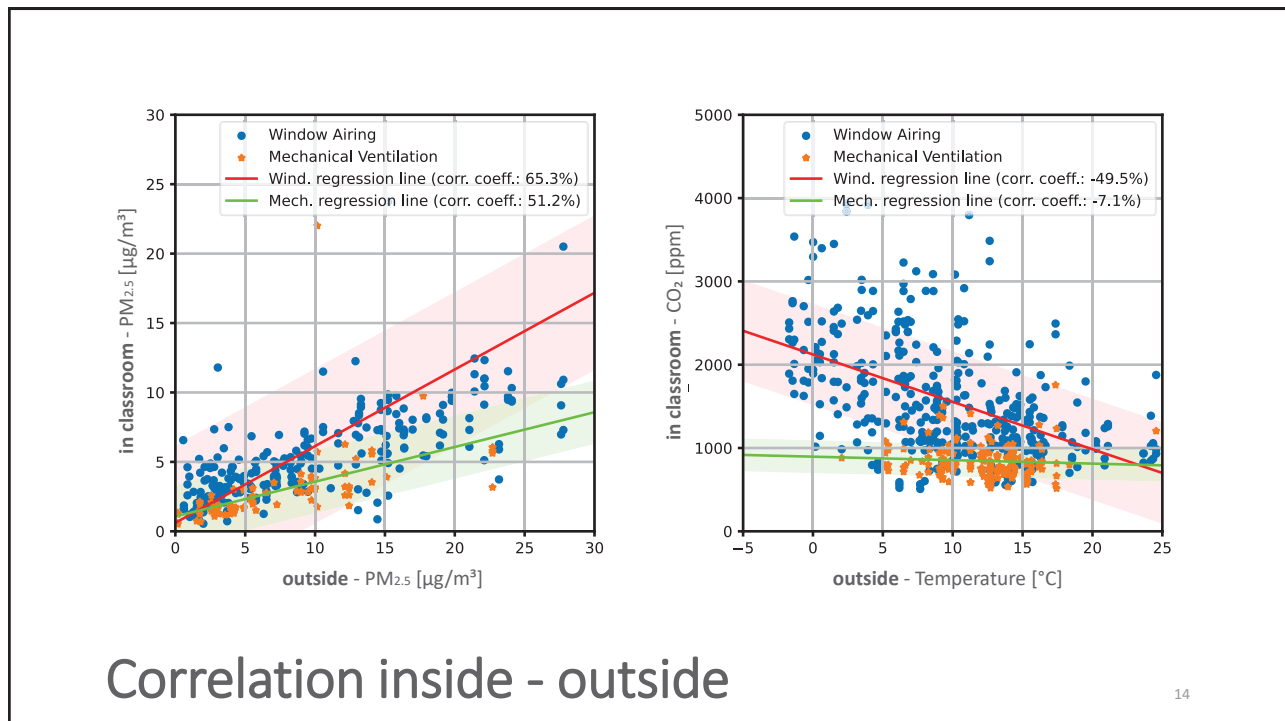
11



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14

Conclusion

36 classrooms measured

Statistical significance found
within spring measurement period

- higher CO₂ concentrations in classrooms **with window airing**
- colder temperature **outside** correlates with **higher CO₂ concentrations in classrooms**
- higher PM_{2.5} concentrations in classrooms **with window airing**

This indicates that **mechanical ventilation** leads to **lower CO₂ and PM_{2.5}** concentrations under **given boundary conditions** (weather and building type)

Table 2: Significance of differences: window airing vs. mechanical ventilation (significance level 5%)

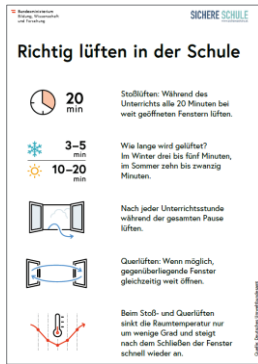
Classroom Avg.	Test-Variant	p-Value Wt*	p-Value MWU**	Result
CO ₂	window airing > mechanical vent	0.0004 %	0.001 %	very significant
PM _{2.5}	window airing > mechanical vent	0.21 %	0.02 %	very significant
b-VOC _{eq}	window airing < mechanical vent	3.74 %	2.51 %	significant
Humidity	window airing > mechanical vent	4.53 %	1.14 %	significant

*Wt ... Welch's t-Test
**MWU ... Mann-Whitney-U-Test



Expanding dataset before and after interventions

Five possible interventions



Raising Awareness



CO₂-Signal



<http://www.coved.tugraz.at/>

Self-made ventilation system



<https://www.rwasysteme.at/-/blog/>

automatic window opener



Source: A. de-Stefani, 2020

Single-Room ventilation system

Problems and upcoming tasks

Problems

- Unsure about **VOC values**, which are calculated by gas sensor
- Temperature readings** off by about 1°C
- Partly heavy **drift of CO₂ sensors**

Work in Progress

- Own **reference measurements** under controlled conditions -> **quantify errors**
- > **generate clean data** from measured offset
- Own **calibration** for next measuring period

Next Steps

- Expanding data set** in coming school years (starting in November)
- Implementing **interventions** and comparing datasets
- Analysis on **influence of boundary conditions** (class size, window orientation, geometry, ...)

Literature and Sources

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Sensirion. (2020b). *Datasheet SPS30: Particulate Matter Sensor for Air Quality Monitoring and Control*. https://sensirion.com/media/documents/8600FF88/616542B5/Sensirion_PM_Sensors_Datasheet_SPS30.pdf

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Measurement of ventilation effectiveness and indoor air quality in toilets at mass gathering events

Ben M. Roberts, Filipa Adzic, E. Abigail Hathway, Christopher Iddon, Benjamin Jones, Malcolm J. Cook, and Liora Malki-Epshtein

Dr Filipa Adzic

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University College London

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Research Background

- Mass-gathering events closed in the UK due to COVID-19 in 2020 & 2021, causing a major economic impact.
- The Events Research Programme was established by the UK Government to safely lift restrictions at events.
- Environmental studies focused on identifying where the risks were in event venues.
- Ventilation and air quality were monitored in 10 venues, with 179 individual spaces over 90 events.
- Monitored spaces within venues included main activity areas, bars, restaurants, private boxes, arrival/departure areas and toilets.
- Opportunity to monitor toilets at events with up to 90 000 attendees!

Methodology

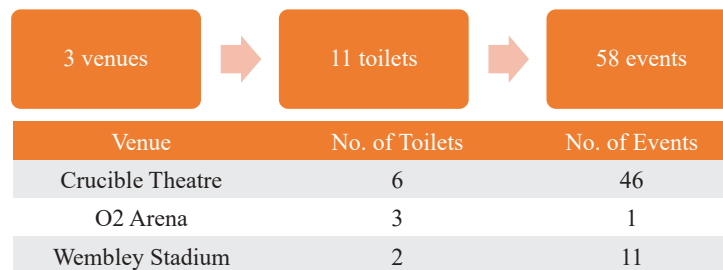
- Introducing sufficient outdoor air into a space is an important measure to reduce the long-range transmission of airborne viruses.
- CO₂ concentration measurements were used as a proxy for ventilation.
- CO₂, temperature and relative humidity were measured with loggers placed on walls at breathing height, away from doors, windows and vents.
- Non-dispersive infrared sensors with a measurement range of 400-5000 ppm and ±30 ppm accuracy were used.
- Occupancy observations were made by researchers on site and confirmed by CCTV imaging and ticket sales.



3

Why are toilets of interest?

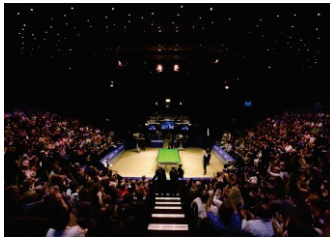
- Toilets are small and densely occupied for brief periods (e.g. half time at a football match).
- Bioaerosols/virus-laden aerosols are dispersed by toilet flushing.
- UK building regulations (Approved Document F) recommend 6 l/s extraction rate per WC pan or urinal.
- All toilets monitored were mechanically ventilated.



4

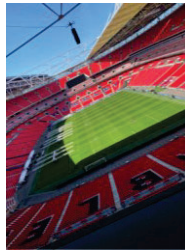
Monitored Venues

Crucible Theatre Sheffield World Snooker Championship



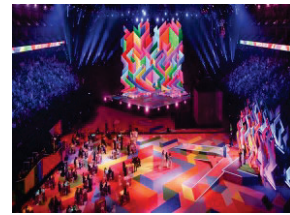
8-88% occupancy, 46 events;
6 toilets, 3 unisex & 3 male.

Wembley Stadium Football Games (EURO 2020)



3-100% occupancy, 11 events;
2 toilets, 1 female & 1 male.

O2 Arena, London Music Awards Ceremony



18% occupancy, 1 event;
3 toilets, 1 female & 2 male.

5

Air Quality Classification

Classification bands were developed during the Events Research Programme for rapid assessment for risk of transmission based on average and maximum CO₂ values recorded during events.

Space classified as band A has high ventilation for monitored occupancy.

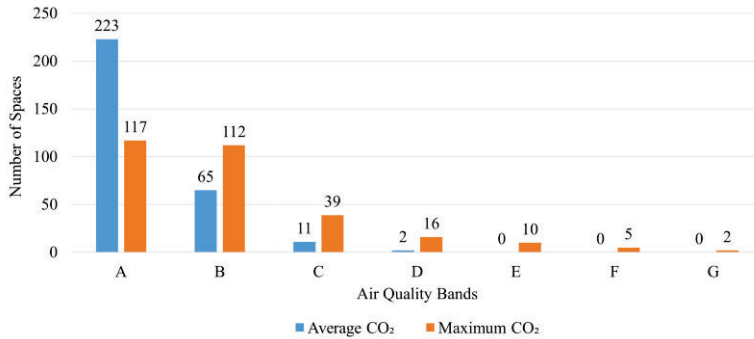
Spaces in bands F & G present with low ventilation and are therefore a priority for improvement.

Malki-Epshtein L, Adzic F, Roberts BM, et al. Measurement and rapid assessment of indoor air quality at mass gathering events to assess ventilation performance and reduce aerosol transmission of SARS-CoV-2. Building Services Engineering Research and Technology. 2023;44(2):113-133.

Air Quality Bands	Class	Range of CO ₂ – Absolute Values (ppm)
At or marginally above outdoor levels	A	400-600
Target for enhanced aerosol generation (singing, aerobic activity)	B	600-800
High air quality design standards for offices	C	800-1000
Medium air quality	D	1000-1200
Design standards for schools pre-Covid	E	1200-1500
Priority for improvement	F	1500-2000
Low ventilation/dense occupancy. Must be improved	G	>2000

6

Overall Results

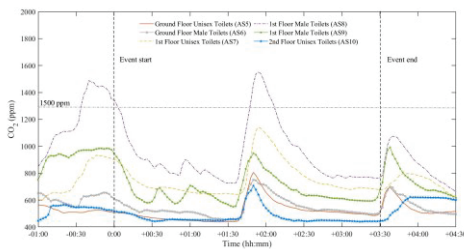


- The spatiotemporal CO₂ average of each toilet over events shows 96% of toilets are A or B. The average includes both low and high occupancy periods.
- Maximum CO₂, observed during high occupancy, shows 76% of toilets are in A or B bands, but 5% of toilets are in E, F and G bands.

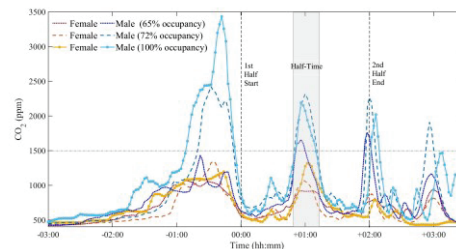
7

Impact of Event Management

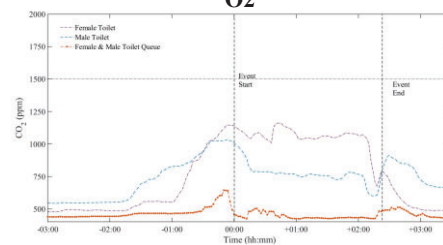
Crucible



Wembley



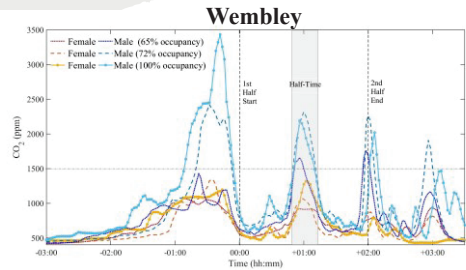
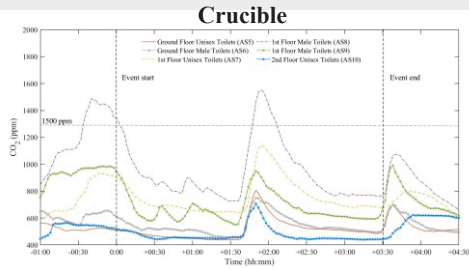
O2



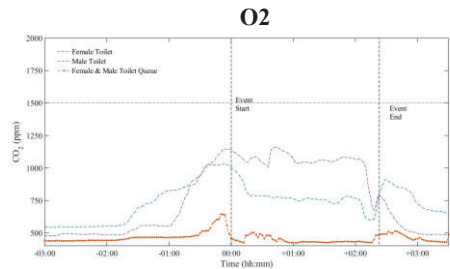
- Three distinct peaks in CO₂ before the event, at the interval and the end of the event noted in Wembley and Crucible events.
- The televised music awards ceremony in the O2 Arena did not have one interval but a series of 15-minute “advert breaks”.

8

Impact of Demographics



- Crucible male toilet on the 1st floor presented with higher values than unisex toilets on the same floor.
- Wembley male designated toilets at full occupancy had a maximum concentration of 3431 ppm, compared to 1320 ppm in female toilets.
- In the O2 arena, female toilets presented with higher CO₂ concentrations than male toilets.

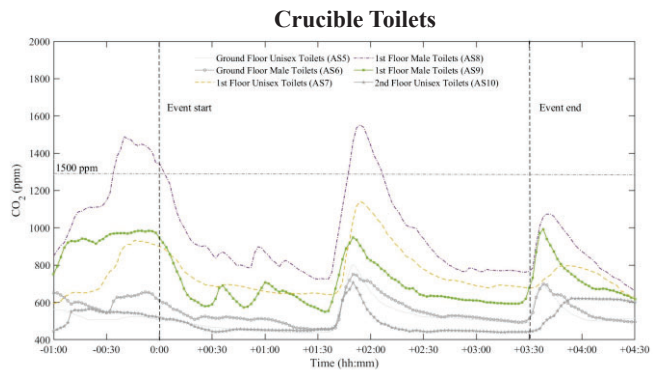


9

Location of Toilets

Toilets on the first floor of Crucible theatre are closer to the main activity area (auditorium) are more frequently visited.

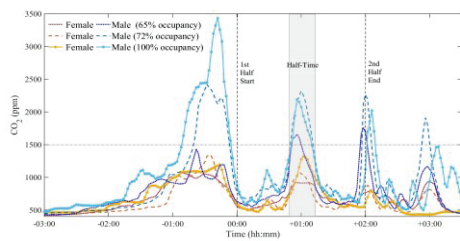
Concentrations in the 1st floor toilets don't fall below 600 ppm because extractor fans draw makeup air from adjacent occupied areas.



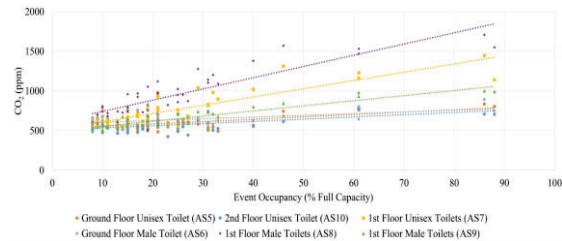
10

Effect of occupancy

Wembley



Crucible



Wembley: CO₂ concentrations continuously over 1500 ppm in male toilets for 44 minutes pre-event, 15 minutes at interval and 8 minutes post-event (full occupancy). Similar levels were observed for 72% occupancy event.

Crucible: Increasing maximum CO₂ concentrations are related to increasing occupancy in the Crucible events, especially toilets close to the main activity area which are more frequently used.

11

Conclusions & Recommendations

- Insufficient ventilation for the occupancy can facilitate airborne transmission of disease.
- Average CO₂ concentrations in most toilets indicated ventilation was sufficient relative to occupancy.
- Maximum CO₂ concentrations revealed that a small number of toilets were prone to poor ventilation.
- Poor ventilation periods usually occur pre or post event and during event intervals.
- The proximity of toilets to the main-activity area has an impact on the occupancy.

Key Recommendations

- Increase ventilation rates or the room volume in toilets most frequently visited.
- Building operators should be mindful of audience demographics and consider changing toilet gender allocation.
- An increased number of event intervals or their length could spread the occupancy of toilets over a longer period.
- Design of ventilation systems in toilets should take occupancy patterns into consideration.

Further development of this work includes ventilation assessment of toilets and toilet queues, as well as microbiological data sampled during events. Existing risk models will be applied to estimate the risk of long-range transmission in toilets at mass-gathering events.

12



Impact of the building airtightness and natural driving forces on the operation of an exhaust ventilation system in social housing in Chile

43rd AIVC -11th TightVent & 9th venticool Conference
October 4-5, 2023

Gilles Flamant^{1,2}, Waldo Bustamante¹, Arnold Janssens², Jelle Laverge²

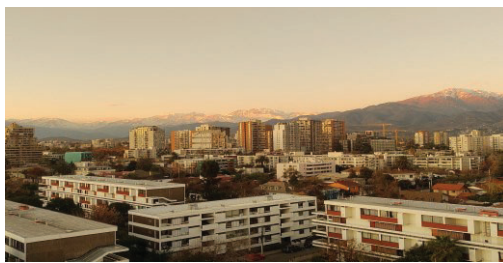
¹ Centre for Sustainable Urban Development (CEDEUS), Pontificia Universidad Católica de Chile, Chile

² Research Group Building Physics, Ghent University, Belgium

1

Context

Residential buildings stock



	CENSO 1992	CENSO 2002	CENSO 2017
Total houses	3369849	4399952	6499355
Single-family houses	3359639	4380822	6486533
Multi-family houses	10210	19130	12822

Less than 2% of homes built meet minimum thermal performance standards.

(Source OECD/LEED 2014)

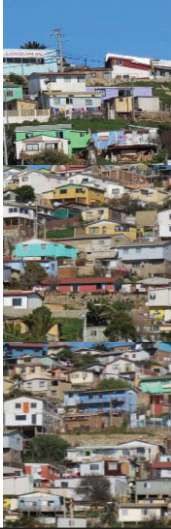
66% have thermal comfort problems. (RedPE, 2019)

2

2

Context

Social housing



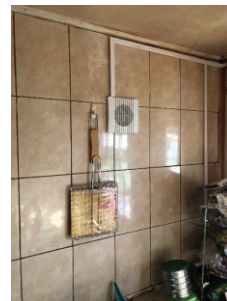
3

3

Context

Retrofitting programs for social housing

Ventilation system



4

4

Research objectives

Evaluate if such mechanical exhaust ventilation system allows ensuring an acceptable IAQ in social housing.

Propose, evaluate and optimize ventilation system and strategies,

- based on a multi-criteria approach: indoor air quality, energy consumption and total costs over the service life of the building
- considering the climatic conditions and outdoor air pollution levels specific to different areas of the country

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Objective of the present study

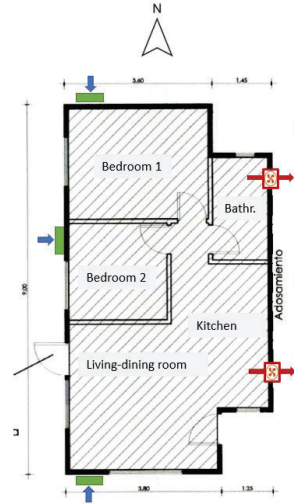
Evaluate by simulation the performances of a mechanical exhaust ventilation system,

- in continuous operation
 - in a representative social housing unit in Chile
- Effects of the airtightness level of the building envelope and natural driving forces on the working of the ventilation system and the IAQ
 - Using the CONTAM program
 - Different outdoor climate data sets

6

6

Case study description



7

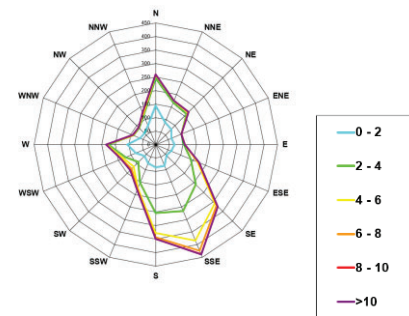
7

Climatic data



Figura 1.6.2.1.1: "Mapa de zonificación climática habitacional de la NCh 1079 - 2008"

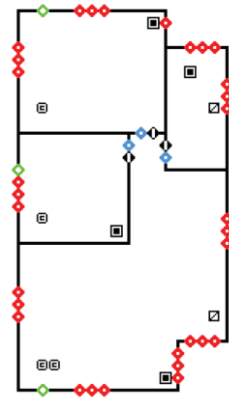
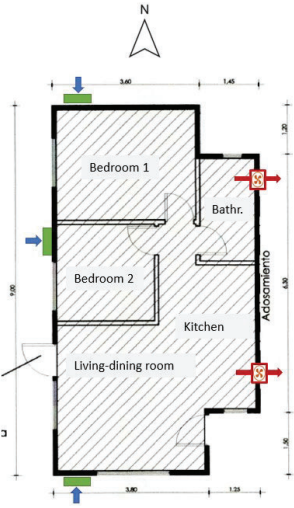
- NORTE LITORAL
- NORTE DESÉRTICO
- VALLES TRANSVERSALES
- CENTRO LITORAL
- CENTRO INTERIOR
- SUR LITORAL
- SUR INTERIOR
- SUR EXTREMO
- ZONA ANDINA



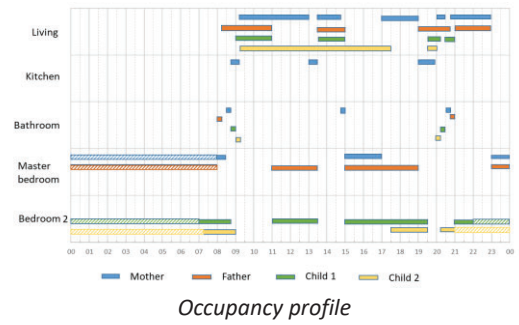
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8

Model and assumptions



3 levels of n_{50} : 10 h^{-1} , 5 h^{-1} , 0 h^{-1}

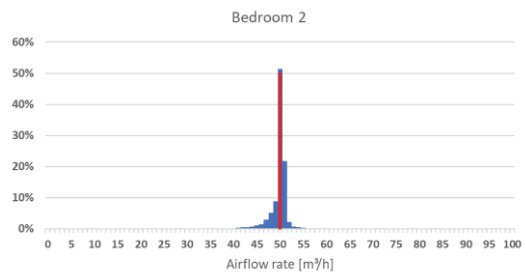
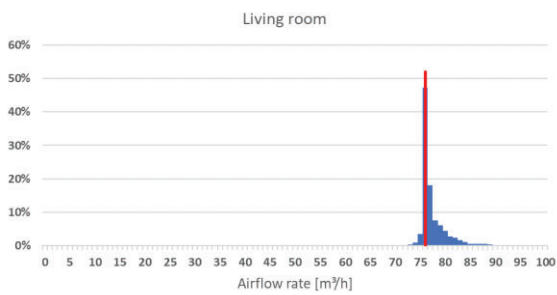
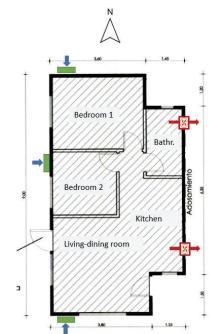


Results

Airflow rates

Perfectly airtight house ($n_{50} = 0 \text{ h}^{-1}$) (ideal case)

With Open interior doors

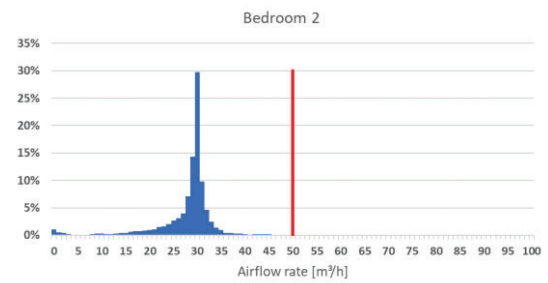
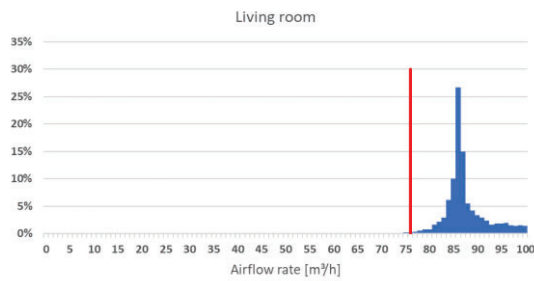
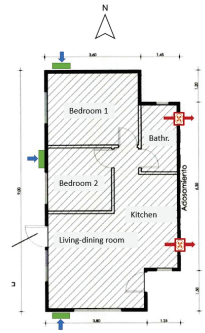


Results

Airflow rates

Leaky house ($n_{50} = 10 \text{ h}^{-1}$)

With Open interior doors



11

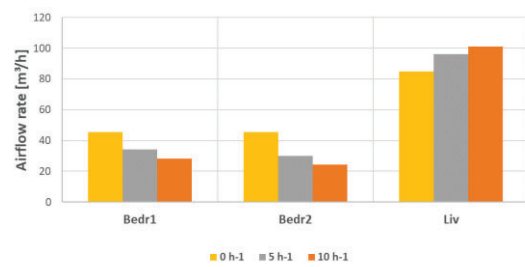
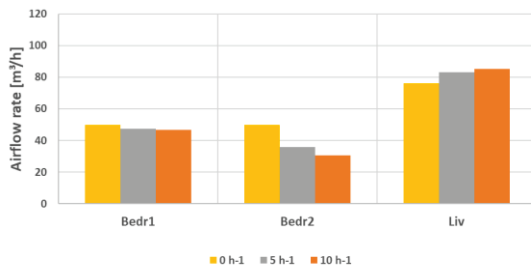
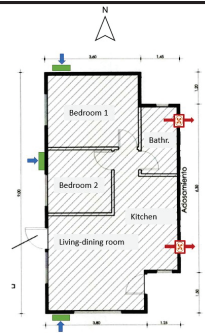
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Results

Airflow rates

With Open interior doors

With Closed interior doors

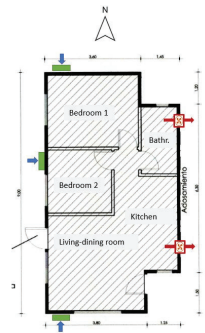
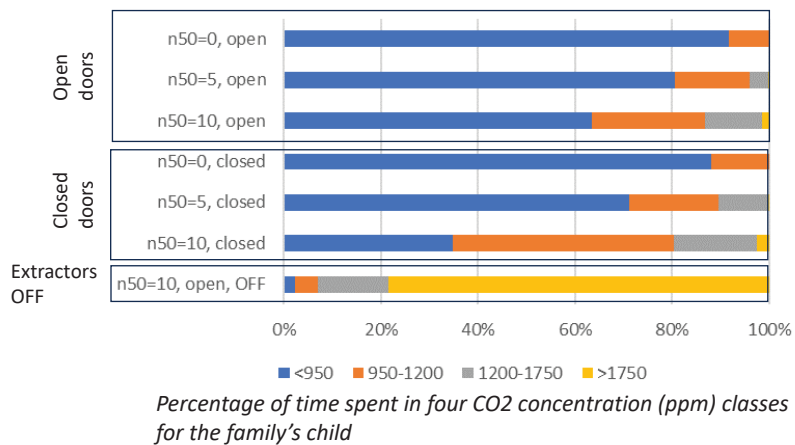


12

12

Results

CO₂-based indicator



13

13

Conclusions

- Significant effect of the building airtightness and natural driving forces on the performance of a mechanical exhaust ventilation system.
 - Average airflow rate is drastically reduced to almost the half of the design value in one of the bedroom.
- Need for improving the level of airtightness of social houses in Chile
- Need for improving the working of the ventilation system : choice, sizing, control strategy (DCV), ...

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Future work

1. Evaluation of the mech. exhaust ventil. system with a discontinuous working, for several climates and several typologies of social housing units in Chile
2. Propose, evaluate and optimize ventilation systems and strategies
 - Criteria : IAQ, Energy, Costs
 - Several climates & air pollution levels



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Thank you very much for your attention !

ACKNOWLEDGEMENTS

This work was funded by the National Research and Development Agency (ANID), Chile, under research grant FONDECYT 1221666. The authors also gratefully acknowledge the research support provided by CEDEUS, ANID FONDAP 1522A0002. The paper was also developed as part of the Ph.D. program at Ghent University, Belgium.



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Metal Oxide Semiconductor sensors (MOS) for measuring Volatile Organic Compounds (VOC) – performance evaluation in residential settings

5.10.2023

Jakub Kolarik
Technical University of Denmark

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1

Introduction - Smart ventilation for residencies ?!



We need suitable sensors!

- Low price
- Low energy use
- Integration with IoT
- Reliable
- Stable over time
- ...

Metal Oxide Semiconductor (MOS) sensors

Figure 1: Based on "Main features of smart ventilation" (Durier et al. 2018)

2

Background - Why MOS VOC sensors?

- Application of MOS VOC sensors seems to be an obvious step towards smart ventilation.
- They offer possibility to not only account for pollution related to occupancy, like CO₂ sensors, but also for diverse odorous events taking place in a space.
- Moreover the MOS technology allows producing sensor units at low price.
- Other advantages claimed by producers include small energy consumption and size.
- This not only makes advanced control achivable, but also allows for use of larger amount of sensors – IoT applications.

3

Background – Challenges, barriers, disadvantages...

- MOS VOC sensors are non-selective = they react to many pollutants!
- MOS VOC sensors provide relative measurement and “non-selectivity” makes calibration difficult.
- Interpretation of measured signal as so called CO₂ equivalent^(*).
- They are cross-sensitive to water vapour/humidity.

4

Objectives

- Examine MOS VOC sensors during operation in realistic residential environment.
- Determine their properties – sensitivity, linearity, hysteresis by comparing their signal with a reference measurement.
- Discuss their suitability for control of residential ventilation.

5

Methods

- Three sensors from different established manufacturers.
- Two specimen of each sensor, integrated on Arduino board with Wi-Fi module.
- Portable photo-ionization (PID) gas detector as a reference measurement (TVOC isobuthylen equivalent).
- Standard IEQ monitoring – temperature, relative humidity, CO₂ concentration.
- Total measurement period of almost one year.

The test site

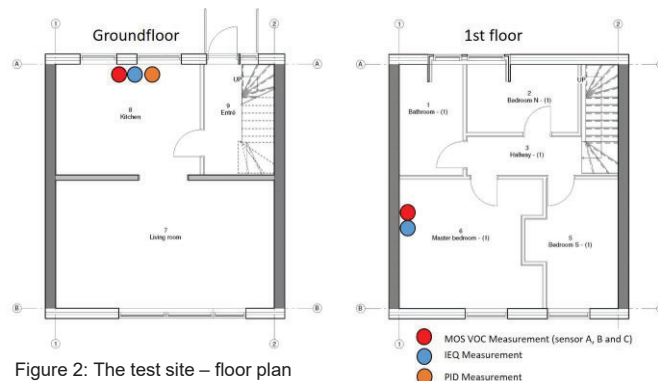


Figure 2: The test site – floor plan

6

Results – Sensor signals

Example for two weeks in October 2021 (the house was unoccupied during the first week) - MOS VOC sensors placed in the kitchen.

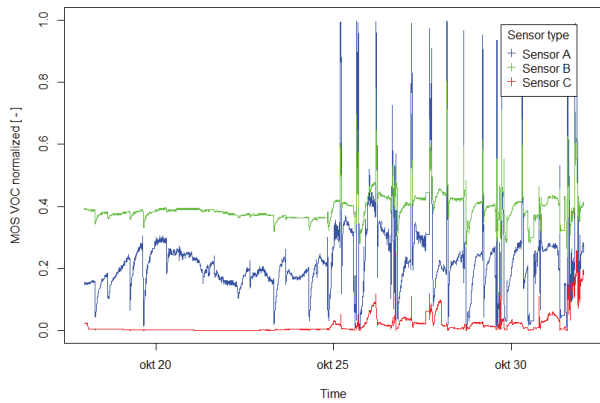


Figure 3: Normalized signals

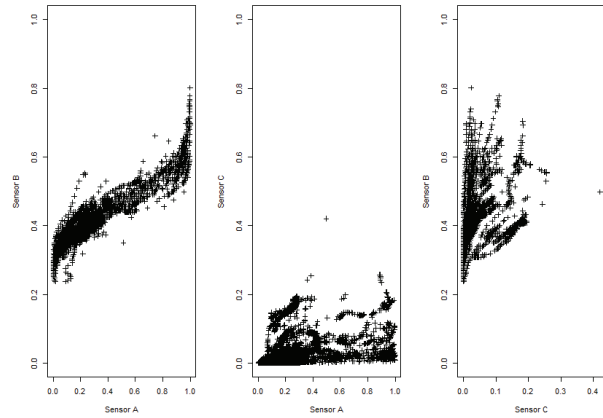


Figure 4: Cross-plot of normalized signals

7

Results – Sensor characteristics

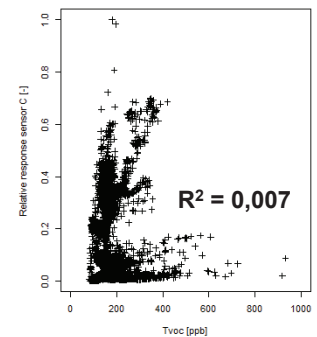
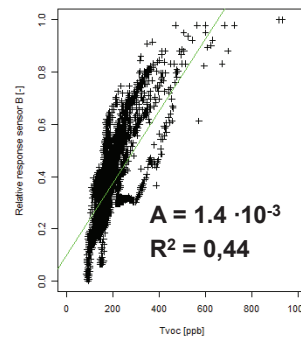
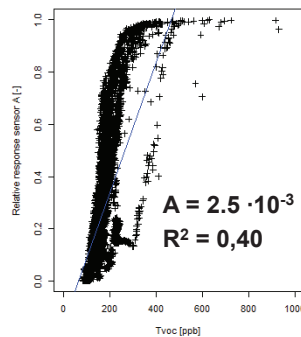
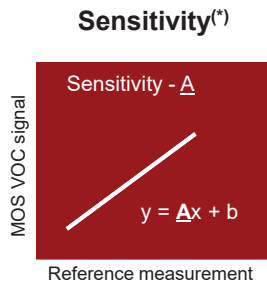
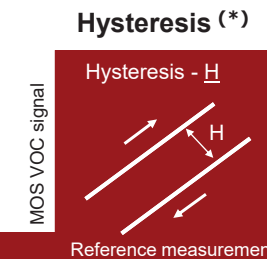


Figure 5: Characteristic curves for tested MOS VOC sensors (PID monitor as reference); period 7.3.2022 – 12.3.2022



- Low linearity of sensor C during longer measurements, comparable to A and B for one-day. Needs more investigation.

- Low hysteresis of 0.123, 0.014 and 0.121 for sensors A, B and C respectively.

(*)Fahlen et al. (1996)

8

Results – CO₂ versus CO₂ equivalent signals

- Close agreement between CO₂ and CO₂eq in the bedroom.
- In the kitchen, CO₂eq substantially drifts from the CO₂ signal.
- Challenge with establishing a set-up to avoid “over ventilation”.

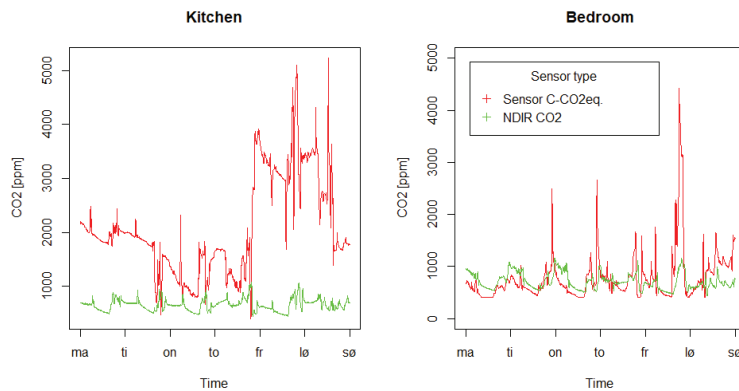


Figure 6: Comparison of CO₂ and CO₂ equivalent signal for measurements in the kitchen and bedroom

Conclusions

- All three tested sensors were able to indicate the pollution events.
- Two of the sensors had comparable behaviour in terms their sensitivity determined using reference PID measurements.
- Cross-relation between their output signals indicates that both sensors would behave in similar manner when used for IAQ control.
- The difference in absolute values of their sensitivity and thus in the amplitude of their response, needs to be taken into account in the case of their use of control.
- The investigated sensors had small hysteresis. The analysis was however conducted on relatively small sample of measurements. Analysis of broader range of build-up and decay periods is needed to confirm the results.

Conclusions

- The CO₂ equivalent signal corresponded to pure CO₂ measurements in the case of measurements in the bedroom.
- In the kitchen where the human bioeffluents were not the main pollution source, there were large discrepancies.
- This is not necessarily a problem for the ventilation control, but careful choice of the set point would be needed to avoid unnecessary ventilation.
- In general the results of the study indicate, that the MOS VOC sensors represent a considerable alternative to currently used sensors. This however requires that their characteristics are properly considered in control algorithms.

References:

- Burdack-Freitag A, Rampf R, Mayer F, Breuer K (2009) Identification of anthropogenic volatile organic compounds correlating with bad indoor air quality. In: Proceedings of the 9th International Conference and Exhibition Healthy Buildings 2009, Syracuse, NY
- Durier, F. Carrié, R., Sherman, M. (2018) What is smart ventilation? Ventilation Information Paper n°38, Air Infiltration and Ventilation Centre, INIVE EEIG, Brussels Belgium
- Fahlen P, Andersson H, Ruud S (1992) Sensor Tests, Demand Control Ventilation Systems, SP Report ISBN 91-7848-331-331-X, Swedish National Testing and Research Institute, Boras, Sweden
- Herberger S, Herold M, Ulmer H, Burdack-Freitag A, Mayer F (2010) Detection of human effluents by a MOS gas sensor in correlation to VOC quantification by GC/MS. Building and Environment, 45, 2430-2439

Acknowledgements:

Funding by Bjarne Saxhofs Fond, Denmark is greatly acknowledged.

TOWARDS PERFORMANCE-BASED APPROACHES FOR SMART RESIDENTIAL VENTILATION

A ROBUST METHODOLOGY FOR RANKING THE SYSTEMS AND DECISION-MAKING

33rd AIVC -11th TightVent & 9th venticool Conference
October 4-5, 2023
Aalborg University, Copenhagen, Denmark

Dr. Baptiste Poirier
Dr. Gaëlle Guyot
Pr. Monika Woloszyn

1

INTRODUCTION

ENERGY, INDOOR AIR QUALITY AND VENTILATION

28% OF THE TOTAL FINAL ENERGY CONSUMPTION IN THE EUROPEAN UNION
(Directorate-General for Energy (European Commission), 2022)

60% TO 90% TIME SPEND INSIDE A BUILDING
For an average European

How to aggregate performance indicators and balance IAQ and energy performance assessment to provide a robust ranking of the ventilation systems?

IMPROVEMENTS OF THE AIRTIGHTNESS AND INSULATION OF THE BUILDINGS to reduce heat losses

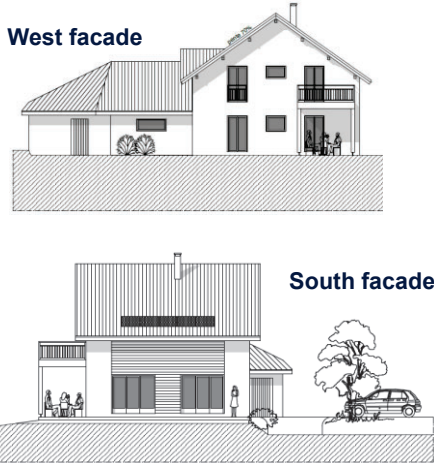
VENTILATION

RENEWAL OF INDOOR AIR source of heat losses and of energy consumption

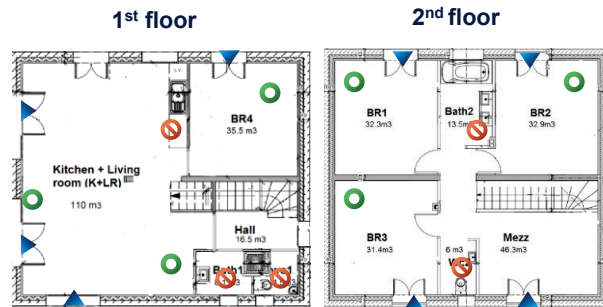
2

INTRODUCTION

A FRENCH LOW ENERGY HOUSE CASE STUDY



Five occupants



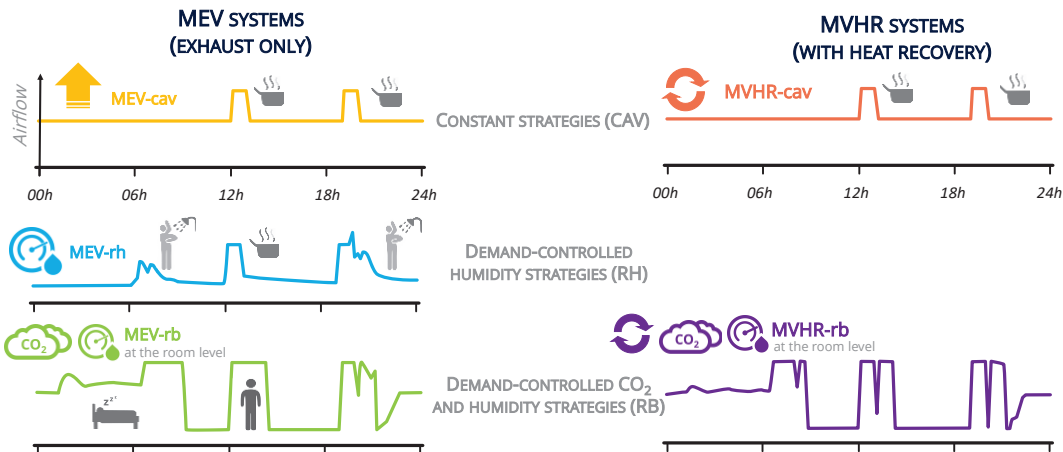
⊘ Exhaust ⊙ Supply ▲ Air-inlets
If Exhaust only ventilation



TOWARDS PERFORMANCE-BASED APPROACHES FOR SMART RESIDENTIAL VENTILATION
A ROBUST METHODOLOGY FOR RANKING THE SYSTEMS AND DECISION-MAKING

INTRODUCTION

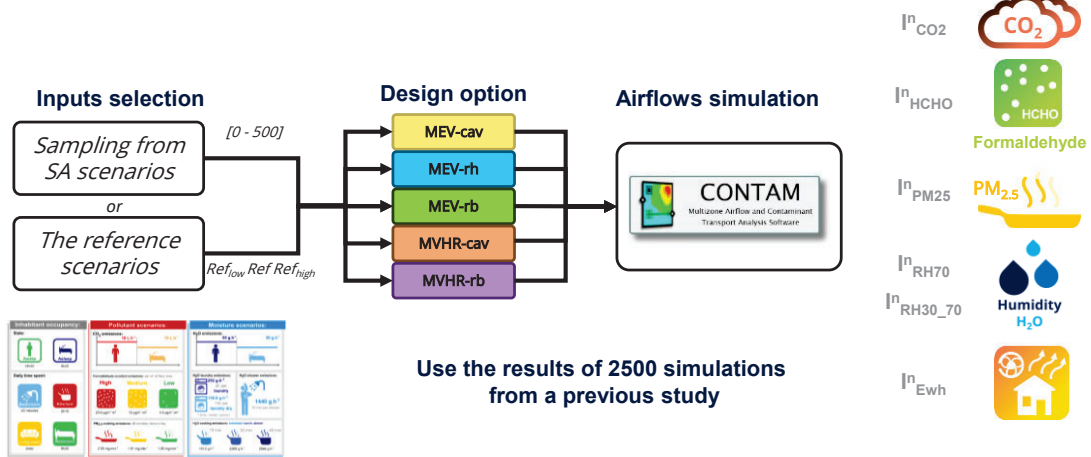
FROM CONSTANT VENTILATION TO SMART VENTILATION



TOWARDS PERFORMANCE-BASED APPROACHES FOR SMART RESIDENTIAL VENTILATION
A ROBUST METHODOLOGY FOR RANKING THE SYSTEMS AND DECISION-MAKING

INTRODUCTION

DESIGN OPTION PERFORMANCE CALCULATION

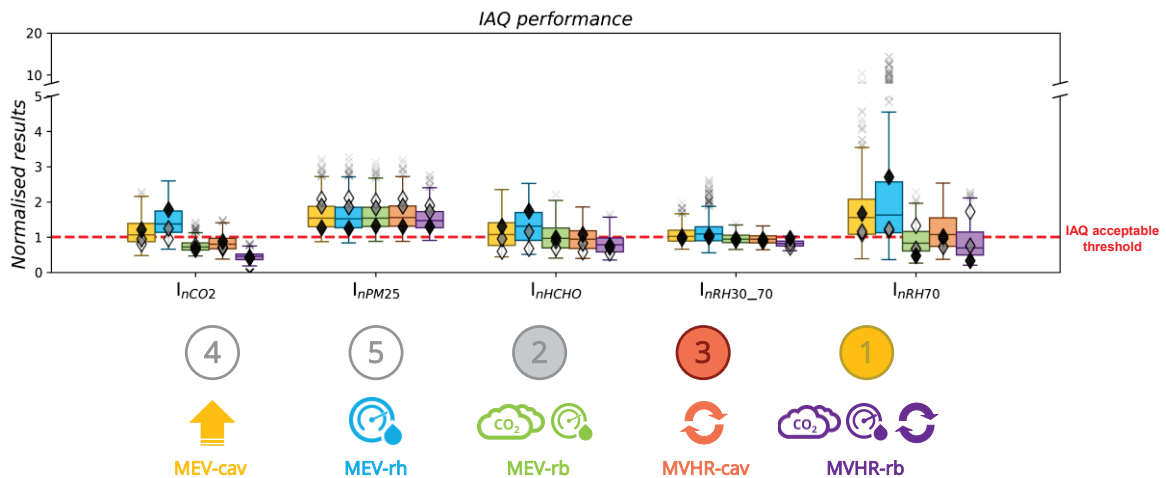


(Poirier et al., 2021b; Poirier, 2023)

5

PERFORMANCE RAKING

BASED ON IAQ PERFORMANCE ?



6

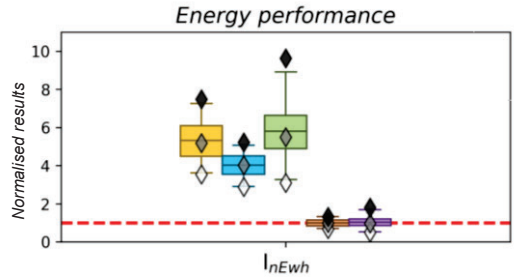
PERFORMANCE RAKING

BASED ON ENERGY PERFORMANCE ?



Heat losses from exhausted airflows

$$H_{th} = \frac{C_{p,m}}{3600} \cdot (1 - \epsilon_{heat_{ex}}) \int q_m(t) \cdot [T_{in}(t) - T_{ex}(t)] \cdot dt$$



MVHR_cav median performance proposed as reference threshold

5



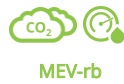
MEV-cav

3



MEV-rh

4



MEV-rb

1



MVHR-cav

2



MVHR-rb

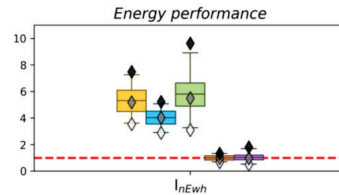
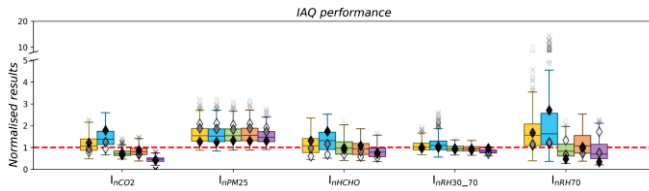


TOWARDS PERFORMANCE-BASED APPROACHES FOR SMART RESIDENTIAL VENTILATION
A ROBUST METHODOLOGY FOR RANKING THE SYSTEMS AND DECISION-MAKING

PERFORMANCE RAKING

BASED ON IAQ AND ENERGY PERFORMANCE ?

How to choose the most relevant one from global performance point-of-view ?



?



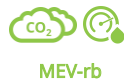
MEV-cav

?



MEV-rh

?



MEV-rb

?



MVHR-cav

?



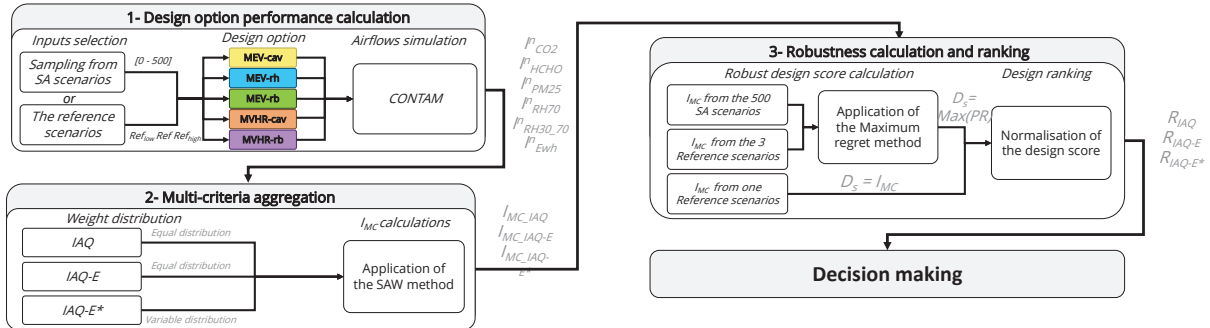
MVHR-rb



TOWARDS PERFORMANCE-BASED APPROACHES FOR SMART RESIDENTIAL VENTILATION
A ROBUST METHODOLOGY FOR RANKING THE SYSTEMS AND DECISION-MAKING

A ROBUST METHOD FOR PERFORMANCE RANKING

A SIMPLIFIED APPROACH IN 3 KEYS STEPS

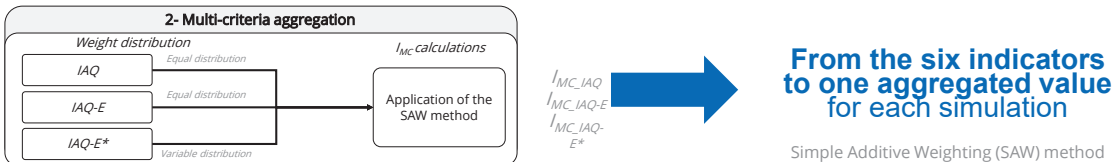


BASED ON EXISTING ROBUST ASSESSMENT METHODS ADAPTED TO BUILDING SECTOR

(Kotireddy et al., 2018; Velasquez and Hester, 2013; Hoes et al., 2009; Sharma and Bhattacharya, n.d.)

A ROBUST METHOD FOR PERFORMANCE RANKING

MULTI-CRITERIA AGGREGATION



From the six indicators
to one aggregated value
for each simulation

Simple Additive Weighting (SAW) method

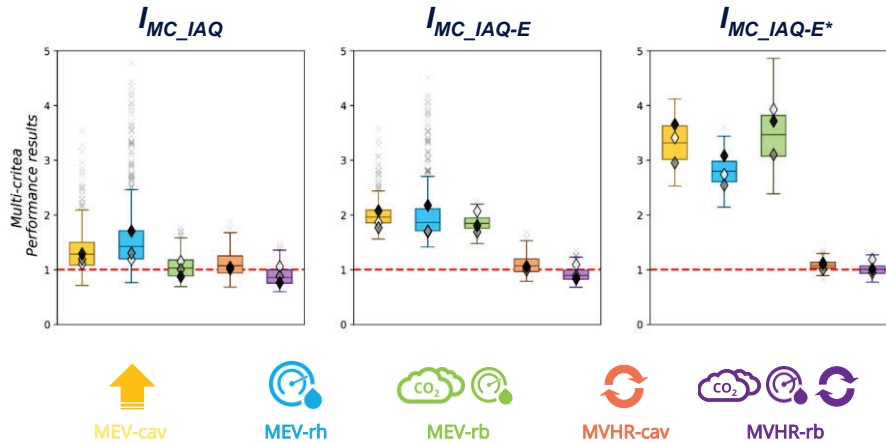
Distribution	Weight ω_i					
	$I_{n_{CO2}}$	$I_{n_{RH70}}$	$I_{n_{RH30_70}}$	$I_{n_{PM25}}$	$I_{n_{HCHO}}$	$I_{n_{Ewh}}$
I_{MC_IAQ}	0.2	0.2	0.2	0.2	0.2	0
I_{MC_IAQ-E}	0.16	0.16	0.16	0.16	0.16	0.16
$I_{MC_IAQ-E^*}$	0.071	0.071	0.071	0.143	0.143	0.5

$$I_{MC} = \sum_i \omega_i \cdot I_i$$

(Podvezko, 2011)

A ROBUST METHOD FOR PERFORMANCE RANKING

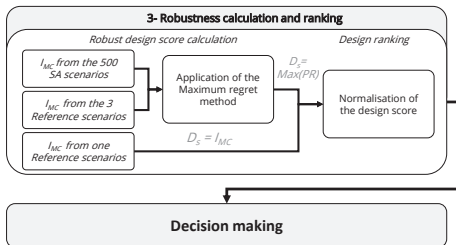
MULTI-CRITERIA AGGREGATION



11

A ROBUST METHOD FOR PERFORMANCE RANKING

ROBUST DESIGN SCORE CALCULATION AND RANKING



Integrating into one design score (D_s) all the individual performance indicators I_{MC} across the tested scenarios.

The minimax regret method

(Kotireddy et al., 2019)

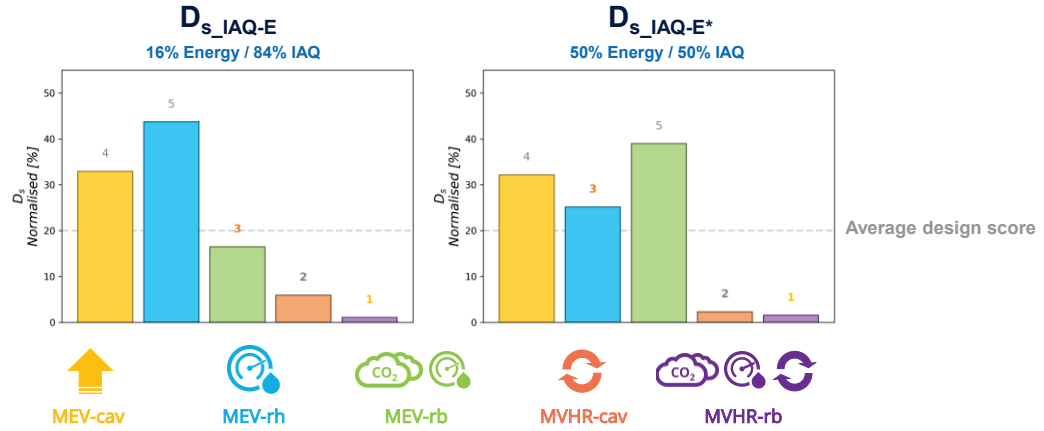
$$PR = I_{MC, D_{opt}, s} - C_s ; \text{ with } C_s = \min_s (I_{MC}(all_{D_{opt}}, s))$$

$$MPR = \max_{D_{opt}} (PR)$$

12

A ROBUST METHOD FOR PERFORMANCE RANKING

DESIGN SCORE RESULTS



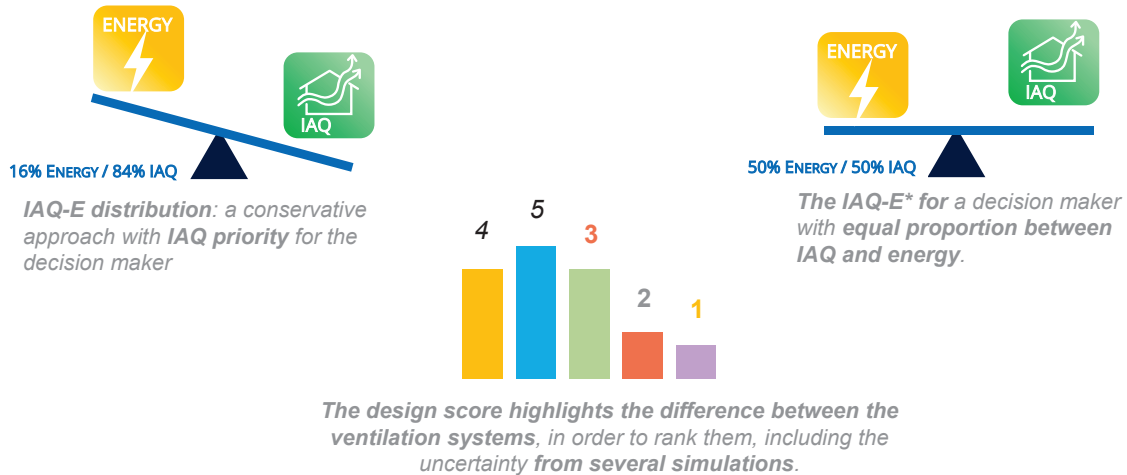
Design score calculation with the minmax regret method

design scores were normalized in [%] by $\sum(D_s)_{D_{opt}}$, the sum of all the design scores

13

CONCLUSION

LEARNINGS REGARDING ROBUSTNESS



The design score highlights the difference between the ventilation systems, in order to rank them, including the uncertainty from several simulations.

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THANK YOU
FOR YOUR ATTENTION



**TOWARDS PERFORMANCE-BASED APPROACHES
FOR SMART RESIDENTIAL VENTILATION**
A ROBUST METHODOLOGY FOR RANKING THE SYSTEMS AND DECISION-MAKING

43rd AIVC, 11th TightVent and 9th venticool Conference: “Ventilation, IEQ and health in sustainable buildings”

IMPORTANCE OF GOOD RESILIENT BUILDING DESIGN AND STANDARDS TO
ENSURE GOOD VENTILATIVE COOLING PERFORMANCE TO REDUCE
OVERHEATING AND ENVIRONMENTAL IMPACT

Introduction to topical session

Christoffer Plesner, Jannick K. Roth
VELUX A/S & WindowMaster International A/S
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jkr.dk@windowmaster.com



1

What is ventilative cooling?

- **Proposed definition of Ventilative cooling; (*)**
 - *“Utilization of outside air at its actual temperature and humidity aimed at improving indoor thermal comfort or decreasing cooling loads. Air transfer may be by natural, mechanical or hybrid means”*
- VC can reduce overheating and deliver good thermal comfort in buildings to achieve good well-being
- The climate target plan towards 2030 in terms of the EPBD revision aims to reduce energy use for heating and cooling by 18% in buildings.
- **Ventilative cooling types**
 - **Natural ventilative cooling** is an aspect of ventilative cooling whose operation is based solely on the effect of wind and stack effect
 - **Mechanical ventilative cooling** is an aspect of ventilative cooling whose operation is based solely on the operation of fans
 - **Hybrid ventilative cooling** is an aspect of ventilative cooling whose operation is based on the combination or alternation of natural and mechanical ventilative cooling

(*) “Ventilative cooling systems – Design” (CEN/TS, draft 2023)

2

What can be done?

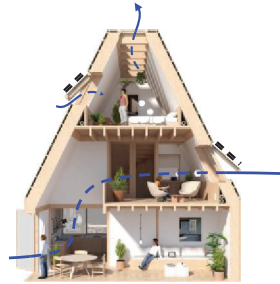
Build sustainable!

VELUX has built Living places in Copenhagen to show sustainable building

- 3x lower carbon footprint than avg. Danish home
- 3x better indoor climate than avg. Danish home
- Scalable solution
- Thoughtful building design ensuring good stack effect and cross ventilation!

Sustainable cooling solutions

- Controlled natural ventilative cooling (façade+roof windows)
- External solar shading
- No mechanical cooling
- Solar cells



3

Human Developments

A growing population, as well as rapid growth in purchasing power in emerging economies and developing countries means that energy demand in buildings could **increase by 50% by 2060 (**)**

Peak and mean summer temperatures will increase by 10C across most European capitals by 2080 (***)

Energy use for cooling of buildings rose +212% from 2010-2019 !. (****)

Researchers at the French National Center for Weather Research have concluded that **if Paris doubles its A/C use by 2030, it could raise outdoor temperatures in the city by 1,6 - 2,2 ° C.**

(*) IEA – The Future of Cooling

(**) www.globalabc.org/ & UN Environment, Global Status Report 2017

(***) Overheating calculation methods, criteria, and indicators in European regulation for residential buildings (Attia, et. Al, 2023)

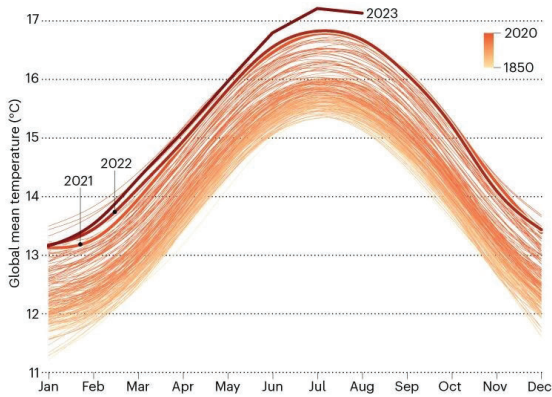
(****) EEA europe

4

Why important – a burning Platform

HEAT RISING

The global average temperatures in the past three months have set new records every month, often by a large margin.



©nature

(*) United in Science 2022, WMO

(**) Overheating calculation methods, criteria, and indicators in European regulation for residential buildings (Attia, et. Al. 2023)

(***) IEA – The Future of Cooling

Alert: Earth Nearing 1.5°C Warming

Model indicate that there's now a 55% likelihood of reaching 1.5°C warming in 2023. What matters most is that this year (2023) is on track to be the hottest on record.

Climate change is expected to drive an increasing frequency of heat waves, which can cause significant morbidity and mortality (**)

3 x increase in the global energy use for space cooling towards 2050 (*) - corresponding to almost a **2 x in CO₂ emissions** for space cooling (***)

All member states must revise their national energy calculation methods and address discomfort problems under climate changes scenarios by the end of 2025 (**)

5

How does Ventilative Cooling fits in this agenda?

Overall



#1: Resiliency

Ventilative cooling can:

Support the robustness and resilience of buildings by having key resilience indicators when designing future buildings for VC

Show resilience by using manual openable windows or by solar-powered solutions

#2: Indoor climate

Ventilative cooling can:

Be an effective measure to reduce buildings energy use, meeting some or all of the cooling requirement limiting use of A/C

Improve IAQ (by reducing CO₂ levels) due to already elevated air change rates

#3: Environmental impact

Ventilative cooling can:

Save resources (energy and material use)

Be considered as a renewable energy solution for cooling according to EU Renewable energy directive II,2018

Have a lowered direct impact on outside temperatures

6

Agenda

Introduction

- 1) Introduction to topical session
Christoffer Plesner, VELUX A/S, Denmark

&

Jannick Roth, Window-Master International A/S, Denmark

#1: Good building design

- 2) Update on resilient cooling and indicators from IEA EBC Annex 80
Ptryk Czarnecki, Institute of Building Research & Innovation, Austria
- 3) Resilient ventilative cooling in Design practice: where next?
Paul O'Sullivan, MeSSO Research at Munster technological university, Ireland

#2: Case studies

- 4) Life cycle assessment: A design element for ventilation system selection
Jannick Roth, Window-Master International A/S, Denmark
- 5) Lessons learned from Irish schools: Early-stage insights on Overheating
Adam O'Donovan, MeSSO Research at Munster technological university, Ireland
- 6) Resilient cooling in office buildings: case study in Belgium
Hilde Breesch, KU Leuven, Belgium

#3: Standards

- 7) Design procedures for ventilative cooling integrated in new standards
Christoffer Plesner, VELUX A/S, Denmark

8) Closing and open Discussion

Facilitated by *Christoffer Plesner and Jannick Roth*

IEA EBC Annex 80 - Resilient Cooling

Patryk Czarnecki
Institute of Building Research & Innovation
Vienna, Austria



Institute of
**Building Research
& Innovation** ZT-GmbH



venticool
the platform for resilient ventilative cooling



Federal Ministry
Republic of Austria
Climate Action, Environment,
Energy, Mobility,
Innovation and Technology

2023-10-04

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1

Topical Session Update on Resilient cooling and indicators from the IEA EBC Annex 80



Institute of
**Building Research
& Innovation** ZT-GmbH



venticool
the platform for resilient ventilative cooling



Federal Ministry
Republic of Austria
Climate Action, Environment,
Energy, Mobility,
Innovation and Technology

2

2

Definition of Resilience

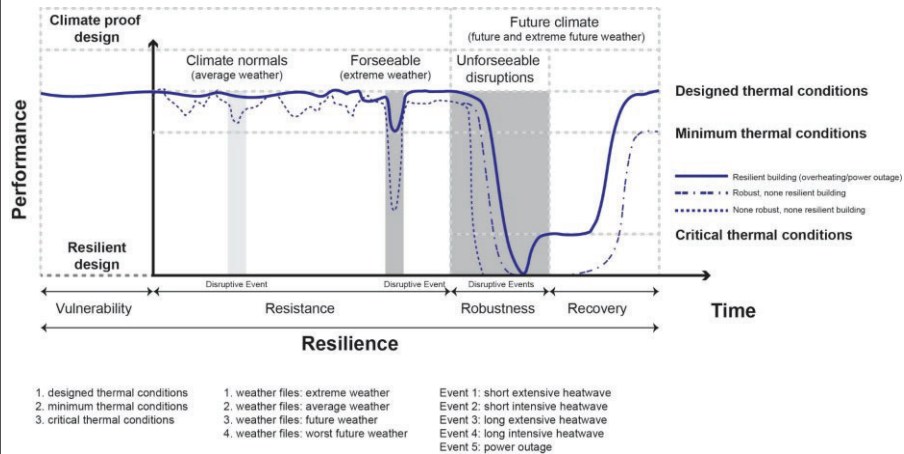
- Resilience is the new sustainability
 - Needs a precise definition

- Annex 80 focused on resilience against heatwaves and power outages

3

3

Definition of Resilience



© Annex 80 - Sub-Group A. Altia et al. (2020); the graph is partially inspired by Mozzani et al. (2019)



4

4

Technology Profiles

What?

- Structured Information about the Essential Resilient Cooling Technologies

Who?

- Planners, Property Owners and Investors, Interested Public

5

5

Technology Profiles - Chapters

1.	Reducing Heat Loads to People and Indoor Environments
1.1.	Solar Shading Technologies
1.2.	Cool Envelope Materials
1.3.	Glazing Technologies
1.4.	Ventilated Façades
1.5.	Green Roofs and Green Façades
2.	Removing Heat from Indoor Environments (Production, Emission and Combined)
2.1.	Ventilative Cooling
2.2.	Thermal Mass Utilization
2.3.	Evaporative Cooling
2.4.	Sky Radiative Cooling
2.5.	Compression Refrigeration
2.6.	Adsorption Chillers
2.7.	Natural Heat Sinks
2.8.	Radiant Cooling
3.	Increasing Personal Comfort Apart from Space Cooling
3.1.	Comfort Ventilation and Elevated Air Movement
3.2.	Micro-cooling and Personal Comfort Control
4.	Removing Latent Heat from Indoor Environments
4.1.	Dehumidification

6

6

Technology Profiles - Chapter Structure

- Description
- Key Technical Properties
- Performance and Application
- Further Reading

7

7

Technology Profiles – Ventilative Cooling

[TPS Ventilative Cooling.pdf](#)

8

8

Resilience of Ventilative Cooling

- Highly resilient (operation without electricity is possible)
- Emergency operation possible (Natural Ventilation)
- Vulnerable against strong heat waves

9

9

Reduce Solar Gain

Focus on what is effective

- (External) Sun Shading
- Glass Surfaces
- 5-10% Openable Surfaces per m²
- Electrical control
- Demand Ventilative Cooling in Building Standards

10

10

Preparing a new Annex-Proposal (?)

Resilient Cooling in Cities

Passive Climate Mitigation: Green, Blue and “White” infrastructures.

Hybrid Cooling: Hybrid Cooling solutions in time and space including latent cooling.

Heat sinks on urban scale: Seasonal storage in soil and water.
Deep Radiative Cooling.

Urban grids: Thermal & electric networks.



Cheonggyecheon stream revitalization project.
Downtown Seoul, South Korea, 2003 – 2005
Source: <https://www.ser-rrc.org/project/south-korea-restoration-of-the-cheonggyecheon-river-in-downtown-seoul/> (04.10.2022), foto credits: City of Seoul.

11

11

Preparing a new Annex-Proposal (?)

If you are interested in joining this new Annex, please contact us!

Peter Holzer, peter.holzer@building-research.at

Philipp Stern, philipp.stern@building-research.at

Patryk Czarnecki, patryk.czarnecki@building-research.at

Institute of Building Research & Innovation
Vienna, Austria

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Thank you for your attention.





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Munster Technological University

MeSSO

MECHANICAL ENERGY SYSTEM SIMULATION OPTIMISATION

Lessons Learned from Irish Schools: Early-stage Insights on Overheating

Adam O' Donovan^{1,2},

Elahe Tavakoli^{1,2}, Paul D O'Sullivan^{1,2}

¹MeSSO Research Group, Department of Process, Energy and Transport Engineering, MTU, Cork, Ireland

²MaREI Centre for Energy, Climate, and Marine, Ireland.

43rd AIVC | 11th TightVent & 9th Venticool Conference | Copenhagen, Denmark, October 4-5, 2023



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Contents



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Ollscoil Teicneolaíochta na Mumhan
Munster Technological University

- Background and context (RESILIENCE and schools)
- Materials and methods (Data, Information and Simulations)
- Results and discussion (**A: Do schools overheat, why and what about the future?**)
- Conclusions and lessons learned (What can we learn from this?)



MECHANICAL ENERGY SYSTEM SIMULATION OPTIMISATION

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2

Background and context

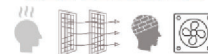
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Mission and Aim

RESILIENCE: MAPPING ADAPTIVE NV INDOOR ENVIRONMENTS IN IRISH LOW ENERGY BUILDINGS (SEAI, RD&D 2019)

Focus on overheating in naturally ventilated (NV) low energy or nZEB built environments in non-residential or commercial buildings. This research aims to:

“Systematically map and quantify how low or Nearly Zero Energy (i.e. nZEB’s) commercial building design, construction and operation in Ireland has affected or will affect indoor thermal environments in buildings that rely exclusively on passive strategies for the supply of fresh air and, more specifically, the removal of heat build-up that would otherwise lead to an unacceptable thermal experience for building occupants.”



4

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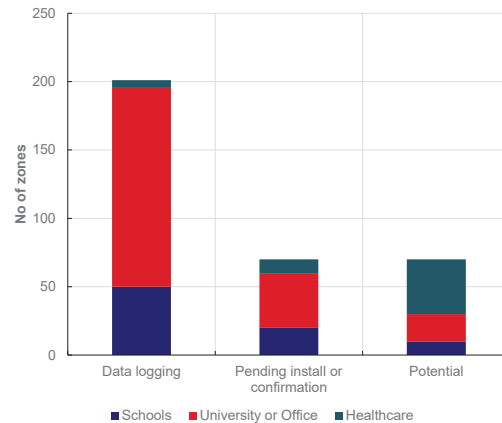
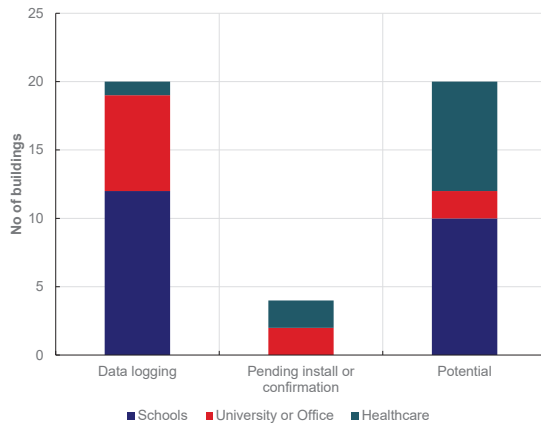
Due for completion in 2024



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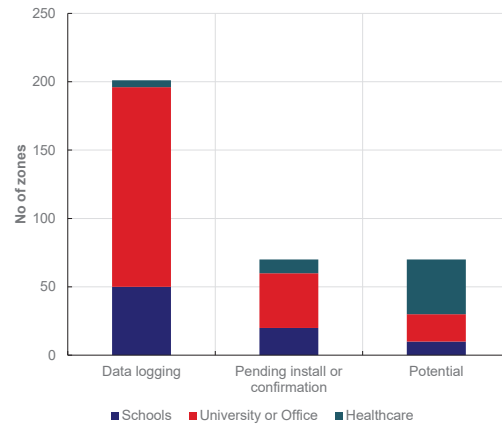
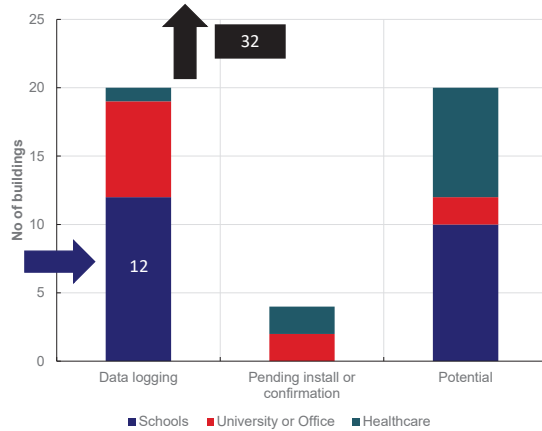
Actual sample (April 2023)



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Actual sample (April 2023)



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Materials and Methods

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Materials and Methods

List of schools in the study



Table 1: List of 12 case study school buildings (as of August 2023) and monitoring systems installed

School Name	Location	Building Type	No of Zones	Monitor Type	Accuracy
PS1	South	New	5	Stand-alone	±0.3°C
PS2	South	New	5	Stand-alone	±0.5°C
PS3	South	New	>5	Wifi-based	±0.1°C
PP1	South-West	Extension	5	Wifi-based	±0.3°C
PP2	Midlands	New	>5	BMS/ Stand-alone	±0.5°C
PS4	Mid-West	Extension/Retrofit	>5	BMS	±0.5°C
PS5	South	New	5	Stand-alone	±0.5°C
PS6	South	Extension/Retrofit	5	Wifi-based	±0.3°C
PS7	North-East	Retrofit	>5	BMS	±0.5°C
PP3	South-East	Retrofit	>5	BMS	±0.5°C
PP4	South-East	Extension	5	Wifi-based	±0.3°C
PS8	West	Existing	>5	BMS	±0.5°C

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Materials and Methods

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PS8	West	Existing	>5	BMS	±0.5°C

60-80 classrooms

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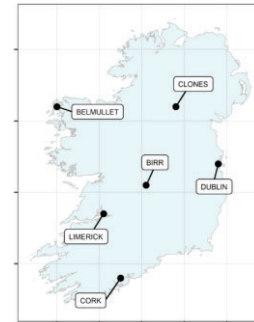
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Materials and Methods

Information, Data, Simulation



- Minimum 1-hour interval space air temperature data
- Building information mapping examples:
 - Area of openings, height
 - Opening types, Cooling Strategy and Principle
 - Number of occupants
- Calibrated Dynamic Simulation into the Future (IES-VE)
 - Design summer year data (2021 to 2071)
 - Climate data for thermal modelling of buildings



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Materials and Methods

Information, Data, Simulation



- Minimum 1-hour interval space air temperature data
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Materials and Methods

Metrics for schools (comfort or performance)

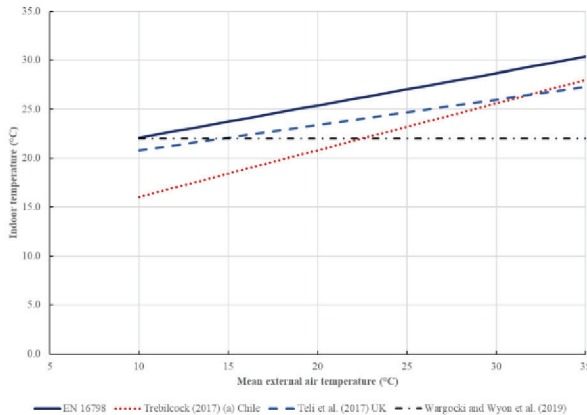


Figure 1: Comparison of adaptive thresholds for adults, children as well as thresholds reflective of relative learning performance

- Limited examples of adaptive comfort in school settings
- Should school be about comfort or performance
- **1.3°C to 3.1°C** difference **between adults and children**
- **0.1°C to 8.4°C** between **comfort and performance**

Materials and Methods

Metrics for schools (comfort or performance)

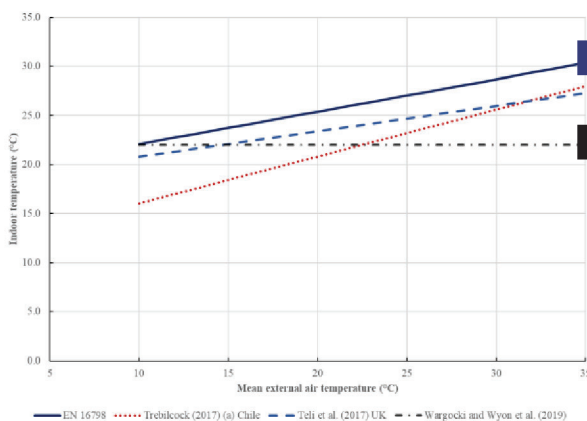


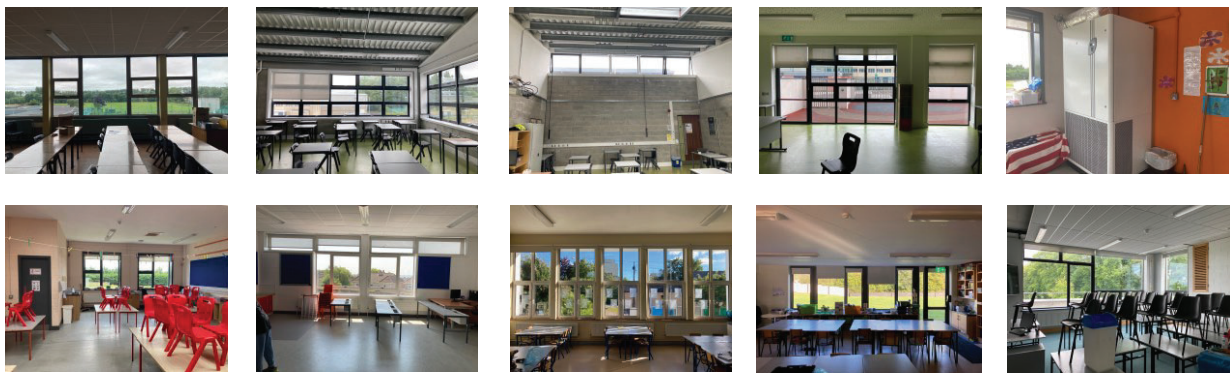
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Results and Discussion

Results and Discussion

Information – examples of strategies



Results and Discussion

Information – building information (preliminary) 50 zones



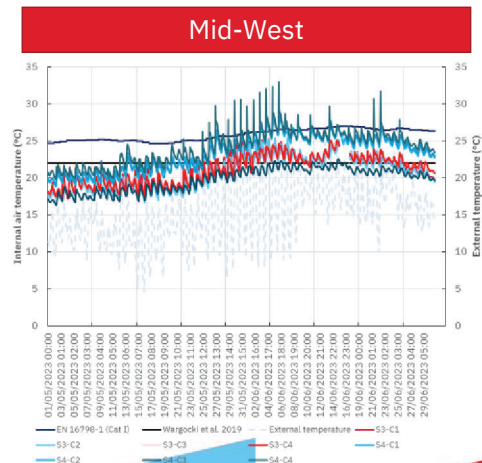
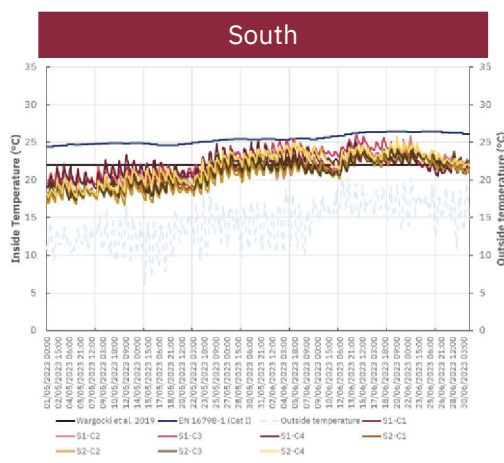
- Strategy – **92%** natural, 4% mechanical, 4% both
- NV principle – **90%** single-sided with multiple openings, 10% use cross
- **6 to 11** openable windows per classroom (mainly top-hung outward opening)
- Openings typically open between **14cm and 32cm** (some exceptions 100cm)
- Opening height **0.57m to 1.4m** (floor-to-ceiling height 2.6m to 3.2m)
- Typically around **26** seating allocations (maximum 35)

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Results and Discussion

Data – May/June 2023 (4 schools in Munster)

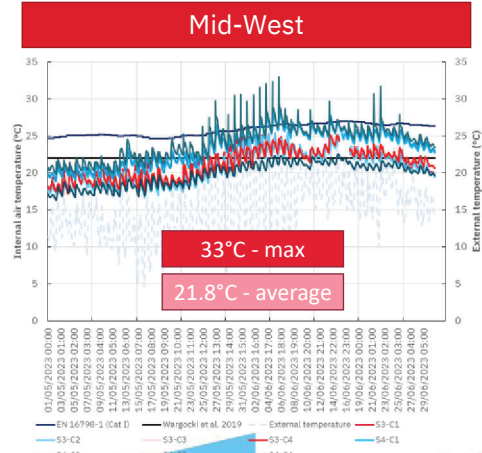
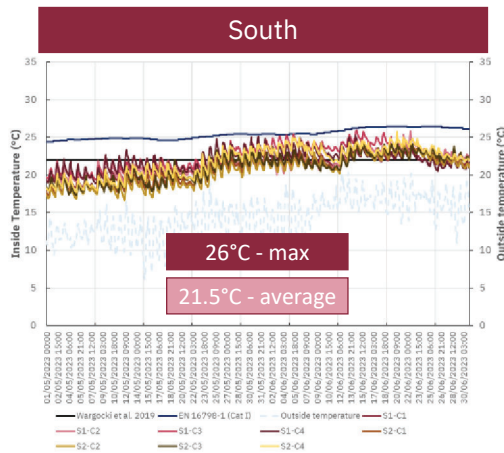


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Results and Discussion

Data – May/June 2023 (4 schools in Munster)



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Results and Discussion

Data – May/June 2023 (4 schools in Munster) - summary



South		
Classroom	> Cat I	> 22°C
S1-C1	0%	44%
S1-C2	0%	50%
S1-C3	0%	65%
S1-C4	0%	63%
S2-C1	0%	37%
S2-C2	0%	29%
S2-C3	0%	42%
S2-C4	0%	59%

Mid-West		
Classroom	> Cat I	> 22°C
S3-C1	0%	40%
S3-C2	0%	32%
S3-C3	0%	46%
S3-C4	0%	46%
S4-C1	8%	67%
S4-C2	10%	66%
S4-C3	0%	2%
S4-C4	22%	79%

*08:00 – 16:00

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Results and Discussion

Data – May/June 2023 (4 schools in Munster) - summary



South		
Classroom	> Cat I	> 22°C
S1-C1	0%	44%
S1-C2	0%	50%
S1-C3	0%	65%
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S2-C2	0%	29%
S2-C3	0%	42%
S2-C4	0%	59%

Mid-West		
Classroom	> Cat I	> 22°C
S3-C1	0%	40%
S3-C2	0%	32%
S3-C3	0%	46%
S3-C4	0%	46%
S4-C1	8%	67%
S4-C2	10%	66%
S4-C3	0%	2%
S4-C4	22%	79%

*08:00 – 16:00

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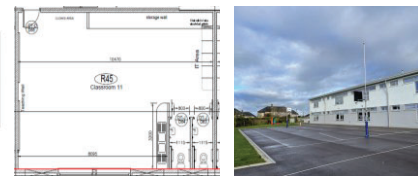
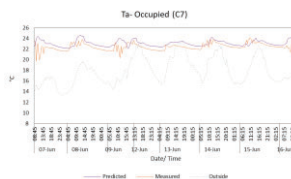
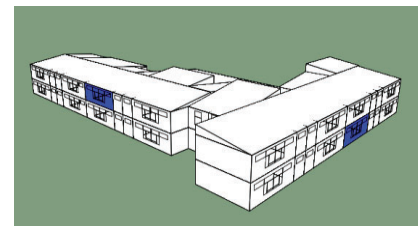
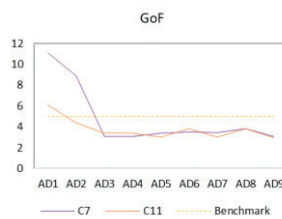
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Results and Discussion

Simulation – Generic Repeatable Design (Calibration)



- Model calibrated with data from two classrooms (**RMSE < 1°C**)
- Model used to simulate performance in 2 locations (Cork and Limerick)
- Two weather files:
 - DSY1-2021 (Moderate overheating)
 - DSY2-2071 (Prolonged overheating events)

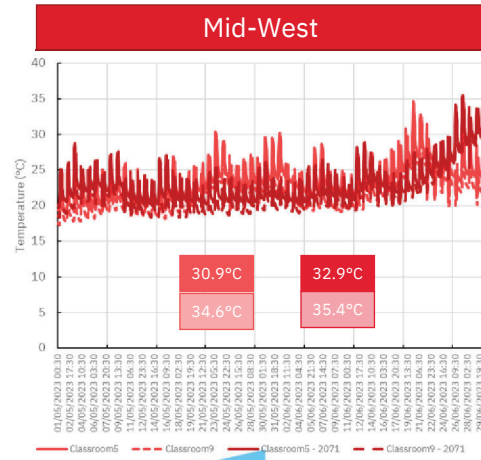
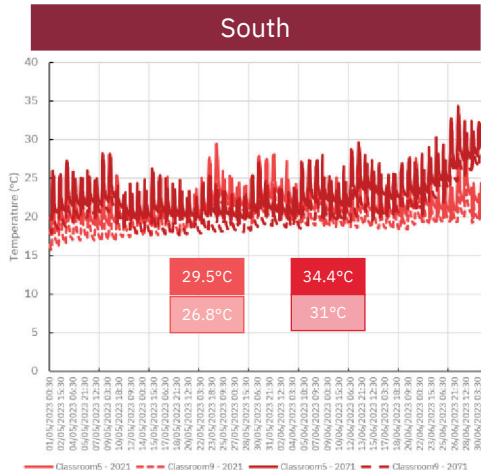


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Results and Discussion

Simulation – Generic Repeatable Design (preliminary)



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Conclusion and lessons learned

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Conclusion and future work

Renewable cooling in Ireland



- **Conclusions**

- **Overheating exists** in some Irish classrooms. **Day-time ventilation only** may not be sufficient **in future**.

- **Lessons learned**

- **SS very common**, limited height of openings, **more use of stack**, atria and guiding components (older buildings not worse in some cases).
- Issues in getting access to BMS data proved troublesome. (**Bespoke data logging required**)
- **Limited** degree of **automation** of natural openings (investment in controls required?)
- Use of mechanical demand controlled VC single-zone could be a useful retrofit solution going forward.
- **New buildings will need further upgrades for heat waves.**

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Thank You, Any Questions?

adam.odonovan@mtu.ie



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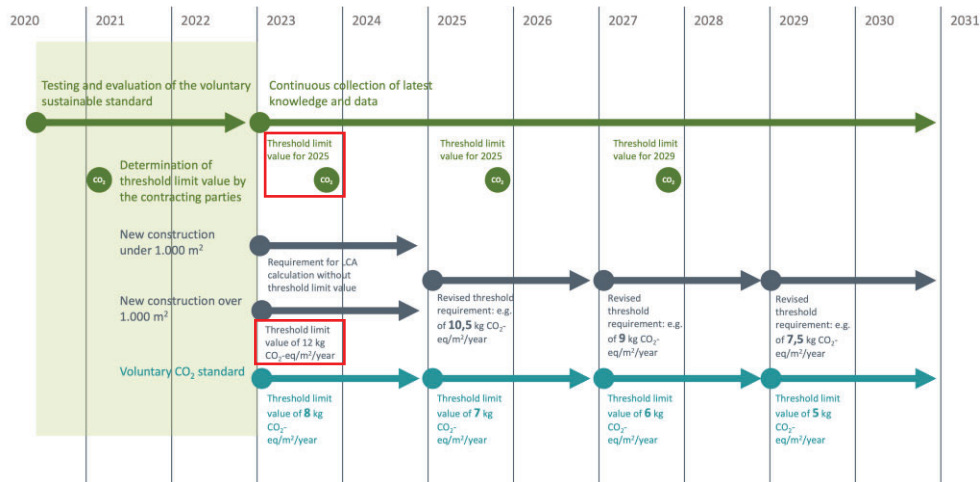
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Life Cycle Assessment: A design element for ventilation system selection



1

Danish Building regulation



2

Office building in Denmark

Comparing LCA (CO₂-eq) and LCC for hybrid and mechanical ventilative cooling

Designed for hybrid ventilative cooling



About the building

- Constructed in 2017
- 1230 m²
- Automated controlled solar shading

Ventilative cooling strategies

Hybrid (HVC)

- Heating season: MVHR
- Cooling season: NVC
Automated natural ventilation via facade- and roof openings

Mechanical (MVC)

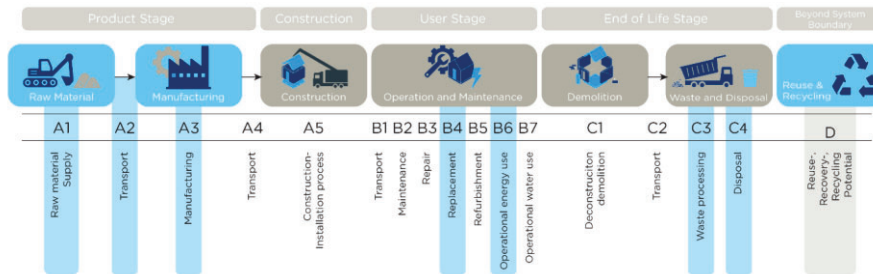
- Heating season: MVHR
- Cooling season: MVHR



Phases included (blue)

Environmental impact over a lifespan of 50 years

According to Danish Building code



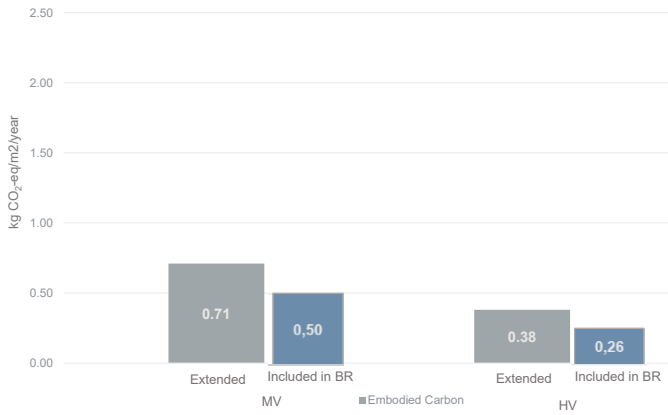
Components assessed

- EPD's
 - Natural ventilation components
 - AHU
- Rest by weight



Results – extended vs. included

Environmental calculation focusing on CO₂-eq



Remarks

- Mecahnical ventilation has a 40% increased CO₂ emission from embodied carbon compared to what should be included according to the Danish Building regulation.

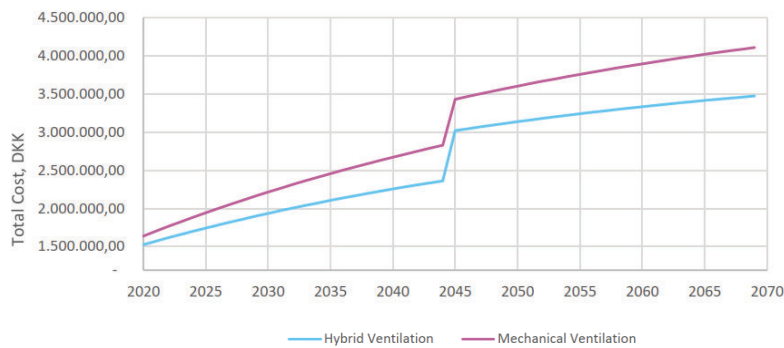
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LCC

Ventilation Systems, Cumulative Cost



Hybrid ventilative cooling offers

- Lower capital and runing cost compared to pure mechanical ventilation.

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Conclusion

- **Designed** hybrid ventilative cooling can reduce carbon emissions and cost compared to a pure mechanical ventilative cooling system.
- By adding embodied carbon from natural ventilative cooling of 0.033 kg CO₂eq/m²/year one can reduce the CO₂-eq:
 - Embodied by more than 40%
 - Electricity by more than 50%
 - In total by more than 30%

Compared to pure mechanical ventilative cooling.



Questions?

Jannick Roth. jkr.dk@windowmaster.com





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Munster Technological University

MeSSO

MECHANICAL ENERGY SYSTEM SIMULATION, OPTIMISATION



Tight Vent
Europe

venticool
the platform for resilient ventilative cooling

Ventilative Cooling Design in Practice: Where Next?

Paul D O'Sullivan

Adam O Donovan, Maha Sohail, Christoffer Plesner, Jannick Roth, Beat Frei, Annamaria Belleri, Valentina Radice

43rd AIVC, Copenhagen, 4th & 5th October 2023

5 October 2023



Department of
Process, Energy &
Transport Engineering
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Agenda & Context

Ventilative Cooling Design in Practice

- Recap from 2022, early-stage intervention in VC design
- Practitioner Perceptions of VC: A survey
- Insights from Interviews with designers
- Resilient VC Design: A two stage design process



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Recap from AIVC 2022

Early-stage intervention to improve resilience



- Need to translate expert knowledge into simple but informed methods to evaluate resilience
- Embed VC & Resilient Principles from the outset of the design
- Architects, Planners, Stakeholders need to be part of the resilience and heat wave/extreme event planning

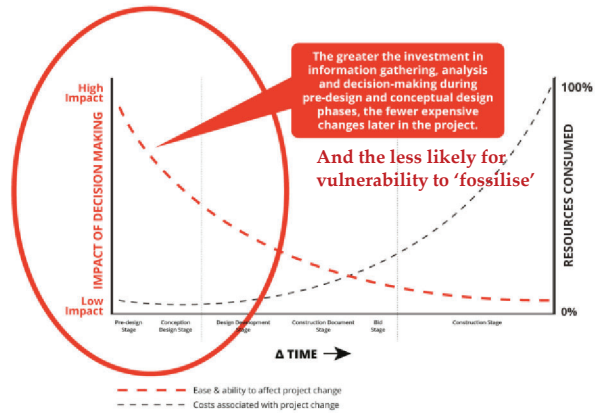


Fig 4: MacLeamy Curve: Influence of Early Effective Decision-making on Project Outcomes

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Designing in Thermal Resilience

Early Stage ~~Non-Expert~~ Interventions



Conceptual stage design frameworks that Resilient VC non-experts (some architects etc) can use to evaluate resilience

- Is early stage interventions in the design process useful to reducing the cooling performance gap?
- Is early stage pre-design identified as an important stage for VC designers?
- How is early stage design done?

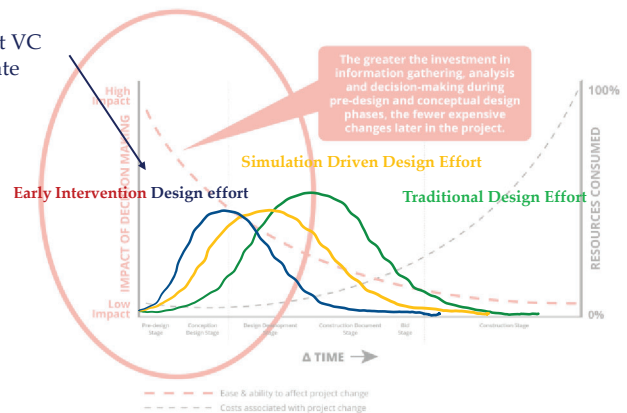


Fig 4: MacLeamy Curve: Influence of Early Effective Decision-making on Project Outcomes

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Practitioner Perceptions of VC: A Survey

5

Practitioner Perspectives

Findings from a UK/Ireland Survey

- Surveyed 71 building design practitioners in Ireland & UK, “Perception of ventilative cooling in building design practices”.
- 57 Questions, 40 respondents completed survey fully (Architects, Engineers etc)
- The aim was an attempt to harvest information about how designers perceive ventilative cooling and how they approach VC design

M., Sohal, A., O'Donovan, C., Plesner, P.D. O'Sullivan, (2023), A survey of building design practitioner perceptions of ventilative cooling in their building design processes. AIVC 2023, Copenhagen, Denmark, October 2023

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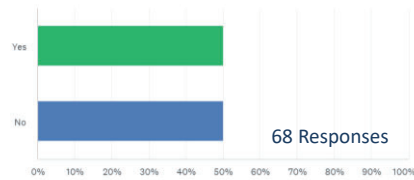
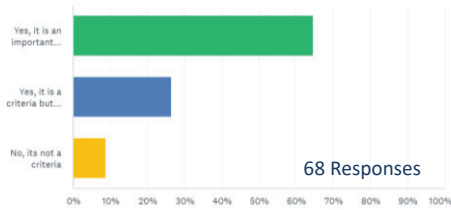
Practitioner Perspectives

Findings from a UK/Ireland Survey



Would you identify cooling of the indoor environment in buildings an important design criteria amongst your profession in your particular climate/country?

Are you familiar with the term "Ventilative Cooling"?



M., Sohall, A., O'Donovan, C., Plesner, P.D. O'Sullivan, (2023), A survey of building design practitioner perceptions of ventilative cooling in their building design processes. AIVC 2023, Copenhagen, Denmark, October 2023

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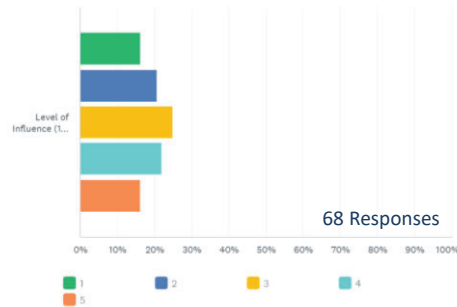
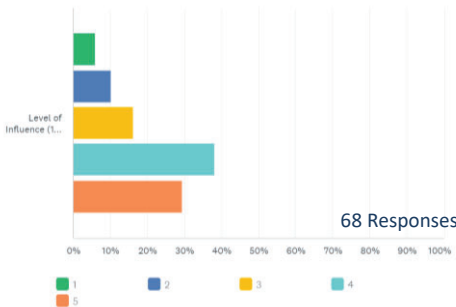
Practitioner Perspectives

Findings from a UK/Ireland Survey



In your experience, is **ventilating of the indoor environment** an influential criteria in the overall building design *at the early stages* of the design process? (i.e. pre-design, schematic design)

In your experience, is **cooling of the indoor environment** an influential criteria in the overall building design *at the early stages* of the design process? (i.e. pre-design, schematic design).



M., Sohall, A., O'Donovan, C., Plesner, P.D. O'Sullivan, (2023), A survey of building design practitioner perceptions of ventilative cooling in their building design processes. AIVC 2023, Copenhagen, Denmark, October 2023

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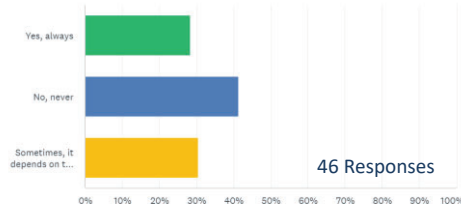
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Practitioner Perspectives

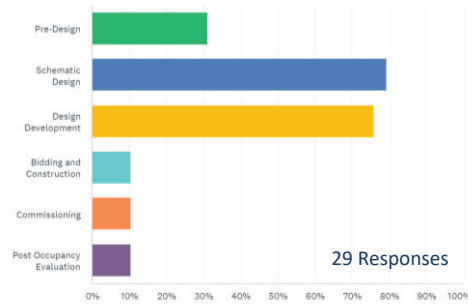
Findings from a UK/Ireland Survey



Do you use any tools or software for making a ventilative cooling design decision?



At what stage(s) of your building design process, have you used this/these tools?



M., Sohall, A., O'Donovan, C., Plesner, P.D. O'Sullivan, (2023), A survey of building design practitioner perceptions of ventilative cooling in their building design processes. AIVC 2023, Copenhagen, Denmark, October 2023

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Insights from Interviews with VC Designers

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Insights From Interviews

The experience of the VC designer [UK & Ireland]



- Interviewed 10 VC design experts in Ireland & UK [summer 2023]
- 40 to 80 minute interviews. Data still being processed, not available yet.
- The aim was an attempt to build from the survey work and get a first-hand impression of how VC design is actually done in practice.

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Insights From Interviews

The experience of the VC designer [UK & Ireland]



“Tools are not the problem, **the people** using them are”

“There are plenty of tools available in the market and each company has in house tools as well, therefore **a new VC tool won't be useful in the industry unless it is part of a building regulation (law)/compliance procedure**”

“**Combination of passive and hybrid ventilation strategies** lead to a resilient VC strategy in view of design practitioners in industry”

“There are buildings existing that have an excellent VC design process that still works, they mostly had **collaboration of all building design stakeholders at early/concept stages**”

“Lack of collaboration between engineers, consultants and the Architect **at early stages** is often the reason of poor building designs as far as VC is concerned”

“The business of design gets in the way of better design”

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Insights From VC Designers

The experience of the VC designer [UK & Ireland]



- Hand sketches (pen and paper) and excel are the two most important tools used in **early concept stages of building design** by practitioners
- IES is the most popular tool in case of non-domestic VC decision making, **but it is mostly used at detailed phases of building design**. PHPP is also very popular for domestic buildings among Architects in particular, also often used at detailed stages.
- **A database of different non-domestic archetypes with VC strategies that work from history can be useful in industry** ← **Informative Annex of Archetypes**
- Experienced practitioners, Architects especially (age 50+) do not always appreciate new tools/forced technological advancements for their own usage, they rather believe in hand sketches or learning from history of each climate for better building design at early stages

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New CEN Ventilative Cooling Technical Specification
//
Resilient VC Design: A Two Stage Process

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New CEN VC Technical Specification

Developing a VC design process/approach



Some challenges

- Early-stage evaluation of complex solutions where performance prediction is difficult without advance tools
- Design is a combination of both iterative processes and sequential stages
- Does a prescriptive approach to VC design exist? (i.e. min recommended ACRs / no performance checks)

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New CEN VC Technical Specification

Identifying the steps...



So far.....

Identified 7 stages in ventilative cooling design [courtesy Beat Frei]:

1. Set performance criteria
2. Specify design assumptions
3. Evaluate cooling potential
4. Define ventilation operating principle
5. Develop ventilation strategy
6. Consider and accommodate the need for supplementary cooling
7. Define controls and operation

These steps continuously repeat with increasing levels of detail throughout the design. Milestones are fluid, design is a live shared space.

Complex solutions to overcome cooling limitations in basic/simple designs are captured in "supplementary cooling"

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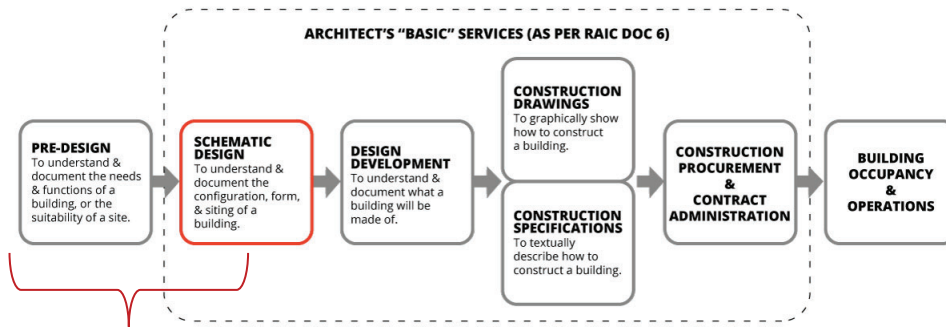


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New VC Technical Specification

Design Stages – Original thinking (AIVC 2022)



Significant Design Opportunity to prevent future vulnerability "lock-in" Fig 5: Early stage intervention in design process

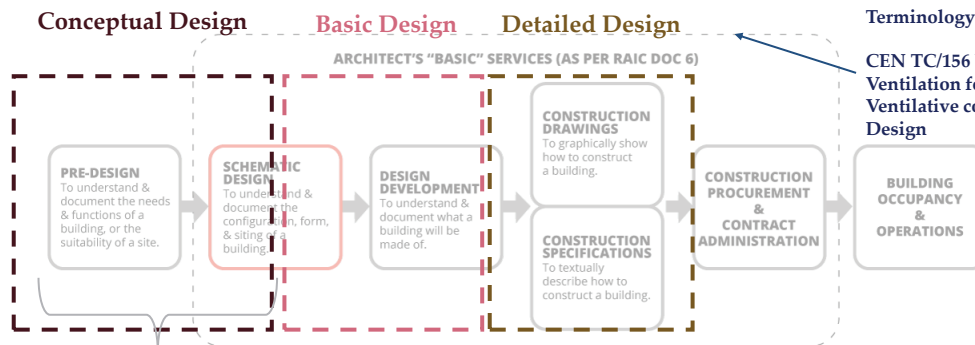
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New VC Technical Specification

Design Stages – Original thinking (AIVC 2022)



Terminology from
CEN TC/156 WG21
Ventilation for buildings –
Ventilative cooling systems –
Design

Significant Design Opportunity to prevent future vulnerability "lock-in" Fig 5: Early stage intervention in design process

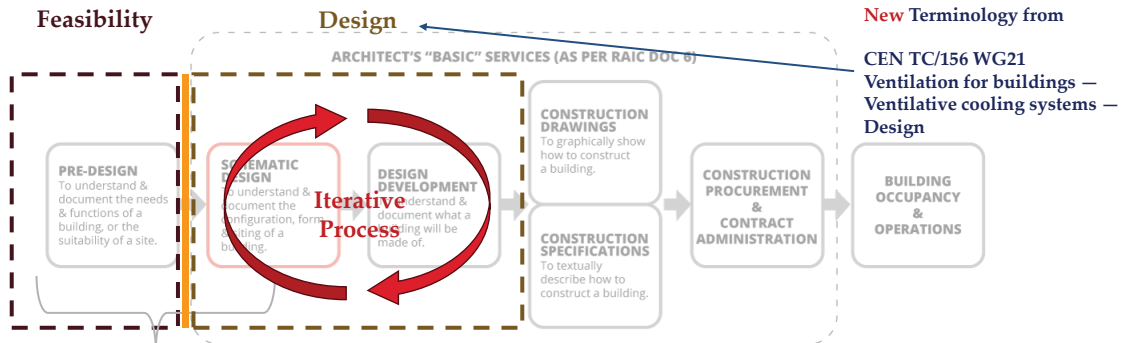
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New VC Technical Specification

A two stage process



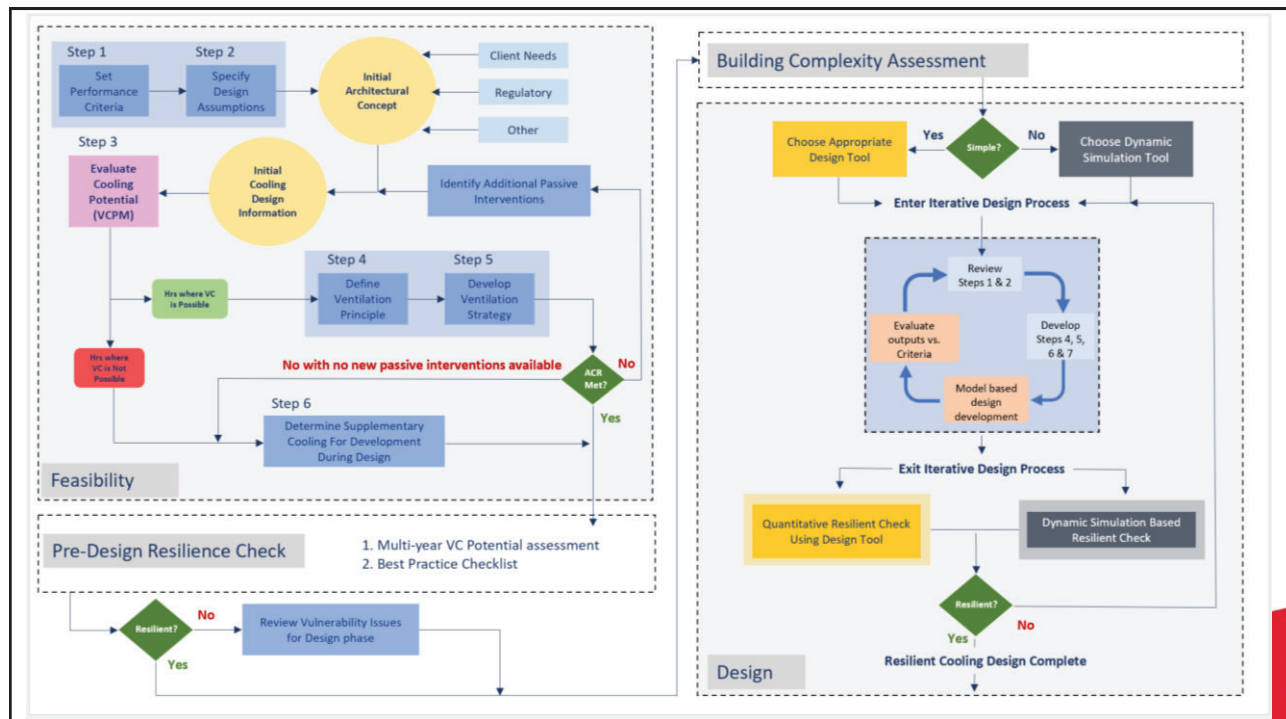
Resilience Check & Complexity Assessment Fig 5: Early stage intervention in design process

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In Closing...



- Easy stage intervention in VC design leads to better design outcomes
- The new Technical Specification will provide a framework for (reasonably) easy assessment of the feasibility of VC designs
- Need early stage guidance that recognises the design culture of VC practitioners
- How best can we address early-stage resilience evaluations?
- Reliable early-stage evaluations of complex supplementary solutions? (i.e. hybrid, double skin facades, cool materials etc)

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MeSSO

MECHANICAL ENERGY SYSTEM SIMULATION OPTIMISATION



Tight Vent Europe

venticool
the pattern for resilient ventilative cooling

Thanks for listening.

paul.osullivan@mtu.ie

6 OCT 2022



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22

Resilient cooling in office buildings: case study in Belgium

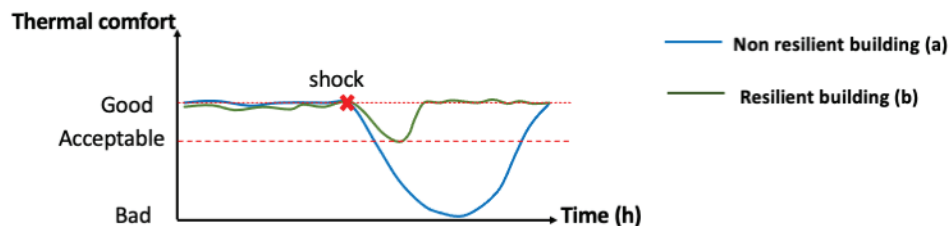


Shiva Khosravi¹, Joost Declercq¹, Abantika Sengupta^{2,3}, Hilde Breesch²
¹ archipelago architects, ² KU Leuven, ³ Ghent University

1

Challenges

- Contemporary buildings vulnerable to face extreme events or **shocks**
- **Design** of buildings resilient to overheating
 - Unexpected events not included in daily design practice
 - No framework in building standards
 - Limited knowledge influencing building design parameters resilient buildings

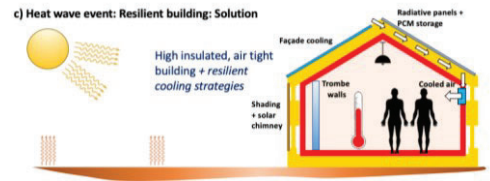


2

2

ReCOOver++

- Aim
 - Improve resilience of buildings to overheating
 - Making resilience tangible & actionable concept for designers & manufacturers
- Case studies



Office buildings



School



Residential



Case study

- New low-tech office building in Leuven (BE)

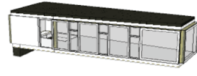
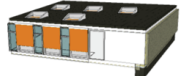
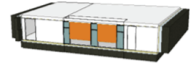


Location	Leuven, Belgium
Building Type	Offices
Floors	2
Occupants in each floor	32
Surroundings (Urban/Rural)	Urban
Year of construction	New construction
Floor Area (m ²)	1019
Windows area to floor area ratio (%)	40%
Window-to-wall ratio (%)	21%
Climate zone	4 (ASHRAE 90.1)

Annex 80: Resilient Cooling Guidelines

Resilience assessment

- Approach
 - Step 1: Analysis passive design strategies current & future weather -> best thermal comfort + lowest energy use
 - Step 2: Resilience assessment
- Shocks
 - Heat wave (current + long term future)
 - Heat wave + 24h power outage
- Indicators
 - Occupied hours above temperature
 - Energy use (kWh)
 - Overheating escalation factor (OEF)

		
Zone A	Zone B	Zone C
+0	+0	-1
reception	open office	open office
89m ²	225m ²	315m ²

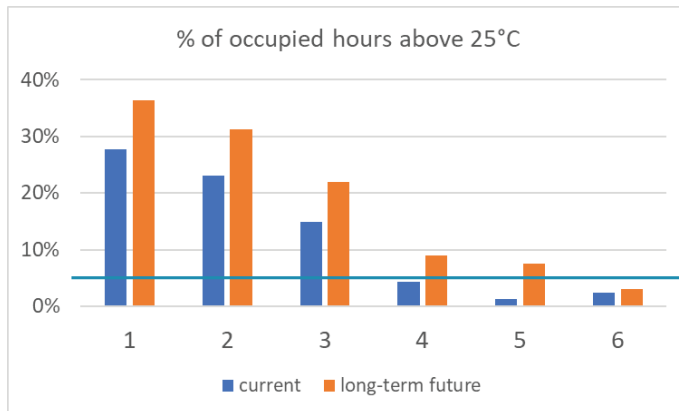
Simulation scenarios

- Cooling strategies
 - Shading
 - Ventilative cooling (+ thermal mass)
 - Mechanical cooling
 - Radiant panel
 - TABS

Scenario	Auto-mated shading	Mechanical ventilative cooling 24/7	Natural ventilative cooling 24/7	Thermal mass (DIN 4108-1)	Mechanical heating and cooling devices
					zone level
1	no	no	no	light	radiant panel
2	yes	no	no	light	radiant panel
3	yes	yes	no	light	radiant panel
4	yes	no	yes	light	radiant panel
5	yes	no	yes	heavy	radiant panel
6	yes	No	yes	heavy	TABS

Step 1: passive design analysis

- Thermal comfort (zone B)



Zone B

7



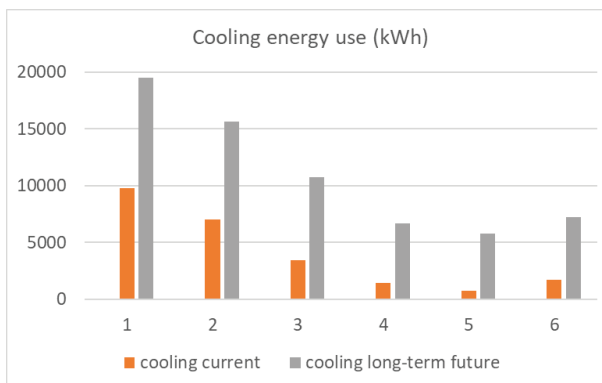
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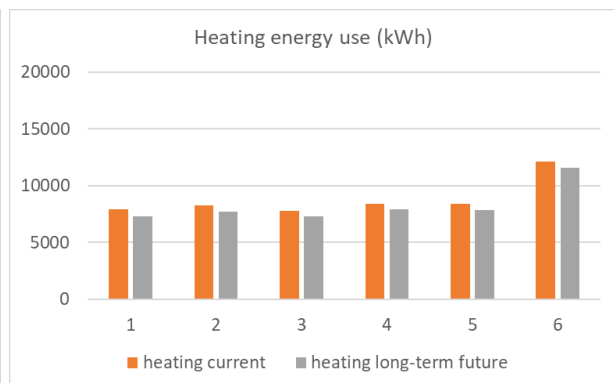
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Step 1: passive design analysis

- Energy use cooling (3 zones)



- Energy use heating (3 zones)



8



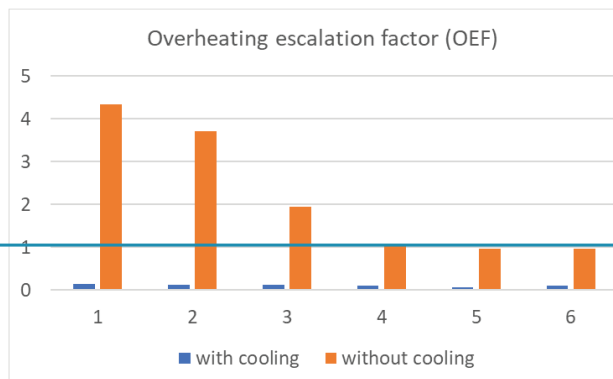
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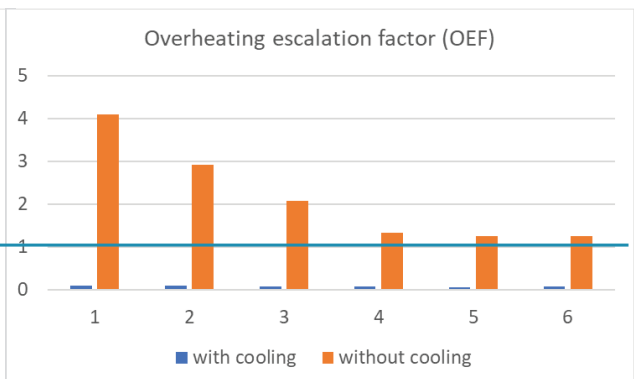
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Step 2: resilience assessment

- Heat wave (current – 25d)



- Heat wave (future long-term – 44d)



9

9

Summary

- Case study office building resilient to overheating
- Passive design analysis: 2 step approach
 - Thermal comfort and energy use
 - Resilience to overheating
- Need for resilience indicator
 - Explaining how buildings react to shocks
 - Communicable to stakeholders

10

10

43rd AIVC, 11th TightVent and 9th venticool Conference: “Ventilation, IEQ and health in sustainable buildings”

IMPORTANCE OF GOOD RESILIENT BUILDING DESIGN AND STANDARDS TO
ENSURE GOOD VENTILATIVE COOLING PERFORMANCE TO REDUCE
OVERHEATING AND ENVIRONMENTAL IMPACT

Design procedures for ventilative cooling integrated
in new standards

Christoffer Plesner
christoffer.plesner@velux.com



1

Projects in CEN & ISO (ventilation)

• What:

- Technical documents in CEN/TC 156 and ISO/TC 205 have started up, dealing with Design of “**Ventilative cooling systems**”
- Plan is to **support content of European EPBD standards** in revision; e.g. EN 16798-1 (indoor air quality, thermal comfort, energy performance)
- These technical documents should:
 - Refer to relevant standards (**performance requirements** and **calculation standards**) but make their own design framework
 - Give extra focus on good design guidance in conceptual design phase to assist in finding the **Ventilative cooling potential** early on

• Overall purpose:

- Make technical documents focusing on setting criteria and giving guidance to design of ventilative cooling systems in buildings for good thermal comfort
- Guide designers/engineers to know what to be aware of when designing ventilative cooling systems in order to fulfill the set criteria
- To make a common design procedure to design ventilative cooling systems in 3 stages

2

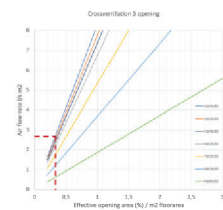
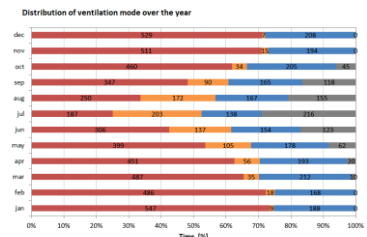
New ventilative cooling projects (P) in CEN & ISO

- **P1: “Ventilative cooling systems – Design” (in all buildings)**
 - Main focus: Thermal comfort (reduce cooling loads and reduce over-heating)
 - Type: A CEN Technical specification
 - Applicable to: Natural, mechanical and a combination (hybrid) ventilative cooling systems & interaction with supplementary and mechanical cooling systems
 - **Wish:** Be a reference in revision of EN 16798-1 (Design and assessment of thermal comfort)
- **P2: “Design process of ventilative cooling systems; Part 1 - non-residential buildings”**
 - Main focus: Thermal comfort (reduce cooling demand and reduce over-heating)
 - Type: An ISO standard (series)
 - Applicable to: Natural, mechanical and a combination (hybrid) ventilative cooling systems & interaction with supplementary and mechanical cooling systems
 - **Wish:** Be a reference in revision of ISO 17772-1 (equivalent to EN 16798-1)

3

Ventilative cooling systems (CEN/TS)

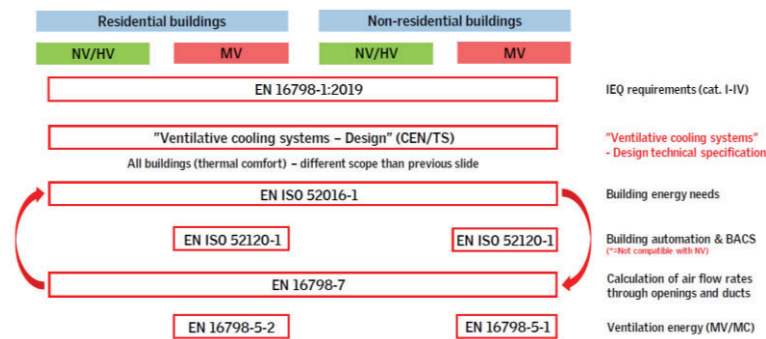
- **Methods to achieve the aim in our new technical documents:**
 - **Common design procedure phases** for ventilative cooling systems
 - **Conceptual** (feasibility)
 - **Basic** (choose air flow rates..)
 - **Detailed** (choose actual components)
 - Continuous **resilience checks** in each design phase (from simple questionnaires to actual Key performance indicators)
 - Use of **Ventilative Cooling Potential Method** in early conceptual design phase for evaluation of the ventilative cooling potential
 - Different ventilative cooling modes, e.g. VC mode 2 = increased air flow rates
 - To hear more: Go to Annamari Belleri's (EURAC) presentation
 - **Method** to find the needed opening area (e.g. for openable windows), based off the needed air change rate (e.g. output from VCPM)
 - EN 16798-7 can be used (see picture)
 - Guidance to **supplementary interventions and modulations (VC+)** on room and building level, e.g. add wall switches, solar shading or thermal mass
 - A **flow diagram overview** for design procedure is in the works for more clarity for designer



6

Ventilative cooling systems (CEN/TS)

- Reference to EPBD standards in CEN/TS
 - Aim for CEN/TS to fit into EPBD standards (e.g. EN 16798-1-2 ; thermal comfort) so referring to compatible standards is important
 - Not full compatibility for natural and hybrid ventilative cooling (to be worked in)



7

Conclusion

- Ventilative cooling can if the building is designed optimally be a good hybrid solution and alternative to mechanical cooling, to save peak load cooling loads and increase the design flexibility in buildings
- Important to have more technical documents highlighting more means like VC to combat the increasing impacts of climate change
- VC+ (ventilative cooling and solar shading) has shown to reduce overheating, often helped by robust early design evaluations like Ventilative cooling potential method
- Our CEN/TS is a good supplement to the EPBD EN standards, mostly telling which IEQ requirements to fulfill and not how
- Implementation of more **human needs** into standards and legislation (part of well-being) is important, e.g. adaptive comfort approach

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Conclusion

- Putting ventilative cooling on the agenda as a good alternative to mechanical cooling
- Presenting a “simplified” evaluation methods in the early conceptual design phase to estimate the ventilative cooling potential (fx for how long can VC be used during different months/years)
- Showcasing that openable windows, louvres, etc can be used in the ventilative cooling system – pretty unclear in other documents
- To be put as reference in revision of EN 16798-1-2 (thermal comfort), when talking of non-mechanically cooled buildings

12

Thank you!

14

43rd AIVC, 11th TightVent and 9th venticool Conference: “Ventilation, IEQ and health in sustainable buildings”

IMPORTANCE OF GOOD RESILIENT BUILDING DESIGN AND STANDARDS TO
ENSURE GOOD VENTILATIVE COOLING PERFORMANCE TO REDUCE
OVERHEATING AND ENVIRONMENTAL IMPACT

Discussion

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1

Questions

- Can ventilative cooling be the optimal solution in future for resilient buildings, if the building is designed correctly?
- Can or should future standards and legislation be more flexible towards human-behaviour etc.?
- How can ventilative cooling best be used in future, with increasing outdoor temperatures?
- What is the main barrier for implementation of ventilative cooling in buildings and standards?



2

Ventilative cooling systems (CEN/TS)

- What do you think of the following global design procedure for design of ventilative cooling systems ?
- General design procedure – key decisions (draft proposal)
 1. Setting performance criteria and requirements.
 2. Specifying design assumptions
 3. Evaluate the ventilative cooling potential
 4. Defining an operating principle
 5. Defining a ventilation strategy for ventilative cooling
 6. Check for interaction between ventilative cooling systems with supplementary and mechanical cooling (if needed).
 7. Defining controls and operation



3

Thank you!

4

Sensitivity Analysis of CO₂ Concentrations as Ventilation Metrics

Dr. Tobi Oke & Dr. Andrew Persily

National Institute of Standards and Technology
Gaithersburg, Maryland USA

43rd AIVC – 11th TightVent & 9th Venticool conference, 2023
Copenhagen, Denmark, 4-5 October, 2023



Courtesy of David Meyer,
Shenandoah University

1

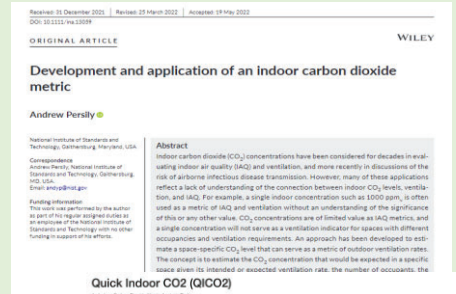
MOTIVATION

- ❑ CO₂ monitoring for ventilation and IAQ assessment often involves a single CO₂ concentration (e.g., 800 ppmv to 1000 ppmv) for all spaces
- ❑ A single value ignores important differences between spaces and occupants.
- ❑ CO₂ metrics for ventilation adequacy need to consider the following :
 - (i) Target ventilation rate
 - (ii) Space volume
 - (iii) Indoor activities performed (metabolic rates)
 - (iv) Occupant characteristics and schedule

2

PRIOR WORK

- ❑ Development of an Approach to Calculating Space-Specific CO₂ Metrics of Ventilation (Persily, 2022)
- ❑ Online tool, QICO2 to facilitate the application of this approach (Persily and Polidoro, 2022)



Quick Indoor CO₂ (QICO2)

An Indoor Carbon Dioxide Mass Analysis Tool

See the documentation of the tool

Building Type: Commercial/Institutional | Model Type: Placeholder | User Defined

User-Defined Commercial Building

Building Description

Building Name: Outdoor CO₂ Concentration: Ceiling Height: Building Floor Area:

Indoor Temperature:

Space Description

Primary Ventilation Rate per Person: Alternate Ventilation Rate per Person: Time to Metrics:

Define Occupants

Sex: Male Female | Age Group: | Mean: | Std: | Activity Level:

Number of Occupants: [Get Occupants](#)

User-Defined Occupants:

[Get Results](#)

- ❑ The concept requires assumptions and we don't know the associated uncertainties

3

OUR AIM

Investigate the uncertainties associated with the CO₂-based ventilation metrics approach

4

METHODS

STEP 1:

□ Single-zone CONTAM simulations of CO₂ concentrations **over a range of input values:**

- Outdoor ventilation rates
- Occupant characteristics
 - Ratio of male to female
 - Body mass (kg)
 - Metabolic rates

STEP 2:

□ A two-level, full factorial design with **a 10-step exploratory data approach (EDA)** to identify:

- The most important factors
- The impact of each input value and interactions between the input values

Output

□ The sensitivity of these factors to the CO₂ concentration after **1 hour after full occupancy** and at **steady state**

5

MODEL INPUTS & DATA SOURCES

□ Single-zone CO₂ concentrations in each space:

$$V \frac{dC}{dt} = Q [C_{out} - C(t)] + G \implies C_{ss} = C_{out} + \frac{G}{Q}$$

□ Individual CO₂ generation rate:

$$V_{CO_2} = BMR * M * (T/P) * 0.000179$$

BMR = Basal metabolic rate ~ f(sex, age, body mass)

□ Volumetric flow of air in & out of the space

$$V_{bz} = R_p \times P_z \times R_a \times A_z$$

ASHRAE Standard 62.1 Ventilation Rate Procedure

DATA SOURCES

Occupant density values

ASHRAE Standards 62.1

Metabolic rates

Adults (Ainsworth et al., 2011)

Youth (Butte et al., 2018)

Body mass

2015-2018 NHANES survey data

6

BASELINE INPUT VALUES: OCCUPANT CHARACTERISTICS AND VENTILATION RATES

SPACE TYPE	OCCUPANT DENSITY (#/100 M ²)	MF RATIO	BODY MASS M/F (KG)	METABOLIC RATE (MET)	VENTILATION RATE (M ³ /H)
Classroom (5 y to 8 y)	25	12 : 12 (1 adult Teacher)	Students: 26.4 / 25.8 Teacher: 93.4 / 79.6	Student: 1.5 Teacher: 2.1	185
Lecture classroom	65	32 : 32 (1 adult Lecturer)	Students: 83.6 / 73.7 Lecturer: 93.4 / 79.6	Student: 1.7 Lecturer: 2.1	277
Restaurant dining room	70	Customer: 33:33 Workers: 2 : 2	93.4 / 79.6	Customer: 1.7 Server: 2.2	356
Conference meeting room	50	25 : 25	93.4 / 79.6	1.7	155
Office space	5	2.5 : 2.5	93.4 / 79.6	1.9	42.5
Active Lobby	150	75 : 75	93.4 / 79.6	2.2	405
Mellow Lobby	150	75 : 75	93.4 / 79.6	1.9	405
Retail	15	7.5 : 7.5	93.4 / 79.6	2.1	117

7

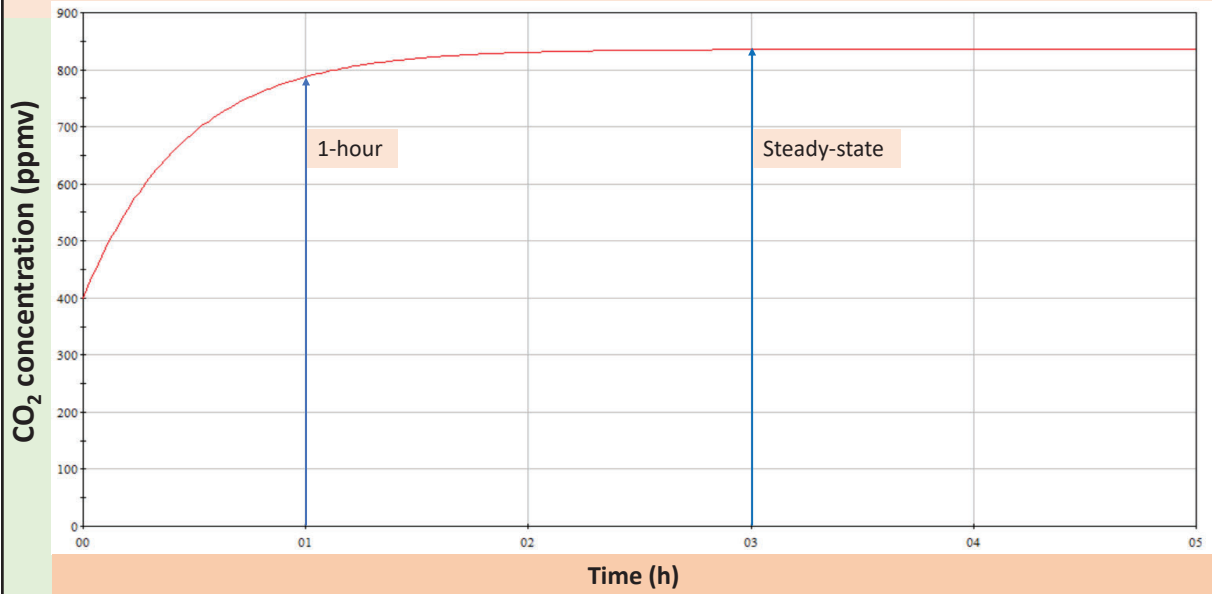
SENSITIVITY ANALYSIS

- ❑ Primary objective is to provide insight into the data, which includes a **graphical** component that can be interpreted more easily.
- ❑ Settings in a **two-level design**, indicating higher (+1) and lower values (-1).
- ❑ Number of simulation based on 2^k factorial design, where k = number of factors ; $2^k = 2^4 = \mathbf{16}$ **for each space**
- ❑ Baseline values of the studied inputs were varied by +/- 20 %.
20 % not based on specific data, but rather to represent realistic variation

X1 Male to female ratio	X2 Body mass	X3 Metabolic rate	X4 Outdoor ventilation rate	Y CO2 concentration
-1	+1	-1	+1	893
+1	-1	-1	+1	890

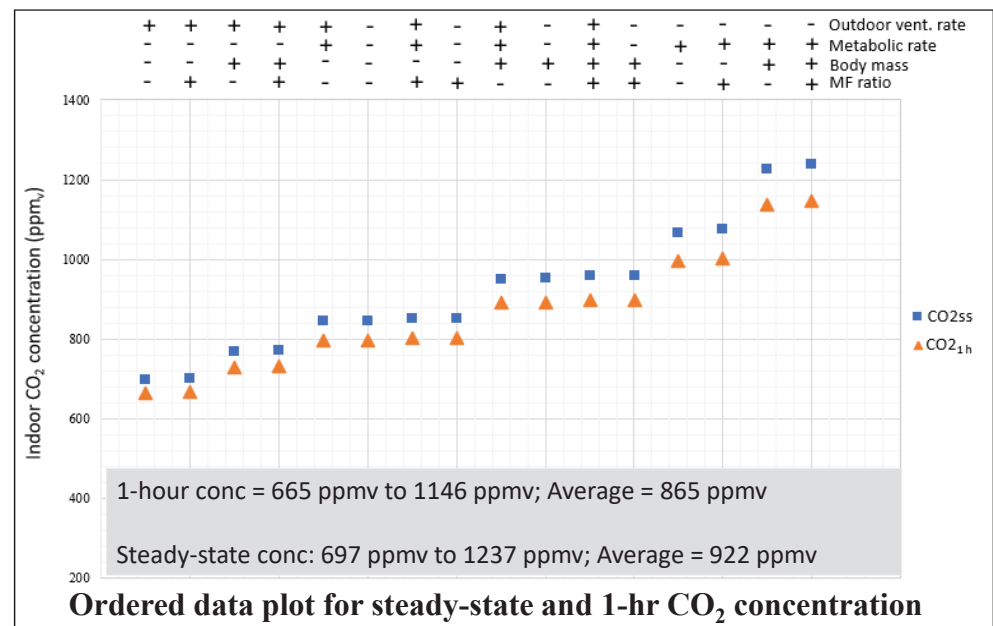
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SIMULATION RESULTS FOR CLASSROOM WITH STUDENTS 5-8 YEARS AND FEMALE TEACHER



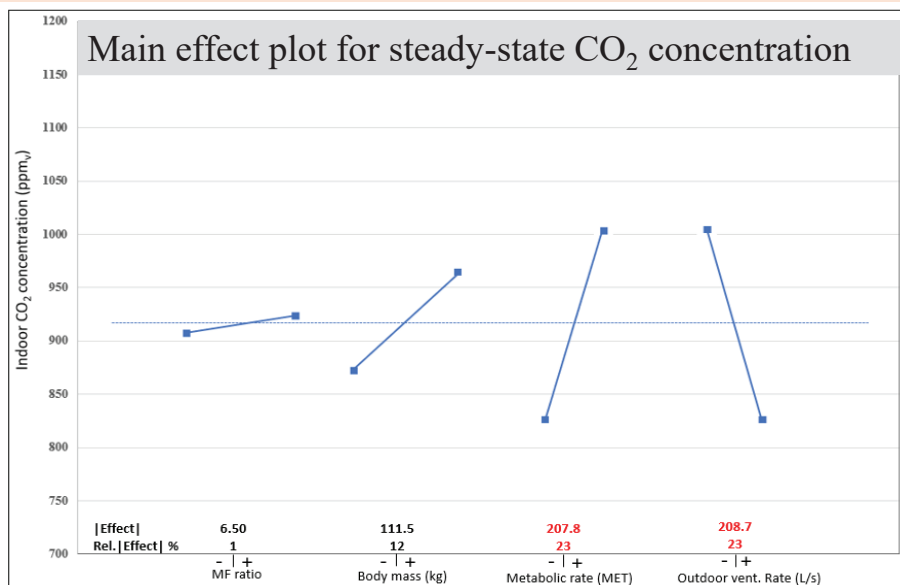
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5-8 YEAR CLASSROOM WITH FEMALE TEACHER



10

5-8 YEAR CLASSROOM WITH FEMALE TEACHER



Main effect = All higher values – all lower values

11

PERCENT DIFFERENCE IN 1-HOUR CONCENTRATIONS FOR +/- 20 % INPUT VARIATION

Space type	Minimum/maximum concentration (mean) ppm _v	MF ratio	Body mass	Metabolic rate	Ventilation rate
Classroom (5 y to 8 y, male teacher)	665/1146 (865)	1	11	21	22
Classroom (5 y to 8 y, female teacher)	659/1129 (855)	1	11	21	21
Lecture classroom (male lecturer)	1175/2825 (1846)	3	21	33	31
Lecture classroom (female lecturer)	1171/2812 (1838)	3	21	33	31
Restaurant dining room	1134/2509 (1700)	3	15	31	31
Conference meeting room	1380/3312 (2170)	4	16	35	33
Office space	595/953 (742)	2	9	17	18
Active Lobby	2146/5456 (3504)	5	18	35	35
Mellow Lobby	1855/4520 (2952)	5	17	33	35
Retail	761/1546 (1074)	7	13	28	25

- ☐ **Ventilation and metabolic rates** impact 1-hour and steady-state CO₂ concentrations more than **body mass and MF ratio**.
- ☐ Varying these two parameters by +/- 20 % results in variations in the CO₂ concentrations of about **20 % to 35 %**.

12

CONCLUSIONS

- The results of this analysis clarify potential range in CO₂ metric values based on variation in key input values
- Outdoor ventilation rates and metabolic rates** have most significant effects on CO₂ metric values.
- The **target ventilation rate** will generally be known with a higher degree of confidence based on the standard or guidance value one is attempting to verify
- The met rate for the space is inherently difficult to determine** as values in literature are based on specific activities; do not address met rates for spaces.
- The uncertainty in the CO₂ metric values associated with variations in met rates needs to be explicitly acknowledged and quantitatively considered.

13

THANK YOU!



14

Topical Session (6A)

The role of carbon dioxide and particulate matter for assessing ventilation and respiratory disease transmission in buildings

Evaluation of Uncertainties of Using CO₂ for Studying Ventilation Performance and Indoor Airborne Contaminant Transmissions

Presenter: Ibrahim Reda, Université de Sherbrooke

Authors: Liangzhu (Leon) Wang^{1,*}, Ibrahim Reda², Shujie Yan¹, Eslam Ali¹,

Dahai Qi², Theodore Stathopoulos¹, and Andreas Athienitis¹

¹Centre for Zero Energy Building Studies, Concordia University, Canada

²Department of Civil and Building Engineering, Université de Sherbrooke, Canada

Date: October 5, 2023
Time: 11:00 – 12:30

Room A

1

Contents

<ul style="list-style-type: none"> • Introduction 	<ul style="list-style-type: none"> • Objectives 	<ul style="list-style-type: none"> • Approach 	<ul style="list-style-type: none"> • Results 	<ul style="list-style-type: none"> • Conclusions
01	02	03	04	05

2

Background

Background

COVID-19 LIVE WORLD MAP/COUNT

October 4-5 2023

43rd IVC
11th TightVent & 8th Venticool Conference

WHO Coronavirus (COVID-19) Dashboard | WHO Coronavirus (COVID-19) Dashboard With Vaccination Data (99) | [LIVE] Coronavirus Pandemic: Real Time Dashboard, World Maps, Charts, News - YouTube

Multiple new variants.. then!?

COVID-19 era

Country	Total Cases	Deaths	Recovered	Active Cases
USA	695,452,047	6,926,553	653,838,170	35,487,324
INDIA	1,000,000,000	100,000	900,000,000	100,000,000
FRANCE	1,000,000	100,000	900,000	100,000
GERMANY	1,000,000	100,000	900,000	100,000
BRAZIL	1,000,000	100,000	900,000	100,000
S. KOREA	1,000,000	100,000	900,000	100,000
JAPAN	1,000,000	100,000	900,000	100,000
ITALY	1,000,000	100,000	900,000	100,000
UNITED KINGDOM	1,000,000	100,000	900,000	100,000
RUSSIA	1,000,000	100,000	900,000	100,000
CHINA	1,000,000	100,000	900,000	100,000
INDONESIA	1,000,000	100,000	900,000	100,000
SPAIN	1,000,000	100,000	900,000	100,000
TURKEY	1,000,000	100,000	900,000	100,000
NET TOTAL	235			

Challenges

Challenges

Uncertainties

- CO₂ monitoring
- Well-mixing tracer
- Mass-balance equation
- Multizone calibration

4

Objectives

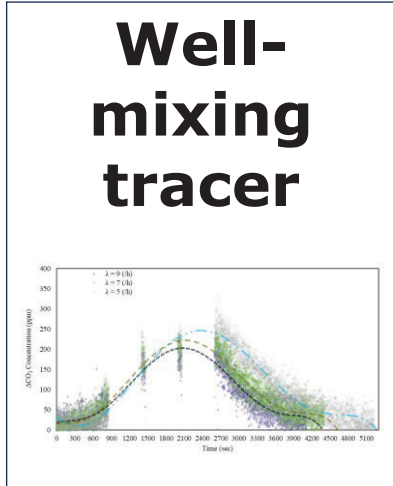


01

02

03

Objectives

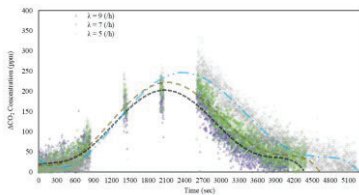


02

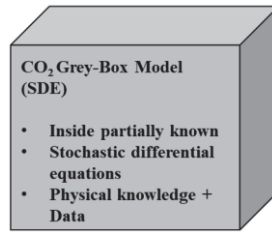
03

Objectives

Well-mixing tracer



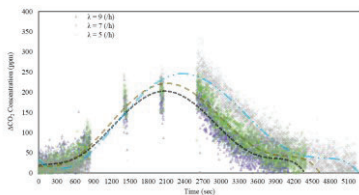
Stochastic grey-box model, integrating Bayesian inference



03

Objectives

Well-mixing tracer



02

03

Modified decay method (MDM)



Challenges

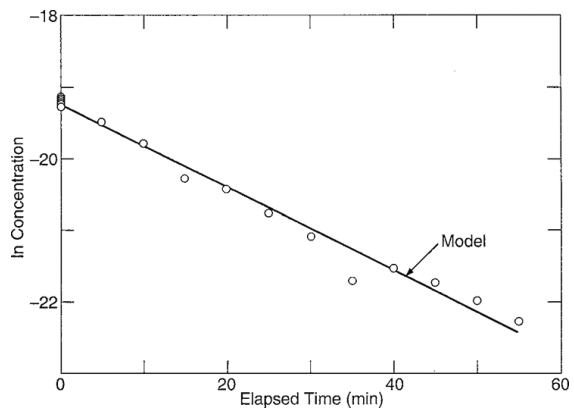


Decay method assumptions and limitations

- Stable and inert tracer.
- No adsorption processes.
- Single zone.
- Constant ventilation rate.
- **Well-mixed space.**

To what extent Well-mixed space is real?

$$\lambda = \frac{\ln C(t_2) - \ln C(t_1)}{t_2 - t_1}$$

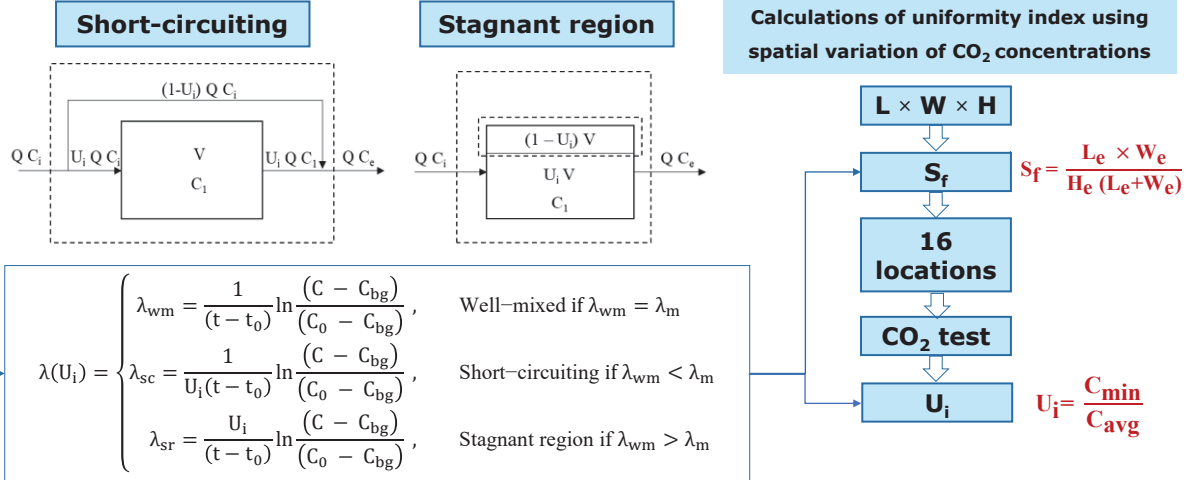


Regression method of estimating air change rates using decay method [ASTM E741].

Solution



Modified decay method and non-uniform mixed space (MDM)

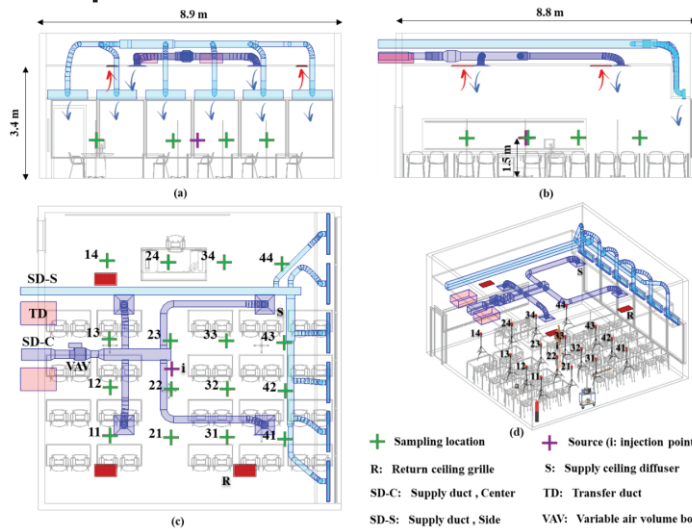


A Modified Decay Method Based on a Proposed Uniformity Index for Measuring Air Change Rates in Non-Uniform Air Mixed Spaces by Ibrahim Reda, Eslam Ali, Dahai Qi, Liangzhu (Leon) Wang, Theodore Stathopoulos, Andreas Athienitis :: SSRN

Case study



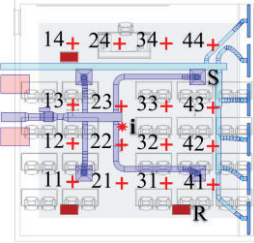
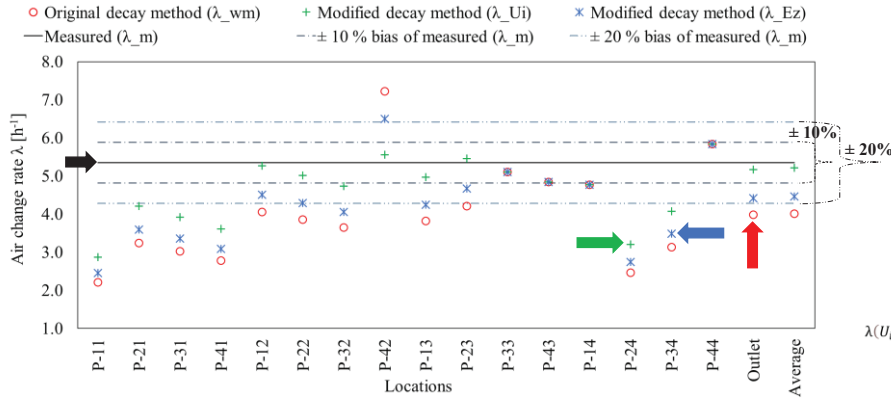
Test setup



Zone description and test setup at Longueuil campus, University of Sherbrooke



Validation of the modified decay method

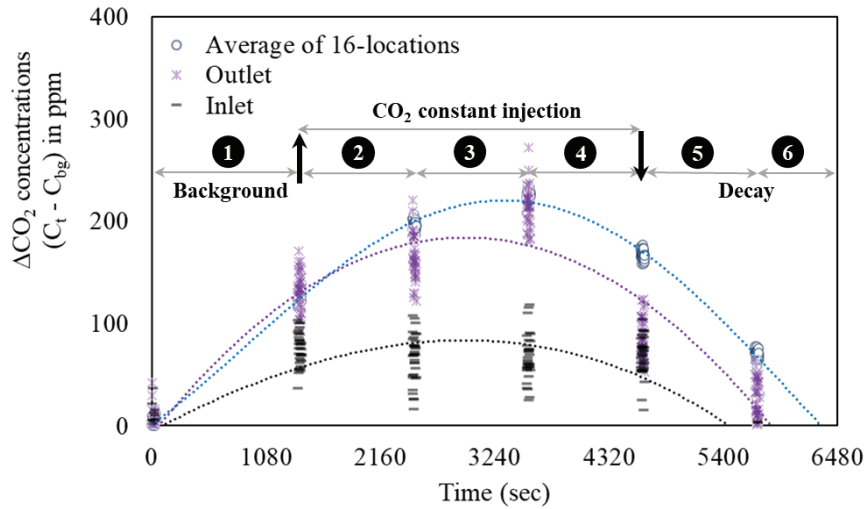


$$\lambda(U_i) = \begin{cases} \lambda_{wm} = \frac{1}{(t-t_0)} \ln \left(\frac{C-C_{bg}}{C_0-C_{bg}} \right), & \lambda_{wm} = \lambda_m \\ \lambda_{sc} = \frac{1}{U_i(t-t_0)} \ln \left(\frac{C-C_{bg}}{C_0-C_{bg}} \right), & \lambda_{wm} < \lambda_m \\ \lambda_{sr} = \frac{U_i}{(t-t_0)} \ln \left(\frac{C-C_{bg}}{C_0-C_{bg}} \right), & \lambda_{wm} > \lambda_m \end{cases}$$

λ_{U_i}	Estimated air change rate using modified decay method and uniformity index [1/h]		
λ_{w_m}	Estimated air change rate using original decay method [1/h]		
λ_{s_c}	Estimated air change rate for a non-uniform mixed space with a short-circuiting of the inlet to outlet location [1/h]		
λ_{s_r}	Estimated air change rate for a non-uniform mixed space with stagnant regions [1/h]		
U_i	Uniformity index	E_z	Zone distribution effectiveness
C	The indoor concentration of tracer gas [g/m ³]	C_{bg}	Background concentration [g/m ³]
C_0	The initial concentration of tracer gas [g/m ³]		

$$\lambda(E_z) = \begin{cases} \lambda_{wm} = \frac{1}{(t-t_0)} \ln \left(\frac{C-C_{bg}}{C_0-C_{bg}} \right), & \lambda_{wm} = \lambda_m \\ \lambda_{zsc} = \frac{1}{E_z(t-t_0)} \ln \left(\frac{C-C_{bg}}{C_0-C_{bg}} \right), & \lambda_{wm} < \lambda_m \\ \lambda_{zsr} = \frac{E_z}{(t-t_0)} \ln \left(\frac{C-C_{bg}}{C_0-C_{bg}} \right), & \lambda_{wm} > \lambda_m \end{cases}$$

Sampling representative



Conclusions



MDM

- [Redacted]
- [Redacted]
- [Redacted]
- [Redacted]

15

Conclusions



MDM

- For the original decay method:**
 - 70% of sampling locations have underestimated λ ($\epsilon = 59\%$)
 - 5% of sampling locations have overestimated λ ($\epsilon = 35\%$)
 - 25% of sampling locations have acceptable λ ($\epsilon = 11\%$)
- [Redacted]
- [Redacted]
- [Redacted]

16

Conclusions



MDM

For the original decay method:
70% of sampling locations have underestimated λ ($\epsilon = 59\%$)
5% of sampling locations have overestimated λ ($\epsilon = 35\%$)
25% of sampling locations have acceptable λ ($\epsilon = 11\%$)

For the modified decay method:
 ϵ decreased from 25.6% to 3.4% at the outlet
 ϵ decreased from 25.0% to 2.5% at the average concentrations

[Redacted]

[Redacted]

Conclusions



MDM

For the original decay method:
70% of sampling locations have underestimated λ ($\epsilon = 59\%$)
5% of sampling locations have overestimated λ ($\epsilon = 35\%$)
25% of sampling locations have acceptable λ ($\epsilon = 11\%$)

For the modified decay method:
 ϵ decreased from 25.6% to 3.4% at the outlet
 ϵ decreased from 25.0% to 2.5% at the average concentrations

For the proposed uniformity index (U_i):
 U_i is more effective than ASHRAE E_z in enhancing the accuracy of λ measurements, with an improvement ranging from 7.8% to 17.5%.

[Redacted]

Future needs



Generalizing the developed method (MDM)

CFD simulations and field tests for various geometrical and ventilation configurations will be performed to develop a referenced data (table) of uniformity index (U_i), which will be used for the proposed novel decay method.



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Behind project



National Research Council
NRC-CNRC
Canada

**Fonds de recherche
Nature et
technologies**

Québec



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Thanks! | Questions

**Corresponding author: leon.wang@concordia.ca*

Presenting author: ibrahim.reda@usherbrooke.ca



Effects of ventilation on airborne transmission: particle measurements and performance evaluation

Huijuan Chen, Caroline Markusson, Svein Ruud

RISE Research institutes of Sweden

2023-10-05

1

Introduction

- Aim: evaluate different ventilation configurations for control of airborne transmission in a room
 - Different set-ups, flow rates, affect ventilation effectiveness on airborne particle removal
- Focus on mixing ventilation
- Part of the Eureka international network project (2020 December - 2024 March)

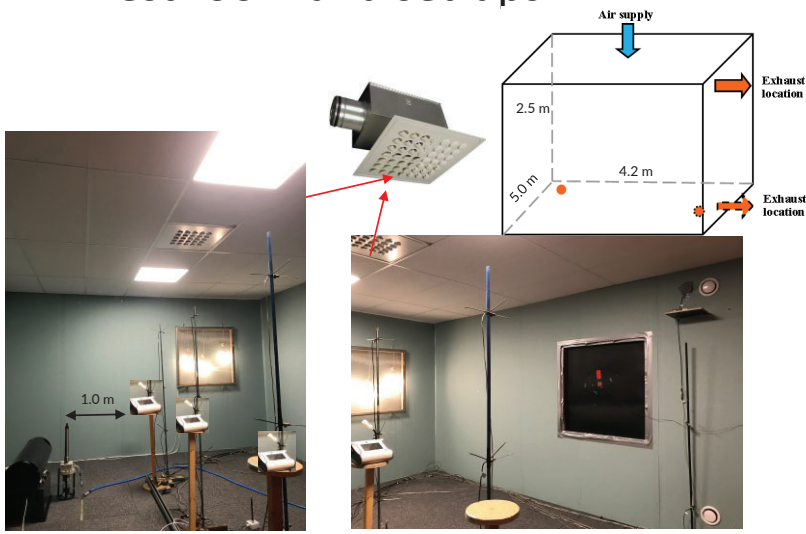
2

Method

- Test method: based on the concept of ventilation effectiveness adapted to aerosol particles.
 - Local ventilation effectiveness: $\varepsilon_p = (C_e - C_s)/(C_p - C_s)$
 - Particles with different size fractions
- Airborne particles up to $5\ \mu\text{m}$ which stay in air for a long time and follow strictly with ventilation air stream.

3

Test room and set-ups



Test room (elderly care home room)

- Size: 4.2 (L) x 5.0 (D) x 2.5 (H)
- Internal heat load
 - Dummy (77 W)
 - Simulated TV (120 W)
 - Lighting (2 * 30 W)
- Mixing ventilation
 - Air supply and exhaust position

Measured parameters

- Particle concentration (4) at 1.2 m above the floor
- Air temperature (PT 100)
 - Air supply, exhaust and inside the test room
 - 0.1, 0.6, 1.1 and 1.7 m
 - Airflow rate

Thermal comfort was NOT Studied.

4

Particle generation and measurement



Palas AGK 2000
KCL solution (NaCl)



- TSI OPS 3330
- 0.3-10 μm up to 16 channels
- 0.3-0.4, 0.4-0.55, 0.55-0.7, 0.7-1, 1-1.3, 1.3-1.6, 1.6-2.2, 2.2-3, 3.0-4.0, 4-5.5, 5.5-7, 7-10.0
- 4 particle counters

5

RISE - Research Institutes of Sweden

RISE

5

Ventilation flow rate

Ventilation level	Flow rate (l/s)
Low	8
Medium	15
High	30
Very high	40

- 8 l/s: The Swedish National Board of housing, Building and Planning (BBR) 0.35 l/s per m²
- 15 l/s: Swedish work environment authority 0.35 l/s per m² + 7 l/s per person
- 30 l/s: double the flow rate
- 40 l/s: even higher flow rate

6

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RISE

6

Measurement cases

Test nr.	Exhaust location	Flow rate (l/s)	Air discharge mode	Δt (room - supply) °C
1	High	8	Swirl	4.5±0.5 (4-5)
2		15		
3		30		
4		40		
5	Low	8	Swirl	4.5±0.5 (4-5)
6		15		
7		30		
8		40		
9		15	Swirl	4.5±0.5 (4-5)

Extra case: exhaust location in relation to ventilation exhaust, tested at 15 l/s

Measurement procedure

- Start ventilation and turn on heat sources and wait until getting stabilized flow and temperature field.
- Inject particles and taking the measurement data until get stabilized particle concentrations in the test room.
- Steady state measurement (1 hour).
- Correlation measurement to reduce relative bias among the particle counters
 - Turn off ventilation and head loads

Results

- Ventilation effectiveness

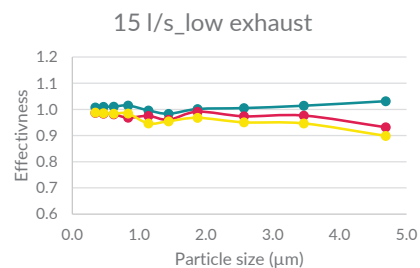
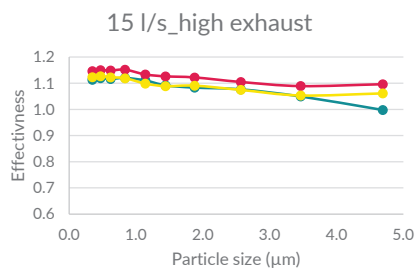
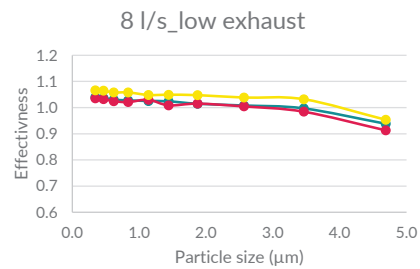
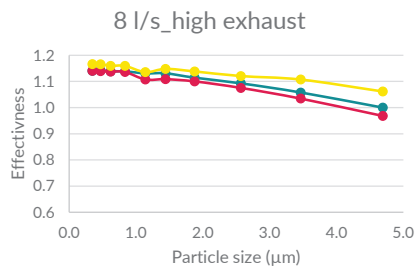
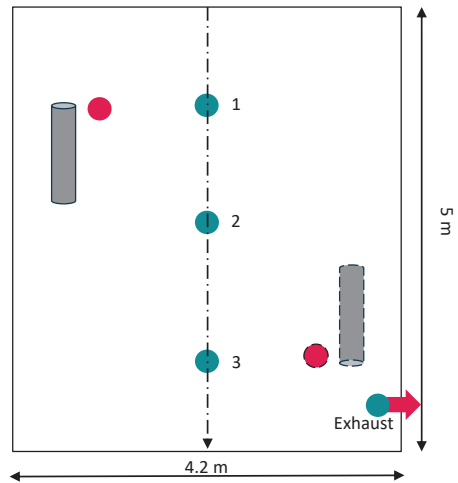
- $\epsilon_p = (C_e - C_s)/(C_p - C_s)$

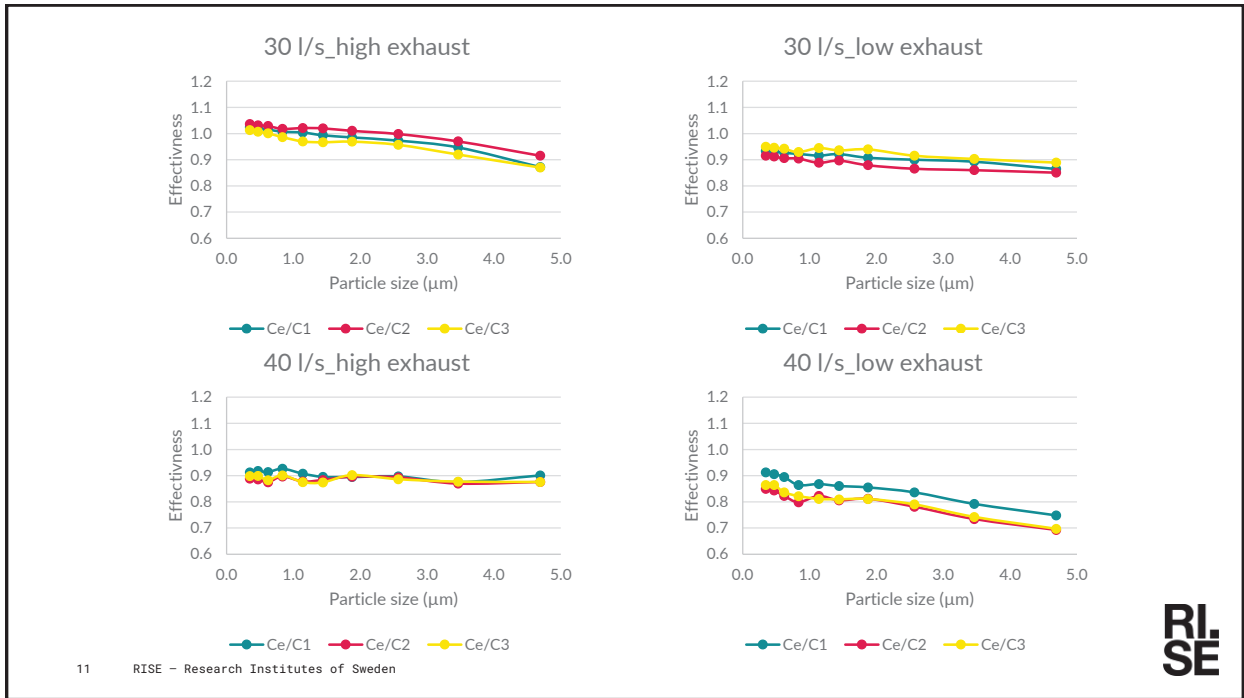
- $C_s = 0$

- $\epsilon_1 = C_e/C_1$

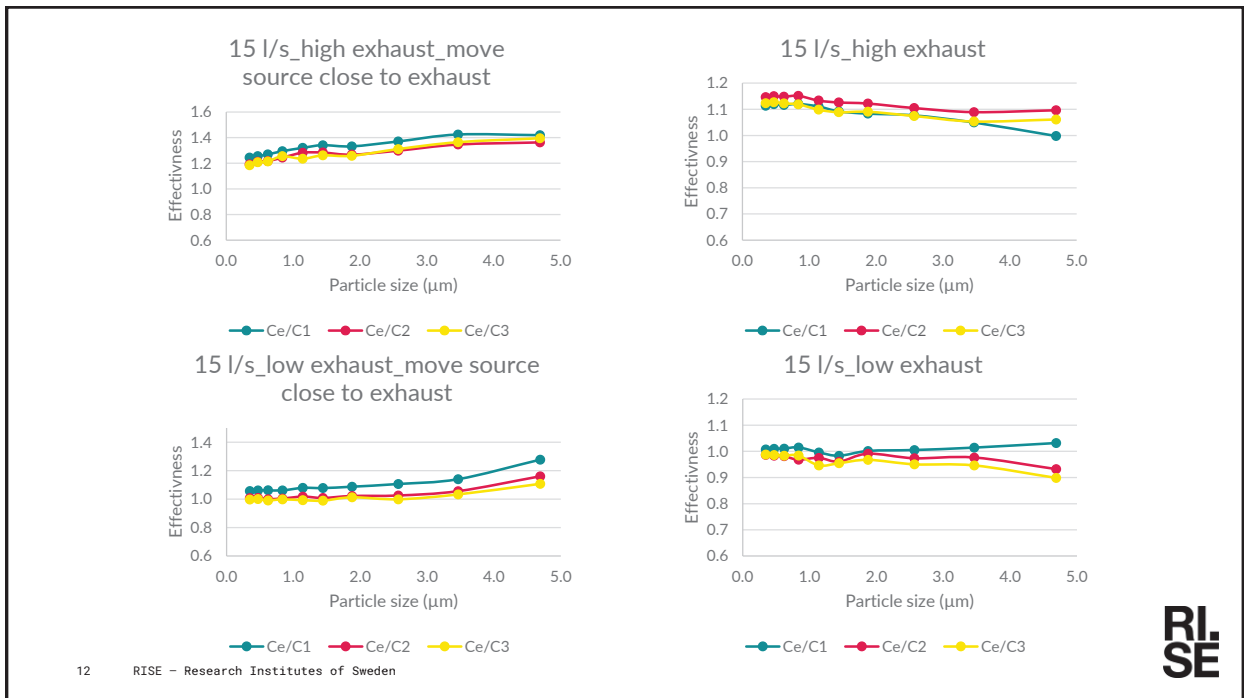
- $\epsilon_2 = C_e/C_2$

- $\epsilon_3 = C_e/C_3$





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12

Conclusions

- High exhaust provides better ventilation effectiveness than that with low exhaust.
- 8 and 15l/s, provide slightly better effectiveness which is due to displacement effect.
 - Not as bad as we thought, compensated by higher effectiveness.
- 30 l/s shows typical effectiveness for mixing ventilation (about 1 for smallest particles), while 40l/s shows less effectiveness due to short circuit.
 - Not as good as we believed due to risk of short circuit.
 - Better dilution effect.
- Balance between the contaminant dilution and removal should be considered when increasing the flow rates to enable sufficient and effective ventilation.
- Re-arranging room layout considering potential source locations could be a measure to reduce airborne transmission.

Thank you for your attention!

Huijuan Chen
Huijuan.chen@ri.se

Impact and benefits of the air cleaning measures in two Canadian schools

Prepared by: Liang Grace Zhou, Chang Shu, Justin Berquist, Greg Nilsson, and Janet Gaskin

Construction Research Centre, National Research Council Canada

Prepared for: AIVC 2023 Topical Session “The role of carbon dioxide and particulate matter for assessing ventilation and respiratory disease transmission in buildings”

October 5, 2023



National Research
Council Canada

Conseil national de
recherches Canada

Canada

1

Overview of the Controlled Intervention Study

A collaboration between the NRC and a Provincial Government

Two elementary schools were selected: each school has ~30 rooms and ~300 students and staff (Grade 1-8); one section with mechanical ventilation and one without; originally built in late 1950s.



Population: The students and staff.

Intervention: Portable air cleaners (PACs) with HEPA filters in each classroom and certain common areas (i.e. library).

Control: Baseline ventilation functioning as in the control school (without PACs).

Outcomes: PM1.0 and PM2.5 concentration; CO₂ level; power consumed by PACs; and filter efficiency; aggregated number of sick days.

Time: 14 weeks April-June 2023 (Control-14 weeks with baseline ventilation; Intervention – 6 weeks with baseline ventilation, followed by 8 weeks with PACs).

2

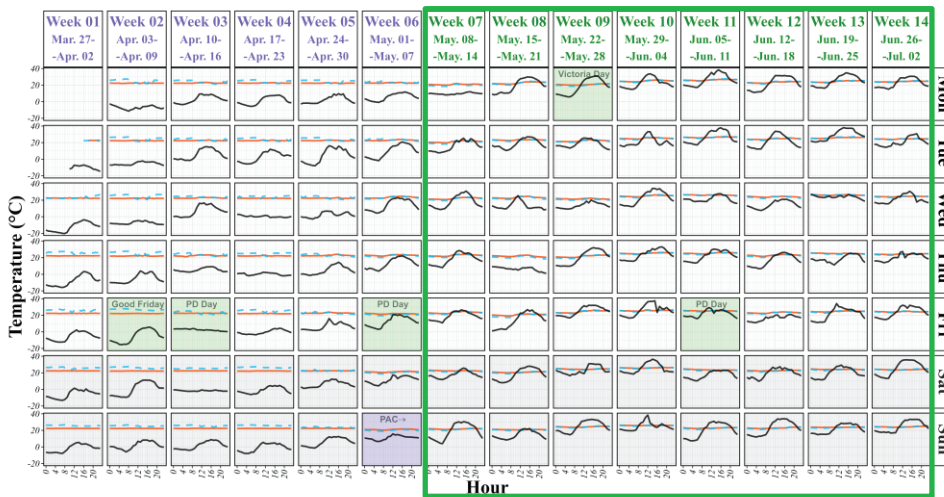
Equipment Installation



AQ sensors in classrooms, library, staff rooms, band rooms, hallways, and outdoors.

- Sound level with PAC on <54 dBA for all classrooms.
- Airflow rates of PACs were measured in each room (~2.5 ACH).
- 11 PACs were installed with a timer (08:00-17:00), 11 PACs were not controlled by a timer, on 24/7.
- Airflow rates from centralized HVAC system supply/return grills were measured.
- Plug power consumption, indoor/outdoor CO₂, PM1, PM2.5, PM10, RH, and T are continuously monitored.

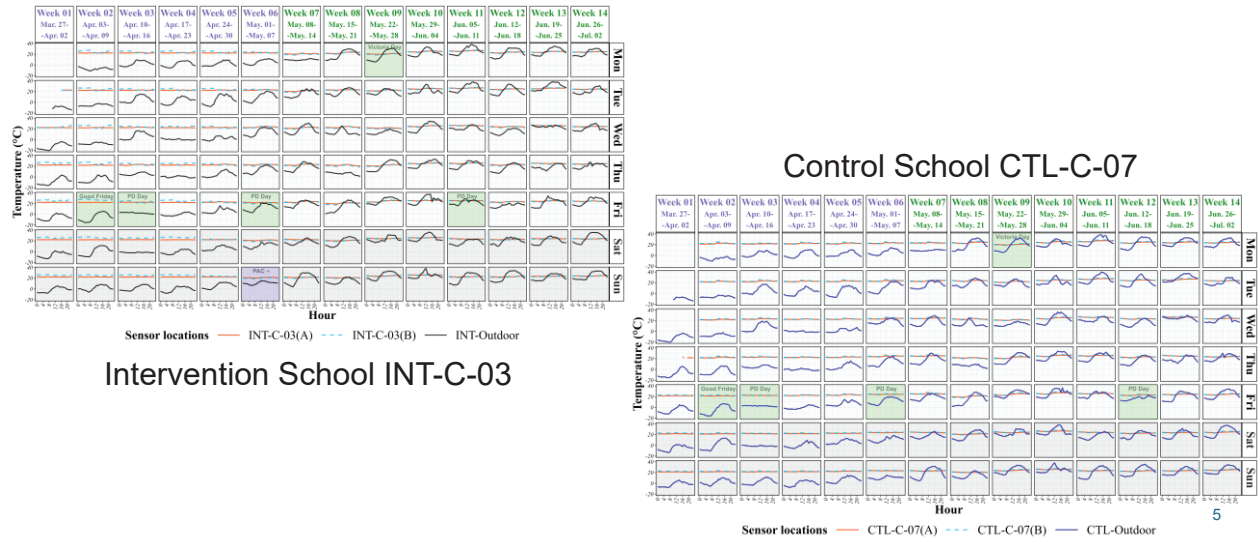
Temperature: a Classroom in Intervention School



Sensor locations — INT-C-03(A) — INT-C-03(B) — INT-Outdoor

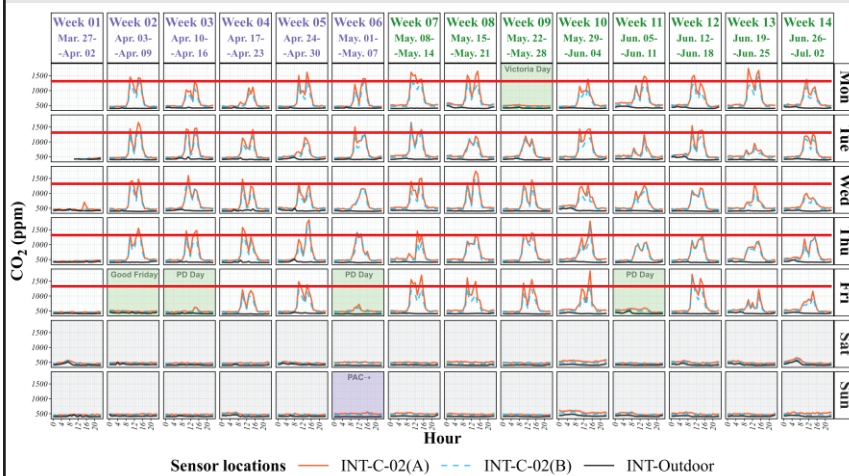
With PAC

Temperature: Intervention vs. Control



5

CO₂ Concentration: Intervention School Original Section, a Classroom with Windows



Space type	CO ₂ concentration above outdoors (ppm)	Time after occupancy (hr)
Classroom (> 9 years)	900	1

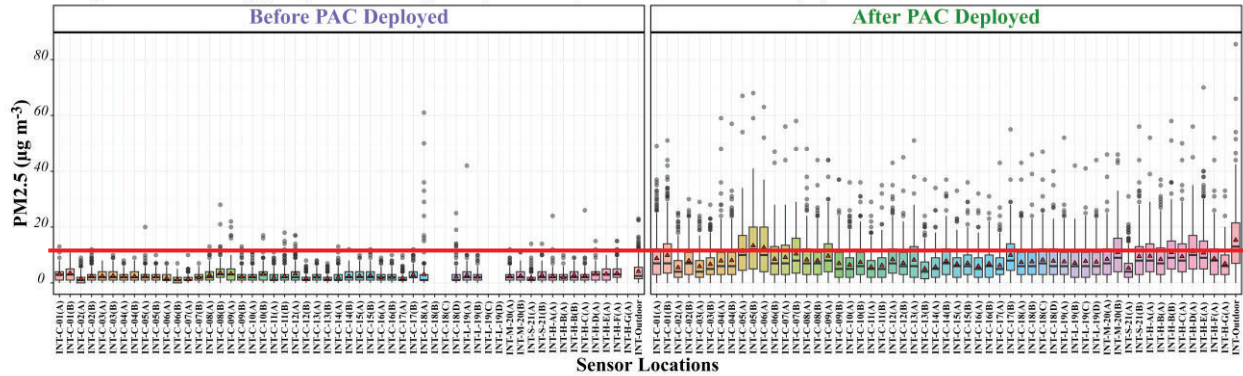
Potential CO₂ concentration metrics based on ASHRAE 62.1 ventilation requirements (Persily, 2022)
Outdoor CO₂ ~450 ppm

This classroom has 29 students, ages 11 to 12, 13 males and 16 females.

6

PM2.5 Concentration: Intervention School

Comparison of PM2.5 ($\mu\text{g m}^{-3}$) Levels During Occupied Hours, Before and After PAC Deployed

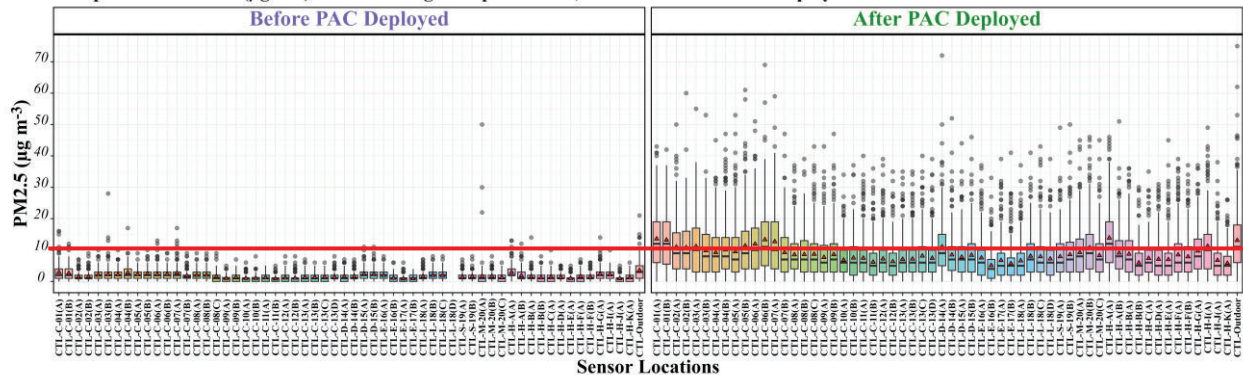


Design limits of PM2.5 – 12 $\mu\text{g/m}^3$ in ASHRAE 62.1



PM2.5 Concentration: Control School

Comparison of PM2.5 ($\mu\text{g m}^{-3}$) Levels During Occupied Hours, Before and After PAC Deployed



Design limits of PM2.5 – 12 $\mu\text{g/m}^3$ in ASHRAE 62.1

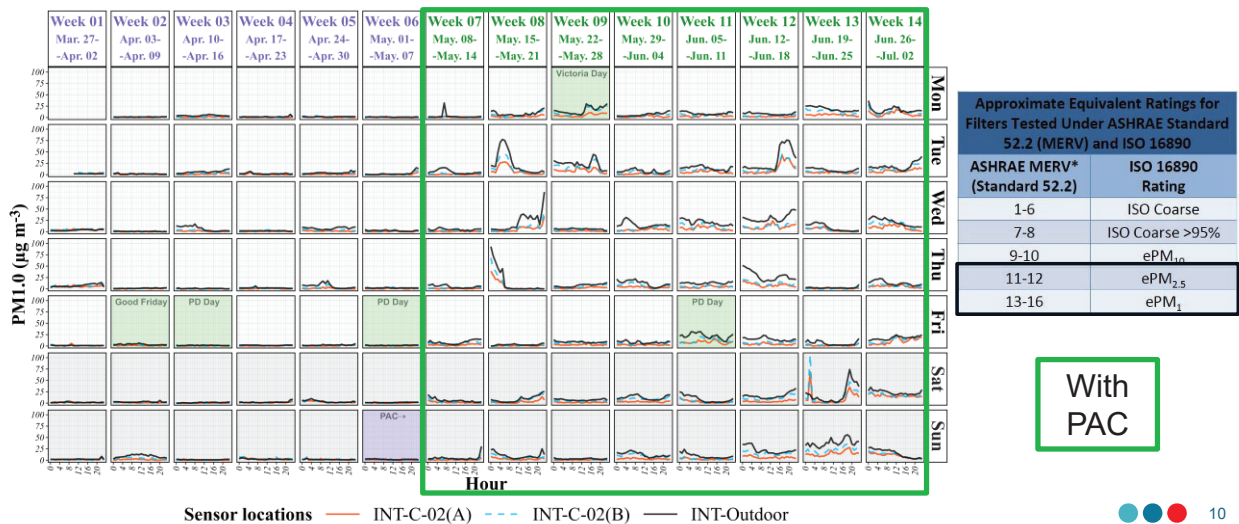


PM Concentration: Outdoor Sources

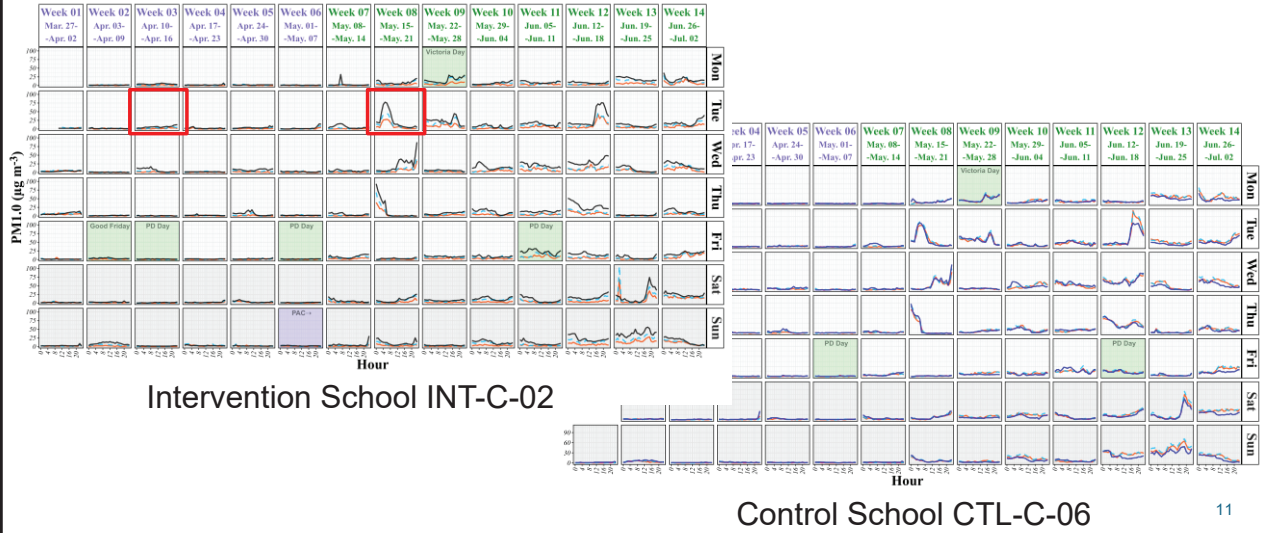


Occupants in the schools didn't smell or see any smoke!

PM1 Concentration: a Classroom with Windows in the Intervention School

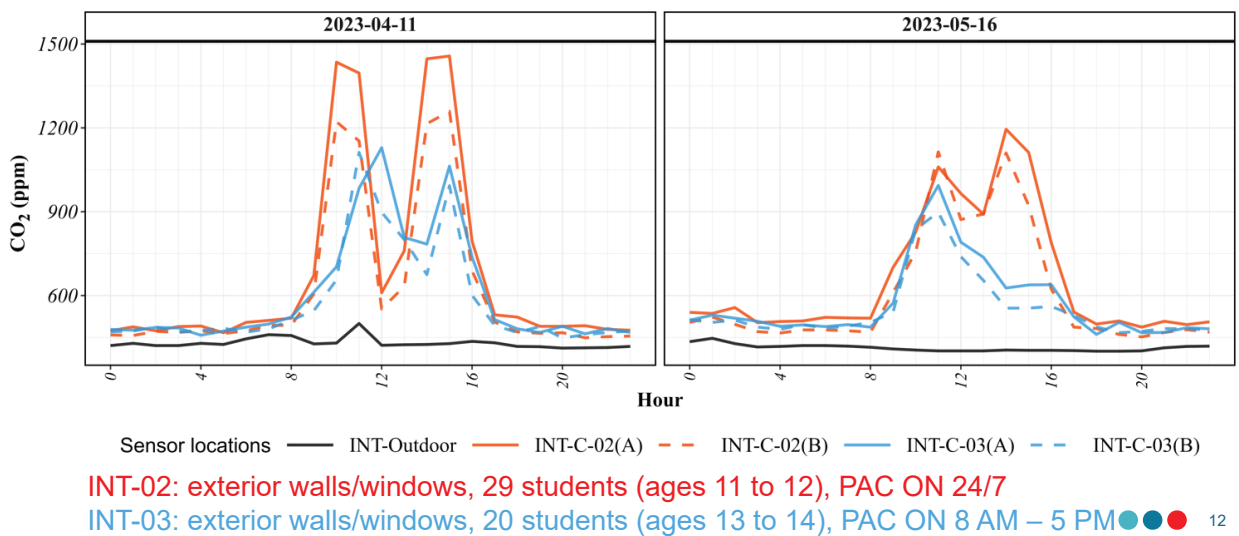


PM1 Concentration: Intervention vs. Control



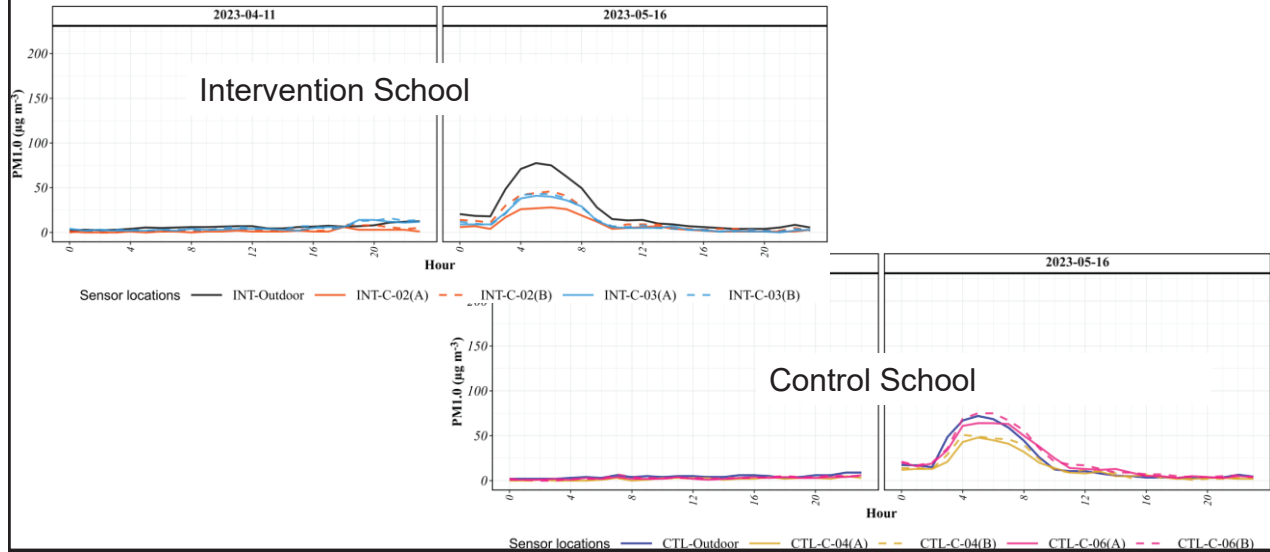
11

Impact of Occupancy on Indoor CO₂ Concentration



12

PAC's Ability to Reduce Indoor PM1 Concentration



13

PAC's Ability to Reduce Indoor PM1 Concentration

$$PM_{removal\ efficiency} = 1 - \frac{PM_{indoor}}{PM_{outdoor}}$$

School	Date	PAC (Y/N)	Outdoor PM1 conc (µg/m³)	Indoor PM1 conc (µg/m³)	Outdoor PM2.5 conc (µg/m³)	Indoor PM2.5 conc (µg/m³)	PM1 removal efficiency	PM2.5 removal efficiency
Control	April 11	N	4.53	1.54	6.70	1.70	0.66	0.74
Control	May 16	N	22.07	14.97	32.11	15.55	0.29	0.48

14

PAC's Ability to Reduce Indoor PM1 and PM 2.5 Concentrations

$$PM_{removal\ efficiency} = 1 - \frac{PM_{indoor}}{PM_{outdoor}}$$

School	Date	PAC (Y/N)	Outdoor PM1 conc (µg/m ³)	Indoor PM1 conc (µg/m ³)	Outdoor PM2.5 conc (µg/m ³)	Indoor PM2.5 conc (µg/m ³)	PM1 removal efficiency	PM2.5 removal efficiency
Control	April 11	N	4.53	1.54	6.70	1.70	0.66	0.74
Control	May 16	N	22.07	14.97	32.11	15.55	0.29	0.48
Intervention	April 11	N	6.22	2.16	8.81	2.31	0.65	0.74
Intervention	May 16	Y	24.65	9.26	34.07	9.69	0.61	0.70

PM1 removal efficiency in control school decrease from 0.66 on April 11 to 0.29 on May 16
 PM1 removal efficiency in intervention school decrease from 0.65 on April 11 to 0.61 on May 16



Observations

- The indoor CO₂ concentrations were dependent on the presence of occupants. The higher the occupant densities, the higher are the CO₂ concentrations.
- The CO₂ concentrations in both schools agreed with the predictions based on occupant characteristics and ASHRAE 62.1 ventilation requirements.
- The outdoor particle sources played the most significant role in deciding the indoor particle concentrations. The presence of exterior walls and windows in a space also affected the indoor particle concentrations.
- The measured PM1 and PM2.5 data with PAC units ON and OFF demonstrates the PAC units in the intervention school were able to remove some of the particles entered indoors.



Future Work

Efforts will be made to verify the cost and benefit of air cleaning and ventilation measures on IAQ and occupants' health in these schools:

- Sick days reported in both schools between April and June 2023 and feedbacks from staff and students (pleasant draft, perceived improvement in IAQ, and white noise)
- Long-term comparative tests: respiratory infection peak seasons and stable outdoor PM levels
- Power consumed by PACs
- PAC filter efficiency
- CO₂ decay after occupants leave the buildings
- Building envelope airtightness
- HVAC system's operating schedule and ventilation rate, in-duct filter efficiency
- Provide PAC manufacturer with feedback re. product improvement

 17

17

NRC-CNRC

 NRC.CANADA.NC

THANK YOU

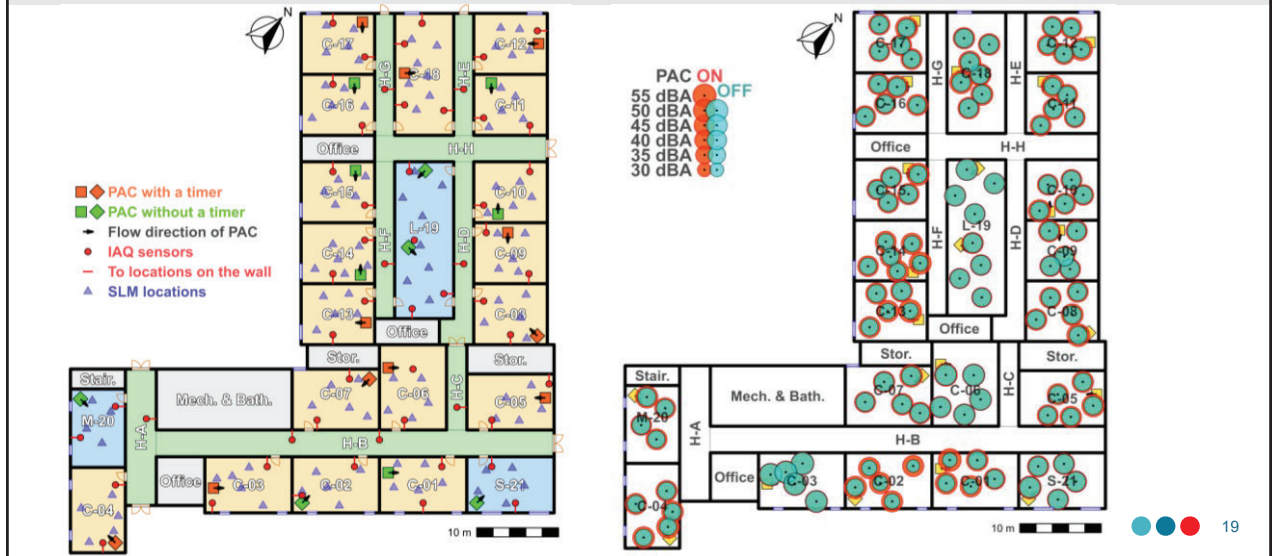
Grace Liang Zhou, Ph.D.
R&D Director - Building Performance and Quality
Senior Research Officer
Liang.zhou@nrc-cnrc.gc.ca

 National Research Council Canada
Conseil national de recherches Canada

Canada

18

Location of PACs, IAQ Sensors, Sound Level in Intervention School



19

Location of Sensors in Control School

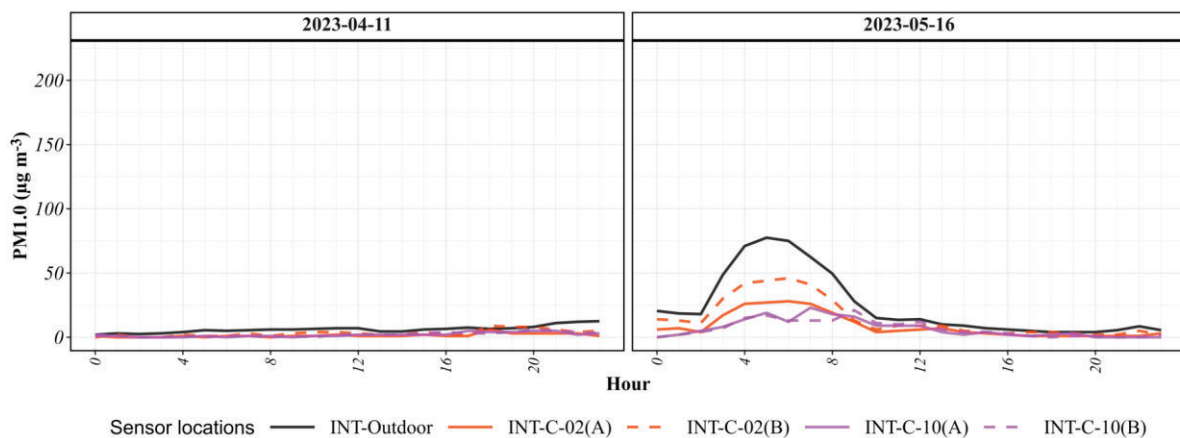


20

Parameters Considered for Data Analysis

- Day and time: minimize the opportunity for opening window and capture events with high outdoor particle concentrations
- Classroom occupant densities
- Classroom with and without exterior walls and windows: impact of leaks/openings through the building envelope on indoor air quality
- PACs controlled by timers (ON 8 AM – 5 PM) or not
- Define and use an index to minimize the impact of outdoor particle concentration on PAC's particle removal efficiency

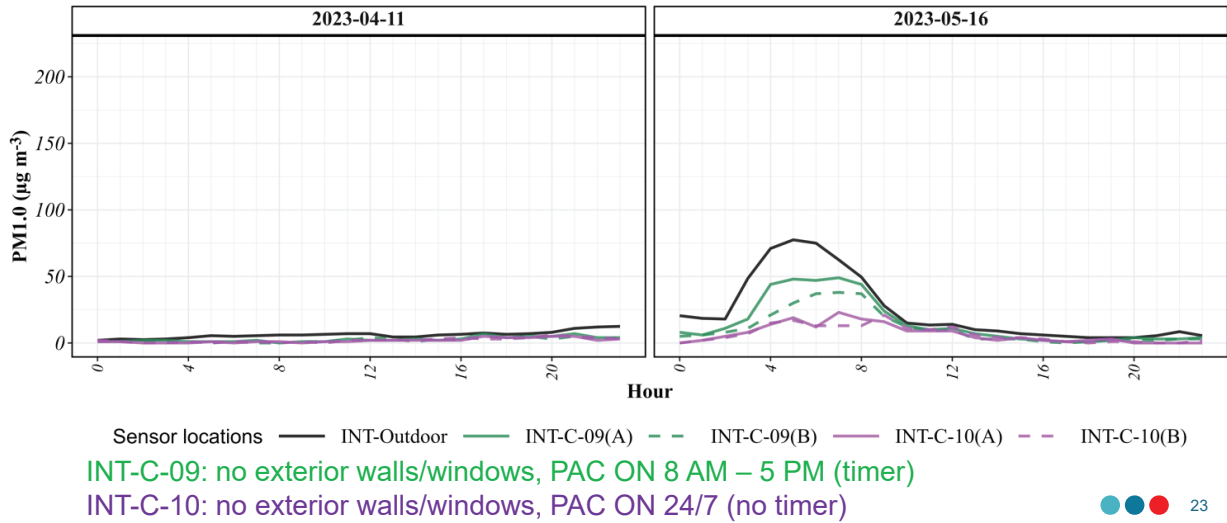
Impact of Exterior Walls and Windows on Indoor PM1



INT-C-02: exterior walls/windows, 29 students (ages 11 to 12), PAC ON 24/7

INT-C-10: no exterior walls/windows, 22 students (ages 6 to 7), PAC ON 24/7

Impact of PAC on Indoor PM1 Concentration



23

PAC's Ability to Reduce Indoor PM1

- Hourly PM1 data from the intervention and the control schools without PACs: weeks 2-6, hours 0:00-24:00

School and PAC Operation	A. Mean / G. Mean ($\mu\text{g}/\text{m}^3$)	Min - Max ($\mu\text{g}/\text{m}^3$)
INT NO PAC	1.3 / 0.18	0.0 - 16.0
CTL NO PAC	1.2 / 0.16	0.0 - 20.0

- Hourly PM1 data from the intervention school PACs ON vs. OFF: weeks 7-14, hours 18:00-7:00

INT PAC Operation	A. Mean / G. Mean ($\mu\text{g}/\text{m}^3$)	Min - Max ($\mu\text{g}/\text{m}^3$)
PACs ON	4.2 / 1.4	0.0 - 106.0
PACs OFF	8.9 / 3.7	0.0 - 101.0

- Hourly PM1 data from the intervention (PACs ON) and the control (PACs OFF) schools : weeks 7-14, hours 0:00-24:00

School and PAC Operation	A. Mean / G. Mean ($\mu\text{g}/\text{m}^3$)	Min - Max ($\mu\text{g}/\text{m}^3$)
INT PACs ON	4.9 / 1.4	0.0 - 106.0
CTL PACs OFF	10.0 / 4.2 ± 10.3	0.0 - 108.0

24

24

Critical reflections on indoor-environmental quality constructs

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1

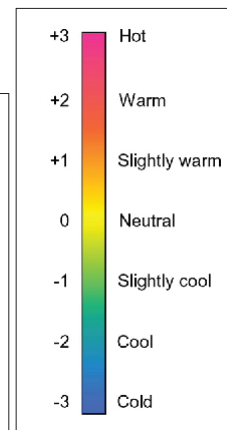
Constructs

... hypothetical variables used in experimental research to capture participants' experience of various aspects of IEQ (e.g., thermal comfort).



<https://cbe.berkeley.edu/wp-content/uploads/2018/09/ankle-draft-chamber-subjects-768x468.jpg>

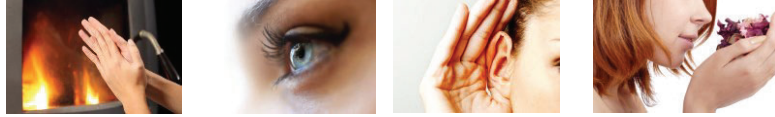
- Independent variables
- Proxies
- Indicators
- Metrics
- ...



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2

Constructs in IEQ research and standardisation

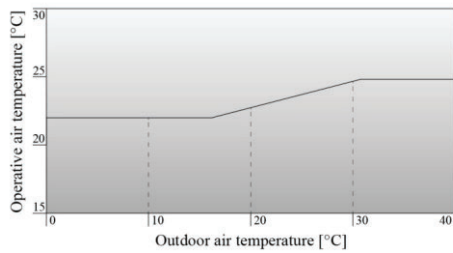


	Thermal	Visual	Auditory	Air Quality
Independent variables	Air and radiant temperatures, water vapor content, air velocity	Illuminance, luminance, contrast, colour temperature	Sound pressure level, reverberation time, frequency (spectrum)	CO ₂ and VOC concentration, Air change rate, Age of air
Constructs	Thermal sensation, thermal comfort	Visual comfort, glare rating	Loudness, annoyance	Air freshness

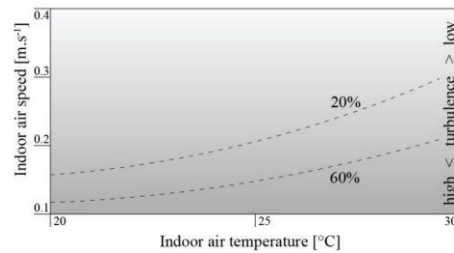
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3

Constructs in IEQ research and standardisation



Operative air temperature and its value as a function of the outdoor air temperature (DIN EN 15251, DIN EN ISO 7730)



Maximum permissible indoor air speed as a function of indoor air temperature for two turbulence intensity levels (DIN EN 15251, DIN EN ISO 7730)

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4

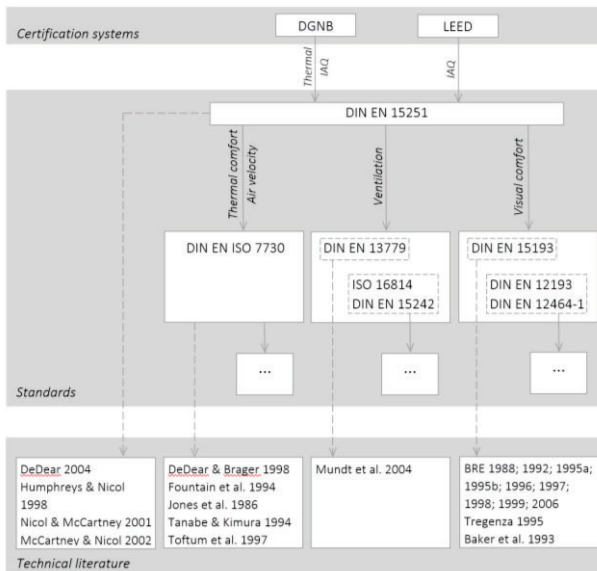
Constructs in IEQ research and standardisation



1. Berger, C., Mahdavi, A., et al. (2023). The role of user controls with respect to indoor environmental quality: From evidence to standards. *Journal of Building Engineering*, 76, 107196. <https://doi.org/10.1016/j.jobe.2023.107196>
2. Berger, C., Mahdavi, A., et al. (2023). Thermal Conditions in Indoor Environments: Exploring the Reasoning behind Standard-Based Recommendations. *Energies*, 16(4). <https://doi.org/https://10.3390/en16041587>
3. Berger, C., Mahdavi, A., et al. (2022). Reflections on the Evidentiary Basis of Indoor Air Quality Standards. *Energies*, 15(20). <https://doi.org/https://10.3390/en1520772>
4. Chinazzo, G., Berger, C., Mahdavi, A. (2022). Quality criteria for multi-domain studies in the indoor environment: Critical review towards research guidelines and recommendations. *BUILDING AND ENVIRONMENT*, 226, 109719. <https://doi.org/https://10.1016/j.buildenv.2022.109719>
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7. Schweiker, M., Berger, C., Mahdavi, A. (2020). Review of multi-domain approaches to indoor environmental perception and behaviour. *BUILDING AND ENVIRONMENT*, 106804-106804. <https://doi.org/https://10.1016/j.buildenv.2020.106804>

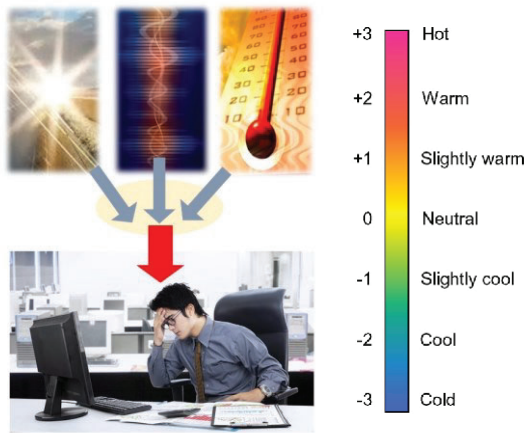
5

In pursuit of evidence...



6

Multi-domain comfort and behavior studies



"... At times, a different number of points and different labels were used, even though the same assessment category was involved. This, as well as the inconsistent use of dimensions in analogue scales, disables the comparison of results from different studies and poses a problem for conducting large-scale meta-analyses."

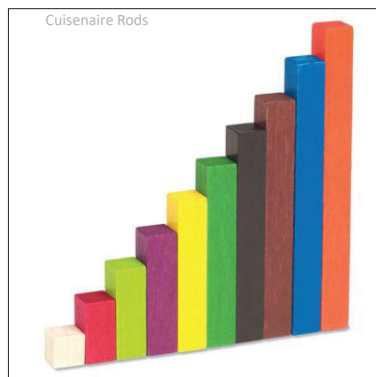
Chinazzo et al. 2022

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7

A fundamental conceptual challenge: Two kinds of measurements

- Representational measurements
- Pragmatic measurements

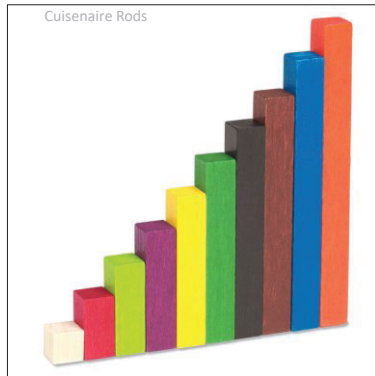


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8

A fundamental conceptual challenge: Two kinds of measurements

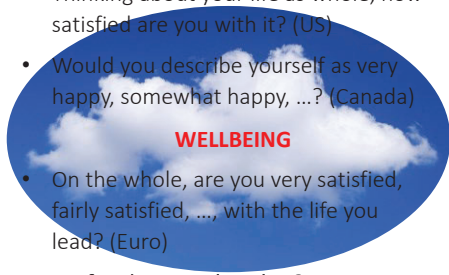
- Representational measurements



D. J. Hand: End to end concatenation and addition

- Pragmatic measurements

- Thinking about your life as whole, how satisfied are you with it? (US)
- Would you describe yourself as very happy, somewhat happy, ...? (Canada)
- On the whole, are you very satisfied, fairly satisfied, ..., with the life you lead? (Euro)

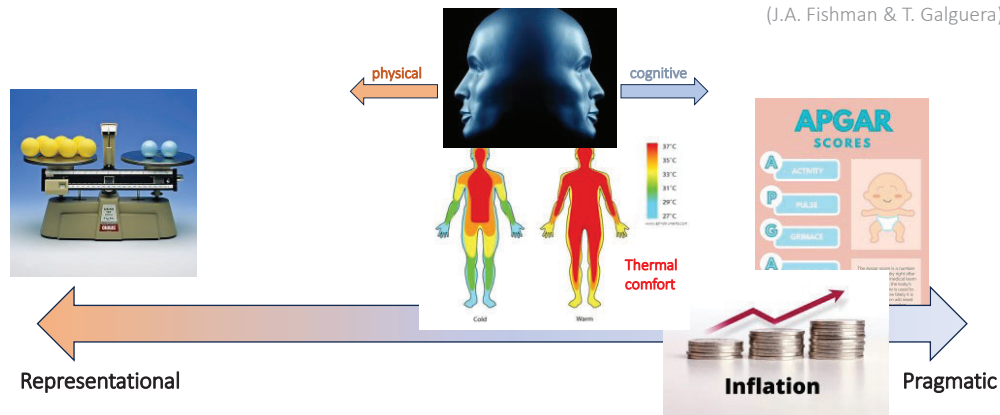


No fundamental reality?

Measurement validity and reliability

... unless one can be sure that an instrument will produce the same results on different applications, one cannot know whether it measures what it purports to measure.

(J.A. Fishman & T. Galguera)



Key takeaways

- Consistency, robustness, accuracy of constructs → reproducibility of findings, possibility of cumulative knowledge, dependability of standards.
- To this end, validation procedures for constructs are essential.

Yet:

- Existing IEQ research barely engages in construct validation: An ongoing review of 30 recent peer-reviewed publications on multi-domain studies shows that none of them included a construct validation effort!

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11

Constructs validations methods and procedures

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- Patten, M.L. (2000). *Understanding research methods* (2nd ed.). Thousand Oaks, CA: Sage.
- Peterson, R.A. (1999). *Constructing effective questionnaires*. Thousand Oaks: Sage.



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12

Critical reflections on indoor-environmental quality constructs

**Thank you for
your attention!**

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Ventilation and sleep quality

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October 4-5, 2023
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1

ASHRAE research project 1837-RP on “The effects of ventilation in sleeping environments”

- Summary of standards defining bedroom conditions
- Summary of literature on ventilation and sleep quality
- Cross-sectional studies in bedrooms to characterize ventilation conditions
- Intervention studies in bedrooms

Specific aim

- Amendment to ASHRAE Standard 62.2 “ Ventilation and Acceptable Indoor Air Quality in Residential Buildings”

2

Additional studies (extensions)

- Survey of bedroom conditions (prior to and after the COVID-19 lockdown)
- Laboratory studies on the effects of ventilation, pure CO₂, temperature and ventilation noise on sleep quality
- Examining bedroom and door opening behavior
- Estimation of CO₂ generation rate for sleeping people (young adults and elderly)
- Estimation of emission rates of bioeffluents from sleeping people using PTR-MS-TOF
- Comparison of performance of different sleep trackers against polysomnograph (PSG)

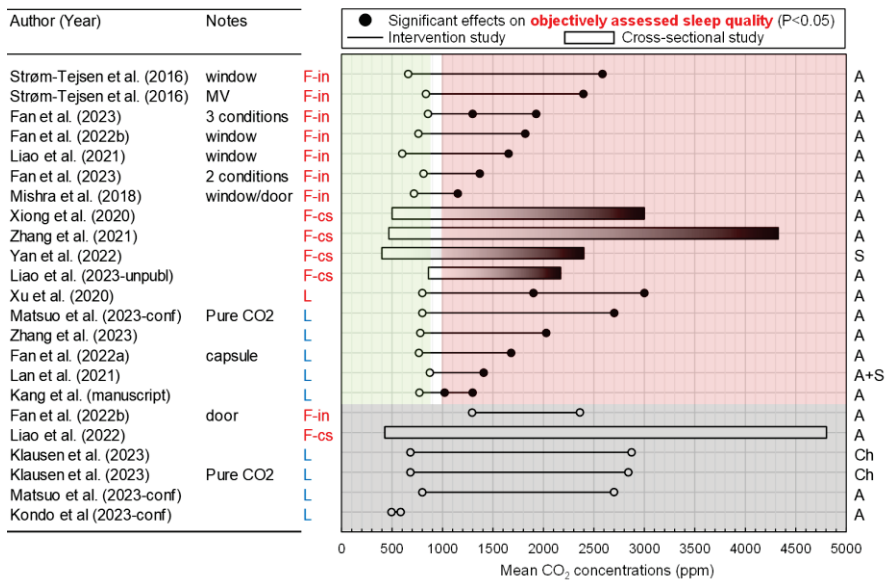


3

How much ventilation is needed in bedrooms?

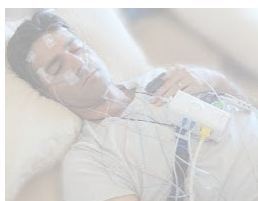
4

Summary of studies examining the effects of ventilation on sleep quality measured objectively



5

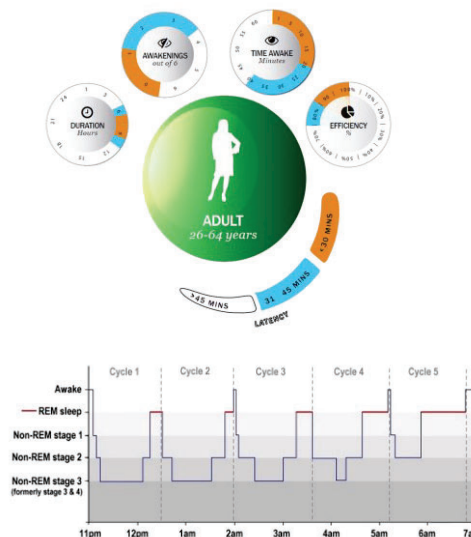
How to objectively measure sleep quality?



Polysomnography

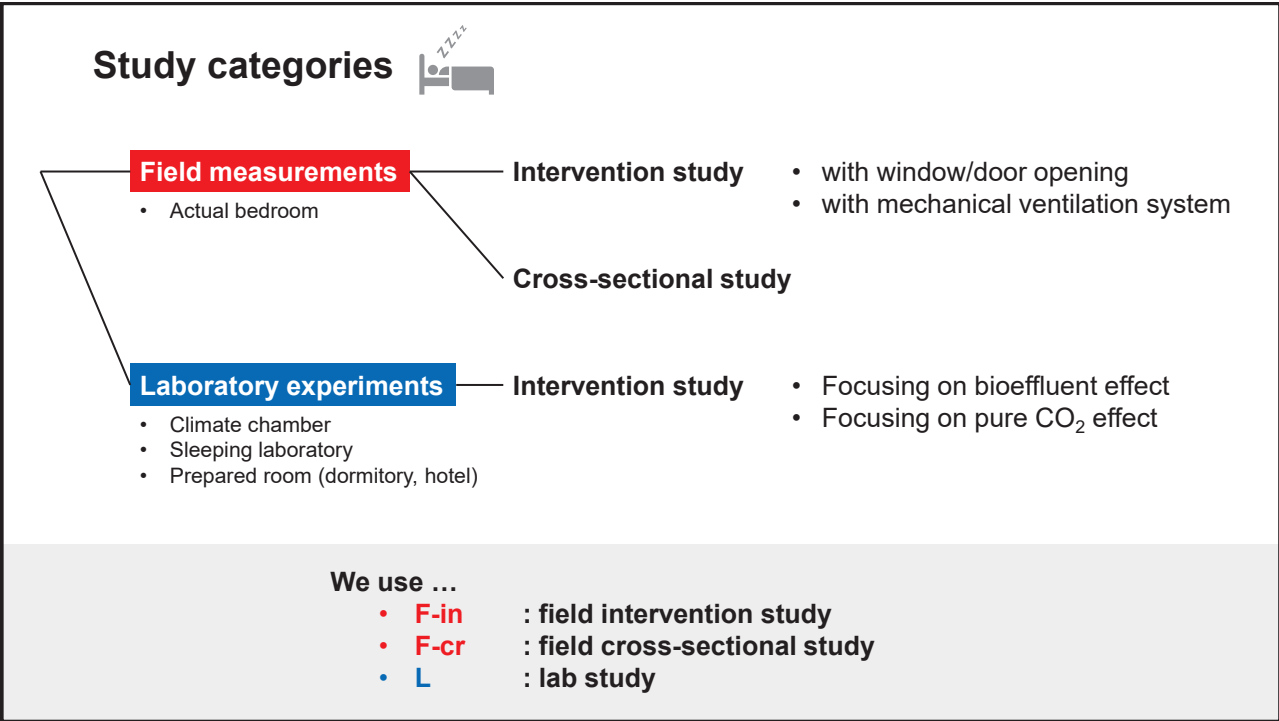


Actigraphy sleep trackers

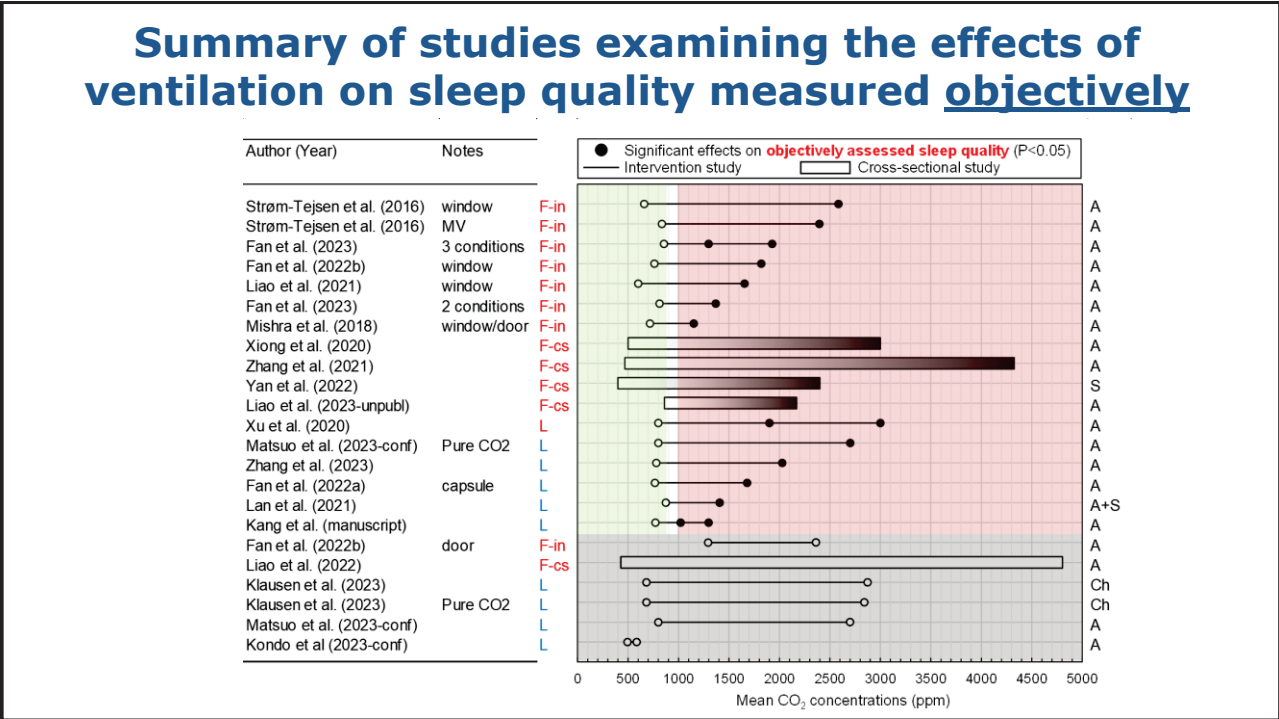


Source: National Sleep Foundation's sleep quality recommendations (2017)

6

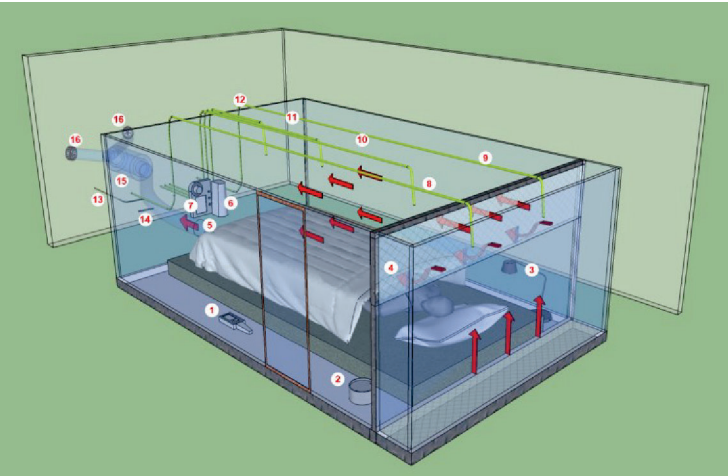


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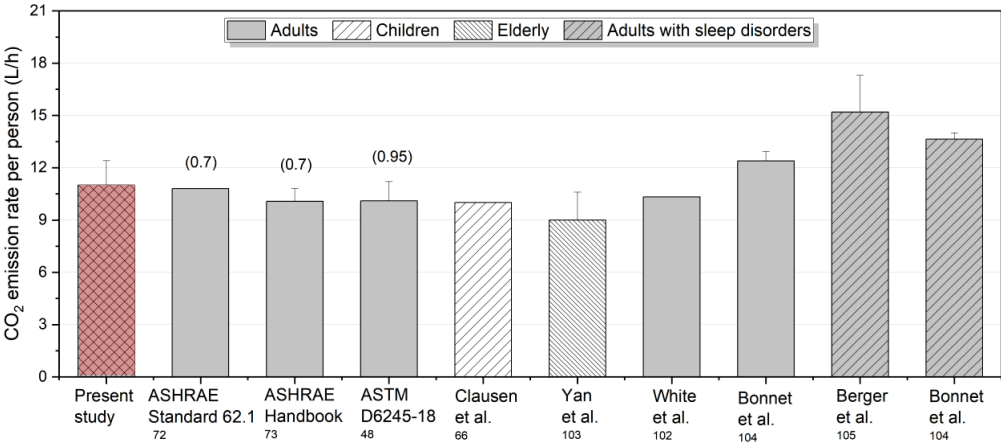
8

Estimating the emission rate of CO₂



9

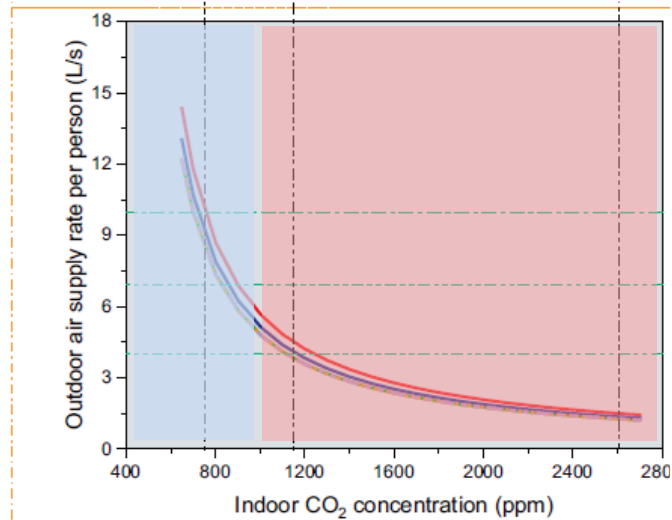
CO₂ emission rates



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Implications

- Current evidence suggest the rates above 10 L/sp (CO₂ levels below 750-800 ppm) will ensure undisturbed sleep
- This would correspond to about 1 h⁻¹



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Bedroom ventilation, standards and measurements: the review

- Wide range of ventilation rates measured in bedrooms using different methods; mostly carbon dioxide concentration and air change rates were measured.
- Reported mean CO₂ concentrations ranged from 428 to 2585 ppm.
- Bedrooms with NV during heating season have highest CO₂ levels and highest temperatures during cooling season
- Reported mean mean air change rates from 0.2 to 4.9 h⁻¹.
- The lowest observed during heating season and in the naturally ventilated bedrooms
- Most existing ventilation standards do not prescribe specific ventilation requirements for bedrooms - ventilation in bedrooms is merely the result of ventilation requirements for the entire dwelling

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Bedroom ventilation: Review of existing evidence and current standards

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ARTICLE INFO

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 Ventilation
 CO₂
 Air exchange rate
 Sleep quality
 Standards

ABSTRACT

Sleep is essential for our health and well-being. Some research suggests that air quality influences sleep quality in bedrooms, but the evidence is limited. Research, until now, has focused on how indoor air quality affects health, comfort, and cognitive performance during waking hours. Less information is available on the levels of indoor air quality and ventilation in bedrooms, or on their consequences for sleep quality and next-day performance. This paper addresses the former by reviewing research published in peer-reviewed journals in this millennium. Bedroom ventilation has been chosen as a specific focus of this review paper, which also includes a review of selected international standards for bedroom ventilation. Aiming out of this review and a comparison of field data with CO₂ and ventilation benchmarks from widely adopted international standards, an attempt is made to generalize the level of bedroom ventilation that exists in practice in residential dwellings and apartments across different seasons and different parts of the world. Based on a limited number of studies dealing with the impact of bedroom ventilation on sleep quality, an attempt is also made to associate measured field data with its potential impact on sleep quality.

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RESIDENTIAL VENTILATION REQUIREMENTS DENMARK

- BR18: 0.3 L/sm²
- DS447 (2021): 0.42 L/sm² (Cat. II)
- Typical Danish bedrooms (Bolius, 2020): 5-20 m² (85%), of which 9-16 m² (61%); average 13.4 m²; average house size is 109 m² with 2 people
- Outdoor air supply rate per bedroom from 2.7 to 4.8 L/s (BR) or 3.8 to 6.7 L/s (DS447, 2021)
- If 0.5 h⁻¹, then (if height 2.5 m) 3.1 to 5.6 L/s

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RESIDENTIAL VENTILATION REQUIREMENTS NORDIC COUNTRIES

- Somewhat unclear and inconsistent
- Around 5 L/s per bedroom
- Except Norway 7.2 L/s per person in bedroom
- Estonia defines 10 L/s per bedroom in major renovations
- Finland defines 4-6 L/s per person for the entire dwelling (or bedroom?), unclear
- According to the present data, ventilation rate shall be >6 L/s per person (<1,000 ppm CO₂)

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BEDROOM VENTILATION REQUIREMENTS US STANDARD 62.2 (2022)

Floor Area, m ²	Bedrooms				
	1	2	3	4	5
<47	14	18	21	25	28
47 to 93	21	24	28	31	35
94 to 139	28	31	35	38	42
140 to 186	35	38	42	45	49
187 to 232	42	45	49	52	56
233 to 279	49	52	56	59	63
280 to 325	56	59	63	66	70
326 to 372	63	66	70	73	77
373 to 418	70	73	77	80	84
419 to 465	77	80	84	87	91

Floor area (m ²)	Bedrooms (L/s/bedroom)				
	1	2	3	4	5
<47	0,8	1,0	1,2	1,5	1,6
47-93	1,2	1,4	1,6	1,8	2,0
94-139	1,6	1,8	2,0	2,2	2,4
140-186	2,0	2,2	2,4	2,6	2,8
187-232	2,4	2,6	2,8	3,0	3,2
233-279	2,8	3,0	3,2	3,4	3,7
280-325	3,2	3,4	3,7	3,8	4,1
326-372	3,7	3,8	4,1	4,2	4,5
373-418	4,1	4,2	4,5	4,6	4,9
419-465	4,5	4,6	4,9	5,0	5,3

US average house: 2273 sq ft. (211 m²)
(median: 2500 sq ft.=232 m²)
Wet-rooms ca. 100 sq ft
US average bedroom: 132 sq ft. (12 m²)
% bedroom floor size (no wet-rooms): 6%

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RESIDENTIAL VENTILATION REQUIREMENTS CHINA

- GB50736 (2012), design code: minimum ACH (h⁻¹) depending on floor size in residential building

Fp - per capita living space (floor area per person)	ACH	L/s
Fp≤10m ²	0.70	<4.9
10m ² <Fp≤20m ²	0.60	4.2-8.3
20m ² <Fp≤50m ²	0.50	6.9-17.4
Fp>50m ²	0.45	>15.6

- GB/T 18883 (2022), air quality standard: 30 m³/hp (8.3 L/sp) and CO₂ ≤ 1,000 ppm
- JGJ 134 (2010), design std. or energy efficiency for hot summer and cold winter regions: 1.0 h⁻¹ (heating or air conditioning)
- JGJ26 (2018), design std. for energy efficiency in severe cold and hot zones: 0.5 h⁻¹

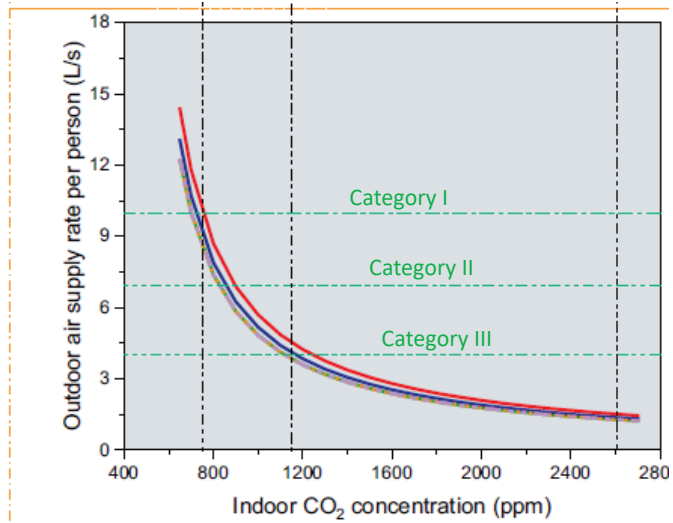
16

BEDROOM VENTILATION REQUIREMENTS EUROPEAN STANDARD EN16798-2020

Category	Design ACO ₂ concentration for living rooms (ppm above outdoors)	Design ACO ₂ concentration for bedrooms (ppm above outdoors)
I	550	380
II	800	550
III	1 350	950
IV	1 350	950

NOTE 1 The above values in Table B.12 correspond to the equilibrium concentration when the air flow rate is 4, 7, 10 l/s per person for cat. I, II, III respectively and the CO₂ emission is 20 l/h per person and 13,6 l/h per person for living rooms and bedrooms respectively.

NOTE 2 For a 10 m² room (room height 2.5 m, 25 m³) 4, 7 and 10 l/s per person correspond, with two persons in the room, to an air change rate of 1,2; 2,0 and 2,9 ACH.



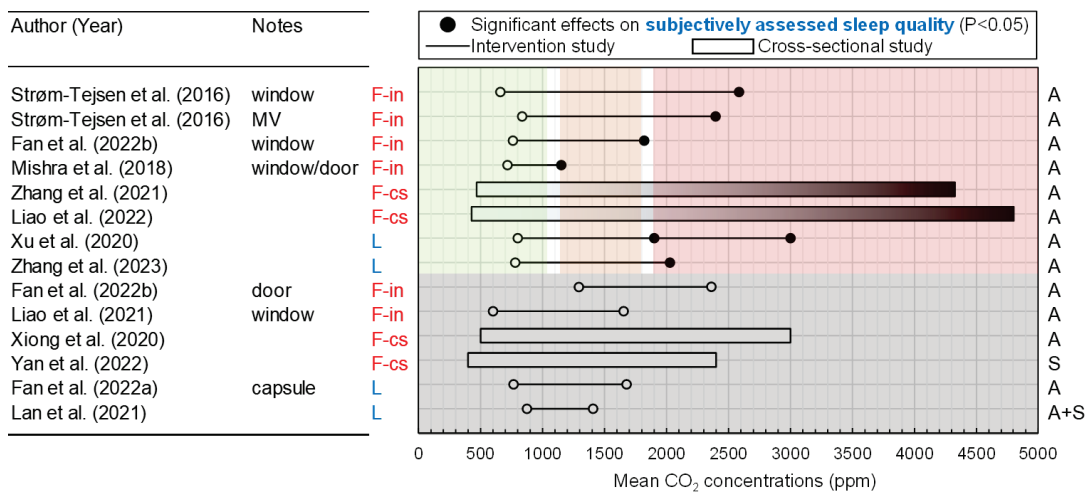
17

Take-away messages

- Changes to standards may be necessary
- Only Cat I (EN 16798) may provide sufficient ventilation in bedrooms
- Air change rates may need to be doubled
- Solution: Intelligent air distribution in housing is needed (following occupants)

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Summary of studies examining the effects of ventilation on sleep quality measured subjectively



How to subjectively measure sleep quality?

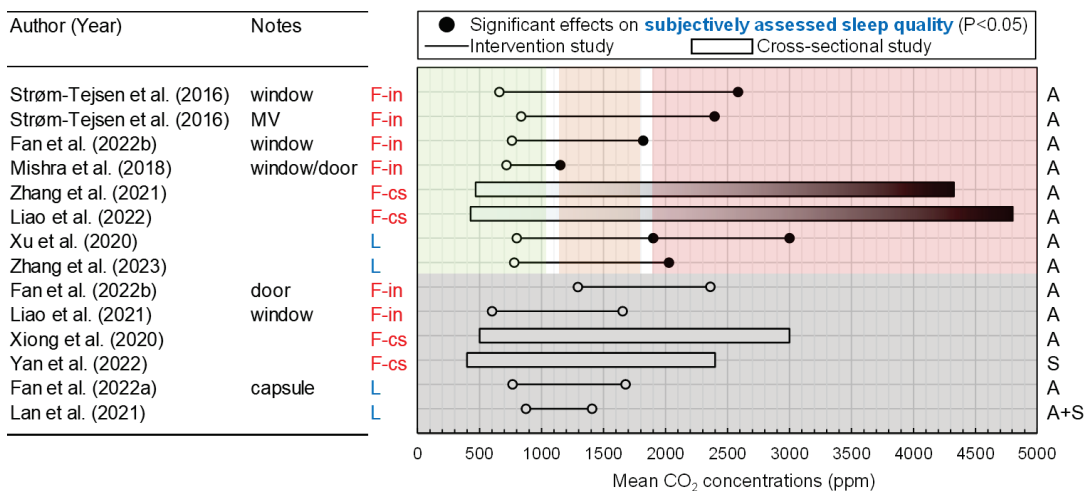
The Groningen Sleep Quality Scale (GSQS)

- Scores range from 0 to 14
- Scores between 0 and 2 indicate normal, refreshing sleep
- Scores ≥ 6 indicate disturbed sleep

I had a deep sleep last night	True	False
I feel like I slept poorly last night	True	False
It took me more than half an hour to fall asleep last night	True	False
I felt tired after waking up this morning	True	False
I woke up several times last night	True	False
I feel like I didn't get enough sleep last night	True	False
I got up in the middle of the night	True	False
I felt rested after waking up this morning	True	False
I feel like I only had a couple hours of sleep last night	True	False
I feel I slept well last night	True	False
I didn't sleep a wink last night	True	False
I didn't have any trouble falling asleep last night	True	False
After I woke up last night, I had trouble falling asleep again	True	False
I tossed and turned all night last night	True	False
I didn't get more than 5 hours sleep last night	True	False

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Summary of studies examining the effects of ventilation on sleep quality measured subjectively



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RESIDENTIAL VENTILATION REQUIREMENTS SWEDEN

- 0.35 L/sm² (w/people) and 0.1 L/sm² (w/o people): code requirement
- 0.35 L/sm² and 0.5 h⁻¹: public health
- Typical bedroom of 13.4 m²
- Bedroom ventilation will be 4.7 L/s bedroom (based on floor area or air change rate)

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RESIDENTIAL VENTILATION REQUIREMENTS NORWAY

- 1.2 m³/hm² (w/people) and 0.7 m³/hm² (w/o people)
- 26 m³/h per person in bedroom
- Typical bedroom of 13.4 m²
- Bedroom ventilation will be 4.5 L/s bedroom (based on floor area) or 7.2 L/s per person

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RESIDENTIAL VENTILATION REQUIREMENTS FINLAND

- 6 L/sp in new buildings (also 0.35 L/sm²); also 18 L/s per dwelling and 12 L/s per bedroom
- 6 L/sp in existing buildings but allowed to go to 4 L/sp
- Typical bedroom of 13.4 m²
- Bedroom ventilation will be 6 L/s per person or 4.7 L/s per bedroom (based on floor area)

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RESIDENTIAL VENTILATION REQUIREMENTS ESTONIA

- 0.42 L/sm² (0.6 h⁻¹) and 7 L/s per person in new residential buildings (EVS-EN 16798)
- 0.42 L/sm² for single family and 0.50 L/sm² for multifamily apartment buildings (energy performance)
- 10 L/s per living room and bedrooms (major renovations) and extracts 10 L/s toilet, 15 L/s (10 L/s) bathroom and 8 L/s (6 L/s) kitchens
- Typical bedroom of 13.4 m²
- Bedroom ventilation will be 7 L/s per person or 5.6 L/s per bedroom (based on floor area) or 10 L/s per bedroom (major renovation)

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BEDROOM VENTILATION REQUIREMENTS EUROPEAN STANDARD EN16798-2020

Category	Total ventilation including air infiltration (1)		Supply air flow per person (2)	Supply air flow based on perceived IAQ for adapted persons (3)	
	l/s,m ²	ach	l/s * per	q _p l/s*per	q _s l/s,m ²
I	0,49	0,7	10	3,5	0,25
II	0,42	0,6	7	2,5	0,15
III	0,35	0,5	4	1,5	0,1
IV	0,23	0,4			

^a Supply air flow for Method 3 is based on Formula (1) from 6.3.2.2.

Category	Design ΔCO ₂ concentration for living rooms (ppm above outdoors)	Design ΔCO ₂ concentration for bedrooms (ppm above outdoors)
I	550	380
II	800	550
III	1 350	950
IV	1 350	950

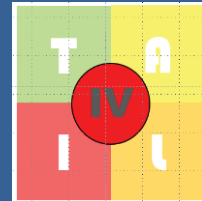
NOTE 1 The above values in Table B.12 correspond to the equilibrium concentration when the air flow rate is 4, 7, 10 l/s per person for cat. I, II, III respectively and the CO₂ emission is 20 l/h per person and 13,6 l/h per person for living rooms and bedrooms respectively.

NOTE 2 For a 10 m² room (room height 2,5 m, 25 m³) 4; 7 and 10 l/s per person correspond, with two persons in the room, to an air change rate of 1,2; 2,0 and 2,9 ACH.

Number of main rooms in the dwelling	Design extract air flow rates in l/s				
	Kitchen	Bathroom or shower with or without toilets	Other wet room	Toilets	
				Single in dwelling	Multiple (2 or more in dwelling)
1	20	10	10	10	10
2	25	10	10	10	10
3	30	15	10	10	10
4	35	15	10	15	10
5 and more	40	15	10	15	10

Category	Airflow rates defined in Table B.13 multiplied by
I	1,4
II	1
III	0,7
IV	0,5

Applicability and sensitivity of the TAIL rating scheme using data from the French national school survey



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43rd AIVC -11th TightVent & 9th Venticool Conference
October 4-5, 2023
Aalborg University, Copenhagen, Denmark



1

NO METHOD FOR LABELING INDOOR ENVIRONMENTAL QUALITY

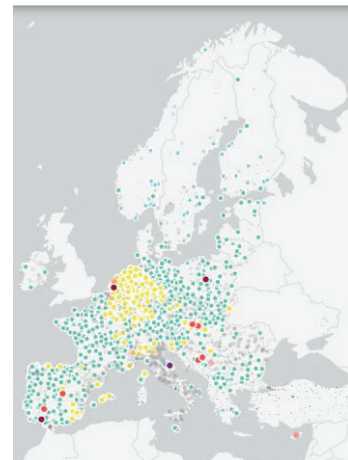
Nutrition Facts	
Serving Size 1 cup (228g) Servings Per Container 2	
Amount Per Serving	
Calories 250	Calories from Fat 110
% Daily Value*	
Total Fat 12g	18%
Saturated Fat 3g	15%
Trans Fat 3g	
Cholesterol 30mg	10%
Sodium 470mg	20%
Potassium 700mg	20%
Total Carbohydrate 31g	10%
Dietary Fiber 0g	0%
Sugars 5g	
Protein 5g	
Vitamin A	4%
Vitamin C	2%
Calcium	20%
Iron	4%

*Percent Daily Values are based on a diet of other people's misdeeds. Your Daily Values may be higher or lower depending on your calorie needs.

	Calories 2,000	2,500
Total Fat	Less than 65g	80g
Sat Fat	Less than 20g	25g
Cholesterol	Less than 300mg	300mg
Sodium	Less than 2,400mg	2,400mg
Total Carbohydrate	300g	375g
Dietary Fiber	25g	30g



European Air Quality Index



2

RATING SCHEMES PROVIDE MONITORING AND DOCUMENTATION OF COMPLIANCE

- Useful data for all building stakeholders and additional incentives for improvement of IEQ
- Create benchmark, reference, building data-base
- Monitor performance – compliance and maintenance
- Input to sustainable investments, and technological advancements
- Input to control and AI
- Input to energy simulation and reduce performance gap
- Input to economic calculations
- Demonstrate invisible - occupants feel secure (no risks)
- Raising awareness



3

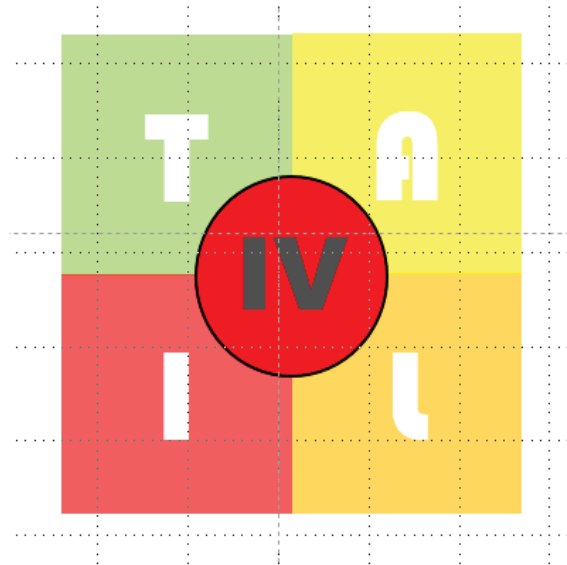
A TAIL SCHEME FOR RATING IEQ

Four components:

- **T**hermal environment
- **A**coustic environment
- **I**ndoor air quality
- **L**ight – Luminous (visual) environment

Overall IEQ:

- (I) (II) (III) (IV)



Source: Wargocki et al. (2020)

4

WHY IS CLASSIFICATION OF A WIDER BENEFITS THAN ENERGY NECESSARY?

- 1) To guarantee that IEQ is not degraded during renovation to satisfy the EPBD mandate.
- 2) To document any improvements in IEQ after renovation.
- 3) To estimate potential additional benefits from renovation including benefits for health and well-being, as well as the financial benefits from improved productivity and increased value of a building on a market.

Source: EU ALDREN Project

5

TWELVE IEQ PARAMETERS in TAIL

	IEQ parameter	Measured	Modelled	Visual inspection
T	Indoor temperature (°C)	x		
A	Noise level (dB(A))	x		
I	CO ₂ (ppm)	x		
	Ventilation rate (L/s)	x		
	Formaldehyde (µg/m ³)	x		
	Benzene (µg/m ³)	x		
	PM _{2.5} (µg/m ³)	x		
	Radon (Bq/m ³)	x		
	Indoor air relative humidity (%)	x		
	Visible mold (cm ²)			x
L	Daylight factor (%)		x	
	Illuminance (lux)	x		

6

RANGES OF PARAMETERS INCLUDED IN TAIL: IAQ

Quality of indoor air quality (I)	Green	Yellow	Orange	Red
Carbon dioxide (concentration above outdoors) ^{1,2}	≤550 ppm	≤800 ppm	≤1350 ppm	If other quality levels cannot be achieved
Ventilation rate ^{3,7}	≥(10 L/s/p + 2.0 L/s/m ² floor)	≥(7 L/s/p + 1.4 L/s/m ² floor) and <(10 L/s/p + 2.0 L/s/m ² floor)	≥(4 L/s/p + 0.8 L/s/m ² floor) and <(7 L/s/p + 1.4 L/s/m ² floor)	If other quality levels cannot be achieved
Relative humidity offices ^{2,4} hotel rooms ^{2,4,5}	>30% <50% >30% and ≤50%	≥25% <60% >25% and ≤60%	≥20% <70% >20% and ≤60%	If other quality levels cannot be achieved
Visible mold ^{6,7}	No visible mould	Minor moisture damage, minor areas with visible mould (<400 cm ²)	Damaged interior structural component, larger areas with visible mould (<2500 cm ²)	Large areas with visible mould (≥2500 cm ²)
Benzene ⁷	<2 µg/m ³	≥2 µg/m ³	no criteria	≥5 µg/m ³
Formaldehyde ⁷	<30 µg/m ³	≥30 µg/m ³	no criteria	≥100 µg/m ³
Particles PM _{2.5} (gravimetric) ⁷	<10 µg/m ³	≥10 µg/m ³	no criteria	≥25 µg/m ³
Particles PM _{2.5} (optical) ⁷	<10 µg/m ³	≥10 µg/m ³	no criteria	≥25 µg/m ³
Radon ^{7,8}	<100 Bq/m ³	≥100 Bq/m ³	no criteria	≥300 Bq/m ³

7

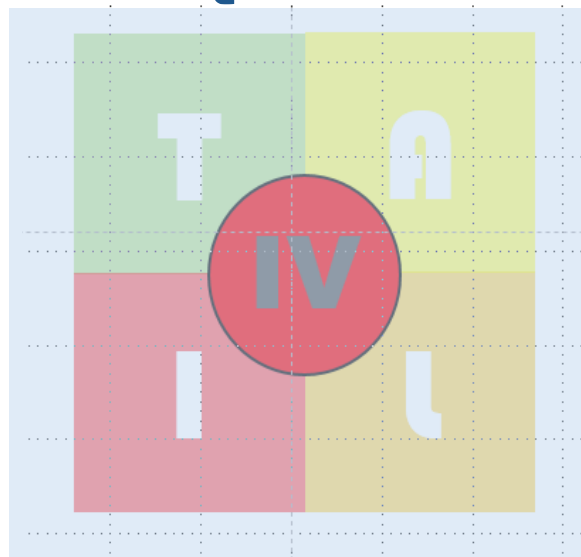
PredicTAIL, the method for rating SIMULATED IEQ

Four components:

- **T**hermal environment
- **A**coustic environment
- **I**ndoor air quality
- **L**ight – Luminous (visual) environment

Overall IEQ:

- (I) (II) (III) (IV)



Source: Wei et al. (2021)

8

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TAIL, a new scheme for rating indoor environmental quality in offices and hotels undergoing deep energy renovation (EU ALDREN project)

Pawel Wargocki^{a,*}, Wenjuan Wei^b, Jana Bendžalová^c, Carlos Espigares-Correa^d, Christophe Gerard^f, Olivier Greslou^g, Mathieu Rivalain^h, Marta Maria Sesanaⁱ, Bjarne W. Olesen^j, Johann Zirngibl^k, Corinne Mandin^l

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ABSTRACT

To avoid health risks and discomfort, the European Energy Performance for Building Directive (EPBD) mandates that “Member States should support energy performance upgrades of existing buildings that contribute to achieving a healthy indoor environment”. There is, however, no widely accepted method for rating the overall level of indoor environmental quality (IEQ), although several different approaches are proposed by standards, guidelines, and certification schemes. To fill this void, a new classification rating scheme called TAIL was developed to rate IEQ in offices and hotels undergoing deep energy renovation during their normal use; the scheme is a part of the energy certification method developed by the EU ALDREN project. The TAIL scheme standardizes rating of the quality of the thermal (T) environment, acoustic (A) environment, indoor air (I), and luminous (L) environment and by using these ratings, it provides a rating of the overall level of IEQ. Twelve parameters are rated by measurement, modeling, and observation to provide the input to the overall rating of IEQ. Their quality levels are determined primarily using Standard EN 15796-1 and World Health Organization (WHO) air quality guidelines and are expressed by colour and human icons to improve communication. The TAIL rating was shown to discriminate IEQ levels when its feasibility was examined in eleven buildings across Europe to provide support for its applicability and input for further modifications. Opportunities for using the scheme in other types of buildings and for its further development and application are discussed.

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1. Introduction

Several policies and actions have been put forward by the European Union (EU) to mitigate and reduce the impact of climate change. One such action is the modernization and renovation of the European building stock, which is responsible for 40% of energy use and 36% of carbon dioxide (CO₂) emissions [1]. The European Commission created instruments to initiate changes in how buildings are constructed, operated, and maintained to achieve significant reductions in energy use. The framework was established by the Energy Performance of Building Directive (EPBD), which was launched in 2002 [1], re-cast in 2010 [1,6], and amended in 2018 [16]. The main purpose of this Directive is to promote improvements in the energy performance of buildings. This applies both to new construction and existing buildings, of which 25% are commercial buildings, 75% are considered to be residential, and about 35% are at least 50 years old.

Despite these high ambitions and good intentions, the implementation of EPBD failed somewhat concerning renovation of the existing building stock. Renovation rates that followed EPBD recommendations have not exceeded 15 to 20% [6], although it is estimated that renovation accounts for 57% of all construction activity, and many renovations do not reach the full amount of energy savings that could be achieved [4]. Renovation rates following EPBD recommendations should reach at least 3% to guarantee that minimum energy reduction goals will be met [11]. One reason for this shortfall could be that renovations, even those leading to reductions in energy

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PredicTAIL, a prediction method for indoor environmental quality in buildings undergoing deep energy renovation based on the TAIL rating scheme

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ABSTRACT

The recently developed TAIL rating scheme enables assessment of the changes in the indoor environmental quality (IEQ) associated with a building's deep energy renovation (DER) and classification of the resulting quality levels of the thermal (T), acoustic (A), and luminous (L) environments and indoor air quality (I). Since the TAIL rating is primarily based on measurement, it cannot be determined prior to renovation operations to help design the IEQ. To fill the gap, the PredicTAIL method was developed in the present study to predict the changes in the twelve TAIL parameters as a result of DER. These parameters are indoor air temperature, relative humidity, sound pressure level, daylight factor, illuminance, and concentrations of carbon dioxide, formaldehyde, benzene, radon, and PM₁₀. No prediction is made for ventilation rate or mold. To examine the feasibility of the PredicTAIL method and the sensitivity of the existing models for quantifying changes in the TAIL parameters corresponding to different renovation strategies, simulations were performed in a hotel and an office building using BRNOVS, IAQ ICE, ACCURATE, MATHE-QAL, and PHAME. These modeling tools were first benchmarked against the TAIL parameters measured in the buildings before renovation. Once the agreement between measurement and modeling was considered acceptable, four pragmatic renovation scenarios were applied, and their impact on the IEQ parameters was quantitatively modeled. The simulations showed that the quality levels of the IEQ parameters were improved or unchanged for some parameters but degraded for other parameters about IEQ. The changes in the IEQ parameters and the TAIL rating depended on the renovation scenario, suggesting that the PredicTAIL method is sufficiently sensitive to guide renovation designs.

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1. Introduction

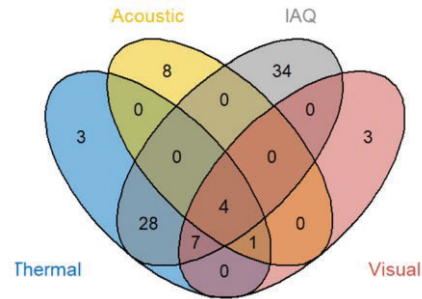
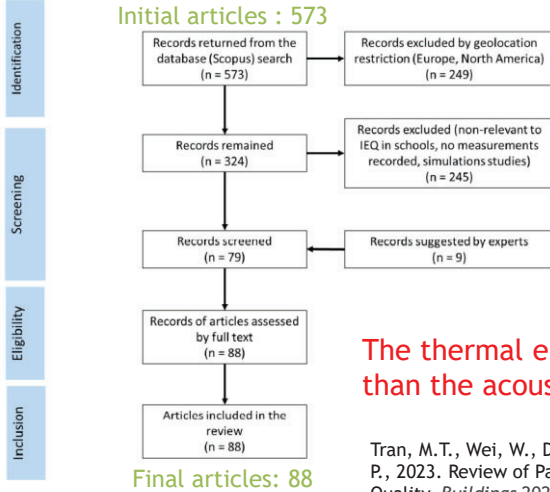
The European Union (EU) put forward a series of directives aimed at developing a sustainable, competitive, secure, and decarbonized energy system, providing objectives for reducing energy consumption by 20% by 2020 and at least 40% by 2030 compared with that in 1990 [1–4]. Given that almost 50% of the EU's final energy consumption is used for heating and cooling, 60% of which is used in buildings [1], performing deep energy renovation (DER) to improve buildings' energy efficiency is a promising way to achieve the EU's energy and climate goals. However, improved insulation may impose risks of higher indoor air humidity, higher indoor pollutant concentrations and overheating, and installing a heating, ventilation and air conditioning (HVAC) system may compromise the indoor acoustic environment [5]. To account for the influence of DER on the indoor environmental quality (IEQ), the latest amendment of the EU Directive on the energy performance of buildings states that “the energy needs for space heating, space cooling, domestic hot water, ventilation, lighting, and other technical building systems shall be calculated in order to optimize health,

9

WHAT ABOUT SCHOOLS?

10

WHAT PARAMETERS HAVE BEEN FREQUENTLY MEASURED IN CLASSROOMS?



The thermal environment and IAQ are assessed more frequently than the acoustic and visual environments in schools.

Tran, M.T., Wei, W., Dassonville, C., Martinsons, C., Ducruet, P., Mandin, C., Héquet, V., Wargocki, P., 2023. Review of Parameters Measured to Characterize Classrooms' Indoor Environmental Quality. *Buildings* 2023, 13(2), 433; <https://doi.org/10.3390/buildings13020433>

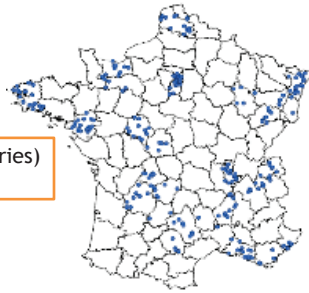
Parameters defining TAIL components for schools

	IEQ parameter	Measured	Modelled	Visual inspection
T	Indoor temperature (°C)	✗		
A	Noise level (dB(A))	✗		
	Reverberation time (s)	✗		
I	CO ₂ (ppm)	✗		
	Ventilation rate (L/s)	✗		
	Formaldehyde (µg/m ³)	✗		
	Benzene (µg/m ³)	✗		
	PM _{2.5} (µg/m ³)	✗		
	Radon (Bq/m ³)	✗		
	Indoor air relative humidity (%)	✗		
	Visible mold (cm ²)			✗
	Nitrogen dioxide (µg/m ³)	✗		
L	Daylight factor (%)		✗	
	Illuminance (lux)	✗		

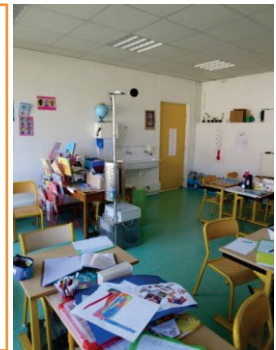
NATIONAL SCHOOLS IEQ SURVEY IN FRANCE

National school survey: (CNE)
 OQAI 2013-2017
 Measurement during 1 schools week.

- 308 schools (elementary & nurseries)
- 602 classrooms



- Measurements values:
- Temperature (°C), relative humidity (%)
 - Volatile organic compounds, semi-volatile organic compounds and metals in settled dust, particulate matter 2.5µm, total particles counts, NO₂, CO₂
 - Background noise level
 - Artificial illuminance
 - Building characteristics



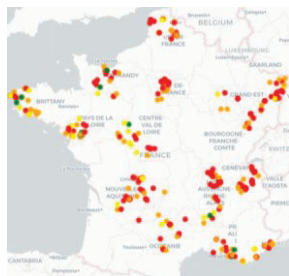
Occupants' perception
 (Teachers, auto complete survey)



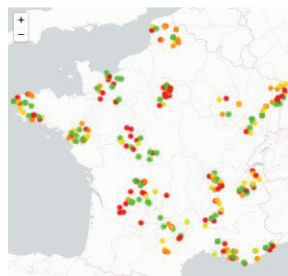
13

ILLUSTRATE INVISIBLE, TAIL FOR 308 SCHOOLS IN FRANCE, EXAMPLE

High, desired Medium Ordinary Low - undesired



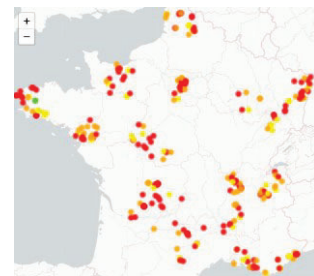
Quality of thermal environment (T)



Quality of acoustic environment (A)



IAQ (I)



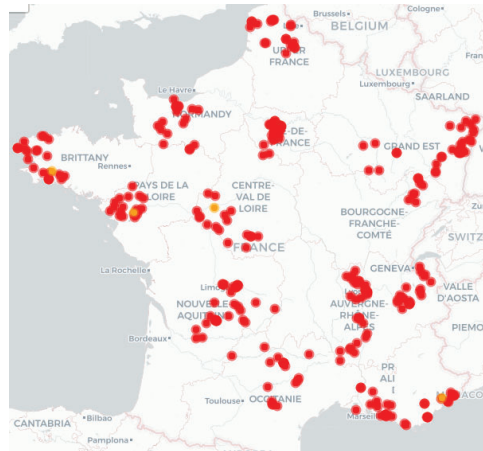
Quality of luminous environment (L)

Source: Tran et al. (2023)

14

ILLUSTRATE INVISIBLE, TAIL FOR 308 SCHOOLS IN FRANCE, EXAMPLE

High, desired Medium Ordinary Low - undesired



Overall quality of indoor environment (TAIL)

Source: Tran et al. (2023)

15

VERIFICATION AND VALIDATION PERCEPTIONS OF TEACHES AND TAIL

	Perception of teachers
Thermal comfort 5 questions	Thermal sensation (cold/warm, satisfactory/unsatisfactory)
	Temperature variation during the day
	Presence of drafts and its perception
Olfactif comfort 4 questions	Presence of odors and its perception
Air quality sensation 3 questions	Sensation of air quality (satisfactory/unsatisfactory, dry/wet, confined/renewed)
Acoustic comfort 29 questions	Feeling the noise in the classroom
	Noise from outside and inside
	Feedback on the furniture layout
Lighting comfort 23 questions	Sensation of the quality of light in the classroom
	Use of solar protection
	Use of artificial lighting

16

CONCLUDING REMARKS

17

SUMMARY

- TAIL and PredicTAIL provide a complete tool allowing characterization of IEQ in buildings.
- They are expected to become a standard method of benchmarking IEQ in buildings when applied.
- They are expected to stimulate actions leading to the general improvement of the IEQ in buildings.

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Minh-Tien Tran[‡], Valérie Héquet[§], Pawel Wargocki^{‡§}, Corinne Mandin[‡], Claire Dassonville[‡], Wenjuan Wei[‡]

‡ French Indoor Air Quality Observatory (OQAI), Health and Comfort Department, Scientific and Technical Center for Building (CSTB), Marne-la-Vallée, France

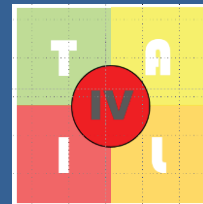
‡‡ Institute for Radiological Protection and Nuclear Safety - IRSN- Paris, France

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Questions



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Aalborg University, Copenhagen, Denmark



20

An investigation of MVHR system performance based on health and comfort criteria in bedrooms of low-carbon social housing in Wales

Faisal Farooq

Welsh School of Architecture, Cardiff University

Simon Lannon, Colin Biggs, Phil Jones



1

Aim

Investigate MVHR system performance in low-carbon housing in the UK based on a health and comfort criteria and propose design solutions.

Motivation

- MVHR has a critical role in the success of low-carbon housing.
- Resolve recurring issues with the performance gap.

2

Background

- Carried systematic literature review on in-situ performance evaluation of different ventilation technologies in low-carbon housing in the UK.
- Selection criteria was dwellings with an airtightness of 5-6 m³/m²h @ 50 Pa or below.
- Meta-analysis showed high concentration of CO₂ in bedrooms without MVHR.
- CO₂ is used as a surrogate for human emitted bio-effluents and commonly forms the basis for ventilation design.

3

Results of meta-analysis

Reference	Sample size & ventilation strategy	Monitoring period	Avg. airtightness (m ³ /h m ² @ 50 Pa)	Occupied time weighted average CO ₂ level (ppm) in bedrooms	ΔCO ₂ level (ppm) in bedrooms above ambient
Howieson (2014b)	20 (trickle vents)	5 months	4-5	1834	1404
Sharpe et al. (2015)	40 (trickle vents)	3 months	<4	1847	1417
Sharpe et al. (2019)	41 (dMEV)	1 week	4-5	1922 (separate bathroom), 1204 (en-suite)	1492 (separate bathroom), 774 (en-suite)
Sharpe et al. (2014)	21 (trickle vents), 5 (MVHR)	7 months	5.33 (trickle vents), 2.13 (MVHR)	1292 (trickle vents), 858 (MVHR)	862 (trickle vents), 428 (MVHR)*
McGill (2015c)	4 (trickle vents), 4 (MVHR)	24 hrs	4.6 (trickle vents), 2.06 (MVHR)	1710 (trickle vents), 875 (MVHR)	1280 (trickle vents), 445 (MVHR)*
Sharpe et al. (2016)	20 (trickle vents), 23 (MVHR)	3 months	3.2	1118 (trickle vents), 762 (MVHR)	688 (trickle vents), 332 (MVHR)*

* CO₂ values for MVHR are under 550 ppm above ambient which represents a ventilation rate of 7l/s per person.

4

Common MVHR shortcomings in UK housing

- Design specification
- Installation
- Commissioning
- Project management
- Occupant knowledge
- Occupant feedback

5

Common MVHR shortcomings in UK housing *Health and Comfort perspective*

- Design specification
 - Supply vent close to bedroom door (Sharpe and Charles, 2015).
- Occupant knowledge
- Occupant feedback
 - Perceived **draught** from supply vent above beds (Sharpe et al., 2018; Gupta et al., 2018; Gupta and Kapsali, 2016; Gupta, 2016).
 - Perceived **noise** when trying to fall asleep (ZCH, 2015; Sharpe et al., 2018; Gupta, 2016).

6

Research questions

- What is the relationship of supply/door undercut arrangement with ventilation effectiveness?
- Do sound levels from an MVHR lie within comfortable range in low-carbon dwellings?
- Under what conditions and circumstances can air from an MVHR supply vent cause thermal (dis)comfort for occupants at night?

7

Case study A



8

Case study B



9

Case study details

- Systems were commissioned to meet the minimum ventilation rates prescribed under UK Building Regulations.
- Rigid ducting was used throughout with minimum bends.
- MVHR unit was located in the thermal envelope for Case Study A.
- Ducting was insulated throughout for Case Study B.

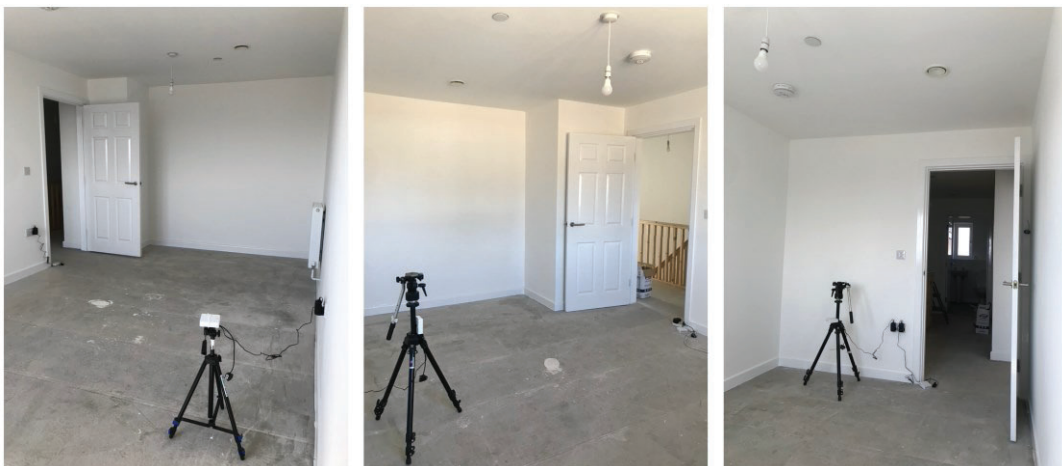
10

Methodology

- **Tracer gas experiments**
 - Ventilation effectiveness & Age of Air (Fisk and Faulkner, 1992).
- **Sound and frequency measurements**
 - Part F of UK Building Regulations
- **Thermal comfort**
 - Predicted Mean Vote experiment (ISO 7730:2005)
 - Continuous Temperature/RH monitoring

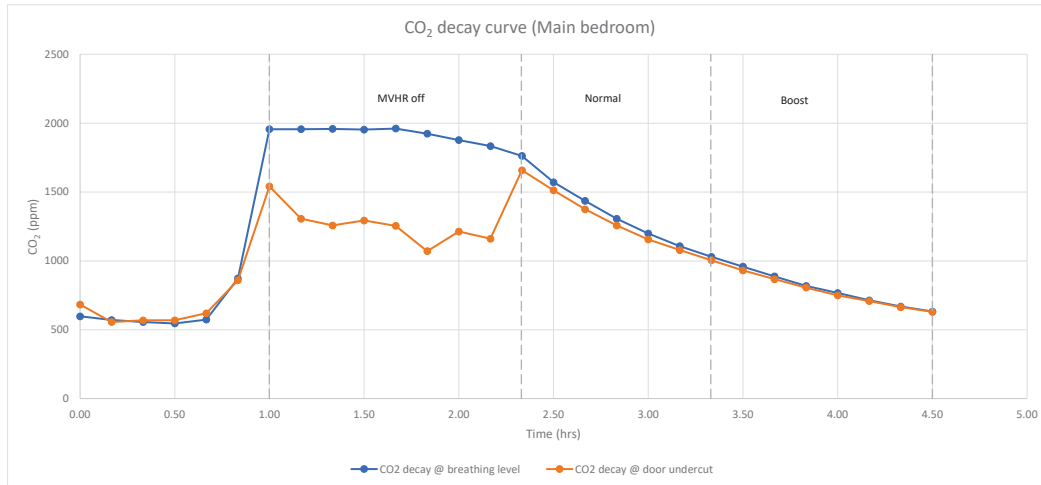
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Tracer gas experiment



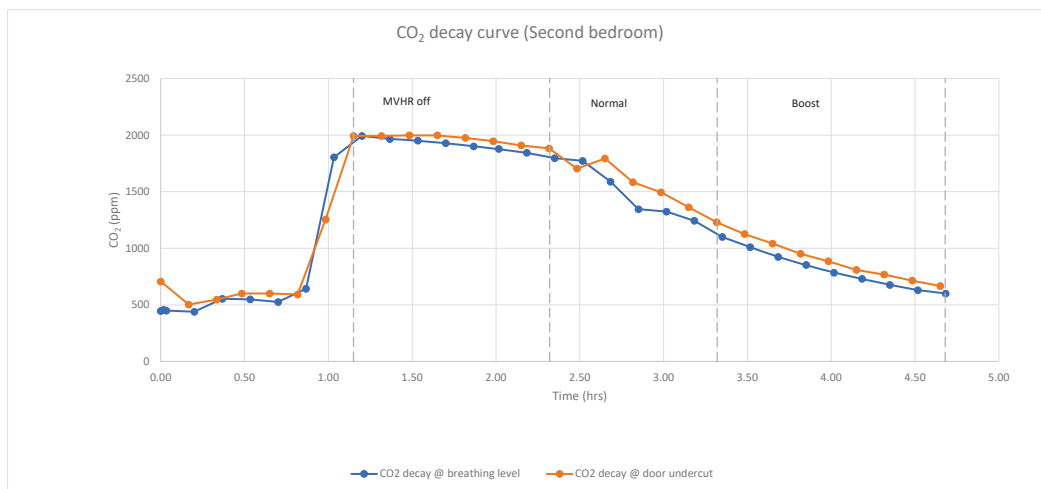
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Decay curve for main bedroom



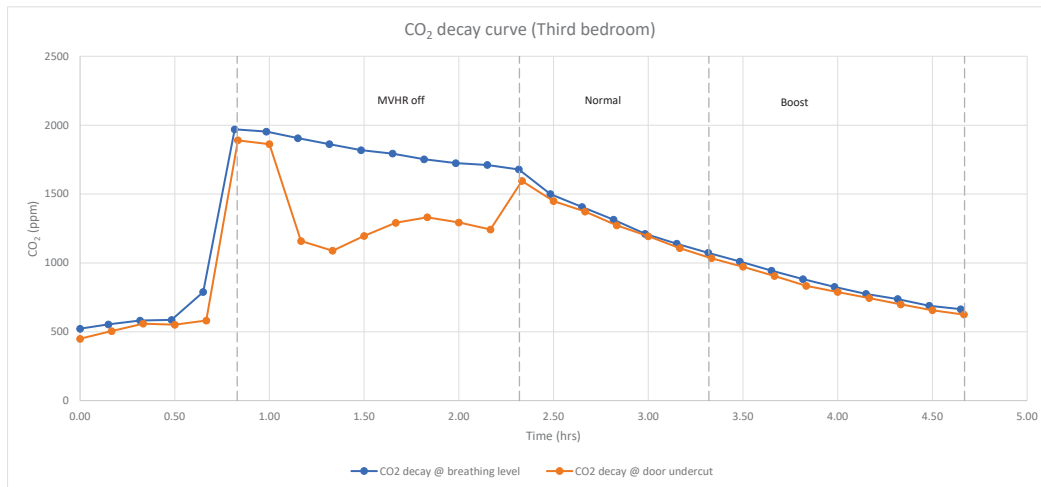
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Decay curve for second bedroom



14

Decay curve for third bedroom



15

Tracer gas experiment results

	τ_{DU}^* (hr)	τ_{BL}^{**} (hr)	E_{ADE}^{***}
Main bedroom	0.78	0.76	1.02
Second bedroom	0.84	0.81	1.04
Third bedroom	0.81	0.79	1.02

τ_{DU}^* is age of air at Door Undercut

τ_{BL}^{**} is age of air at Breathing Level

E_{ADE}^{***} is Air Diffusion Effectiveness

$E_{ADE} > 1$ is displacement flow pattern

$E_{ADE} = 1$ is perfect mixing

$E_{ADE} < 1$ is short-circuiting

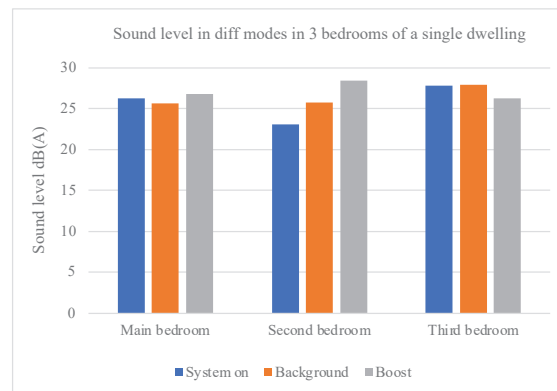
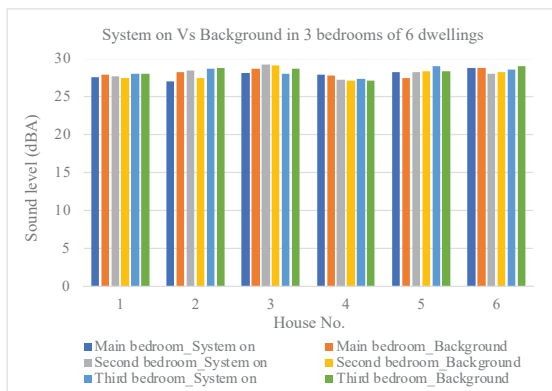
16

Sound measurements



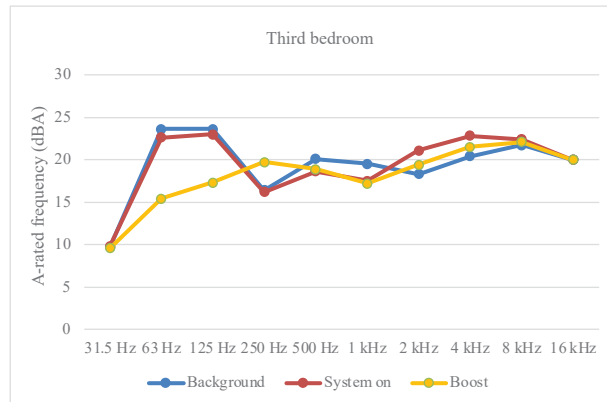
17

Sound measurements results in dB(A)



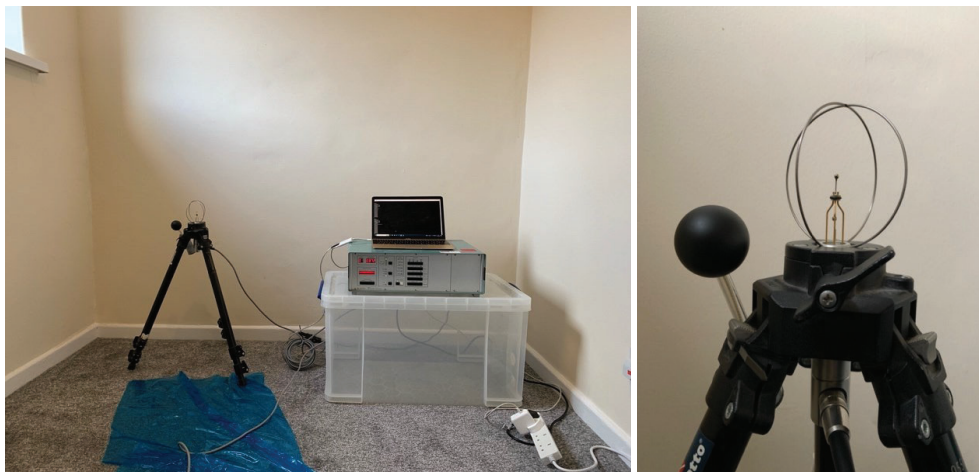
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Frequency measurement in one bedroom



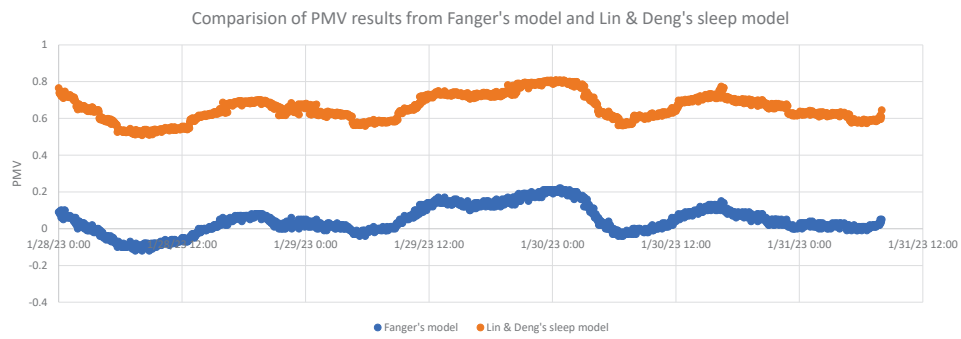
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Thermal Comfort experiment



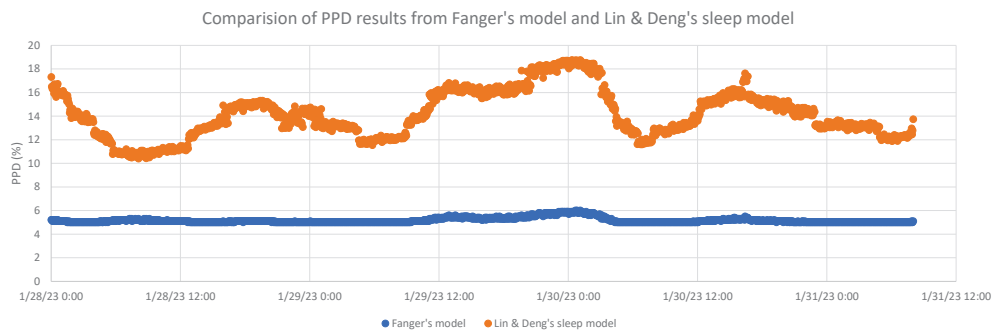
20

Predicted Mean Vote results



21

Predicted Percentage Dissatisfied results



22

Conclusions

- In terms of ventilation effectiveness, displacement flow pattern was observed, despite the proximity of the vent to the door.
- This is attributed to low velocities and shape of supply vent.
- Noise levels were under 30dB(A) in all modes of operation.
- A rumble in the low frequency range (50-500Hz) was observed in one of the bedrooms under boost mode.
- From the PMV experiment, 80% of occupants are predicted to be thermally satisfied under these environmental conditions.
- This is attributed to proper installation and commissioning practice at the two case study sites.

23

Further work – CFD modelling

- Vent closer to the door
- Different seasons
 - Continuous monitoring data
- Unbalanced system
- Door undercut
- Furniture/bed



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Thank you! 😊

Contact: farooqf@cardiff.ac.uk

Impact of optimized residential ventilation with energy recovery on health and well-being

Martin Kremer, Kai Rewitz, Dirk Müller

RWTH Aachen University, E.ON Energy Research Center, Institute for Energy Efficient Buildings and Indoor Climate

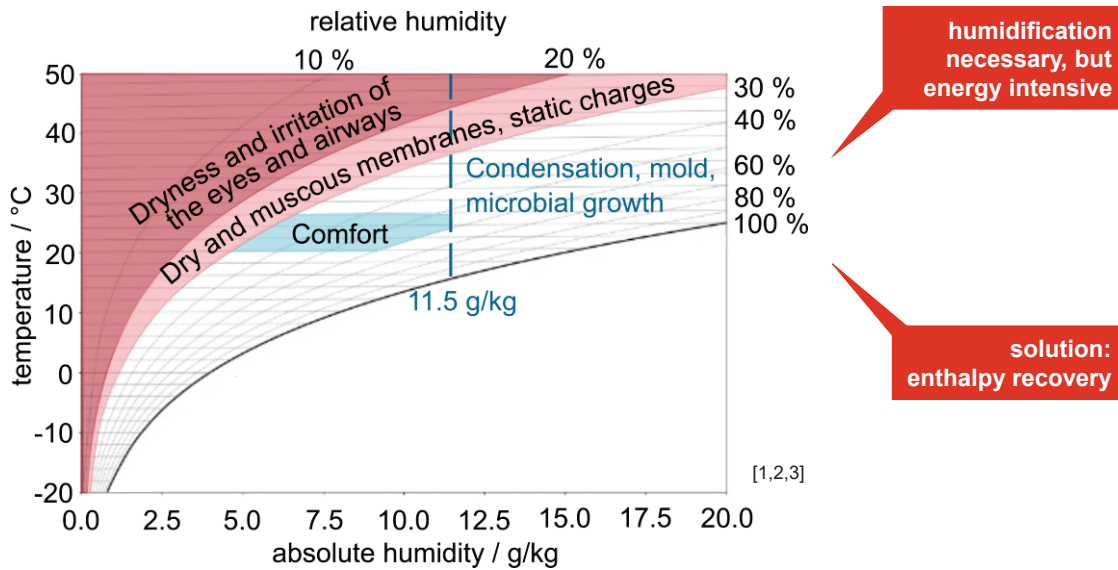
EBC | Institute for Energy Efficient Buildings and Indoor Climate



1

Motivation

Why optimization of energy recovery is necessary?



2

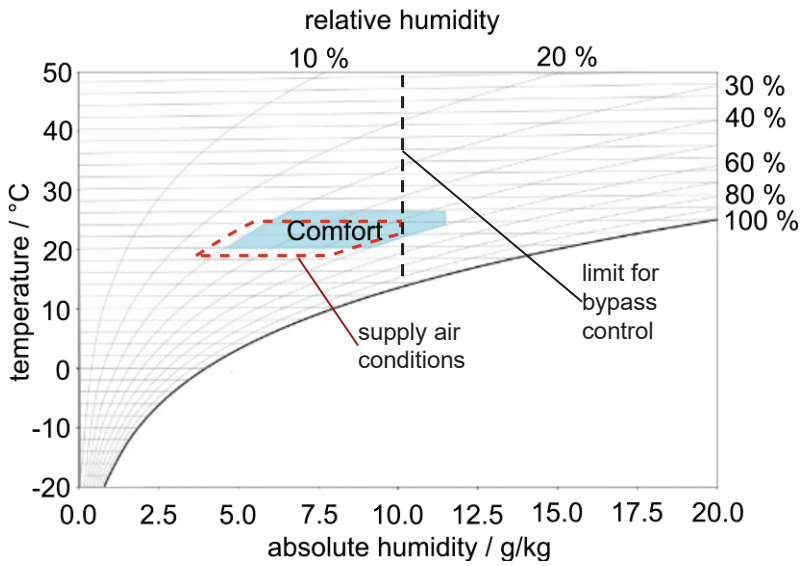
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2

Motivation

Why optimization of energy recovery is necessary?

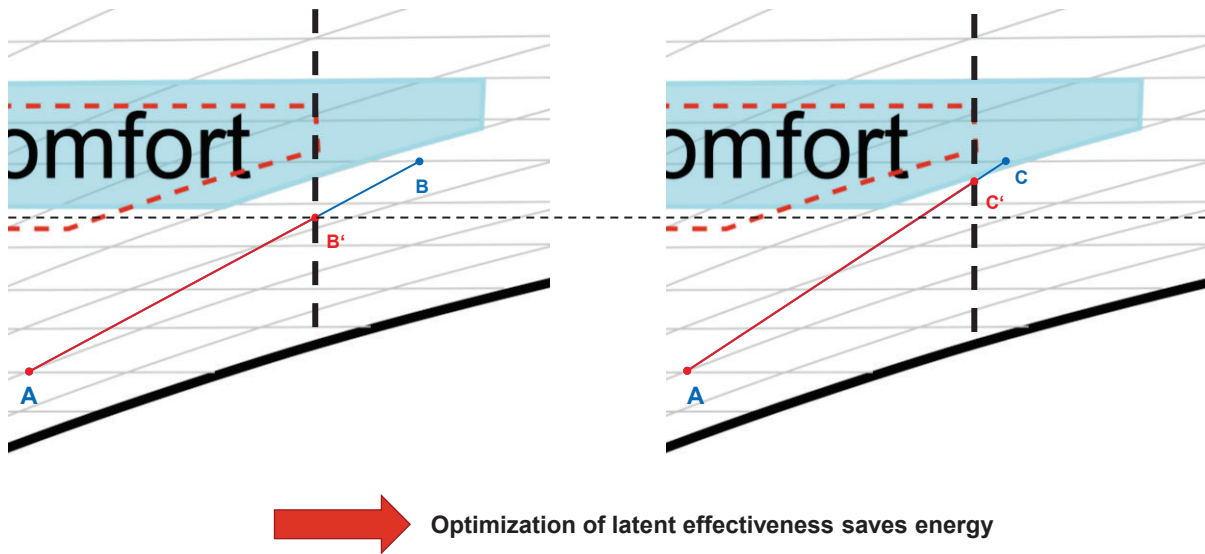


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Motivation

Why optimization of energy recovery is necessary?

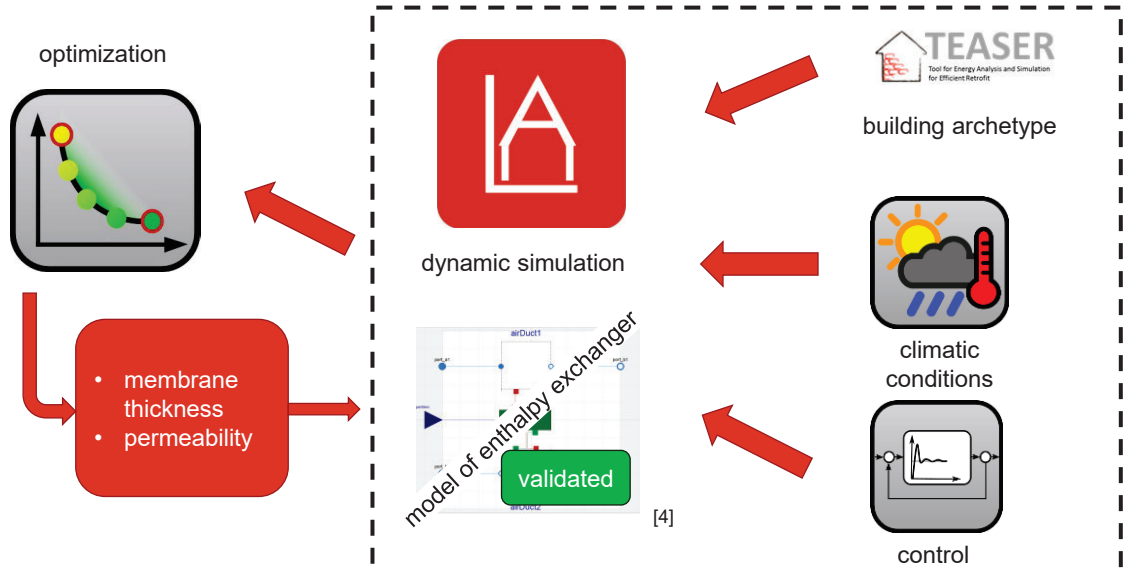


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4

Methodology

Design optimization with dynamic simulation models

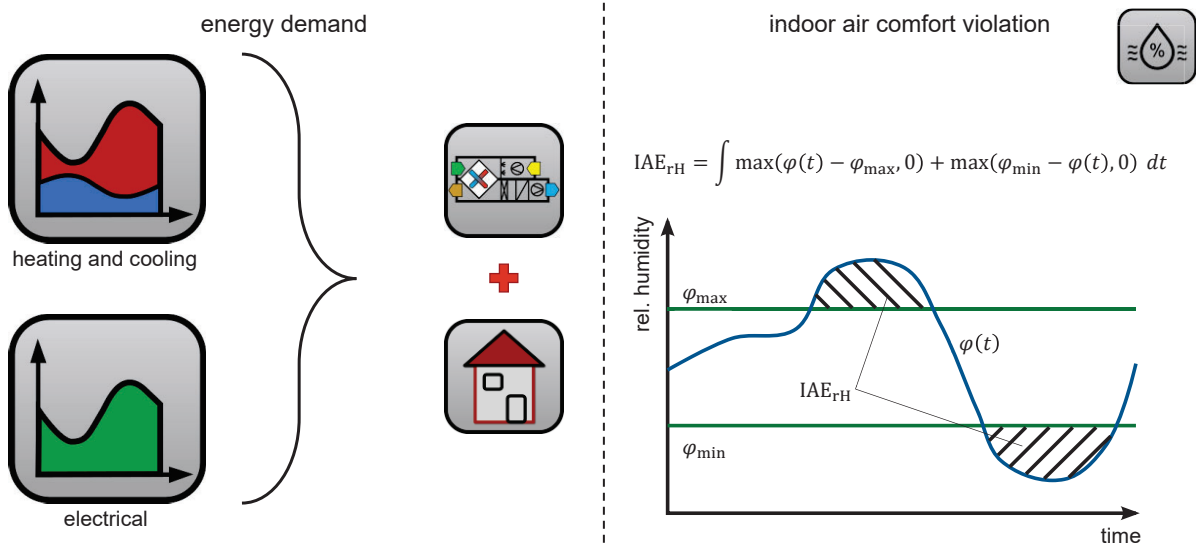


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5

Methodology

Multicriterial optimization regarding energy demand and comfort

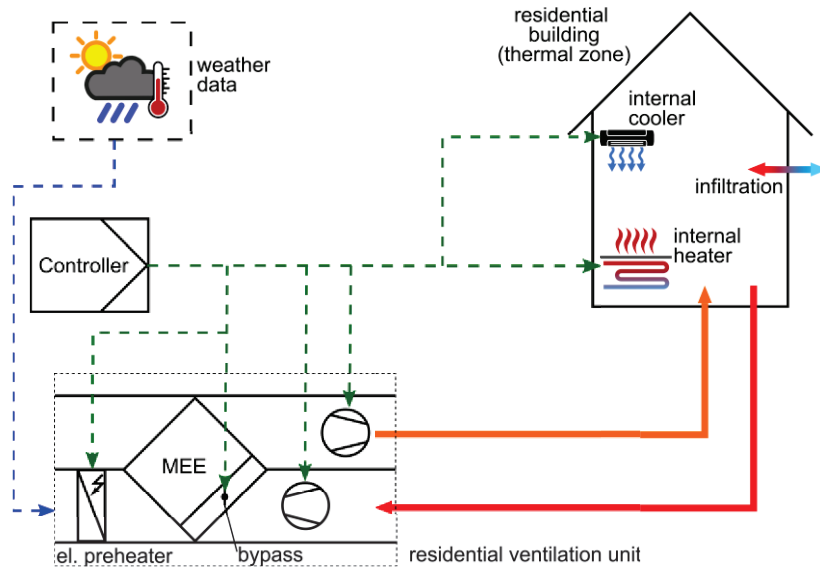


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6

Case Study

Simulation model of residential building with central ventilation unit



7

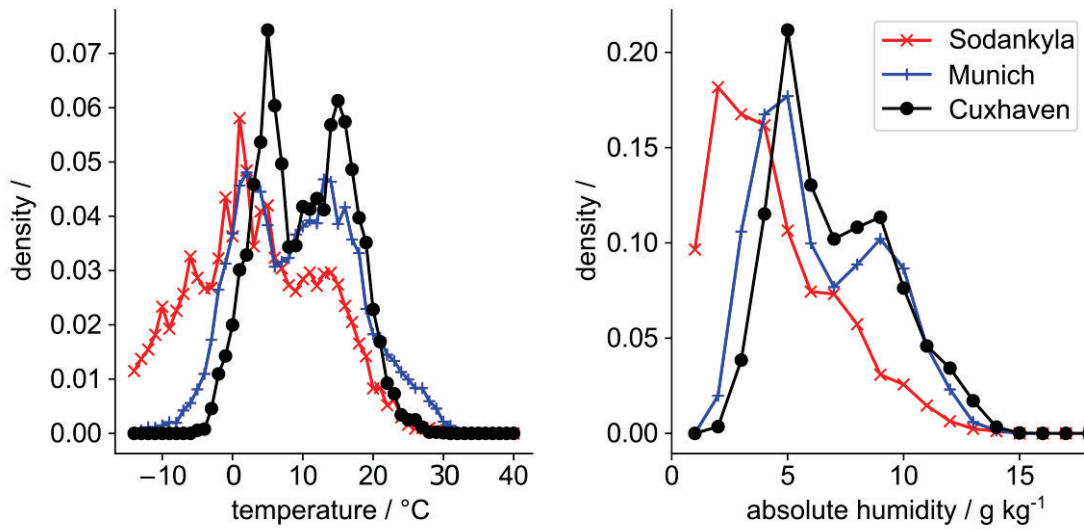
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7

Case study

Investigated locations – differences in temperature and humidity



8

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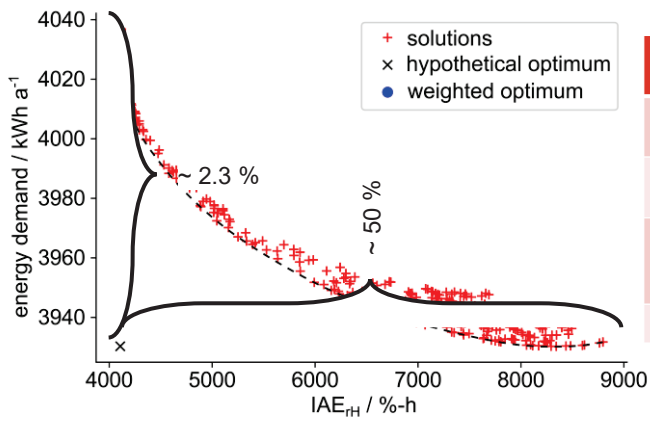


8

Results

Energetic and comfort optimum differ: weighting necessary

Munich



	energetic optimum	comfort optimum	weighted optimum
membrane thickness / μm	155	20	20
permeability / $\text{mol m}^{-1} \text{s}^{-1} \text{Pa}^{-1}$	$1,67 \cdot 10^{-10}$	$2,65 \cdot 10^{-10}$	$2,27 \cdot 10^{-10}$
energy demand / kWh a^{-1}	3930	4030	4021
IAE_{rH} / %·h	8343	4107	4124

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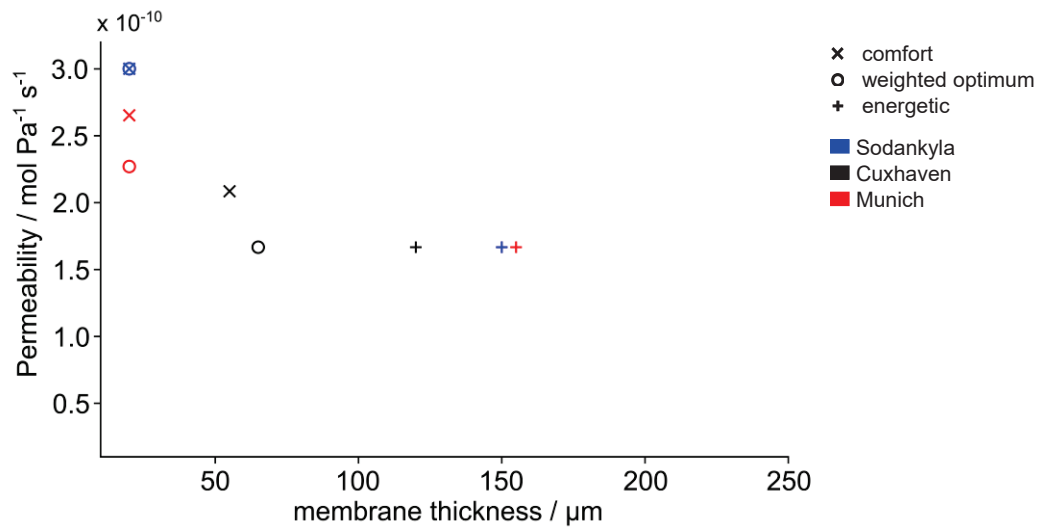
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9

Results

Influence of the location: moderate climates need lower latent effectiveness



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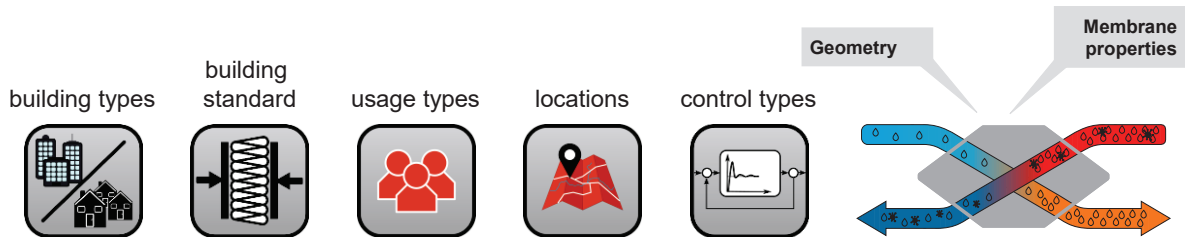
Conclusion and Outlook

Comfort increase with optimal enthalpy exchanger design depending on location

Conclusion

- Energetic Optimum: thicker and less permeable membranes
- Comfort Optimum: thinner and more permeable membranes
- Relative impact
 - ≡ Energy demand: 0.5 – 2.3 %
 - ≡ Comfort: 34 – 50 %

Outlook



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11

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[2]: Sterling, E.M.; Arundel, A.; Sterling, T.D. (1985): *Criteria for human exposure to humidity in occupied buildings*. In: ASHRAE transactions 91.1, S. 611-622

[3]: Sunwoo, Y.; Chou, C.; Takeshita, J.; Murakami, M.; Tochihara, Y. (2006): *Physiological and subjective Responses to low relative humidity*. In: Journal of physiological anthropology 25 (1), S. 7-14

[4]: Kremer, Martin; Mathis, Paul; Müller, Dirk (2019): *Moisture Recovery - A Dynamic Modelling Approach*. In: CLIMA 2019 Congress : Bucharest, Romania, May 26-29, 2019 / S.I Tanabe, H. Zhang, J. Kurnitski, M.C. Gameiro da Silva, I. Nastase, P. Wargocki, G. Cao, L. Mazzarela and C. Inard (Eds.).
<https://doi.org/10.1051/e3sconf/201911101099>



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Tight Vent Europe

venticool
the platform for passive ventilative cooling

A detailed investigation of the impact of an innovative dynamic façade system on indoor environmental quality in offices

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5th October 2023

Built Environment Department, Building Performance Group

TU/e EINDHOVEN UNIVERSITY OF TECHNOLOGY

1

Outline

- ✓ FacelNQ project
- ✓ Research objectives
- ✓ Methodology
- ✓ Results
- ✓ Summary & conclusions

2

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5th October 2023

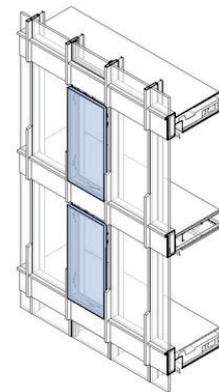
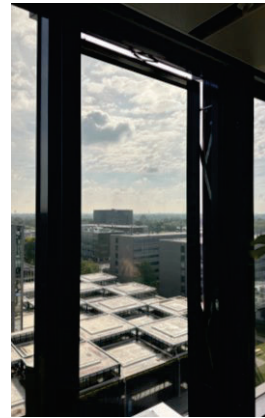
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FaceINQ project

Innovative dynamic Façade systems for INdoor environmental Quality



Sources: <https://www.archdaily.com/>
Zhijian Wang

Research objectives



Develop **new operational strategies and designs** for innovative building façade systems that:

- Ensure IEQ appropriate to users,
 - Limit building related health risks and
 - Reduce energy consumption.
-
- Provide detailed IEQ parameters in offices,
 - Capture user's perception of the indoor environment,
 - Provide data for development of validated CFD/BES models.



Methodology

The methodology cycle consists of three main stages:

- Building:** A photograph of a modern building with a glass facade. Source: Bart van Overbeeke.
- Measurement:** Images showing experimental setups with sensors and equipment in a laboratory setting.
- Simulation:** 3D thermal and airflow simulation models of the building interior, showing temperature gradients and air flow patterns.

Below the simulation models is an icon representing two people, likely indicating human presence or survey data.

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Measurements & surveys

Air temperature	Air velocity	Background questions
RH	CO ₂ concentration	Thermal comfort
Boundary air temperature	Boundary air velocity	Air quality
Weather conditions		General comfort

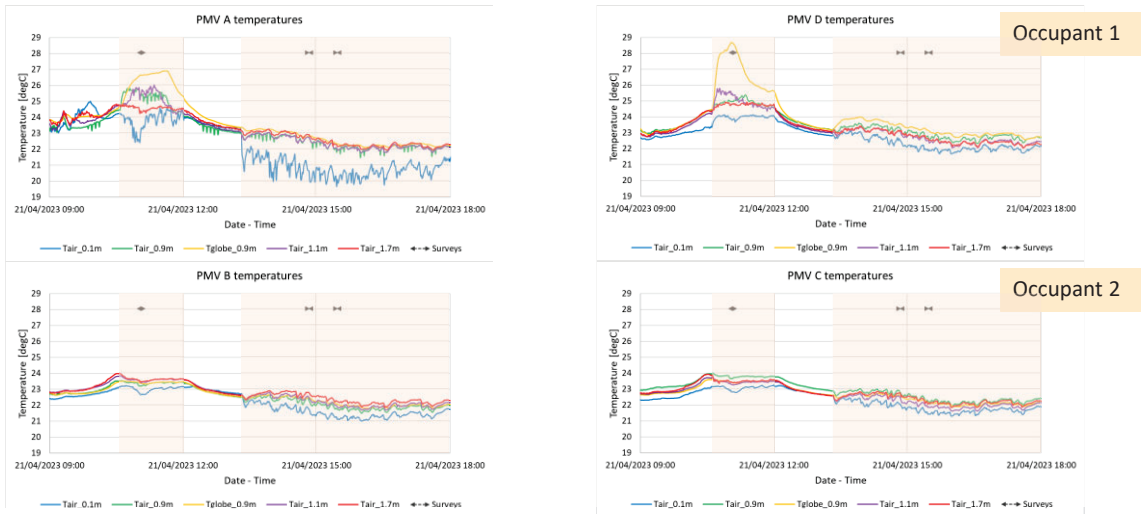
The floor plan diagram shows a room with an 'Openable window' at the top and a 'Door' at the bottom. A North arrow (N) is located in the top left corner. Measurement locations are marked as follows:

- PMV A, B, C, D:** Four measurement points for Predicted Mean Vote, located near the window and door.
- V1, V2, V3, V4, V5:** Five measurement points for air velocity, distributed throughout the room.

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Results – Indoor air temperatures



7

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Results – Draught risks

		PMV A		PMV B		PMV C		PMV D	
		V_0.1m	V_1.1m	V_0.1m	V_1.1m	V_0.1m	V_1.1m	V_0.1m	V_1.1m
Window closed (9:00-10:35)	MEAN	0.07	0.15	0.08	0.08	0.09	0.08	0.08	0.08
	Tu	16	34	56	38	60	37	28	23
	DR	3	13	6	6	8	6	5	4
Window open (10:36-12:00)	MEAN	0.17	0.16	0.09	0.09	0.08	0.08	0.11	0.10
	Tu	74	41	56	36	32	27	31	28
	DR	24	15	8	7	5	5	9	7
Window closed (12:01-1:19)	MEAN	0.10	0.10	0.06	0.09	0.08	0.07	0.10	0.08
	Tu	21	40	18	22	24	21	23	26
	DR	7	8	3	6	5	4	7	4
Window open (1:20-6:00)	MEAN	0.30	0.20	0.14	0.09	0.08	0.08	0.09	0.09
	Tu	64	34	59	29	32	43	33	28
	DR	50	19	16	6	5	6	6	6

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Results – Occupants' perception

Survey	Morning		Afternoon	
	Occupant 1	Occupant 2	Occupant 1	Occupant 2
How are you feeling now?	<i>Slightly warm</i>	<i>Slightly warm</i>	Neutral	Neutral
How do you find this thermal environment?	Comfortable	Comfortable	Comfortable	Comfortable
Please state how you would prefer to be now.	A little colder	Neutral	Neutral	A little colder
Do you feel local heat or cold on some parts of your body?	No	No	No	No
Do you experience draught on some parts of your body?	No	No	No	No
I am satisfied with the air quality in this room.	Neutral	Agree	Agree	<i>Slightly agree</i>
The air in this room is not stale.	<i>Slightly agree</i>	Agree	Agree	Agree
There is a lot of fresh air in the room.	<i>Slightly disagree</i>	Strongly agree	Agree	Agree
The room is properly ventilated.	Neutral	Strongly agree	Agree	Strongly agree
The room has a pleasant smell.	<i>Slightly agree</i>	Neutral	Agree	Agree
The scent of the air in the room does not distract me.	Agree	Agree	Agree	Strongly agree
The air is not dry in the room.	Agree	Agree	Agree	Strongly agree
The air is not dusty in the room.	Agree	Agree	Agree	Strongly agree
Is the window open at this moment?	Yes	Yes	Yes	Yes
At this moment, do you prefer the window open or closed?	I prefer it open	I prefer it open	I prefer it open	I prefer it open
Are there any other factors in the room that bother you?	No	No	No	No
Are you currently experiencing any other symptoms?	No	No	No	No
How are you feeling at the moment?	Neither alert nor sleepy	Alert	Very alert	Some signs of sleepiness

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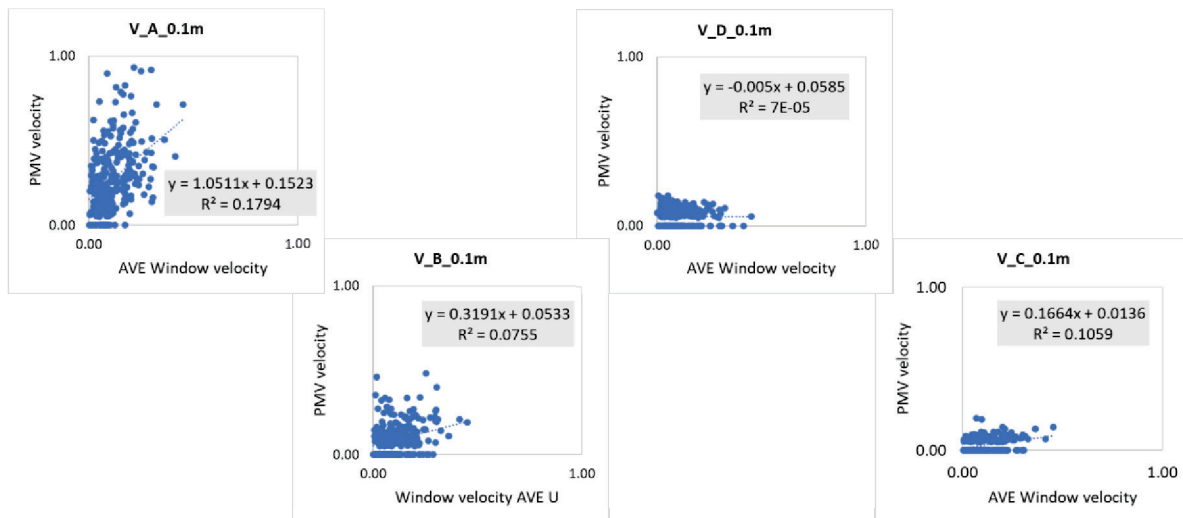
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Results – Air velocity correlation



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Summary & conclusions

- Research objectives, measurement protocol and the initial results of the IEQ investigation in an office room.
- Point-in-time characterisation of the façade operation and its impact on the occupants.
- Measured data to support validation of CFD and building energy models.
- Leading to optimised design and operation of adaptive façade systems.

Thank you for your attention!

m.hajdukiewicz@tue.nl



M. Hajdukiewicz & M.G.L.C. Loomans

'A detailed investigation of the impact of an innovative dynamic façade system on indoor environmental quality in offices'

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TECHNOLOGY

Methodology for evaluating the ventilative cooling potential in early-stage building design

43rd AIVC Conference “Ventilation, IEQ and health in sustainable buildings”

Valentina Radice Fossati, Annamaria Belleri, Dick van Dijk

4-5th October 2023

1

Outline

- Context
- Methodology for evaluating ventilative cooling potential (VC)
- Validation of the methodology
- Results
- Conclusions and next steps

2

Context

What's ventilative cooling?

Ventilation used to **reduce the cooling loads and limit the use of mechanical cooling** in buildings by **utilizing differences between indoor and outdoor temperatures for airborne cooling**

Ventilative cooling potential methodology

Calculation method that allows **engineers/designers/experts** to assess the potential of ventilative cooling in early-stage design

Version 1 developed within the **IEA - EBC Annex 62 - "Ventilative Cooling"**

Version 2 involved the **CEN/TC 156/WG21 - "Ventilative Cooling Systems - Design"**

Calculation Methodology

Ventilative Cooling Potential (VCP) Methodology

Calculation method to evaluate VCP based on a **single-zone thermal model** applied to user-input climatic data on hourly basis

From EN ISO 52016, the **detailed thermal balance calculation was reduced to essential (lumped) parameters** and applied to calculate:

- Free-floating room temperature
- Heating/cooling loads (with and without VC contribution)

1 RC Lumped parameter model

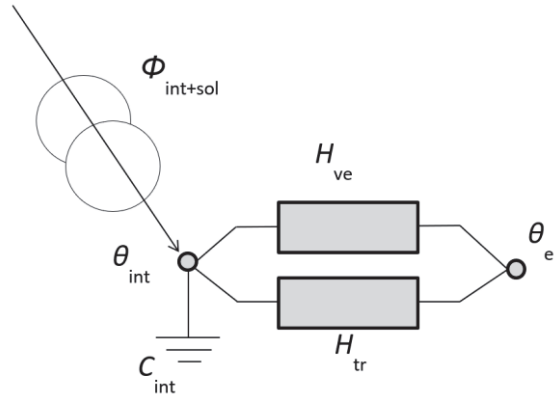


Fig. 1 RC model selected as the most suited model for the early design phase

Overall Energy Balance Equation

At each time interval the heat balance of the building according to the 1 RC model is calculated

$$\left[\frac{C_{int}}{\Delta t} + H_{ve;t} + H_{tr} \right] \theta_{int;t} = \frac{C_{int}}{\Delta t} \cdot \theta_{int;t-1} + [H_{ve;t} + H_{tr}] \cdot \theta_{e;a;t} + \Phi_{int;t} + \Phi_{sol;t} + \Phi_{HC;t}$$

Labels for the equation terms:

- $\frac{C_{int}}{\Delta t}$: (lumped) internal thermal capacity
- $H_{ve;t}$: overall heat transfer coefficient by ventilation
- H_{tr} : overall heat transfer coefficient by transmission
- $\theta_{int;t}$: internal air temperature
- $\theta_{int;t-1}$: Internal air temperature at previous time interval
- $H_{ve;t} + H_{tr}$: overall heat transfer coefficient by transmission
- $\theta_{e;a;t}$: external air temperature
- $\Phi_{int;t}$: internal gains
- $\Phi_{sol;t}$: solar gains
- $\Phi_{HC;t}$: heating/cooling loads
- Δt : timestep

$$\rightarrow A_t \theta_{int;t} = B_t + \Phi_{HC;t}$$

Input and Output

Input

- Climatic conditions
- Building data (building typology and geometry)
- Technical specifications
 - U-value of opaque and transparent envelope
 - g-value of glazing system
 - Shading control setpoint
 - Minimum required ventilation rate
 - Gains
 - Time control

Output

- Hourly distribution of ventilative cooling modes over the year
- Frequency of ACH required to provide potential comfort (VC-mode[2])
- Monthly and annual sensible energy needs of heating/cooling with and without VC

Evaluation Criteria for VCP

Equations based on heating and/or cooling loads

- **VC-mode [0]** – ventilative cooling is not required

$$\text{IF } \theta_{int;0;t} < \theta_{int;set;H} \text{ THEN } q_{V;t} = q_{V;min}$$

- **VC-mode [1]** – potential comfort achieved with minimum airflow rate

$$\text{IF } \theta_{int;set;H} < \theta_{int;0;t} < \theta_{int;set;C} \text{ THEN } q_{V;t} = q_{V;min}$$

- **VC-mode [2]** – potential comfort achieved with increased airflow rate

$$\text{IF } \theta_{int;0;t} > \theta_{int;set;C} \text{ AND } \theta_{e;a;t} \leq (\theta_{int;set;C} - \Delta\theta_{crit}) \text{ AND } \Phi_{HU;e;a;t} \leq \Phi_{HU;max} \\ \text{THEN } q_{V;t} = q_{V;VCS}$$

- **VC-mode [3]** – residual discomfort hours

Methodology Validation

9

Methodology

Comparison of the results of the methodology with those indicated in EN ISO 52016-1:2017

2 test cases consisting of a single thermal zone with different thermal capacity were analysed in the climate of Denver, USA

- **Lightweight case (BESTEST 640)** → wood-based construction system
- **Heavyweight case (BESTEST 940)** → concrete-based construction system

Alignment of the input* according to EN ISO 52016-1:2017

Calculation of statistical error (**CV(RMSE)**) to assess uncertainty degree of the methodology

Validation output:

- **Monthly and annual** sensible energy needs for **heating** ($Q_{H,nd}$) and **cooling** ($Q_{C,nd}$)
- **Monthly average values of operative temperature** ($\theta_{op;av}$)

* for further details see the paper

10

Statistical error

CV(RMSE) (Coefficient of Variation of the Root Mean Square Error)

Errors that allows to delete cancellation errors and measure the model uncertainty degree

$$CV(RMSE) = \frac{RMSE}{a} 100[\%] \quad RMSE = \sqrt{\frac{\sum_{i=1}^n (s_i - m_i)^2}{n}}$$

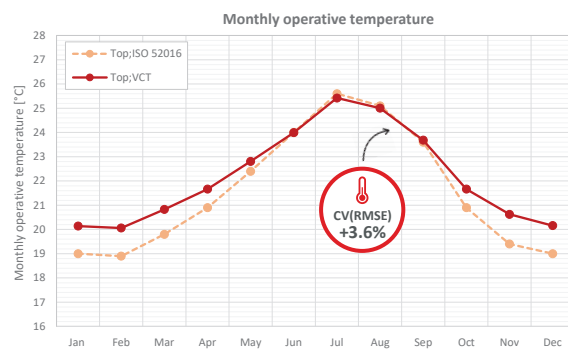
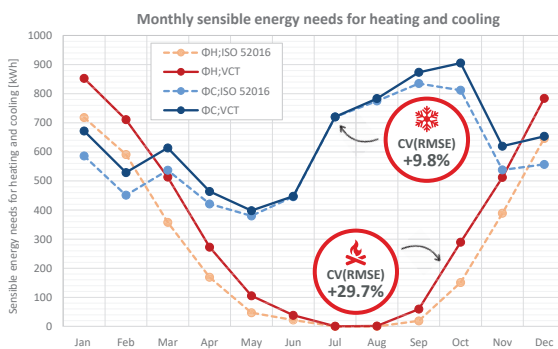
- Acceptable limits according to ASHRAE Guideline 14:

	Monthly criteria	Hourly criteria
CV(RMSE)	15%	30%

* The mentioned statistical errors are generally used to calibrate and validate dynamic simulation software. Since the tested methodology refers to early stage design phase, these errors are just taken as reference.

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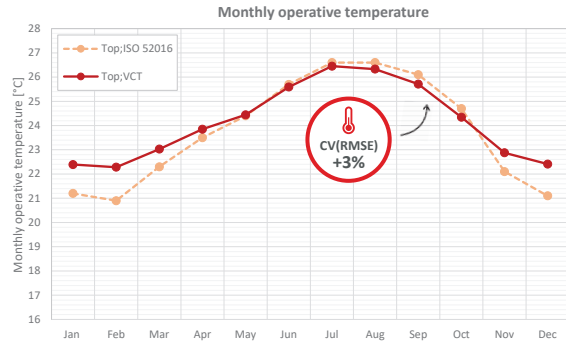
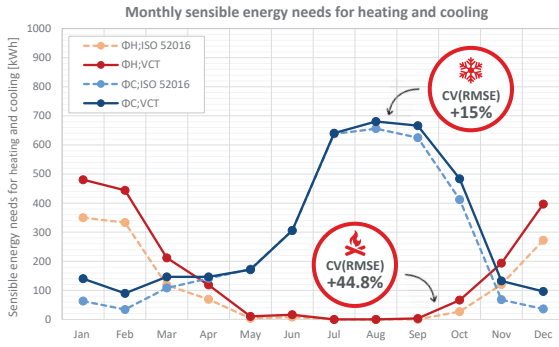
Results - BESTEST 640 (Lightweight)



- $\Phi_{H;annual;VCT} = 4141 \text{ kWh vs } \Phi_{H;annual;bestest} = 3110 \text{ kWh}$
- $\Phi_{C;annual;VCT} = 7680 \text{ kWh vs } \Phi_{C;annual;bestest} = 7058 \text{ kWh}$

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Results - BESTEST 940 (Heavyweight)

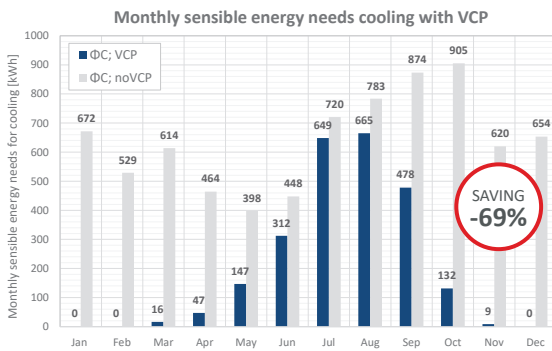


- $\Phi_{H;annual;VCT} = 1941 \text{ kWh vs } \Phi_{H;annual;bestest} = 1301 \text{ kWh}$
- $\Phi_{C;annual;VCT} = 3699 \text{ kWh vs } \Phi_{C;annual;bestest} = 3260 \text{ kWh}$

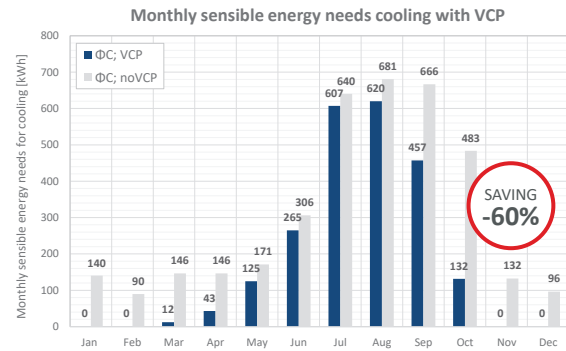
Results - Ventilative cooling potential

- Cooling loads comparison considering the potential of ventilative cooling

BESTEST 640 (Lightweight)

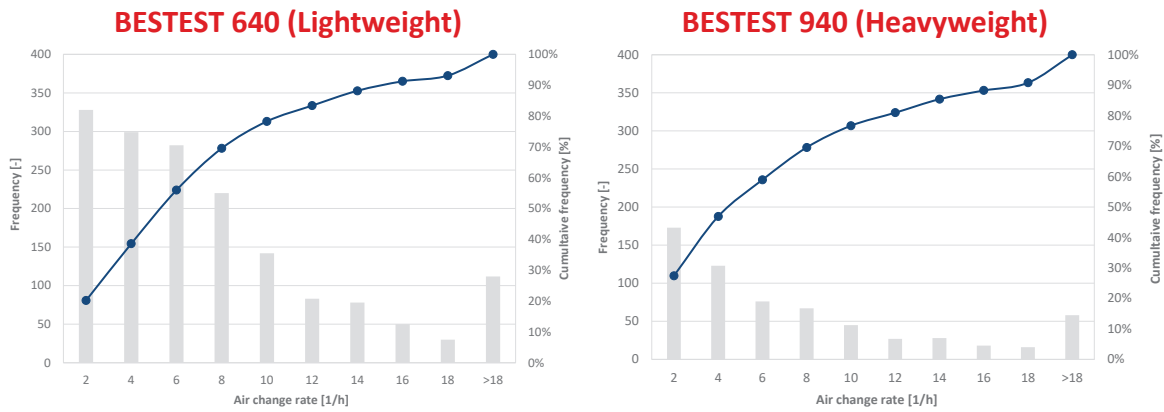


BESTEST 940 (Heavyweight)



Results - ACH required for comfort

- Frequency of ACH required to provide potential comfort (VC-mode[2])



Conclusions and next steps

Conclusions and next steps

Conclusions

- **Validation results are promising** since the level of detail of input data required for the 1RC model is **very low**
- Ventilative cooling tool is useful to **compare the ventilative cooling capacity for different building typologies and thermal capacities**
- The methodology enables to **analyse the effect of other energy efficiency measures on ventilative cooling effectiveness**, such as internal gains reduction, solar gains control and envelope performance

Next steps

- Modelling a shoebox and **compare software and methodology results**
- Implement the **calculation method in Python and release on GitHub**
- Implement **resilience check**

Thank you!

Valentina Radice Fossati, Annamaria Belleri, Dick van Dijk

The authors would like to thank the CEN/TC 156/WG21 TG on "*Ventilative Cooling Systems - Design*" experts for their contributions to the concept development and the interesting discussions.

4-5th October 2023

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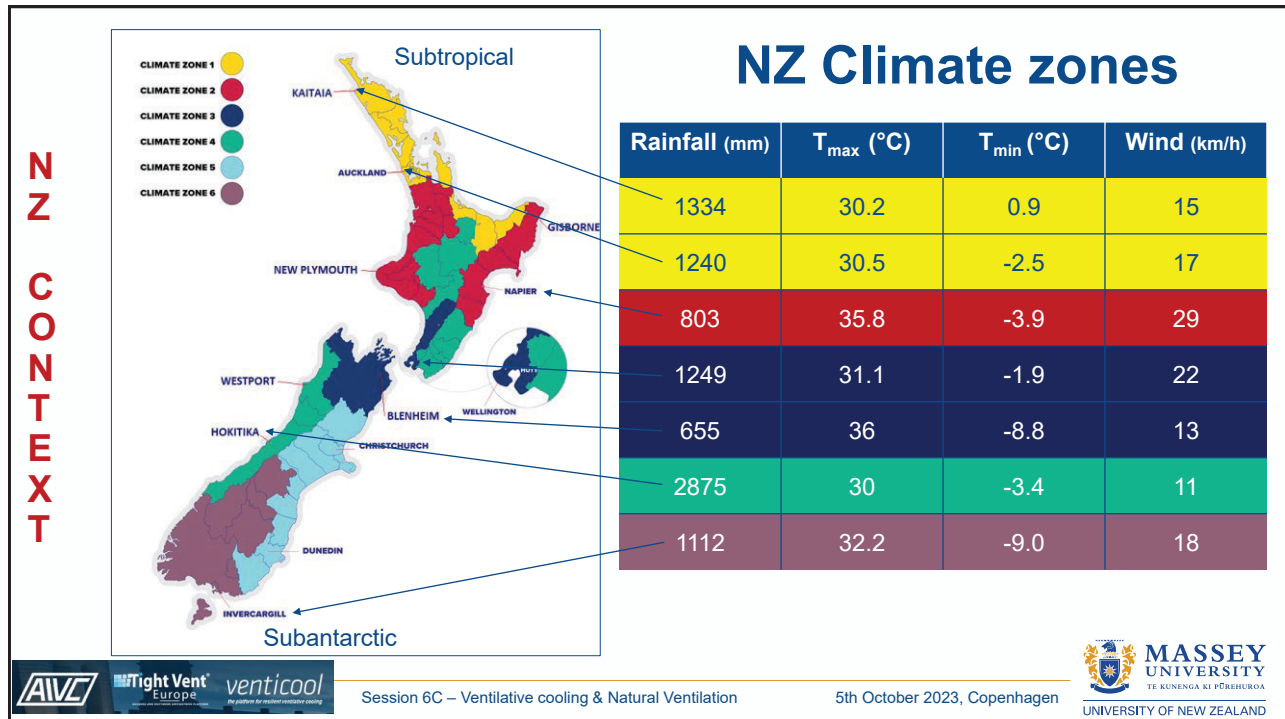
Ventilation reliability: A pilot study on window opening behaviour in a primary school

L Tookey

M Boulic, B McDonald, W Page, P Wargocki, H van Heerden



1



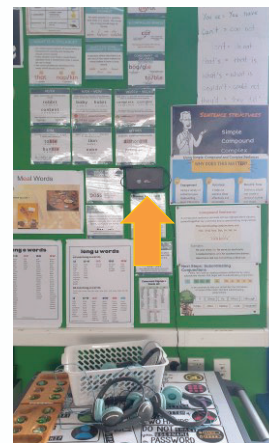
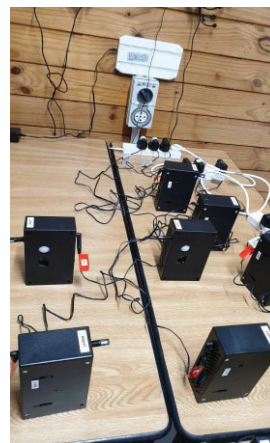
2

METHODS



- Built in 1965
- Six classrooms in one pilot study primary school located in Auckland (Zone 1),
- Data collected in NZ Autumn (March 2023), over three days,
- Visual observation forms completed,
- Discussions with teachers clarifying reasons for window operations,...

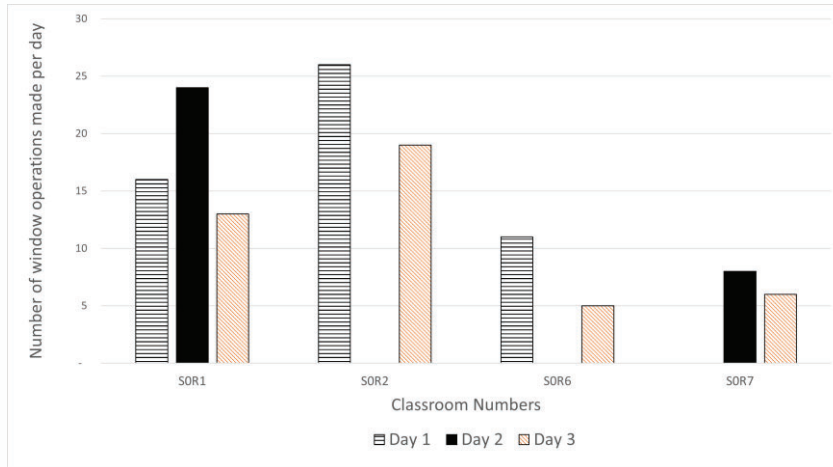
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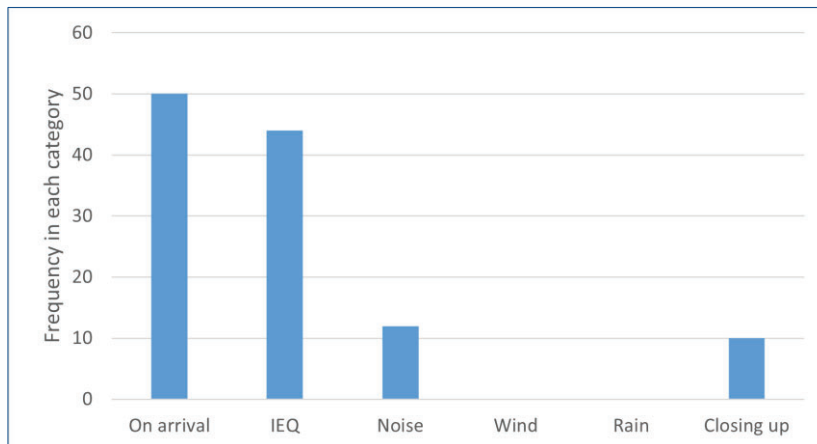
- Windows opening monitored using magnetic sensors,
- Carbon dioxide (CO₂), temperature and relative humidity monitored,
- Ambient measurements from local met station.

4

Number of daily window operations

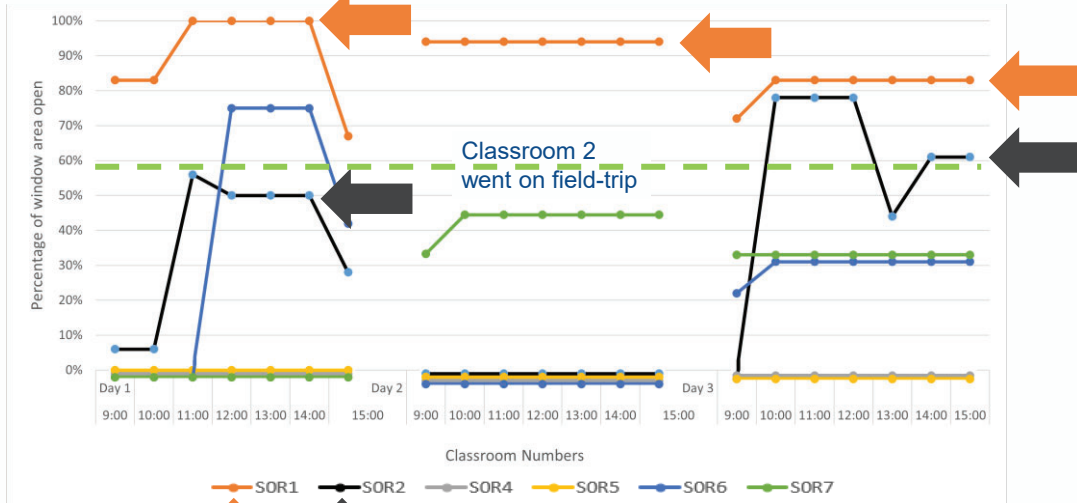


Reasons for window operations



RESULTS

Percentage of window area that is open



Session 6C – Ventilative cooling & Natural Ventilation

5th October 2023, Copenhagen



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Session 6C – Ventilative cooling & Natural Ventilation

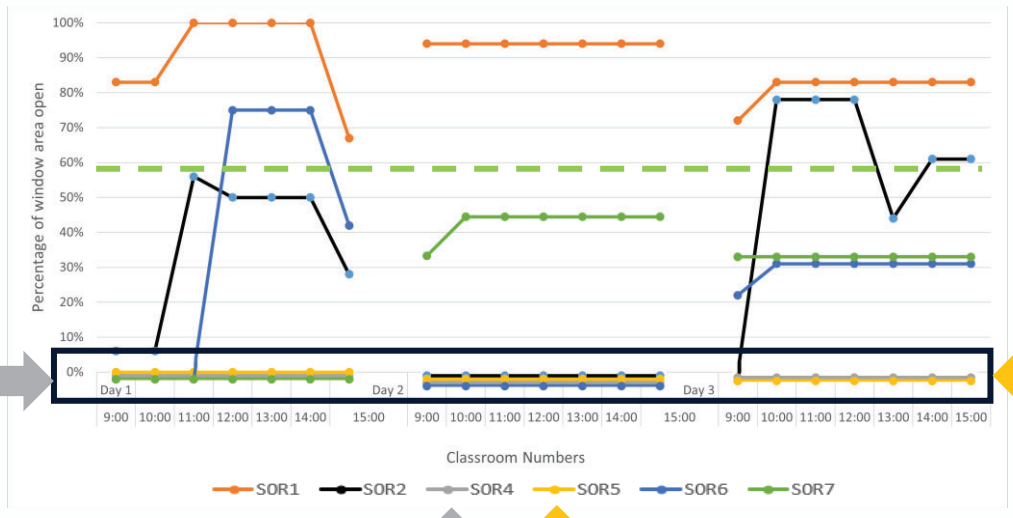
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RESULTS

Percentage of window area that is open



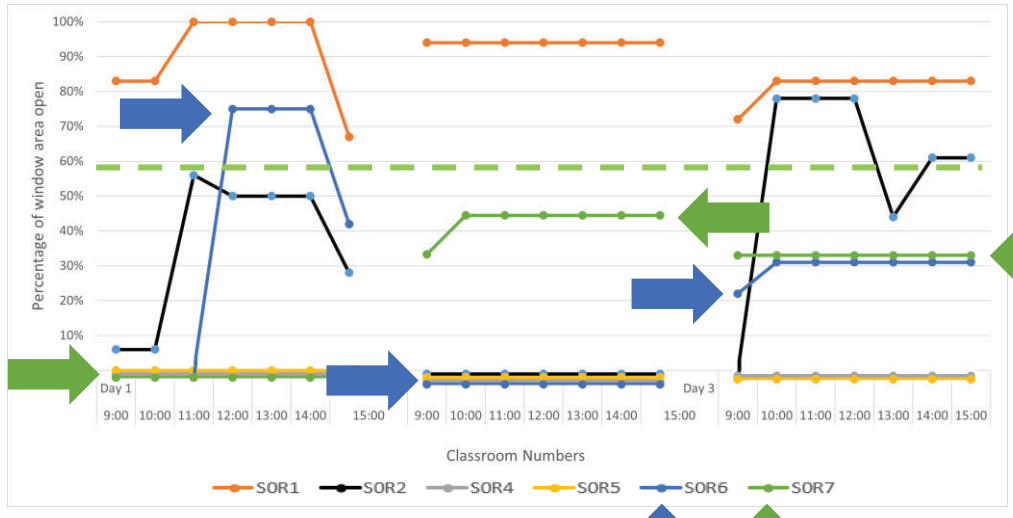
Session 6C – Ventilative cooling & Natural Ventilation

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RESULTS

Percentage of window area that is open



Session 6C – Ventilative cooling & Natural Ventilation

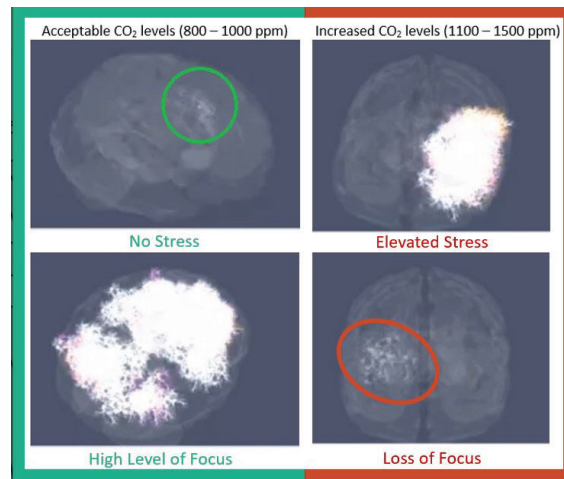
5th October 2023, Copenhagen



CONCLUSIONS

- Extreme difference between six teachers in one school.
- Behaviour difference continues after receiving ventilation best practice notes.
- Observed correlations support changes
 - likely to impact variations in indoor temperature & CO₂ levels.
- Preliminary results good agreement
 - between observations & data from window sensors.
- Next
 - evaluate how indoor temperature & CO₂ levels change based on window opening behaviour.

Next Steps



Ventilation reliability: A pilot study on window opening behaviour in a primary school

QUESTIONS

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“A survey of building design practitioner perceptions of ventilative cooling in their building design processes”

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Contents

- Background
- Overview of the Survey
- Results
- Conclusion
- Future Work and Discussion
- Q/A
- References

Background: Definition of Ventilative Cooling

International Energy Agency's Energy in Buildings and Communities Annex 62 State of the Art Review report (2018) recently defined Ventilative Cooling (VC) as, *'The application of ventilation flow rates to reduce the cooling loads in buildings. VC utilizes the cooling and thermal perception potential of outdoor air. The air driving force can be natural, mechanical or a combination'*.

(<https://venticool.eu/>)

3

Background

Critical Review of Design Processes

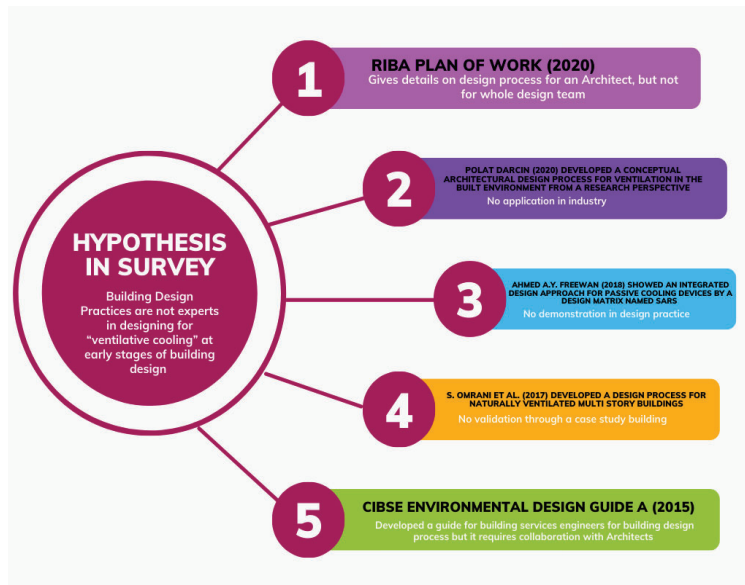


Fig 1: A critical review of a few design processes developed in literature pertaining to ventilation

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Overview of the Survey

- **Title:** "Perception of ventilative cooling in building design practices"
- **Tool:** Survey Monkey
- **Estimated Completion Time:** 17-35 minutes
- **Results Presented in Conference Paper:** 14 completed responses out of 32 in first three weeks (15 March to 7th April 2023)
- **Target Location:** UK and Ireland
- **Time Run:** March 15th to 30th September 2023
- **Question Types:** A mix of open ended and closed ended questions, only closed ended responses are presented in this paper.

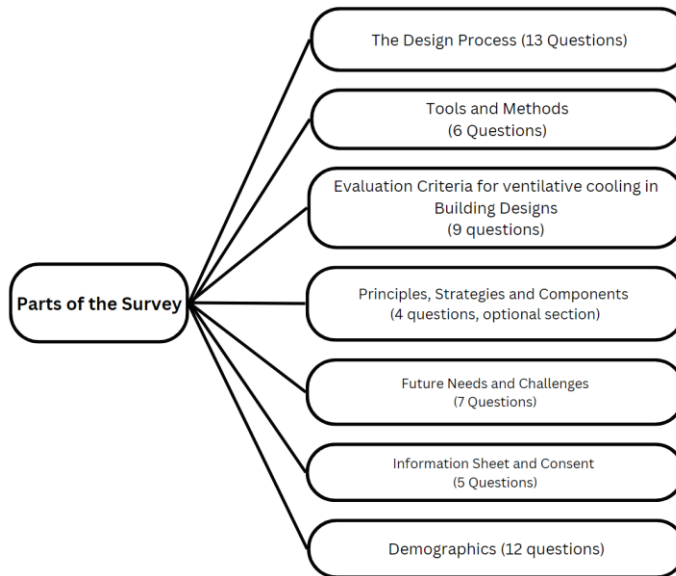


Figure 2: Summary of 57 questions used in the survey

09/10/2023

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Results: Demographics of Survey Respondents

Table 1: Demographics of Survey Respondents

Demographic	Architect	Engineer	Architectural Technologist	Other	Total
Gender					
Male	7	2	1	1	11 (78%)
Female	2	0	0	0	2 (14.3%)
Prefer Not to Say	1	0	0	0	1 (7%)
Age					
35-44	2	0	0	0	2 (14%)
45-54	4	1	0	1	6 (43%)
55-64	4	1	1	0	6 (43%)
65+					
Highest Qualification					
PhD	0	0	0	1	1 (7%)
Masters or Postgraduate	7	1	0	0	8 (57%)
Bachelors	2	1	0	0	3 (21%)
Diploma	0	0	1	0	1(7%)
Other	1	0	0	0	1(7%)

09/10/2023



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Results: The Design Process

Reasons of not agreeing:

- “Design is an iterative process, not a single process can be standardized..”
- “The diagram seems biased towards a commercial building”.
- “additional stages are involved e.g., tender, building regulations etc.”

10/9/2023

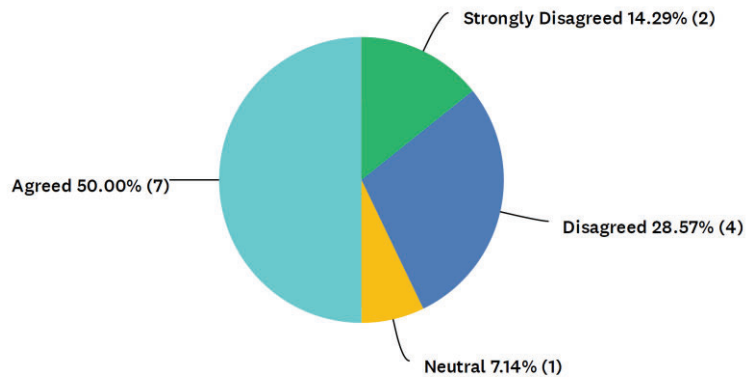


Figure 3: The extent to which respondents agreed or disagreed with the building design process diagram presented to them.

Scan the QR code to see the design process diagram presented in the survey

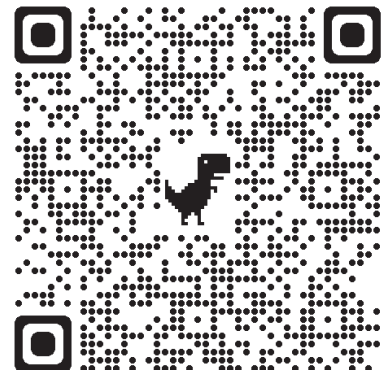


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Results: Tools and Methods

- Around **21%** of the respondents do not use any design tools, **50%** sometimes do and **29%** use a tool normally
- The most popular software used by the respondents is **Passive House Planning package (PHPP)** with around **50%** of the respondents using it, and the second most popular tool is **Integrated Environmental Solutions (IES)** with around **33%** of the respondents using it in their design practices
- **54%** of the respondents are not satisfied with the tools for taking ventilative cooling decision (**this is one of the results, that is changed now**)
- Other than the tools, around **64%** of the respondents rely on their **professional experience**, and **28%** rely on other ways to take a ventilative cooling decision

09/10/2023



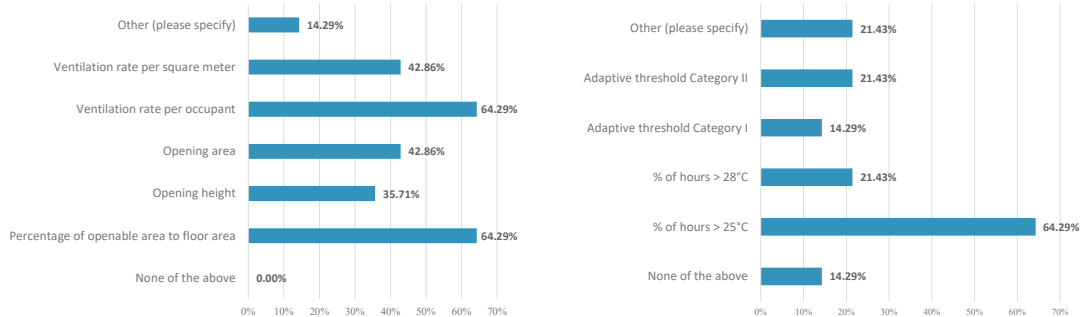
Scan the QR code to see the list of tools presented in the survey

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Results: Evaluation Criteria for Ventilative Cooling in Building Designs

Figure 4: Results of the questions, “Which of the following metrics or variables do you consider at the early design stage for ventilative cooling systems?” and “When evaluating comfort performance in your designs, what upper temperature thresholds do you consider?” from left to right respectively



Results: Future Needs and Challenges

Table 2: Responses of Questions related to Future Needs and Challenges of Ventilative cooling in Building Designs in the Survey

Questions	Answer Option 1	Answer Option 2
Do you think that the built environment design professionals are prepared for accounting for extreme future climate events, such as heat waves, while designing buildings today that will be used many years from now?	Yes (28.57%)	No (71.43%)
In your experience does Natural Ventilation work satisfactorily as a cooling solution against future climate change?	Yes (42.86%)	No (57.14%)
Can Ventilative Cooling play a role in delivering a carbon neutral built environment?	Yes (92.86%)	No (7.14%)

Conclusion of the Survey

- Lack of collaboration at early stages by core design team can lead to poor Ventilative Cooling decisions
- Building design is an iterative process and good ventilative cooling design practices require time, expert advice and often cost in non-domestic buildings at early stages
- Post Occupancy Evaluation or site monitoring is critical in ensuring good ventilative cooling performance after building hand over

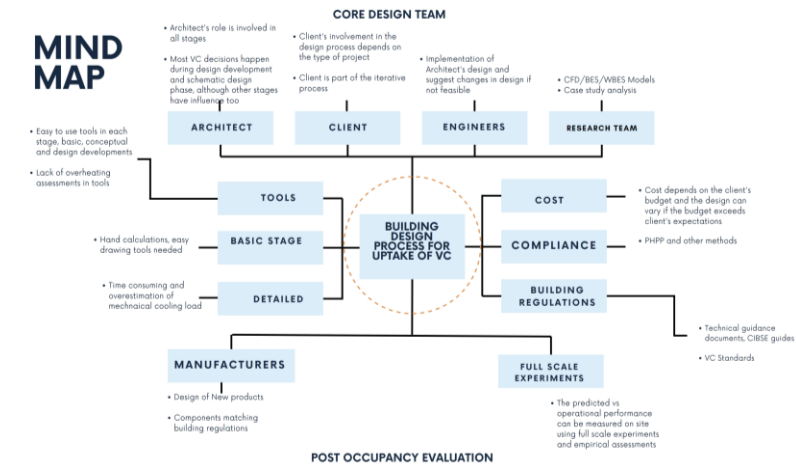


Figure 5: Visualization of the survey responses of how information flows in the design process

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Future Work and Discussion

- Many results have changed in the survey (**40/71** complete responses received by far)
- Survey was too long for many practitioners to complete
- **10** interviews have been completed by far (6 Architects, 4 Engineers)
- Analyse the survey and interview data collectively
- Propose an improved design process diagram and demonstrate it through case studies by numerical modelling

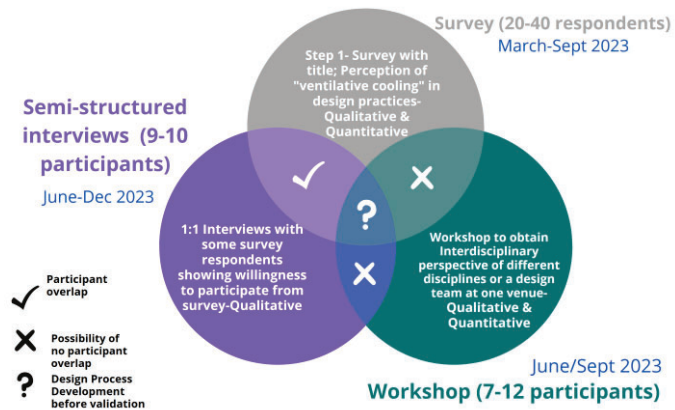


Figure 6: Planned methodology for the study for participatory action research

09/10/2023

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The End Any Questions?

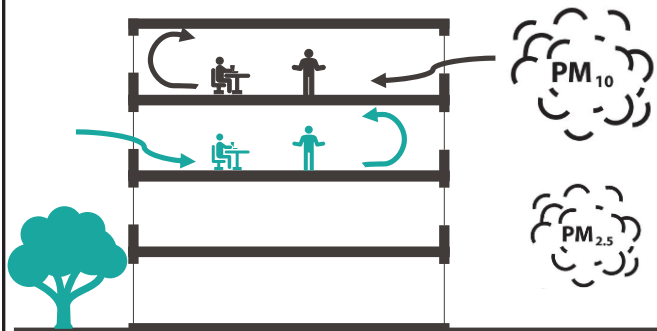
Maha Sohail

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Technological University, Cork, Ireland

5th October 2023

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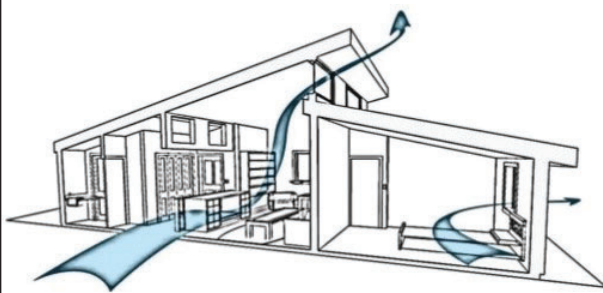


Can naturally ventilated office buildings cope with dusty outdoor air?

Evangelos Belias^{1,2}, Flourentzos Flourentzou², Dusan Licina¹

1 Human-Oriented Built Environment Lab, EPFL

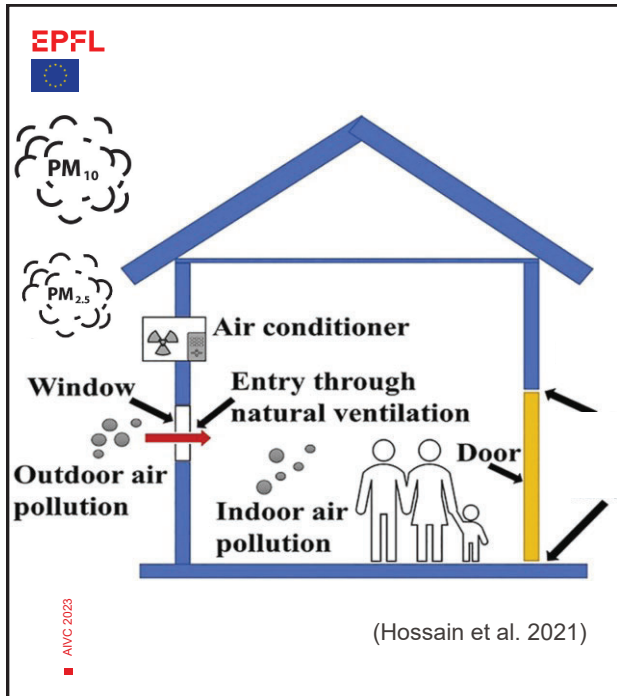
2 Estia SA, Lausanne, Switzerland



Natural ventilation advantages

- Comfort
- Indoor air quality (IAQ)
- Save energy
- Low operational and grey energy demand

(Flourentzou et al., 2017)



Natural ventilation and outdoor air pollution

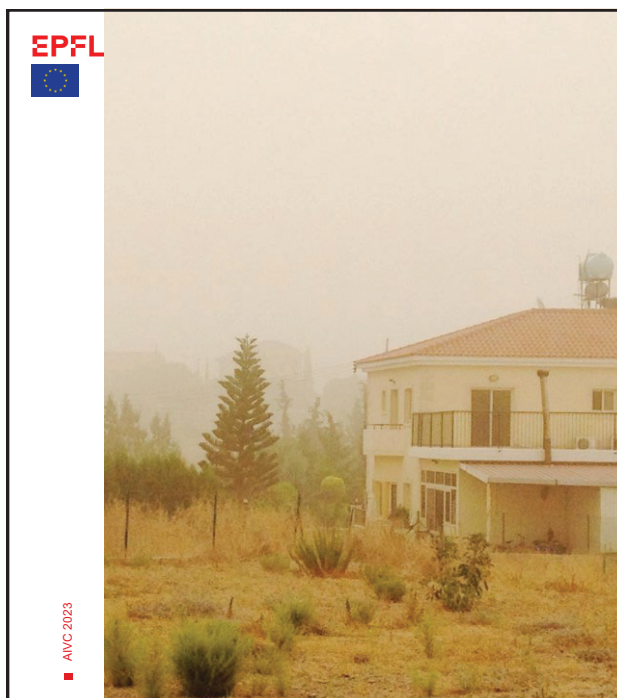
- The use of NV increase the penetration of outdoor air pollutants

(Stabile et al., 2017)

- Particulate matter (PM) provokes severe health impacts

(Logue et al., 2012)

- Lack of studies that examine the resilience of NV in polluted areas



Dust episodes, arid areas and climate change

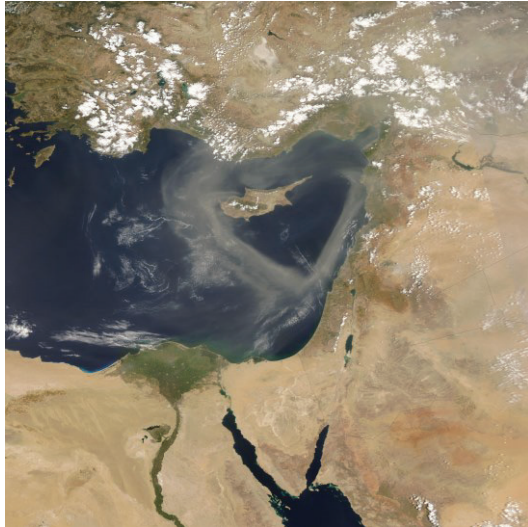
- ~40% of the planet's surface

- 2.1 billion people

(Katra & Krasnov, 2020)

- Need to examine the resilience of NV in such environments

Can naturally ventilated buildings cope with dusty environments?

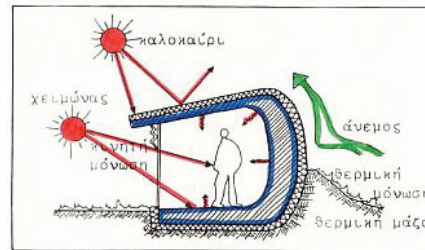


AVC 2023

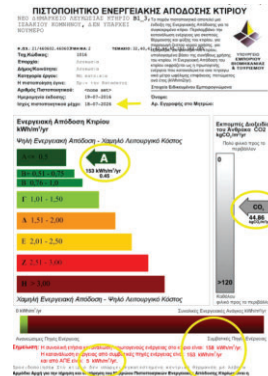
<https://earthobservatory.nasa.gov/>

Investigated building

- Bioclimatic principles
 - High thermal mass/insulation
 - NV – many window opening positions
 - Ceiling fans
 - Solar protection
- AC units for heating/cooling
- Occupant-centered control
- Measurements
 - Energy consumption
 - IEQ – Subjective + Objective

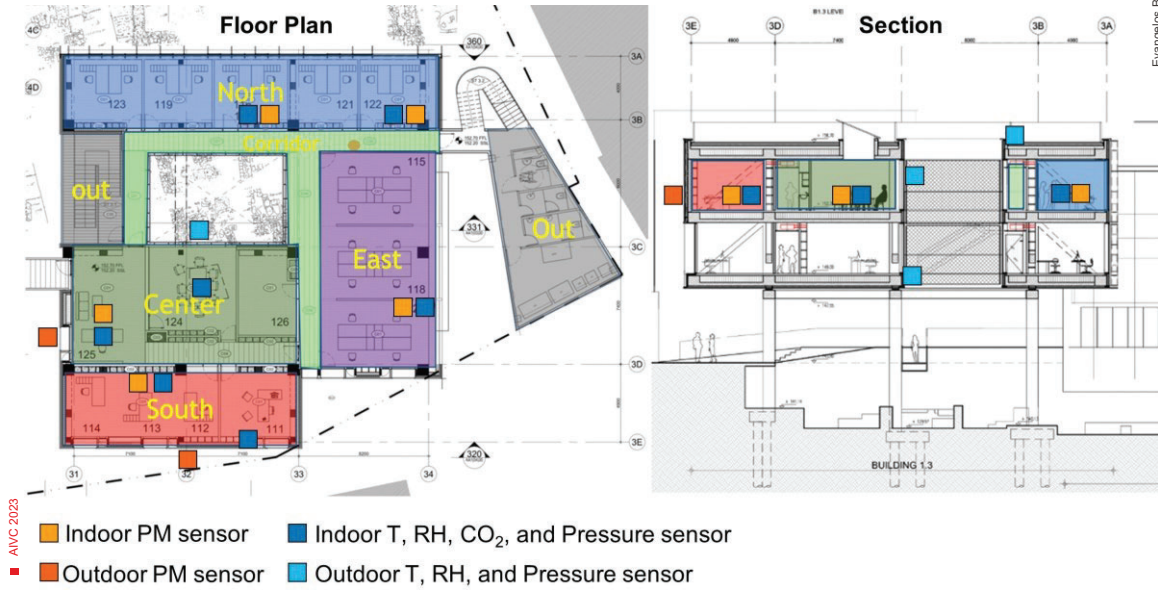


Technical Chamber of Greece, 20702-5/2010



AVC 2023

Investigated building – IEQ monitoring plan

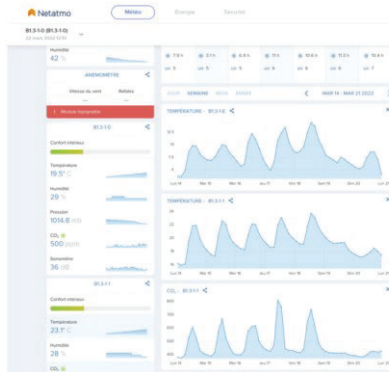


Used sensors

OPC-R2 sensor



<https://www.alphasense.com/>



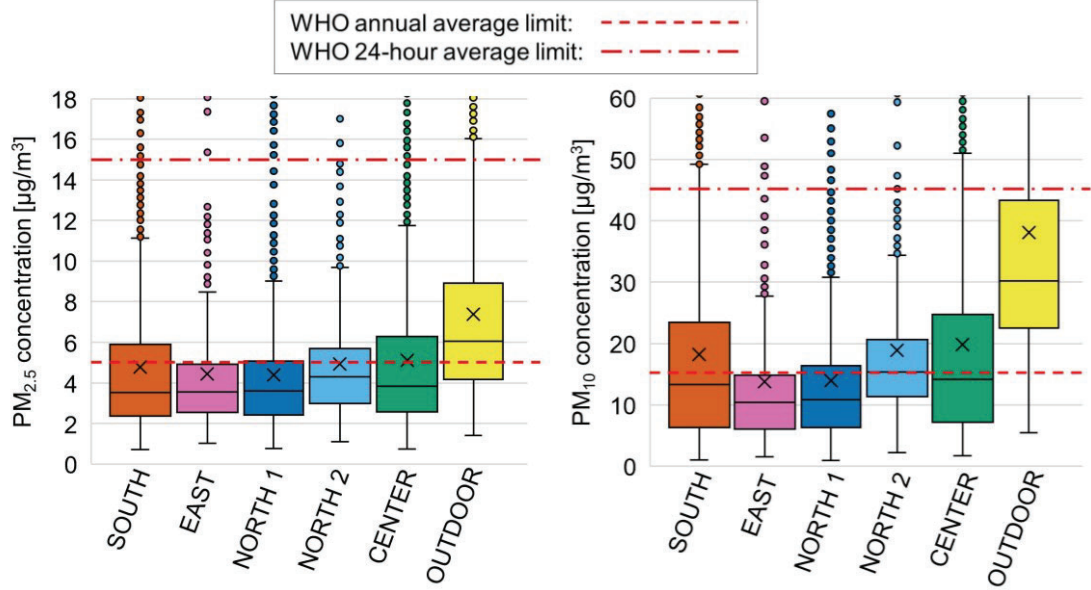
<https://www.netatmo.com/>



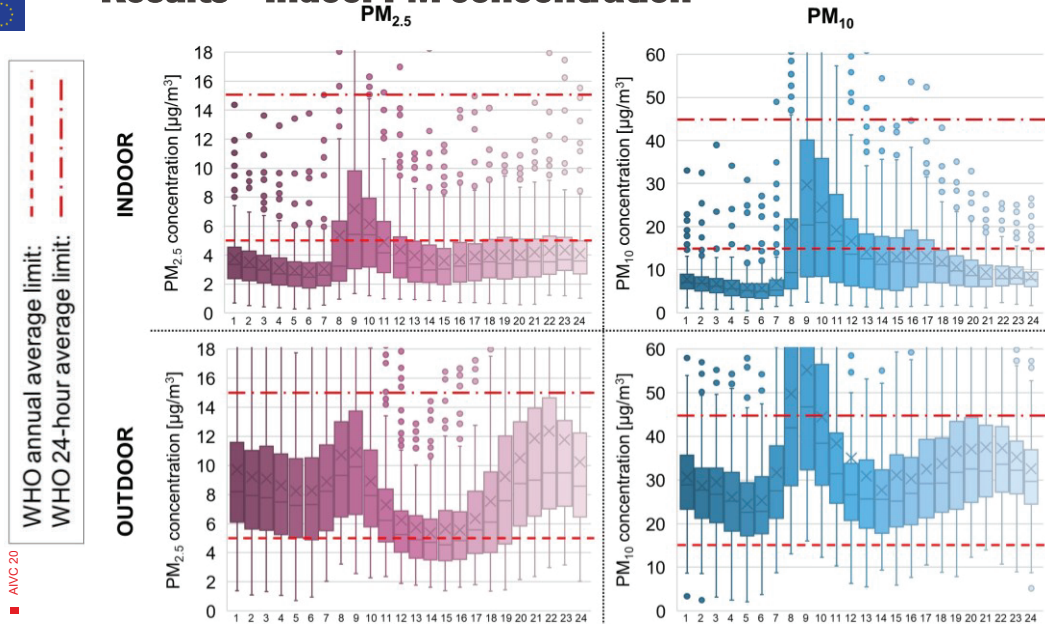
Sufficient accuracy compared to high grade sensors

(Demanega et al., 2021)

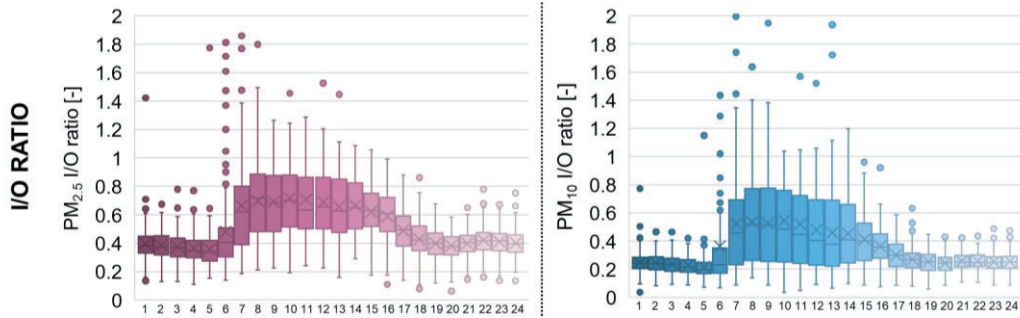
Results - Indoor PM concentration



Results - Indoor PM concentration



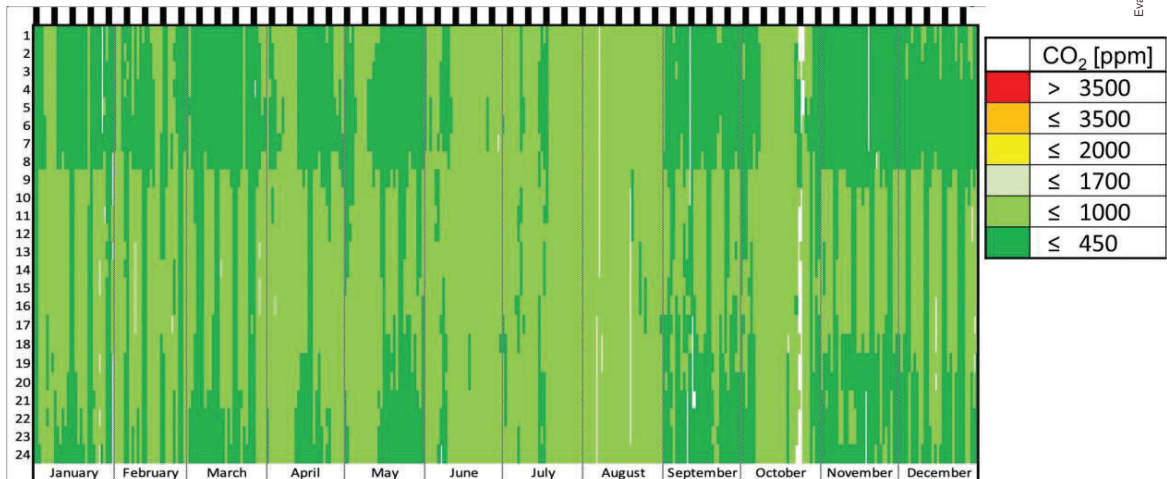
Results – Indoor to outdoor PM ratio



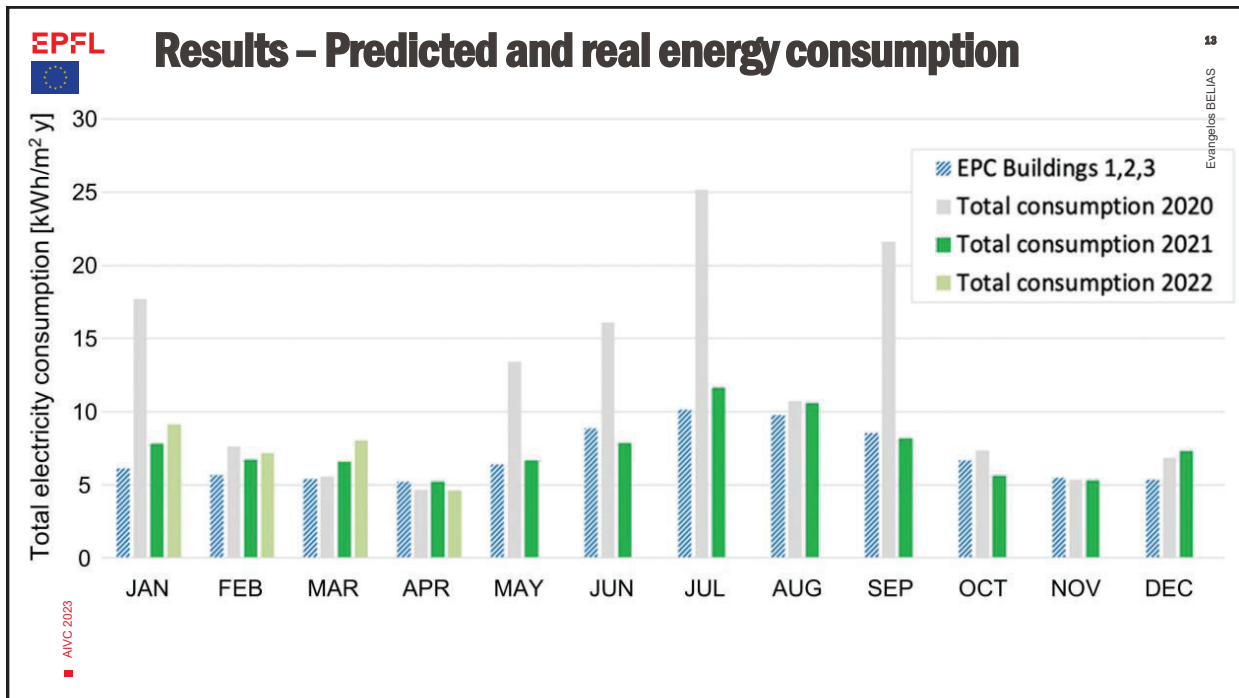
- Outdoor air is the main PM source
- During occupied hours higher I/O due to ventilation and indoor sources

■ ANVC 2023

Results – Indoor CO₂



■ ANVC 2023



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EPFL **Summary – Conclusions** 14

Evangelos BELIAS

■ AIVC 2023

- Indoor PM_{2.5} levels complied with the WHO air quality guidelines in 4 out of 5 monitored spaces
- PM₁₀ levels respected these limits in 2 out of 5 monitored spaces
- Indoor CO₂ concentrations remained below the 1000 ppm limit for more than 90% of the time
- More than 90% of the interviewed occupants evaluated the IEQ as exceptional or very good
- **NV buildings can offer high IEQ with low energy usage even in dusty environments**

Merci. Questions?

14

Distribution of Particulate Matter Concentration and Temperature Stratification Examined by Zonal Model and Experimental Measurements in Room with A Novel Portable Displacement Ventilation Cooling Unit



Session 6C
(Ventilative cooling & Natural Ventilation)
11:00-12:30



OSAKA UNIVERSITY
Toshio YAMANAKA



1

Objectives



Osaka University

• Objective:

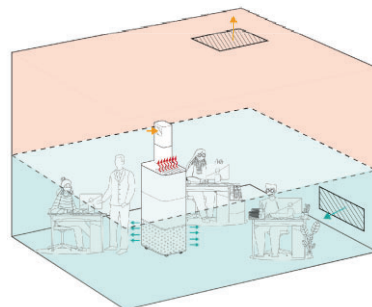
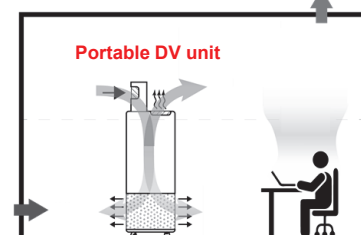
- To investigate the improvement effect of ventilation effectiveness by a novel **portable DV unit** (PDV unit) in the room with displacement ventilation

• Evaluation:

- Vertical temperature distribution
- Vertical distribution of concentration of particle concentration

• Method:

- Zonal Model
- Full-scale experiment



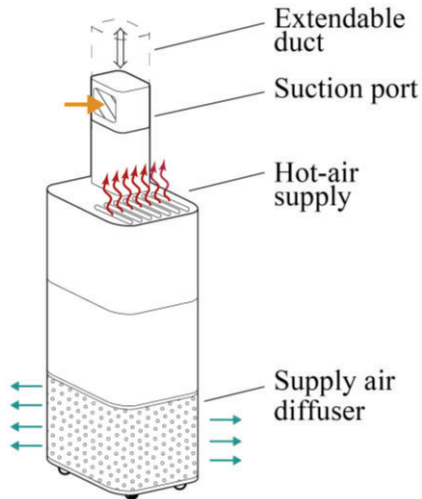
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2

Portable DV unit : Conceptual Design

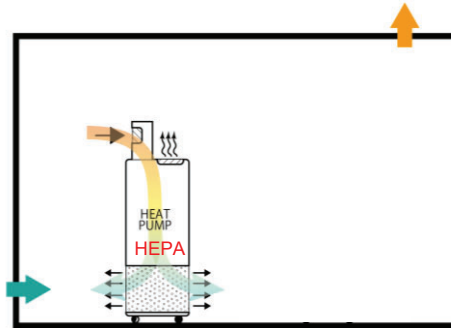


Osaka University



Prototype design of PDV unit

- PDV unit has two functions:
- The first function is cooling and filtering, that is, air cooler and HEPA filter.
- The second function is that exchanged heat is exhausted in the upper zone **to boost the temperature stratification**.



Function 1:
Cooling + Filtering

Function 2:
Hot air exhaust

3

Combination of DV and PDV unit

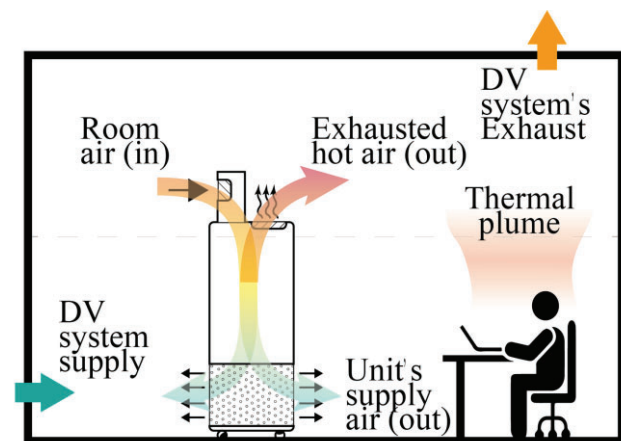


Osaka University

PDV unit is

“Portable air purifier with air cooling and heating function for displacement ventilation”

- Basically, PDV unit is supposed to be placed in the room with DV (Displacement Ventilation).

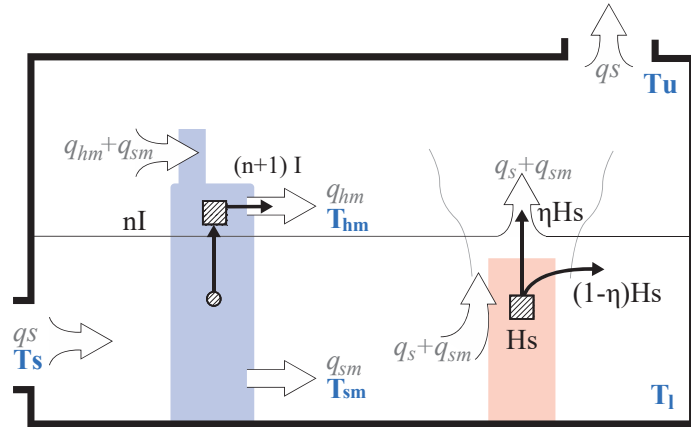


4

Zonal Model



- Basically, this model is based on two zones, that is, the upper zone and the lower zone.
- The model consists of heat balance equations and airflow balance equations.
- This model is used to determine the conditions of parameters for experiment.

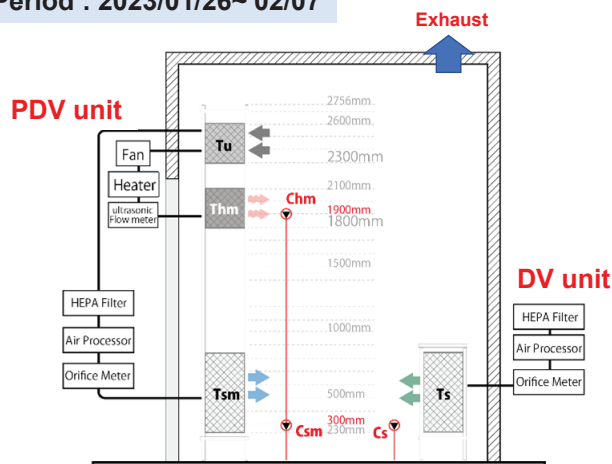


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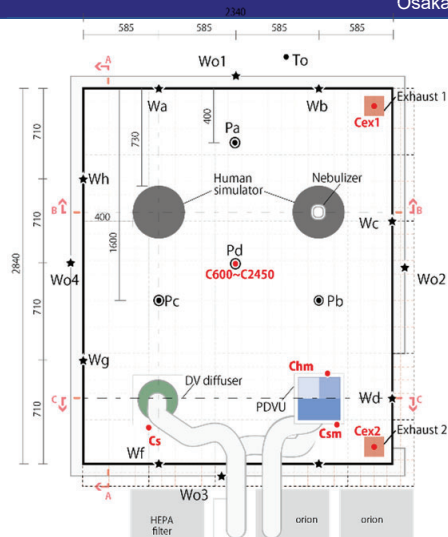
Experimental chamber for full-scale experiment



- Period : 2023/01/26~ 02/07



Section of experimental chamber

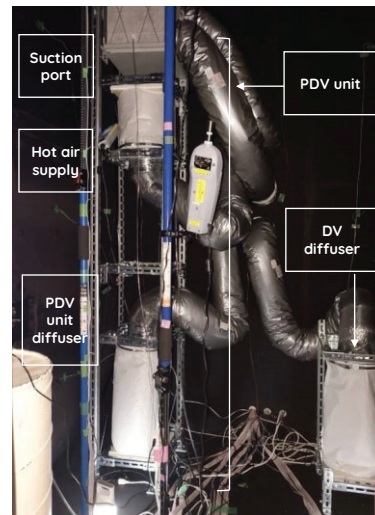
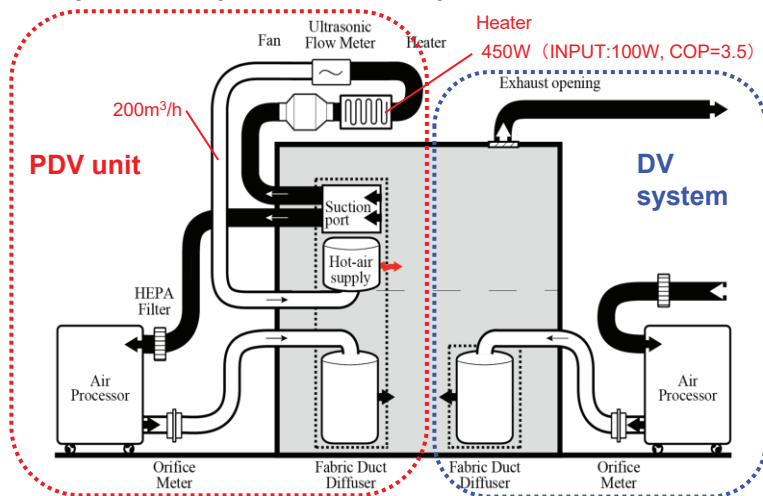


Plan of experimental chamber

6

Experiment equipments

- In fact, the PDV unit is not an all-in-one unit, but composed of separate components in the experiment,

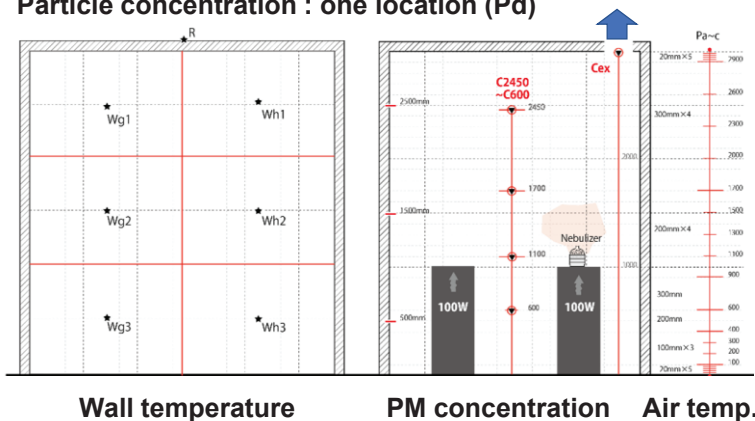


Inside view of the chamber

7

Measurement points of temperature and PM

- Temperature gradient : three locations (Pa, Pb, Pc)
- Particle concentration : one location (Pd)

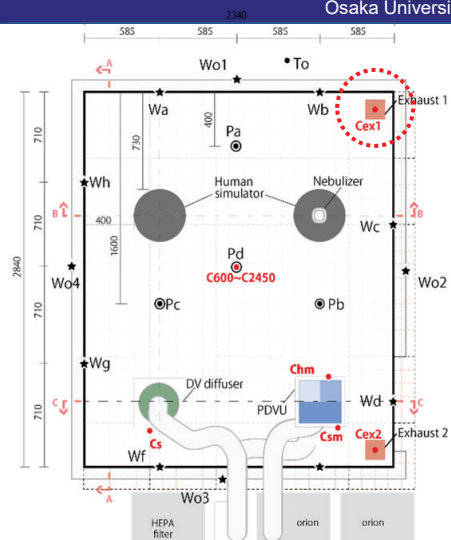


Wall temperature

PM concentration

Air temp.

Section of experimental chamber



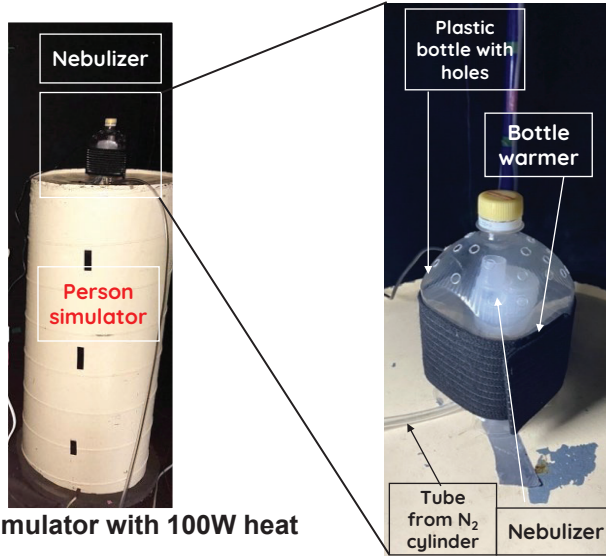
Plan of experimental chamber

8

Two person simulators and nebulizer

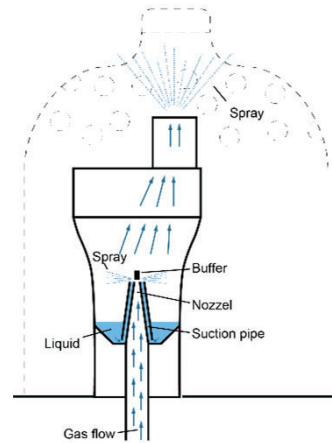


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Person simulator with 100W heat emission

Artificial saliva was used with 12 g salt (NaCl) and 76 g glycerine for 1 L of distilled water.



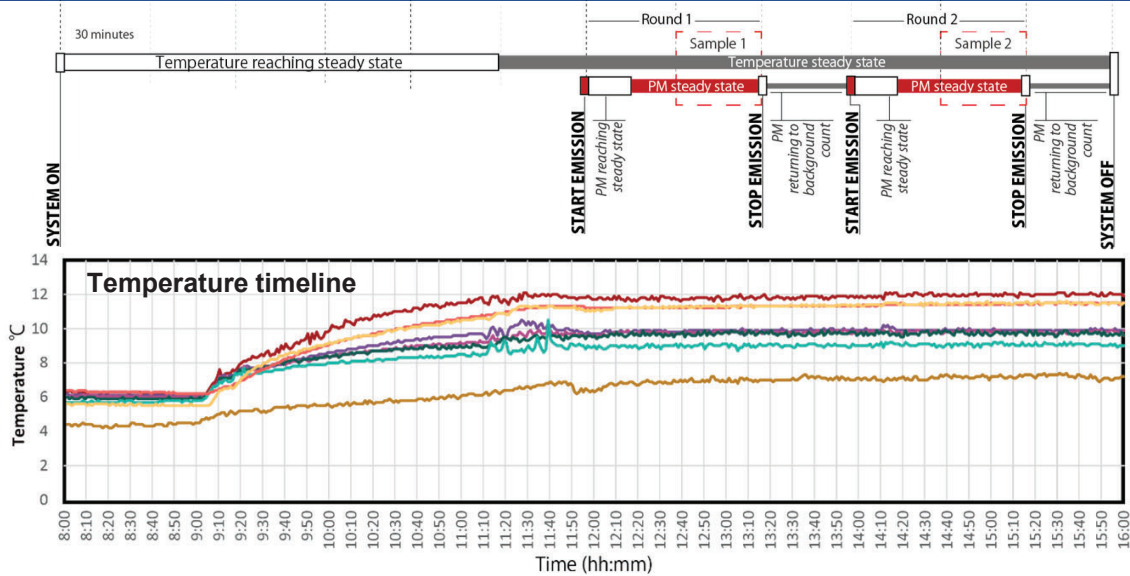
Section of nebulizer

9

Time schedule of experiment



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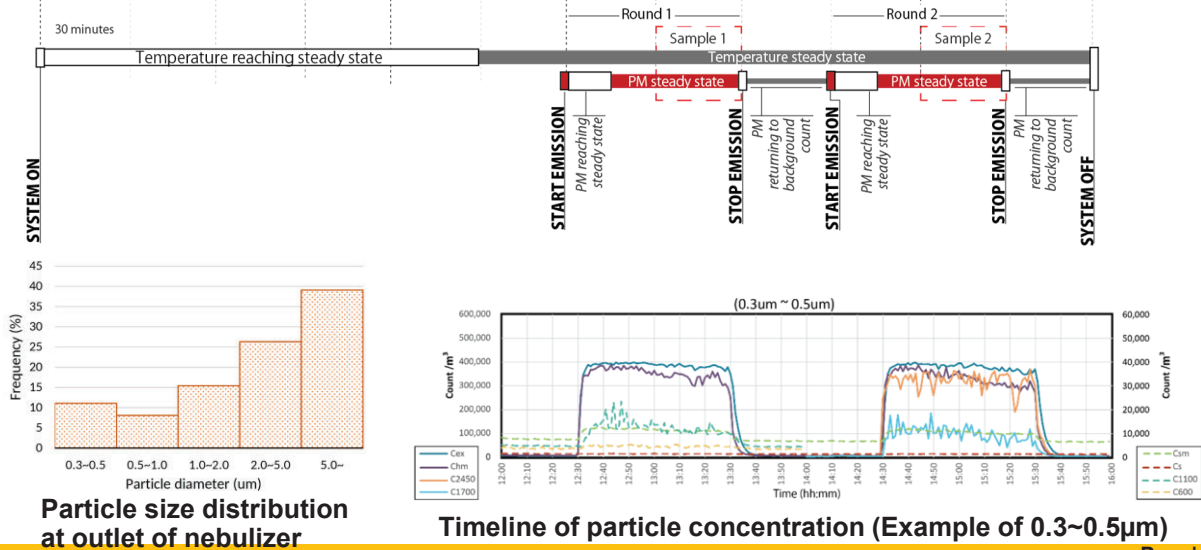


10

Experiment timeline and PM concentration



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11

Experimental cases



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Case		Flow rate (m ³ /h)	
		qsm	qs
1	DV200	0	200
2	DV200_PU100	100	200
3	DV200_PU200	200	200
4	DV200_PU300	300	200
5	DV100_PU200	200	100
6	DV100_PU100	100	100
7	PU200_Ex1	200	0

- Airflow rate of PDV unit
- Airflow rate of DV
- ➔ Base case of DV 100m³/h only
- ➔ DV200 +PDV unit 100~300 m³/h
- ➔ DV100 +PDV unit 100~200 m³/h
- ➔ Special case of PDV 200m³/h only without any ventilation

12

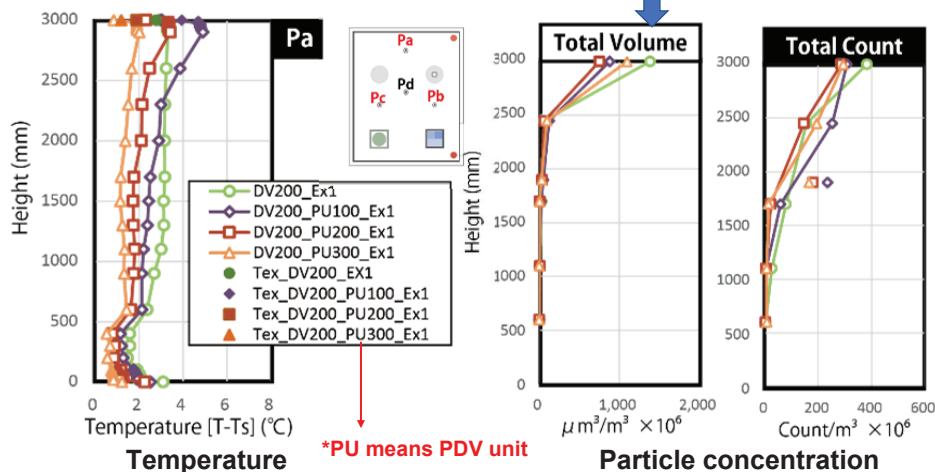
- **Three groups for investigation:**

- **Group 1** : Changing PDV unit's supply flowrate (q_{sm}) with the same DV flowrate
- **Group 2** : Changing DV supply flow rate (q_s) with the same PDV unit flowrate
- **Group 3** : Changing the ratio of q_s & q_{sm} under the same total flow rate

Group 1: Effect of Changing PDV unit's supply flowrate

- Constant DV flowrate (200 m³/h)
- Varied PDV unit flow rate (0~300 m³/h)

Total volume of particles with different diameters



Temperature:

- It is observed that cases with PDV unit of 100 or 200 m³/h have a stronger temperature stratification.
- When PDV unit flowrate is the larger, the overall temperature becomes the lower.

Particle concentration:

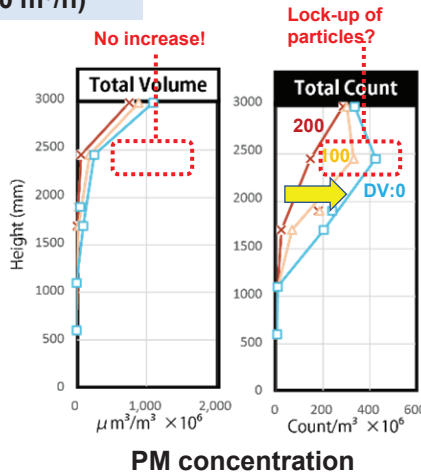
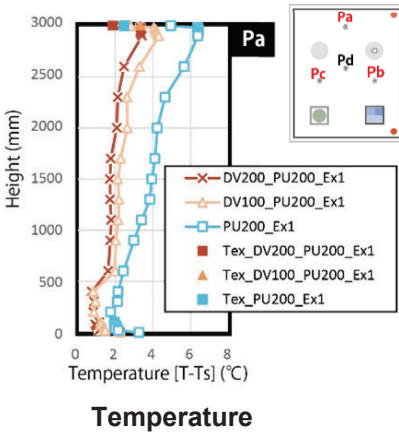
- The effect of the PDV unit's flowrate on the particle concentration is not so large.

Group 2: Effect of Changing DV supply flowrate



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- Constant PDV unit flowrate (200 m³/h)
- Varied DV unit flow rate (0~200 m³/h)



Temperature:

- Decreasing the DV supply flowrate from 200 to 0 m³/h cause a shift in the temperature distribution.
- Especially, the case without DV (that is PDV only) shows the increase of temperature ranging from 1 to 2 °C with a stronger temperature stratification.

Particle concentration:

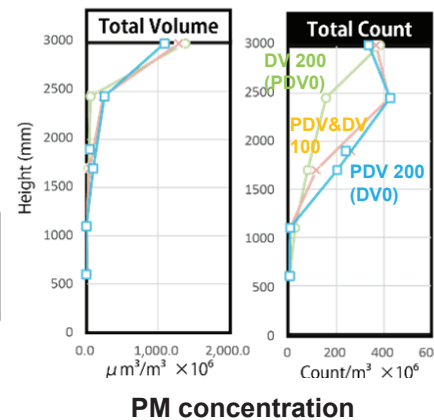
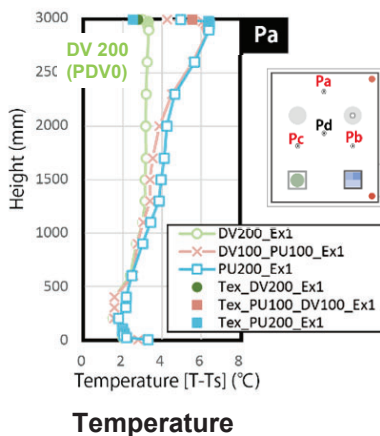
- Decreasing the DV airflow rate brings the increase of the particle count and slight increase of volume concentration.
- There is no particles in the lower zone even if no ventilation.
- PDV unit has the distinct effect of cleaning air in occupant zone and increase of stability due to strong temperature stratification.

Group 3: Effect of Changing flowrate ratio



Osaka University

- DV flowrate + PDV unit flow rate is constant at 200 m³/h



Temperature:

- The case of DV200 shows the lowest temperature in upper zone.

Particle concentration:

- Increasing PDV unit airflow rate causes an increase in the top zone particle count.
- It can be said that DV is more effective than the PDV unit if the airflow rate is the same, but PDV unit has the capacity of air purification in the occupancy zone.
- Purification capacity of PDV unit depends on suction location!

INPUT and Imaginary COP in Experiment



Osaka University

b. PDV unit capacity, COP, and required input power

Cases		Experiment			
		Q _{sm} (W)	Q _{hm} (W)	COP(-)	I(W)
Far Exhaust	DV200_PU100_Ex1	217	411	1.1	195
	DV200_PU200_Ex1	379	329	-7.6	-50
	DV200_PU300_Ex1	428	407	-20.7	-21
	DV100_PU200_Ex1	316	454	2.3	138
	DV100_PU100_Ex1	228	417	1.2	189
	DV300_PU300_Ex1	348	445	3.6	97
PU only	PU200_Ex1	507	385	-4.2	-122

Input power and COP:

- Supposed Input power(100W) and COP(3.5) of the cooling unit are turned out to be not the same as the calculated ones based on the measurement data in the experiment. This might be caused by energy loss through ducts.

Zonal model

Q _{sm} (W)	Q _{hm} (W)	COP(-)	I(W)
350	450	3.5	100

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Concluding remarks



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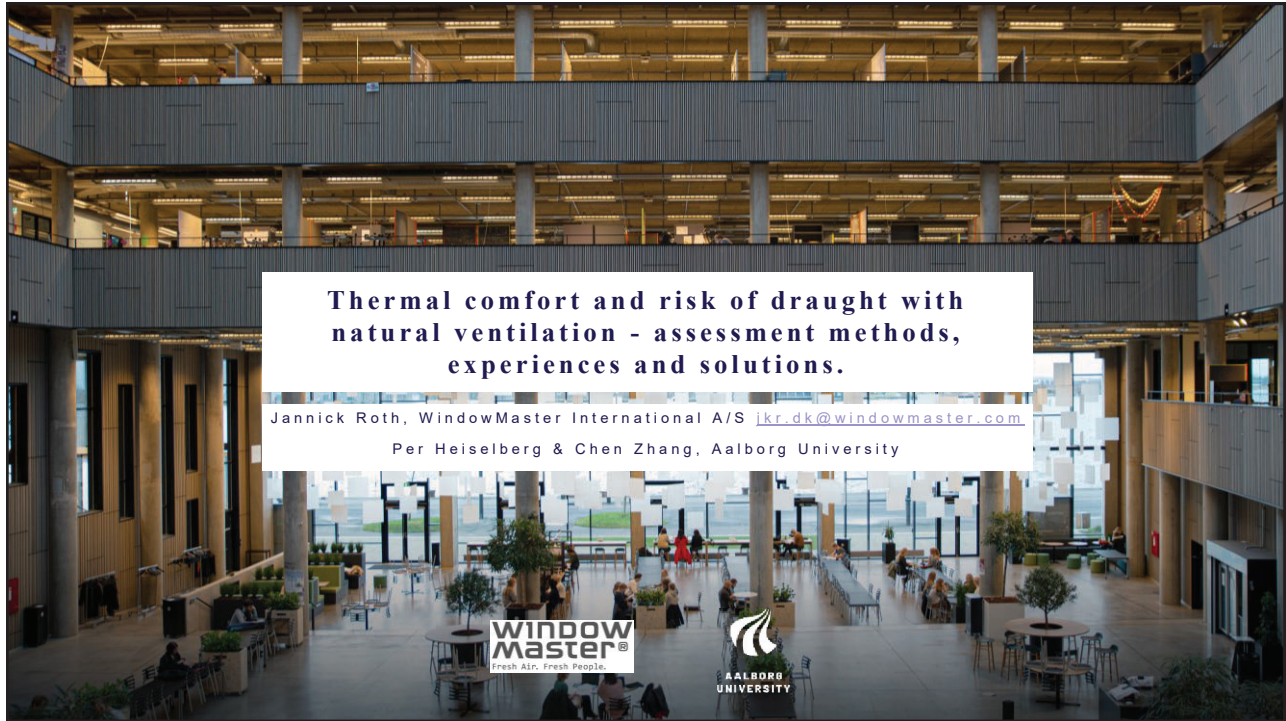
- **The novel PDV unit** can enhance the temperature stratification and reduce the particle concentration in the occupied zone with Displacement Ventilation.
- Additionally, **this PDV unit** could work as a replacement system for Displacement Ventilation system.
- **As for the effect of parameters:**
 - **Increasing PDV unit flowrate** reduces temperature at all heights
 - **Decreasing DV flowrate** supplemented by the PDV unit have no effect on occupied zone (lower zone)
 - **PDV unit alone is** potentially effective in cooling and filtering the occupied zone, and the height and location of suction opening is important.
- **Improvement:**
 - **Filtration** will be recommended to be applied for hot air exhaust as well.

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Thank you for listening!






Thermal comfort and risk of draught with natural ventilation - assessment methods, experiences and solutions.

Jannick Roth, WindowMaster International A/S jkr.dk@windowmaster.com

Per Heiselberg & Chen Zhang, Aalborg University



1



Objectives


Estimating the risk of draught for natural ventilation systems

The key objectives

- Is the current (Fangers) Draught Rate method suitable for the evaluation of natural ventilation and are there currently other more appropriate methods for assessing the risk of draught?
- What are the main findings and experiences until now and to what extent can we use these?
- Examples of solutions for ensuring thermal comfort in cold periods.

Why

- The majority of research and hence the assessment methods and tools for thermal comfort assessment of ventilation systems are not based on findings for natural ventilation solutions and do not take into account the specific characteristics of natural ventilation.
- This has created a lack of suitable methods for the assessment and performance evaluation of natural ventilation.

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2

Assessment methods - Fanger

Fanger – risk of draught

Predicted percentage dissatisfied (%) expressed by:

$$DR = (34 - ta,l)(\bar{v}a,l - 0.05)^{0.62}(0.37 \cdot \bar{v}a,l \cdot Tu + 3.14)$$

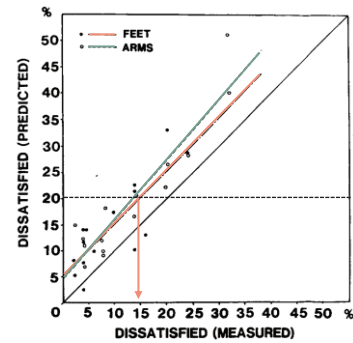
Where:

- *DR* is the predicted percentage dissatisfied, %
- *ta,l* is the local air temperature, in degrees Celsius, 20 °C to 26 °C;
- *$\bar{v}a,l$* is the local mean air velocity, in metres per second, < 0,5 m/s;
- *Tu* is the local turbulence intensity, in percent, 10 % to 60 %

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Observations

- Model applies to people with sedentary activity and with a neutral thermal sensation for the whole body.
- Model is designed to predict the draught rate at the neck level.
- Overestimation is expected when predicting the draught at the arm or feet level.



3

Assessment methods - Berkeley

Berkeley (Schiavon et al 2016; Liu et al 2017)

- The maximum air speed at ankle level can be derived using:
- $V_{\text{ankle}} < 0.35 \cdot TS + 0.39$

Where:

- V_{ankle} air speed at 0.1m above floor, m/s
- *TS* whole body thermal sensation; Equal to PMV calculated using the input air temperature and speed averaged over two heights: 0.6m and 1.1m for seated occupants and 1.1m and 1.7m for standing occupants.

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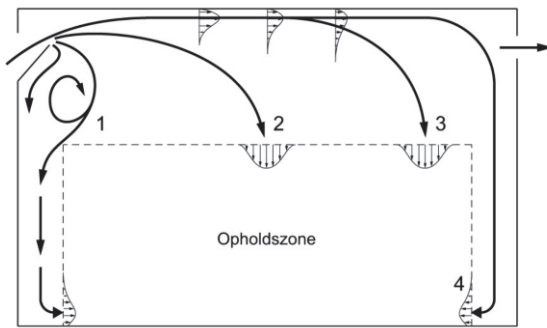
Observations

- A new draught risk assessment method has been added in the recent ASHRAE 55-2020
- The method assesses the risk of draught at the ankle region, 0.1m above floor level, valid for clo < 0.7 and met < 1.3.

4

Airflow pattern with high-positioned openings

Typical air distribution conditions



Explanation of air distribution conditions

Flow regime	Typically occurring	Flow pattern
1	Small driving forces (0.2 – 0.4 Pa) or low outdoor temperature supply air (high indoor/outdoor temperature difference).	Air distribution in the room will follow the displacement principle and the draught risk will be highest along the floor.
2 & 3	Driving forces ($\Delta p > 4-6$ Pa) or higher outdoor temperatures ($\Delta t < 5$ K).	Air distribution in the room will act as a thermal jet and traditional jet theory can be used to predict airflow path and draught risk will typically be highest at neck level.
4	For bottom-hung windows close to the ceiling during warmer outdoor temperatures	Air distribution in the room will act as an isothermal jet.

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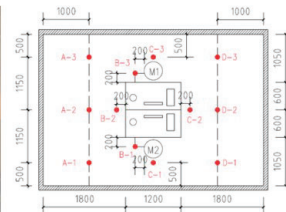


5

Laboratory measurements

From research projects

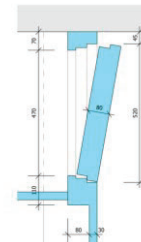
“Natural cooling and ventilation through diffuse ceiling supply and thermally activated building constructions” – AAU uni.



- Scenario 1: Heat load 30 W/m², a constant **ACH of 2** (85,5 m³/h)
- Scenario 2: Heat load 60 W/m² and a constant **ACH of 4** (171 m³/h).
- Temperature difference (indoor/outdoor air) varied from **0-32K**.
- Tu = 20%
- Named hereafter: **Zhang et al.**

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”Naturligvis” (Naturally) - SINTEF



- Scenario 1: a constant **ACH of 2** (50,4 m³/h)
- Scenario 2: a constant **ACH of 4** (104 m³/h)
- Temperature difference (indoor/outdoor air) varied from **23-25K**.
- Tu = 20%
- Named hereafter: **Gunnar et al.**



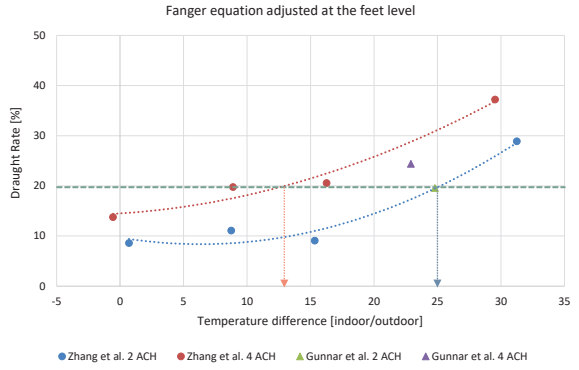
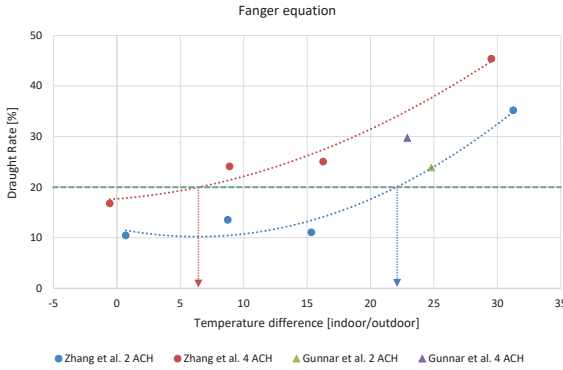
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Fangers equation

Based on the highest DR in the room

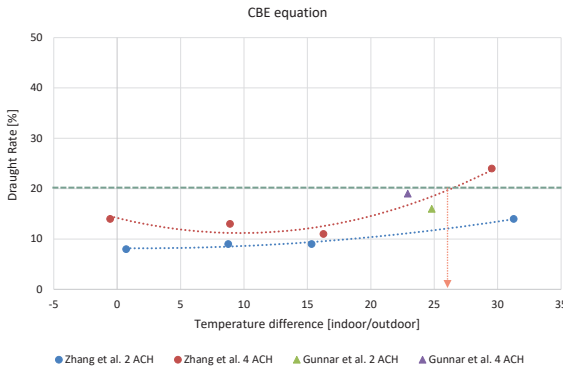
A draught rate < 20% can be achieved at a temperature difference below 6 K and 22K for 4 and 2 air changes, respectively.

A draught rate < 20% can be achieved at a temperature difference below 13 K and 25K for 4 and 2 air changes, respectively.



Berkeley (Schiavon et al 2016; Liu et al 2017)

A draught rate < 20% can be achieved at a temperature difference below 26K for 4 air changes.

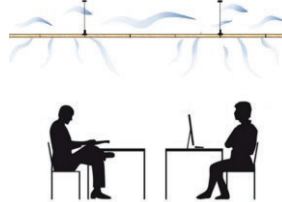


Solutions coping with draught

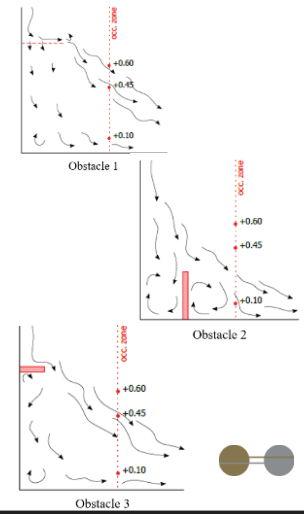
Examples

Goal	Solution
Higher air inlet temperature	<ul style="list-style-type: none"> Double skin façade solutions
Lowered air velocities	<ul style="list-style-type: none"> Diffuse ceiling supply Obstacles e.g. a perforated plate like a window sill. Radiator below incoming air.
Higher air inlet temperature and lowered air velocities	<ul style="list-style-type: none"> Diffuse ceiling supply

Zhang et al. – Diffuse ceiling ventilation

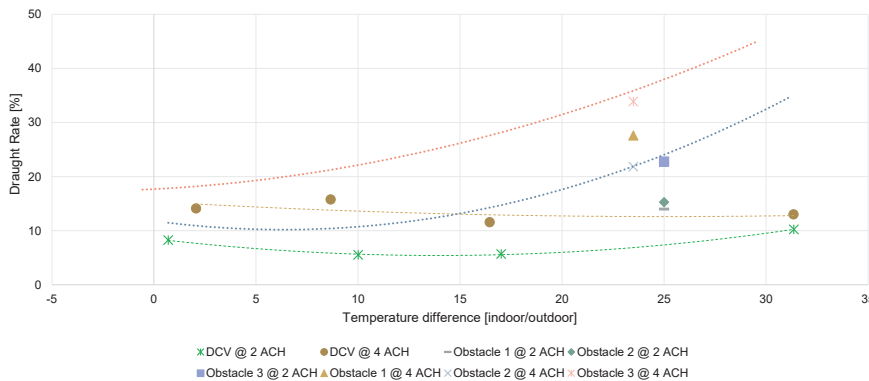


Gunnar – obstacles

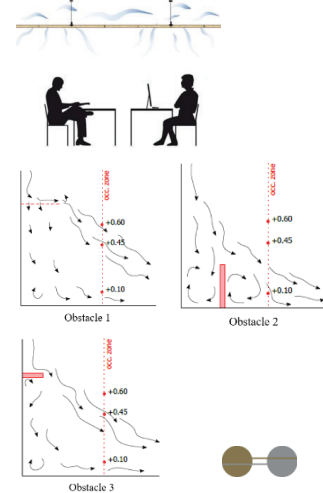


Solutions coping with draught

Solutions (Fangers equation)



Solutions – DCV, obstacle 2, 1, 3



Conclusion & acknowledgement

Main findings

- The commonly used **Fanger's method tends to overestimate** the draught risk associated with natural ventilation, especially in systems using displacement air distribution patterns.
- The results indicate **significant deviations in predicted draught rate** when different methods are utilized.
- **Further investigation** is needed to identify the most suitable method for evaluating draught risk in natural ventilation systems.

Acknowledgement

- This work was supported by the project: Hybridene – Optimal hybrid ventilasjon i fremtidens bygg, financed by The Research Council of Norway (project nr. 327591).



Questions?



Evaluation of sensor-based air cleaners to remove PM_{2.5} and TVOC from indoors with pollutant sources of smoking and burning candles

- Kathrine Andersen, Aarhus University,
- Stig Koust, Danish Technological Institute
- Freja Rasmussen, Danish Technological Institute
- Li Rong, Aarhus University



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1

Motivation

- 90 % of time indoor and air pollution is a large cause of mortality.
- Light candles and smoke cigarettes
 - Increases the emissions of particles.
 - Smoke: PM_{2.5}
 - Candles: Ultrafine
 - Both: TVOCs
- To combat these emissions: employ portable air cleaners
- To simplify the control: embedded IAQ sensor with auto mode

2

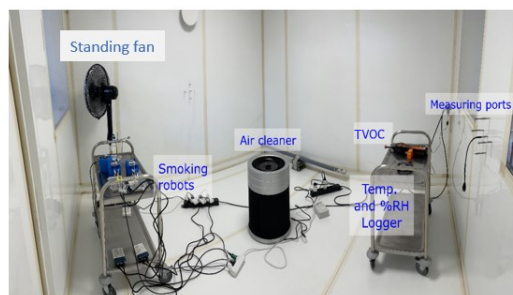
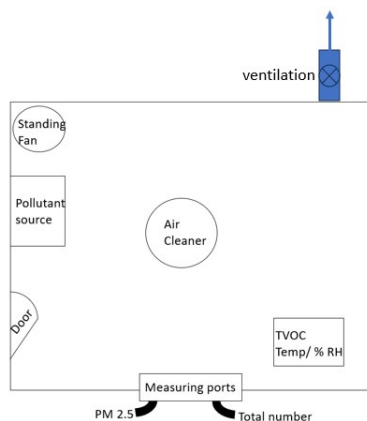
Objectives of this study

- 1) Accuracy of embedded sensors in air cleaners
 - Cigarettes smoke
 - Burning candles
- 2) Regulation of fan speed by auto mode
 - PM2.5
 - TVOCs

3

Methodology

- An airtight test chamber of 20 m³ with Teflon surfaces.
- The set-up



4

Air cleaners and types of experiments

- Register and control from outside of the test chamber
- Two TVOC sensors
- Types of experiments:
 - 2 types of smoke experiments
 - 1 type of candles experiments.
- For more details about each air cleaner and the procedure see the paper.

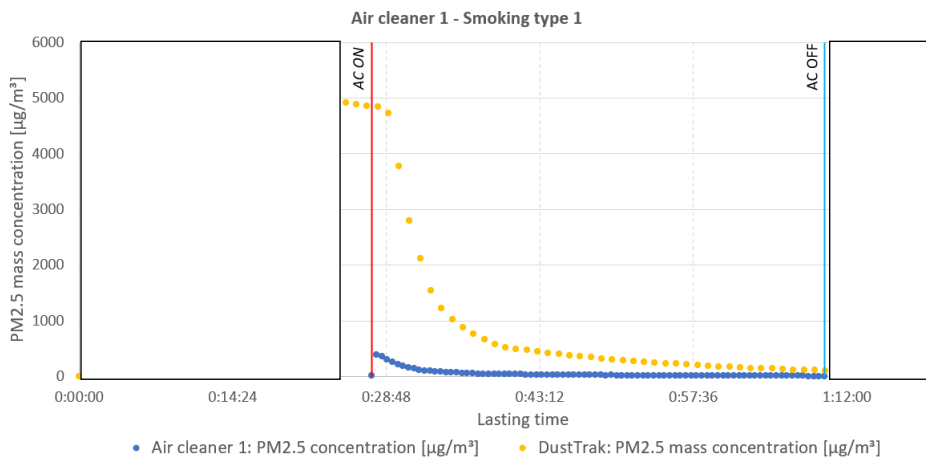


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5

Smoke experiment – Air cleaner 1

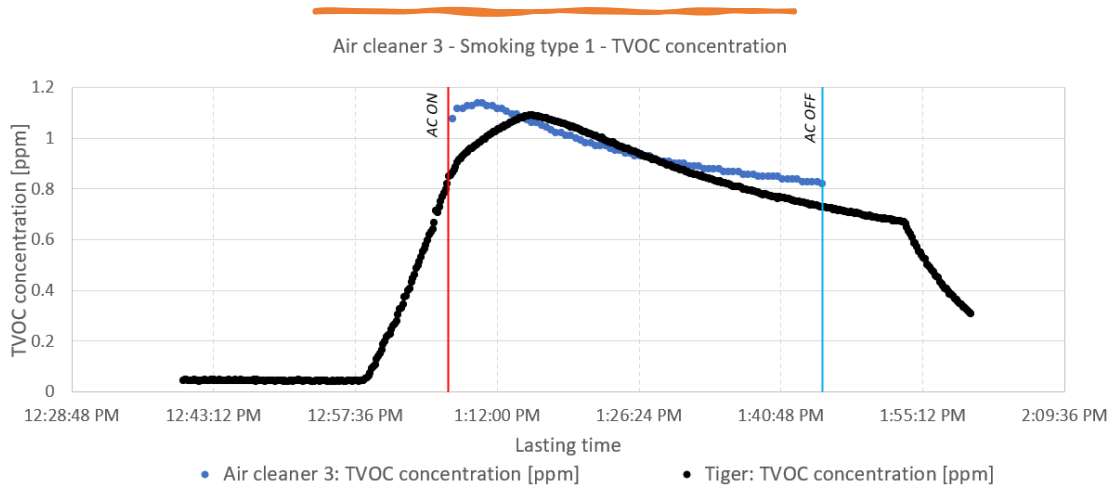


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Smoke experiment – Air cleaner 3

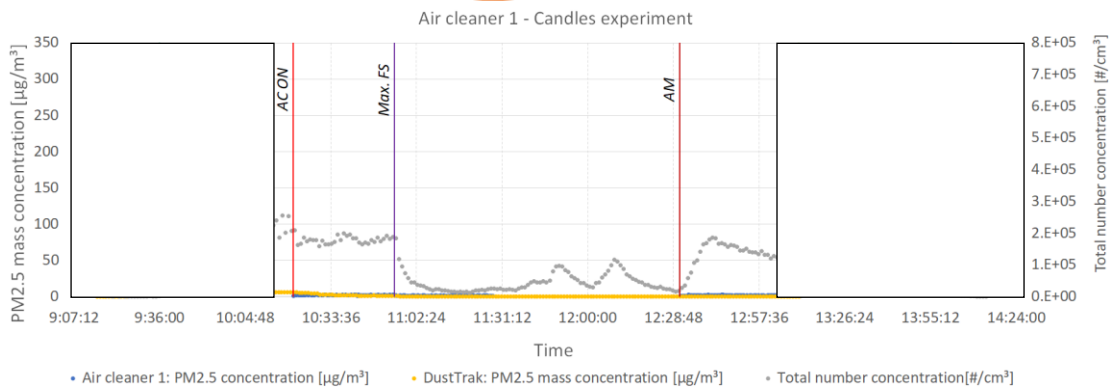


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Candles experiment – Air cleaner 1



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8

Conclusions

- All embedded sensors detected the PM2.5 mass concentration emitted by smoke.
 - Underestimated, to different degrees, compared to the advanced instruments.
 - Affected the adjustment of fan speed
 - Extended exposure time.
- None of the sensors detected the rise of the ultrafine particles from candles
 - No automatically adjustment of fan speed
 - All the air cleaners: able to reduce
 - Reduction rate increased with the fan speed
 - Not done by auto mode
 - Auto mode is not preferred for high concentrations of ultrafine particles
- The TVOC sensor with exacted data
 - Register the TVOC concentration for smoke
 - No influence on the fan speed adjustments

ACKNOWLEDGEMENTS

This study was part of a larger project conducted by Danish Technological Institute founded by RealDania and Grundejernes Investeringsfond:

<https://www.teknologisk.dk/projekter/mobile-luftrensere-hvor-godt-virker-de/44379>



Thank you for your attention!

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Developing methodology for testing of gas-phase air cleaners based on perceived air quality

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Technical University of Denmark, DTU Sustain



43rd AIVC -11th TightVent & 9th venticool Conference
October 4-5, 2023
Aalborg University, Copenhagen, Denmark

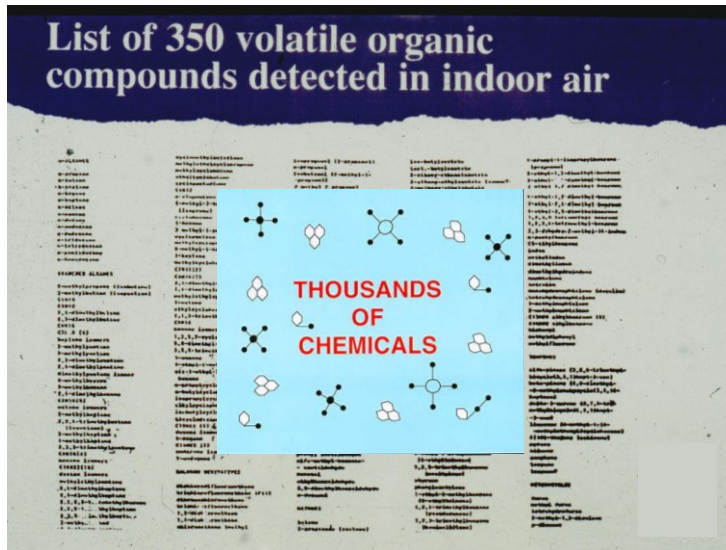


1

Background

2

Organic chemicals in indoor air



3

Methods and standards for testing gas-phase air cleaners

Standard/Protocol	Methods	Challenge Gaseous	Measured Gaseous	Performance index
Air cleaner, Standardization Administration of China (GB/T-18801)	Pulldown	Single species gas e.g.,	Formaldehyde toluene	CADR
Air cleaner, Standardization Administration of China (GB/T-18801)	Singlepass	Single species gas e.g.,	Formaldehyde toluene	Single-pass efficiency
Reduced Energy Use Through Reduced Indoor Contamination in Residential Buildings, NCEMBT (NCEMBT 061101), US report	Pulldown	Eight VOCs mixture	TVOC _{toluene} formaldehyde	CADR
Air cleaner, Japanese Standard Association (JIS C 9615-2007)	Singlepass	NO ₂ , SO ₂	NO ₂ , SO ₂	Single-pass efficiency
Air cleaners of household and similar use, Japan Electrical Manufacturers Association (JEM 1467-1995)	Pulldown	Tobacco smoke	Ammonia, acetaldehyde, and acetic acid	Removal rate
Independent air purification devices for tertiary sector and residential applications - Test methods - Intrinsic performances, Association Française De Normalisation (XP B44-200)	Singlepass	Four VOCs mixture	Acetone, acetaldehyde, heptane, and toluen	Single-pass efficiency , CADR
Test method for assessing the performance of gas-phase air cleaning media and devices for general ventilation ISO 29464:2017	Singlepass	VOCs, acids, bases, and others	VOCs, acids, and bases, and others	Afshari et al. (2022) Single-pass efficiency

Source: Afshari et al. (2022)

4

Byproduct generation Incomplete oxidation

Aldehydes → **formaldehyde**, formic acid, CO

Alcohols → aldehydes → acids → shorter carbon chain alcohols and acids → **formaldehyde**, methanol → CO₂ and H₂O

Benzene → phenol

1-Butanol → butanal (butyraldehyde), butanoic acid, ethanol, acetaldehyde, (propanal (propionaldehyde) and propanol, propanoic acid) → (ethanol, **formaldehyde**) → methanol, **formaldehyde** and formic acid

Ethanol → methanol, acetaldehyde, **formaldehyde**, acetic acid, formic acid

Methanol → methyl formate (measured in liquid form only), **formaldehyde**, methylal (formaldehyde dimethyl acetal)

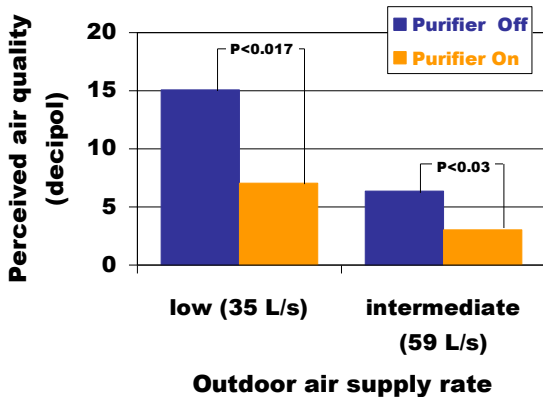
Toluene → benzaldehyde, benzoic acid, cresol, benzyl alcohol, phenol, benzene, formic acid

Source: Mo et al. (2009)

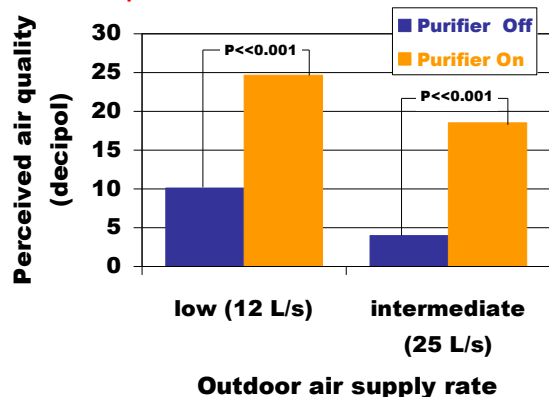
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Detection of by-products using sensory assessments

🟢 improved PAQ (lower decipol) w/build. materials



🔴 reduced PAQ (higher decipol) w/human bioeffluents

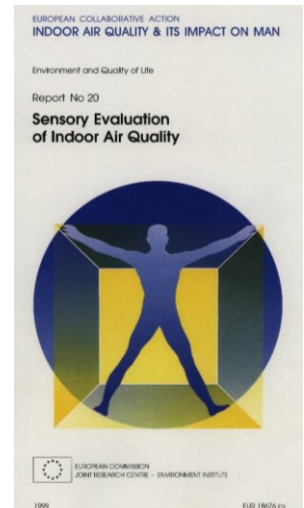


Source: Kolarik and Wargocki (2006)

6

Sensory evaluations of air quality

- Sensory evaluations of air quality have been used routinely in indoor air research for the past 25 years.
- Perceived air quality has been used to define ventilation rates prescribed in the majority of present standards (eg. 16798, ASHRAE 62.1)
- Perceived air quality has been used to examine emissions from building materials, it is included as a part of testing in few labelling schemes for building and furniture materials (Finnish M1 Label; Danish Indoor Climate Label, and German AgBB Scheme) and the standard describing sensory testing in connection with emission testing (ISO 16000-30)
- Perceived air quality has been used extensively in the past in field studies as a measure of air quality in rooms and buildings (eg. Wargocki et al., 2004)

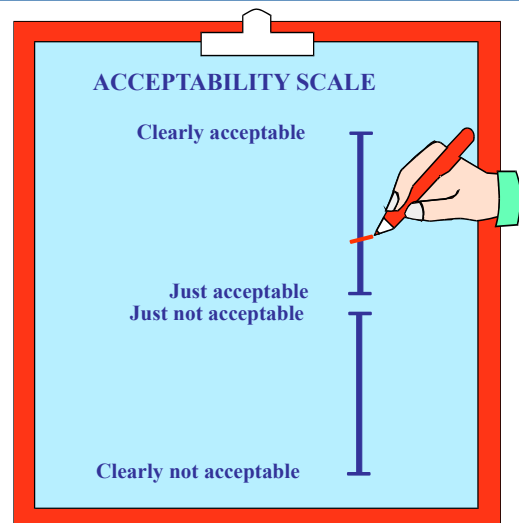


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Continuous acceptability scale

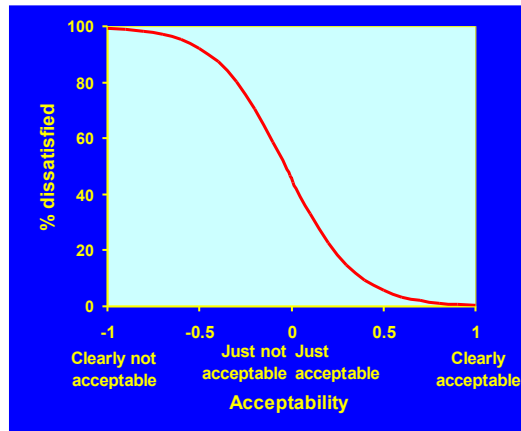
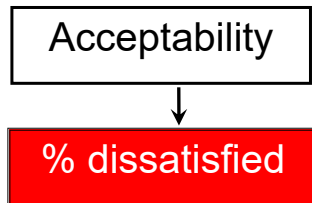
How do you assess the air quality?

Pay attention to the dichotomy between acceptable and not acceptable.



8

Measuring metric



Development of a standard method for assessing the performance of gas-phase air cleaning

Overall objective

- Develop the protocol for rating gas-phase air cleaners
- Examine the protocol by performing the actual measurements
- Use the protocol in the Round-Robin tests
- Implement the protocol in the standard for testing the performance of air cleaners.

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General approach

- Two-stage-testing:
 - Stage 1: Pass/no pass with respect to the effect on indoor air quality
 - Stage 2: Determine clean air delivery rate (CADR) and compare with equivalent ventilation requirements

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Specific objective

Examine the protocol by testing different types of air purifiers

Perform sensory evaluation and chemical measurements

Test two types of pollution sources: building materials (building) and human bioeffluents (humans) and their combination

Determine the effect of air cleaners on air quality in terms of clean air delivery rate

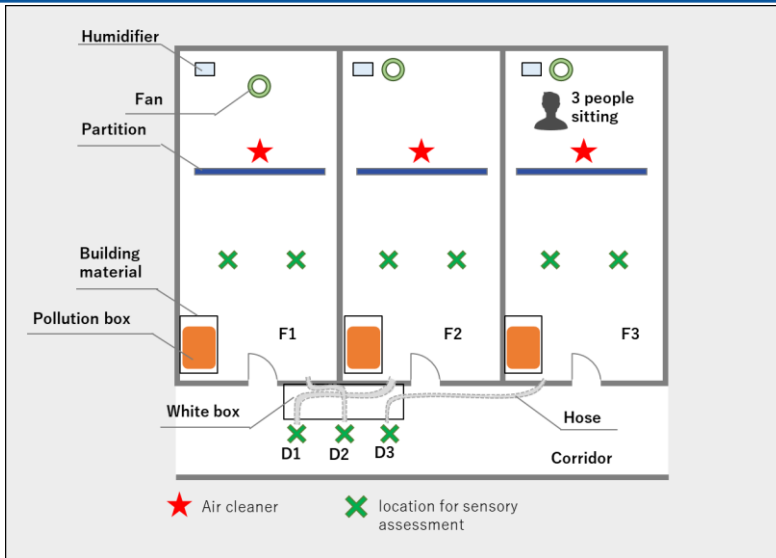
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Overall protocol

- Portable air cleaners were tested; all operated at close to the maximum capacity
- Air cleaners were challenged with different types of pollutants representing people and building materials
- Conditions under test: ca. 23°C (73°F) and 50%RH
- Up to four levels of ventilation with outdoor air were tested
- Different number of air cleaners were placed in the rooms during testing
- Measurements of air quality were performed with air cleaners idled and in operation

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Set-up



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Sensory assessments



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ASHRAE 62.1, ventilation rate procedure

People Component



Building Component

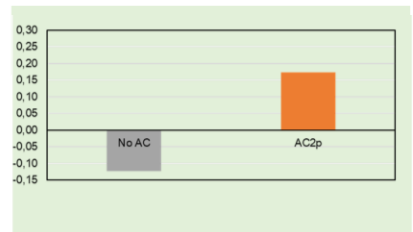
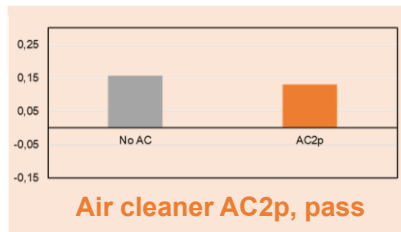
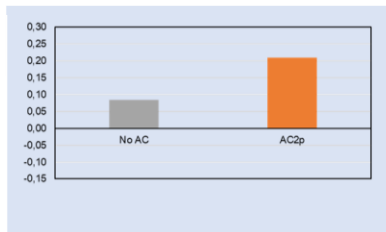
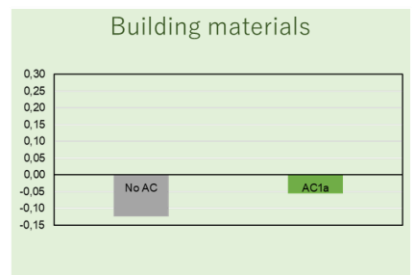
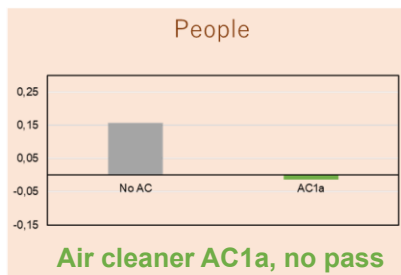
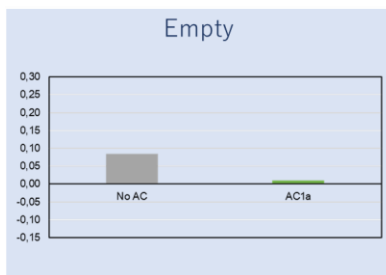


People + Building



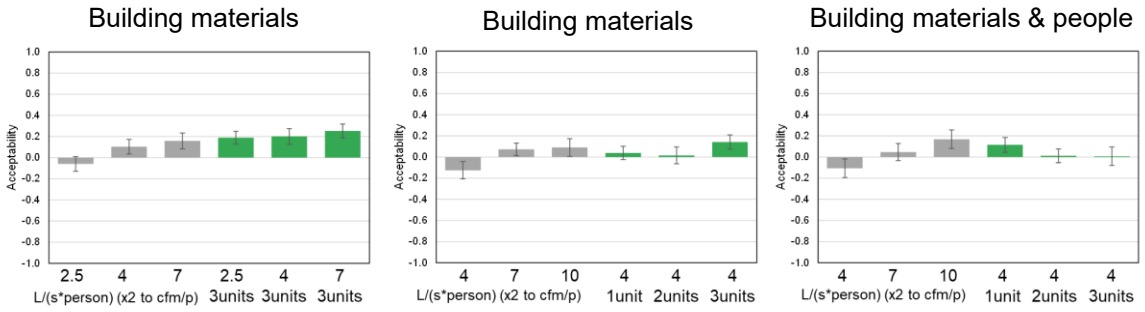
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Stage 1 – pass/no pass, example



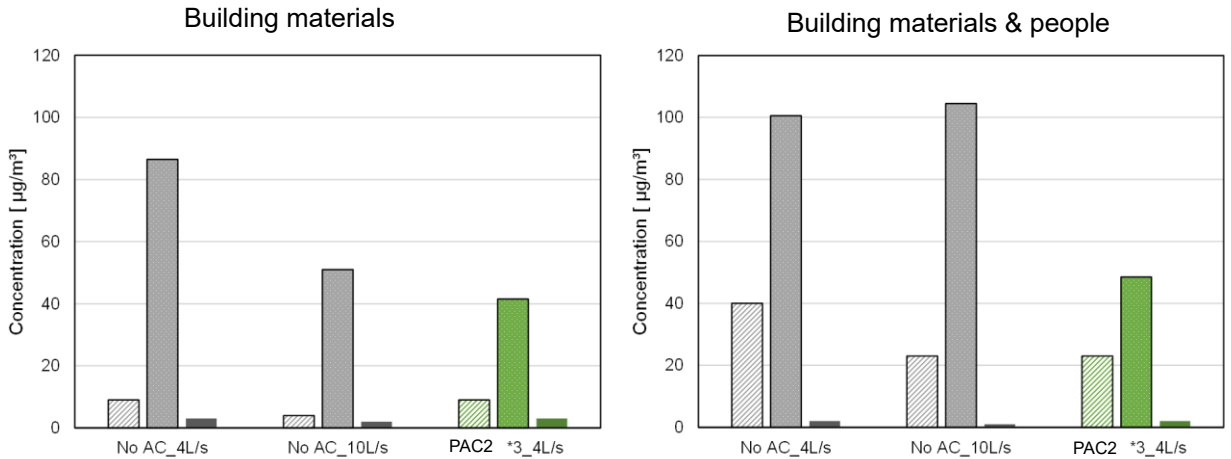
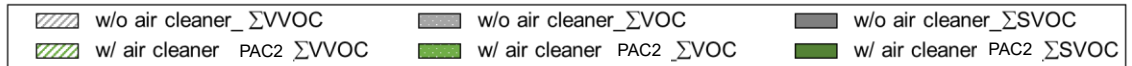
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Stage 2 – testing, sensory assessments, example




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Stage 2 – testing, chemical measurements




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Stage 2 – testing, chemical measurements



 w/o air cleaner

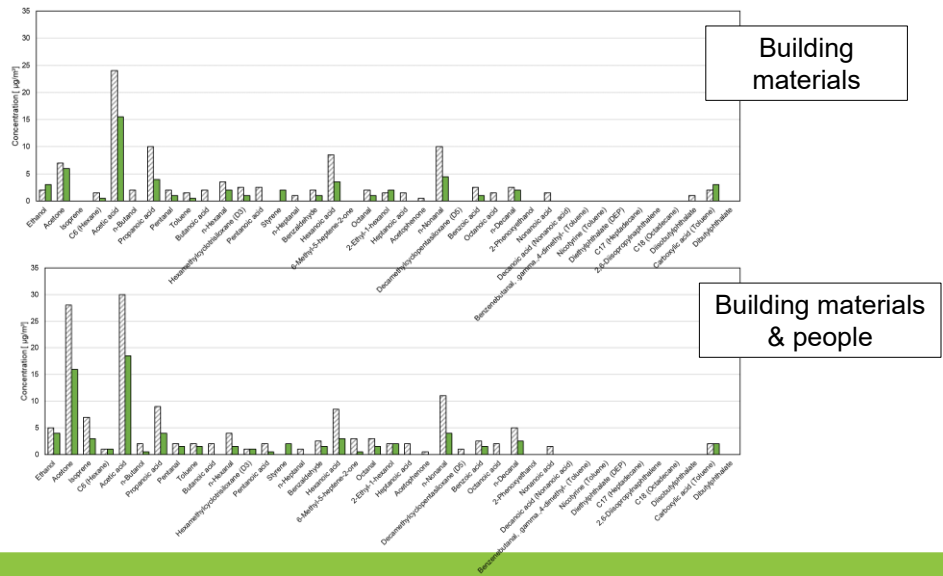
 4 L/s per person



 w/ air cleaner PAC2

 (3 units)

 4 L/s per person



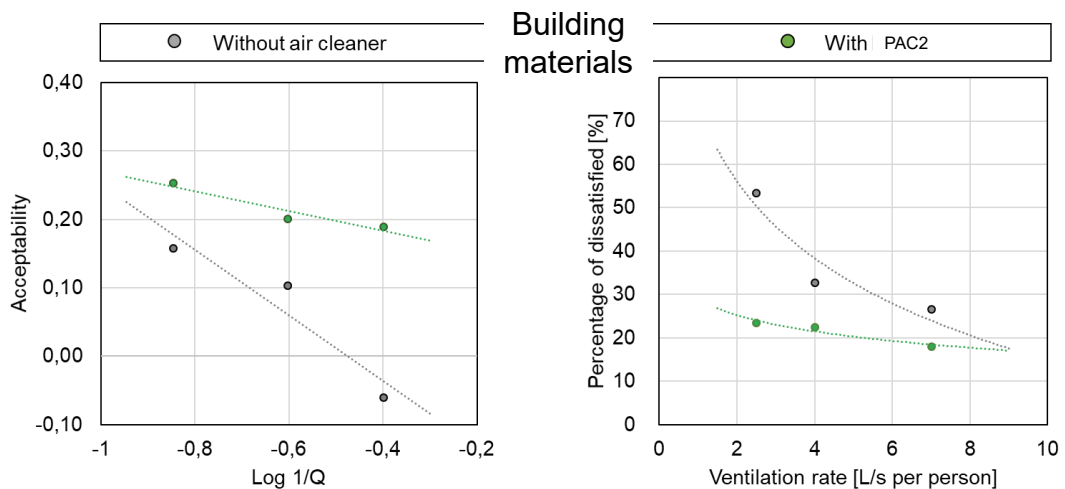
Concluding remarks

Preliminary conclusions

- We successfully examined methodology for testing gas-phase air cleaners
- Passive air cleaners performed better than active air cleaners
- There were some inconsistencies between chemical analyses and sensory assessments
- More analyses are in progress

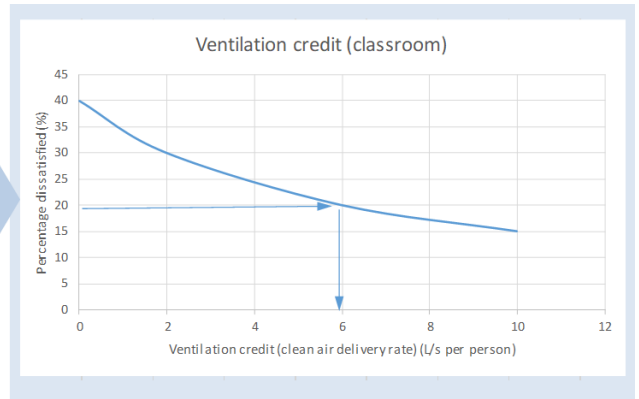
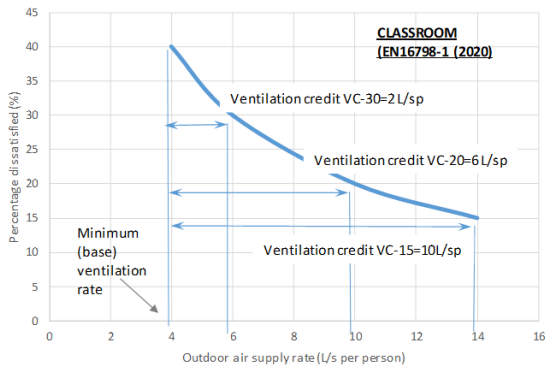
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Determination of clean air delivery rate



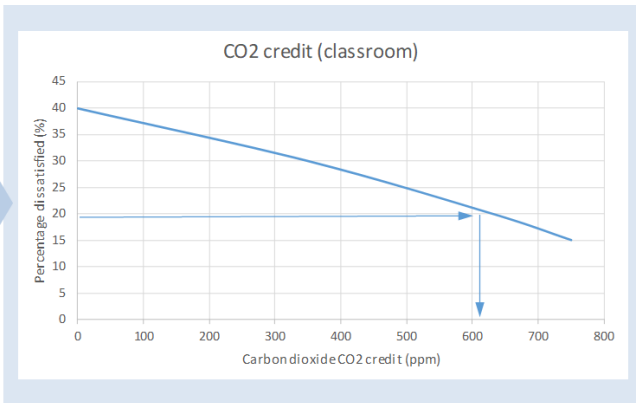
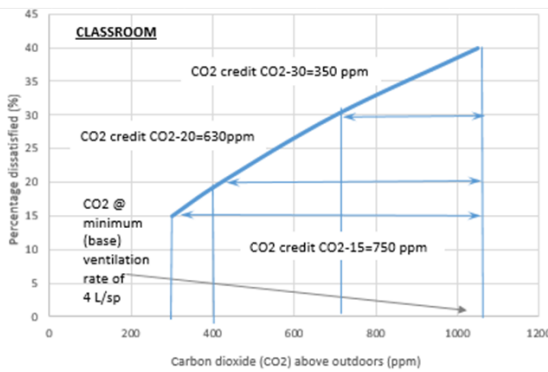
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New concept, ventilation credit



25

New concept, CO₂ credit



26

Questions



pawar@dtu.dk

EVALUATING THE IMPACT OF AIR CLEANING ON BIOAEROSOLS AND OTHER IAQ INDICATORS IN BELGIAN DAYCARE FACILITIES

Sarah L. Paralovo (presenter)
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Arnold Janssens
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Maarten Spruyt
Marianne Stranger

9/10/2023

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1

INTRODUCTION

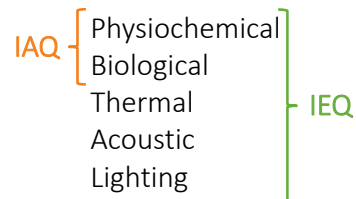
COVID-19 pandemic increased awareness about the spread of viral particles through the air^{1,2}



Ventilation and air cleaning emerged as main strategies to reduce infection risk.

But health and comfort of occupants are influenced by many different parameters:

A study was conducted in Flanders (Belgium) to investigate the impact of ventilation and air cleaning strategies on IEQ in public spaces, to enable an objective evaluation of the effectiveness and impact of such risk reduction methods



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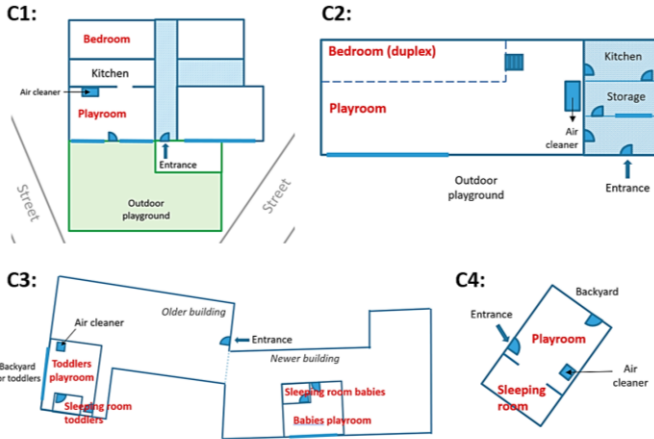
2/9

2

MATERIAL AND METHODS

Focus: One category of public space (daycare facilities) and selected IAQ parameters

Facilities assessed in spring (C1/C2) and summer (C3/C4)



Continuous monitoring of:

- T/RH
- CO₂ (by stationary sensors)
- PM_x

Scheduled* measurements of:

- ACHs (via artificially injected CO₂ decay test³)
- Air tightness (via blower door test)
- Pathogenic bioaerosols (via collection by cyclonic impinger⁴)

*under different and pre-determined ventilation and air cleaning scenarios.

RESULTS AND DISCUSSION

Ventilation characterization:

- C1 and C4 had no mech. vent. system;
- C3 had two separate mechanical balanced vent. systems (newer and higher flow rates in K2/K4);
- C2 had a mech. vent. system, but measurements showed it was not operational;
- None of the rooms reached the Belgian guideline for COVID-19 protection of **40 m³ h⁻¹ per person** (considering each room's occupancy) under the most common scenarios;
- In C2, C4 and the newer part of C3, the guideline is reached when **cross-ventilation** is provided.

	Air tightness (window tilted in room next to the assessed room)	'Normal' scenario		'Summer' scenario					
		ACH (h ⁻¹ 50Pa)	Vol. flow (m ³ h ⁻¹)	ACH (h ⁻¹)	Vol. flow (m ³ h ⁻¹)				
C1	n50	ACH	Vol. flow	ACH	Vol. flow				
		(h ⁻¹ 50Pa)	(m ³ h ⁻¹)	(h ⁻¹)	(m ³ h ⁻¹)				
K1	8.7	1.10	75.6	5.40	372				
	Air tightness	All doors and windows closed		'Winter'		'Enhanced winter'		'Summer'	
		ACH	Vol. flow	ACH	Vol. flow	ACH	Vol. flow	ACH	Vol. flow
C2	n50	ACH	Vol. flow	ACH	Vol. flow	ACH	Vol. flow	ACH	Vol. flow
		(h ⁻¹ 50Pa)	(m ³ h ⁻¹)	(h ⁻¹)	(m ³ h ⁻¹)	(h ⁻¹)	(m ³ h ⁻¹)	(h ⁻¹)	(m ³ h ⁻¹)
K1/K2	3.1	0.18	38.0	0.82	174	3.61	768	13.2	2807
	Air tightness	All doors and windows closed		Door to corridor open		Sliding window open		Door + sliding window open	
		ACH	Vol. flow	ACH	Vol. flow	ACH	Vol. flow	ACH	Vol. flow
C3	n50	ACH	Vol. flow	ACH	Vol. flow	ACH	Vol. flow	ACH	Vol. flow
		(h ⁻¹ 50Pa)	(m ³ h ⁻¹)	(h ⁻¹)	(m ³ h ⁻¹)	(h ⁻¹)	(m ³ h ⁻¹)	(h ⁻¹)	(m ³ h ⁻¹)
K1	-	0.67	167	1.20	299	2.01	501	1.80	449
K2	-	1.82	363	2.93	585	2.30	458	4.44	886
K3	-	1.23	50.0	2.57	104	-	-	-	-
K4	-	5.34	138	12.0	310	-	-	-	-
	Air tightness	All doors and windows closed		Door to backyard open		Window to the street side open		Door to backyard + window to street side open	
		ACH	Vol. flow	ACH	Vol. flow	ACH	Vol. flow	ACH	Vol. flow
C4	n50	ACH	Vol. flow <td>ACH</td> <td>Vol. flow <td>ACH</td> <td>Vol. flow <td>ACH</td> <td>Vol. flow </td></td></td>	ACH	Vol. flow <td>ACH</td> <td>Vol. flow <td>ACH</td> <td>Vol. flow </td></td>	ACH	Vol. flow <td>ACH</td> <td>Vol. flow </td>	ACH	Vol. flow
		(h ⁻¹ 50Pa)	(m ³ h ⁻¹)	(h ⁻¹)	(m ³ h ⁻¹)	(h ⁻¹)	(m ³ h ⁻¹)	(h ⁻¹)	(m ³ h ⁻¹)
K1	-	<1	<140	3.33	465	1.11	155	12.0	1669

RESULTS AND DISCUSSION

Temperature and RH:

- T and RH are important comfort parameters, but RH also impacts airborne pathogen transmission (influences evaporation kinematics and particle growth, lower RH increases sensitivity of airways^{5,6});
- Temperatures in C2, C3 and C4 were most of the occupied time in accordance with the Flemish Indoor Environment Decree (20-24°C in cold season; 22-26°C in warm season);
- In C1 the temperatures were < 20°C for over a quarter of the occupied time in both rooms;
- RH in C3 and C4 was most of the occupied time in accordance with the Flemish Indoor Environment Decree (30-70% in warm season);
- In C1's playroom the median RH was below the recommended level (40-60% in cold season), and in C2 the median RH was < 25% in all assessed rooms.

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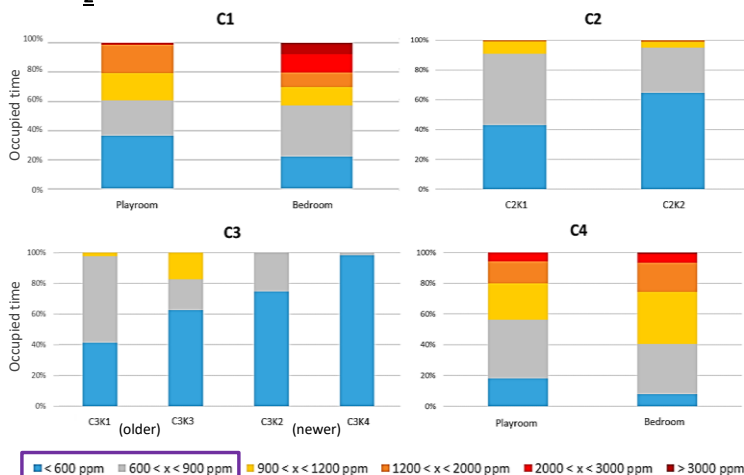
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5

RESULTS AND DISCUSSION

CO₂ concentrations:



- Belgian guideline for COVID-19 protection sets a maximum of 900ppm;
- C1 had the lowest occupancy, but the highest CO₂ concentrations (peaks in the bedroom consistently >3500 ppm);
- In C4 the CO₂ concentrations were the most frequently > 900ppm, with greater exceedances in the second (colder) week;
- Although C4 allowed high ACHs by cross ventilation, this is not often applied;
- C2 consistently had peaks >900 ppm, but overall situation was good despite faulty mech. vent.;
- C3 had the lowest CO₂ concentrations, but a clear difference was noticed between the two parts of the facility.

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6

RESULTS AND DISCUSSION

PM_x concentrations:

Important air cleaning evaluation parameter.

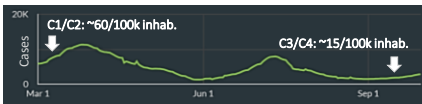
	C1		C2		C3		C4	
	Air cleaner status OFF	ON	Air cleaner status OFF	ON	Air cleaner status OFF	ON	Air cleaner status OFF	ON
	I/O ratios: Median (P-75)							
I/O TSP	0,6 (1,2)	1,0 (2,0)	0,3 (0,6)	0,4 (1,0)	2,8 (5,5)	2,7 (5,8)	2,9 (5,1)	2,6 (4,8)
I/O PM ₁₀	0,8 (1,1)	1,2 (1,9)	0,9 (1,6)	1,3 (1,8)	1,5 (2,3)	1,4 (1,9)	1,5 (2,0)	1,1 (1,5)
I/O PM _{2.5}	0,8 (1,0)	0,9 (1,1)	0,8 (1,1)	0,9 (1,0)	0,6 (0,8)	0,6 (0,8)	1,5 (1,8)	0,9 (1,1)
I/O PM ₁	0,7 (0,9)	0,6 (0,8)	0,7 (0,8)	0,8 (0,9)	0,5 (0,6)	0,5 (0,6)	1,9 (2,4)	1,1 (1,5)

- I/O ratios >1 indicate either indoor sources of PM or an accumulation of outdoor PM indoors;
- Only in C4 a consistent decrease of I/O ratios was observed when the air cleaner was switched on (up to 40% reduction for PM_{2.5} and PM₁);
- I/O ratios were either the same or slightly higher with the air cleaner switched on in C1 (except for PM₁), C2 and C3 (except for PM₁₀);
- Potential reasons:
 - Inadequate configuration of air cleaner
 - Different activities in the morning and in the afternoon that generate different levels of resuspension
 - Different ventilation/airing during morning and afternoon.

RESULTS AND DISCUSSION

Pathogens in air:

- CT-value > 35.0 considered as limit value for SARS-CoV-2 (very low viral load, higher uncertainty);
- Official COVID-19 incidence in Belgium:



		Week 1				Week 2			
		Day 1	Day 2	Day 3	Day 4	Day 1	Day 2	Day 3	Day 4
C1	K1 AC ON	LV* (CT 36,6) ¹	Negative	LV* (CT 37,0)	Positive (vI* < 1000, CT 33,9) ^{1,3,4,5}				
	AC OFF	-	Negative	LV* (CT 34,7)	Negative				
	K2 -	Negative ^{1,2}	-	LV* (CT 35,3) ¹	-				
C2	K1 AC ON	Negative	Negative	LV* (CT 36,2)	LV* (CT 39,8)				
	AC OFF	-	Negative	Negative ¹	Negative				
	-	Negative	Negative	Negative	LV* (CT 36,8)				
C3	K1 AC ON	Negative ^{6,7}	LV* (CT 39,9) ^{1,2}	Negative ¹	LV* (CT 40,9) ^{1,3}				
	AC OFF	Negative	Negative ^{1,7}	LV* (CT 38,4) ¹	Negative ⁷				
	K2 -	Negative ¹	Negative	Negative	Negative				
C4	K1 AC ON	Positive (vI* < 1000, CT 30,7) ²	Positive (vI* < 1000, CT 32,2) ^{3,4}	Positive (vI* < 1000, CT 29,8) ^{1,2}	Positive (vI* < 1000, CT 29,6) ^{1,3,4,5}				
	AC OFF	Positive (vI* > 1000, CT 27,0) ²	Positive (vI* > 1000, CT 29,0) ^{1,2,7}	Positive (vI* < 1000, CT 30,3) ^{1,2}	Positive (vI* > 1000, CT 28,1) ^{1,3,10,11}				
	K2 -	-	Positive (vI* > 1000, CT 27,6) ^{1,3,11}	-	-				

*LV = limit value
vI = viral load

1. Streptococcus pneumoniae DNA positive 4. Enterovirus RNA at LV 7. Bordetella parapertussis DNA positive 10. Bocavirus DNA positive
2. Corona virus OC43 RNA at LV 5. Influenza A virus RNA at LV 8. Bordetella parapertussis DNA LV 11. Adenovirus DNA LV
3. Rhinovirus positive 6. Rhinovirus LV 9. Bocavirus DNA LV

- At least one pathogen detected in **67%** of C1, **31%** of C2, **73%** of C3 and **100%** of C4 samples.
- Ventilation seems to be more effective than air cleaning in C1, C2 and C3;
- Air cleaning seems to slightly decrease the viral load of SARS-CoV-2 and the presence of other pathogens in C4.

CONCLUSIONS

- Comprehensive IAQ assessment in four daycare facilities, with physiochemical (T, RH, ACH, CO₂ and PM_x) and biological measurements (collection of air samples for detection of over 20 respiratory pathogens);
- Most of the time the airflow rates per person were below the recommended in Belgium regarding COVID-19, but could be improved by airing;
- Higher CO₂ concentrations in the facilities without mechanical ventilation, and the lowest in the facility with effective mechanical ventilation system;
- Pathogenic bioaerosols were detected more frequently and in larger quantities in the less ventilated facility;
- Where indoor PM_x concentrations correlated well with air cleaning, the same effect was also noticeable in the pathogens presence;
- Expected positive impact of ventilation over IAQ is corroborated, but impact of air cleaning was not as consistent in all facilities.

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9

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- ²Randall, K. ; Ewing, E.T. ; Marr, L.C. ; Jimenez, J.L. ; Bourouiba, L. (2021). How did we get here: what are droplets and aerosols and how far do they go? A historical perspective on the transmission of respiratory infectious diseases. *Interface Focus*, 11: 20210049. DOI: 10.1098/rsfs.2021.0049
- ³Paralovo, S. L ; De Jonge, K. ; Laverge, J.; Janssens, A. (2021). Ventilation assessment in three teaching spaces at a Belgian university. *Proceedings of Healthy Buildings 2021 America*. Presented at the Healthy Buildings 2021 America, Hawaii, USA (Virtual).
- ⁴Bertin. 2012. Coriolis® μ user manual. Manual code: 05027-006-DU002-F ENG. Revised: November 2021. Bertin Technologies
- ⁵Ahlawat, A., Wiedensohler, A. and Mishra, S.K. (2020). An Overview on the Role of Relative Humidity in Airborne Transmission of SARS-CoV-2 in Indoor Environments. *Aerosol Air Qual. Res.* 20: 1856–1861. <https://doi.org/10.4209/aaqr.2020.06.0302>
- ⁶Keetels, G. H., Godderis, L. and van de Wiel, B. J. H. (2022). Associative evidence for the potential of humidification as a non-pharmaceutical intervention for influenza and SARS-CoV-2 transmission. *Journal of Exposure Science & Environmental Epidemiology*. 32: 720 – 726. <https://doi.org/10.1038/s41370-022-00472-3>

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Thank you!

Contact: sarah.limaparalovo@vito.be

REMOVAL OF ODORANTS IN NURSING HOMES USING AIR CLEANERS

By Freja Rydahl Rasmussen (frer@dti.dk) & **Stig Koust**, Ph.D. Nanoscience (stko@dti.dk)
Danish Technological Institute
Clean Air Technology

AIVC October 5, 2023

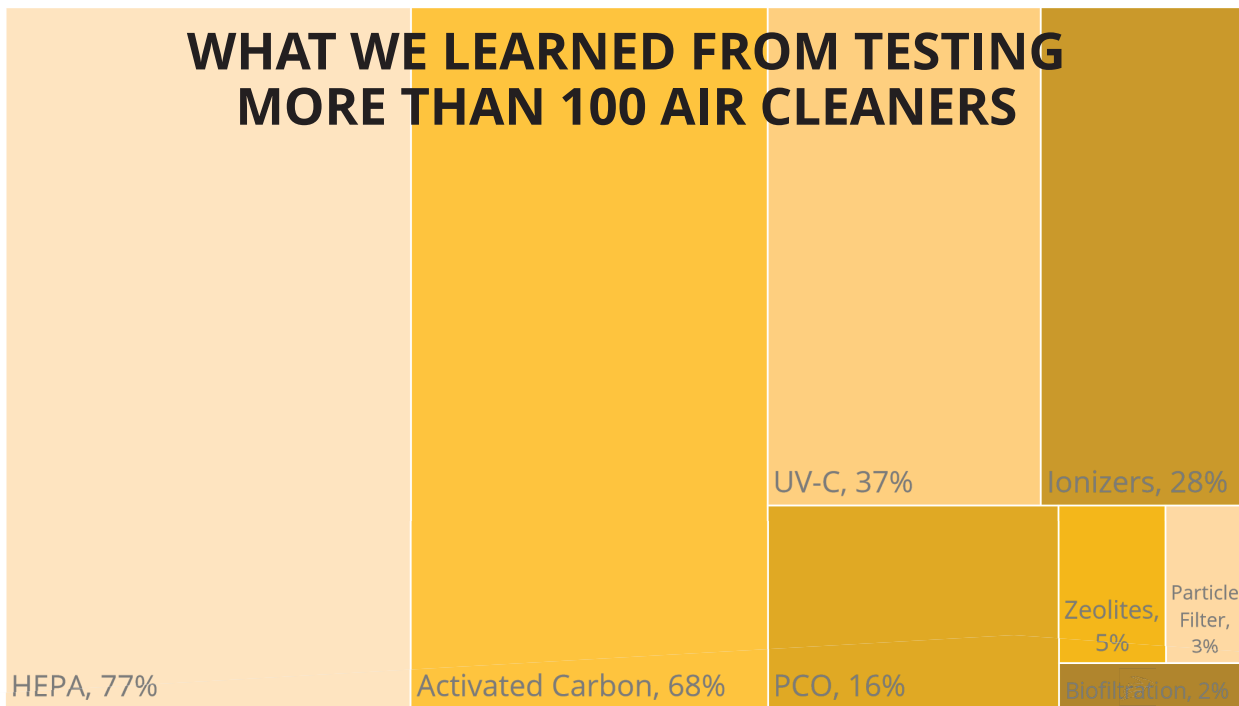


AARHUS
KOMMUNE



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WHAT WE LEARNED FROM TESTING MORE THAN 100 AIR CLEANERS



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HOW MUCH CLEAN AIR WILL 1 DKK BUY YOU?

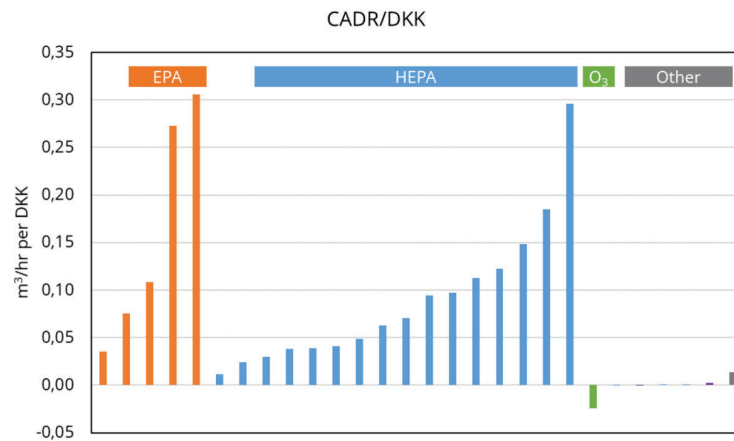
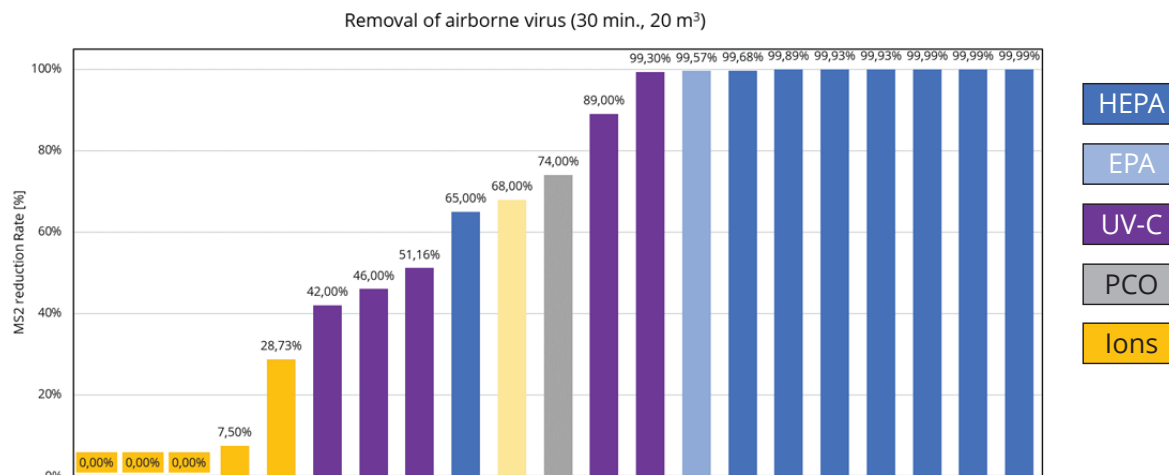


Figure 6: Overview over the cost-effectiveness of all products, calculated as amount of clean air per 1 DKK (CADR [PM2.5] / purchase cost)



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AIR CLEANERS FOR PANDEMIC PREPAREDNESS?

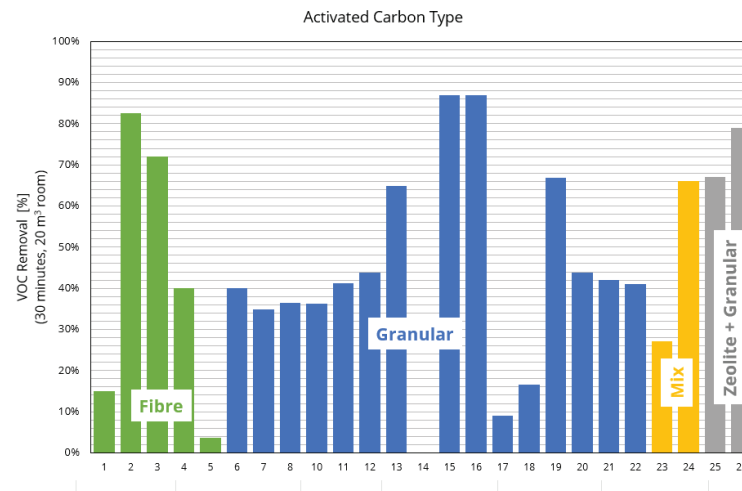


According to ISO 16000-36:2018 - Indoor air — Part 36



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WILL ACTIVATED CARBON GUARANTEE GAS REMOVAL?



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THE ISSUE WITH ODORANTS



- Nursing homes have problems with foul scents from laundry and linen rooms and this smell can often spread to the surrounding common areas
 - This leads to a poor working environment and is a great inconvenience for caregivers and residents
-
- How do we quantify smell and can air cleaners alleviate the problem?



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RESULTS FROM OUR APPROACH

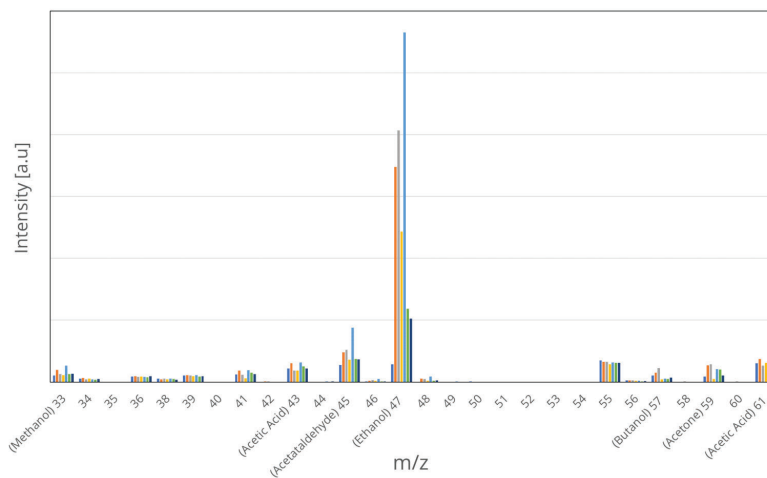


1. Testing selected air cleaners in the laboratory
2. Effect on work environment after implementation at 4 selected nursing homes
3. Lifetime of selected air cleaners



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WHAT TO TEST FOR IN THE LAB?



- ✓ VOC screening in nursing homes
- ✓ Air samples analyzed using Proton-Transfer-Reaction Mass Spectrometry (PTR-MS)
- ✓ Acetaldehyde as proxy for odorants



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PURIFICATION ABILITY IN THE LABORATORY

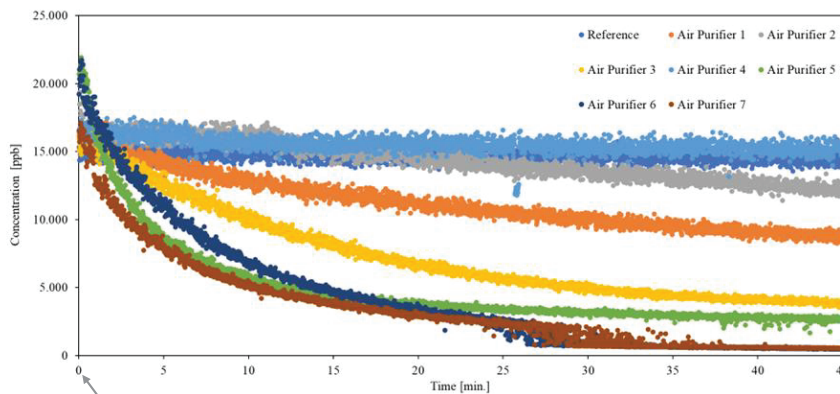


- ✓ Acetaldehyde as proxy for odorants
- ✓ 20 m³ specially designed test chamber
- ✓ Proton-Transfer-Reaction Mass Spectrometry (PTR-MS)
- ✓ Pulldown method – Performance Index %-reduction
- ✓ Byproduct formation
- ✓ Degassing from filter



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ACETALDEHYDE REMOVAL



	Technology				Removal [%]
	Activated Carbon	PCO	Ionizer	HEPA	
1	X		X	X	44%
2	X		X	X	31%
3	X			X	73%
4		X			8,0%
5	X			X	86%
6	X	X	X	X	97%
7	X (+ zeolites)			X	96%

Air Purifier turned on

Link to project report (Danish):
<https://www.teknologisk.dk/ydelsler/udvikling-og-test-af-luftrensere-og-filtersteknologier/39676>



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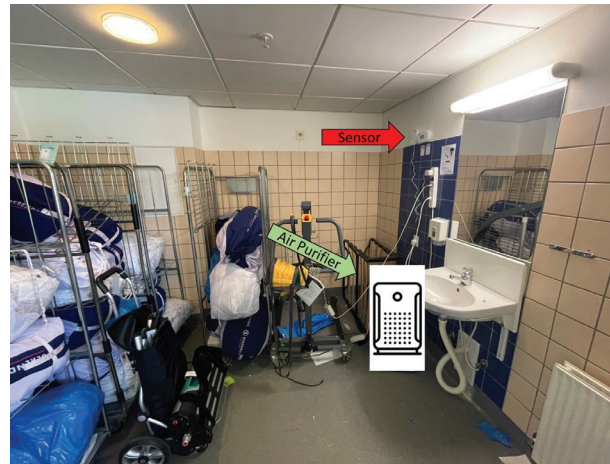
RESULTS FROM OUR APPROACH



1. Testing selected air cleaners in the laboratory
2. Effect on work environment after implementation at 4 selected nursing homes
3. Lifetime of selected air cleaners

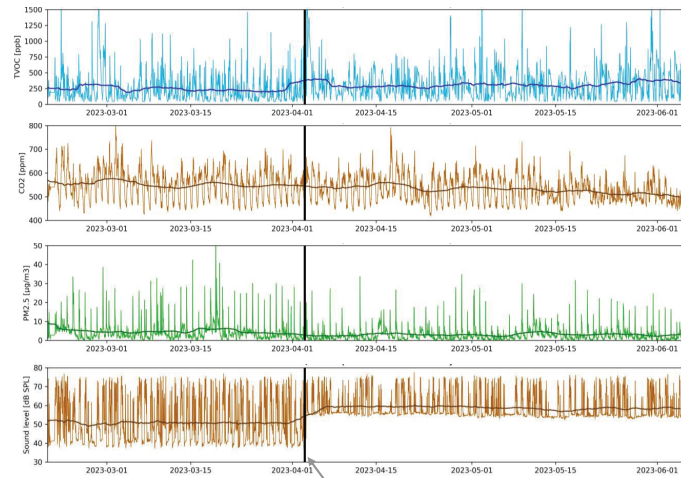
REMOVAL OF ODORANTS IN NURSING HOMES

- Four different locations
 - Low-cost sensors (TVOC as proxy for smell)
 - Air Samples analyzed using PTR-MS



REMOVAL OF ODORANTS IN NURSING HOMES

- Four different locations
- **Low-cost sensors (TVOC as proxy for smell)**
- Air Samples analyzed using PTR-MS



Air Cleaner installed

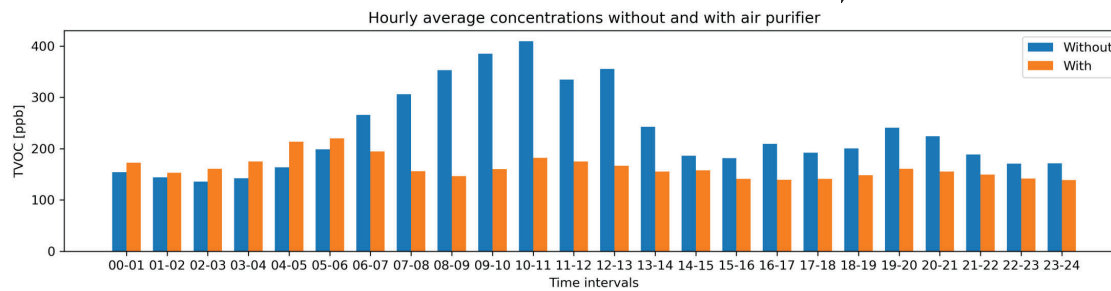


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REMOVAL OF ODORANTS IN NURSING HOMES

- Four different locations
- **Low-cost sensors (TVOC as proxy for smell)**
- Air Samples analyzed using PTR-MS

Peak levels reduced
Background unchanged

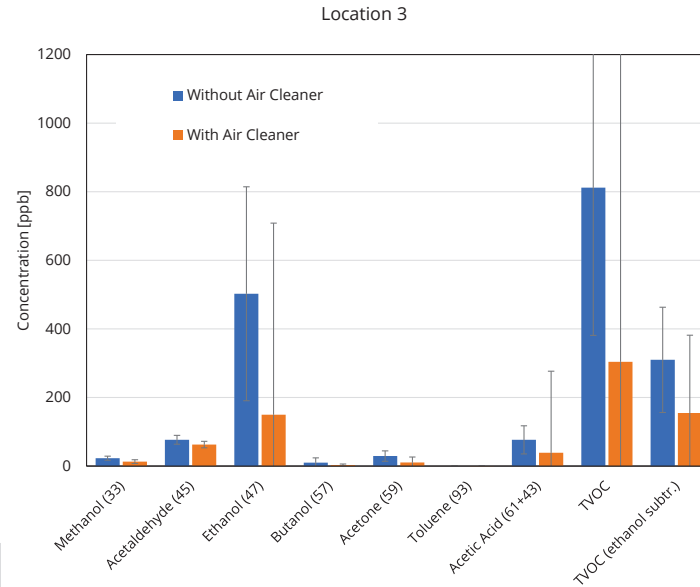


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REMOVAL OF ODORANTS IN NURSING HOMES

- Four different locations
- Low-cost sensors (TVOC as proxy for smell)
- Air Samples analyzed using PTR-MS

- ! Air Quality snapshot on 5 different days
- ! Large variations
- ! Ethanol from hand sanitizers
- ✓ TVOC reduction 17 - 57 %



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CONCLUSIONS



- ✓ VOC removal using activated carbon is a multi-complex issue
- ✓ Continuous and real-time work environment measurements
- ✓ Evaluation of possible technical solutions to remove odorants
- ✓ Demonstrated improvement of air quality in laundry rooms



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**TECHNOLOGY FOR A
BETTER FUTURE**

THANK YOU

stko@dti.dk



RESULTS FOR THREE DIFFERENT STEPS



1. Testing selected air purifiers in the laboratory
2. Effect on work environment after implementation at 4 selected nursing homes
3. **Lifetime of selected air purifiers**



LIFETIME OF SELECTED AIR PURIFIERS

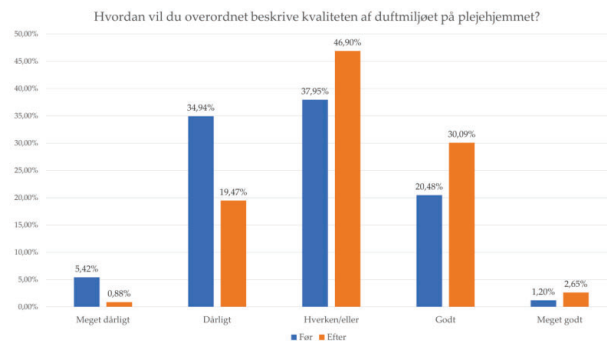
- After operation in nursing homes for 2 months
- Similar test as before implementation

Air Purifier	Reduction in cleaning performance, acetaldehyde (%)	Reduction in CADR (particles) (%)
3	12	57
5	23	44
6	13	3
7	24	8



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"PERCEIVED" AIR QUALITY



Hjemlige duft- og lydmiljøer - Et interventionsstudie på aarhusianske plejehjem
Sofie König Wilms, Sissel Raahede Lundgård, Mads Duevang Dahl, Anne-Sofie Udsen, Amalie Rævsbæk
Birch, Susanne Højlund*, Johanne Korsdal Sørensen* og Marie Koldkjær Højlund*, MANTRA, Aarhus
University



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What can CO₂ measurements tell us about ventilation and infection risk in classrooms?

Carolanne Vouriot

43rd AIVC - 11th TightVent & 9th Venticool Conference



05/10/2023

1

Air quality in classrooms matters



11 million pupils and staff in the UK
(15% of the population)



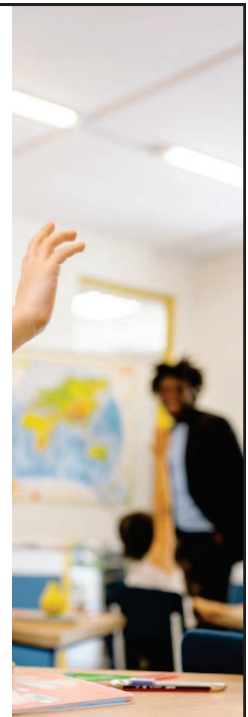
Spending 20% of their time inside classrooms



Direct impact on health and academic performance



Previous campaigns have shown insufficient ventilation provision



2

CO₂ monitoring in classrooms

Part of the pandemic response and becoming widespread in classrooms.

Potential for long-term and large-scale monitoring.

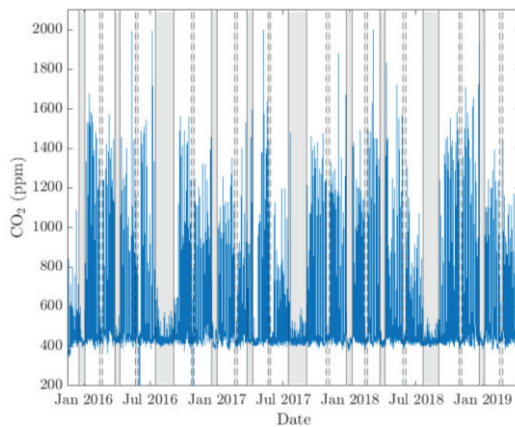
What can we learn from those CO₂ measurements?



3

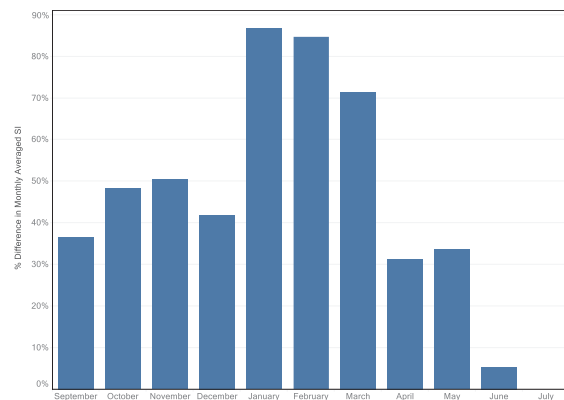
Predicting airborne infection risk

Measurements in 45 classrooms from 11 schools in England (November 2015 – March 2020) ⁽¹⁾



⁽¹⁾ Data provided by Monodraught, with the assistance of Nick Hopper and Nyssa Hayes, and by the K2n platform, with the assistance of Professor Ian Knight.

Rebreathed fraction calculated from CO₂ Number of secondary infections (SI) from one infector



Vouriot, C.V.M., Burrige, H.C., Noakes, C.J. and Linden, P.F., 2021. Seasonal variation in airborne infection risk in schools due to changes in ventilation inferred from monitored carbon dioxide. *Indoor Air*. <https://doi.org/10.1111/ina.12818>

4

CO₂ as a proxy for airborne infection risk

Assumptions:

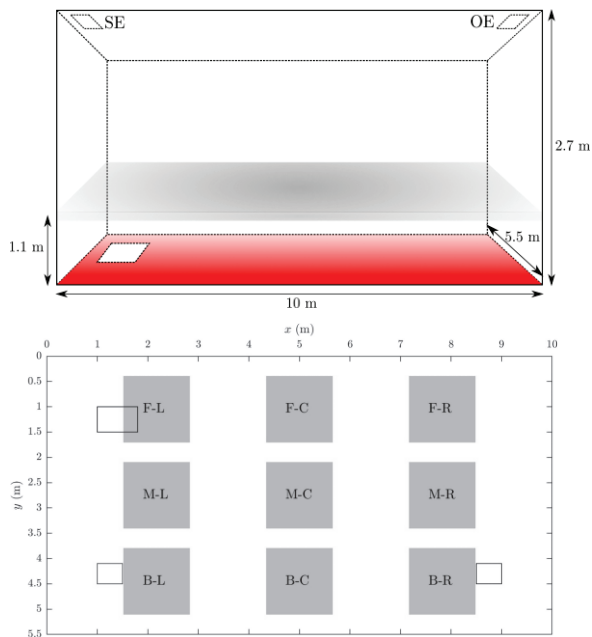
1. Known number of occupants
2. Quanta generation rate
3. Single point measurement

90% of classrooms in the UK are naturally ventilated.

CFD simulations : what is the impact of the well-mixed assumption?



5

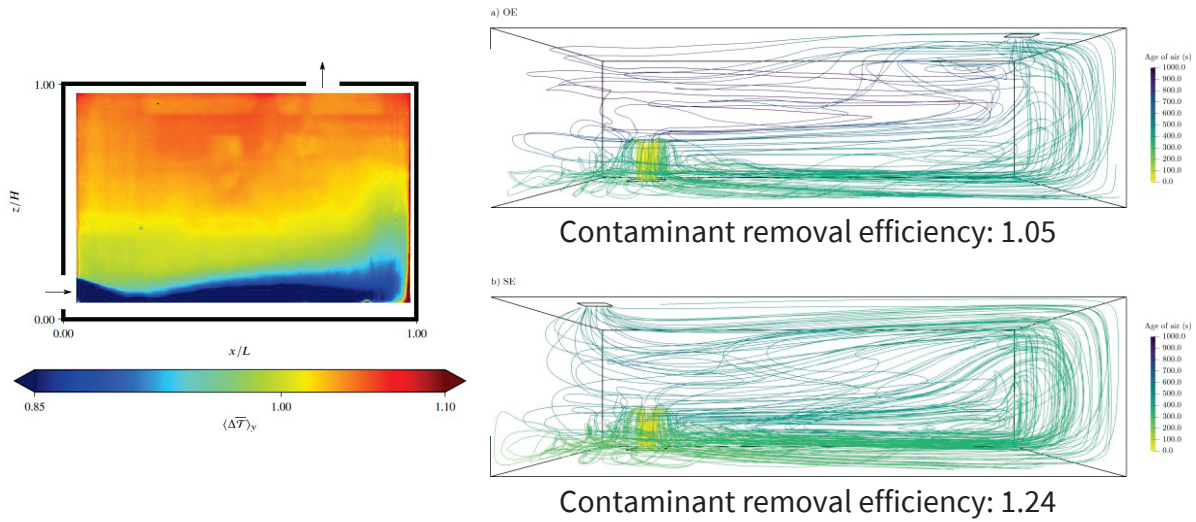


Numerical set-up

- Idealised primary classroom in wintertime.
- Buoyancy driven ventilation with a distributed heat source.
- High and low-level vents in two configurations.
- Passive scalars introduced at breathing height.
- OpenFOAM RANS simulations using buoyantPimpleFoam.

6

Vent position impact on ventilation efficiency



Vourirot, C.V., Higton, T.D., Linden, P.F., Hughes, G.O., van Reeuwijk, M. and Burridge, H.C., 2023. Uniformly distributed floor sources of buoyancy can give rise to significant spatial inhomogeneities within rooms. *Flow*, 3, p.E18. <https://doi.org/10.1017/flo.2023.11>

7

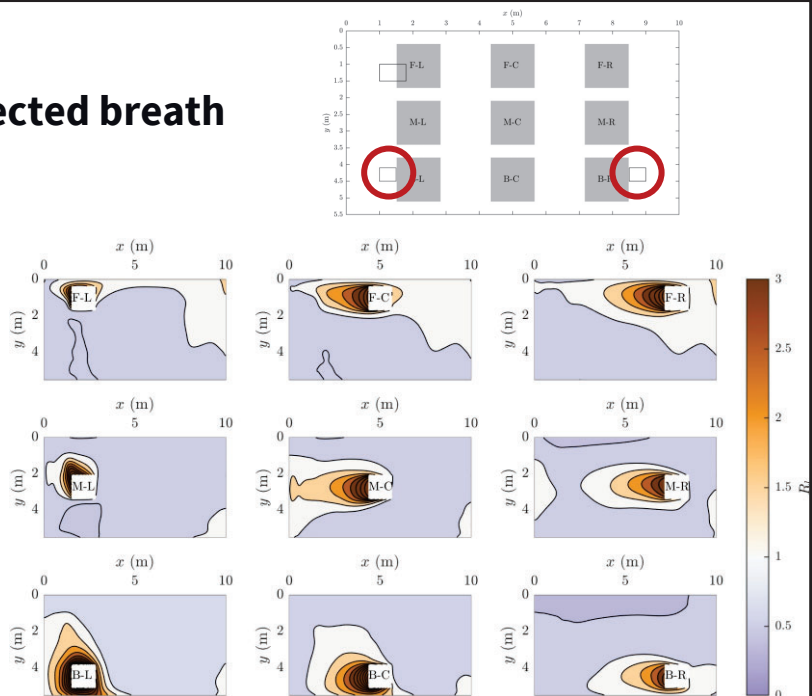
CO₂ as a proxy for infected breath

R_l ratio of:

- the infected breath proxy exposure (infected scalar concentration)
- to the CO₂ proxy exposure (local CO₂ concentration)

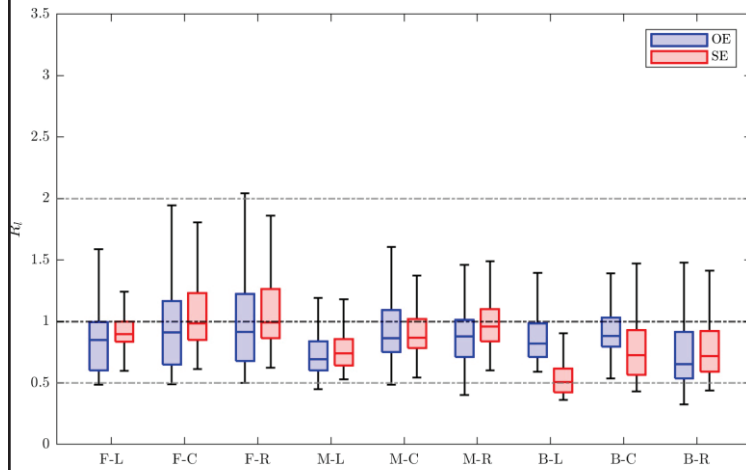
Accuracy depends on:

- Infector location
- Position of the vents



8

Conclusions and future work



Non uniform CO₂ distribution.
 Uncertainty due to a single point measurement will depend on the:

- Ventilation configuration
- Infector location

Exposure calculated from overall CO₂ within a factor 2 of exposure from infected breath.

Quanta generation rate varies by 4 orders of magnitude

9

Conclusions and future work

Ventilation can be estimated from steady-state concentration with $Q_c = \frac{NG}{C_c - C_a}$.

Accuracy of the CO₂ measurement matters along with accurate estimates of the **occupancy levels and CO₂ generation rates**.

SAMHE project aims to gather more contextual information:



10

Carolanne Vouriot
c.v.vouriot@sheffield.ac.uk



Indoor air modelling and infection risk assessment in a naturally ventilated patient room

Natalia Lastovets, Mohamed Elsayed, Ville Silvonen, Anni Luoto, Topi Rönkkö and Piia Sormunen



1

Motivation

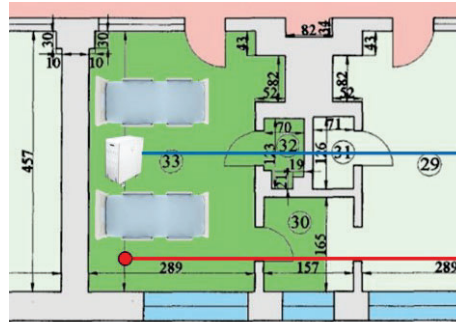
- **Sufficient ventilation in hospitals** is crucial to reduce **airborne virus exposure** for occupants.
- Current **infection risk models** favor **mechanically ventilated** facilities.
- Finnish hospitals use mechanical ventilation, while **southern and eastern European countries** like Romania rely more on **natural ventilation**.
- The research aims to create an **infection risk assessment method** for naturally-ventilated healthcare buildings.



2

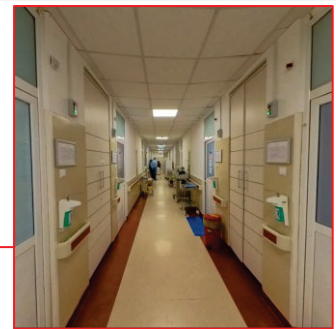
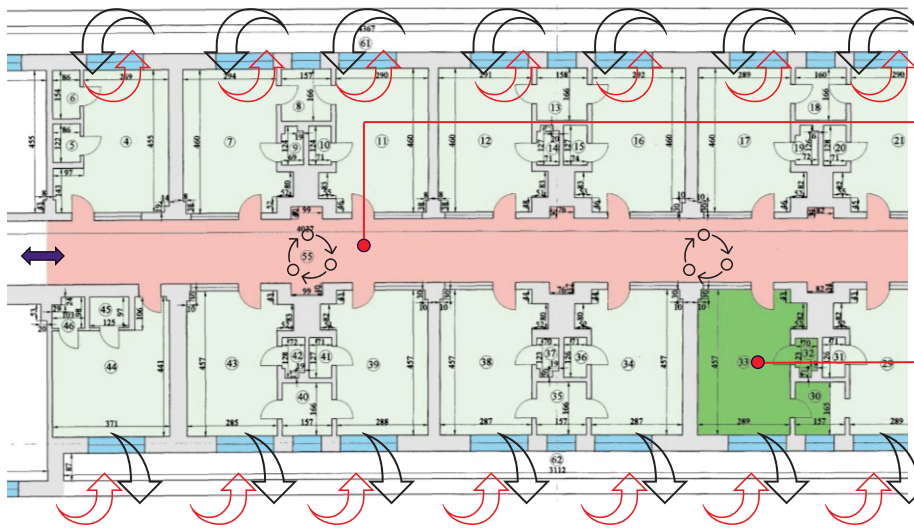
Case study

- Matei Bals hospital, Romania, Bucharest built at the beginning of the 20th century
- Studied patient room is located on the second floor of a four-story hospital building
- A closed hallway separates the patient rooms on opposite sides of the building.
- The building is ventilated only by natural ventilation



3

Covid ward space



Corridor

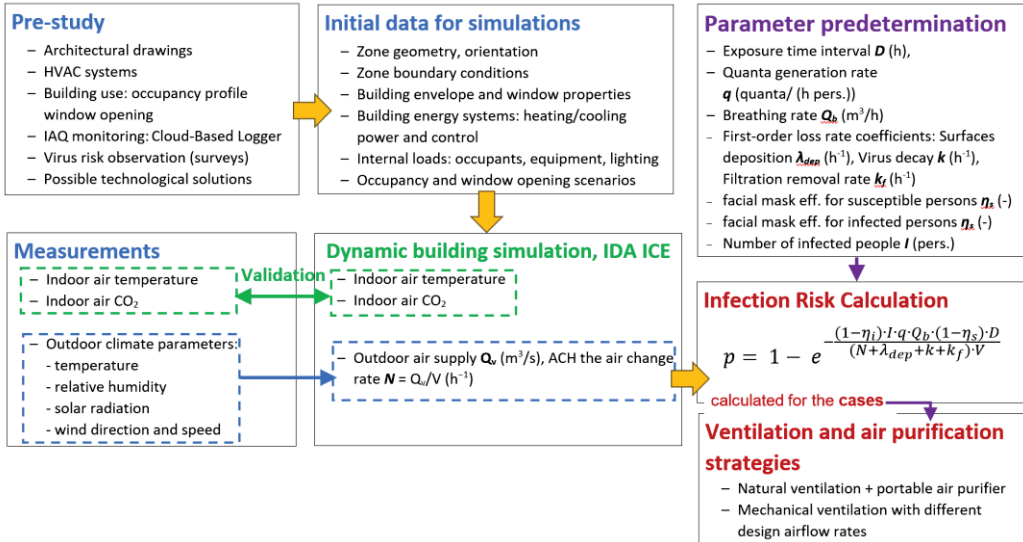


Covid room



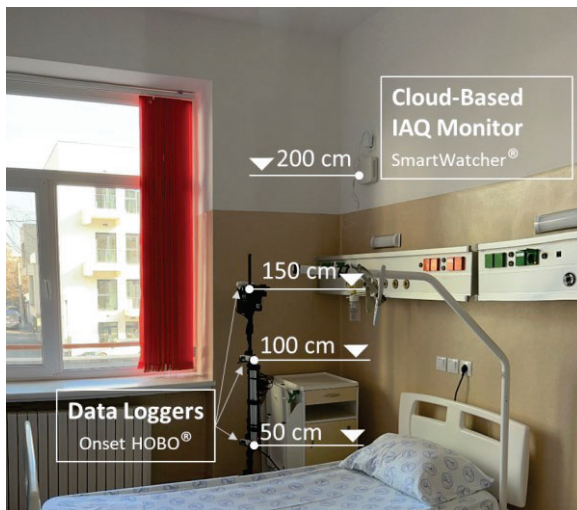
4

Methods

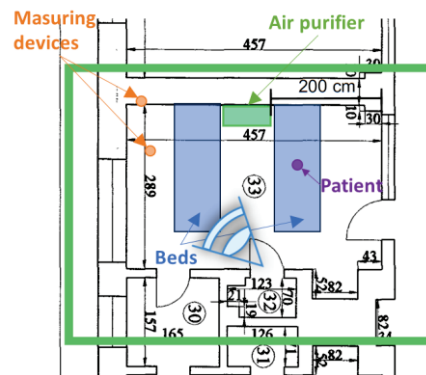


5

Measurements



- CO₂
- Indoor air temperature
- Outdoor air temperature data were taken from the local meteorological institute



6

Dynamic simulation of air exchange

Building constructions:

- outdoor walls, brick
U-value = 0.95 W/(m²K)
- windows: 2-plane glazing, no shading, U-value = 2.9 W/(m²K)
g-value = 0.7, area = 3.6 m²

Infiltration:

- Building leakage rate n₅₀ = 4 ACH
- Equivalent Leakage Area at ΔP₄ = 4 Pa, C_d = 1: ELA = 0.004 m²
- Height from the ground z_{out} = 4 m
- Height from the floor z_{in} = 1m

Wind pressure:

For city centre location:

- Wind profile coefficient: a₀ = 0.47
- Wind profile exponent: a_{exp} = 0.35
- Wind surface pressure coef. C_{d,wind}

Outdoor climate:

- Measured outdoor climate data Bucharest, Romania:
Outdoor temperature T_{out}, relative humidity H_{out}, solar radiation, wind speed v_{ref} at reference level (H_{ref} = 10m) and wind direction

Zone outlet:

- ζ_{term,in} = 1
- Z_{exh} = 2.9 m

Air exhaust through ventilation shaft

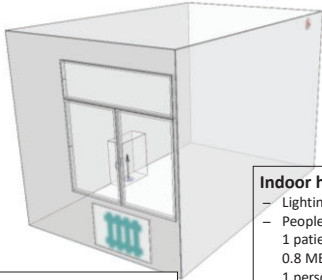
- shaft length dz_{shaft} = 7m
- shaft area A_{sh} = 0.023 m²
- roughness ε = 0.003 m
- Pressure loss coef. ζ_{term,out} = 0.6
- shaft top height dz_{out} = 12.9 m

Heating, water radiator

- Q_{heat} = 60 W/m²

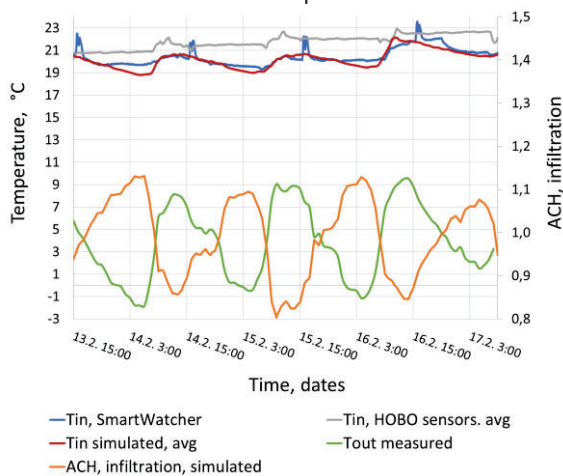
Indoor heat gains:

- Lighting: 164 W
- People:
1 patient (constant): 0.8 MET and 0.61 clo
1 personnel (periodic): 1.2 MET and 0.8 clo
- Air purifier: 51 W

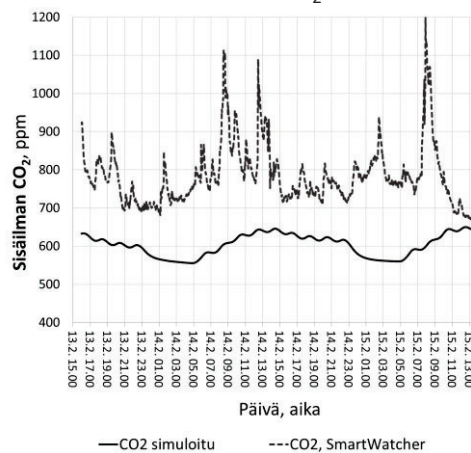



Results

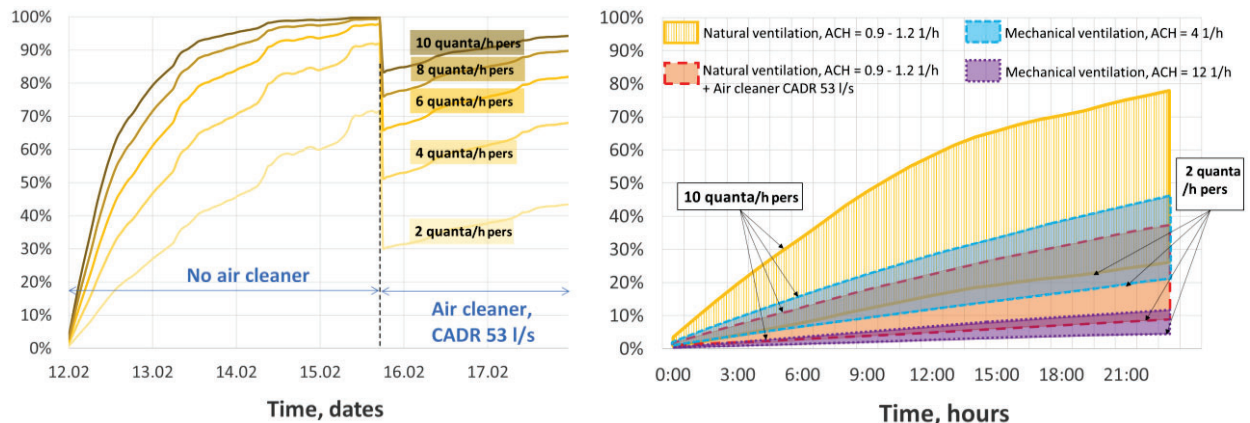
Indoor and outdoor air temperatures and ACH



CO₂



Infection risk probability



9

Conclusions

- The simulation model calculated dynamic air flows using natural ventilation to assess infection risk.
- Based on the case study results, natural ventilation alone does not effectively dilute airborne pollutants.
- Combination of ventilation strategy and air purification are needed in the patient room to prevent infections.
- The simulation tool can be used for infection risk analysis to optimize the operation of the hospital ventilation system.



10

43rd AIVC October 4-5, 2023

Performance of Local Ventilation System Combined with Underfloor Air Distribution as Preventative Measures for Infectious Diseases in Consulting Room



2023/10/5



Jun Yoshihara
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Graduate School of Engineering, **Osaka University**, JAPAN

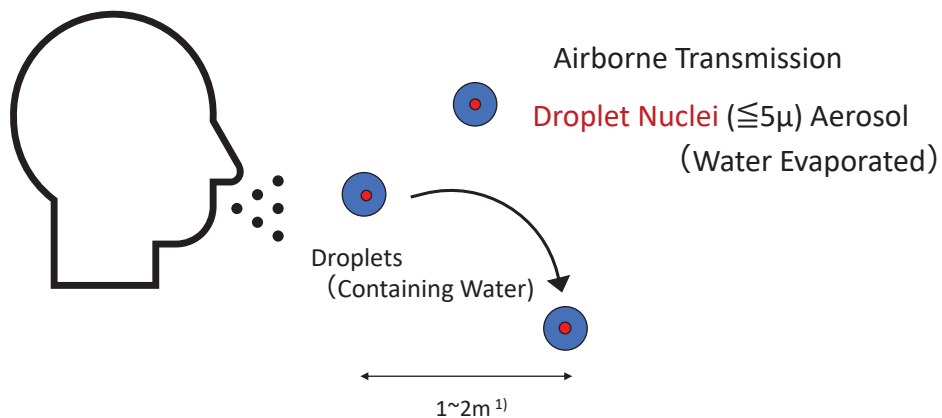
1

Introduction

2

Pandemic of COVID19 (2019.10~)

- Infection Route (Droplets and **Droplet Nuclei**)¹⁾



¹⁾ X. Xie¹, Y. Li¹, "How far droplets can move in indoor environments – revisiting the Wells evaporation–falling curve" Indoor Air 2007; 17: 211–225

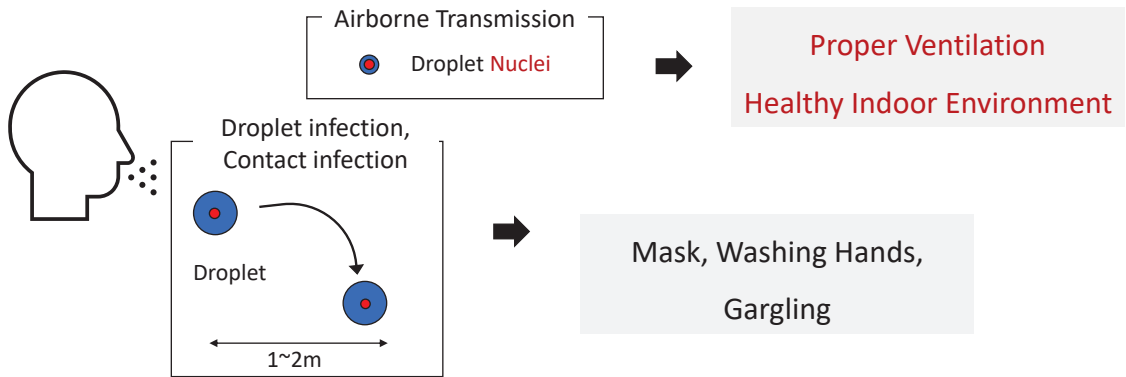
2

Introduction

3

Pandemic of COVID19 (2019.10~)

- Infection Route (Droplets and Droplet Nuclei) ¹⁾

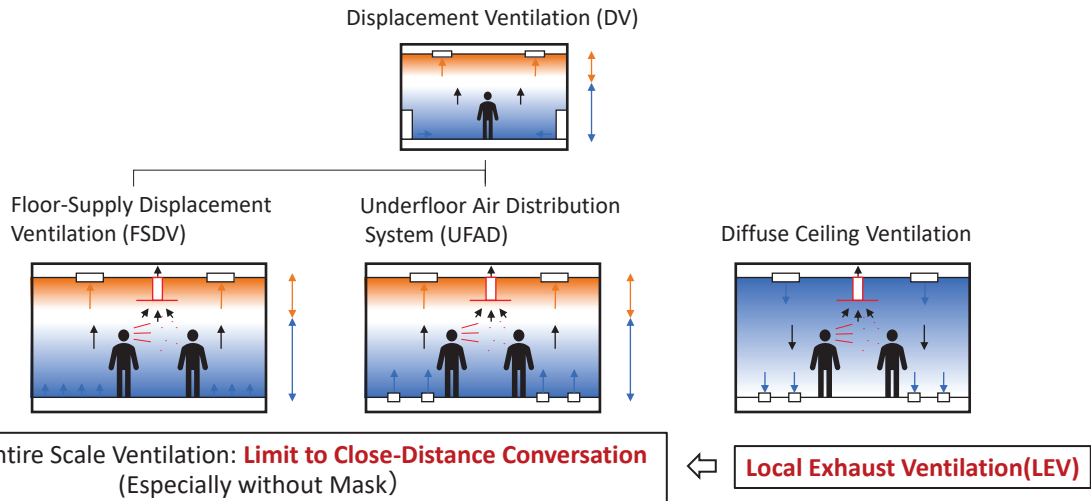


3

Introduction

4

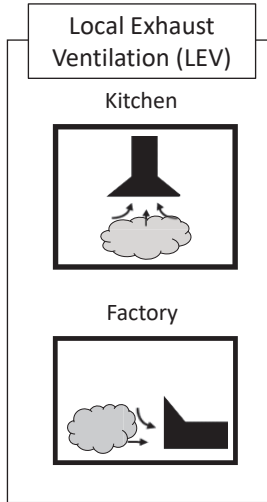
Various Ventilations to Prevent Airborne Transmission



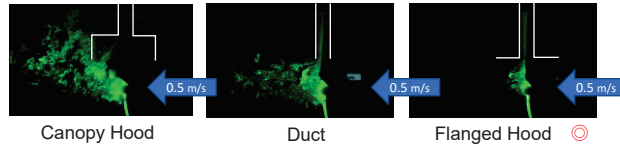
4

Introduction

5



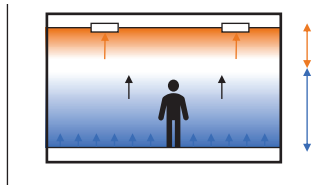
Previous Resrch²⁾
Pollutant collection performance of local exhaust system under crosswind



Performance : Largely affected by surrounding airflow

FSDV: Form a calm airflow field

Floor-Supply Displacement Ventilation (FSDV)



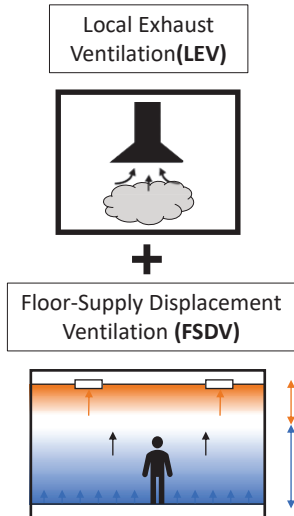
2) M. Komori, T. Yamanaka, T. Kobayashi, N. Choi, N. Kobayashi : Study on Pollutant Collection Performance of Local Exhaust System under Side Wind Airflow (Part 2) Effects of Hood Shape and Pollution Source Heat Generation on Pollutant Collection Performance

5

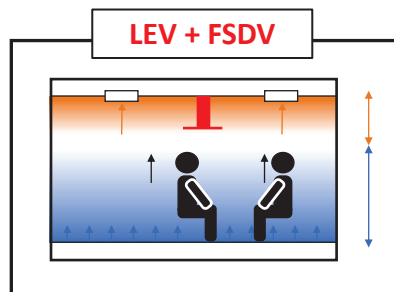
Introduction

6

For close-distance conversation...



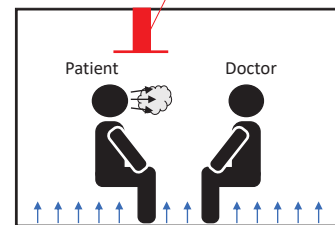
Proposed Ventilation System



Assumed Place

Consulting Room

LEV (hood)



► High Potential to Meet Infectors

Easy to Assume The Infectors Position

6

Experiment Method

7

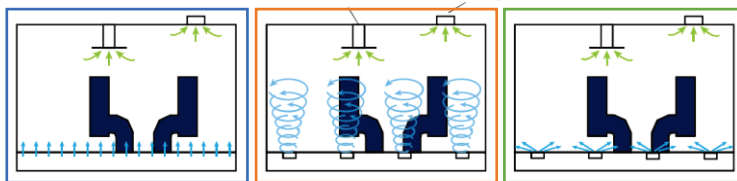
Method (Parameter)

8

Parameter : Under-floor Flow Supply Manner → Verified the significance of air conditioning with full floor ventilation

Case A: Consulting Room (Hood: Above the patient)

Hood (0,50,100,150,200,300,400,500 m³/h) Ceiling exhaust (air flow balance)



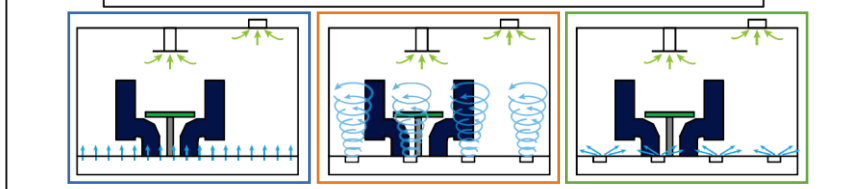
Floor-Supply Ventilation Swirling Flow Type Diffuser Displacement Flow Type Diffuser

* Flow Rate
1,000 m³/h (=50ACH)



Swirling Flow Type Diffuser*)

Case B: General Conversation (Hood: Middle of the Manikins)



Displacement Flow Type Diffuser *)

Total 48

*) 空研工業HP電子カタログ P10, https://www.kuken.com/wp-content/uploads/2020/08/FukidashiSuikomi_20200728P.pdf

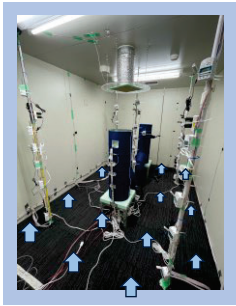
8

Method (Parameter)

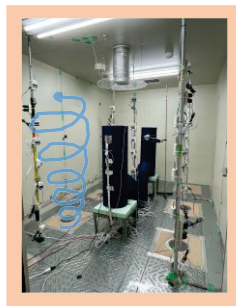
9

Picture of Experiments

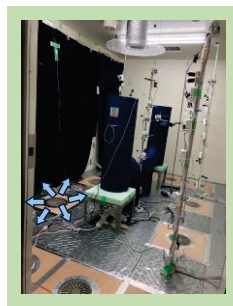
Case A: Consulting Room (Hood: Above the patient)



Floor-Supply Ventilation



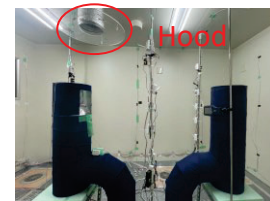
Swirling Flow Type Diffuser



Displacement Flow Type Diffuser

Side View

Case A: Consulting Room



Case B: General Conversation

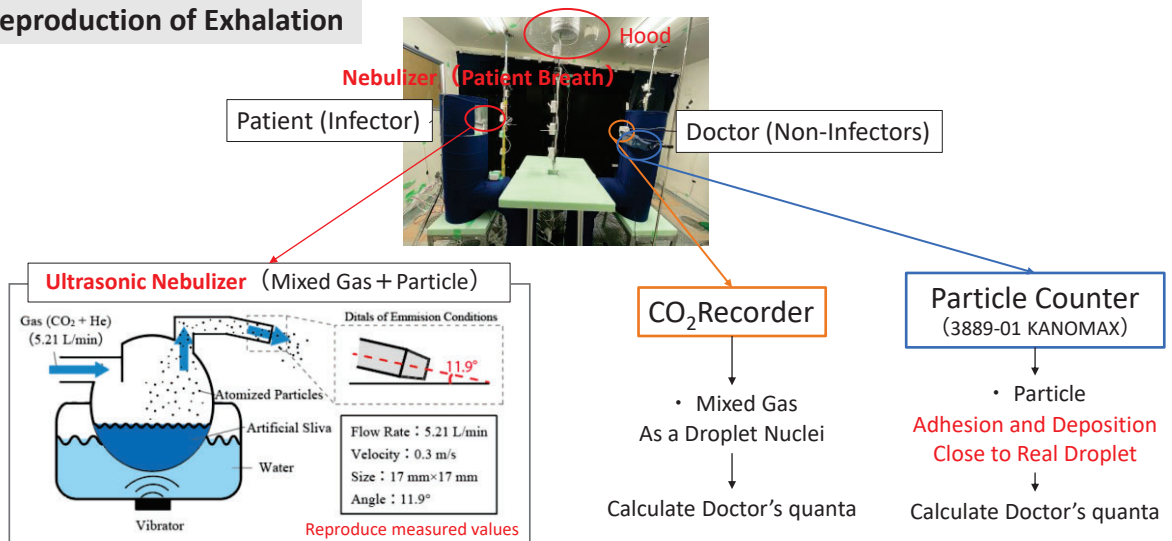


9

Method (Reproduction of Exhalation)

10

Reproduction of Exhalation



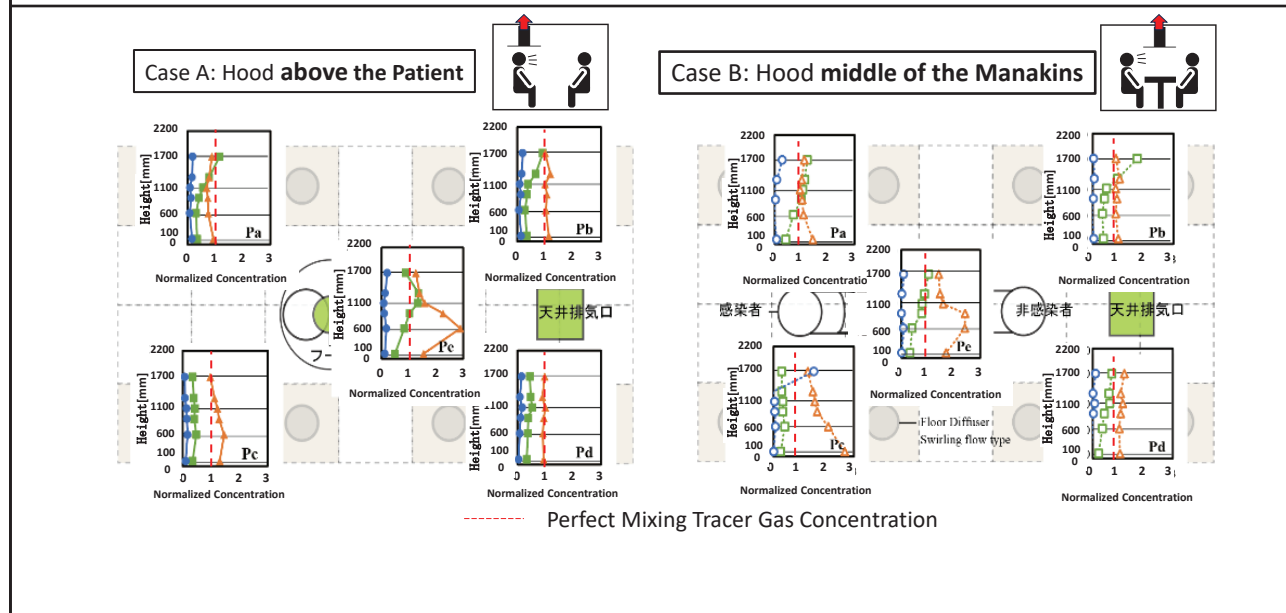
10

Results

11

Results (Vertical Distribution of Tracer Gas)

12

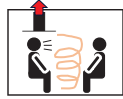


12

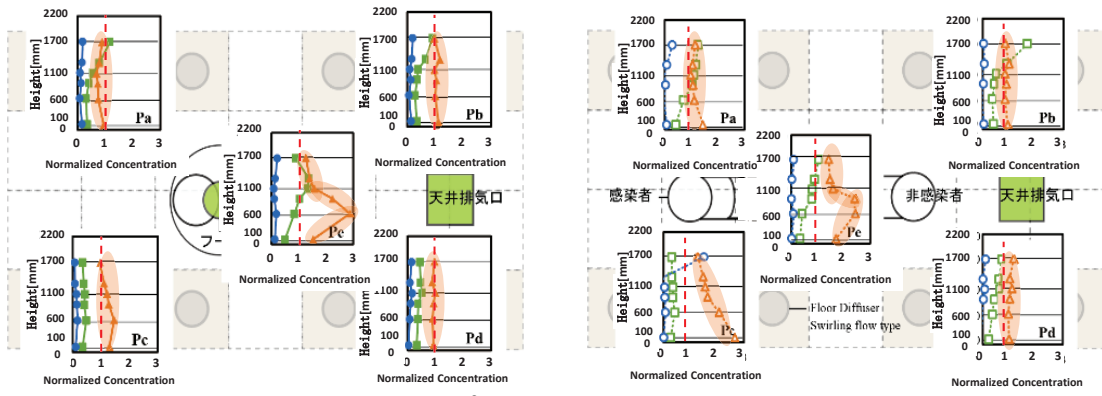
Results (Vertical Distribution of Tracer Gas)

13

Case A: Hood above the Patient



Case B: Hood middle of the Manakins



----- Perfect Mixing Tracer Gas Concentration

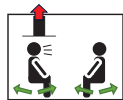
Swirling Flow Type Diffuser : Normalized Concentration = 1 → The Room : Perfect Mixing

13

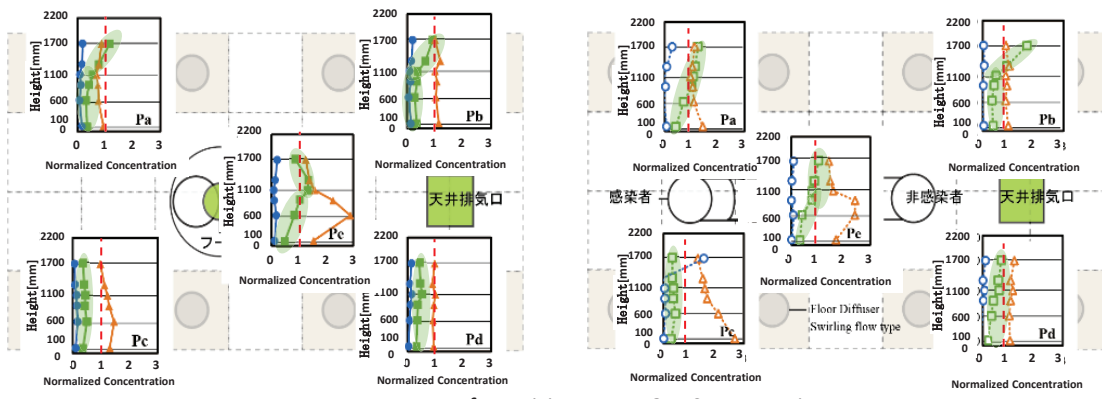
Results (Vertical Distribution of Tracer Gas)

14

Case A: Hood above the Patient



Case B: Hood middle of the Manakins



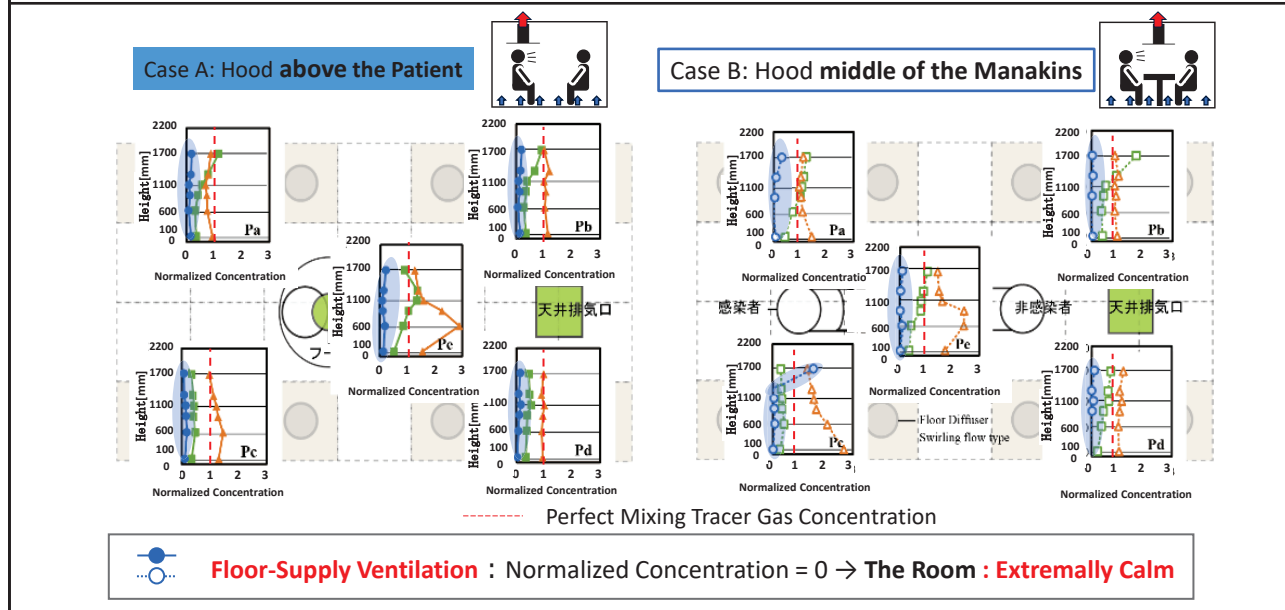
----- Perfect Mixing Tracer Gas Concentration

Displacement Flow Type Diffuser : Normalized Concentration = 0.5~1 → The Room : Relatively Calm

14

Results (Vertical Distribution of Tracer Gas)

15



15

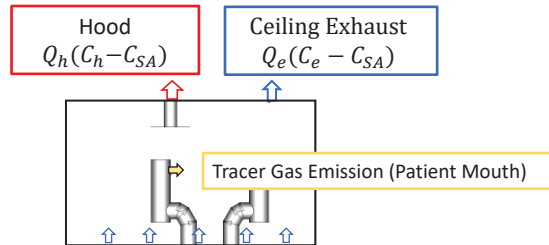
Method (Calculation)

16

Capture Efficiency

$$\eta = \frac{Q_h(C_h - C_{SA})}{Q_h(C_h - C_{SA}) + Q_e(C_e - C_{SA})}$$

η [-]: Hood capture efficiency
 Q_h [m³/h]: Hood flow rate
 Q_e [m³/h]: Ceiling exhaust flow rate



C_{qd} : Quanta concentration in front of the doctor's mouth [quanta/m³]

$$C_{qd} = q \cdot \frac{C_d}{M} = \frac{\text{(quanta Emission Rate)}}{\text{[quanta/h]}} \times \frac{\text{(Tracer Gas Concentration in front of Doctor's mouth) [m³/m³]}}{\text{(Tracer Gas Emission Rate) [m³/h]}}$$

Assume Conversation → $q = 42$ [quanta/h] ⁴⁾

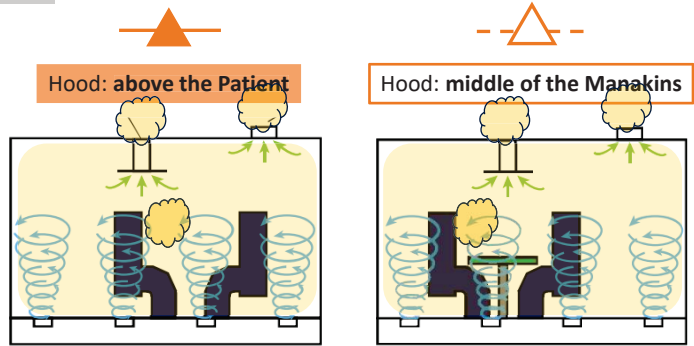
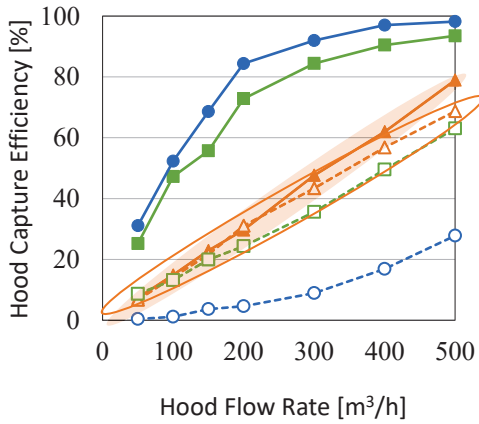
4) REHVA:COVID-19 guidance document version 4.2021

16

Results (Hood Capture Efficiency of Tracer Gas)

17

Hood Capture Efficiency of The Tracer Gas



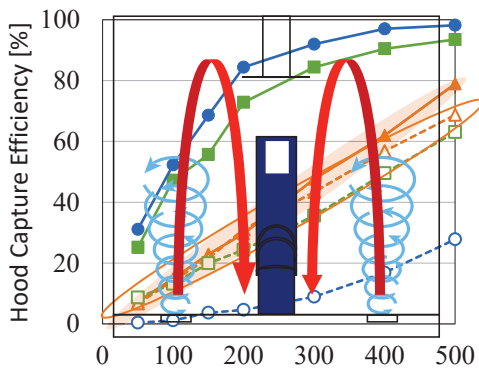
Tracer Gas : **Mixed Well**
 → **The Difference** of Hood Capture Efficiency : **Small**

17

Results (Hood Capture Efficiency of Tracer Gas)

18

Hood Capture Efficiency of The Tracer Gas



* In order to Visualize : Breath Flow Rate was a little Big

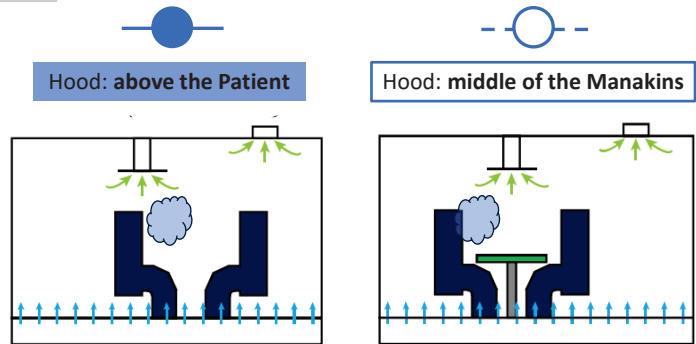
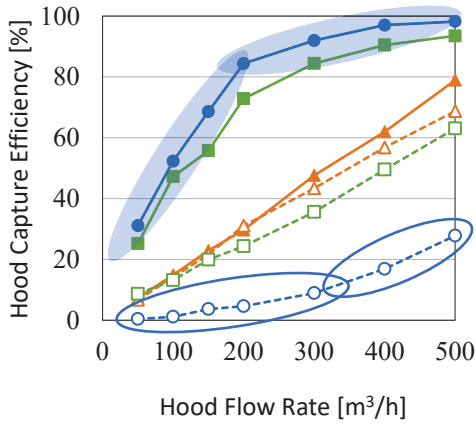
Swirling Flow type at 100 m³/h = 50 ACH → Supplying up Flow > Exhaust → **Circulation Flow ?**

18

Results (Hood Capture Efficiency of Tracer Gas)

19

Hood Capture Efficiency of The Tracer Gas



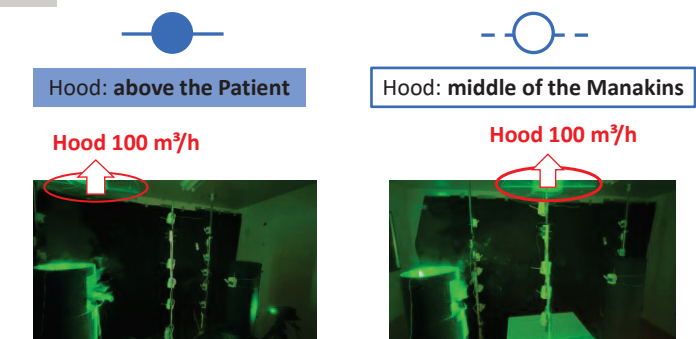
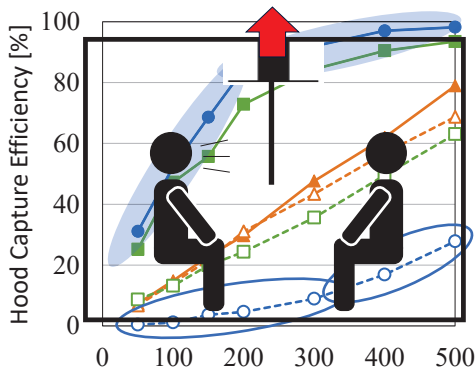
Tracer Gas : **Go Up** After Emitted
 → **The Difference** of Hood Capture Efficiency : **Large**

19

Results (Hood Capture Efficiency of Tracer Gas)

20

Hood Capture Efficiency of The Tracer Gas



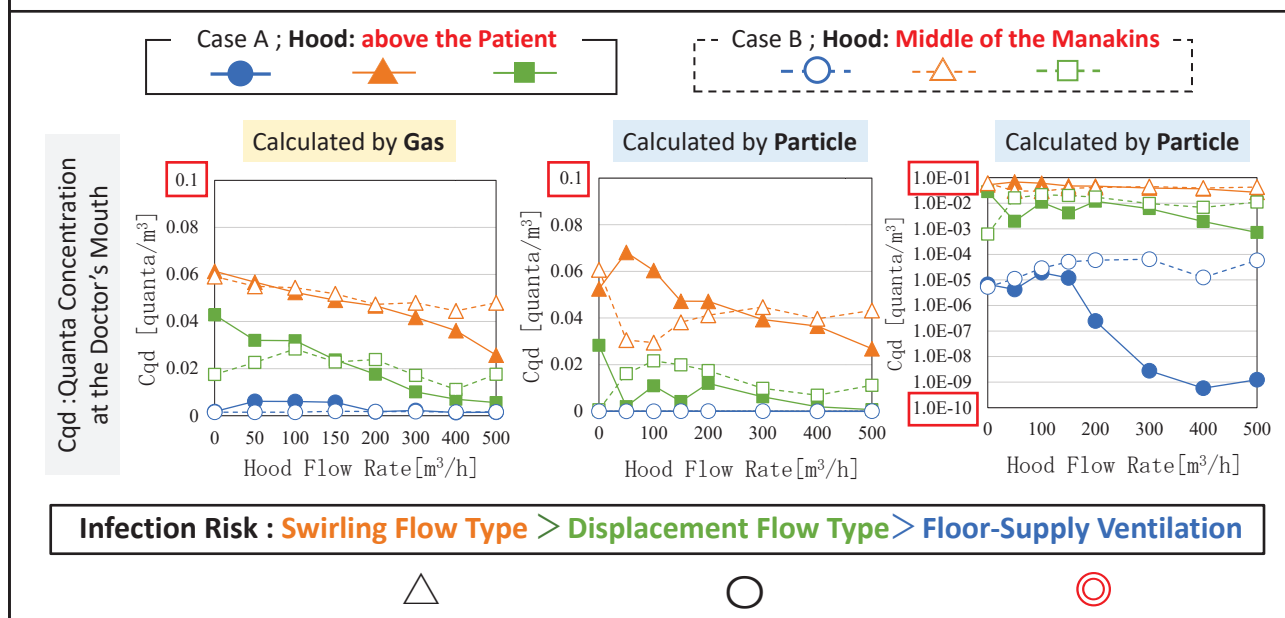
* In order to Visualize : Breath Flow Rate was a little Big

Hood couldn't affect the doctors Breathing Zone → **Other ways should be considered to** Install in a Meeting Room

20

Results (Infection Risk: Gas and Particle)

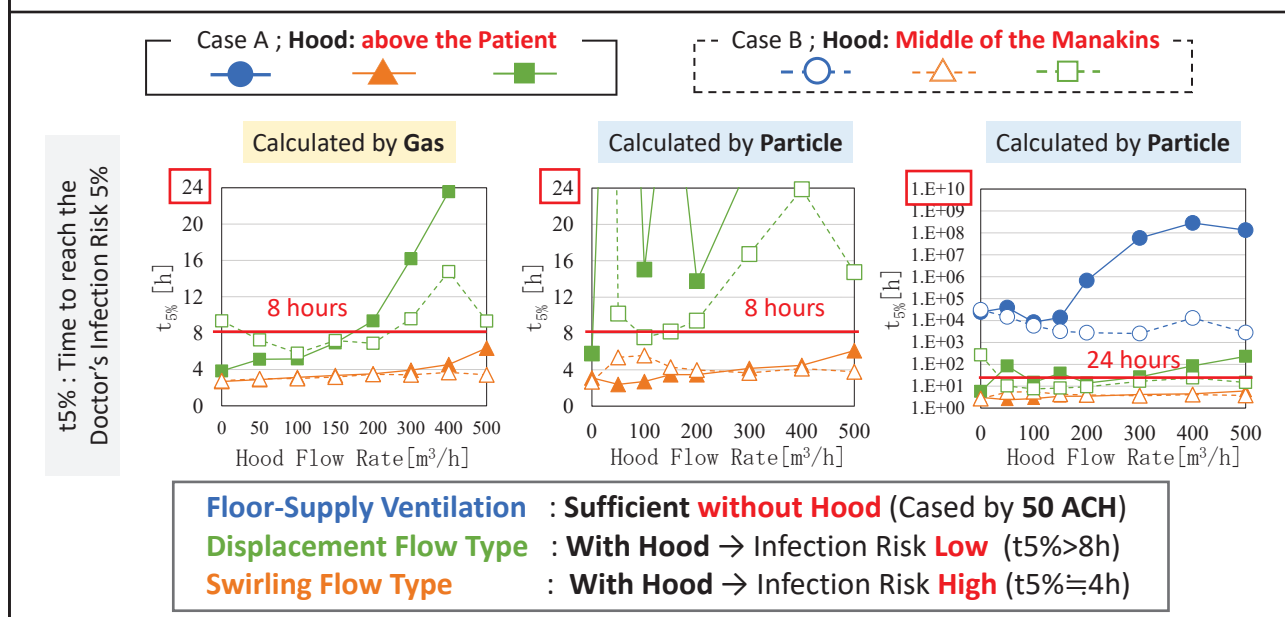
21



21

Results (Infection Risk: Gas and Particle)

22



22

Summarization

23

Air Flow Condition

Floor-Supply Ventilation : Normalized Concentration = 0 → **The Room : Extremely Calm**

Displacement Flow Type : Normalized Concentration = 0.5~1 → **The Room : Relatively Calm**

Swirling Flow Type : Normalized Concentration = 1 → **The Room : Perfect Mixing**

Infection Risk

Floor-Supply Ventilation : **Sufficient without Hood** (Cased by **50 ACH**) (t5%>24h)

Displacement Flow Type : **With Hood** → Infection Risk **Low** (t5%>8h)

Swirling Flow Type : **With Hood** (500 m³/h) → Infection Risk **High** (t5%≐4h)

Case B (Hood: Middle of the Manakins): **Hood couldn't work properly**

→ **Other ways should be considered** in a Meeting Room, Restaurant

* **Limitation: 50 ACH (1,000 m³/h)** (Decided by the Experimental Settings)

Future View: Experiments at 6~12 ACH, Large Hood, Air Purifier, Partitions

23

Thank you for your attentions!



Ph.D.student Jun Yoshihara

OSAKA UNIVERSITY

24

The numerical investigation of human micro-climate with different human simulators

Haruna Yamasawa (Osaka University)
Sung-Jun Yoo (Kyushu University)
Kazuki Kuga (Kyushu University)
Kazuhide Ito (Kyushu University)

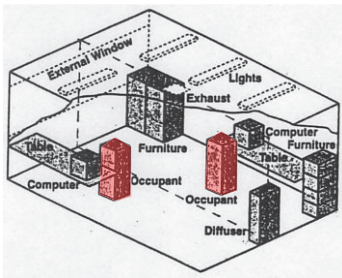
1

Introduction: General background

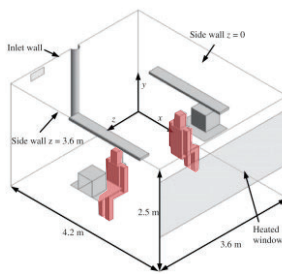
2



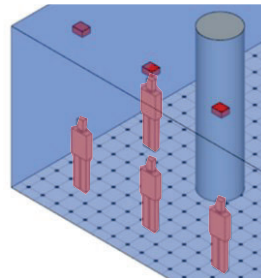
- ✓ The computational fluid dynamics (CFD) is commonly used in various studies
- ✓ Indoor environment: Human simulator (HS) is needed to be installed



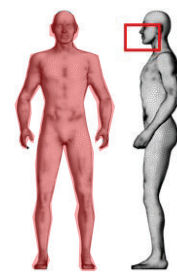
Yuan et al. (1999)



Chen et al. (2013)



Yin et al. (2021)



Yoo et al. (2022)

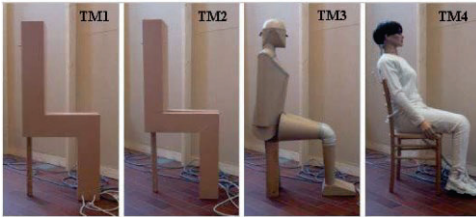
- ✓ The researchers develop HS by themselves addressing previous works
- ✓ The **geometries** varies by the year of publications and the aim of the studies
- ✓ The thermal plume is important especially in high-efficiency vent. systems

Computer
Simulated
Person

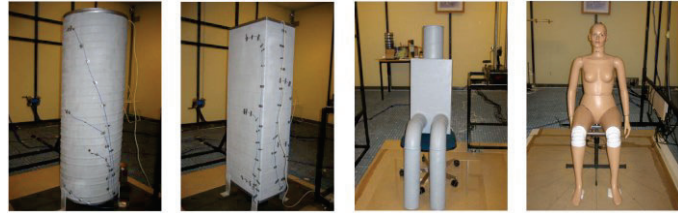
2

Research question:

What influence does the HS geometries have on the micro-climate?



Nielsen et al. (2003)

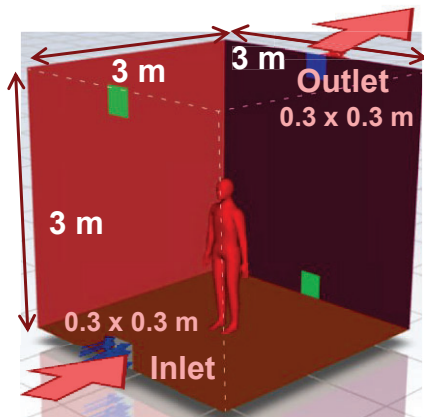


Zukowska et al. (2007)

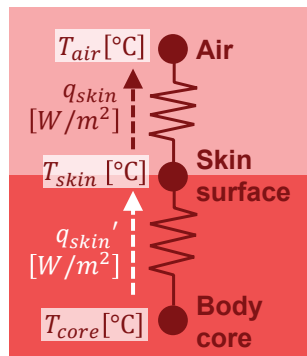
Present work:

CFD analysis is conducted under steady state

→the heat generation and thermal plume from human simulators



Fanger's model



Analytical domain & conditions

Analytical conditions

Turbulence model	SST <i>k-ω</i> model
Radiation	S2S model
Inlet	0.1 m/s, 20°C
Outlet	Gradient zero
Skin surface	Fanger's model (Fanger, 1972)
Other walls	Gradient zero (Adiabatic)

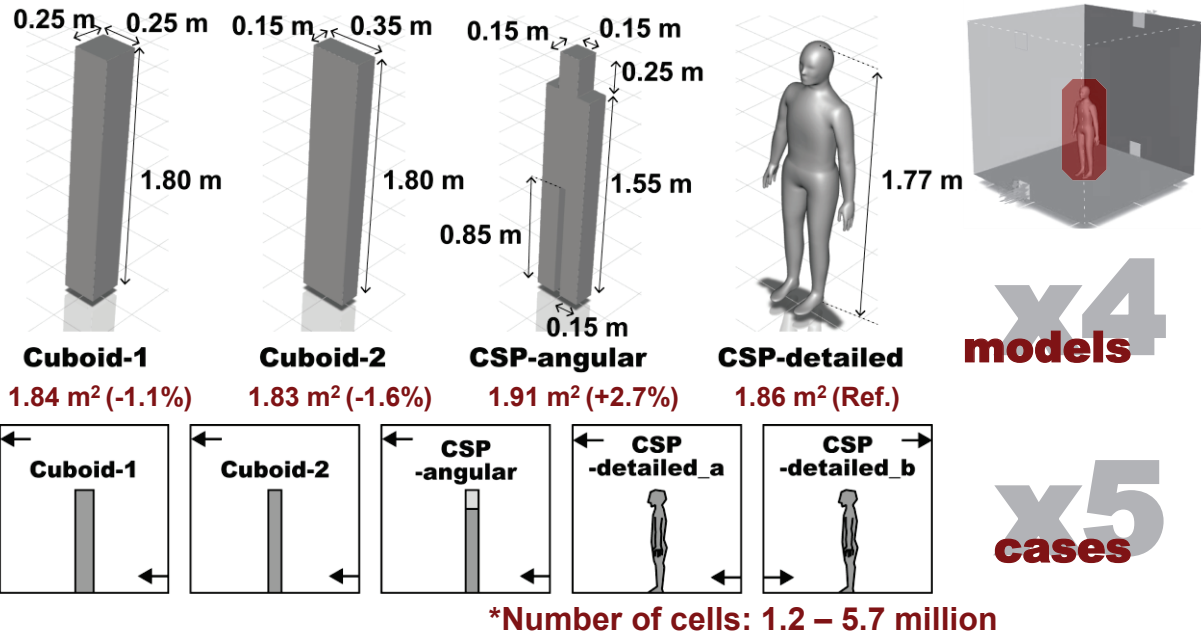
Heat flux through the skin surface [W/m²]

$$q_{skin} = q'_{skin} = K(T_{core} - T_{skin})$$

$$*K = 18.5 [W/m^2°C] \quad *T_{core} = 36.4 [°C]$$

Studied cases

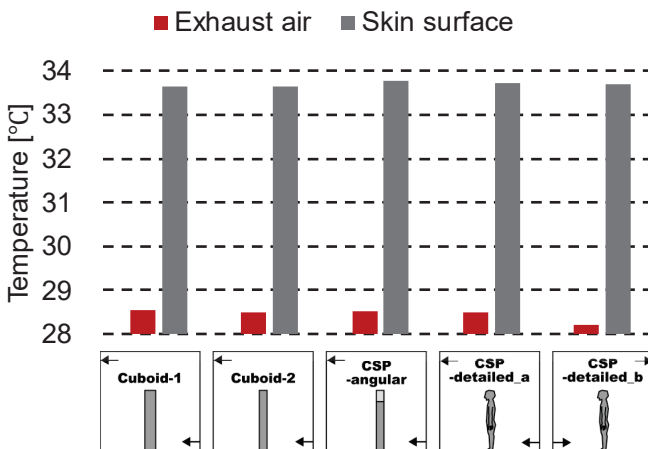
5



5

Results: Temperature

6



Even the simple model worked well regarding the temperature!

However, the points are:

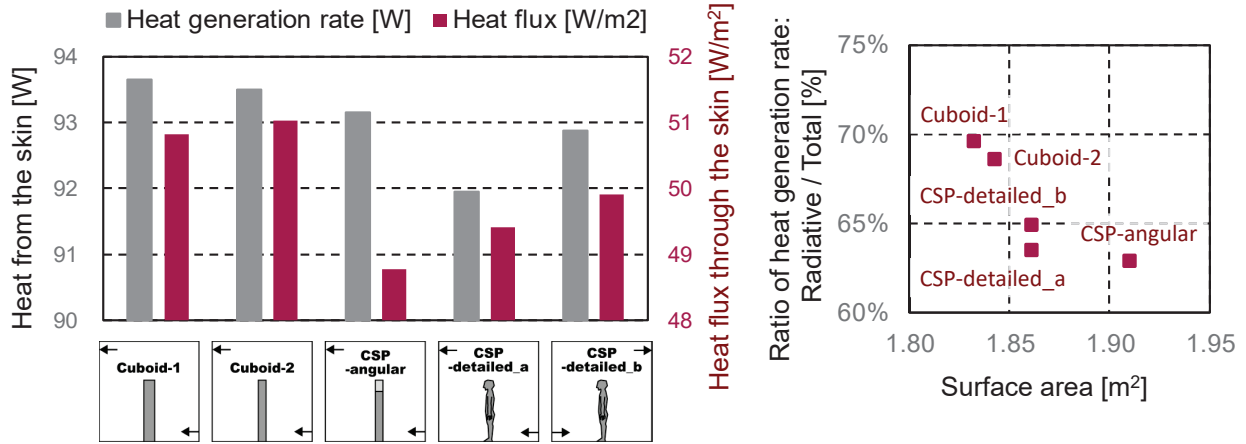
- ✓ Heat generation rate
- ✓ Thermal plume

✓ The exhaust and skin surface temperature are almost the same among all the cases

6

Results: Heat generation from the skin

7

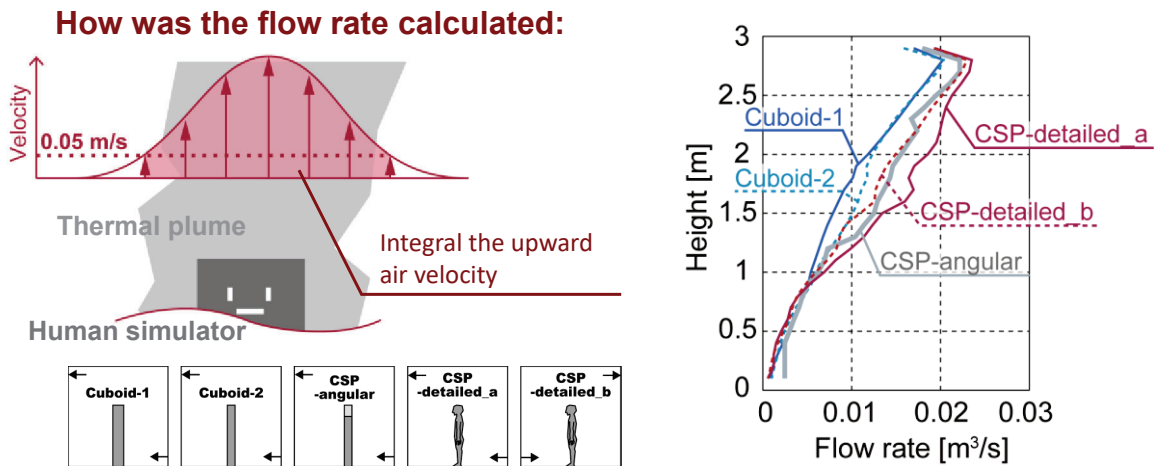


- ✓ The heat generation is all most the same
- ✓ The ratio between rad.-convec. is different
- ✓ The surface area seems to have some influence on the rad.-convec. ratio?

7

Results: Flow rate of thermal plume

8



- ✓ The flow rate profile of Cuboid-1 and Cuboid-2 are similar to each other
- ✓ The flow rate profile of CSP-angular is similar to that of CSP-detailed_b
- ✓ There are some discrepancies between cuboids and CSPs (Due to rad.-convec. ratio? Geometry?)

8

The numerical investigation of human micro-climate with different human simulators

Conclusions & perspectives

§ The studied models

- ✓ two kinds of cuboids, one simple CSP, and one detailed CSP

§ Results and discussions

- ✓ It is shown that although the total sensible heat generation is almost the same among all the cases, however, the ratio between radiative and convective heat transfer differed.
- ✓ This also led to the difference in thermal plume flow rate.

§ Perspective

- ✓ HSs with absolutely the same surface area
 - ✓ Contaminant concentration
 - ✓ Clothing
- etc...

IEA EBC Annex 87

Energy and Indoor Environmental Quality Performance of Personalised Environmental Control Systems (PECS)

Bjarne W. Olesen and Ongun Berk Kazanci
Intl. Centre for Indoor Environment and Energy, Technical University
of Denmark

AIVC2023, October 2023, Copenhagen

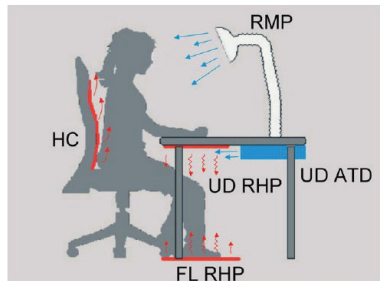
WHAT IS PECS?

- Personal Environmental Control System (PECS) with the functions of heating, cooling, ventilation, lighting and acoustic has advantages of controlling the localized environment at occupant's workstation by their preference instead of conditioning an entire room.
- This improves personal comfort, health and energy efficiency of the entire heating, ventilation and air-conditioning (HVAC) system substantially.
- Personalized ventilation will also protect against cross contaminations, which are critical in open plan offices and work places with close distance.





Source: Melikov 2010



Source: Watanabe et al. 2010



Source: Zhang et al. 2010

Why PECS?

- Has several benefits compared to ambient (total volume) conditioning systems
 - Improved comfort, health and productivity
 - Higher satisfaction with the indoor environment, due to
 - Improvements in the immediate indoor environment experienced by the occupants
 - Possibility of personalized control
 - Potential energy savings
 - Increasing focus on individual differences between people → PECS can address these individual differences
 - Even more relevant due to COVID-19 (pandemic-proofing)

Why PECS?

- Not entirely new – significant amount of research exists
- Despite the proven benefits
 - No design guide or manual for such systems and their integration in building HVAC systems
 - Far from "solved", still several issues to be addressed
 - Not at the level of a common solution in buildings
 - Very limited "real world" and commercial examples

Meetings

- 2022 Preparation phase
 - May 20th, 2022 Rotterdam
 - October 17-18th, Copenhagen
- 2023-2025 Working Phase
 - 1. meeting of the working phase of IEA EBC Annex 87, 19th and 20th of May 2023, Tokyo
 - 2. meeting in September 11-12, 2023 (Lausanne, Switzerland, before CISBAT2023 Conference)
- 2026 Reporting Phase

OBJECTIVES

- Establish design criteria and operation guidelines for PECS
- Quantify the benefits regarding health, comfort and energy performance.
- Control concepts and guidelines for operating PECS in spaces with general ambient systems for heating, cooling, ventilation and lighting.

SCOPE

- Includes all types of PECS for local heating, cooling, ventilation, air cleaning, lighting and acoustic.
- Includes desktop systems, which are mounted on desks or integrated in a furniture
- Chairs with heating/cooling and ventilation.
- Wearables, where heating/cooling and ventilation are included in garments or devices attached to occupants' body.
- [Not including cars](#)

TARGET AUDIENCE

- Manufacturers (who need design guidelines)
- Building owners and consultants (who need information on performance, advantages, problems, operation, how PECS is operated together with other building systems)
- Users (need same info as building owners and for home workplaces)
- Standardisation Bodies (revision of standards for indoor environmental quality).

Activities

- Seminar in CLIMA2022 (ca. 30 people attended)
 - https://clima2022.org/extra_content/seminar-new-iea-ebc-annex-on-personalized-environmental-control-systems-pecs/
- Topical session in AIVC2022 (ca. 45 people attended)
 - <https://aivc2022conference.org/topical-session-08/>
- AIVC Webinar on Monday, 12th of Dec 2022
 - Registration link:
 - <https://inive.webex.com/inive/j.php?RGID=rd4ce219c23589874419137a1bff98911>

Subtask A: Fundamentals

- **Leader**
 - **Mariya P. Bivolarova, Technical University of Denmark, Denmark**
- **Co-leader:**
 - **Dolaana Khovalyg, EPFL, Switzerland**
- **Activity A1:** Definition and identification of the requirements of PECS in terms of localized and background Indoor Environmental Quality (IEQ) i.e., thermal, air quality, lighting, and acoustics.
- **Activity A2:** Outline the benefits of PECS regarding comfort, health and productivity based on literature and new research.
- **Activity A3:** Outline the minimum energy cost requirements for PECS.

Subtask B: Applications and Technologies

- **Leader:**
 - **Kai Rewitz, RWTH Aachen University, Germany, pending**
- **Co-leader**
 - **Joyce Kim, University of Waterloo, Canada**
- **Activity B1:** Summarize the working principles, capabilities and limitations of existing PECS, based on literature.
- **Activity B2:** Identify future development and improvement suggestions for PECS for optimal energy, IEQ and cost performance.

Subtask C: Control, operation and system integration

- **Leader:**
 - **Joon-Ho Choi, University of Southern California, USA**
- **Co-leader**
 - **Wooyoung Jun, University of Arizona, USA**
- **Activity C1:** Identify and summarize existing methods for controlling PECS (including sensors used for control).
- **Activity C2:** Develop guidelines on integrating PECS with ambient conditioning systems in buildings.

Subtask D: IEQ and Energy Performance evaluation

- **Leader:**
 - **Douaa Al-Assad, KU-Leuven, Belgium**
- **Co-leader**
 - **Marco Perino, Politecnico di Torino, Italy**
- **Activity D1:** Collection of existing methods of studying and testing PECS.
- **Activity D2:** Identification of generic power requirements for PECS to achieve energy savings compared to ambient conditioning systems.
- **Activity D3:** Development of universal and standardized ways of evaluating and reporting performance of PECS.

Subtask E: Policy and advisory actions

- **Leader:**
 - **Rajan Rawal, CRABSE, CEPT University, India**
- **Co-leader:**
 - **TBD**
- **Activity E1:** Summary of national and international building codes and standards regarding PECS.
- **Activity E2:** Develop ways of overcoming current barriers for a wide implementation of PECS in buildings.
- **Activity E3:** Provide input to existing national and international standards about requirements, characteristics, and performance of PECS.

DELIVERABLES

1. Guidebook on requirements for PECS
2. State-of-the-art report on PECS
3. Guidebook on PECS design, operation and implementation in buildings (including integration of PECS with ambient conditioning systems)
4. Report on test methods for performance evaluation of PECS
5. Universal criteria about requirements, characteristics, and performance of PECS to be used in national and international standards

Participating countries

- Australia, Belgium, Brazil, Canada, China, Denmark, Finland, France, Germany, Italy, Netherlands, Republic of Korea, Singapore, Switzerland, Turkey, USA

Further information

- www.iea-ebc.org/projects/project?AnnexID=87

Subtask D: IEQ and Energy performance evaluation of PECS

Douaa Al-Assaad¹, **Hilde Breesch**¹, Marco Perino²

¹ KU Leuven, ² Politecnico di Torino

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Aim & activities

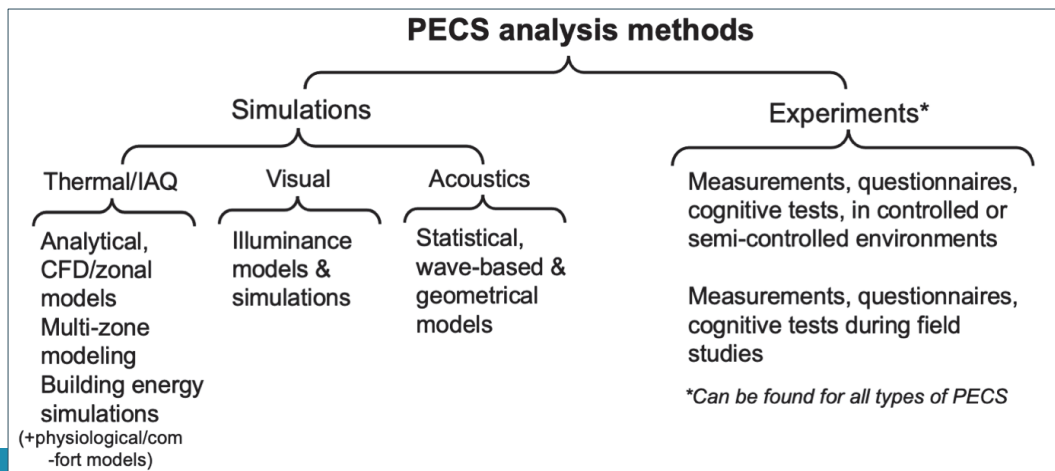
- Aim
 - Review methods evaluating performance of PECS: IEQ (thermal, air quality, acoustics, lighting) and energy performance
 - Identify missing performance evaluation methods.
 - Performance evaluation: qualitative and quantitative KPIs from literature and new metrics (Subtask B)
- Activities
 - D1: Collect existing methods studying and testing PECS
 - D2: Identify energy requirements PECS
 - D3: Develop universal and standardized ways evaluating and reporting IEQ performance PECS

2

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Activity D.1 Literature review performance methods

- Overview methods analyzing PECS' performance



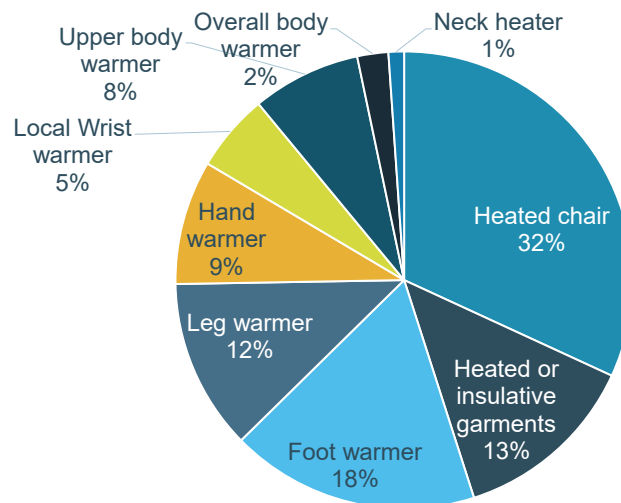
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Activity D.1 Literature review performance methods

- Example heating PECS:
review 69 studies



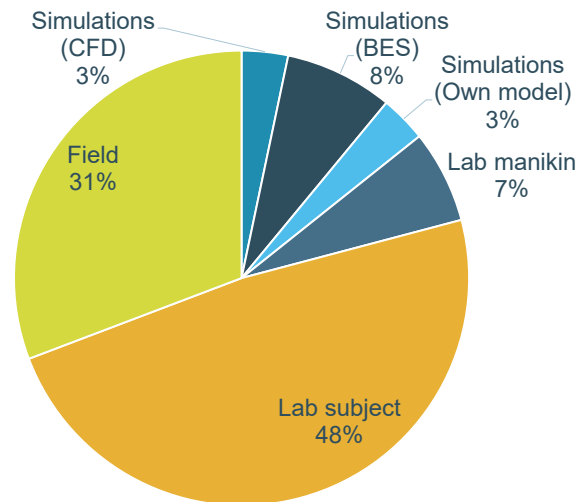
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Activity D.1 Literature review performance methods

- Example heating PECS:
review 69 studies

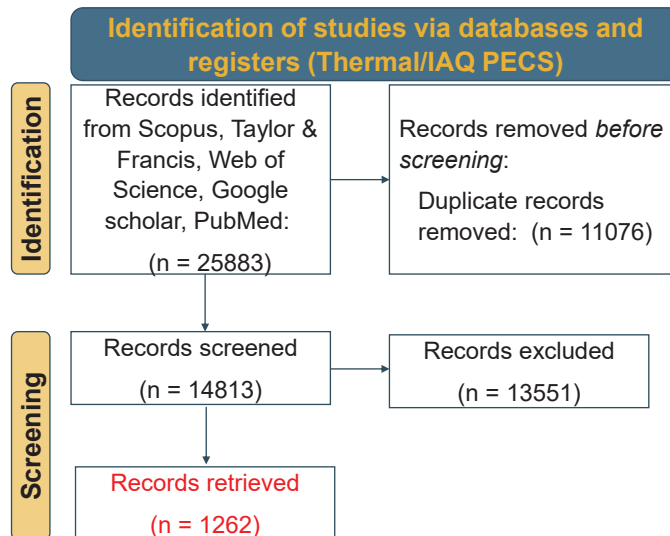


Activity D.1 Literature review performance methods

- Review objectives
 - Overview different methods analyzing PECS' performance
 - How to use these methods effectively to obtain accurate performance output?
 - How to ensure quality of results?
 - Which method most suited for specific analysis? Or specific PECS?
 - Challenges and/or limitations using this method? -> Agenda for future activities

Activity D.1 Literature review performance methods

- Systematic review process
 - cfr. Prisma guidelines
 - Next step:
 - Split into different analysis methods
 - Extract info methodological elements



Activity D.1 Literature review performance methods

- Examples methodological elements for thermal/IAQ PECS
 - Analytical methods/ CFD
 - Numerical methods
 - discretization schemes
 - convergence criteria
 - occupant modeling
 - modeling HVAC system diffusers and terminals
 - boundary conditions
 - Building Energy Simulations (BES)
 - Building type
 - Location
 - envelope characteristics
 - modeling of energy systems
 - thermal zoning
 - schedules and patterns
 - modeling occupant behavior

Activity D.1 Literature review performance methods

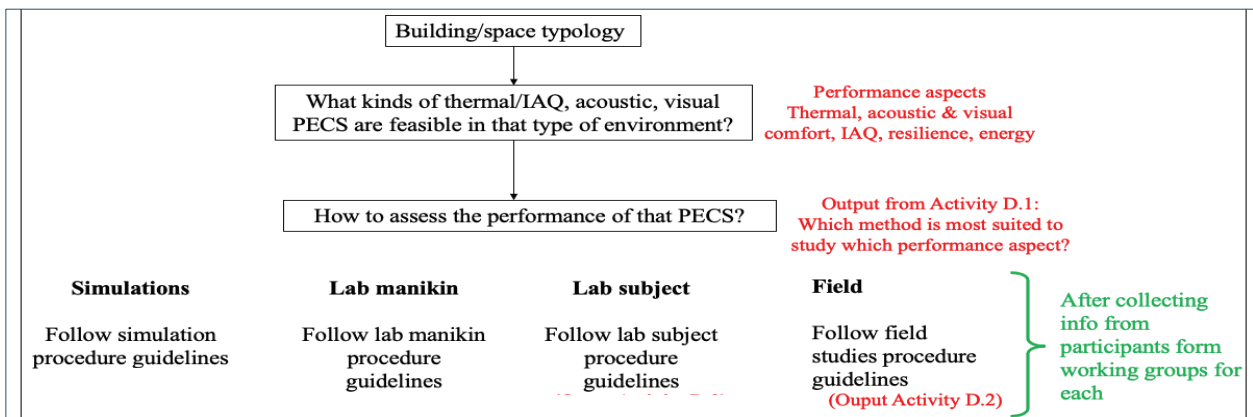
- Examples methodological elements for thermal/IAQ PECS
 - Chamber studies
 - Chamber characteristics and conditioning
 - human subjects or manikin details
 - measurements' locations, Intervals, durations
 - Field studies
 - Building type
 - outdoor climatic conditions
 - number of PECS installed
 - occupancy schedules, patterns, density
 - questionnaires and response rate
 - measurement locations, intervals and durations

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Activity D2 & D3: guidelines evaluating and reporting performance PECS



Procedure on how to combine the different methods to holistically assess the performance of any PECS in any type of space

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Subtask D: IEQ and Energy performance evaluation of PECS

Douaa Al-Assaad¹, Marco Perino², Hilde Breesch¹

¹ KU Leuven, ² Politecnico di Torino



1



Dragos-Ioan Bogatu

drabo@dtu.dk

International Centre for Indoor Environment and Energy – ICIEE, DTU SUSTAIN, Technical University of Denmark

Physiological sensing for thermal comfort assessment

2

Highlights and agenda

- Systematic review of data-driven thermal comfort prediction for HVAC control.
- Overview of physiological indicators, sensing strategies, and derived indicators.
- Statistical analysis of the influence of features on model prediction performance.
- Discuss principles and examples of automatic control for PECS

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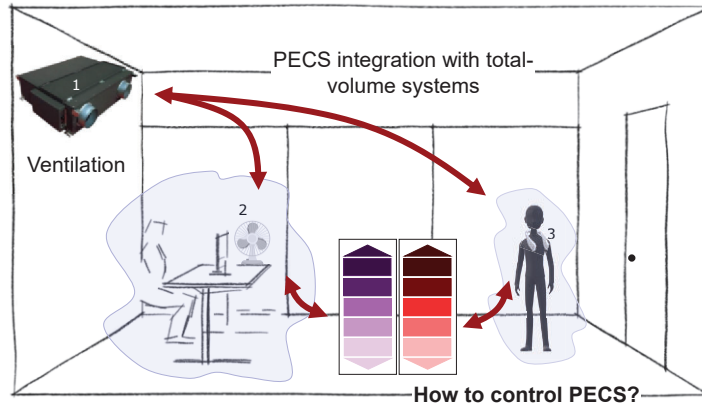
Personalised environmental control system

“A **personalised Environmental Control System (PECS)** is a system that can provide individually controlled thermal, air quality, acoustic, or luminous environments in the immediate surroundings of an occupant, without affecting directly the entire space and other occupants’ environment.”

IEA EBC - Annex 87 - Energy and Indoor Environmental Quality Performance of Personalised Environmental Control Systems, <https://annex87.iea-ebc.org/>

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Personalised environmental control system



Is the occupant feeling hot, cold, or neutral?

¹ Mitsubishi Electric; ² Freepik.com; ³ Coolpriser.dk Neck Fan

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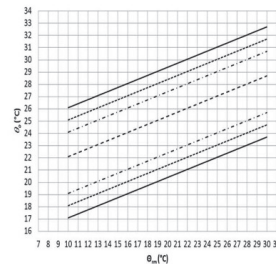
Existing comfort models

”The **PMV** is an index that predicts the **mean value of the votes of a large group of persons** on the 7-point thermal sensation scale, based on the heat balance of the human body”

$$PMV = [0,303 \cdot \exp(-0,036 \cdot M) + 0,028] \cdot \left[\begin{aligned} &(M - W) - 3,05 \cdot 10^{-3} \cdot [5733 - 6,99 \cdot (M - W) - p_a] - 0,42 \cdot [(M - W) - 58,15] \\ &- 1,7 \cdot 10^{-5} \cdot M \cdot (5867 - p_a) - 0,0014 \cdot M \cdot (34 - t_a) \\ &- 3,96 \cdot 10^{-8} \cdot f_{cl} \cdot [(t_{cl} + 273)^4 - (\bar{t}_r + 273)^4] - f_{cl} \cdot \epsilon_c \cdot (t_{cl} - t_a) \end{aligned} \right]$$

CEN, EN ISO 7730, 2006; EN 16798-1:2019

Adaptive: indoor temperature as a function of running mean outdoor temperature (takes into account adaptation)



6

Personal comfort models

- Adequately represent the need of a specific individual
- Input is required from the occupant as comfort is a function of both thermoregulatory and psychological aspects
- Need for subjective thermal response emulators to reduce interaction with the occupant
- Indicators can be derived from **environmental** changes, and **thermoregulatory** reactions, **behavior**, and occupant differences in body type (i. e., **anthropometric** information).

Bogatu et al., Human physiology for personal thermal comfort-based HVAC control – A review, B&E, 2023, <https://doi.org/10.1016/j.buildenv.2023.110418>

Physiological indicators

Physiological signals

Physiological indicators gathered for thermal comfort models stem from signals representative of human thermoregulation.

- **Neural activity:** brainwave [Hz] Alpha (8–14), Beta (13–35), Delta (0.5–4), and Gamma (35–45)
- **Heartbeat:** HR, HRV, SpO₂
- **Blood flow:** blood volume changes, blood pressure
- **Activity:** MET, activity level, standardized values
- **Temperature:** T_{SK} , T_{MSK} , T_{CORE}
- **Sweat:** EDA, RH_{SK}

May be used as indicators. Some signals are representative of stress, excitement, emotion

Bogatu et al., Human physiology for personal thermal comfort-based HVAC control – A review, B&E, 2023, <https://doi.org/10.1016/j.buildenv.2023.110418>

Derived indicators from physiological signals

Time, spectral, and non-linear domains

- Changes in time or over body & general statistics (mean, min, max)
- Representation of a signal or data in terms of its frequency components
- Short and long term signal perturbations and predictability of the signal

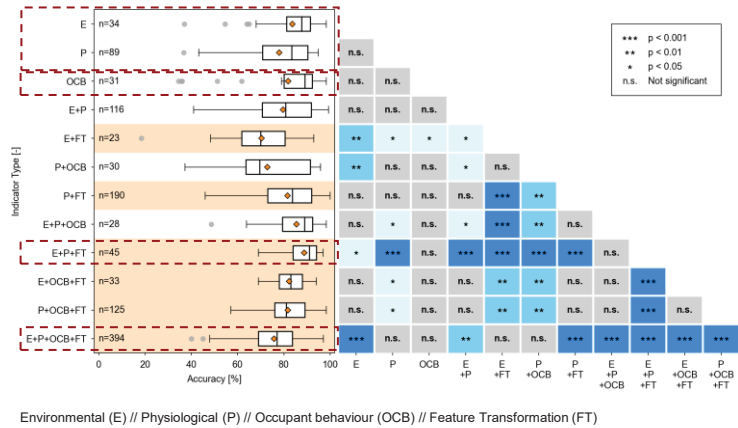
Examples: LF/HF, heat loss, gradient of skin temperature

Complemented by Environmental (E), Occupant behaviour (OCB), and Feature Transformation (FT)

Bogatu et al., Human physiology for personal thermal comfort-based HVAC control – A review, B&E, 2023, <https://doi.org/10.1016/j.buildenv.2023.110418>

Model performance (ML based)

- E > P
- Clothing and activity important
- FT important
- Including all (E, P, AB, FT) may not necessarily lead to a high accuracy



Bogatu et al., Human physiology for personal thermal comfort-based HVAC control – A review, B&E, 2023, <https://doi.org/10.1016/j.buildenv.2023.110418>

Model performance and physiological indicators

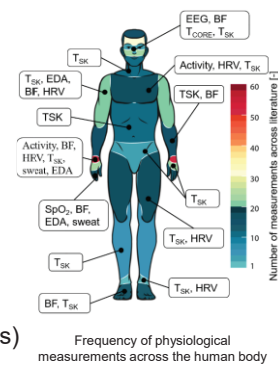
- An accuracy of 70–80% should be attainable. KNN potential baseline, however **no definitive list of physiological indicators** is available.
- **Sweat, RH_{SK} , and SpO_2** are **biased** to the hot part of the thermal sensation spectrum.
- **T_{SK} and heart** signals present **higher variability across thermal sensation states**.
- So far **only skin temperature and heart rate** were **included in control strategies**.
- The usefulness and availability of each indicator is **challenged** though **by the measuring strategy and location**.

Physiological sensing

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Physiological sensing

- Most measurements made at wrist level
- T_{SK} most investigated for each body part
- **Neural activity (EEG)** – light, easy to set up, intrusive
- **HRV (ECG)** – electrodes, least intrusive chest
- **HR** (PPG/RGB/Doppler), **SpO2** (PPG) – smartband
- **MET** (cardiopulmonary tester), **activity** (3-axial acc.)
- T_{SK} (thermocouples, button devices, smartbands, IR cameras)
- T_{CORE} (radio-pills, IR cameras, T_{SK} and heat flux)
- RH_{SK} , **sweat rate**, **EDA** (humidity, electrodes, durometer)



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Capabilities and limitations of current sensing strategies

- Wearable devices (cheap and mature products); wired/wireless (probes, smartwatch, headset)
- Contactless (RGB, IR cameras, Doppler)
- Not always 'plug and play'

Device	Capabilities	Limitations
Wearable	<ul style="list-style-type: none"> • May be integrated in a device attached to an occupant • May measure multiple parameters • Clothing does not interfere with the measurement • Mature products • May be relatively cheap • Can be placed directly on the skin • Can be placed on different and multiple body areas • Can be connected via cloud-based solutions 	<ul style="list-style-type: none"> • Measurement length dependent on the battery life and data storage capacity of the equipment • Could be intrusive and invasive (e.g., chest strap) • Accuracy issues (improper use, movement, fastening option) • Sensor accuracies may be unknown • Covers body area where measurement is made • Narrow operating ranges • Single point measurement • Influenced by physical pressure, insulation from fitting material and thermal inertia of the sensor • Inconvenient if wired
Contactless	<ul style="list-style-type: none"> • Can gather data on body areas not covered by clothing, e.g. face • Non-invasive and non-intrusive • Can capture a bigger surface area • Can detect changes from the skin naturally and directly impacted by the surrounding environment 	<ul style="list-style-type: none"> • May require a complex system consisting of multiple nodes (e.g. depth and thermal image camera) • Privacy concerns • Little flexibility regarding placement • Correction regarding clothing might be required • May not be suitable for multi-occupancy spaces due to the limited field of view • Error in detecting area of interest for measurements • Can be difficult to implement in building due to size and compatibility issues (e.g. in Building Management Systems). • Higher cost compared to wearable sensors

Bogatu et al., Physiological sensing for thermal comfort assessment, AIVC, 2023

Physiological sensing

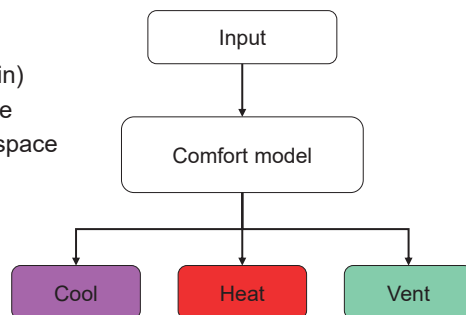
- **Further development required** – not all devices are 'plug-and-play' and designed for thermal comfort
- **Contactless devices can only** be used to acquire **skin temperature and HR** (low resolution, error correction required)
- Measurement **accuracy rarely provided** for consumer products
- **Smartbands acquire multiple physiological indicators** (consumer oriented devices protected by 3rd party protocols)
- Wearable devices are connected to the occupant which makes it **difficult to determine ownership, operation, and maintenance responsibility**

Automatic control

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Automatic control

- Trained model predicts state **cold / neutral / warm**
- Air temperature changed in **increments**, usually ± 1 K, in direction of comfort
- **Setpoint changes:**
 - made at $\Delta t = x$ min (usually 30 min)
 - when occupant provided a change
 - when occupants entered/left the space



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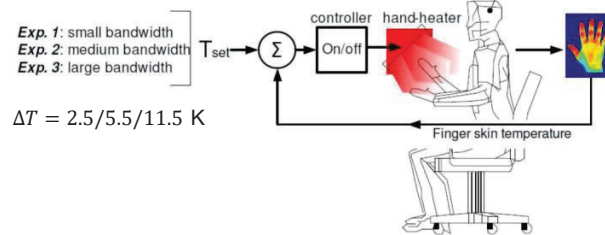
Examples

Automatic control

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Personal heater control

- Two incandescent reflector heating lamps (98 W)
- $T_{room} = 20\text{ }^{\circ}\text{C}$
- Input: finger skin temperature



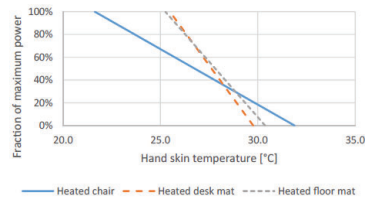
- In general it was possible to maintain neutral or higher TSV .

Zeiler et al., Thermal response of different body parts: the fingertip as control sensor for personalized heating, EnergyProcedia 78, 2015, 10.1016/j.egypro.2015.11.622

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Personalised heater control

- 18 °C, heated chair, desk mat, floor mat
- Personalized heating controlled proportionally to T_{SK} (hand, dorsal)
- Setting adjusted every 2 s.



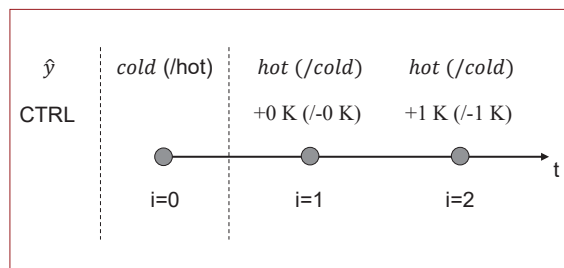
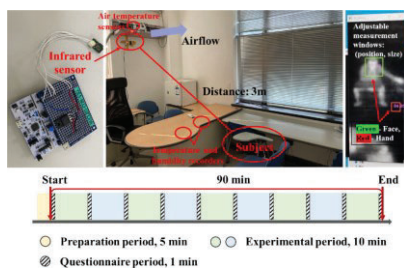
- Automatic control provided **same level of thermal comfort** as user control
- Average user control input similar to automatic control



Vesely et al., Personalized heating – comparison of heaters and control modes, Build. Environ., 2017, <https://doi.org/10.1016/j.buildenv.2016.11.036>.

Air conditioning control

- Single-person office with AC
- $22\text{ °C} < T_{room} < 30\text{ °C}$



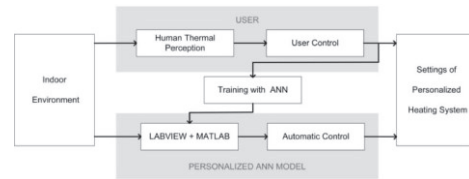
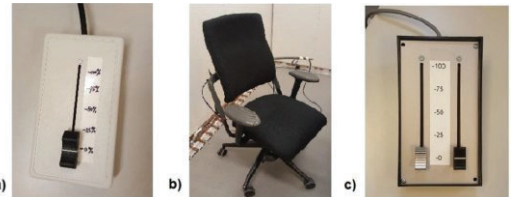
$\overline{TSV} = [-0.5, 0.5]$; $TCV = \text{not always 'comfortable'}$

- Algorithm: RF; Input: $\overline{T_{hand}}, \overline{T_{face}}, T_{room}$; Output: $\hat{y} = TSV$; $ACC = 84\%$
- ΔT : -1 K ($\hat{y} = \text{hot}$) ; $+1\text{ K}$ ($\hat{y} = \text{cold}$) ; $+0\text{ K}$ ($\hat{y} = \text{neutral}$)

Yeyu et al., Development of personal comfort model and its use in the control of air conditioner, E&B, 2023, <https://doi.org/10.1016/j.enbuild.2023.112900>

Personal comfort model for PECS

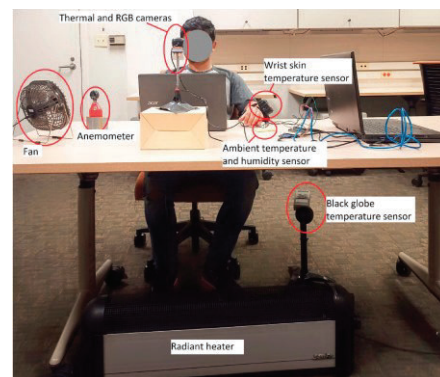
- 20 °C, 2 people, 14 test sessions (4h)
- Inputs: environmental (T_{air} , RH , \bar{T}_r); Output: setting of PECS
- The control voltage was used as the target of the predictive model
- Both test subjects felt comfortable throughout the tests and expressed their satisfaction with the automatic control



Katic et al., Neural network based predictive control of personalized heating systems, E&B, 2020, <https://doi.org/10.1016/j.enbuild.2018.06.033>

Personal comfort model for PECS

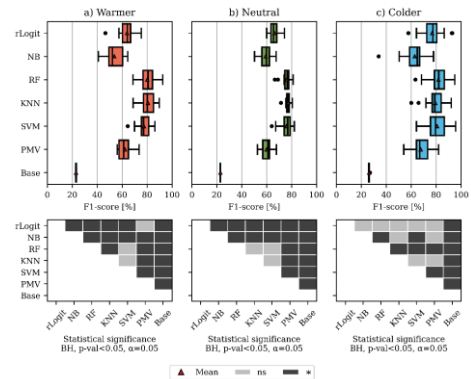
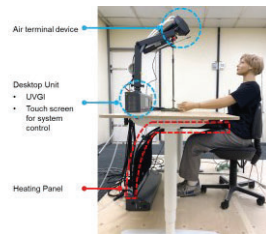
- 19 to 31 °C with 1°C/10 min, fan (1.2 m/s) or heater (1000 W), 0.57 clo
- PECS require separate comfort models
- Comfort models built without PECS may be enhanced by including PECS power/operation
- 2-5% increase in accuracy with environmental instead of physiological data (T_{SK})
- Model accuracy decreased with PECS (better sensing and modelling methods required)



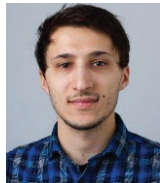
Aryal et al., Thermal comfort modeling when personalized comfort systems are in use: comparison of sensing and learning methods, B&E, 2020

Personalised comfort model for PECS

- 18 to 28 °C, steady-state, w/o and with PECS, 24 subjects
- Grouped according to general thermal preference
- Integrating self-assessed general thermal preference leads to higher prediction performance (11 to 18 %) than the PMV.
- Environmental indicators had the highest importance



Bogatu et al., Personalised Thermal Comfort Model for Automatic Control of a Newly Developed Personalised Environmental Control System (PECS), IAQVEC, 2023



drabo@dtu.dk

Bogatu et al., 'Physiological sensing for thermal comfort assessment', AIVC, 2023

Bogatu et al., "Human physiology for personal thermal comfort-based HVAC control – A review", <https://doi.org/10.1016/j.buildenv.2023.110418>

ASHRAE STANDARD 241

Max Sherman, SSPC241, Vice Chair
AIVC Conference; October 2023

1

ASHRAE Standard 241-2023 *Control of Infectious Aerosols*

Purpose

- Requirements for control of infectious aerosols to reduce risk of airborne transmission
- Occupiable space in existing and new buildings, additions, and major renovations
- Non-residential, residential, and health care spaces
- Covers outdoor air and air cleaning system design, installation, commissioning, operation, maintenance
- Specify *equivalent clean air* to be provided in *infection risk management mode*

Scope

- Based on reduction of *long range transmission* risk
- Does not establish overall requirements for acceptable indoor air quality but requires IAQ as a pre-requisite

2

2

BACKGROUND

Request from White House

- Quickly develop standard for future epidemics
- Hard deadlines

ASHRAE authorizes abbreviated process

- For first version only
- Expedites resources

“Tiger Team” effort

- 6 committed Working Groups.
- Done in 4 months vs. more normal 4 years

ASHRAE commits to

- Normalize standard (i.e. use normal process)
- Continually improve standard and fix gaps

3

3

Overview

1. Assess facility - condition and existing equivalent clean air delivered
2. Determine target equivalent clean air required by space and system
3. Determine need for additional equivalent clean air
4. Determine the best option for providing required equivalent clean air using outdoor air, particle filtration, and air cleaners tested as required, and operational measures
5. Prepare a Building Readiness Plan to document assessment and decisions
6. Perform repair and maintenance as needed and required
7. Implement upgrades if needed

4

4

Infection Risk Management Mode (IRMM)

The mode of operation in which measures to reduce infectious aerosol exposure documented in a building readiness plan are active

Decision on IRMM Enable / Disable: **Not 24/7**

- Public health official
- Owner
- Occupant

Why not all the time?

- Potential Energy use and cost increase
- Infection risk and consequences of infection vary over a wide range

An example of **resilience** applied to IAQ



Normal

IRMM

5

5

Equivalent Clean Airflow (ECA)

The flow rate of pathogen-free air that, if distributed uniformly within the breathing zone, would have the same effect on infectious aerosol concentration as the sum of actual outdoor airflow, filtered airflow, and inactivation of infectious aerosols

Concept on which the entire standard depends

- Determine ECA for infection risk mitigation (ECA_i)
- Determine total ECA for spaces, systems (V_{ECA})
- Analyze options to reach target in IRMM

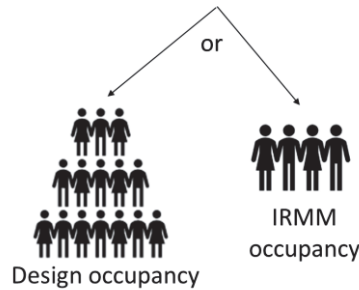
Also adopted from Epidemic Task Force guidance (same as *equivalent outdoor air*)

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ECAi depends on space type,
number of people, activity

$$V_{ECAi} = ECAi \times P_{Z, IRMM}$$



Double table ECAi for high vocalization spaces

Occupancy Category	ECAi	
	cfm/person	L/s/person
Correctional Facilities		
Cell	30	15
Dayroom	40	20
Commercial/Retail		
Food and beverage facilities	60	30
Gym	80	40
Office	30	15
Retail	40	20
Transportation waiting	60	30
Educational Facilities		
Classroom	40	20
Lecture hall	50	25
Industrial		
Manufacturing	50	25
Sorting, packing, light assembly	20	10
Warehouse	20	10
Health Care		
Exam room	40	20
Group treatment area	70	35
Patient room	70	35
Resident room	50	25
Waiting room	90	45
Public Assembly/Sports and Entertainment		
Auditorium	50	25
Place of religious worship	50	25
Museum	60	30
Convention	60	30
Spectator area	50	25
Lobbies	50	25
Residential		
Common space	50	25
Dwelling unit	30	15

7

Meeting the ECA target

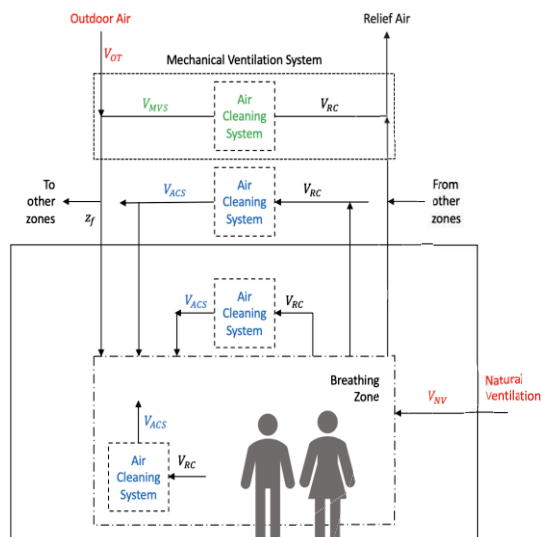
Requirement can be met by

- Outdoor airflow - mechanical/natural
- ECA from multi-zone air cleaning systems
- ECA from in-room air cleaning systems

Approach allows maximum flexibility to user

Limitations on compliance

- Must have prerequisite minimum outdoor air
- To receive credit toward meeting requirements, mechanical filters must be MERV-A 11 or higher or equivalent (ePM2.5>50%)
- MERV 11 acceptable until 1/1/2025
- Hope to raise to ePM1>50%



8

8

Air Cleaning

Reducing infectious aerosol concentration through capture and removal or inactivation

Air cleaning technologies

- Mechanical filters (including electret media)
- Germicidal ultraviolet light
- Reactive species - ionizers, photocatalytic oxidation, other oxidants

Mention of specific technologies in the standard is not endorsement!



9

9

Air Cleaning Effectiveness and Safety

Lack of information and standards related to air cleaning systems was a major problem during the COVID19 pandemic:

- Effectiveness - ability to remove or inactivate infectious aerosols
- Safety - adverse effects of direct exposure (UV-C, oxidants), secondary contaminants (particles, ozone); not required for pure mechanical filtration

Standard 241 establishes backup minimum requirements for effectiveness and safety testing in Normative Appendix A

- For when no applicable standard exists

Goal is a level playing field for all technologies

10

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Special Considerations: Dwellings



Building Readiness Plan (BRP)

Reduced detail for dwellings



ASHRAE EPIDEMIC TASK FORCE
BUILDING READINESS | Updated 8/17/2022

General Information

- Building Readiness Intent
- Building Readiness Team
- Building Readiness Plan

Epidemic Conditions in Place (E-CIP)

- Systems Evaluation
- Building Automation Systems (BAS)
- Ventilation per Code / Design
- Increased Ventilation Above Code
- Increased Ventilation Control
- Building and Space Pressure
- Flushing Between Occupied Periods
- Equipped Outdoor Air
- Upgrading and Improving Filtration
- Filter Single Pass Efficiency Particle Size Expectations
- Energy Service Considerations
- Extended Air Recirculation
- Emergency Recovery Ventilation Systems Operation Considerations
- HVAC Systems
- Domestic Water & Plumbing Systems
- Mechanical Scheduling
- Shutdown & Building Temporary-FAQ
- System Manual
- Respecting During Epidemic Conditions in Place

Post-Epidemic Conditions in Place (P-E-CIP)

- P-E-CIP: Prior to Occupancy
- P-E-CIP: Operational Considerations once Occupied
- P-E-CIP: Ventilation
- P-E-CIP: Filtration
- P-E-CIP: Building Maintenance Program
- P-E-CIP: Systems Manual

Additional Information

- Acknowledgments
- References
- Disclaimer

Information in this document is provided as a service to the public. While every effort is made to provide accurate and reliable information, this is advisory, is provided for informational purposes only, and may represent only one person's view. They are not intended and should not be relied upon as official statements of ASHRAE.

Assessment, planning, and implementation

Builds on ASHRAE Epidemic Task Force Building Readiness guidance

Applies commissioning practices to infection risk mitigation systems

Requirements for developing the Building Readiness Plan

- optional for dwellings

Assessment of existing V_{EACI} to determine need for additional controls

Supporting information

- Tracer particle test procedure for determining V_{EACI} in-place (appendix)
- Checklists for assessment and commissioning (appendix)
- Building Readiness Plan template (appendix)
- Equivalent clean air calculator (download at ashrae.org/241-2023)
- Guidance on assessing energy recovery ventilators (download)
- Guidance on preventing re-entry of contaminated air (download)

13

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Operations

(Requirements are optional for dwellings)

BRP on site, accessible, current

Essential supplies stocked

Operating modes defined:

- Normal - occupied/unoccupied
- IRMM - occupied/unoccupied
- Temporary shutdown

Temperature and humidity - maintain design set points when occupied

Operating schedules

- On for all occupied hours
- No on-off control of HVAC fans

Flushing not required between occupancy periods

Operator training

Occupant communication

14

14

Maintenance

Maintenance tasks and frequencies for all occupancies and system types follow ASHRAE/ACCA Standard 180 plus additional requirements →

Frequency of some checks increased for IRMM

Table 9-2 Minimum Maintenance Activity and Frequency for Additional Engineering Controls and Associated Components While in Use

Engineering Control	Inspection/Maintenance Task	Frequency
In-room air cleaners	Verify unit is in appropriate location and operating as intended per the <i>BRP</i> . Confirm that the air cleaner is operating at the speed or setting assumed in the $V_{EC,At}$ calculation. Maintain systems and equipment and verify performance per manufacturer's instructions. Visually inspect intake for debris and clean as necessary.	Monthly
Ultraviolet (UV) germicidal irradiation	Maintain systems and verify performance and safety per manufacturer's instructions and in accordance with ANSI/IES RP-44-21 ¹¹ and ANSI/IES RP-27.1.22 ²⁰ or equivalent. Adjust, clean, and replace equipment as needed.	Assess quarterly or per manufacturer's recommended interval
All air cleaning systems and equipment (including in-room, in-duct, and UV air cleaners)	Maintain systems and equipment and verify performance per manufacturer's instructions. Adjust, clean, and replace equipment as needed. If equipment cannot be repaired, remove equipment from service and use a substitute engineering control to maintain $V_{EC,At}$ in occupied space.	Assess quarterly or per manufacturer's recommended interval
Separation space	The designated temporary separation areas shall be tested for negative pressure whenever an infected individual is present.	As used

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15

Thank You

QUESTIONS

16

Can the Wells-Riley model universally assess airborne pathogen infection risk?

Dr Benjamin Jones
Associate Professor
University of Nottingham

Dr Chris Iddon

Prof. Max Sherman



1

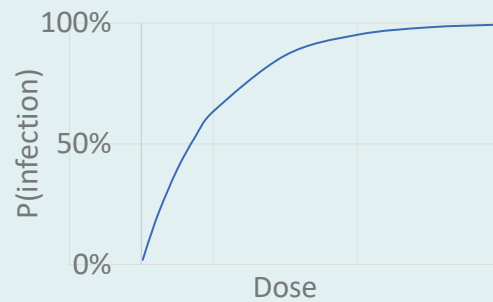
NO



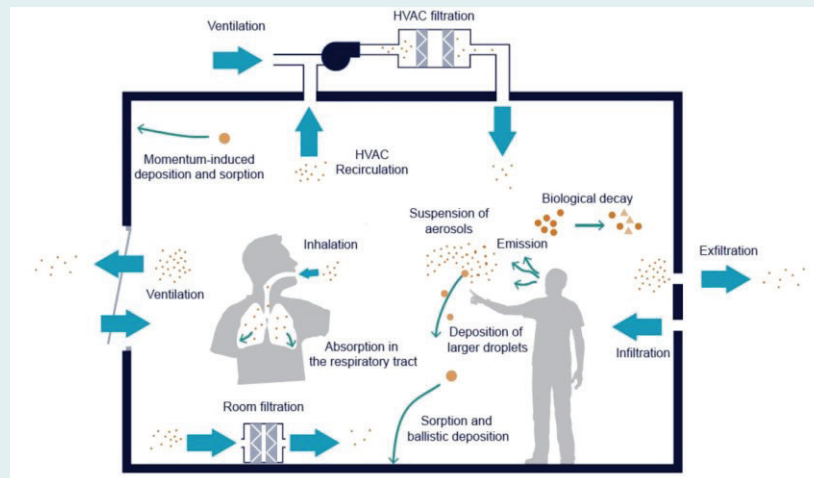
2

Wells-Riley model

$$P(\text{infection}) = 1 - \text{EXP}(-n)$$

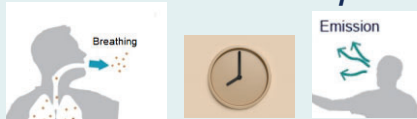


Virus dynamics

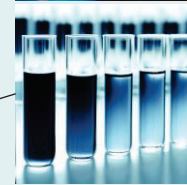
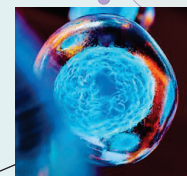
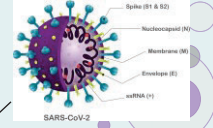
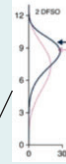


Terms

$$n = \frac{-QDq}{\phi V}$$



Emission

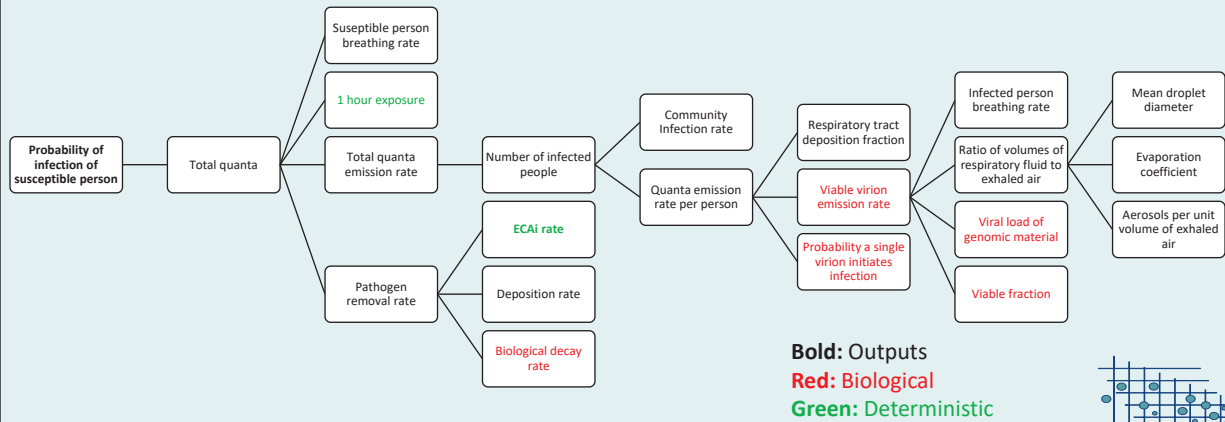


$$q \equiv kG/K$$

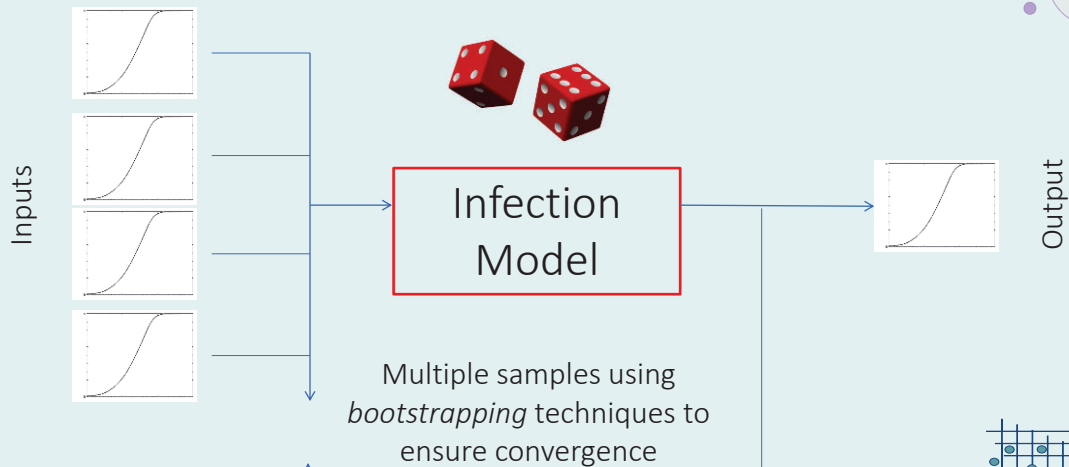
$$G \equiv QV_{drop}^*Lv$$

$$V_{drop}^* = \frac{\pi}{6} (\bar{d}E)^3 C_{drop}$$

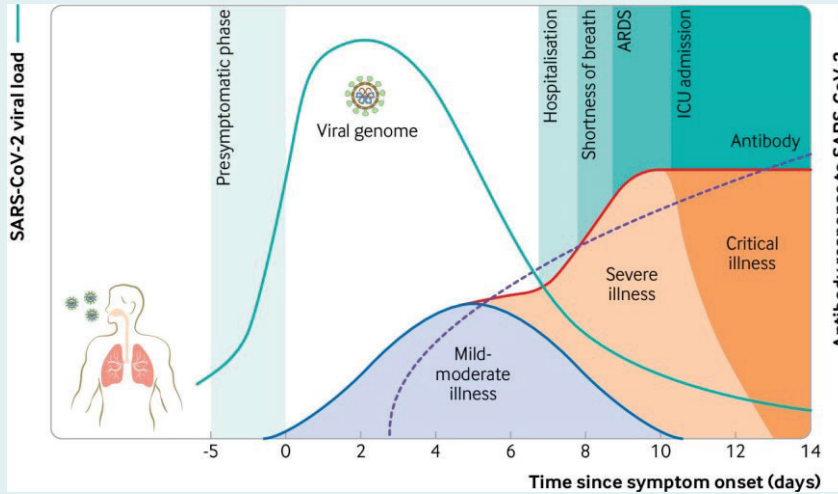
The system



Variance using Monte Carlo



Viral load

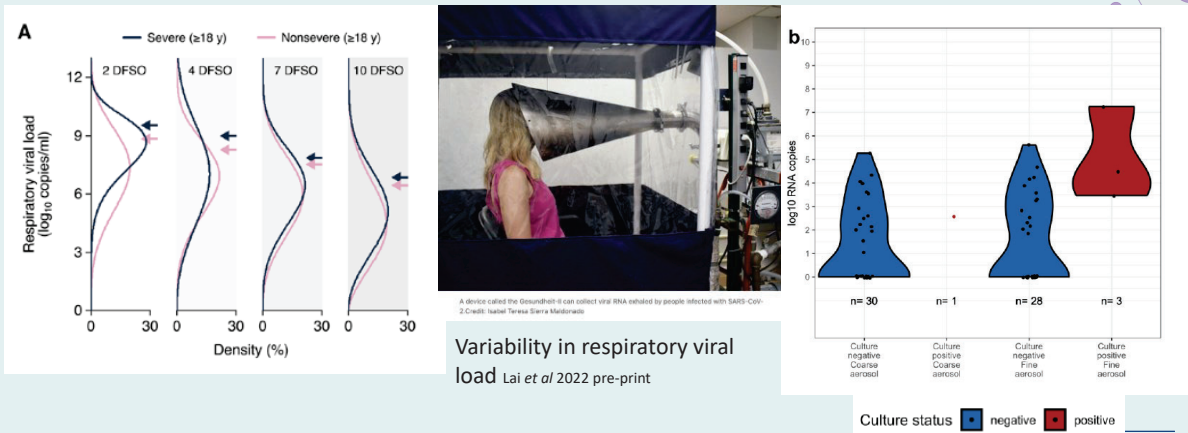


AIRBDS

Cevik, M. *et al.* (2020) 'Virology, transmission, and pathogenesis of SARS-CoV-2', *BMJ (Clinical research ed.)*, 371, p. m3862. doi: 10.1136/bmj.m3862

9

Viral load



AIRBDS

Chen *et al* (2021) SARS-CoV-2 shedding dynamics across the respiratory tract, sex, and disease severity for adult and pediatric COVID-19 <https://doi.org/10.7554/eLife.70458>

10

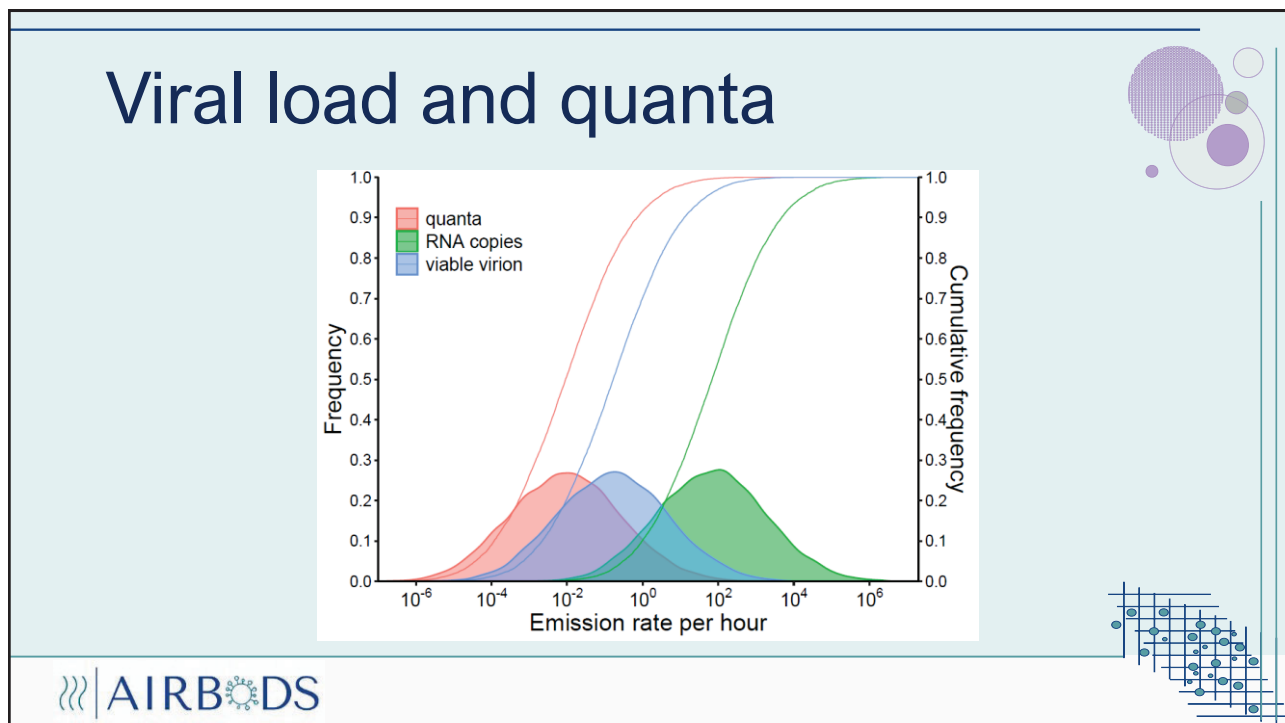
Table 1: Uncertainty in input parameters

Variable	Values	Source
Breathing rate, Q ($\text{m}^3 \text{h}^{-1}$)	LN(0.56, 0.056)	(Adams 1993)
Respiratory activity, <i>breathing: talking</i> (%)	75:23	(Morawska <i>et al.</i> , 2009; Iddon <i>et al.</i> 2022)
Aerosol concentration in exhaled air, C_{drop} (respiratory particles m^{-3})	LN(1.54×10^5 , 1.54×10^4)	(Morawska <i>et al.</i> , 2009; Iddon <i>et al.</i> 2022)
Mean aerosol diameter, \bar{d} (m)	LN(1.91×10^{-6} , 1.91×10^{-7})	(Morawska <i>et al.</i> , 2009)
Aerosol evaporation factor, E	B(2.0, 5.0) [2.0, 5.0]	(Nicas <i>et al.</i> 2005)
Viral load, L (\log_{10} RNA copies ml^{-1})	N(7.0, 1.4)	(Chen <i>et al.</i> 2021)
Viable fraction, v	B(2.0, 5.0) [10^{-4} , 10^{-2}]	(Killingley <i>et al.</i> 2022)
Respiratory tract absorption fraction, k	U(0.43, 0.65)	(Darquenne 2012)
Dose constant, K	U(5, 15)	(Killingley <i>et al.</i> 2022)
Number of infected people, j	1	
Number of occupants	50	
Space volume, V (m^3)	1350	
Exposure duration, D (h)	8	
Outside airflow rate (1 s^{-1} per person)	10	
Outside air change rate, ψ (h^{-1})	1.33	
Biological decay rate, λ (h^{-1})	LN(0.63, 0.43)	(Van Doremalen <i>et al.</i> 2020)
Surface deposition rate, γ (h^{-1})	U(0.42, 0.61)	(Thatcher <i>et al.</i> 2002)

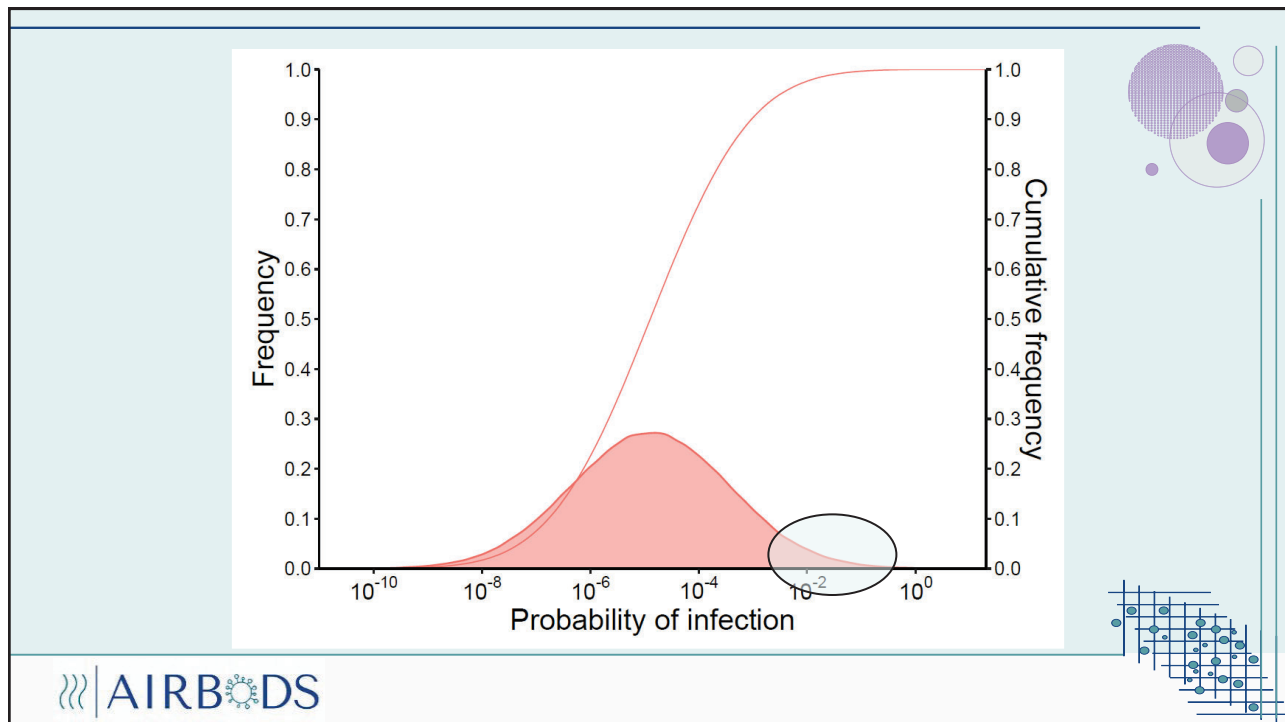
$N(\mu, \sigma)$, normal(mean, standard deviation); LN(μ, σ), log-normal; U(max, min), uniform; B(α, β) [min, max], beta.

Note that L needs to be converted into RNA copies per m^3 by multiplying by 10^6 .

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Problems

- Using outbreak data doesn't account for key factors that introduce significant bias into estimations
 - Number of index cases
 - Transmission pathways
 - Time varying virus dynamics
 - Inter-personal emission and immunity differences
 - Oral activity and breathing rate
- Personal quantum emission rate varies of time
- Population of individual emission rates has a broad distribution
- Do not understand the true underlying distributions
- Uncertainty is high so must be accounted for because data not enough to reduce it, unless it is cancelled

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A solution

$$P(\text{infection}) = 1 - \text{EXP}(-n)$$

When want to know when the $P(\text{infection})$ is low when

$$P(\text{infection}) \approx n$$

A simple first order % reduction in risk

$$REI = 1 - \left(\frac{\phi_{pre} N_{post}}{\phi_{post} N_{pre}} \right)$$

The End



FLOW DYNAMICS OF HUMAN COUGH AND MEASURING TECHNIQUES: A REVIEW

CHEN ZHANG*, PETER NIELSEN, SIMON MADSEN, LI LIU, CHUNWEN XU, ZHENGTAO AI
DEPARTMENT OF THE BUILT ENVIRONMENT, AALBORG UNIVERSITY

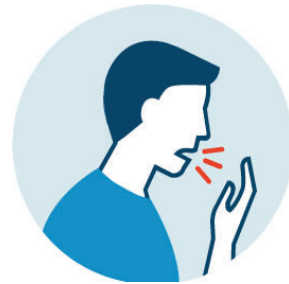


1

Background

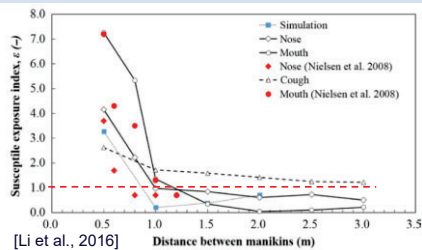
Why cough is important?

- Symptom of many respiratory infections and diseases, such as influenza, tuberculosis, COVID-19
- Critical vector in droplet and airborne transmission

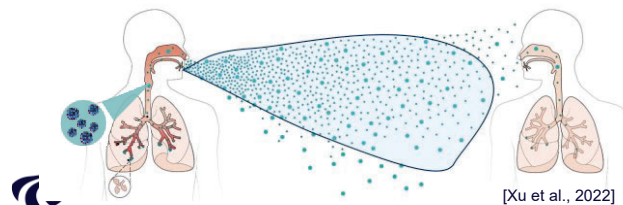


'2 meters (6-feet) rule' might not apply to situations involving increased exhalation, e.g. coughing

- Centers for Disease Control and Prevention (CDC)



[Li et al., 2016]



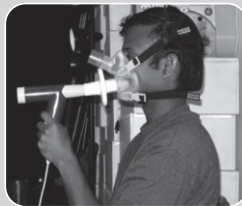
2

Literature review –Flow dynamics of human cough

Reference	Technique	Subjects	Cough Duration [s]	Cough peak velocity [m/s] or flow rate [l/s]	Peak velocity time [ms]	Cough Volume [l]	Flow direction	Mouth geometry
Mahajan et al. 1994						0.5-5		
Khan et al. 2004			0					
Zhu 2006	PIV	3 (3 M)	0			0.8-2.2 Average 1.4		
Gupta et al. 2009	Spirometer	25 (13 M and 12 F)	0.35-0.75	M: 0.3 L/s F: 1.6-6 L/s	M: 0.7-0.9 F: 57-110	M: 0.4-1.6 F: 0.25-1.25	$\theta_1=15\pm 5^\circ$; $\theta_2=40\pm 4^\circ$ (vertical spread angle 25°)	M: 400 ± 95 mm ² F: 337 ± 140 mm ²
Chao et al. 2009	PIV	11 (3 M and 8 F)		M: 13.2 m/s F: 10.2 m/s Average 11.7 m/s				
Lindsay et al. 2010	Spirometer	58 (47 influenza-positive and 11 influenza-negative)	0.9	Influenza-positive 7.1 l/s Influenza-negative 7.6 l/s		Influenza-positive 2.7 Influenza-negative 3.1		
Vansciver et al. 2011	PIV	29 (10 M and 19 F)	0.55	1.15-28.8 m/s Average 10.2 m/s				706 mm ² (D= 30 mm)
Tang et al. 2012	schlieren imaging technique	20 (10 M and 10 F)	0.2-0.35	M: 3.2-14 m/s F: 2.2-5.0 m/s;				
Kwon et al. 2012	PIV	26 (17 M and 9 F)		M: 15.3 m/s F 10.6 m/s			Vertical spread angle M: 38° F: 32°	
Wang et al. 2020	PIV	4		15 m/s				
Han et al. 2021	PIV	10 (5 M and 5 F)	0.52-0.56	M: 6.4-18.6 m/s F: 5.0-15.7 m/s	M: 8-35 F: 8-39		Vertical spread angle M: 15.3° F: 15.6° Horizontal spread angle M: 13.3° F: 14.2°	M: 47 mm F: 39.4 mm (Mouth width)

3

Measuring techniques



[Gupta et al., 2009]



[NIST]



[Han et al., 2021]

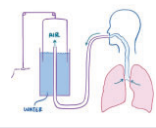
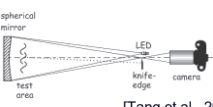
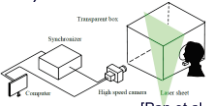
Spirometer

Schlieren imaging technique

Particle image velocimetry (PIV)

4

Measuring techniques

Techniques	Principle	Advantages	Disadvantages
 <p>[Osmosis]</p>	Poiseuille's equation $Q = f(\Delta p)$	<ul style="list-style-type: none"> Widely available Easy and low cost 	<ul style="list-style-type: none"> Need a mouthpiece Disturb exhaled flow development
 <p>[Tang et al., 2011]</p>	Visualize variations in refractive index caused by density gradients	<ul style="list-style-type: none"> Visualizes flow Non-intrusive technique No safety concern 	<ul style="list-style-type: none"> No direct quantitative measurements of flow velocity or flow rate Require temperature difference between exhaled flow and surrounding
 <p>[Pan et al., 2023]</p>	Captures flow velocity by tracking the motion of tracer particles in a fluid	<ul style="list-style-type: none"> Visualizes flow Quantitative measurement High Spatial Resolution 	<ul style="list-style-type: none"> Need tracer particles High intensity laser light cause safety concern Safety protections might constrain and disturb the exhaled flow

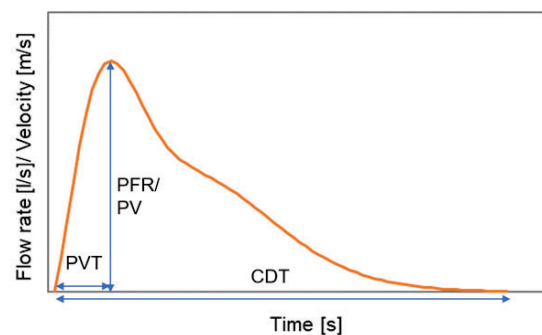
5

Flow characteristics

- Peak flow rate (PFR) or peak velocity (PV)
1.15 -28.8 m/s, mean 10~11 m/s
- Peak velocity time (PVT)
10-100 ms, mean 50 ms
- Cough duration time (CDT)
200 - 900 ms, mean 500 ms
- Cough expired volume (CEV)
0.5-5 l, mean 1.4 l

Large deviations on cough profile, due to:

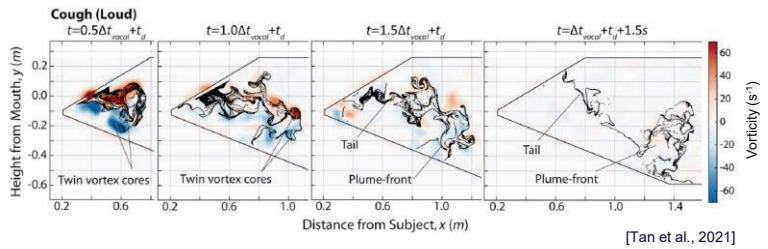
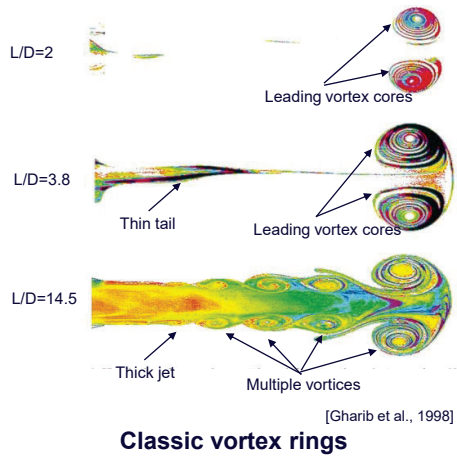
- Gender
- Body mass
- Health
- Measuring techniques



Gamma-probability distribution function

6

Vortex Structure

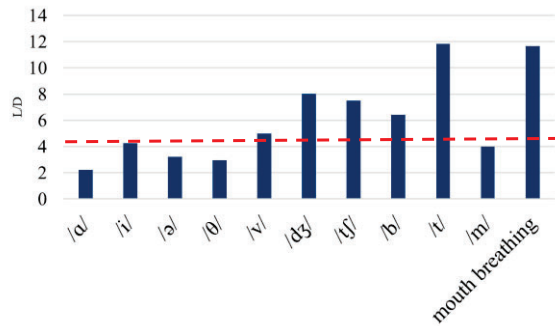
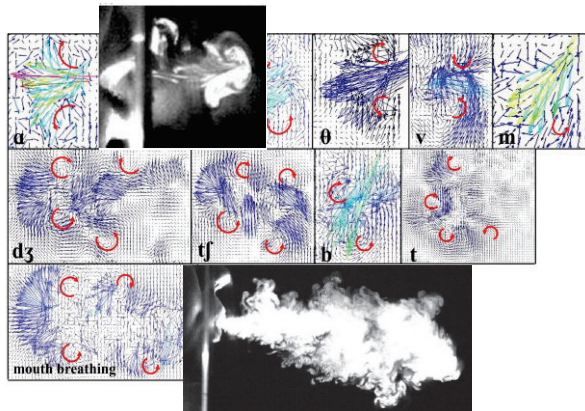


$$\text{Stroke ratio}(L/D) = \frac{\text{propagation distance}}{\text{mouth diameter}} > 100$$



7

Vortex Structure



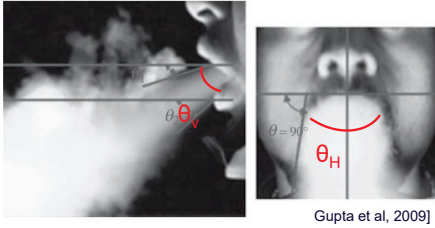
[Pan et al., 2023]



8

Flow direction

- Flow visualization



Gupta et al, 2009]

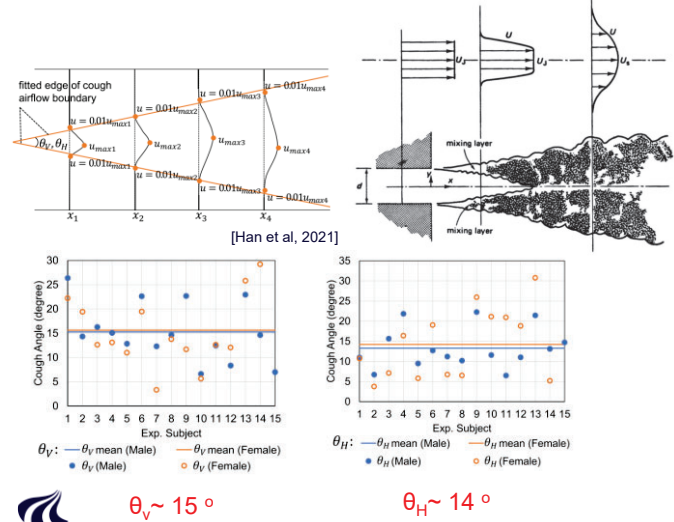
$$\theta_1 = 15^\circ \pm 5^\circ$$

$$\theta_2 = 40^\circ \pm 4^\circ$$

$$\theta_v \sim 25^\circ$$

$$\theta_H \sim 0^\circ$$

- Velocity boundary layer



9

Cough simulator

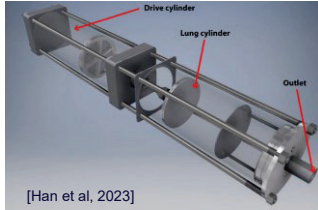
Author	Year	Peak flow	Cough volume [l]	Duration [s]	Mouth opening diameter [mm]	Cough medium	Equipped with thermal manikin	Refer to human subject measurement
Bolashikov et al.	2012	10 l/s	2.5	0.5	21	CO2, 100%	Yes	-
Lindsley et al.	2013	11.4 l/s	4.2		21	Cell culture medium	No	Lindsley et al. 2010
Liu et al.	2014	6.08 m/s		1	24	Particles (0.77 μ m, 2.5 μ m and 7 μ m)	No	-
Pantelic et al.	2015		1.4	0.5		Water 90% and glycerin 10%	Yes	Zhu et al. 2006
Cao et al.	2015	6 m/s	1.4	1		Particles (0.77 μ m)	Yes	Zhu et al., 2006; Gupta et al., 2009
Licina et al.	2015	10 m/s				Mixture of water (94%) and glycerin (6%)	No	Zhu et al. 2006
Hall et al.	2022		4.2		40	Aqueous solution of 0.1% fluorescein	No	Lindsley et al. 2010
Thacher et al.	2022		1.23±0.09		25.5	Fluorescent polyethylene	Yes	Gupta et al., 2009



10

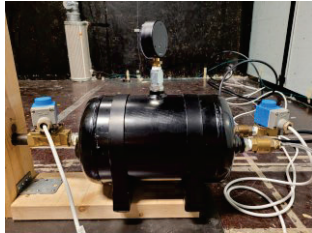
Cough simulator

- Drive cylinder

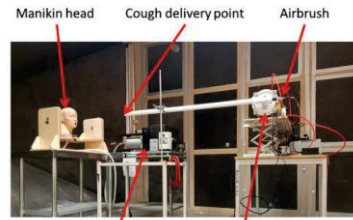


[Han et al, 2023]

- Pressurized gas tank



- Without manikin



Breathing machine Cough simulator

[Han et al, 2023]

- With manikin



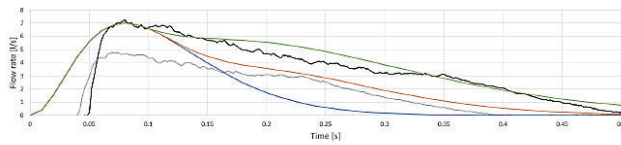
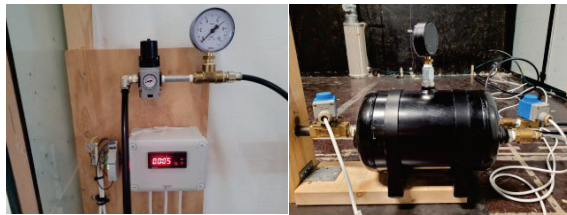
[PT Teknik]



[Nielsen et al, 2008]

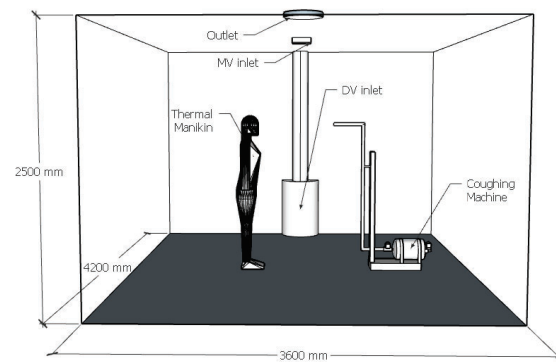
11

Cough simulator



— Cough Profile 1
 — Gupta (2009) $a_2=1.68$ $b_2=6.83$ $c_2=0.31$ — Gupta (2009) $a_2=10.0$ $b_2=5.0$ $c_2=0.45$
 — Gupta (2009) $a_2=8.0$ $b_2=3.2$ $c_2=0.82$

Cough flow profile generated by the cough simulator compared with profiles from literature [Gupta et al., 2009].



12

Conclusions

- The airflow characteristic of human cough is normally measured by three techniques, including the **Schlieren imaging technique, spirometer** and **PIV**. The uncertainties of measuring techniques require further investigation.
- Cough profile can be described by a **gamma-probability distribution function**.
- Large deviations are due to subjects' **gender, age, body mass surface**, and **measuring techniques**.
- It is difficult to create a '**standard**' cough profile.
- Cough airflows can be described as an **impulsive jet**, vortex ring structures **prevent the rapid dilution of aerosol plume**.
- Development of **realistic cough simulator** is needed.



13

THANK YOU FOR YOUR ATTENTION!



14

EVALUATING THE IMPACT OF AIR CLEANING AND VENTILATION OF AIRBORNE PATHOGENS AND HUMAN BIO-EFFLUENTS AT TWO PRIMARY SCHOOLS IN BELGIUM

Ghent University

VITO

Liantis

Jessa Hospital Clinical Lab

Antwerp University Hospital

Klaas De Jonge, Jelle Laverge and Arnold Janssens

Marianne Stranger, Sarah L. Paralovo, Maarten Spruyt, Borislav Lazarov

Tom Geens

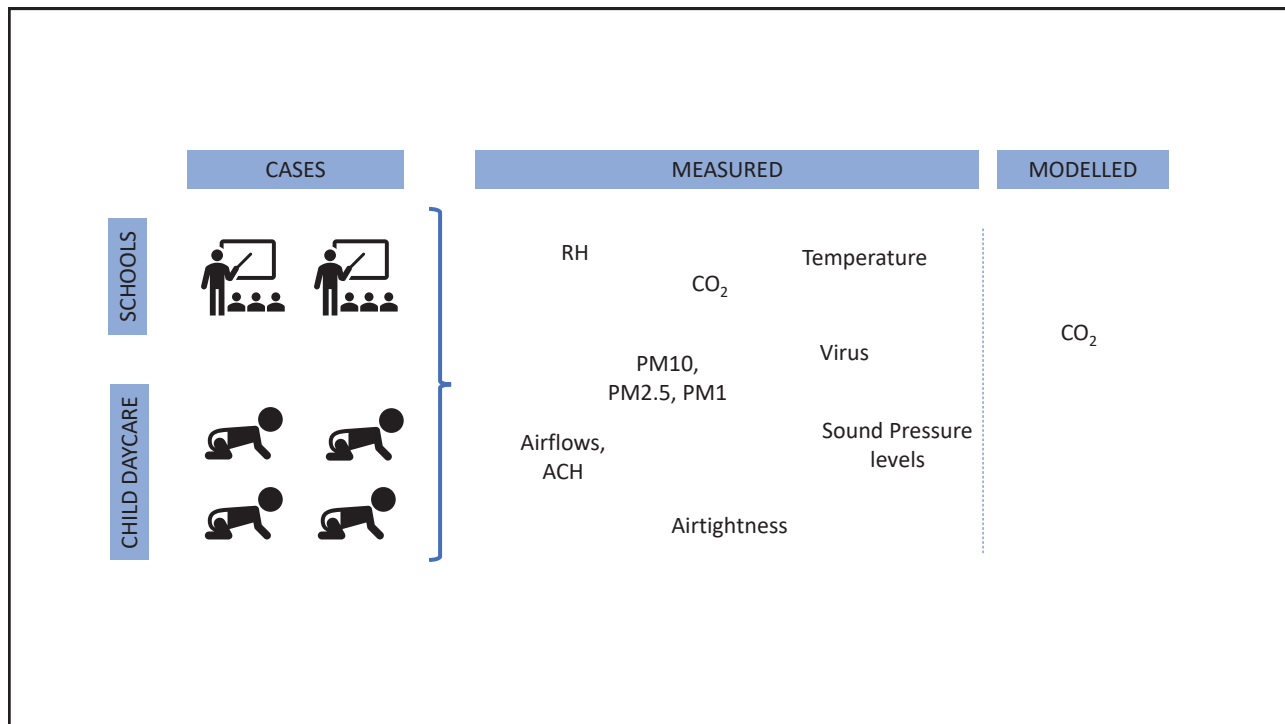
Reinoud Cartuyvels

Koen Van den Driessche

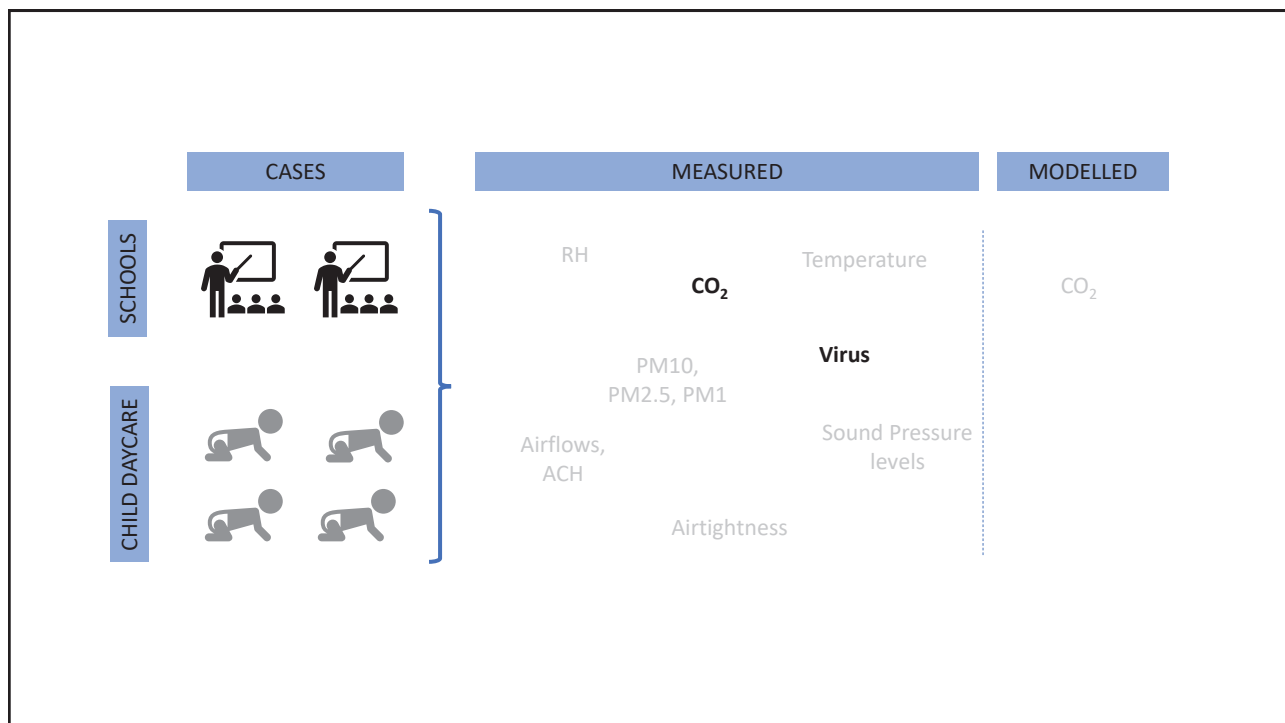
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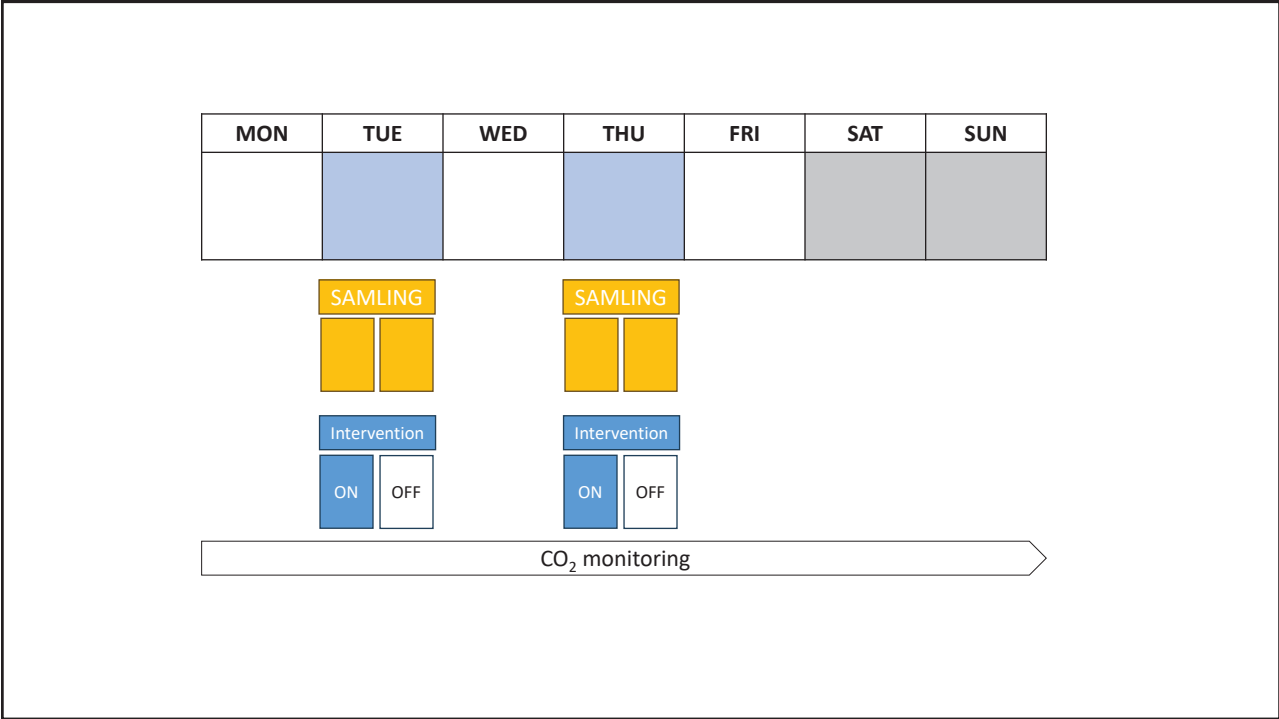
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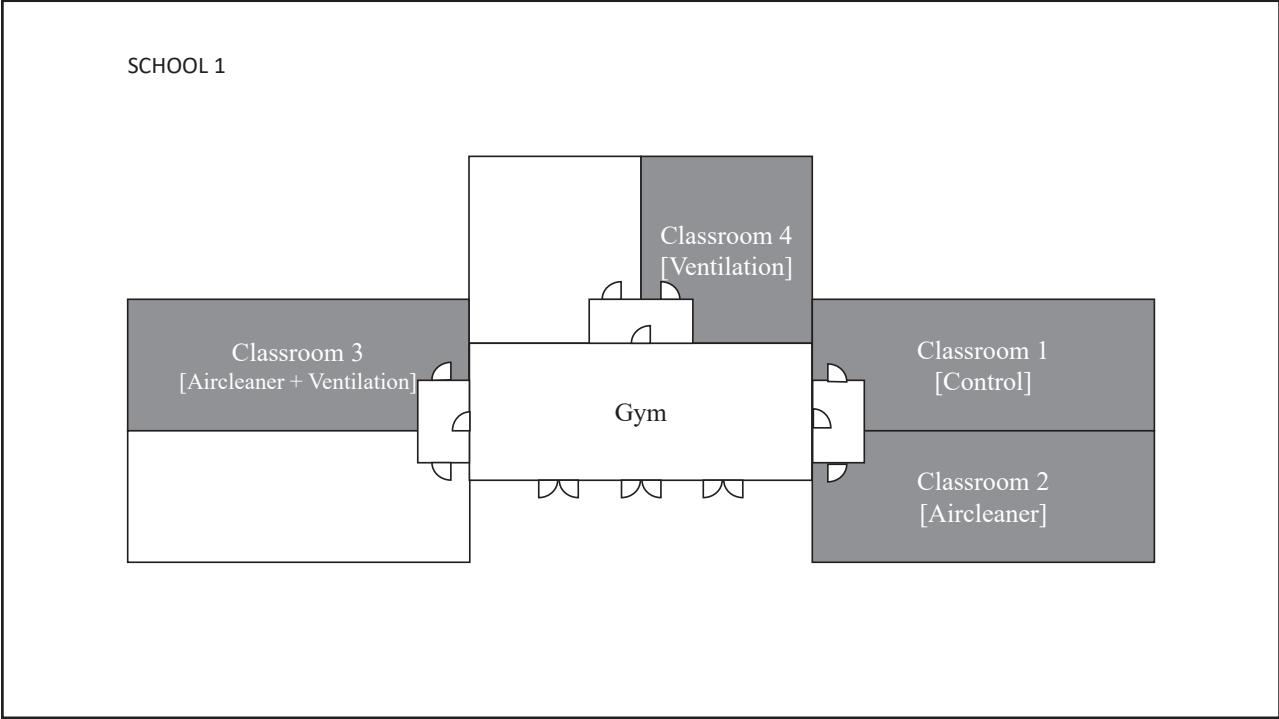
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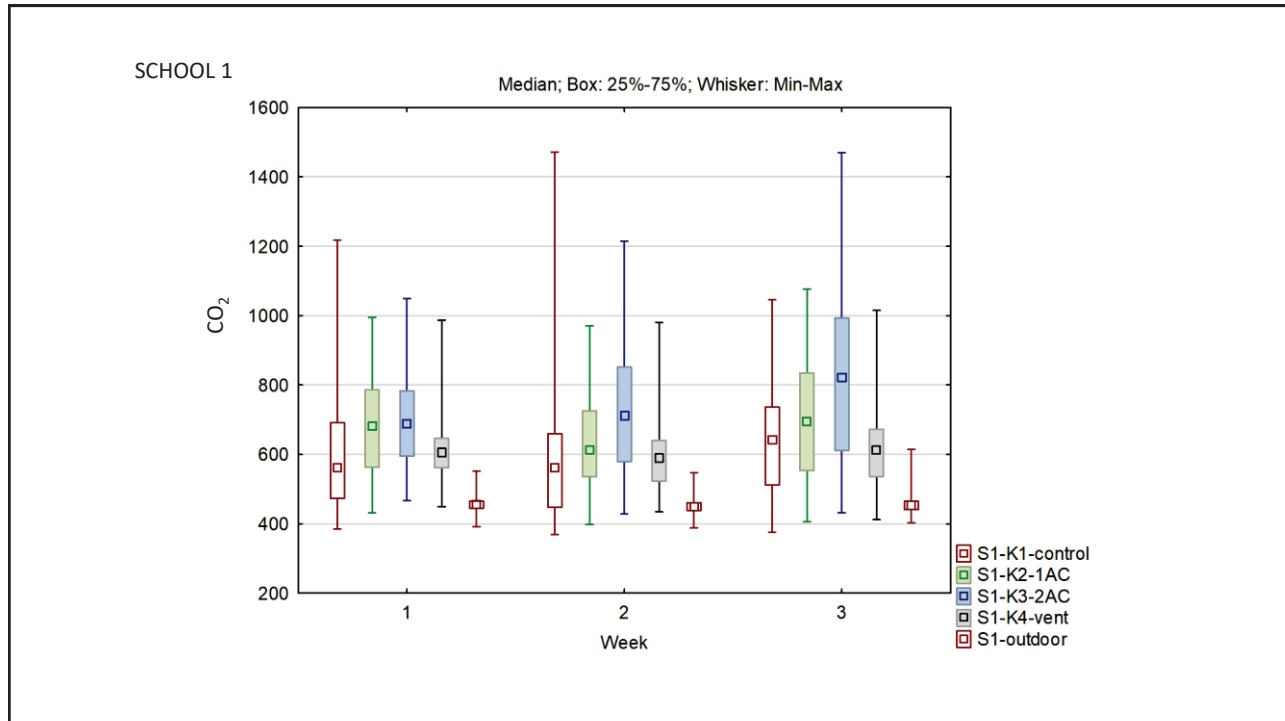
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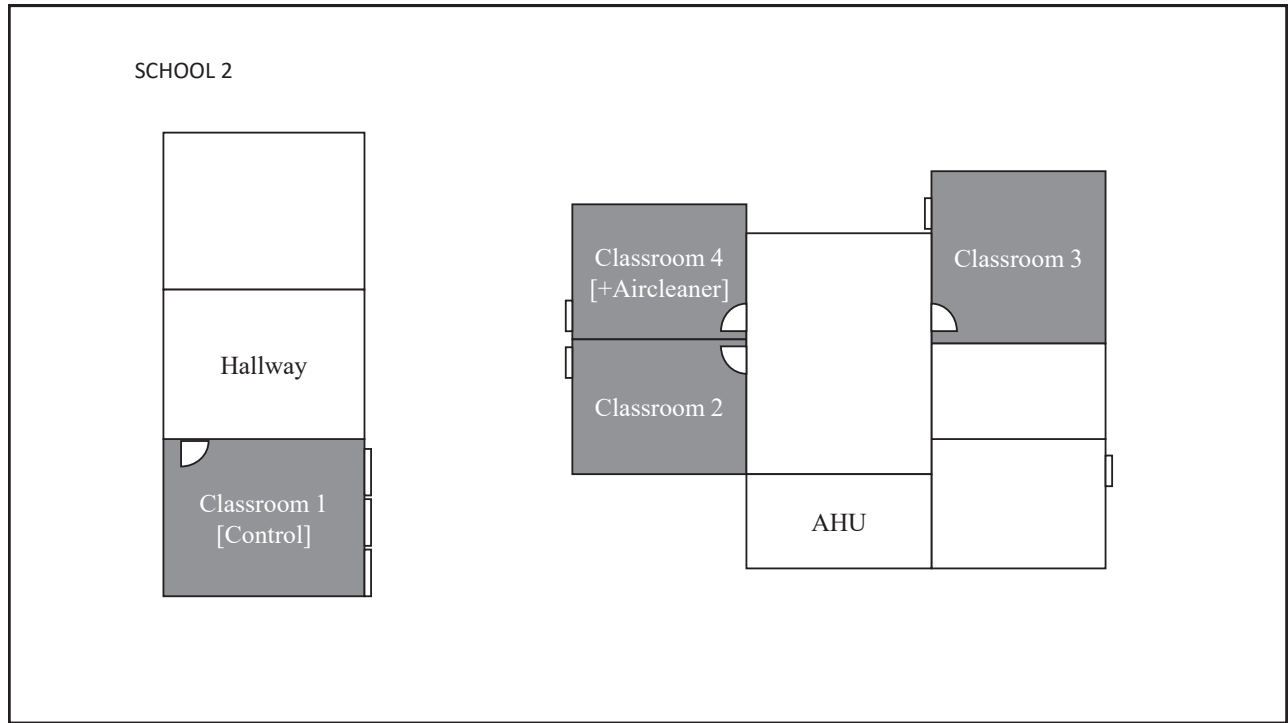


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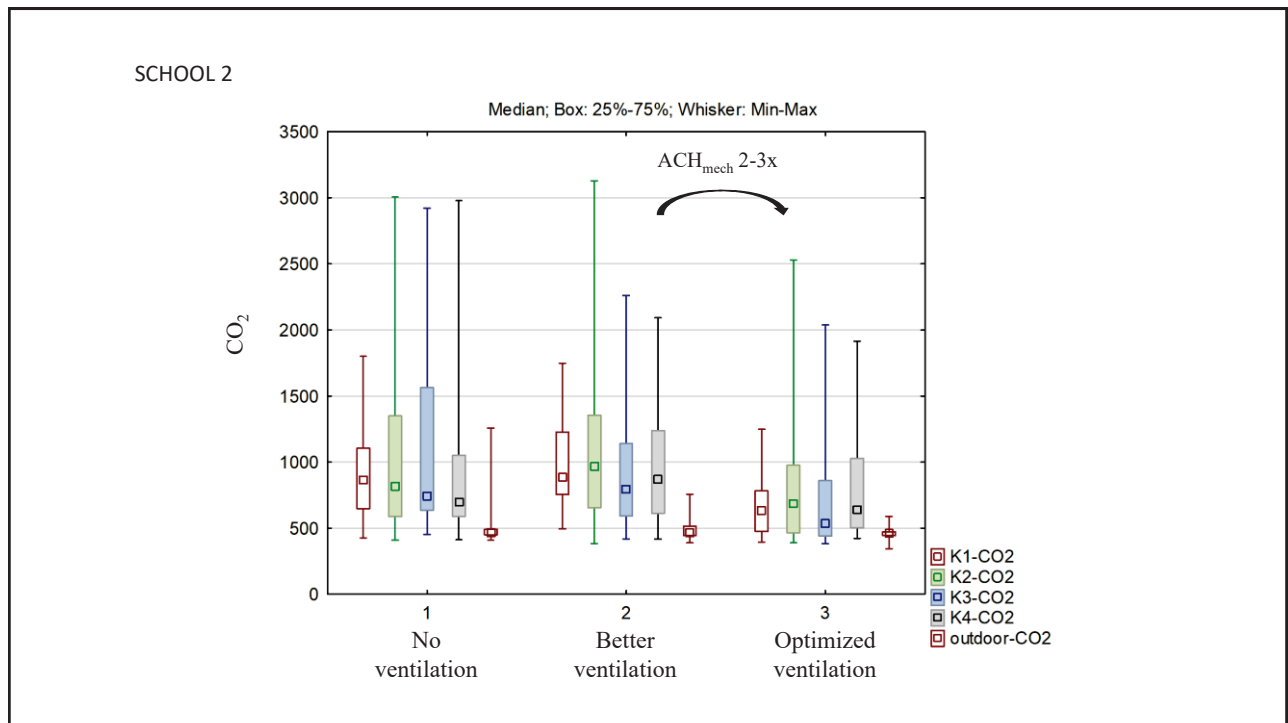
SCHOOL 1

SARS-CoV-2		Tuesday		Thursday	
		Intervention ON	Intervention OFF	Intervention ON	Intervention OFF
Week 1	Class. 1 - Control			Limit (ct 39.3)	
	Class 2 - Air cleaner			Neg.	Neg.
	Class 3 - Air clean + Vent.			Limit (ct 36.9)	Limit (ct 37.0)
	Class. 4 - Vent.			Neg.	
Week 2	Class. 1 - Control	Neg.		Neg.	
	Class 2 - Air cleaner	Neg.		Neg.	Neg.
	Class 3 - Air clean + Vent.	Neg.	Limit (ct 38.7)	Neg.	
	Class. 4 - Vent.	Neg.	Neg.	Neg.	
Week 3	Class. 1 - Control	Neg.		Neg.	
	Class 2 - Air cleaner	Neg.		Neg.	Neg.
	Class 3 - Air clean + Vent.	Limit (ct 39.4)	Neg.	Neg.	Limit (ct 38.3)
	Class. 4 - Vent.	Neg.	Neg.	Neg.	

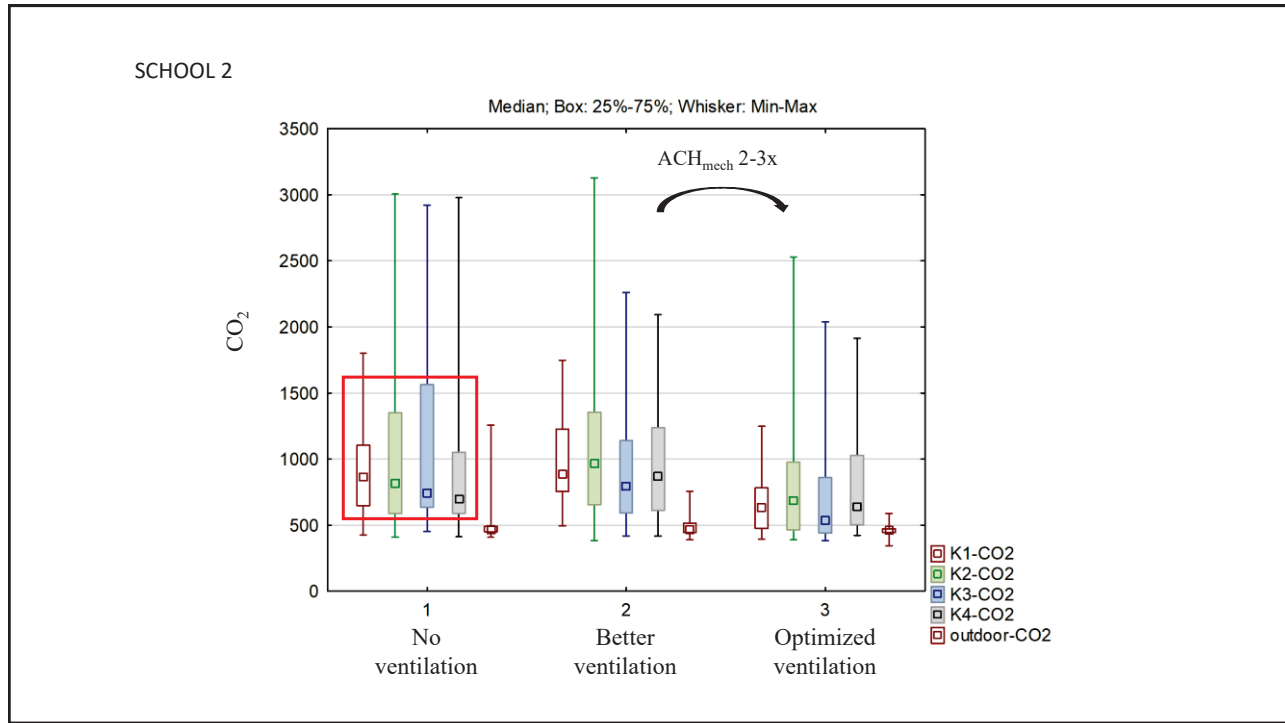
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SCHOOL 2

SARS-CoV-2

		Tuesday		Thursday		
		Intervention ON	Intervention OFF	Intervention ON	Intervention OFF	
Better ventilation	Week 1	Class. 1 - Control		Limit (ct 36.0)	Limit (ct 36.7)	
		Class 2 - Mech. Vent.	Limit (ct 36.7)	Limit (ct 38.3)	Neg.	Limit (ct 38.9)
		Class 3 - Mech. Vent.		Neg.	Neg.	Limit (ct 38.5)
		Class 4 - Vent.+Air cleaner				Limit (ct 35.4)
	Week 2	Class. 1 - Control	Limit (ct 38.9)		Limit (ct 39.7)	
		Class 2 - Mech. Vent.	Neg.	Limit (ct 38.7)	POS. (ct 33.0)	
		Class 3 - Mech. Vent.	Limit (ct 40.5)		Limit (ct 37.9)	Limit (ct 36.5)
		Class 4 - Vent.+Air cleaner	Limit (ct 40.7)	Neg.	Limit (ct 38.9)	Neg.
	Week 3	Class. 1 - Control	Neg.		Limit (ct 40.4)	
		Class 2 - Mech. Vent.	Limit (ct 39.3)	Limit (ct 39.2)	Limit (ct 38.3)	Neg.
		Class 3 - Mech. Vent.	Limit (ct 38.5)		Limit (ct 37.4)	Limit (ct 38.2)
		Class 4 - Vent.+Air cleaner	Limit (ct 38.2)	Neg.	Limit (ct 40.0)	
Maintenance and commissioning to AHU						
Optimized ventilation	Week 4	Class. 1 - Control	Limit (ct 39.1)			
		Class 2 - Mech. Vent.	Neg.			
		Class 3 - Mech. Vent.	Neg.	Neg.	School closed	
		Class 4 - Vent.+Air cleaner	Limit (ct 41.1)	Neg.		
	Week 5	Class. 1 - Control	Neg.		Limit (ct 37.0)	
		Class 2 - Mech. Vent.	Neg.		Limit (ct 40.0)	
		Class 3 - Mech. Vent.	Limit (ct 38.9)	Neg.	Limit (ct 38.2)	
		Class 4 - Vent.+Air cleaner	Limit (ct 39.0)	Limit (ct 37.6)	Neg.	Neg.
	Week 6	Class. 1 - Control			Limit (ct 39.5)	
		Class 2 - Mech. Vent.	Limit (ct 37.3)		Limit (ct 40.2)	Neg.
		Class 3 - Mech. Vent.	Neg.	Limit (ct 40.1)	Neg.	Limit (ct 40.0)
		Class 4 - Vent.+Air cleaner				

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CONCLUSIONS

- For both schools, **no measurable effect** of increased ventilation or air cleaning on the occurrence of SARS-CoV-2 could be **objectively observed** for these in-situ and in-use situations.
- **importance of proper commissioning and maintenance** to mechanical ventilation systems and show an overall better expected perceived indoor air quality when the ventilation system works properly
- **manual airing** through the opening of windows **can achieve the same level of expected perceived indoor air quality** if operated correctly

REVIEW OF INTERNATIONAL STANDARDS

DESCRIBING AIR CLEANER TEST METHODS

H. Scheipers, A. Janssens, J. Laverge
05/10/2023

1

Context

- Increasing use of air cleaners
- Multiple test standards used

2

Investigated standards



AHAM AC-1 (2020)
AHAM AC-4 (2022)
AHAM AC-5 (2022)



ASHRAE 52.2 (2017)
ASHRAE 145.1 (2015)
ASHRAE 145.2 (2016)
ASHRAE 185.1 (2020)
ASHRAE 185.2 (2020)



IEC 63086-1 (2020)



EN ISO 10121-3 (2022)
ISO 16000-36 (2021)



EN 16846-1 (2017)



NF B44-200 (2016)



3

3

Investigated standards



AHAM AC-1 (2020)
AHAM AC-4 (2022)
AHAM AC-5 (2022)



ASHRAE 52.2 (2017)
ASHRAE 145.1 (2015)
ASHRAE 145.2 (2016)
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ASHRAE 185.2 (2020)



IEC 63086-1 (2020)



EN ISO 10121-3 (2022)
ISO 16000-36 (2021)



EN 16846-1 (2017)



NF B44-200 (2016)



Standalone/portable air cleaners
In duct or air handling systems

4

4

Investigated standards



AHAM AC-1 (2020)
AHAM AC-4 (2022)
AHAM AC-5 (2022)



ASHRAE 52.2 (2017)
ASHRAE 145.1 (2015)
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IEC 63086-1 (2020)



EN ISO 10121-3 (2022)
ISO 16000-36 (2021)



EN 16846-1 (2017)



NF B44-200 (2016)



Specifies the used technology
Does not specify the used technology

5

5

Investigated characteristics

Technical characteristics

- Test method
- Definition of performance
- Duration of the test
- Test pollutants

Knowledge gaps and challenges

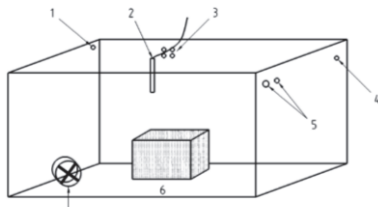


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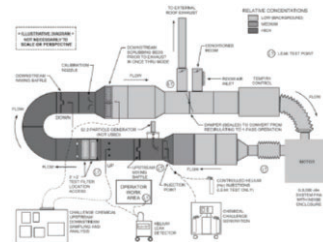
Test apparatus

Test chamber



NBN EN 16846-1 (2017)

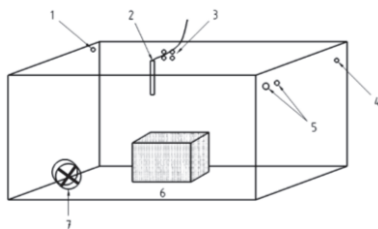
Test duct



ANSI/ASHRAE 145.2-2016

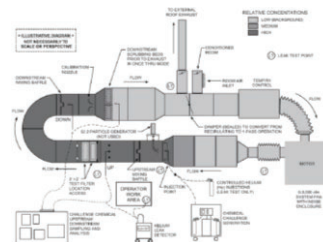
Definition of performance

Test chamber



NBN EN 16846-1 (2017)

Test duct



ANSI/ASHRAE 145.2-2016

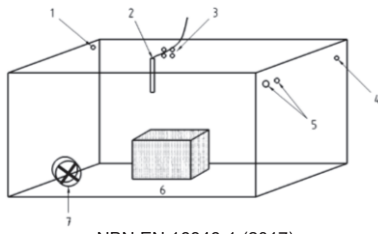
CADR

Clean Air Delivery Rate

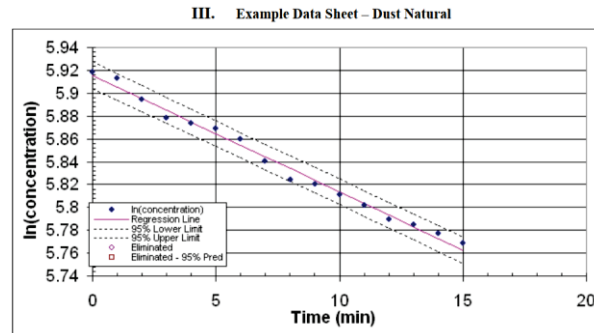
Single pass efficiency

Test apparatus

Test chamber



NBN EN 16846-1 (2017)



Decay rate (AHAM AC-1 (2020))

CADR

Clean Air Delivery Rate

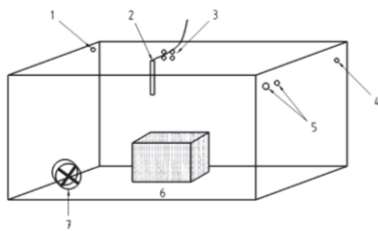


9

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Definition of performance

Test chamber



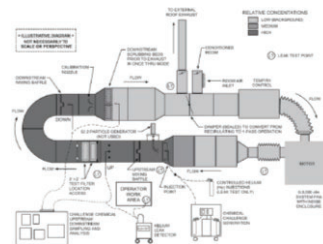
NBN EN 16846-1 (2017)

CADR

Clean Air Delivery Rate



Test duct



ANSI/ASHRAE 145.2-2016

Single pass efficiency

10

10

Investigated standards



AHAM AC-1 (2020)
AHAM AC-4 (2022)
AHAM AC-5 (2022)



ASHRAE 52.2 (2017)
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ASHRAE 185.2 (2020)



IEC 63086-1 (2020)



EN ISO 10121-3 (2022)
ISO 16000-36 (2021)



EN 16846-1 (2017)



NF B44-200 (2016)



Test chamber + CADR
Test duct + single-pass efficiency

11

11

Test pollutants

- VOCs, aerosols, (synthetic) dust, (acid) gases and microorganisms
- Limited scope of pollutants vs. broader range
- Different concentrations



12

12

Investigated characteristics

Technical characteristics

- Test method
- Definition of performance
- Duration of the test
- Test pollutants

Knowledge gaps and challenges



13

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Knowledge gaps and challenges

~~By-products~~

- Short duration
- Non-realistic test conditions
- Specific test pollutants
- Small test chamber
- ...



Real life performance?
Long-term performance?



14

14

Conclusion

- Main differences
 - Type of air cleaner being tested
 - Test apparatus used + definition of performance
 - Test pollutants
- Knowledge gaps

Further research

- Creation of a new test standard
 - Non-targeted analysis
 - Real life & long-term performance

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Rethinking different ventilation strategies in a post- pandemic era: a CFD assessment

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 alicia.murga@harbor.kobe-u.ac.jp

Kyushu University, Kazuhide Ito (ito@kyudai.jp)
 Riken/Kobe University, Makoto Tsubokura (tsubo@tiger.kobe-u.ac.jp)

43rd AIVC -11th TightVent & 9th venticool Conference. Ventilation, IEQ and health in sustainable buildings. Copenhagen, 2023

Aim

- Use breathing-zone volume ventilation by CFD (RANS) to create a focus on infection control:
 - purging effectiveness
 - thermal comfort

Methodology

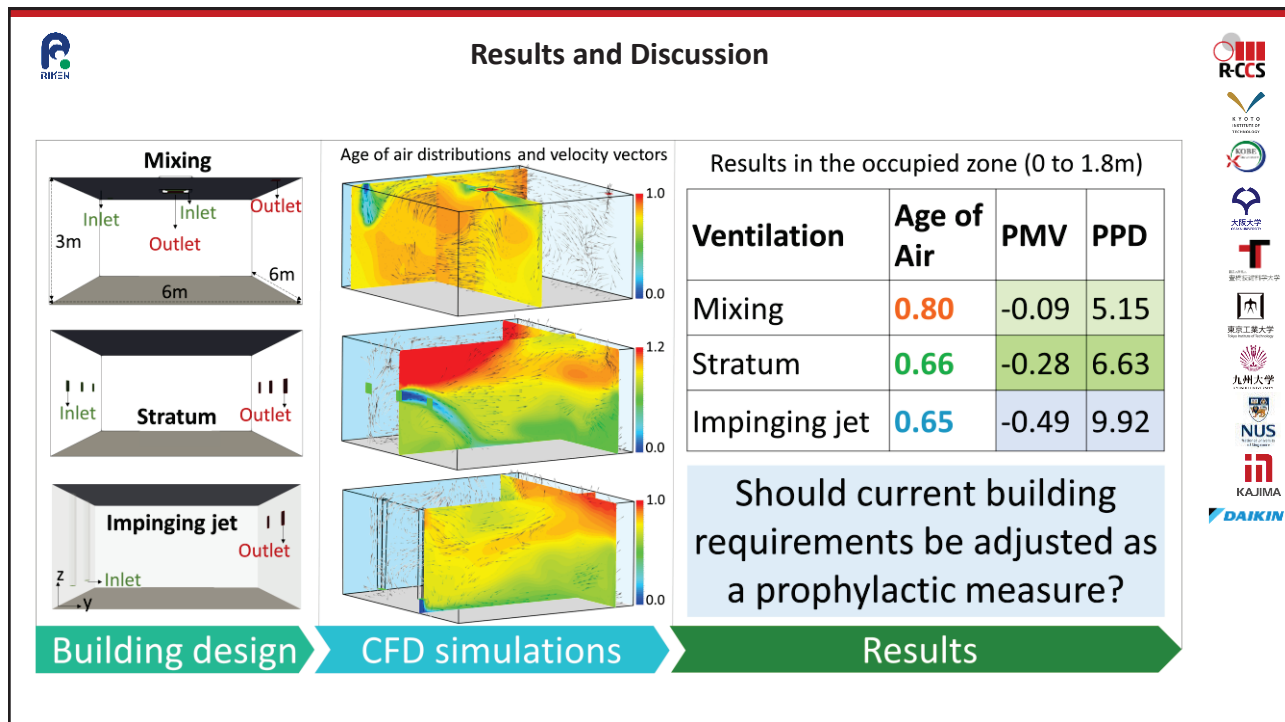
3 types of ventilation strategies:
 General-volume focused: **mixing**

Advanced:
 Breathing-zone focused: **stratum & impinging jet**

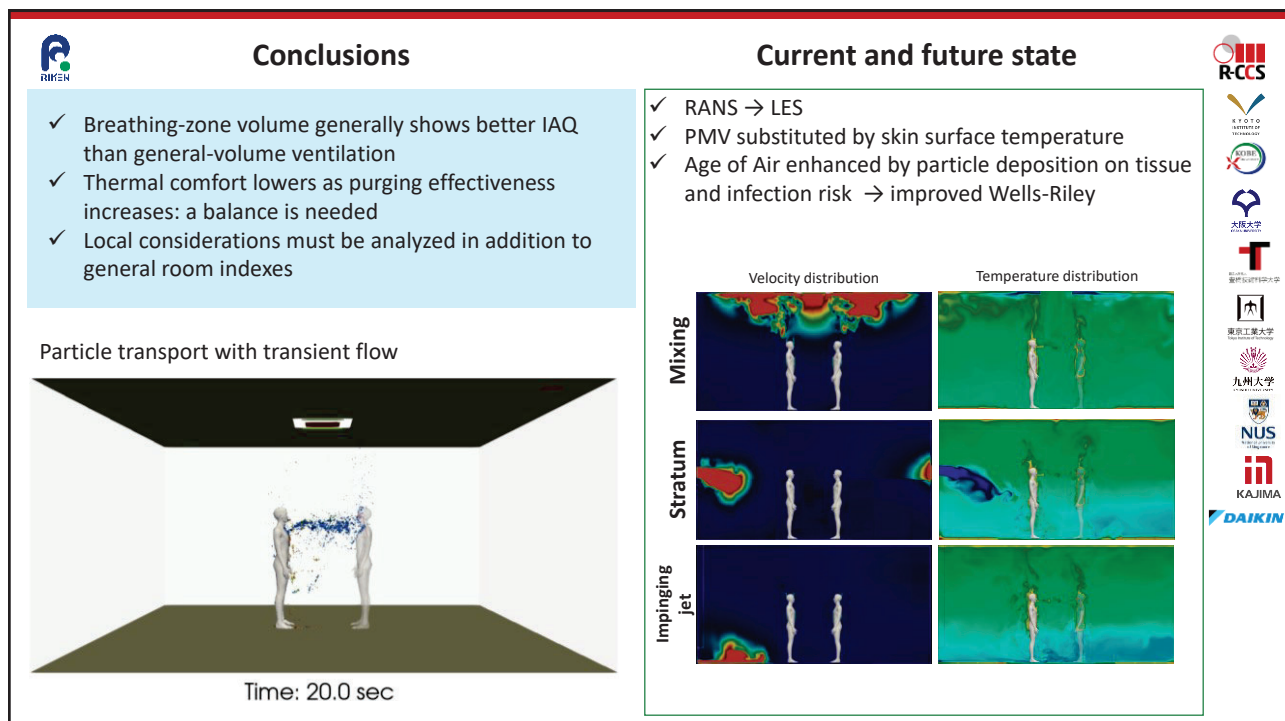
Airflow
 ASHRAE 62 $Q = (Q_p \times N) + (Oc \times A)$

Heat Load
 $h = Q\rho C_p \Delta T$

Age of Air
 Scale for ventilation efficiency 3 $SVE3 = \frac{C'_x(X)}{C_s}$
 $C_s = \frac{q}{Q}$



3



4



Acknowledgments

Fugaku computational resources were provided by the HPCI System Research Project (ID: hp210086) and “Program for Promoting Researches on the Supercomputer Fugaku” (MEXT) and used computational resources provided by RIKEN Center for Computational Science.

This research was partially supported by JST CREST “Creation of fundamental technologies by interdisciplinary research to coexist with infectious diseases including COVID-19” (Issue name: Development of the integrated risk assessment system for the viral droplet infection on a supercomputer and its social implementation), Grant Number JPMJCR20H7, Japan.

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Riken/Kobe University, Makoto Tsubokura (tsubo@tiger.kobe-u.ac.jp)



How the COVID Pandemic and the Energy Crisis Have Influenced Indoor Environmental Conditions in non-residential Buildings

Aurora Monge-Barrio¹, Ainhoa Arriazu-Ramos¹, María Fernández-Vigil¹, and Ana Sánchez-Ostiz¹
 1. School of Architecture, Universidad de Navarra, Spain. amongeb@unav.es



1

HOW the COVID Pandemic and the Energy Crisis Have Influenced Indoor Environmental Conditions in non-residential Buildings

STUDY BASED ON ANALYSIS OF MONITORING DATA, ENERGY CONSUMPTION AND SURVEYS

Two High Schools and a Museum **naturally ventilated**

Winter	Situation	IES_NV Monitoring data	IES_PC Monitoring data	MN Monitoring data
2019-20	COVID, schools closed at March 15th 2020	3-15/03/20	NA	12/19 - 2/20
2020-21	COVID Schools opened in Spain*	12/20 - 2/21	NA	12/20 - 2/21
2021-22	COVID Schools opened in Spain*	12/21 - 2/22	12/21 - 2/22	12/21 - 2/22
2022-23	Post Covid & Energy crisis	12/22 - 2/23	12/22 - 2/23	12/22 - 2/23



Aurora Monge-Barrio¹, Ainhoa Arriazu-Ramos¹, María Fernández-Vigil¹, and Ana Sánchez-Ostiz¹
 1. School of Architecture, Universidad de Navarra, Spain.



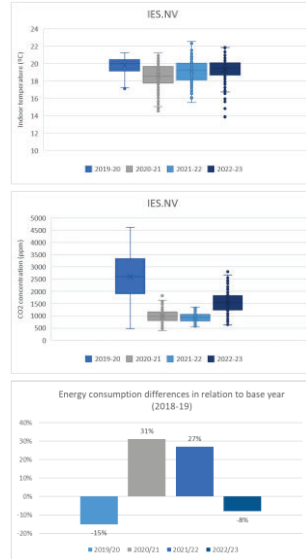
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HOW the COVID Pandemic and the Energy Crisis Have Influenced Indoor Environmental Conditions in non-residential Buildings

- ALL CLASSROOMS ARE MONITORED
- STAFF AND TEACHERS AWARE OF DATA, AND ENGAGE WITH BEST PRACTICES



Aurora Monge-Barrio¹, Ainhoa Arriazu-Ramos¹, María Fernández-Vigil¹, and Ana Sánchez-Ostiz¹
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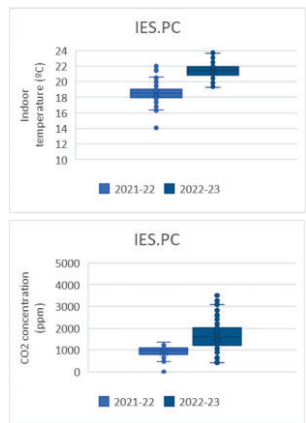
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HOW the COVID Pandemic and the Energy Crisis Have Influenced Indoor Environmental Conditions in non-residential Buildings

- SELECTED CLASSROOMS ARE MONITORED
- STAFF AND TEACHERS AWARE OF DATA, AND ENGAGE WITH BEST PRACTICES



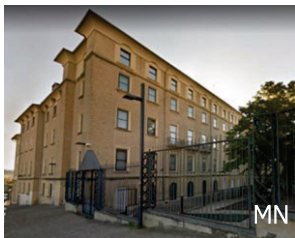
Aurora Monge-Barrio¹, Ainhoa Arriazu-Ramos¹, María Fernández-Vigil¹, and Ana Sánchez-Ostiz¹
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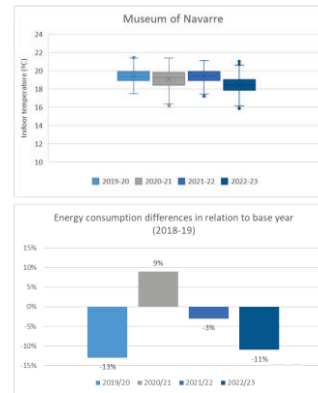
4

HOW the COVID Pandemic and the Energy Crisis Have Influenced Indoor Environmental Conditions in non-residential Buildings

- SELECTED ROOMS ARE MONITORED
- MAIN FOCUS ON THE CONSERVATION OF COLLECTIONS



MN



Aurora Monge-Barrio¹, Ainhoa Arriazu-Ramos¹, María Fernández-Vigil¹, and Ana Sánchez-Ostiz¹
 1. School of Architecture, Universidad de Navarra, Spain.



5

HOW the COVID Pandemic and the Energy Crisis Have Influenced Indoor Environmental Conditions in non-residential Buildings

MAIN CONCLUSIONS of the study:

- NV buildings have had to balance minimum thermal comfort, high levels of ventilation and high energy costs.
- High level of CO₂ concentration prior to COVID has been followed by two years of low and adequate levels for NV buildings during COVID (1000ppm)
- After the pandemic, CO₂ concentration has risen due to concerns on energy costs in case studies (1500ppm)
- Indoor temperatures have been balanced during COVID with high energy consumption (30%), accepting low temperatures for the benefits of face to face learning
- The energy crisis has reduced energy consumption since pre-COVID (10%), affecting IEQ in both kinds of buildings
- The relevance of IAQ has risen through the COVID pandemic, but the perception is that "everybody" has forgotten it
- The perception of the need to a HVAC upgrade is mainly focused on the heating system and renewable energies, but not on the installation of HRV
- The study of existing NV buildings dealing with external events helps in understanding the potential and benefits of NV, which should be considered in their upgrade

Thank you!
 (amongeb@unav.es)

Aurora Monge-Barrio¹, Ainhoa Arriazu-Ramos¹, María Fernández-Vigil¹, and Ana Sánchez-Ostiz¹
 1. School of Architecture, Universidad de Navarra, Spain.



6

The impact of increased occupancy on particulate matter concentrations in mechanically-ventilated residential buildings in a subtropical climate.

German Hernandez^{1,2}, Terri-Ann Berry¹,
Rafael Borge² and Dan Blanchon¹

¹ UNITEC, Institute of Technology, New Zealand

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² Universidad Politecnica de Madrid, Spain

43rd AIVC -11th TightVent & 9th venticool Conference, 2023

1

Outline

Objective

Background Information

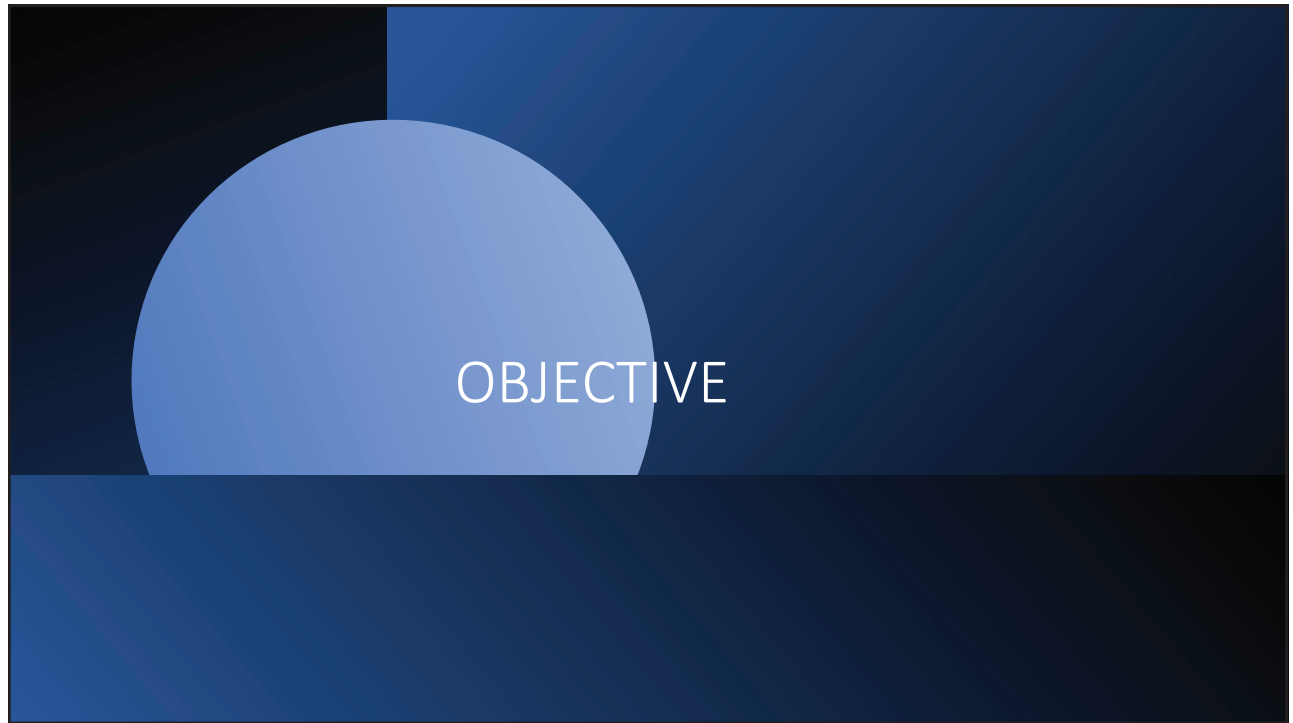
- Mechanical Ventilation Systems
- Factors Affecting IAQ
- Study Location
- 2021 COVID-19 Lockdown

Methodology / Pilot Study Location

Results

Conclusions & Recommendations

2



3

43rd AIVC -11th TightVent & 9th venticool Conference, 2023

Investigate impact of high occupancy levels, caused by stay-at-home orders under a COVID-19 lockdown, on IAQ in mechanically-ventilated residential buildings.

43rd AIVC -11th TightVent & 9th venticool Conference, 2023

4

RESEARCH BACKGROUND

5

43rd AIVC-11th TightVent & 9th venticool Conference, 2023

Mechanical Ventilation Systems



On average, kiwis spend 70% of their time indoors.

Growing pressure to combat climate change:

- Reduce energy consumption
- Improve energy efficiency

Some Energy-efficient characteristics can negatively impact IAQ.

- Increase air exchange and reduce moisture.
- Can improve indoor air quality

6

Factors Affecting IAQ



Season & weather conditions



Building characteristics



Proximity to roads/traffic & industry

Study Location

Auckland

Largest city in NZ

Population over 1.6 million (Stats NZ, 2022)

Humid, subtropical climate – warm, humid summers and mild winters (Hessell, 1988).



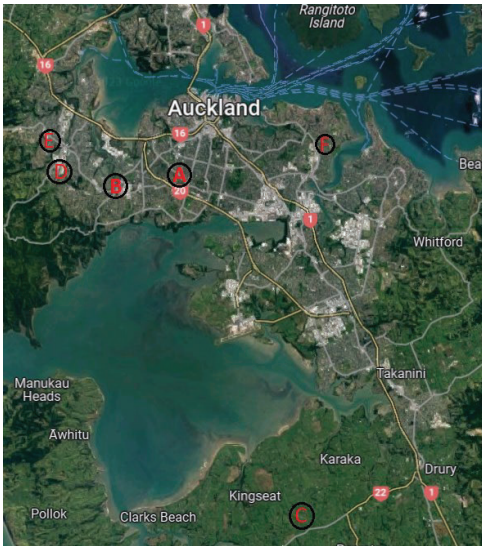


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Methodology



6 houses, mech ventilation
 3-week period: L4 lockdown +/- 1 wk
 Winter period
 Low-cost monitors for indoor/outdoor AQ
 High accuracy: tested by AQ-SPEC
 Outdoor data also from council stations.

11

Pilot Study

Auckland average house size 117 m²

3 indoor monitors:

- Main bedroom, second bedroom, kitchen/living space

1 outdoor monitor:

- South facing

Sensors:

- PM_{2.5}, PM₁₀, Temp, RH



12

RESULTS & ANALYSIS

13

43rd AIVC-11th TightVent & 9th venticool Conference, 2023

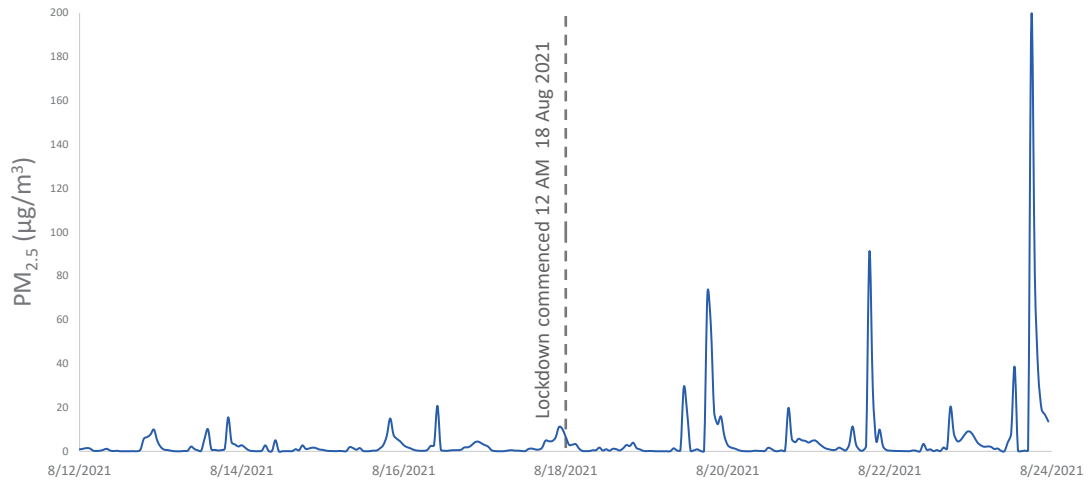
Change in Indoor PM_{2.5} Concentrations

- Significant increase in 50% houses, despite mechanical ventilation.
- Only 1 house exceeded WHO guideline limit.

	PM _{2.5} (µg/m ³)		PM ₁₀ (µg/m ³)	
	Pre-lockdown	During lockdown	Pre-lockdown	During lockdown
House A	0.55	0.80	0.93	0.58
House B	0.73	1.73	1.08	0.36
House C	20.62	21.21	23.51	21.50
House D	4.20	5.24	4.69	6.13
House E	4.96	8.01	5.52	6.05
House F	4.37	5.78	9.00	7.71

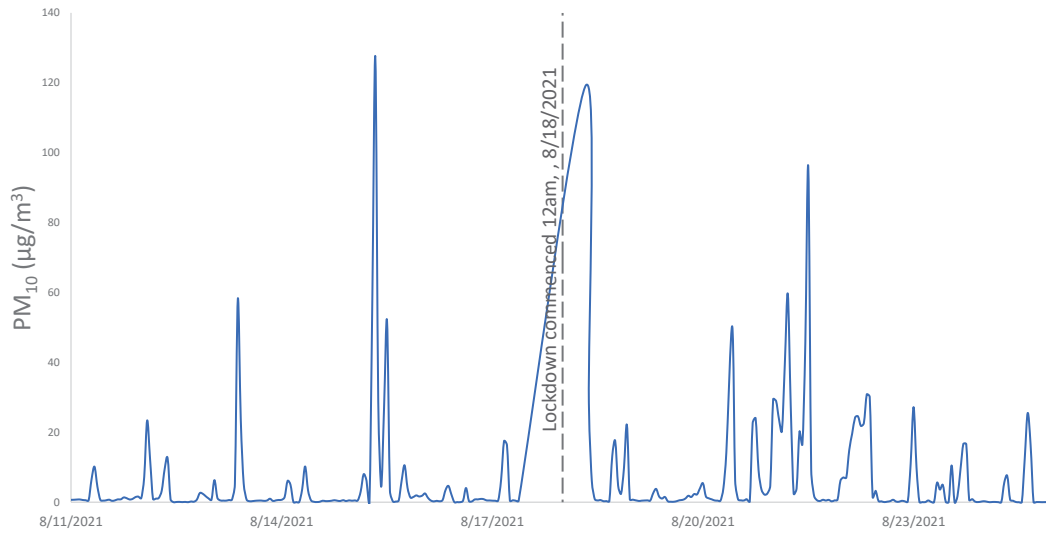
14

1 Week Before/After Lockdown (House D)



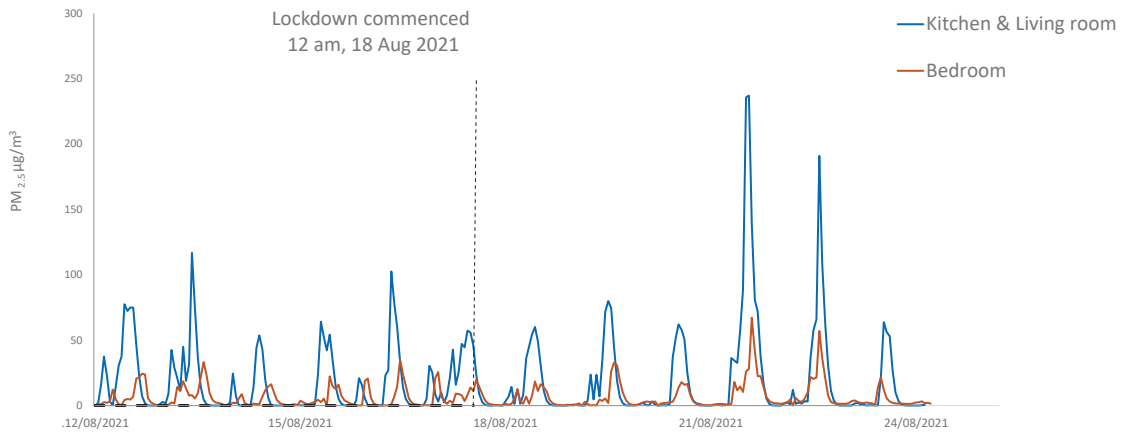
15

1 Day Before/After Lockdown (House E)



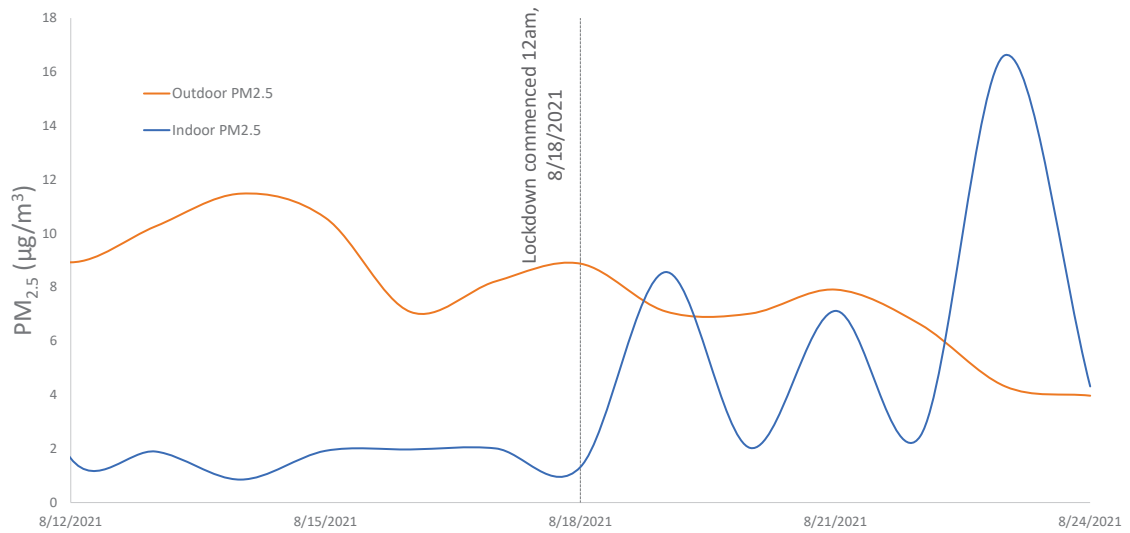
16

Indoor PM_{2.5} Variability



17

Indoor vs Outdoor



18

CONCLUSIONS & FUTURE STUDIES

19

Conclusions

- Outdoor $PM_{2.5}$ decreased 34%
- Indoor $PM_{2.5}$ 25% to 62% higher, suggesting internal sources.
- Importance of ventilation, especially with higher occupancy.



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20

Current and Next

- Conducted Experiments in Winter 2023 and 2024
- Roof Air Quality Assessment in 2023 and 2025
- Examined Different Seasons in 2023
- Increased House Sample to 15 in 2023 and 2024
- Evaluated Sustainable Home Characteristics in 2023 and 2024
- Expanding Study to Other Cities in 2024



21



POLITÉCNICA



Environmental Solutions
Research Centre

43rd AIVC-11th TightVent & 9th venticool Conference, 2023

Thank You!

German Hernandez

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Universidad Politécnica de Madrid, Spain



23

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AIVC 2023 Conference

On-site Capture efficiency of Kitchen Range Hood based on Particle Diameters and Exhaust Flow Rates

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Contents

- 1. Introduction
- 2. Materials and Methods
- 3. Results
- 4. Conclusions

Introduction

- 1. Background
- 2. Research Objective

3

1 Introduction

1 Background

Cooking is one of the biggest contributors to particle concentration
→ Kitchen range hood removes the particles directly and efficiently.

1. Factors affecting the Efficiency of the Hood

Particle Diameter

- Particles with various diameters can be generated during cooking.
(Turning on gas, sautéing, frying)
- exhibit different CEs depend on diameter.

Exhaust rate

- Higher exhaust rate is more effective.

Kitchen range hood
Capture Efficiency (CE)



4

1 Introduction

1 Background

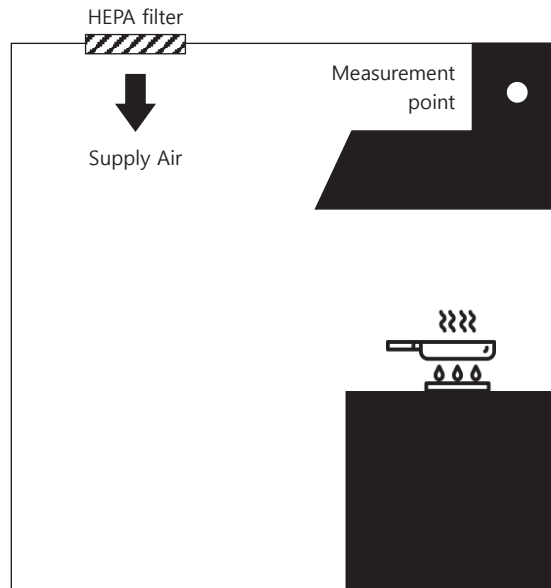
2. Limitations of evaluating CE in ordinary households

1. Difficulty to adhere to standard

- ASTM E3087-18
 - measure concentration inside the kitchen range hood
 - measure concentration of gas
- Measuring inside the exhaust hood is difficult.
- Especially when it comes to measuring particles.

2. The challenge of minimizing outdoor particle infiltration

- HEPA filter in the path of the supply air.



5

1 Introduction

2 Objectives of the study

1. Factors affecting efficiency of the Hood

- Particle diameter
- Exhaust rates

2. Limitations of evaluating CE in ordinary household

- Difficulty to adhere to standard
- Challenging to minimize the outdoor particles

OBJECTIVE

Compare the performance of kitchen range hoods in relation to **particle diameters** and kitchen **exhaust rates** when the kitchen is affected by outdoor air particulate concentration

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Materials and Methods

1. Testbed
2. Measurement
3. Cooking procedure
4. Calculation of capture efficiency
5. Experimental cases

7

2 Material and Methods

1 Testbed

Experimental Overview

Date	January – February, 2020.
Location	A testbed in Seoul
Volume	20.1 m ³ (kitchen)

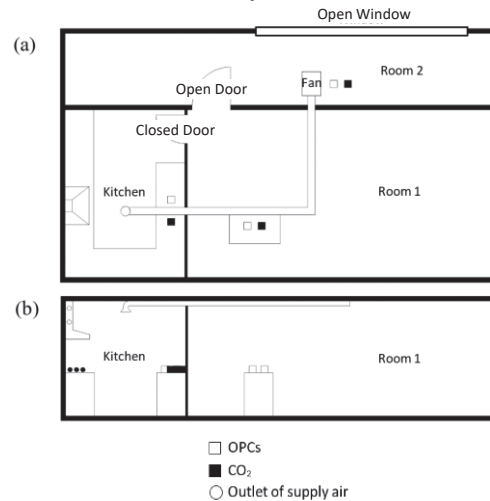
Kitchen Range hood

Type	Chimney type
Cabinets	Two cabinets installed on both side
Height	60 cm above the cooktop
Dimension	90 cm × 55 cm

Cooktop

Type	Electric cooktop
Dimension	60cm × 51 cm
Power Consumption	220 Wh/kg
Used burner	Front burner

Schematic layout of the Testbed



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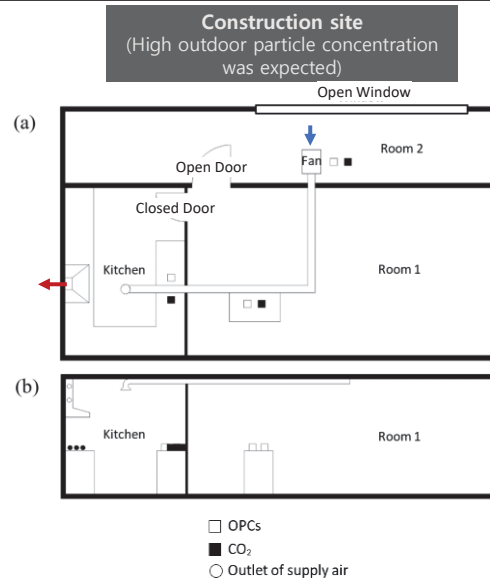
2 Material and Methods

1 Testbed

1. Construction site was 30 m away
→ High outdoor particle concentration was expected.
2. Window and door in room 2 was always opened to draw in outdoor air.
3. Fan in the room 2 supplied outdoor air to the kitchen via ductwork.
There was no filter in the fan.
4. Pressure difference between the kitchen and room 1 was maintained near zero by adjusting the supply air volume flow rate.

→ Particles generated from construction site was introducing to the kitchen.

(∵ Testbed was modeled after a typical household, not a chamber or laboratory)



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2 Material and Methods

2 Measuring instruments

1. Particle counter □

Measuring location	Kitchen, Room 1, Room 2
Instrument	Optical Particle Counter (OPCs) Aerotrak 9306 (TSI)
Sampling rate	1 min

2. CO₂ concentration

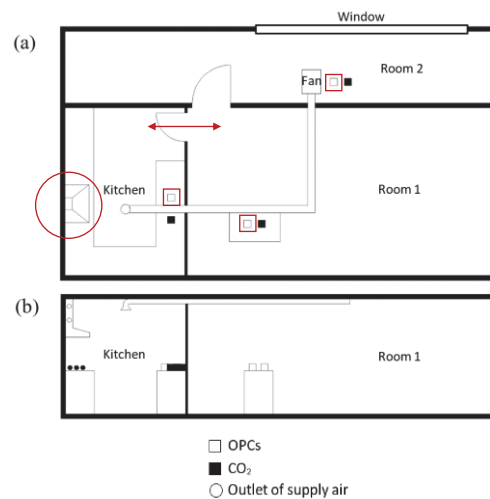
Measuring location	Kitchen, Room 1, Room 2
Instrument	MCH-383SD, Lutron Electronic

3. Pressure difference ↔

Measuring location	Kitchen and Room 1
Instrument	C310 (Kimo)

4. Volumetric flowmeter ○

Measuring location	Kitchen range hood
Instrument	420 (Testo)



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2 Material and Methods

3 Cooking Procedure

Cooking ingredients

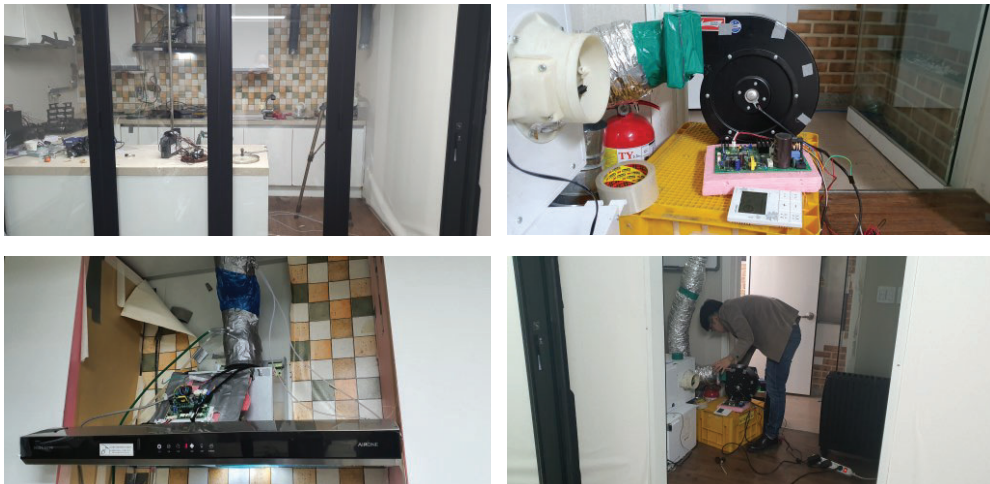
Bacon frying to ensure the even size, weight, and distribution of fat

Cooking Procedure

1. Measure flow rate of the kitchen range hood	Turn on the hood and measure the flow rate.
2. Measure the weight of the bacon	Weigh each piece of bacon before every experiments. The weight was approximately 40 g.
3. Preheat the pan	Until the surface temperature reached 210 – 230 °C (took about 3 min)
4. Cook bacon	For 3 min
5. Turn over the bacon	For another 2 min
6. Cover the pan and remove it from the kitchen	Prevent further particle generation
7. Wait for the concentration return to the initial concentration	
8. Clean up	Remove any oil and particles attached to the walls, cooktop, and kitchen range hood with wet tissues.

2 Material and Methods

3 Cooking Procedure



2 Material and Methods

4 Calculation of Capture Efficiency

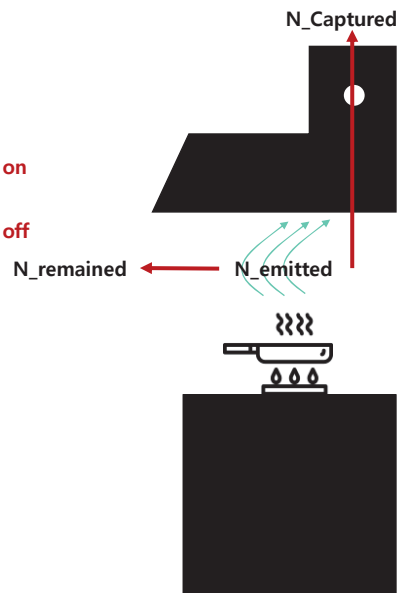
Capture Efficiency

Ratio of the particle captured by the kitchen range hood and the particle emitted from the source.

$$CE = \frac{N_{\text{captured}}}{N_{\text{emitted}}} = 1 - \frac{V \int_{t_0}^{t_b} (C_{\text{remained}}) dt}{V \int_{t_0}^{t_b} (C_{\text{emitted}}) dt}$$

N _{captured}	<ul style="list-style-type: none"> Number of particles captured during experiment → Can be obtained when the hood is on
N _{emitted}	<ul style="list-style-type: none"> Number of particles emitted from cooking during experiment → Can be obtained when the hood is off
V	Volume of the kitchen
t ₀ , t _b	Initial time and the time taken to return to initial concentration

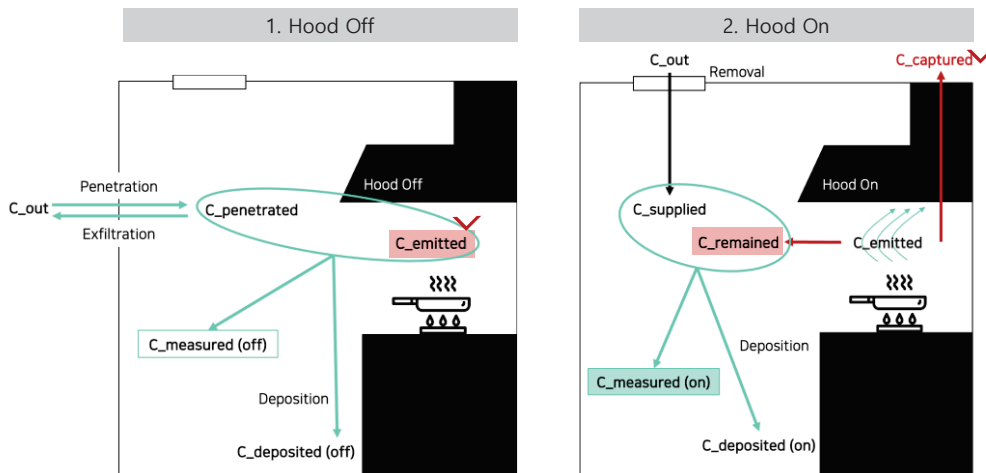
To obtain N_{captured} and N_{emitted},
C_{captured} and C_{emitted} should be calculated beforehand.



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3 CONTAM model and Validation

4 Calculation of Capture Efficiency



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2 Material and Methods

4 Calculation of Capture Efficiency

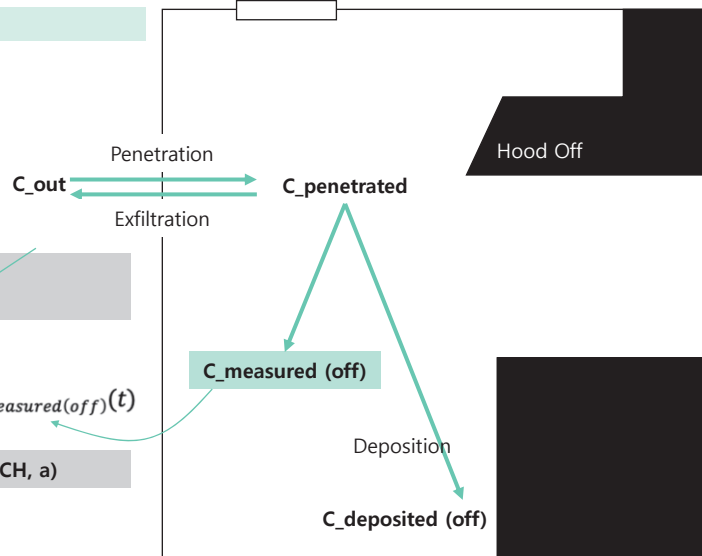
1. C_emitted (Hood off)

Penetration Coefficient (P),
Deposition rate (K)

$$\frac{dC_{measured(off)}(t)}{dt} = aPC_{room2}(t) - (a + K)C_{measured(off)}(t)$$

Air change rate per hour (ACH, a)

$$ACH(t) = -\frac{1}{C_{CO_2}(t)} \frac{dC_{CO_2}(t)}{dt}$$



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2 Material and Methods

4 Calculation of Capture Efficiency

1. C_emitted (Hood off)

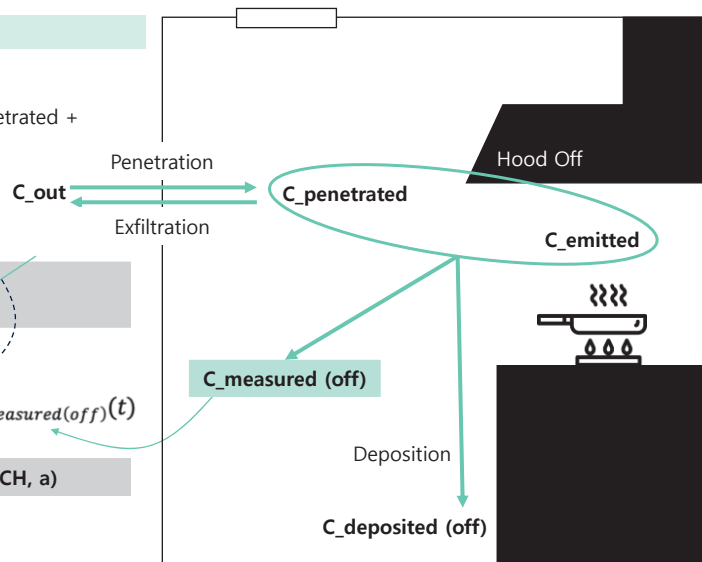
$$C_{emitted} = C_{measured} + C_{deposited} - C_{penetrated} + C_{exfiltrated}$$

Penetration Coefficient (P),
Deposition rate (K)

$$\frac{dC_{measured(off)}(t)}{dt} = aPC_{room2}(t) - (a + K)C_{measured(off)}(t)$$

Air change rate per hour (ACH, a)

$$ACH(t) = -\frac{1}{C_{CO_2}(t)} \frac{dC_{CO_2}(t)}{dt}$$



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2 Material and Methods

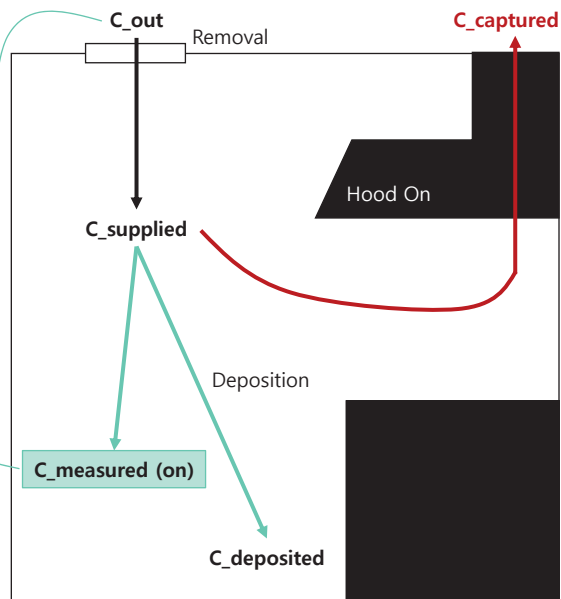
4 Calculation of Capture Efficiency

2. C_{captured} (Hood on)

Removal efficiency (ε), Deposition rate

$$\frac{dC_{measured(on)}(t)}{dt} = (1 - \epsilon) \frac{Q}{V} C_{room2}(t) - \left(\frac{Q}{V} + K \right) C_{measured(on)}(t)$$

ε	Removal efficiency of the fan and duct [-]
K	Deposition rate [-]
Q	Volumetric flow rate of the supply air [m ³ /h]
V	Volume of the kitchen [m ³]



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2 Material and Methods

4 Calculation of Capture Efficiency

2. C_{captured} (Hood on)

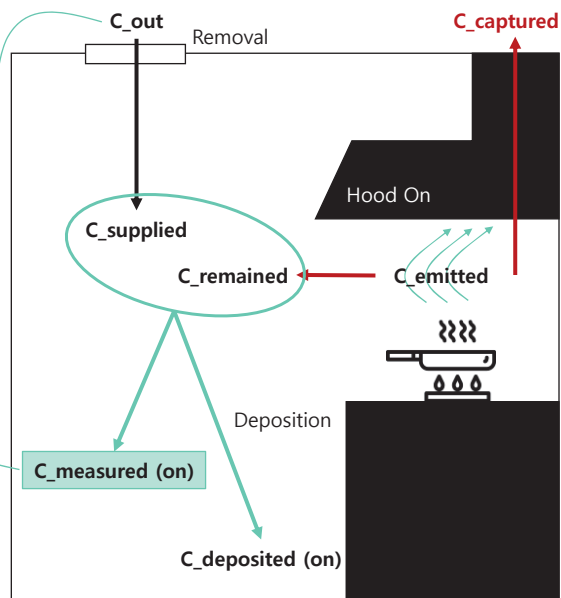
$$C_{captured} = C_{emitted} - C_{remained}$$

$$C_{remained} = C_{measured} + C_{deposited} - C_{supplied}$$

Removal efficiency (ε), Deposition rate

$$\frac{dC_{measured(on)}(t)}{dt} = (1 - \epsilon) \frac{Q}{V} C_{room2}(t) - \left(\frac{Q}{V} + K \right) C_{measured(on)}(t)$$

ε	Removal efficiency of the fan and duct [-]
K	Deposition rate [-]
Q	Volumetric flow rate of the supply air [m ³ /h]
V	Volume of the kitchen [m ³]



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2 Material and Methods

5 Experimental cases

Experimental Cases

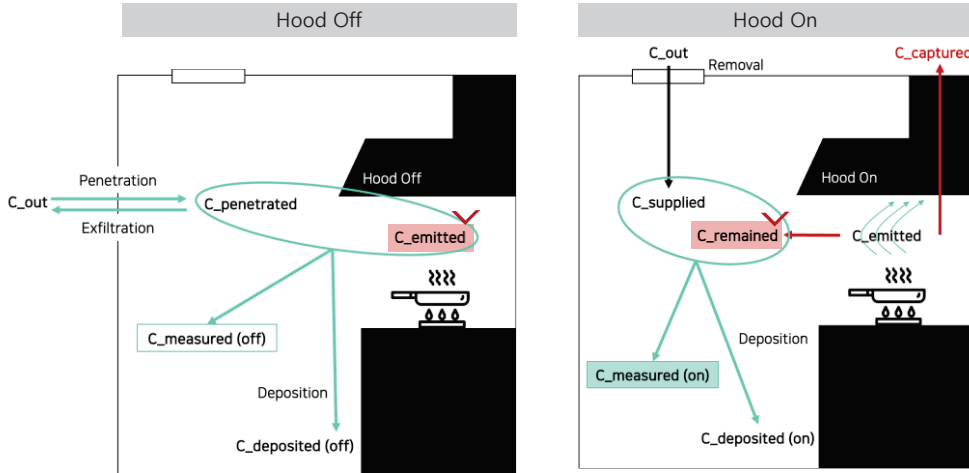
Case	Supply Fan	Exhaust Flow Rates	Repeated #	Estimated Parameters	What to obtain
Hood off	Not operated	-	2	Penetration Coefficient, Deposition Rate	C_emitted
Hood on	Operated	250 m ³ /h	3	Removal Efficiency of the fan and the duct, Deposition rate	C_remaining
		350 m ³ /h	3		

Results

1. Estimated Particle concentration
2. Capture efficiency

3 CONTAM model and Validation

1 Calculated particle concentration

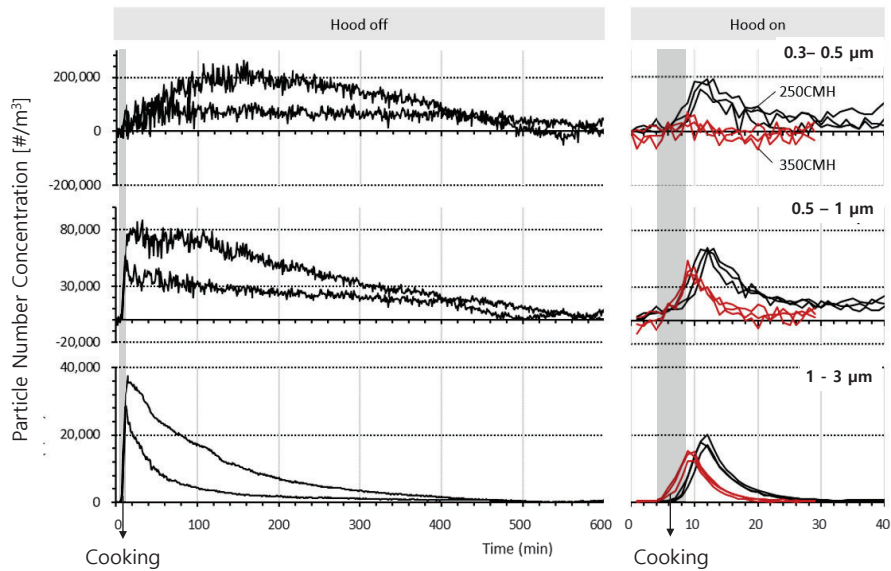


Calculated time-series $C_{emitted}$ and $C_{remaining}$ to compare the hood performance based on the concentration affected by hood only.

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3 CONTAM model and Validation

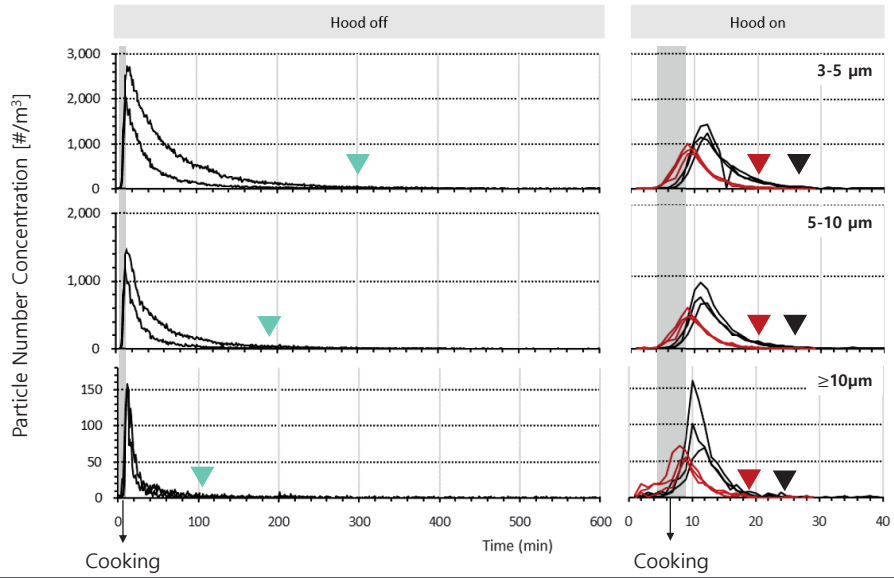
2 Estimated particle concentration



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3 CONTAM model and Validation

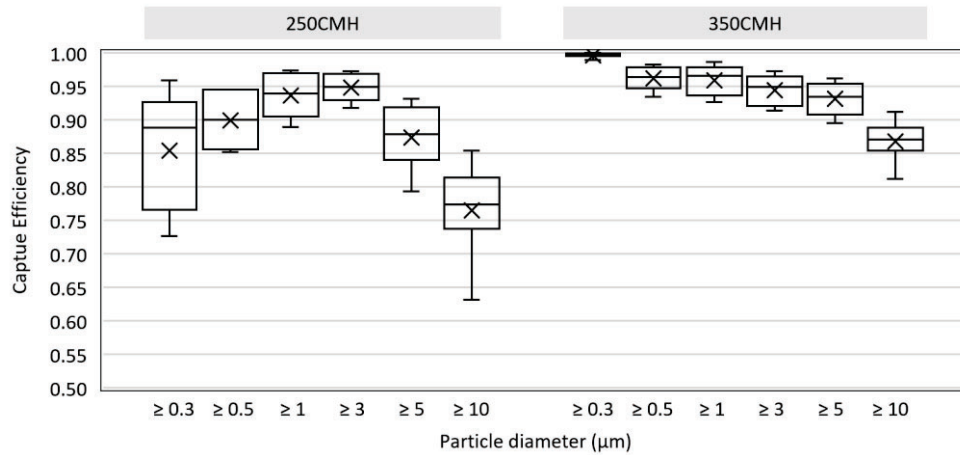
2 Calculated particle concentration



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3 CONTAM model and Validation

3 Capture efficiency



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Conclusion

1. Conclusion
2. Limitations and future studies

25

5 Conclusion

1 Conclusion

This study investigated the particle removal performance of a kitchen range hood depend on the particle diameters and exhaust rates, in terms of capture efficiency. There was experimental attempt to find out the capture efficiency in a typical household setting. In this way, we had to subtract any particles besides generated from cooking.

- Kitchen range hood could reduce the time taken to return to the initial concentration by about 1/5.
- Regardless of the particle size, as the exhaust rate increased, the peak concentration and the time taken to return to the initial concentration decreased.
This leads to the fact that larger exhaust flow rate results in better capture efficiency. ($\leq 350 \text{ m}^3/\text{h}$)
- To remove particles smaller than $1 \mu\text{m}$, larger exhaust flow rate is much more effective.

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5 Conclusion

2 Limitation and future studies

Limitations

- Assumed complete mixing.
 - forced well-mixing was avoid (e.g., using mixing fan) because mixing fan does not operate while typical cooking activity.

Future studies

- Compare and validate the effectiveness of the proposed method in this study using the HEPA filter

Thank you for your Attention

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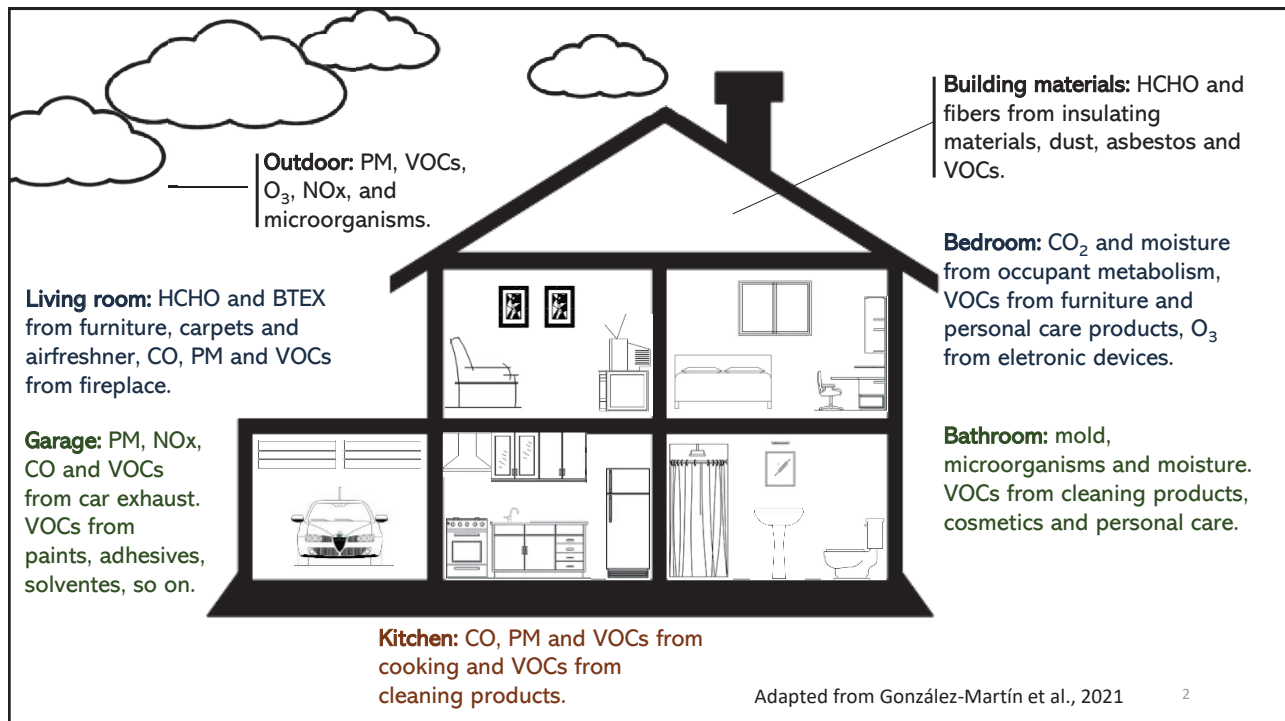
MAIN INDOOR POLLUTANTS IN-KITCHEN

Daniela Mortari
Gaëlle Guyot
Nathan Mendes

Copenhagen, 2023




1



2

COOKING PROCESS



- ✓ Type of stove/fuel
- ✓ Procedure
- ✓ Type of food

3

3

TYPE OF STOVE/FUEL

- ✓ Challenge: Great **diversity of customs** within each country;
- ✓ 3 billion people cook using open fires or simple stoves:
wood, charcoal, animal dung, crop waste, coal, and kerosene.
WHO (2022)


12 major cities (5 houses each),

+

4 global regions:

→

natural gas (NG),
 liquefied petroleum gas (LPG), kerosene, ethanol, electric + different combinations



Kumar et al., 2022

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TYPE OF STOVEN/FUEL

Fuel	Main emissions	Authors
Natural gas (NG)	CO ₂ , CO, steam, NO, NO ₂ , CH ₂ O, CH ₄ and PM	(Lebel et al., 2022; Singer et al., 2017; Zheng et al., 2022)
Liquefied petroleum gas (LPG)	CO ₂ , CH ₄ , CO, SO ₂ , NO, NO ₂ , PM ₁₀ , PM _{2.5} and HC	(Oke et al., 2020; Permadi et al., 2017)
Charcoal	CO ₂ , CH ₄ , N ₂ O, PM ₁₀ , PM _{2.5} , SO ₂ , CO, NO _x , black carbon (BC), organic carbon (OC), non-methane volatile organic compounds (NMVOC)	(Permadi et al., 2017)
Ethanol	CO, CO ₂ , PM _{2.5}	(Chomanika et al., 2022)
Kerosene	CO ₂ , CH ₄ , N ₂ O, PM, SO ₂ , CO, NO _x , BC, OC, NMVOC	(Permadi et al., 2017)
Coal	CO ₂ , CH ₄ , PM, SO ₂ , CO, NO _x , BC, OC, NMVOC	(Permadi et al., 2017)
Raw wood-based fuel	CO ₂ , H ₂ O, NO, NO ₂ , SO _x , CO, CH ₂ O, PM and VOCs	(Kuye and Kumar, 2023)

5

TYPE OF STOVE/FUEL



Kenya Ceramic Jiko (KCJ) (photo source: Njenga et al., 2017)

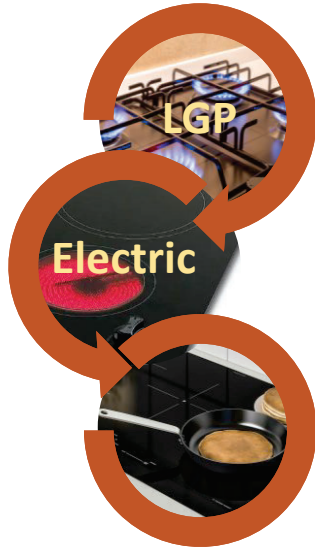
Fuel	Charcoal	Ethanol briquettes
Consumption (g of fuel per kg of cooked food (g/kg))	665	453
CO emissions rate (g/min)	20.71	0.029
PM _{2.5} emissions rate (mg/min)	1398.03	0.0931

(source: Chomanika et al., 2022)

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6

TYPE OF STOVE/FUEL



- ✓ CO, CO₂, NO, NO₂, and TVOC were reduced by around 40%, 26%, 86%, 42% and 48%, respectively (Zhao and Zhao (2018))
- ✓ CO₂ emissions reached 3500 ppm to LPG stove while the concentrations to induction and electric stove were near 500 ppm
- ✓ The CO concentration reached 23 ppm to LPG stove and was lower than 4 ppm to induction and electric stove (Martínez-Gómez et al., 2016)



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PROCEDURE

Reported methods: boiling, grilling, steaming, stewing, braising, stir-frying, pan-frying, deep-frying and roasting (Zhao and Zhao, 2018)

Main results: frying and grilling produce more PM than boiling and steaming food, especially if the food is Maillard browned or charred

- Type of pan and the use of seasonings also affect the indoor air pollutants

8



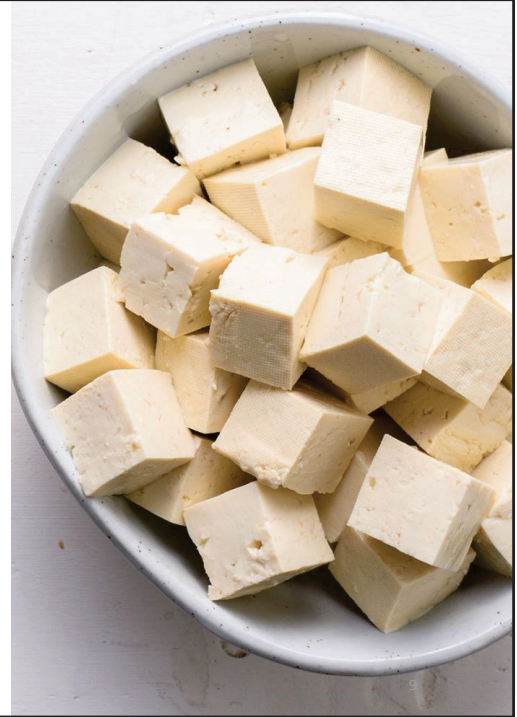
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PM_{2,5} AVERAGE MASS CONCENTRATIONS

- ✓ **Steaming:** 65,7±7.6 µg/m³
- ✓ **Boiling:** 81.4±9.3 µg/m³
- ✓ **Stir-frying:** 120±13 µg/m³
- ✓ **Pan-frying:** 130±15 µg/m³
- ✓ **Deep-frying:** 190±20 µg/m³.

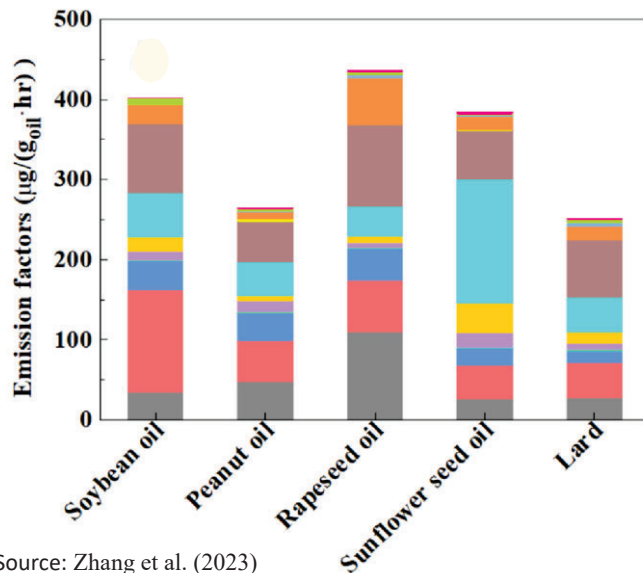
During the cooking process of 150 g of tofu

Balasubramanian 2008 (apud Zhao and Zhao 2018)



9

TYPE OF FOOD – EFFECT OF EDIBLE OILS



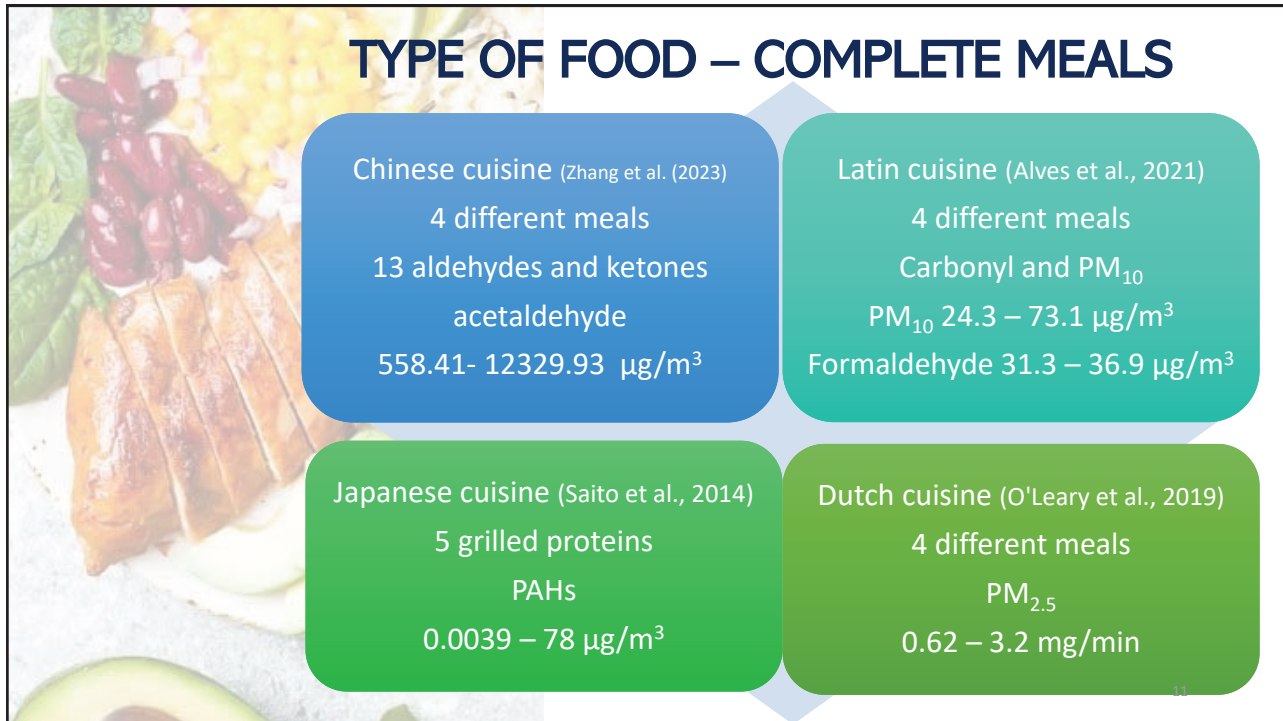
Frying is the procedure that emits more pollutants – many studies investigating edible oils

- Methyl isobutyl ketone
- 3 - Pentanone
- 2 - Pentanone
- Butanone
- Methyl vinyl ketone
- Acetone
- Hexanal
- Amyl aldehyde
- n-Butyl aldehyde
- Methylacrolein
- Propanal
- Acrolein
- Acetaldehyde

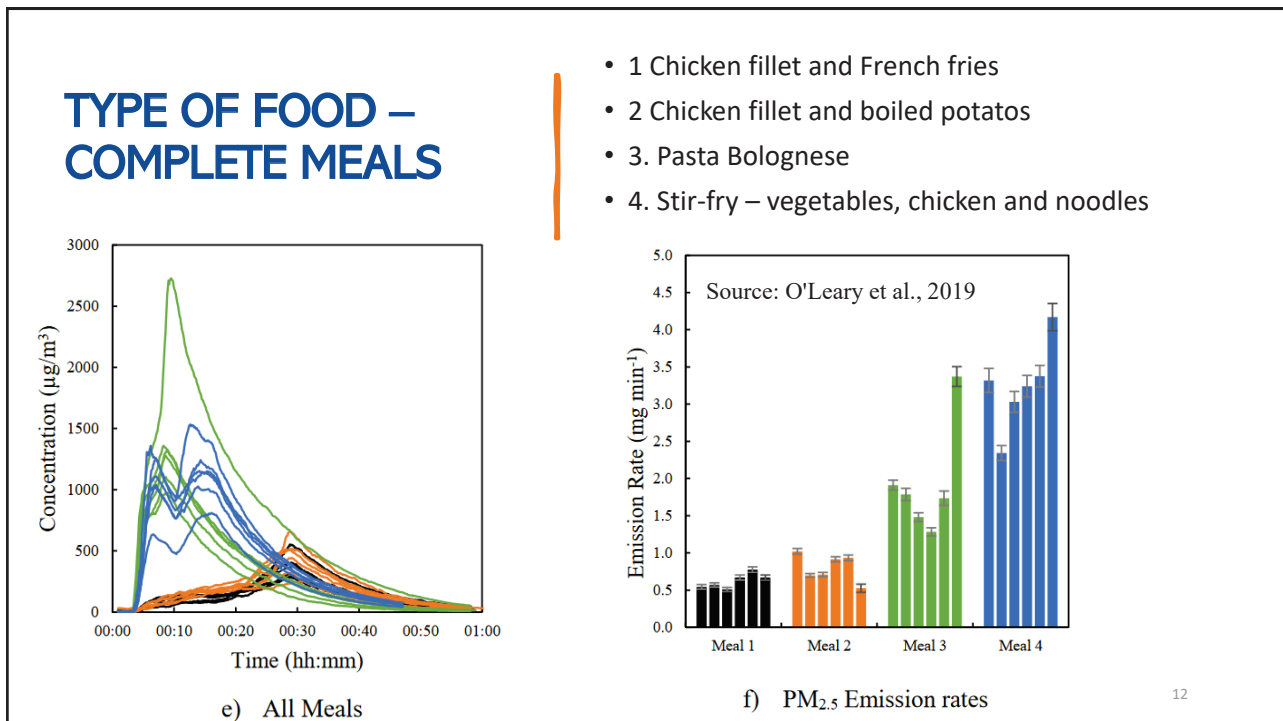
Source: Zhang et al. (2023)

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DISCUSSION

- Various contaminants exceeded the maximum threshold concentrations defined by the World Health Organization (WHO) based on health effects.
- In different cultures the cooking process is repeated three or more times a day, every day, and in many situations, performed daily by the same person.
- More than 3 million people die annually worldwide due to illnesses caused by fumes and toxins released during cooking processes .
- While source reduction is a key in reducing avoidable pollution, ventilation is a key in reducing unavoidable pollution, such as cooking emissions.
- Challenge: Manual activation (USA: > 80% of houses built after 2003 have the devices)

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CONCLUSIONS

- **Ventilation** appears as a key in reducing inevitable pollution.
- However, when it is **manually activated**, its use is decreased and only 30% of households reports to use the devices.
- This decrease also indicates the lack of knowledge of cooking pollutant hazards.
- In this context, **smart ventilation** associated with **education** on cooking pollutants is promising in improving IAQ.

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THANK YOU FOR YOUR ATTENTION!



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Auvergne-Rhône-Alpes



Cerema



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THANK YOU FOR YOUR ATTENTION!



MORTARI, Daniela



GUYOT, Gaëlle



MACHADO, Denner



BATISTELA, Marcos



MENDES, Nathan

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Alves, C.A., Vicente, E.D., Evtyugina, M., Vicente, A.M.P., Sainnokhoi, T.-A., Kováts, N., 2021. Cooking activities in a domestic kitchen: Chemical and toxicological profiling of emissions. *Science of The Total Environment* 772, 145412.

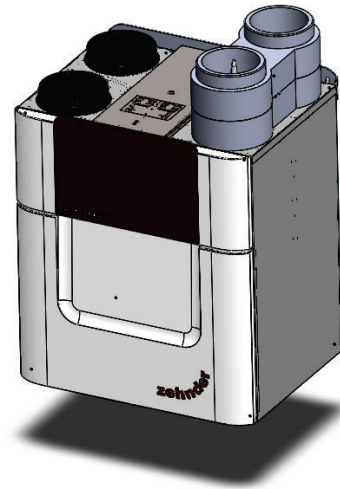
Saito, E., Tanaka, N., Miyazaki, A., Tsuzaki, M., 2014. Concentration and particle size distribution of polycyclic aromatic hydrocarbons formed by thermal cooking. *Food Chemistry* 153, 285–291.

AIVC Conference 2023 in Copenhagen

Fine dust (PM) measurement in ducts of balanced ventilation systems

Bart Cremers (in cooperation with Jan de Vries)
Knowledge Consultant Ventilation technologies

zehnder

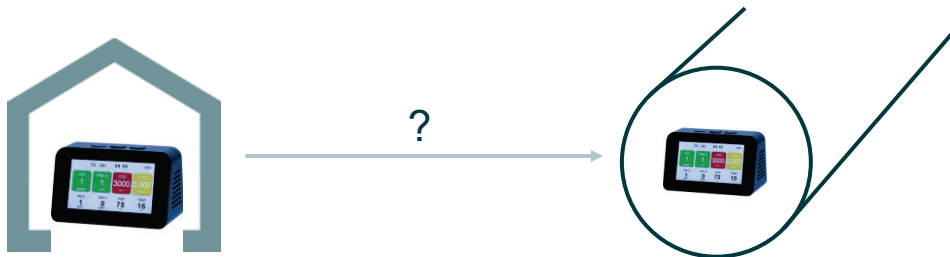


1

Introduction: PM measurement in ducts

All currently available particulate matter (PM) sensors are designed to measure in slow-moving air (<math><1\text{m/s}</math>).

zehnder

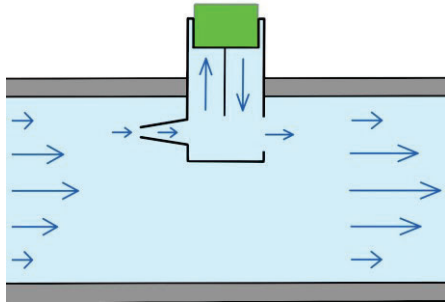


2

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2

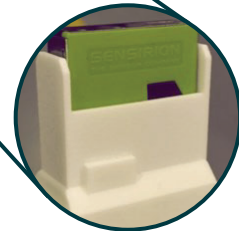
Technology of measurement in ducts



1. isokinetic sampling of air
2. slowing down below 1 m/s
3. traversing PM sensor



Rapid prototype
without PM sensor



detail, with PM sensor

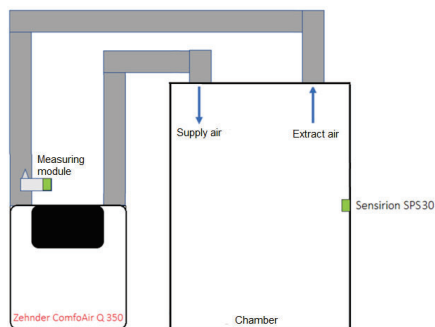
3

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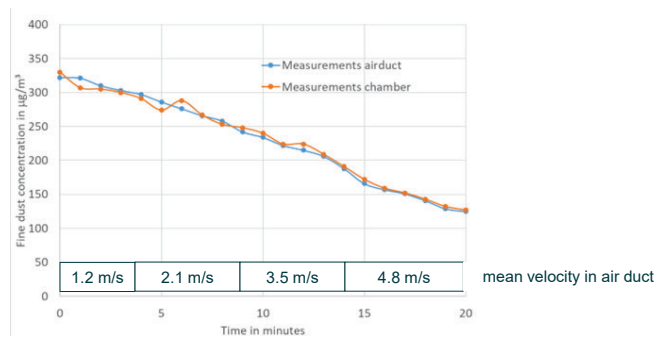
3

Measurement in ducts compared to measurement in chamber

Experimental setup



Typical decay pattern of incense smoke



PM measurement in ducts can reproduce the values as measured in a test chamber (difference 2% to 4%)

4

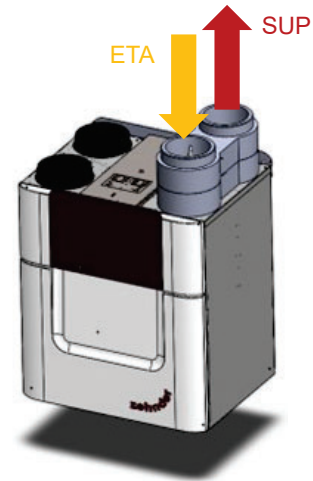
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Measurement in ducts of balanced ventilation system

Measurement of PM2.5 in extract air duct and in supply air duct

- Measurement in extract air is average of all rooms with extract
- Measurement in supply air is filtered outdoor air
- Measurement in supply air is after heat recovery, so in less variable thermal conditions



5

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Test in three operational balanced ventilation units in three locations in NL



6

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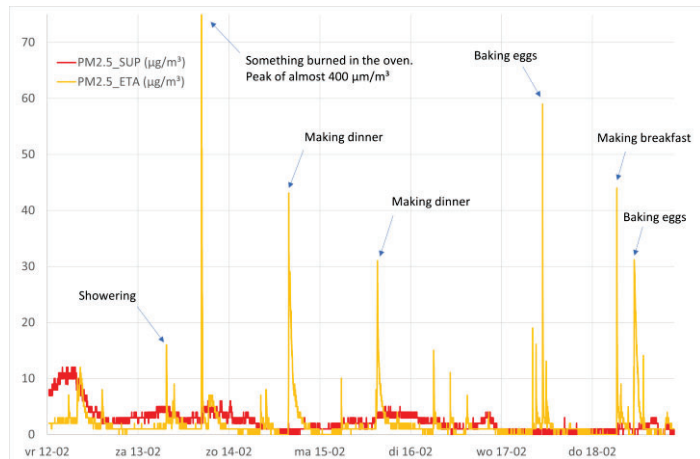
Test in Helmond (NL)

Extract

- large fine dust peaks caused by cooking
- small fine dust peaks caused by showering and steam ironing

Supply

- Small variation of fine dust values in supply air (because of outdoor variation)



7

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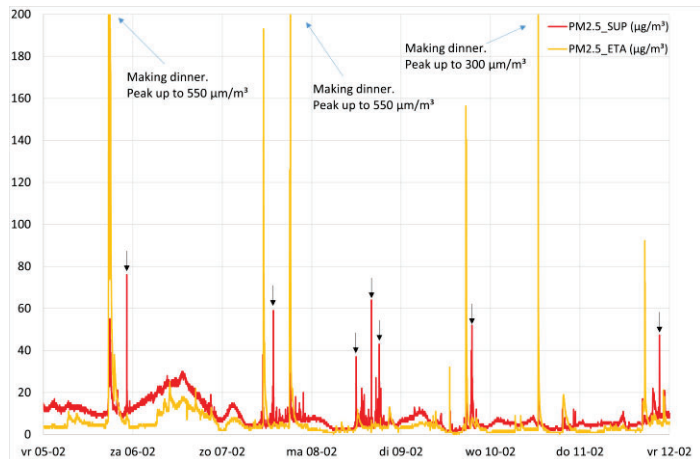
Test in Lettele (NL)

Extract

- large fine dust peaks caused by cooking

Supply

- Fine dust peaks in supply air are caused by re-intake of exhaust air by a wood stove with unfavourable winds
- Elevated levels in supply air lead to elevated levels in extract air, therefore indoor levels are result of indoor sources and outdoor sources (and ventilation rate)



8

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8

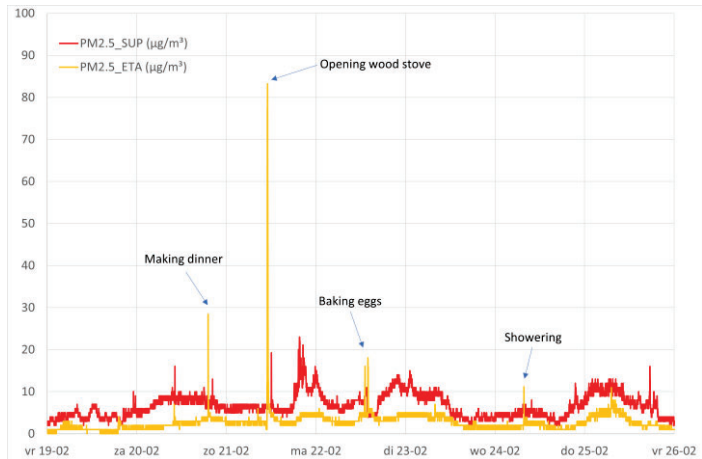
Test in Zwartsluis (NL)

Extract

- large fine dust peaks caused by cooking (lower values because of good cooking hood)

Supply

- Fine dust peaks in supply air are caused by re-intake of exhaust air by a wood stove with unfavourable winds

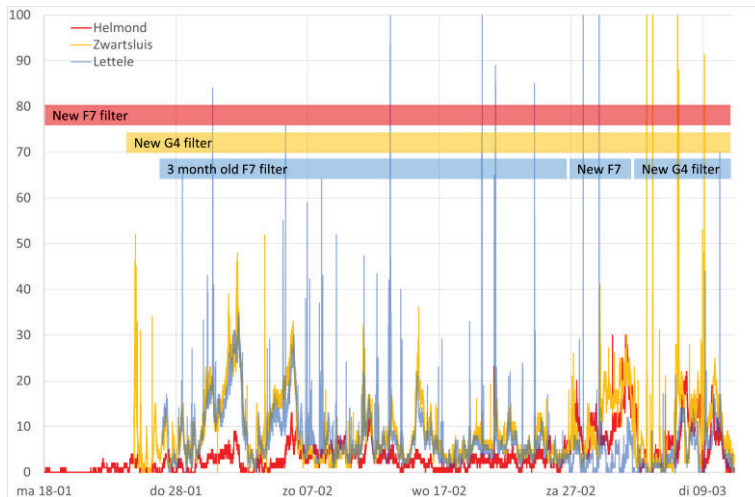


9

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Comparing fine dust in supply air at three locations in NL



- similar pattern of fine dust values in all three locations
- ePM1/F7 filter of 3 months or older does not filter efficiently anymore
- influence of filter type (ePM1/F7 or Coarse/G4) and filter age (new or old)

10

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10



Conclusions

- Fine dust concentrations coming from outside of the house can be measured qualitatively in the supply air duct, with the measurements depending on
 - outside activity, and
 - the filter type and filter age

- Fine dust concentrations coming from inside the house can be measured qualitatively in the extract air duct, with the measurements depending on
 - type/duration of inside activities, and
 - the amount of air-extraction per room



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43rd AIVC - 11th TightVent & 9th venticool Conference

The Impact of Deep Energy Renovations on Indoor Air Quality and Ventilation in Irish Dwellings (ARDEN)

*Hala Hassan, Asit Kumar Mishra, Hilary Cowie, Emmanuel Bourdin,
Brian McIntyre, Marie Coggins**

University
ofGalway.ie

1



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Outline/Agenda

- Background
- Study objective & methodologies
- Preliminary results - Indoor air pollutants and ventilation measured
- Conclusions



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- Evaluate the impact of deep energy retrofit on IAQ and thermal comfort in a sample of Irish dwellings.
- Assess the performance of the mechanical ventilation systems installed as part of the retrofit

3



Irish Residential Sector



27% energy use



28% of Ireland's total energy emissions



Strong tradition using fossil fuels

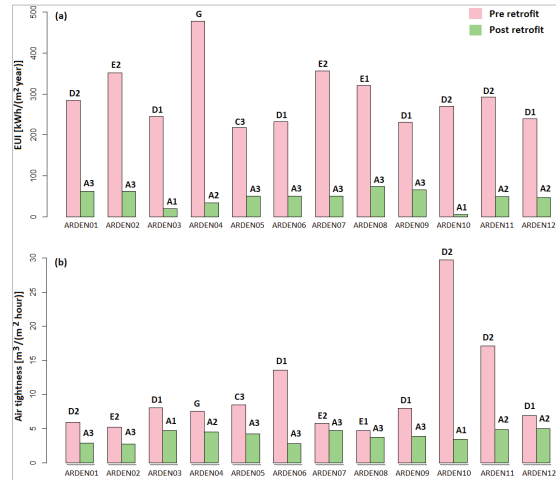


Irish homes use 7% more energy and emit 60% more CO₂ than average EU home

4

Participating Dwellings

- N=26 (N=11)
- SEAI Deep Retrofit pilot programme
- 14 – 59 years old constructions
- 3 yr + post retrofit
- Non-smoking
- Detached/semi-detached, Cavity wall
- Floor area approx. 50 – 283 m²



5

Pollutants of interest and measuring equipment



6

Particulate Matter PM_{2.5}

TSI AM520 monitor

(Monitored for 3 days)



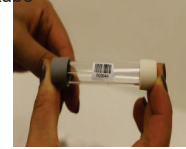
Formaldehyde UMEX 500 badge

(3 days)



Nitrogen Dioxide (NO₂) Passive tube

(3-4 weeks)



Carbon dioxide (CO₂)
Carbon monoxide (CO)
Total Volatile Compounds (TVOCs)
Temperature
Relative Humidity
GrayWolf

(Monitored for 3 days)



BTEX TD passive tubes

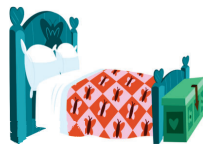
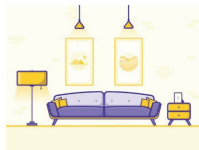
(3-4 weeks)

Radon closed alpha-track detector (*Radtrak*)

(3 months)

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Environmental Survey, Questionnaire, Occupant Diary



Contextual information sheet

ARDEN

ID # _____
Date _____
Pre retrofit _____
Post retrofit _____

UCG Galway
seai

		DAY ONE					
IN THE MAIN LIVING ROOM Did any of the following happen? (Please tick all that apply)		9am-10am	10am-11am	11am-12noon	12noon-3pm	3pm-5pm	5pm-8pm
Windows opened							
Cool/Wood/Peat fire burning							
Candles/incense burning							
Gas hob used							
Electric cooker used							
Toaster/grill used							
Extractor fan on							
Cleaning/polishing							
Vacuum cleaner used							
Pets were present							
4 or more people were present							
Wall or window vents present/open/closed							

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Ventilation

DCV (n=9)
Humidity controlled
Manometer (KimoMP50)
Ventilation extracts in wet
rooms(kitchen/bathrooms/utility)

MVHR (n=2)
Flowfinder
Air extract and supply terminals

ACH
Bedroom CO₂ steady-state data



Preliminary results

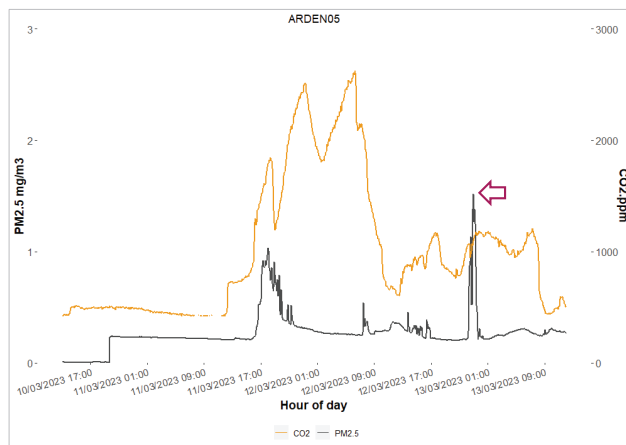
Results – Effect of Retrofit on Indoor Air Pollutants

	Pre- Retrofit		Post-retrofit	
	Bedrooms	Living rooms	Bedrooms	Living rooms
PM _{2.5} [$\mu\text{g m}^{-3}$]	11.2 (7.5,16.7)*	9.9 (6.6,14.7)	27.3 (18.3,40.8)	24.0 (16.1,35.8)
CO ₂ [ppm]	960 (860,1060)	825 (725, 925)	903 (803, 1003)	768 (668, 868)
Temperature [°C]	19.2 (18.5,19.8)	19.7 (19.0,20.3)	21.4 (20.7,22.1)	21.9 (21.2,22.6)
Relative Humidity [%]	53.8 (51.1,56.5)	52.5 (49.8,55.2)	50.3 (47.6,53.0)	49.0 (46.3,51.7)

* Least-square means (LCL,UCL)

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PM_{2.5} and occupants activities



- Average 24-hr post-retrofit:
Bedrooms: 4.8 – 318 $\mu\text{g}/\text{m}^3$
Living rooms: 3.1 – 258 $\mu\text{g}/\text{m}^3$
- Highest 24-h average (318 $\mu\text{g}/\text{m}^3$) detected at **ARDEN05** bedroom
- Peak in 2nd night (1,518 $\mu\text{g}/\text{m}^3$)
- Door closed, pets present in room over-night
- Under-ventilated (ACH = 0.3 h⁻¹)
- No ensuite ~ no extract vent
- One extract in bathroom – faulty reading

12



Ventilation



Ventilation – reported issues

- Poor maintenance
- Visible dust on vents/fan blocked
- Noise
- 25% homes reported no handover post-retrofit
- In most cases lack of understanding of the systems





Since ARDEN Commenced

- Technical Guidance Document (TGD) F - Ventilation (2019) to the Irish Building Regulations
 - requirement for a third party to validate that ventilation system has been installed, balanced and commissioned to achieve the design flow rates
- Presence control indicators to indicate to occupant that system is operating correctly and if fault occurred
- National Promotion Campaign (video/guidance brochure) to raise awareness on the importance of proper ventilation in homes
- NSAI S.R.54 currently under review
 - more emphasis on IAQ



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Conclusions

- Higher concentrations of some IAPs detected in bedrooms and living rooms
- Warm temperatures detected in bedrooms and living rooms during summer time
 - Promote natural ventilation/more airing
- Public awareness with regard to IAPs
- Need for further studies on occupant behavior and it's impacts on IAQ and ventilation



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Acknowledgement

- Project participants
- ARDEN project team, Dr Marie Coggins (University of Galway), Dr Nina Wemken (University of Galway), Dr Asit Kumar Mishra (UCC), Hilary Cowie (Institute of Occupational Medicine Edinburgh)
- Sustainable Energy Authority of Ireland – Brian McIntyre, Conor Hanniffy
- Irish Department of Housing, Local Government and Heritage – Dr Emmanuel Bourdin

This project has been funded by the Sustainable Energy Authority of Ireland under the SEAI Research, Development & Demonstration Funding Programme 2019, Grant number 18/RDD/204.



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17



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Thank you !

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18

Questions ?



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Financial impact of leaky ductwork in buildings – a calculation tool to raise awareness

AIVC 2023 CONFERENCE COPENHAGEN
5TH OCTOBER 2023

NOLWENN HUREL - PLEIAQ
VALÉRIE LEPRINCE – CEREMA
MARCUS LIGHTFOOT - UBBINK

5th October 2023

Nolwenn Hurel - PLEIAQ

1

Introduction

“Build tight, ventilate right”

- Buildings more and more airtight (less energy losses)
- Mechanical ventilation systems to ensure a good IEQ

But in practice: ductwork leakages issues

- Low awareness
- Consequences on energy use and/or IAQ



Credit: Ubbink

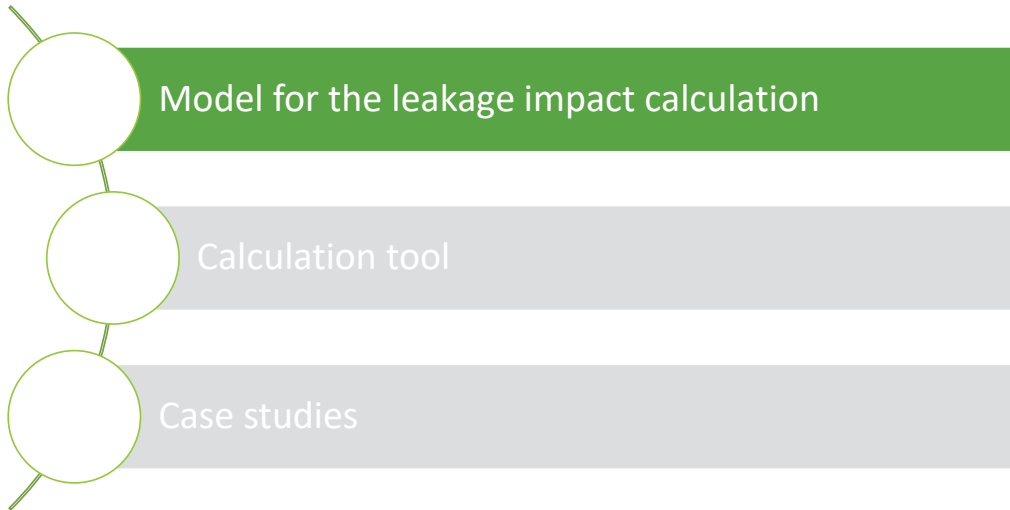
→ **Objective:** Development of a **simple tool** to estimate the **financial impact of leaky ductwork** in buildings over their whole life

5th October 2023

2

Nolwenn Hurel - PLEIAQ

2



Fan energy use

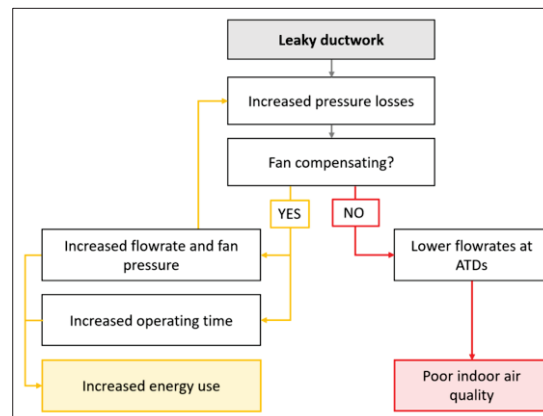
Electrical fan power:

$$P_{AHU} = \frac{\Delta p_{AHU} \times Q_{AHU}}{\eta \times 3600}$$

Pressure → Δp_{AHU} Flowrate → Q_{AHU}
 efficiency → η

Assumption:

Full compensation of leakage



Pressure losses

Friction losses:

$$\Delta p_f = \frac{1000 f L}{D_h} * \frac{\rho V^2}{2}$$

Labels: Friction factor (f), Length (L), Velocity (V), Diameter (D_h)

Dynamic losses :

$$\Delta p_t = \frac{C \rho V^2}{2}$$

Labels: Total loss coeff. (C), Velocity (V)

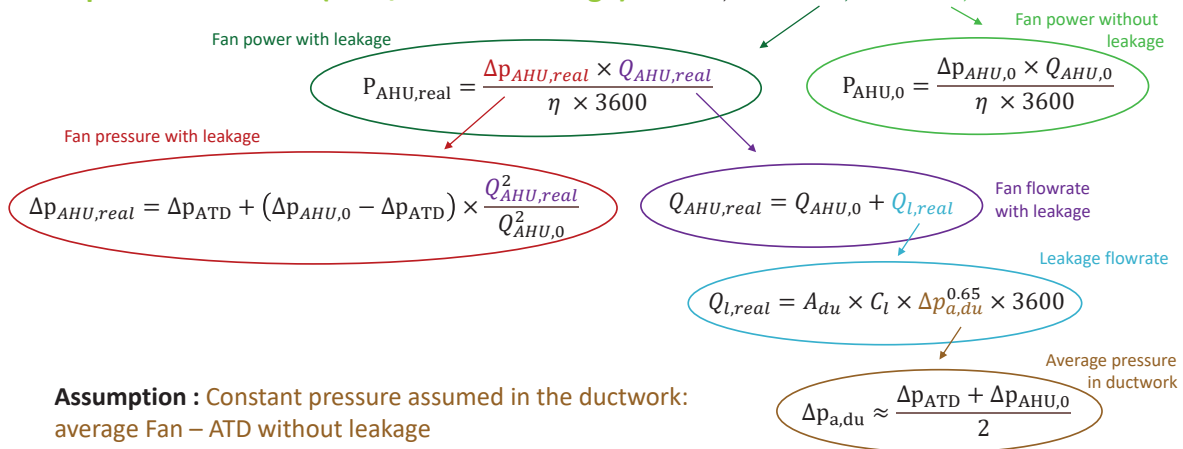
Total pressure loss in the ductwork:

$$\Delta p_{loss} = \left(\frac{1000 f L}{D_h} + \sum C \right) \left(\frac{\rho V^2}{2} \right)$$

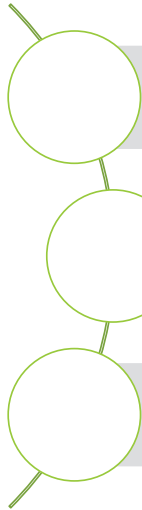
5

Electrical overconsumption

Fan power difference (with/without leakage) : $\Delta P_{AHU,real} = P_{AHU,real} - P_{AHU,0}$




6



Model for the leakage impact calculation


Calculation tool

Case studies



Calculation tool - Ductwork airtightness impact on energy use

Version : 1.0 Date : June 2023 By : PLEIAQ




Color code	Value to be completed	Value calculated (do not modify)	Cell to be ignored
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Inputs						
	Balanced		EXHAUST		SUPPLY	
	Data	Estimation	Unit	Data	Estimation	
Flowrate at the fan/AHU (designed):	225		m ³ /h	225		
Pressure at the fan/AHU (designed):	110		Pa	110		
Pressure at the ATD:	10		Pa	10		
Power of the fan/AHU (designed):	26		W	26		
Ductwork area:	calculation	14,72	m ²		14,72	
Annual operating time:	estimation	8760	h/year			
National cost of a kWh:		0,28	€/kWh			
Years of operation:	known	80	years			

Results					
Airtightness classes		Annual energy overcons.		Leakage cost	
Prev. name	New name	Value (kWh)	% fan cons.	Annual	Total
3A	-	452	99%	126 €	10 117 €
2,5A	ATC 6	361	79%	101 €	8 095 €
1,5A	-	199,5	43,8%	56 €	4 469 €
A	ATC 5	127,5	28,0%	36 €	2 855 €
B	ATC 4	40,1	8,8%	11,2 €	898 €
C	ATC 3	13,1	2,9%	3,7 €	294 €
D	ATC 2	4,3	1,0%	1,22 €	97 €
-	ATC 1	1,4	0,3%	0,40 €	32 €


Estimated values						
Ductwork area 14,72 m ²						
EXHAUST ductwork						
Type 1	Shape	Side A (mm)	Side B (mm)	Diam. (mm)	Length (m)	Area (m ²)
Type 2	none				0,00	0,00
Type 3	none				0,00	0,00
Type 4	none				0,00	0,00
Annual operating time 8760 hours/year						
nb of hours / day 24 hours/day						
nb of days / week 7 days/week						
nb of weeks / year 52,143 weeks/yes						
SUPPLY ductwork 14,72 m ²						
Type 1	Shape	Side A (mm)	Side B (mm)	Diam. (mm)	Length (m)	Area (m ²)
Type 2	none				0,00	0,00
Type 3	none				0,00	0,00
Type 4	none				0,00	0,00
Years of operation: 80 years						

Color code	Value to be completed	Value calculated (do not modify)	Cell to be ignored
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Calculation tool - Ductwork airtightness impact on energy use

Version : 1.0 Date : June 2023 By : PLEIAQ




Color code	Value to be completed	Value calculated (do not modify)	Cell to be ignored
------------	-----------------------	----------------------------------	--------------------

Inputs			
Ventilation system:	Balanced	EXHAUST	SUPPLY
	Data	Estimation	Unit
Flowrate at the fan/AHU (designed):	225		m ³ /h
Pressure at the fan/AHU (designed):	110		Pa
Pressure at the ATD:	10		Pa
Power of the fan/AHU (designed):	26		W
Ductwork area:			
Annual operating time			
National cost of a kWh			
Years of operation:			

Results					
Airtightness classes		Annual energy overcons.		Leakage cost	
Prev. name	New name	Value (kWh)	% fan cons.	Annual	Total
3A	-	452	99%	126 €	10 117 €
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1,5A	-	199,5	43,8%	56 €	4 469 €
A	ATC 5	127,5	28,0%	36 €	2 855 €
				€	898 €
				€	294 €
				€	97 €
				€	32 €

Publication on [Ubbink.com/International](https://ubbink.com/International) expected soon

Estimated values						
Ductwork area						
EXHAUST ductwork 14,72 m ²						
Shape	Side A (mm)	Side B (mm)	Diam. (mm)	Length (m)	Area (m ²)	
Type 1	circular		75	62,5	14,72	
Type 2	none					0,00
Type 3	none					0,00
Type 4	none					0,00
Annual operating time 8760 hours/year						
nb of hours / day 24 hours/day						
nb of days / week 7 days/week						
nb of weeks / year 52,143 weeks/yez						
SUPPLY ductwork 14,72 m ²						
Shape	Side A (mm)	Side B (mm)	Diam. (mm)	Length (m)	Area (m ²)	
Type 1	circular		75	62,5	14,72	
Type 2	none					0,00
Type 3	none					0,00
Type 4	none					0,00
Years of operation: 52,143						



Model for the leakage impact calculation

Calculation tool

Case studies

Case studies: 4 houses & 1 school

Medium-sized house

- heat recovery
- diameter: 75 mm
- length: 125 m

Medium-sized house

- heat recovery
- 6 m of DN160 mm
- 40 m of DN 125 mm

Large house

- heat recovery
- diameter: 75 mm
- length: 200 m

Large house

- Extract only
- Pressure at ATD: 70 Pa
- Area : 7,4 m²

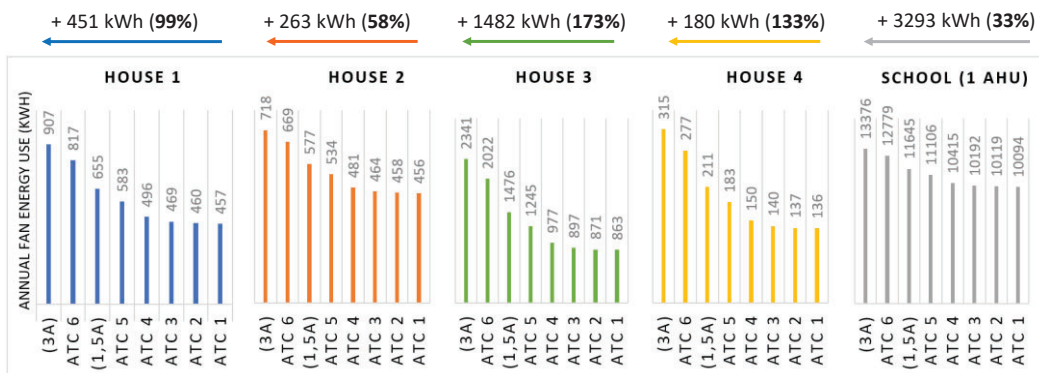
	House 1	House 2	House 3	House 4	School (1 AHU)	
	Exh./Sup.	Exh./Sup.	Exh./Sup.	Exh.	Exh.	Sup.
Fan flowrate (m ³ /h)	225	225	300	100	13500	11855
Fan pressure (Pa)	110	110	160	150	130	130
Pressure ATD (Pa)	10	10	10	70	43	35
Fan power (W)	26	26	49	15.4	4701	4143
Ductwork area (m ²)	14.72	9.36	23.6	7.4	317	279
Operating time (h/y)	8760				1140*	

* 2000 hours (10 h/d; 5 d/w; 40 w/y) x 57% due to CO₂ regulation (value from the Avis Technique of the ventilation system)

School : 1 of the 10 AHU of a 4500 m² French school (500 students) – PromevenTertiaire project

Annual energy use

Additional annual energy use due to ductwork leakage (Class 3A):



Annual and total cost of ductwork leakage

Airtight. classes		House 1		House 2		House 3		House 4		School (1 AHU)		
Prev.	New	Annual	Total	Annual	Total	Annual	Total	Annual	Total	Annual	Total	
	3A	-	126 €	10 117 €	73,5 €	5 881 €	415 €	33 199 €	50,4 €	4 035 €	922 €	73 772 €
	2,5A	ATC 6	101 €	8 095 €	59,7 €	4 772 €	326 €	26 071 €	39,8 €	3 182 €	755 €	60 415 €
	1,5A	-	55,9 €	4 469 €	33,9 €	2 713 €	173 €	13 834 €	21,3 €	1 706 €	437 €	34 999 €
	A	ATC 5	35,7 €	2 855 €	22,0 €	1 760 €	108 €	8 655 €	13,4 €	1 073 €	287 €	22 924 €
	B	ATC 4	11,2 €	898 €	7,1 €	565 €	33,1 €	2 645 €	4,1 €	331 €	93,3 €	7 463 €
	C	ATC 3	3,7 €	294 €	2,3 €	186 €	10,7 €	856 €	1,3 €	107 €	30,8 €	2 468 €
	D	ATC 2	1,22 €	97 €	0,77 €	62 €	3,53 €	282 €	0,44 €	36 €	10,26 €	820 €
	-	ATC 1	0,40 €	32 €	0,26 €	21 €	1,17 €	94 €	0,15 €	12 €	3,41 €	273 €

Conclusions

Simple tool developed to estimate the financial impact of leaky ductwork

- Help to raise awareness on the ductwork leakage issue
- Encourage the installation of airtight ductwork systems

Case studies for 4 houses and 1 AHU of a school:

Very leaky ductwork (Class 3A) induce:

- an increase of fan energy use ranging between **58%** and **173%** for the 4 single-family houses (180 to 1482 kWh) and of **33%** for the school AHU (3293 kWh).
- a financial impact for 80 years of operation ranging between **4.0 k€** to **33.2 k€** for the 4 single-family houses and reaching **74 k€** for the school AHU (out of the 10 AHU).

- **Significant financial impact for all type of buildings, including single-family houses**

Publication of the tool on Ubbink/International expected soon

- Send me **an email** to be informed: nolwenn.hurel@pleiaq.net



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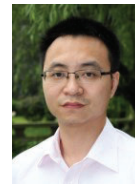
Qihong Deng,
Vice President, Practice



Dustin Poppendieck,
Secretary



James McGrath,
Treasurer



Weiwei Liu,
Treasurer Elect

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INTERNATIONAL SOCIETY OF INDOOR AIR QUALITY AND CLIMATE

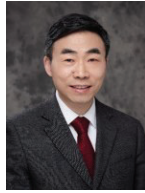
Non-voting Members



Corinne Mandin,
Past President



Yuguo Li,
ISIAQ Journal



Yingping Zhang,
President of the
Academy of Fellows



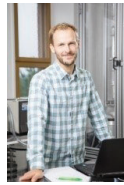
Richard Shaughnessy,
Coordinator of STCs



Lada Hensen,
Coordinator of Chapters



Rita Lam,
Student Representative



Martin Täubel,
Relations with
Other Societies



Tengfei Zhang,
President
HB2023-Asia



Marcel Schweiker,
President
HB2023-Europe



Nadia Boschi,
Trustee

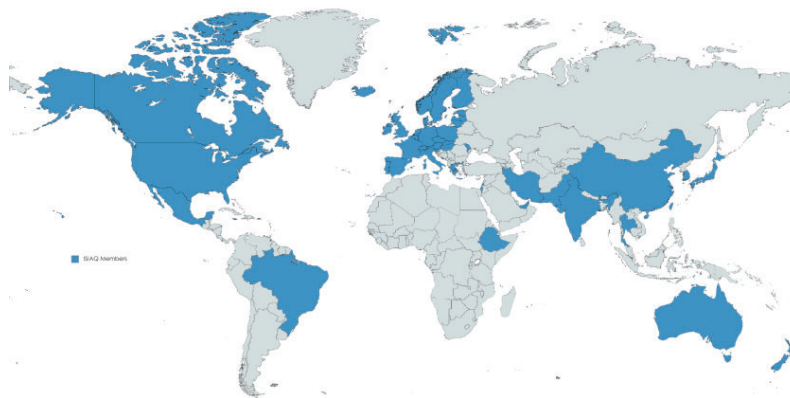


Chungyoon Chun,
Trustee



INTERNATIONAL SOCIETY OF INDOOR AIR QUALITY AND CLIMATE

50 countries represented and growing!



769
INDIVIDUAL MEMBERS

264
STUDENT MEMBERS

9
AFFILIATE MEMBERS

15
EARLY CAREER MEMBERS

15
CORPORATE MEMBERS



INTERNATIONAL SOCIETY OF INDOOR AIR QUALITY AND CLIMATE

Conferences

INDOOR AIR



Held in even-numbered years; rotation between Asia/Pacific Rim – Europe/Africa – America

HEALTHY BUILDINGS



Held in odd-numbered years; regional conferences; targeted to practice

5



INTERNATIONAL SOCIETY OF INDOOR AIR QUALITY AND CLIMATE

2023 Conferences



HEALTHY BUILDINGS EUROPE 2023
BEYOND DISCIPLINARY BOUNDARIES

June 11-14, 2023
Aachen, Germany



HEALTHY
BUILDINGS
ASIA 2023

July 16-19, 2023
Tianjin, China



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INTERNATIONAL SOCIETY OF INDOOR AIR QUALITY AND CLIMATE

2024 Conference



18TH CONFERENCE OF THE INTERNATIONAL SOCIETY OF INDOOR AIR QUALITY & CLIMATE

INDOOR AIR 2024

July 7-11, 2024 🌺 Honolulu, Hawaii, USA



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INTERNATIONAL SOCIETY OF INDOOR AIR QUALITY AND CLIMATE

ISIAQ Conferences



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INTERNATIONAL SOCIETY OF INDOOR AIR QUALITY AND CLIMATE

Scientific and Technical Committees (STCs)

Roles of STCs:

- **To review and report new and emerging research directions in indoor air sciences** to be published in the ISIAQ newsletter or submitted to the ISIAQ journal
- **To organize symposiums and workshops** during ISIAQ conferences
- All individuals are **welcome to request to join** any STCs.



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INTERNATIONAL SOCIETY OF INDOOR AIR QUALITY AND CLIMATE

9 Scientific and Technical Committees (STCs)

STC11 Source, monitoring and evaluation: chemical pollutants (55 members)

Chairperson: Ying Xu, Tsinghua University, China

STC12 Source, monitoring and evaluation: aerosols (20 members)

Chairperson: Chair: Brandon E. Boor, Purdue University, USA

STC13 Microbes in indoor environments (15 members)

Chairperson: Ju-Hyeong Park, NIOSH/CDC, USA

STC21 Ventilation (33 members)

Chairperson: Brent Stephens, Illinois Institute of Technology, USA

STC22 Air cleaning (36 members)

Chairperson: Alireza Afshari, Aalborg University, Denmark

STC31 Health effects and epidemiology (31 members)

Chairperson: Yuexia Sun, Tianjin University, China

STC32 Environmental/climate impacts (18 members)

Chairperson: Zhiwen Luo, California Department of Public Health, USA

STC33 Thermal comfort (25 members)

Chairperson: Yingxin Zhu, Tsinghua University, China

STC34 IEQ Guidelines (13 members)

Chairperson: Ulla Haverinen-Shaughnessy, The University of Oulu, Finland

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INTERNATIONAL SOCIETY OF INDOOR AIR QUALITY AND CLIMATE

Indoor Air Journal

- Wiley-owned *Indoor Air* journal had been the official journal of ISIAQ for 30 years.
- In late 2022, Wiley would transition *Indoor Air* to an Open Access journal as part of their new partnership with Hindawi.
- ISIAQ decided not to enter any association with Wiley-Hindawi publishing
- *Indoor Air* is no longer the “official journal” of ISIAQ.
- ISIAQ chose Elsevier publishing
- **A new ISIAQ-owned journal, *Indoor Environments*, will be launched this year**



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INTERNATIONAL SOCIETY OF INDOOR AIR QUALITY AND CLIMATE

Library of Specialized Webinars

ISIAQ Members have access to our library of specialized webinars. Recent webinars include:

- The Recognition of Airborne Transmission of Infectious Diseases
- Managing buildings in the era of COVID-19
- Health and contamination: New perspectives for the future
- A Modular Mechanistic Framework for Assessing Human Exposure to Chemicals in Materials, Products and Articles – Next Steps
- Cleaning for Health: Viruses / Biological Contamination on Surfaces and How to Meet the Challenges to Achieve Effective Cleaning
- Assessing Human Exposure to Chemicals in Materials, Products and Articles: A Modular Mechanistic Framework

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INTERNATIONAL SOCIETY OF INDOOR AIR QUALITY AND CLIMATE

Leading the Field

INTERNATIONAL SOCIETY OF INDOOR AIR QUALITY AND CLIMATE

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Jobs and Resources

Click here to view:
ISIAQ JOB BOARD

ISIAQ Members may submit job posts by logging in to the website. Under Member Menu, click on Job Post and select ISIAQ Job Board. Or email the ISIAQ Secretariat.

PROFESSIONAL RESOURCES

Airborne Transmission of COVID-19, Interview with Professor Morawska [Click here to view the interview](#)

Past President Lidia Morawska was selected by Time Magazine as one of the one hundred most influential people in the world in 2021. See our interview with her on our website.

Airborne Transmission of COVID-19, Interview with Professor Morawska

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INTERNATIONAL SOCIETY OF INDOOR AIR QUALITY AND CLIMATE

Promoting Knowledge with IEQ Global Alliance



ISIAQ and the Indoor Environmental Quality Global Alliance (IEQ-GA) have partnered to create a unique monthly show called **Indoor Environments: Global Research to Action**

This new video show & podcast explores at how research can be translated to practice on a variety of topics related to indoor environments.

Watch the replays on:

<https://global.healthyindoors.com/c/indoor-environments/>



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Student Activities

- **Mentorship programme:** ISIAQ brings together students and mentors in the IAQ field. All ISIAQ members are eligible to participate.
- **Student Conference Support Awards** are available to ISIAQ student members for each conference.
- The Indoor Air conference includes **Summer School** sessions preceding the conference, available to any students that wish to participate.



Student and Early Career Professionals Webinar Series

ISIAQ has launched a webinar series aimed at providing resources for student and early career professionals. Recent talks include:

- Transitioning from Academia to Industry
- Careers in Academia: How to land the perfect PhD and Post-doc position at your dream institution
- Research and Careers in the Indoor Environment: Learn how ISIAQ can help you



ISIAQ Chapters

5 national chapters:

FISIAQ (Finland)

ISIAQ.nl (the Netherlands)

SWESIAQ (Sweden)

NIO (Norway)

ISIAQ.is (Iceland)



FINNISH SOCIETY OF
INDOOR AIR QUALITY
AND CLIMATE



INTERNATIONAL SOCIETY OF
INDOOR AIR QUALITY
AND CLIMATE



NORSK INNEMILJØORGANISASJON

- Activities at national level
- Meetings (sharing experience) during ISIAQ conferences
- ISIAQ welcomes applications for new chapters!



Online Resources

- **Quarterly newsletter** with updates on ISIAQ conferences, chapters activities, webinars in replay, publication of new reports or PhD thesis.
- **IEQ Guidelines Database:** ISIAQ STC 34 has developed a database to share worldwide IEQ guidelines. The database is accessible through the new website, www.ieqguidelines.org. On the database page, you will have access to the entire database in a tabular format where you can search, sort, and filter by any fields available. On the map page, you will have access to the national guidelines by country on an interactive map.

INTERNATIONAL SOCIETY OF INDOOR AIR QUALITY AND CLIMATE

Online Resources: Job Board

INTERNATIONAL SOCIETY OF INDOOR AIR QUALITY AND CLIMATE

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Jobs and Resources

Click here to view:
[ISIAQ JOB BOARD](#)

ISIAQ Members may submit job posts by logging in to the website. Under Member Menu, click on Resources and select ISIAQ Job Board. Or email the ISIAQ Secretariat.

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INTERNATIONAL SOCIETY OF INDOOR AIR QUALITY AND CLIMATE

Opportunities for Membership

ISIAQ has several tiers of membership available:

- Ordinary Members - \$135
- Early Career Members - \$70
- Student Members - \$30
- Affiliate Members - \$30

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INTERNATIONAL SOCIETY OF INDOOR AIR QUALITY AND CLIMATE

Connect with ISIAQ

The screenshot shows the ISIAQ website header with the logo and title. Below the header, there are sections for 'Upcoming Events', 'Recent News & Announcements', and 'Tweets by @isiaq'. A tweet from Andrew Hoxington is visible. To the right, there is a social media profile for ISIAQ (@isiaq) with a banner that reads 'CELEBRATING 30 YEARS INTERNATIONAL SOCIETY OF INDOOR AIR QUALITY AND CLIMATE 1992-2022'. The profile also shows the text 'Official tweets by the International Society of Indoor Air Quality and climate (ISIAQ). isiaq.org' and 'Joined September 2012'.

Connect with us through our website, our social media, and LinkedIn



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INTERNATIONAL SOCIETY OF INDOOR AIR QUALITY AND CLIMATE

30th Anniversary Webinar Series

To celebrate ISIAQ's 30 years, we took the opportunity to consider what we know/what are the main milestones of the past research, and what are now the major challenges for the coming years.

ISIAQ organized a webinar series. Each webinar brought together key researchers who had a major contribution to indoor air sciences and who played an important role within our society, and young brilliant researchers who are preparing the future of research in our field.



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INTERNATIONAL SOCIETY OF INDOOR AIR QUALITY AND CLIMATE

Webinar 1. 30+ Years of Knowledge Creation: Indoor Air 1991-2021



William W. Nazaroff

Distinguished Professor Emeritus
University of California, Berkeley, USA



Marina Vance

Assistant professor
University of Colorado Boulder, USA

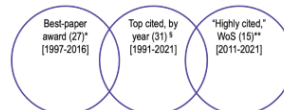
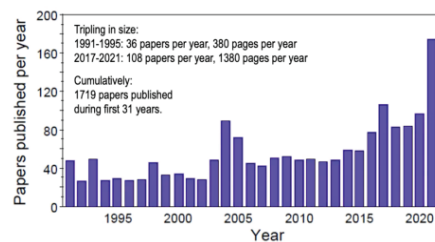
23



INTERNATIONAL SOCIETY OF INDOOR AIR QUALITY AND CLIMATE

Indoor Air — an international, multidisciplinary, research journal

- Established in 1991
- Aims and Scope: provide a healthy and comfortable environment for building occupants
- Steady growth in scale and influence
- **64 key papers — identify 7 major topics (1991-2021)**



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Four themes remain vigorous

- Dampness, mold, microbiomes, and health
- Indoor environmental quality in schools and student achievement
- Infectious disease transmission
- Residential air quality



New themes

- Climate change adaptation and mitigation in indoor environments
- IEQ improvements through advanced sensing and personalized controls
- Autoimmune, autoregulatory, and cognitive disorders as influenced by indoor exposures to endocrine disrupting chemicals

Five principles to achieve good IAQ

- 1) Minimize indoor emissions
- 2) Keep it dry
- 3) Ventilate well
- 4) Protect against outdoor pollution
- 5) Satisfy occupant needs



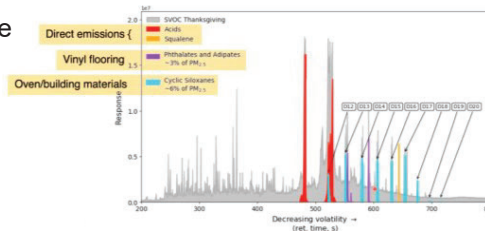
Examples of recent collaborative work related to three of Bill's five principles

The HOMEChem study: home observations of microbial and environmental chemistry

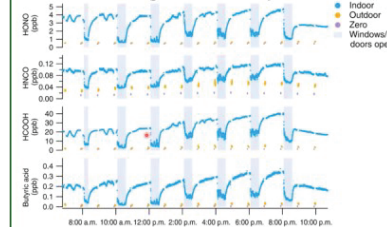
1. Minimize indoor emissions: particulate matter, SVOCs
2. Ventilate well
3. Protect against outdoor pollution



Particle-phase SVOCs enhanced during cooking



Indoor mixing ratios of some acids during ventilation experiments



Wang et al. (2020) Sci Adv



INTERNATIONAL SOCIETY OF INDOOR AIR QUALITY AND CLIMATE

Webinar 2. What We Know and What We Should Know About Indoor Environmental Quality



Pawel Wargocki

Associate Professor
Technical University of Denmark



Brandon Boor

Associate Professor
Purdue University, USA

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INTERNATIONAL SOCIETY OF INDOOR AIR QUALITY AND CLIMATE

IAQ and its contribution to health and comfort became a significant topic of investigation 30 years ago. Topics of initial research included:

- Formaldehyde/Radon
- Sick building syndrome
- Volatile organic compounds
- Allergies
- Thermal comfort
- Bioaerosols



Indoor air pollution is responsible for the deaths of **3.8 million** people annually



SOURCE: International Journal of Environmental Research and Public Health

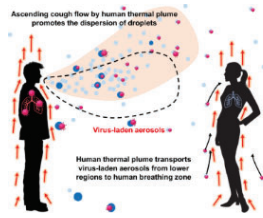


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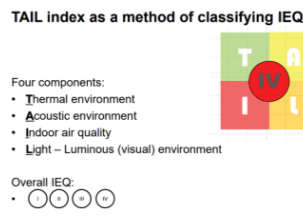


Areas of research to push the field forward:

- Identification of new and potential pollutants (toxicity, transport, level)
 - Human bio-effluents
 - Nano-size particles
 - Persistent organic pollutants
- Building resilience to climate change, low carbon requirement, pandemic
- Establishment of the consensus indices or metric for IEQ
- Focus on sensitive population, e.g., the elderly and infants
- Sleep IAQ and its relationship with health



(Sun et al., 2021)



(Wargocki et al., 2021)



Webinar 3. Dermal - the often-overlooked exposure route

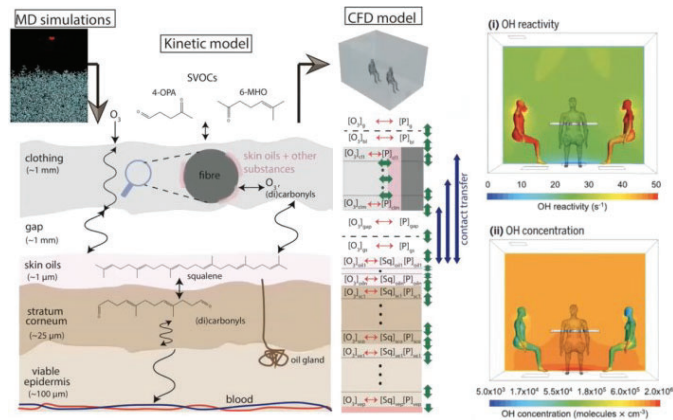


Charles Weschler
Adjunct Professor
Rutgers University, USA



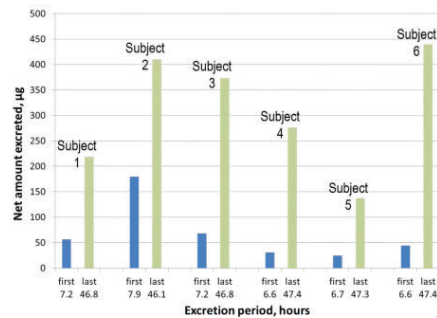
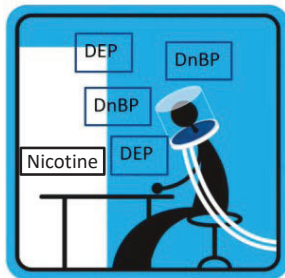
Manabu Shiraiwa
Professor
University of California, Irvine, USA

While traditionally ozone was thought to be the major oxidant indoors, recent studies have found that skin is a source of OH radicals indoors.



33

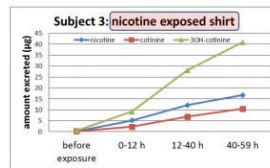
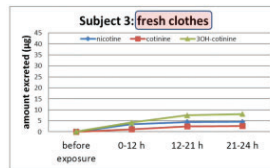
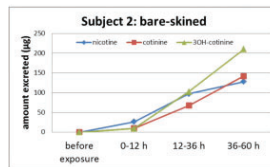
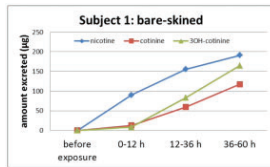
Experiments: dermal uptake from air



- Relatively large dermal uptake from air or clothing has been measured for DEP, DnBP, and nicotine etc.
- Skin is a reservoir – delivery continues after exposure ends

Weschler et al., 2015; Morrison et al., 2016

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- Modeling indicates that dermal uptake is substantial for some SVOCs (e.g., parabens, synthetic musks, chlorpyrifos, Texanol)
- Looking to the future, long-term measurements and big data sets are needed to connect indoor air pollutants to specific health effects

Bekö et al., 2017



Webinar 4. Winter is coming: challenges for indoor air sciences in times of energy crisis, pandemics & climate change



Tunga Salthammer
Professor
Fraunhofer WKI, Germany



Wenjuan Wei
Research Scientist
Scientific and Technical Centre for Building, France



Current challenges facing indoor air sciences include insufficient insulation, viral load in indoor environments, and the impact of climate change on indoor air quality and ventilation.

Potential solutions exist, such as mobile air cleaners, manual ventilation, and permanently installed fans to reduce risks of infection, heat, and pollutants. However, testing is needed to check for efficacy and safety from harmful byproducts.

Better communication channels are needed to share scientific results with the public.



Webinar 5. From Research to Practice: Past Successes and Remaining Gaps



Nadia Boschi

Director

Sustainability Italy & Continental
Europe Lendlease, Italy



James McGrath

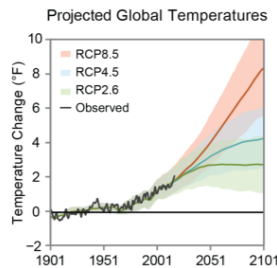
Assistant Professor

Maynooth University,
Ireland



Lifestyle trends of increasing urbanism, increasing population, aging demographics, increased stress, and a sedentary lifestyle all contribute to a considerable amount of time spent indoors.

Sustainability and climate change must be considered as strategies to promote indoor air quality.



Ban tobacco smoke in apt/condo buildings <small>(if allowed in some states)</small>	Compartmentalization to prevent odor or unwanted air transfer <small>(e.g. shop-vacuum air pollutants)</small>
Local exhaust	Eliminate lead and mitigate asbestos and radon
Switch to 100% renewable energy supply (H/C/E)	Retrofit favoring the installation of controlled mechanical ventilation systems
Install windows with trickle air vents	Install CO2 monitor



Communication of risks and opportunities is important – for example, a labeling system could allow consumers to assess health risks and make informed choices among building materials and consumer products used.





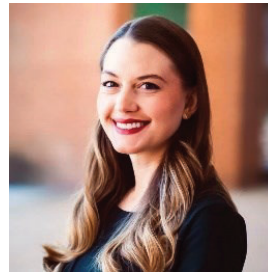
Webinar 6. The long history of airborne infection transmission: why don't we use the knowledge we have



Lidia Morawska

Professor

Queensland University of Technology, Australia



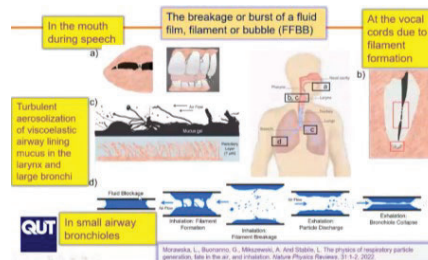
Kristen Coleman

Assistant Professor

University of Maryland School of Public Health, USA



We are a source of **respiratory particles**, which are **small** (the vast majority are $<10 \mu\text{m}$), **light** (can stay suspended in the air) and **may contain viruses**.



July 2020: It is time to address airborne transmission of COVID-19. 239 scientist from 34 countries



7 July 2020: "The World Health Organization acknowledged *"evidence emerging"* of the airborne spread of the novel coronavirus, after a group of scientists urged the global body to update its guidance on how the respiratory disease asses between people.."

<https://www.who.int/emergencies/diseases/novel-coronavirus-2019/guidance-covid19-airborne-transmission>



The closing webinar of the series discussed the importance of ventilation, the use of mechanical manipulation and germicidal UV air disinfection, and the need for regulations and guidelines to ensure indoor air quality standards are met.

There is considerable work on health and the aerosolization of human and animal pathogens at the animal interface, which could potentially lead to the rapid spread of novel viruses being done, which should continue to inform developments moving forward.



2024 Conference



18TH CONFERENCE OF THE INTERNATIONAL SOCIETY OF INDOOR AIR QUALITY & CLIMATE

INDOOR AIR 2024

July 7-11, 2024 🌺 Honolulu, Hawaii, USA





INTERNATIONAL SOCIETY OF INDOOR AIR QUALITY AND CLIMATE

Thank you for your attention!

<https://isiaq.org/>





University of
Nottingham
UK | CHINA | MALAYSIA

AIVC 2023 Smart Ventilation, IAQ, & Health

Benjamin Jones

1



University of
Nottingham
UK | CHINA | MALAYSIA

Recent AIVC publications

ANNEX 5




AIVC Technical Note 72
Ventilation Requirements and Rationale
behind. Standards and Regulations of
dwellings, office rooms and classrooms

April 2023

Main Authors

Wim de Gooijer, Netherlands
Wouter Bouwboom, TNO, Netherlands


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University of Nottingham
UK | CHINA | MALAYSIA

Annex 78 – Gas Phase Air Cleaners

[IEA EBC HOME](#) | [LINKS](#) | [SEARCH](#) | [SITE MAP](#) | [EBC-LOGIN](#)





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IEA EBC Annex 78 - Supplementing Ventilation with Gas-phase Air Cleaning, Implementation and Energy Implications

The proposed Annex should bring researchers and industry together to investigate the possible energy benefits by using gas phase air cleaners (partial substitute for ventilation) and establish procedures for improving indoor air quality or reduced amount of ventilation by gas phase air cleaning. The project shall also establish a test method for air cleaners that considers the influence on the perceived air quality and substances in the indoor air.

ANNEX INFO & CONTACT
Status: Ongoing (2018 - 2023)

OPERATING AGENTS

Bjarne Olesen
 Technical University of Denmark
 Kongens Lyngby
 DENMARK
[Email](#)

Dr. Pawel Wargocki
 Technical University of Denmark
 Kongens Lyngby
 DENMARK
[Email](#)

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University of Nottingham
UK | CHINA | MALAYSIA

The End

The End

6

SUMMARY OF AIRTIGHTNESS TRACKS

Valerie Leprince

October 5th, 2023



1

SUMMARY OF AIRTIGHTNESS TRACKS

Airtightness, still a growing concern around the world

Yes airtightness matters!

New update on regulations and programs

But how to evaluate it?

Blowerdoor test – benchmark solution but still a topic of research

New alternative methods

Predictive methods?

2

Airtightness, still a growing concern around the world

Summary of Airtightness tracks – AIVC 2023
Valérie Leprince

3

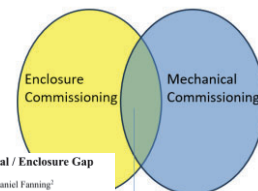
On the importance of airtightness...



Yes building and ductwork airtightness matters!

It impacts on:

- System sizing
- The quality of ventilation



Bridging The Mechanical / Enclosure Gap
David de Sola¹, Nathaniel Fanning²

¹ AIVE LLC
338 Windsor Street
Cambridge MA 02141 USA
Email: david.deSola@aive.com

² Filimonov & Tracy Associates, Inc.
300 Union Park Drive, 2nd Floor
Boston, MA 02118 USA
Email: ntfanning@fa.com

- Air Leakage
- Durability
- Candidate Evaluation
- Energy Efficiency

4

Impact of the building airtightness and natural driving forces on the operation of an exhaust ventilation system in social housing in Chile.

Gilles Flamant^{*1,2}, Waldo Bustamante¹, Arnold Janssens², and Jelle Laverge²

Poor building airtightness induces under-ventilated bedroom with exhaust only systems!

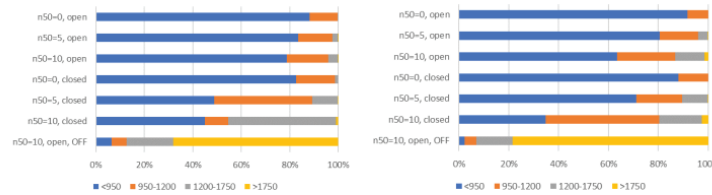
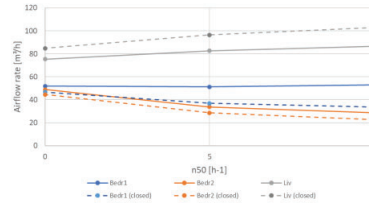


Figure 11: Percentage of time spent in four CO₂ concentration (ppm) classes for the father (left) and child (right).

5

Financial impact of leaky ductwork in buildings – a calculation tool to raise awareness

Nolwenn Hurel^{1*}, Valérie Leprince² and Marcus Lightfoot³

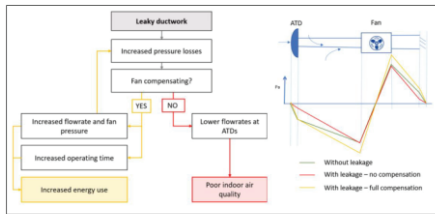


Figure 1- Impact of leaky ductwork

Simple calculation tool for building designers to evaluate the impact of ductwork airtightness on fan energy use for houses

- If the fans fully compensate for leakage, the annual energy use increase for a very leaky ductwork (3A) compared to an airtight one ranges between 58% and 173% for the 4 single-family houses, and is of 33% for the school AHU.

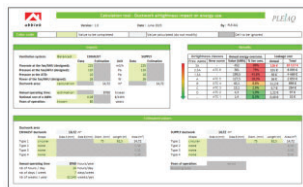


Figure 2 - Calculation tool (for the case study of house 1)

Table 4 - Annual and total (for 80 years of operation) cost of ductwork leakage according to the ductwork airtightness level for the 5 case studies

Airtight. classes	House 1		House 2		House 3		House 4		School (1 AHU)		
	Prev.	New	Annual	Total	Annual	Total	Annual	Total	Annual	Total	
3A	-	126 €	10 117 €	73,5 €	5 881 €	415 €	33 199 €	50,4 €	4 035 €	922 €	73 772 €
2,5A	ATC 6	101 €	8 095 €	59,7 €	4 772 €	326 €	26 071 €	39,8 €	3 182 €	755 €	60 413 €
1,5A	-	55,9 €	4 469 €	33,9 €	2 713 €	173 €	13 834 €	21,3 €	1 706 €	437 €	34 999 €
A	ATC 5	35,7 €	2 855 €	22,0 €	1 760 €	108 €	8 655 €	13,4 €	1 073 €	287 €	22 924 €
B	ATC 4	11,2 €	898 €	7,1 €	565 €	33,1 €	2 645 €	4,1 €	331 €	93,3 €	7 463 €
C	ATC 3	3,7 €	294 €	2,3 €	186 €	10,7 €	856 €	1,3 €	107 €	30,8 €	2 468 €
D	ATC 2	1,22 €	97 €	0,77 €	62 €	3,53 €	282 €	0,44 €	36 €	10,26 €	820 €
-	ATC 1	0,40 €	32 €	0,26 €	21 €	1,17 €	94 €	0,15 €	12 €	3,41 €	273 €

6

More publications expected (around 16 countries)

- Already **8 published papers** (on the AIVC website: <https://www.aivc.org/collection-keys/vip>)
 - Estonia (VIP 45.1)
 - Spain (VIP 45.2)
 - Czech Republic (VIP 45.3)
 - Belgium (VIP 45.4)
 - Latvia (VIP 45.5)
 - France (VIP 45.6)
 - Greece (VIP 45.7)
 - China (VIP 45.8)

Update on regulations on airtightness >

- **VIP Content**
For both **BUILDING** and **DUCTWORK** airtightness, it details :
 - **national requirements and drivers**: airtightness indicator, requirements in the regulation, energy programs, airtightness justifications, sanctions, etc.;
 - if it is included in the **energy calculations** and how;
 - the **airtightness test protocol**: qualification for the testers, guidelines, requirements on measuring devices;
 - **tests performed**: tested buildings/ductworks, database, evolution with time;
 - **guidelines** to build airtight buildings/ductworks.

Summary of Airtightness tracks – AIVC 2023
Valérie Leprince

7

KEY REQUIREMENTS IN COUNTRIES

In Norway

- “All new buildings shall be tested”
 - $n_{50} \leq 0.6$ /h for all dwellings
 - Less than 100% is probably measured in practice..

In Netherland

- Measurement according NEN 2686 (that mostly follow EN ISO 9972)
- The Building Decree demands based on building volume
 - $< 500 \text{ m}^3$: $qv_{10} = 0.2 \text{ m}^3/\text{s}$ (200 l/s) as qv_{10} value
 - $> 500 \text{ m}^3$: $qv_{10 \text{ kar}} = qv_{10} (500/Vb)$

Summary of Airtightness tracks – AIVC 2023
Valérie Leprince

8

KEY REQUIREMENTS IN COUNTRIES

In Spain



- Window permeability regulation since 1975
- For the envelope (only for dwellings >120m²)
 - n₅₀<6 (for V/A<=2) and n₅₀<3 for V/A>=4
- 2 options for justification: measurement or calculations

$$n_{50} = 0.629 \cdot \frac{C_0 \cdot A_0 + C_h \cdot A_h}{V}$$

In Latvia



- Mandatory airtightness requirements (in q₅₀) for all new buildings but no mandatory testing



In Montenegro



- Requirement
 - without mechanical ventilation ACH50=3 h⁻¹
 - with mechanical ventilation ACH50=1.5 h⁻¹
- No mandatory test
- Changing windows successfully divided by 9 the airtightness in a dwelling!!



How to evaluate airtightness?

Blowerdoor test according ISO 9972



Is it the perfect solution?

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ISO 9972: FAN PRESURIZATION METHOD

- Describes measurement procedure and calculation methods for determining airtightness
- To obtain comparable and credible results, it needs to be
 - **Reliable** and valid for different kinds of buildings
 - **Reproducible** under challenging environmental conditions
 - **Applicable** in any conditions
 - **Consistent** with other standards
- Recent scientific works + more experience in field testing → **need to improve parts of ISO 9972!**

- No regulations
- Airtightness recommended values
- Airtightness default values that can be improved
- Airtightness requirements
- Mandatory systematic testing



Poza-Casado et al. (2020)



12

REVISION OF ISO 9972



13

On the integration of envelope pressure inhomogeneity and autocorrelation in fan pressurization uncertainty analysis

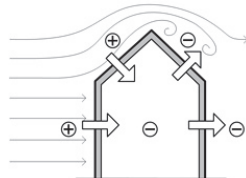
Martin Prignon¹

Repeatability to test the autocorrelation

Neither fully correlated nor fully uncorrelated
 Needs to be taken into account for uncertainty calculation

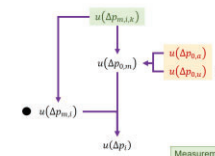
Inhomogeneity

Could be included as uncertainty term taking into account the maximum and the minimum value



Two novelties in the uncertainty estimation process

1. Test: measurement of $\Delta p_{m,i,k}$
2. Zero-flow pressure: $\Delta p_{fm} = \frac{\Delta p_{fm,1} + \Delta p_{fm,2}}{2}$
3. At each pressure station: $\Delta p_{m,i} = \frac{1}{N} \sum_{k=1}^N \Delta p_{m,i,k}$
4. $\Delta p_i = \Delta p_{m,i} - \Delta p_{fm}$



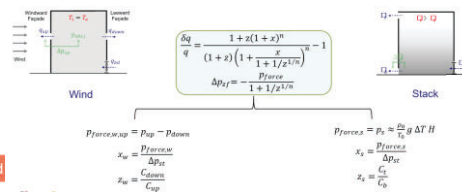
Measurement uncertainties
 Assumption uncertainties
 Propagation law

Objective: Having less uncertainty in the uncertainty calculation!

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FLOW ERROR ANALYSIS: ISO 9972 & ZERO-FLOW CONSTRAINTS

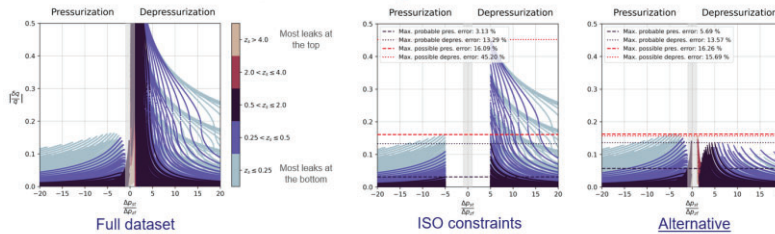
UNDERSTANDING ZERO-FLOW PRESSURE



Benedikt Kölsch, Valérie Leprince, Adeline Mélois, Bassam Moujalled

Alternative constraints to ISO 9972 ones exist to measure high-rise buildings!

Stack measurement position of Δp_0

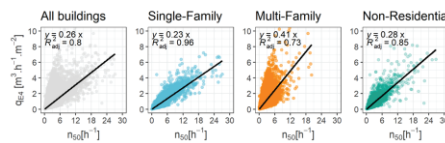


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Valérie Leprince

Statistical analysis of the correlations between buildings air permeability indicators

Bassam Moujalled^{1,2*}, Benedikt Kölsch^{1,3}, Adeline Mélois^{1,2}, Valérie Leprince³

Correlations between five indicators (specific air leakage rates $q_{E4} / q_{E10} / q_{E50}$ and air change rates n_{10} / n_{50}) have been calculated



• Correlations between q_{E4} and n_{50} depending on building type and building compactness

Correlation	Building type	Correlation depending on building type and building compactness			Correlation depending on building type		General correlation	
		Compact.	Reg. coef.	Conf. Int. 95%	Reg. coef.	Conf. Int. 95%	Reg. coef.	Conf. Int. 95%
$q_{E4} = \text{Coef} * n_{50}$	Single-Family houses	(0.07,0.78]	0.264 ($r^2=0.965$)	[0.264,0.265]	0.228 ($r^2=0.964$)	[0.228,0.228]	0.26 ($r^2=0.801$)	[0.26,0.261]
		(0.78,0.83]	0.23 ($r^2=0.983$)	[0.23,0.231]				
		(0.83,0.88]	0.219 ($r^2=0.984$)	[0.218,0.219]				
		(0.88,2]	0.2 ($r^2=0.977$)	[0.2,0.2]				
	Multi-Family apartments	(0.02,0.29]	0.91 ($r^2=0.929$)	[0.906,0.914]	0.409 ($r^2=0.728$)	[0.407,0.41]		
		(0.29,0.46]	0.517 ($r^2=0.958$)	[0.516,0.519]				
		(0.46,0.70]	0.33 ($r^2=0.966$)	[0.329,0.331]				
		(0.70,1.94]	0.237 ($r^2=0.964$)	[0.236,0.238]				
	Non-Residential buildings	(0.08,0.64]	0.418 ($r^2=0.9$)	[0.412,0.423]	0.284 ($r^2=0.851$)	[0.282,0.287]		
		(0.64,0.78]	0.276 ($r^2=0.971$)	[0.274,0.278]				
		(0.78,0.85]	0.233 ($r^2=0.973$)	[0.231,0.234]				
		(0.85,6.52]	0.192 ($r^2=0.936$)	[0.19,0.194]				



Summary of Airtightness tracks – AIVC 2023
Valérie Leprince

Correlation analysis between ACH50 and Air permeability considering the floor area of a residential buildings

Su-Ji Choi¹, and Jae-Hun Jo^{*2}

Classification of dwellings into 3 categories (Small, Medium and Large)

The average s/v ratio for small is 1.11, medium is 1.05, and large is 1.02, with ratios closer to 1 as the floor area increases

- More reasonable to evaluate the airtightness by air permeability (q_{E50}),
- smoother change in airtightness value
- consider the effect of leakage rate through the envelope

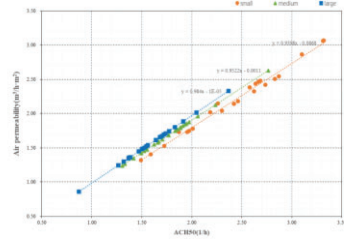


Figure 6: Correlation of ACH50 and air permeability

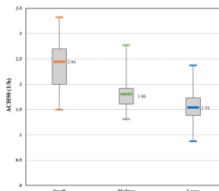


Figure 3: ACH50 by floor area

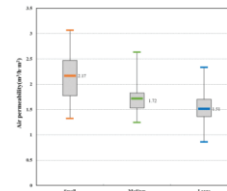


Figure 4: Air permeability by floor area



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Alternative methods



Are they reliable and implementable on site?



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Pulse tests in highly airtight Passivhaus standard buildings

Xiaofeng Zheng^{*1}, Luke Smith², and Christopher Wood¹



Complete 40L Pulse setup



Window mounted blower door

New setup adapted to very tight building: smaller tank to avoid saturation and leave a sufficient decay period

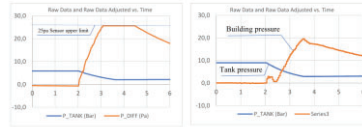


Figure 2: Example of unsuccessful pulse tests in highly airtight dwellings

On 11 passive houses results in line with BLD test

- Average difference between the 2 methods
 - 0.0003m³/h/m²@4Pa (11%)
 - 0.12 @50Pa

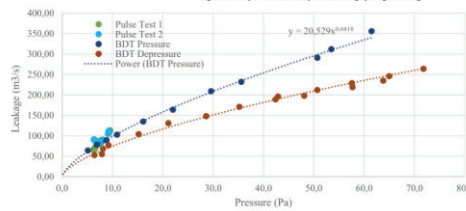


Figure 6: Property 004 blower door fan test power law extrapolation, with poor agreement between the pressurisation and depressurisation curves



Acoustic method for measurement of airtightness – field testing on three different existing office buildings in Germany

Björn Schiricke^{*1}, Benedikt Kölsch²



Figure 2: Measurement setup: pair of loudspeakers inside (left), microphone array or acoustic camera outside (centre), visualization of the loudest noise sources on the facade (right).

Do not necessitate large volumes of air movement through the building envelope,

- Test under naturally occurring unpressurized conditions.

Do not rely on closed volume

- Test during the construction or renovation

Table 3: Evaluation of acoustic signals for each measured room in Building E

Room	note	0.8	1.3	1.6	2	2.5	3.2	4	5	6.3	8	10	12.5	16	20	25	Multi Frequency Assessment Score
E-Büro 2	2.floor (DG)	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	14
E-Büro 1	2.floor (EG)	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	20
E-Büro	2.floor (EG)	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	5
E-Büro	1.floor (UG)	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	8
E-Bepr.	1.floor (UG)	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	9
E-Büro 2	1.floor (UG)	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	21
E-Büro 1	1.floor (UG)	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	15
E-Büro	2.floor (EG)	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	4
E-Aufenth.	2.floor (EG)	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	10
E-Büro	1.floor (UG)	○	○	○	○	○	○	○	○	○	○	○	○	○	○	○	13

Quantification of leakage

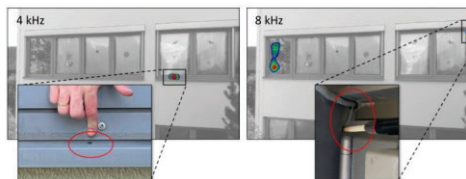


Figure 4: Examples of confirmed plausibility for air leakage as cause for sound peaks in room 106 (building D, east facade)



Predictive methods



Is it really possible to predict airtightness?



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Airtightness predictive model from measured data of residential buildings in Spain

Irene Poza-Casado¹, Pilar Rodríguez-del-Tío², Miguel Fernández-Temprano², Miguel Ángel Padilla-Marcos¹, and Alberto Meiss¹

Table 5: Equation of the final GLM predictive model for airtightness.

In Spain, maximum envelope permeability values but 2 solutions to justify it

- pressurization tests (rarely done)
- Use of reference values ... which were proved to be inaccurate

The predictive model developed

- explains 42.9% of the variability of the response
- contain 12 main effects and 2 interactions.

➢ Not to replace airtightness test

➢ But useful for decision-making process before building construction or retrofiting actions

$$Y = \beta_0 + \sum_{i=1}^p \beta_i X_i + \sum_{i < j} \tau_{ij} (X_i X_j) + \varepsilon$$

Parameter	Coefficient	Parameter	Coefficient
Intercept	0.273	Shutter position. P04	0a
Retrofitting state. Original	0.137**	False ceiling. FC0	-0.313***
Retrofitting state. Retrofitted	0a	False ceiling. FC1	-0.264***
Climate zone. A3	0.346**	False ceiling. FC2	0a
Climate zone. B4	0.545***	Typology. Multifamily	0.412**
Climate zone. C1	0.273	Typology. Single-family	0a
Climate zone. C2	0.630***	Heating system. No heating	0.074
Climate zone. C3	0.053	Heating system. Underfloor heating	-0.041
Climate zone. D2	0.575***	Heating system. Ducts	0.261***
Climate zone. u3	0a	Heating system. Other systems	0.173
Period of construction. Before 1980	-0.329***	Heating system. Heating units	0a
Period of construction. Since 1980	0a	Number of bathrooms. 0	0.610**
Window permeability. Class 0 or 1	0.596***	Number of bathrooms. 1	0.347***
Window permeability. Class 2	0.322***	Number of bathrooms. 2	0.183
Window permeability. Class 3	0.255***	Number of bathrooms. 3	0.090
Window permeability. Class 4	0a	Number of bathrooms. 4 or 5	0a
Window material. Steel	0.071	Share of windows	0.045***
Window material. Aluminium	0.074	Share of opaque envelope	0.003
Window material. Wood	0.298***	Period of construction. Before 1980 * Share of opaque envelope	0.010***
Window material. PVC	0a	Period of construction. After 1980 * Share of opaque envelope	0a
Shutter position. P01	0.195*	Typology. Multifamily * Share of opaque envelope	-0.009**
Shutter position. P02	0.144**	Typology. Single-family * Share of opaque envelope	0a
Shutter position. P03	-0.123		

a. This parameter is set to 0 as it corresponds to the reference class of the variable.

* stands for p-value ≤ 0.1, ** for p-value ≤ 0.05 and *** for p-value ≤ 0.01

where: Y is the response variable to be predicted, X_i with $i=1, \dots, p$ are the explanatory variables, β_i are the main effects of the explanatory variables on the response, τ_{ij} are the first-order interactions among variables X_i and X_j , and ε are the random independent homoscedastic normal perturbations. For the qualitative explanatory variables, the usual decomposition in dummy indicator variables has been considered.



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43rd AIVC 11th TightVent & 9th venticool Conference



Summing up Resilient ventilative cooling

Hilde Breesch
(KU Leuven, venticool)

INIVE



venticool
the platform for resilient ventilative cooling

TightVent Europe
the platform for resilient ventilative cooling



1

Overview resilient ventilative cooling track

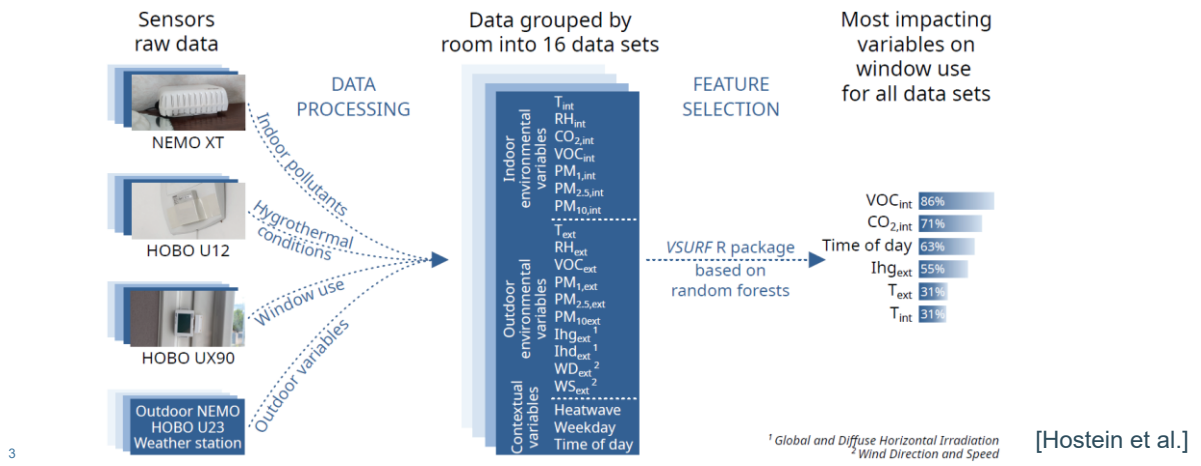
October 4th	October 5th
1C: Topical Session: Summer comfort and energy efficiency in hot periods: interest of mixed mode cooling and need of occupant feedback	5C: Topical Session: Importance of good resilient building design and standards to ensure good ventilative cooling performance to reduce overheating and environmental impact
2C: Climate change & Resilient cooling	6C: Ventilative cooling & Natural Ventilation
3C: Topical Session: Resilient Cooling of Buildings meets Resilient Cooling in Cities	7C: Topical session: Personalized Environmental Control Systems (PECS) operation and evaluation
4B: Ventilation strategies & thermal comfort	

2

2

Window opening behaviour

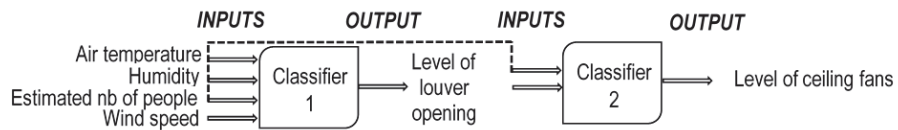
- Window use drivers
 - [Hostein et al.], [Tookey et al.], [Berger et al.]



3

Window opening behaviour

- [Payet et al.] Decision tree & Random forest model: windows and ceiling fans occupant behaviour

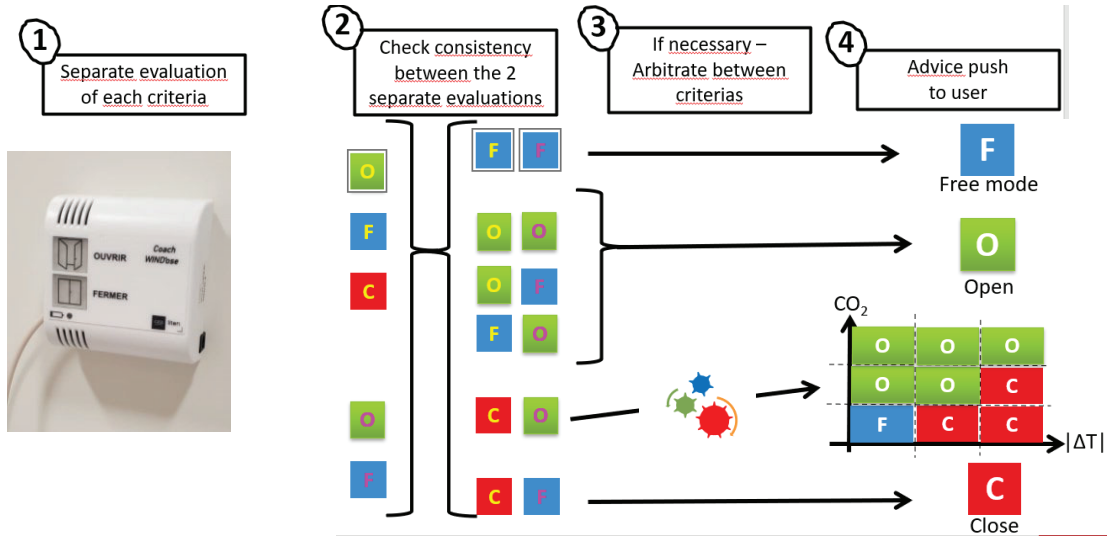


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4

Window opening behaviour

- [Jay et al.] Prototype coach to help occupants opening/closing windows

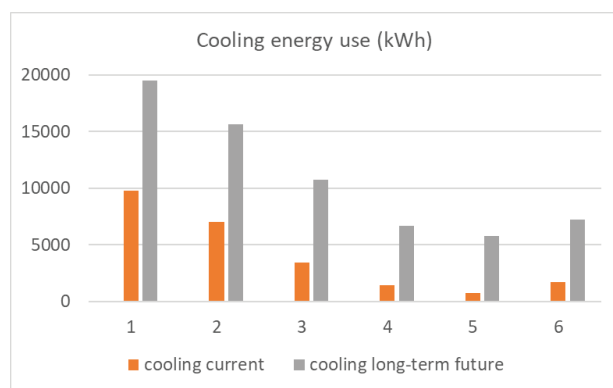


5

5

Assessment: use of future weather data

- Resilience [Sengupta et al.], [Khosravi et al.]
- Thermal comfort [Sengupta et al.], [Kolokotroni et al.], [Khosravi et al.]
- Energy use/demand [Kolokotroni et al.], [Khosravi et al.], [Romero-Lara et al.] [O'Donovan et al.]



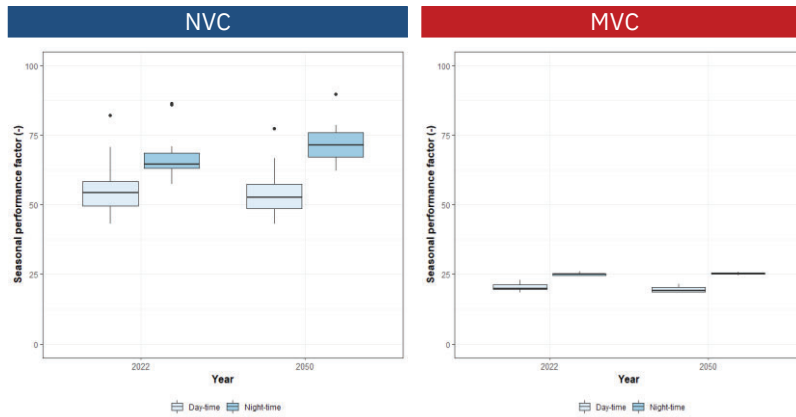
[Khosravi et al.]

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Assessment: use of future weather data

- SPF natural & mechanical ventilative cooling [O'Donovan et al.]

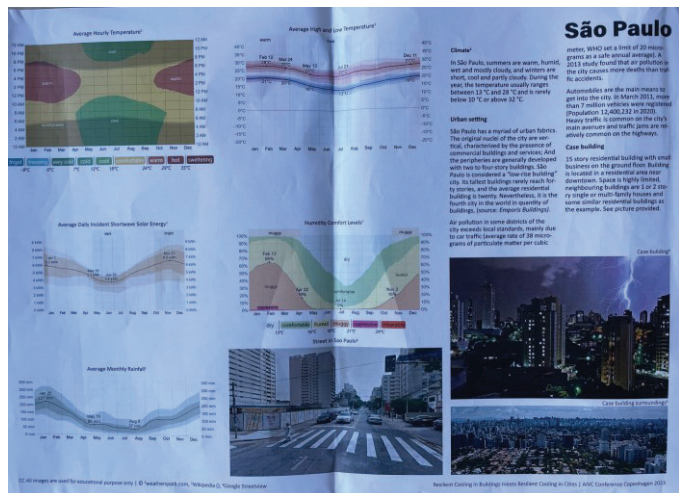


7

7

Design resilient cooling/ventilative cooling

- Resilient cooling design exercise
 - Climates
 - Sao Paulo
 - Los Angeles
 - Abu Dhabi
 - Measures
 - Building envelope
 - Interior & operation
 - Surroundings
 - Other: technical political, societal



8

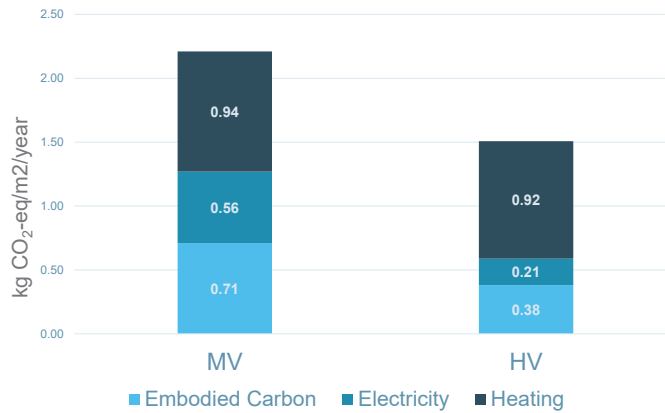
[Czarneci et al.]

8

Design resilient cooling/ventilative cooling

- LCA

- [Roth] example office building comparing CO₂-eq mechanical <> hybrid ventilative cooling



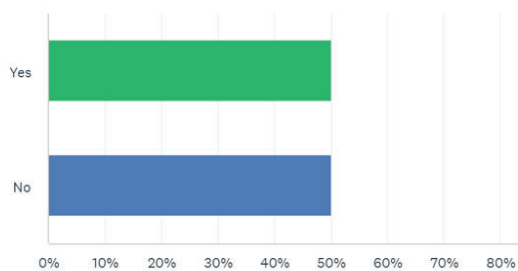
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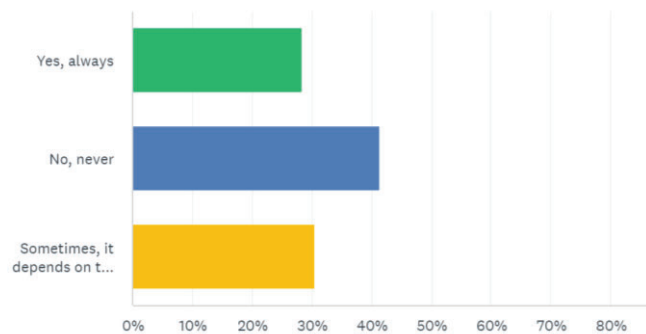
Design resilient cooling/ventilative cooling

- Design practice: survey design practitioners UK&IE [Sohail et al.]

- Are you familiar with term “ventilative cooling”?



- Do you use tools ventilative cooling design?

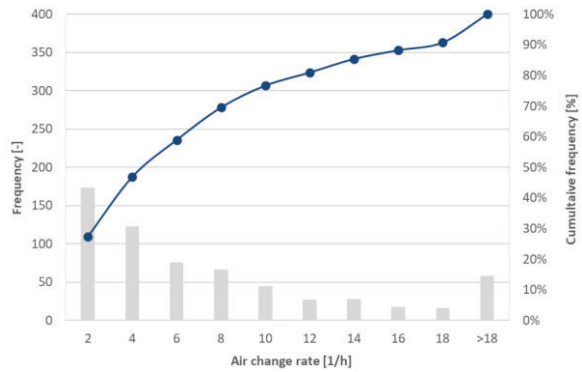
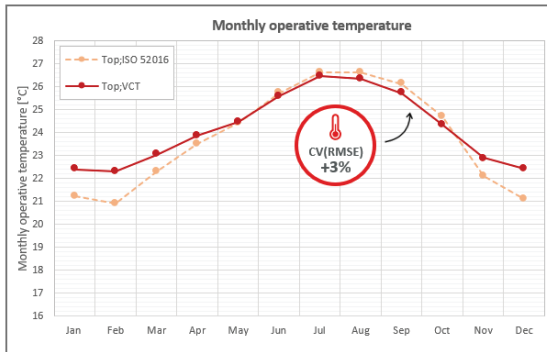


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10

Design resilient cooling/ventilative cooling

- Update early-stage design tool ventilative cooling [Radice Fossati, et al.]
 - Validated BESTEST heavyweight

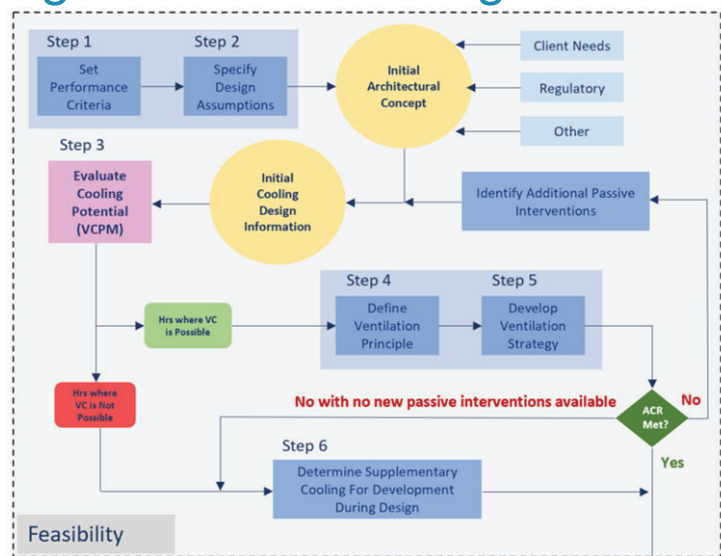


11

11

Design resilient cooling/ventilative cooling

- new CEN VC technical specification: framework (reasonably) easy assessment feasibility VC designs [Plessner], [O'Sullivan]



12

[O'Sullivan]

12



43rd AIVC
 11th TightVent
 & 9th venticool
 Conference

Ventilation, IEQ and health
 in sustainable buildings

October
4-5

Aalborg
 University
 Copenhagen

CONFERENCE ORGANIZERS

INIVE

AIVC

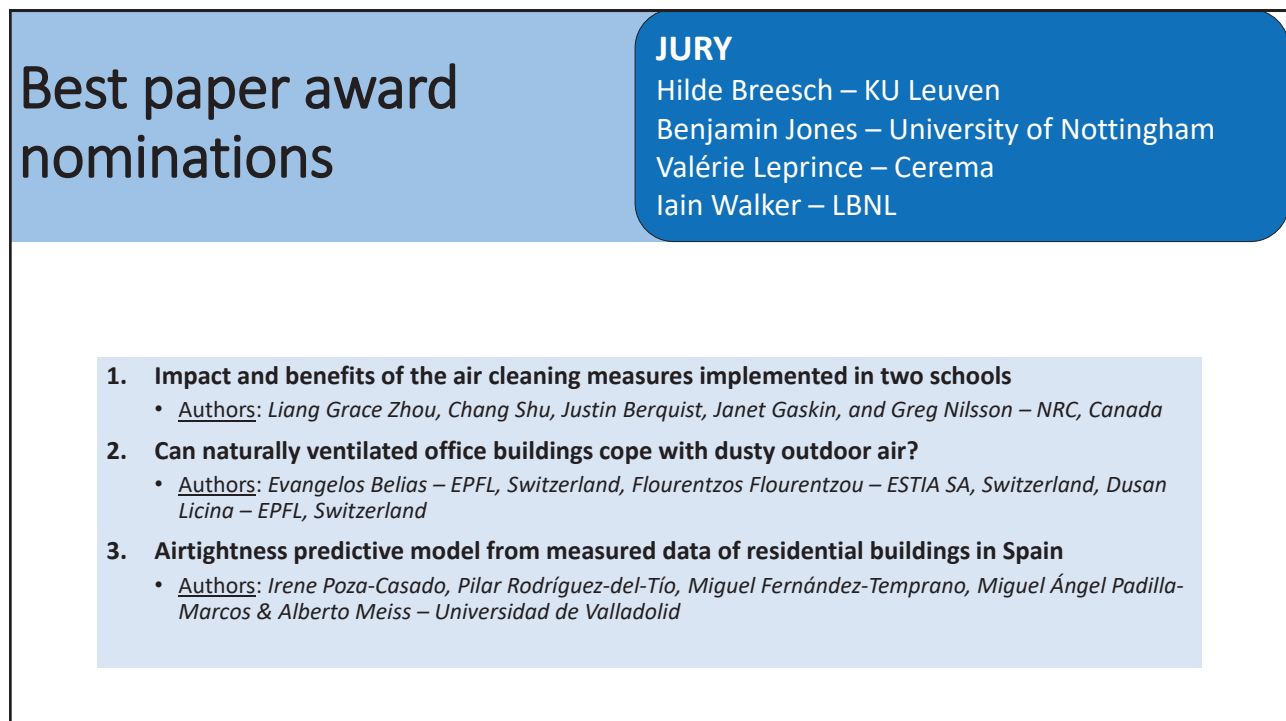
venticool
the platform for resilient ventilable cooling

TightVent
 Europe

DEPARTMENT OF THE BUILT ENVIRONMENT
 AALBORG UNIVERSITY

Best paper/poster award &
 Student Competition awards

1



Best paper award nominations

JURY
 Hilde Breesch – KU Leuven
 Benjamin Jones – University of Nottingham
 Valérie Leprince – Cerema
 Iain Walker – LBNL

- Impact and benefits of the air cleaning measures implemented in two schools**
 - Authors: Liang Grace Zhou, Chang Shu, Justin Berquist, Janet Gaskin, and Greg Nilsson – NRC, Canada
- Can naturally ventilated office buildings cope with dusty outdoor air?**
 - Authors: Evangelos Belias – EPFL, Switzerland, Flourentzos Flourentzou – ESTIA SA, Switzerland, Dusan Licina – EPFL, Switzerland
- Airtightness predictive model from measured data of residential buildings in Spain**
 - Authors: Irene Poza-Casado, Pilar Rodríguez-del-Tío, Miguel Fernández-Temprano, Miguel Ángel Padilla-Marcos & Alberto Meiss – Universidad de Valladolid

2

Best paper award

Can naturally ventilated office buildings cope with dusty outdoor air?

Authors: *Evangelos Belias – EPFL, Switzerland, Flourentzos Flourentzou – ESTIA SA, Switzerland, Dusan Licina – EPFL, Switzerland*

Can naturally ventilated office buildings cope with dusty outdoor air?

Evangelos Belias^{1,2}, Flourentzos Flourentzou², Dusan Licina¹

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² ESTIA SA, EPFL Innovation Park, 1015 Lausanne, Switzerland
 *Corresponding author: evangelos.belias@epfl.ch

ABSTRACT

Naturally ventilated (NV) buildings, when well designed and operated, can provide adequate indoor environmental quality (IEQ) while reducing the building energy demand. However, in dusty outdoor air, this ventilation technique may increase the penetration of outdoor particulate matter (PM) indoors, leading to adverse health effects. Given the increasing frequency of outdoor dust episodes in Mediterranean climates, an important research question is whether NV buildings can provide adequate indoor air quality (IAQ) during increased outdoor air dust episodes. We monitored indoor and outdoor concentrations of size-resolved PM for six months in an occupant-operated NV low-energy office building in Cyprus, an island with frequent episodes of airborne dust. In parallel, the building was monitored for its energy consumption, indoor air temperature, relative humidity, and CO₂ concentrations. We also interviewed the building occupants regarding their perceived IEQ conditions. The results revealed that the NV provided adequate IAQ conditions in 4 out of 5 investigated indoor spaces for PM_{2.5} and in 2 out of 5 investigated spaces for PM₁₀. The average indoor concentrations were in the range of 4.4-5.1 µg/m³ for PM_{2.5} and 13.8-19.9 µg/m³ for PM₁₀, while the average outdoor concentrations for the same period were 7.4 µg/m³ for PM_{2.5} and 38.1 µg/m³ for PM₁₀. Additionally, unlike the outdoor air, the indoor PM concentrations respected the WHO short-term 24-hour limits, indicating that the building addressed well the dusty days. In terms of other IEQ parameters, the CO₂ levels remained below 1000 ppm for more than 90% of the time, while more than 90% of the occupants were satisfied with the thermal comfort conditions. The final actual energy consumption was -164 kWh/m²/y, drafting only by 7% from the predicted energy use. The results of this case study indicated that well-designed low-energy NV office buildings can provide adequate IEQ conditions, even in outdoor environments with dusty air.

KEYWORDS

passive technologies, office buildings, pollution penetration, I/O ratio, climate change

1 INTRODUCTION

Energy-efficient buildings are necessary to limit the energy demand and reduce greenhouse gas emissions globally. Natural ventilation (NV), when adequately designed and operated, can contribute to reducing the operational and grey energy demand in buildings while, in parallel, improving thermal comfort and indoor air quality (IAQ) (Flourentzos et al., 2017). Nevertheless, several studies criticize naturally ventilated buildings for not providing adequate protection to their occupants concerning outdoor air pollution, as the higher ventilation rates

3

Best poster award

JURY

Jaap Hogeling – REHVA
 Max Sherman – Nottingham University

Energy implications of increased ventilation in commercial buildings to mitigate airborne pathogen transmission

Authors: *Sean M. O'Brien, David Artigas, Ece Alan – Simpson Gumpertz & Heger Inc., New York, USA*

Energy Implications of Increased Ventilation in Commercial Buildings to Mitigate Airborne Pathogen Transmission



Sean M. O'Brien, P.E., David Artigas, P.E., & Ece Alan
 Simpson Gumpertz & Heger Inc., New York, NY USA

Objective

Analyze the energy costs due to and feasibility of increased mechanical filtration and ventilation to mitigate airborne pathogen spread in commercial buildings

Method

- Whole-building energy modeling using eQuest v. 3.65
- 2-story case and 22-story case
- New York, NY (heating climate) and Miami, FL (cooling climate)
- IECC minimum ventilation rate, up to 2x IECC rate, and 5 ACH (US CDC recommended rate)

Purpose

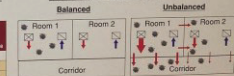
- Increased ventilation is a common recommendation to reduce airborne pathogen spread
- Limited evidence of effectiveness
- Heating and cooling are major contributors to building energy use
- Little thought given to environmental concerns with this mitigation strategy

Results: Increased Filtration

Impact on Fan Power and Building EUI

Filtration Level	System Fan Power (kW)	% Increase	Building Energy Demand (kWh)	% Increase
None	3.1	0%	200	0%
MERV 13-16	5.9	87.1%	192	4.3%

Unbalanced System



Results: Increased Ventilation

Impact on Energy and Building EUI

Low Rise, New York	Heating Energy (kWh)	% Increase	Cooling Energy (kWh)	% Increase	Energy Demand (kWh)	% Increase
2.38 IECC	48.6	-	32.2	-	28.7	-
4.72 (2x IECC)	36.8	-26.3%	32.2	0%	32.2	11.2%
9.44 (4x IECC)	28.8	-40.7%	34.7	7.2%	34.4	19.0%
High Rise, New York						
2.38 IECC	322	-	378.5	-	219.7	-
4.72 (2x IECC)	426.7	31.6%	422.8	11.2%	288.8	31.3%
Low Rise, Miami						
2.38 IECC	0	-	84.1	-	236.7	-
4.72 (2x IECC)	0	-	86.3	2.8%	251.7	6.8%
High Rise, Miami						
2.38 IECC	0	-	822.4	-	319.8	-
4.72 (2x IECC)	0	-	822.1	-0.03%	342.9	7.9%

Conclusions

- Feasible increase in ventilation results in significant increases in heating and cooling loads, but not overall EUI
- CDC-recommended ventilation rates likely are not feasible without replacing mechanical systems and significantly increase EUI
- Previous research indicates that increased ventilation has a short term and limited effect on infection spread, at best
- Increased ventilation requires rebalancing the system, otherwise it could lead to increased pathogen spread

4

Student Competition award

JURY

Alireza Afshari – Aalborg University
Maria Kolokotroni – Brunel University

Method for Evaluating an Air-Conditioning System with Natural Ventilation by Coupled Analysis of a Building Energy Simulation Tool and Computational Fluid Dynamics

Authors: *Ryuichi Yasunaga & Yasuyuki Shiraishi – The University of Kitakyushu, Japan*

Long-term energy performance of dew-point indirect evaporative cooler under the climate change world scenario

Authors: *María Jesús Romero-Lara, Francisco Comino & Manuel Ruiz de Adana – University of Cordoba, Spain*

Method for Evaluating an Air-Conditioning System with Natural Ventilation by Coupled Analysis of a Building Energy Simulation Tool and Computational Fluid Dynamics

Ryuichi Yasunaga¹, Yasuyuki Shiraishi²
^{1,2} Kitakyushu, Fukuoka, Japan

ABSTRACT

In office buildings, air-conditioning systems with natural ventilation can reduce cooling loads and create a comfortable indoor environment. However, it is difficult to predict the performance of such systems and thermal energy flows for the entire building, not only in normal indoor thermal environments. In this paper, we propose a method for analyzing the performance of natural ventilation air-conditioning systems by coupling a building energy simulation tool and computational fluid dynamics. In addition, we analyze the office building in which the system was installed and verified the prediction accuracy of the proposed method by comparing the simulation results with actual measurements. It was confirmed that the proposed method for natural ventilation.

KEYWORDS

Natural ventilation, Office building, Building energy simulation tool, CFD, Coupled analysis

1 INTRODUCTION

In recent years, air-conditioning (AC) systems that incorporate natural ventilation to reduce cooling loads and achieve a comfortable indoor environment have attracted increasing attention. However, because the amount of natural ventilation is greatly affected by weather conditions, it is difficult to predict quantitatively, and there is a concern that the natural ventilation will create an indoor thermal environment. In addition, no unified performance evaluation method for such systems has been established. Building energy simulation (BES) tools have attracted attention as a comprehensive analysis tool for evaluating the performance of buildings. However, such tools represent the physical quantity of the system as a single node and cannot consider the non-homogeneity of the indoor environment.

In this paper, we propose a method for evaluating the performance of an AC system by the integration of the indoor environment by coupling the BES tool (Building Energy Simulation Tool) and CFD. In addition, we analyzed a real office building as was installed and verified the prediction accuracy of the proposed method simulation results with actual measurements.

CASE STUDY

Using a typical office building, the indoor air is introduced through ceiling air conditioning units. These units are installed at the ceiling and outside air is introduced through a shaft in the center of the workplace and exits are not installed and verified the prediction accuracy of the proposed method simulation results with actual measurements.

Long-term energy performance of dew-point indirect evaporative cooler under the climate change world scenario

María Jesús Romero-Lara¹, Francisco Comino², and Manuel Ruiz de Adana³
¹Departamento de Ingeniería Técnica, Universidad de Córdoba, Córdoba, España
²Departamento de Ingeniería Técnica, Universidad de Córdoba, Córdoba, España
³Departamento de Ingeniería Técnica, Universidad de Córdoba, Córdoba, España

ABSTRACT

The projections estimate that the global average surface air temperature will increase by 1.5 to 2.0 °C by the end of the century. This increase in temperature will have a significant impact on the energy performance of buildings. In this research, the long-term energy performance of a dew-point indirect evaporative cooler (DEIC) was analyzed under the climate change world scenario. The results show that the DEIC can reduce the cooling load and energy consumption of a building. In addition, the DEIC can reduce the cooling load and energy consumption of a building. The results show that the DEIC can reduce the cooling load and energy consumption of a building. The results show that the DEIC can reduce the cooling load and energy consumption of a building.

KEYWORDS

Climate change, Long-term energy performance, Building energy simulation tool, Dew-point indirect evaporative cooler, Climate change world scenario

1 INTRODUCTION

Climate change has several implications for both human health and the energy performance of buildings (Comino et al., 2020; Pomeroy et al., 2020). Research papers have highlighted the significant contribution of buildings to global energy consumption and carbon emissions (Comino et al., 2020). Buildings are responsible for approximately 40% of the global final energy and are responsible for nearly 40% of total CO₂ emissions (Comino et al., 2020). The impact of climate change on the energy performance of buildings is a significant concern (Comino et al., 2020). The impact of climate change on the energy performance of buildings is a significant concern (Comino et al., 2020). The impact of climate change on the energy performance of buildings is a significant concern (Comino et al., 2020).



44th AIVC, 12th TightVent & 10th Venticool Conference
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'Retrofitting the Building Stock for Good Indoor Air Quality'

1



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2



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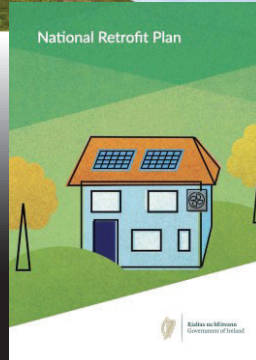
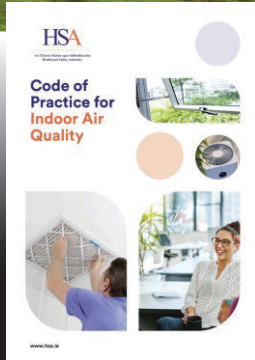
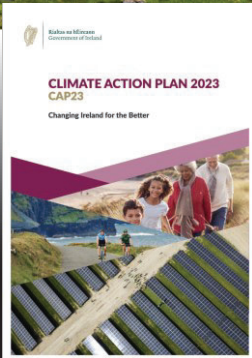
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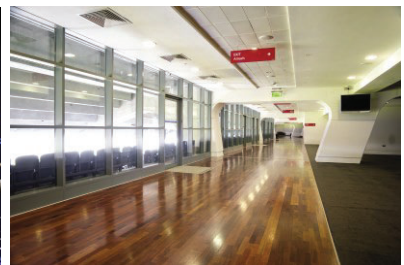
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5

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7

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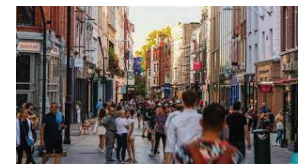
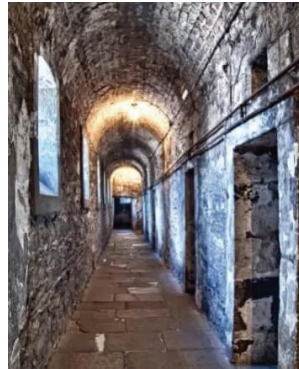


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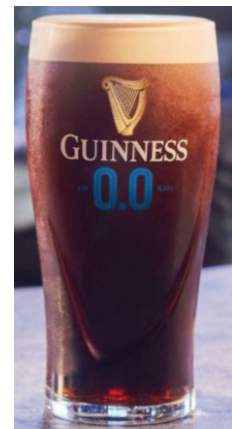
- Scientific Programme
- Culture Experience



9

AIVC 2024

- Scientific Programme
- Culture Experience
- A REAL pint of Guinness




10

AIVC 2024

- Scientific Programme
- Culture Experience
- A REAL pint of Guinness
- The 'Craic'



11



**Feicfidh mé thú i mBaile
Átha Cliath
(See you in Dublin!)**

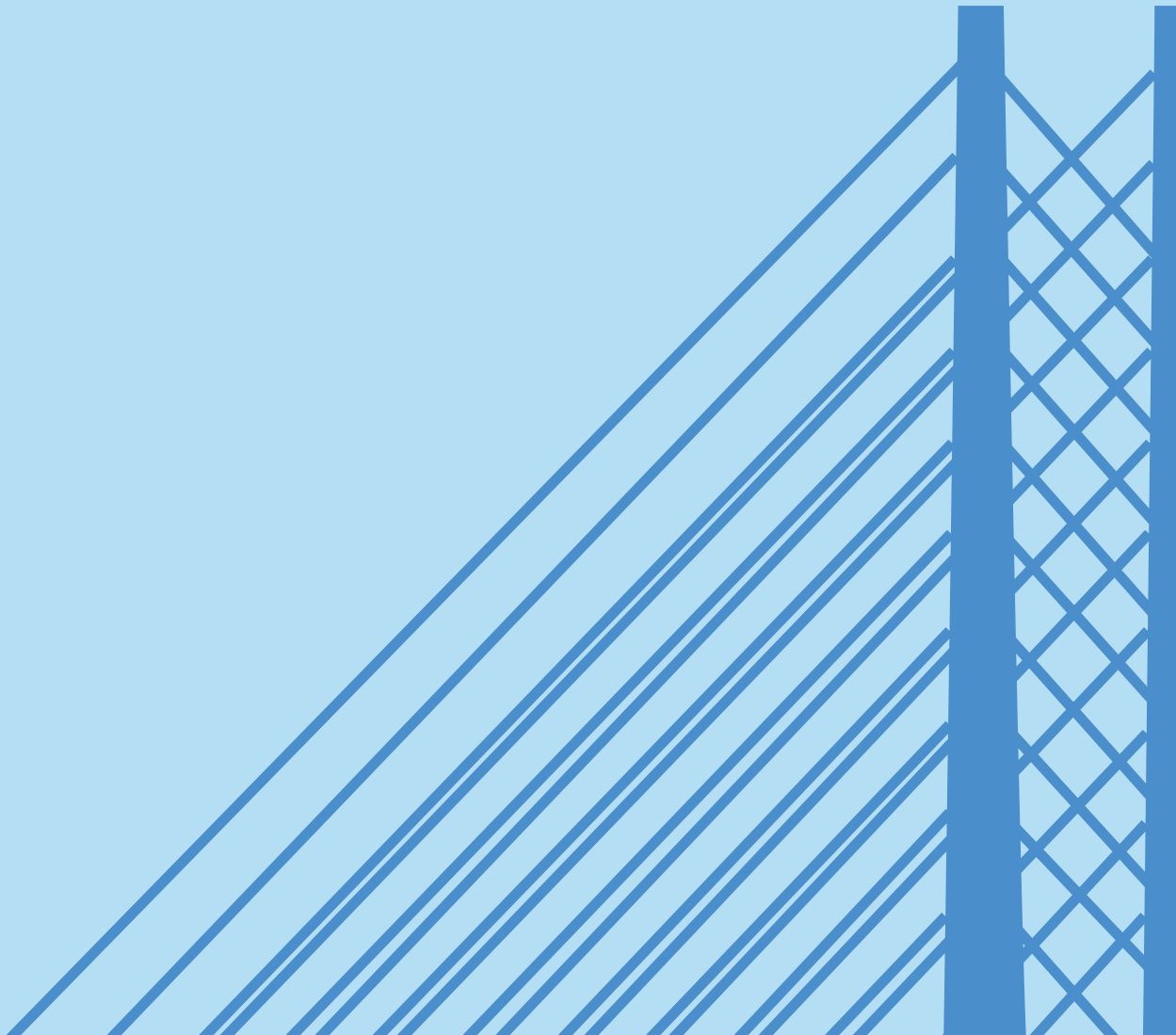
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